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(54) **FLUID FLOW NOZZLE**

FLÜSSIGKEITSSTROMDÜSE

BUSE D'ÉCOULEMENT DE FLUIDE

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Description

Reference to Related Application and Priority Claim

[0001] This application is a utility filing of and claims priority to co-pending provisional application No. 61/816,596, filed on April 26, 2013.

Background

[0002] The present disclosure relates to fluid flow nozzles, and particularly to nozzles for use in accelerating water flow.

[0003] Fluid flow devices such as hose or wand attachments are well-known. Many such attachments are provided to accelerate the fluid or water flow from the hose or wand for various tasks. The desirable flow velocity usually depends on the nature of the task, for instance lawn watering versus power washing. In the former case a wider lower velocity flow pattern is desirable while in the latter case a high velocity narrower flow pattern is preferred.

[0004] It is known from basic physics that the velocity of fluid flow through a nozzle increases as the inner diameter decreases. Thus, nozzles by necessity include an inlet having a larger inner diameter than the outlet. How this diameter change is accomplished varies among fluid flow devices. Some devices utilize a stepped down diameter outlet bore but this approach leads to significant fluid resistance and reduced flow volume. Consequently, most devices provide a tapered bore that tapers from the larger inlet diameter to the smaller outlet diameter. Other devices utilize a spherical bore from the larger inlet to the smaller outlet diameter.

[0005] One typical problem is that at higher flow velocities the fluid flow can be more turbulent or may tend to diverge. Both problems are counter to the straight powerful flow streams that are desired for power spraying tasks, such as power washing. Consequently, there is a need for a fluid flow nozzle that can achieve high flow velocities while reducing turbulence and divergence of the fluid stream.

[0006] WO 99/02271 discloses an infusion nozzle comprising a body having an internal wall whicj forms a flow passage which extends between an entry port into, and an exit port from, said flow passage. The wall is circularly-sectioned add extends along a central axis between the ports, the internal wall including an entry portion extending from the entry port and a constricting portion extending from its junction with the entry portion to the exit portion. The constricting portion decreases in diameter from the junction to the exit port. A plurality of vanes extend from the junction into the entry port and into the constricting portion, the vanes being equally spaced around the central axis, and including a deflection face facing toward the end port and a crest rising at a crest angle to the central axis toward the central axis and fairing into the constricting portion at a point spaced from the exit port. The vanes are disposed at a deflection angle to a plane that contains the central axis and the intersection of the respective vane with the junction.

Summary of the Disclosure

[0007] A fluid flow nozzle is provided that is configured to increase flow velocity while reducing turbulence and divergence of the discharge stream. In one aspect the fluid flow nozzle includes an elongated body having an inlet end and an outlet end, the inlet end configured for engagement therethrough from the inlet end to the outlet end; the channel including an inlet channel adjacent the inlet end and an outlet channel adjacent the outlet end, the inlet channel defined at an inlet diameter and the outlet channel defined at an outlet diameter that is less than the inlet diameter; the channel further defining a tapered channel extending from the inlet channel to the outlet channel and having a length between the inlet and outlet channels; and the elongated body further defining a plurality of vanes circumferentially spaced around the tapered channel and extending along at least a portion of the length of the tapered channel, the fluid flow nozzle being characterised in that the plurality of vanes are circumferentially spaced around the tapered channel in a substantially spiral shape following a radius that is approximately equal to the length between the inlet and outlet channels to impart a rotational momentum on fluid flowing through the channel. The vanes help ensure linear flow to reduce divergence of the discharge stream. Curved vanes impart a rotational movement to the fluid.

[0008] The fluid flow nozzle may include a series of stages from the inlet to the outlet to sequentially increase the flow velocity without increasing turbulence or divergence of the discharge stream. In such embodiment, two stages have a constant diameter whilst three stages step down the diameter between the constant diameter stages.

[0009] A selectable orifice attachment may be provided that allows the user to select among a plurality of orifice shapes and sizes. The attachment may be mounter to the discharge nozzles to further alter the discharge stream as desired by the user.

Description of the Figures

[0010]

FIG. 1 is a perspective view of a fluid flow nozzle which is not part of the present invention.

FIGS. 2(a)-(d) are engineering views of the nozzle shown in **FIG. 1** including a top, outlet end, inlet end and cross-sectional views.

FIG. 3 is perspective partial cut-away view of a fluid flow nozzle according to an aspect of the present disclosure.

FIG. 4 is an end view of the fluid flow nozzle shown in **FIG. 3**.

FIG. 5 is perspective partial cut-away view of a fluid flow nozzle which is not part of the present invention.

FIG. 6 is a side cross-sectional view of a fluid flow nozzle which is not part of the present invention.

FIG. 7 is an end view of a selectable outlet opening attachment for engagement to a fluid flow nozzle in one feature of the present disclosure.

FIG. 8 is an enlarged view of the outlet end of the fluid flow nozzle shown in **FIG. 6** with the selectable outlet opening attachment shown in **FIG. 7** mounted thereto.

Detailed Description

[0011] For the purposes of promoting an understanding of the principles of the invention, reference will now be made to the embodiments illustrated in the drawings and described in the following written specification. It is understood that no limitation to the scope of the invention is thereby intended. It is further understood that the present invention includes any alterations and modifications to the illustrated embodiments falling within the scope of the appended claims and includes further applications of the principles of the invention as would normally occur to one skilled in the art to which this invention pertains.

[0012] A fluid flow nozzle **10** includes an inlet end **11** that may be threaded for engagement to a garden hose, wand or other fixture, an elongated body **12** and an outlet end **13**, as shown in **FIG. 1** and **FIGS. 2(a)-(d).** The nozzle is hollow from the inlet end to the outlet end, defining an inlet channel **15**, followed by a first tapered channel **16,** a second tapered channel **17** and an outlet channel **18**. The inlet and outlet channels **15, 18,** respectively, have generally constant diameters, with the inlet having a greater diameter than the outlet. In one specific embodiment, the inlet channel may have a diameter of about 19.3mm and the outlet channel may have a diameter of about 4.7mm, for an approximate 4 to 1 reduction in diameter. Since the fluid flow rate is proportional to the square of the diameter, this reduction leads to an approximate 16 fold increase in flow velocity from the inlet to the outlet.

[0013] The first and second tapered channels **16, 17** are contiguous and are tapered at the same angle from the inlet channel to the outlet channel. In one specific embodiment the channels **16, 17** may be tapered at an angle of about 13.3° for a combined length of about 62.5mm. The tapered channels thus combine to gradually reduce the flow diameter, and thereby gradually increase the flow velocity. In the specific embodiment the outlet channel may have a length of about 25mm, or about 40% of the length of the tapered channels. The length of the tapered channels helps increase the flow velocity without turbulence, while the length of the outlet channel helps maintain a laminar flow exiting the nozzle **10**. The outlet channel also helps maintain the outlet stream as narrow as possible - i.e., as close to the outlet diameter as possible. However, as with prior art nozzles, the length and diameter relationships alone are not sufficient to ensure a non-diverging outlet stream.

[0014] In order to further reduce divergence of the outlet stream, the first tapered channel **16** is provided with linear vanes **20** that extend parallel to the length of the nozzle and extend generally radially inward from the inner surface of the channel. The vanes extend from the inlet channel **15** along the length of the first tapered channel **16** and essentially have an inversely tapered height, meaning that the vanes taper from a maximum height at the inlet channel to a zero height at the junction between the first and second tapered channels. In one specific embodiment, the inner edges **21** of the vanes **20** may be defined at a diameter of about 9.9mm. The first tapered portion with the vanes extends along about two-third (2/3) of the combined length of the two tapered portions, which in the specific embodiment provides a length of the first tapered portion of about 42.4mm. This configuration of vanes straightens the fluid flowing through the nozzle so that the discharge stream does not diverge significantly and maintains a generally straight stream.

[0015] The body **12** of the nozzle may be tapered from the inlet to the outlet, generally parallel to the taper of the first and second tapered channels. In order to strengthen the nozzle the body **12** may be provided with outer ribs **25** running the length of the body. The nozzle may be fabricated from a suitable material, such as molded from a hard plastic material. The inlet end **11** may include external threads, as shown in **FIG. 1** or may incorporate another feature for engagement to a hose, wand or similar fluid flow device. Alternatively, the entire nozzle may be integrally formed with the discharge end of a fluid flow device or may be overmolded onto the discharge end of the device.

[0016] A fluid flow nozzle **50** shown in **FIGS. 3** and **4** is similar to the nozzle **10** in that the nozzle includes vanes in a tapered channel. The nozzle **50** includes an inlet end **51** and an outlet end **52**. For clarity, the inlet end **51** is illustrated without any fitting for engagement to a hose, wand or other fluid flow device. However, it is understood that the nozzle **50** may incorporate a fitting or may be engaged as shown to a fluid flow device in a suitable manner. The nozzle **50** includes a tapered channel **55** extending from the inlet end **51** to an outlet channel **56** at the outlet end **52**. The outlet channel may have a constant diameter while the tapered channel **55** is tapered from the larger diameter of the inlet end to the smaller diameter of the outlet end. As with the nozzle **10**, the nozzle **50** may have an inlet to outlet diameter ratio of 4:1.

[0017] The nozzle **50** further includes curved vanes **58** disposed within the tapered channel **55**. The height to the edge **59** of the vanes decreases from the inlet end **51** to the outlet channel **56**, similar to the vanes **20** of the nozzle **10**. Thus, the height at end **60** is greater than the vane height at end **61**. Unlike the vanes **20**, the vanes **58** do not reduce to a zero height at end **61** but instead may have a non-zero height, as depicted in **FIG. 3**. The vanes **58** extend along the length of the tapered channel **55** and curve in the shape of a gradual spiral from inlet to outlet end. The vanes **58** follow a radius that is approximately equal to the length of the tapered channel **55**, which in a specific example can be about 90mm. As can be seen in **FIG. 4**, the ends **60** and **61** for each vane are at the same angular location in the nozzle, or in other words the outlet end **61** of the vane **58** is not angularly offset relative to the inlet end **60**. In the illustrated embodiment, four vanes **58** are evenly spaced around the circumference of the tapered channel. The width of the vanes is sufficient to maintain rigidity under high flow velocities but sufficiently narrow so as not to reduce the flow area significantly.

[0018] The curvature of the vanes imparts rotational momentum to the fluid flowing through the nozzle, while the tapered channel gradually increases the flow velocity. The rotational momentum helps keep the fluid flow collimated or helps prevent the fluid stream from diverging when it exits the nozzle **50**.

[0019] While the nozzle **50** includes radially inwardly directed vanes, the nozzle **70** shown in **FIG. 5** incorporates radially outwardly formed grooves **78** defined in the tapered channel **75** of the nozzle. The nozzle **70** includes a tapered channel **75** from the inlet end **71** to an outlet channel **76** at the outlet end **72**, in a manner similar to the nozzle **50.** The grooves **78** have a depth that is between one-third (1/3) and one half (1/2) of the wall thickness of the nozzle **70** at the tapered channel **75**. The width of the channels may be between 50% and 100% of the depth. In a specific example, the grooves have a width and depth of about 1.5mm. The grooves are curved in the form of a gradual spiral. Unlike the vanes **58** of the nozzle **50**, the ends of the grooves **78** may be angularly offset from each other. Since the grooves are recessed into the wall of the nozzle, the grooves do not impede the fluid flow or reduce the flow area. The grooves do impart rotational momentum to the fluid flow; however, the recessed nature of the grooves can reduce the ability to impart rotational momentum relative to the vanes of the embodiment of **FIG. 3**. In order to improve the ability to impart rotation to the fluid flow, a larger number of grooves **78** are provided in the nozzle **70** than vanes in the nozzle **50**. At least six grooves are provided and in a specific example eight grooves are uniformly spaced around the circumference of the tapered channel **75**, as shown in **FIG. 5**.

[0020] The nozzle **100** shown in **FIG. 6** includes an inlet channel **101** and an outlet channel **102** that can have a diameter ratio similar to the nozzles discussed above in order to achieve flow velocity increases of the magnitudes described herein. In order to achieve a non-turbulent linear discharge stream, the nozzle **100** incorporates staged reduction in flow area. In the illustrated example, the nozzle contemplates five stages from the inlet channel to the outlet channel. The first, third and fifth stages **104**, **106**, **108** are tapered channels while the second and fourth stages **105**, **107** are constant diameter stages. The tapered stages gradually step down the inner diameter from the diameter of the inlet channel **101** to the diameter of the outlet channel **102**. In one example, the diameter of the second stage channel **105** is about two-thirds (2/3) the diameter of the inlet channel, while the diameter of the fourth stage channel **107** may be about one-third (1/3) the inlet channel diameter. The tapered channels are thus configured to reduce the diameter by about one-third (1/3) at each stage.

[0021] The length of the stages may be calibrated to help reduce turbulent flow in the reducing stages **104, 106, 108** and to help maintain linear, non-turbulent flow through the constant diameter stages **105, 107**. In one example, the length of the constant diameter stages increases as the diameter of the stages decreases. Thus, the second stage channel **105** is longer than the inlet channel **101**, and the fourth stage channel **107** is longer than the second stage channel **105**. In one specific example, the constant diameter stage lengths can increase by about ten percent (10%). The tapered flow area reducing stages **104, 106, 108** may all have the same length, which in a specific embodiment may be about half the length of the inlet channel 101.

[0022] The nozzles **10, 50, 70, 100** may be provided with an attachment having selectable discharge orifices, such as the attachment **120** shown in **FIG.7** and shown engaged to the nozzle **100** in **FIG. 8.** The attachment includes a circular body **121** that can be mounted to a nozzle, such as nozzle **100** at a pivot point **126.** A separate mounting attachment (not shown) may be provided that clamps onto the nozzle and rotatably supports the attachment **120** at the pivot point **126.** The attachment includes a plurality of differently sized and shaped discharge orifices **122a-122h.** Each of the orifices includes a mating face **123** that may match the shape and diameter of the outlet channel **102.** The body **121** thus defines a tapered channel **124** from the mating face to the particular orifice. Some orifices may not incorporate a tapered channel, such as the orifice **122a** that includes a constant diameter feature. The attachment **120** is configured to create a fluid-tight seal between the outlet channel, such as channel **102** of nozzle **100,** and the selected orifice. Thus, the attachment may include seal rings between the nozzle and attachment, and/or the attachment may be formed of a self-sealing material, such as rubber.

[0023] While the invention has been illustrated and described in detail in the drawings and foregoing description, the same should be considered as illustrative and not restrictive in character.

Claims

1. A fluid flow |nozzle (50) comprising:

an elongated body having an inlet end (51) and an outlet end (52), the inlet end (51) configured

for engagement to a fluid supply, the elongated body defining a channel extending therethrough from said inlet end (51) to said outlet end (52);
said channel including an inlet channel adjacent said inlet end (51) and an outlet channel (56) adjacent said outlet end (52), said inlet channel defined at an inlet diameter and said outlet channel (56) defined at an outlet diameter that is less than said inlet diameter;
said channel further defining a tapered channel (55) extending from said inlet channel to said outlet channel (56) and having a length between said inlet and outlet channels; and
said elongated body further defining a plurality of vanes (58) circumferentially spaced around said tapered channel (55) and extending along at least a portion of the length of said tapered channel (55),
the fluid flow nozzle being **characterized in that** the plurality of vanes (58) are circumferentially spaced around said tapered channel (55) in a substantially spiral shape following a radius that is approximately equal to the length between said inlet and outlet channels to impart a rotational momentum on fluid flowing through the channel.

2. The fluid flow nozzle (50) of claim 1, wherein said tapered channel includes a first tapered channel adjacent said inlet channel and a second tapered channel adjacent said outlet channel, said plurality of vanes (58) defined only in said first tapered channel.

3. The fluid flow nozzle (50) of claim 2, wherein said first and second tapered channels are defined at the same taper angle.

4. The fluid flow nozzle (50) of claim 3, wherein said taper angle is about thirteen degrees (13°).

5. The fluid flow nozzle (50) of claim 2, wherein said first tapered channel extends along about two-thirds (2/3) of the length of said tapered channel.

6. The fluid flow nozzle (50) of claim 2, wherein said plurality of vanes (58) are tapered from a maximum height adjacent said inlet channel to substantially zero height adjacent said second tapered channel.

7. The fluid flow nozzle (50) of claim 1, wherein said inlet channel has a substantially constant diameter equal to said inlet diameter and said outlet channel has a substantially constant diameter equal to said outlet diameter.

8. The fluid flow nozzle (50) of claim 7, wherein said inlet diameter is about four (4) times greater than said outlet diameter.

9. The fluid flow nozzle (50) of claim 7, wherein said outlet channel has a length from said second tapered channel to said outlet end (52) that is about forty percent (40%) of the length of said tapered channel.

10. The fluid flow nozzle (50) of claim 1, wherein the outer surface of said elongated body is tapered from said inlet end (51) to said outlet end (52) and said body further defines strengthening ribs (25) extending along said outer surface from said inlet end (51) to said outlet end (52).

11. The fluid flow nozzle (50) of claim 1, wherein said plurality of vanes extend along said tapered channel (55) from said inlet end (51) to said outlet end (52).

12. The fluid flow nozzle (50) of claim 11, wherein said plurality of vanes (58) have a first end (60) adjacent said inlet end (51) of said nozzle (50) and a second end (61) adjacent said outlet end (52) of said nozzle (50), said first end (60) and said second end (61) being arranged at substantially the same angular position around the circumference of said tapered channel (55).

13. The fluid flow nozzle (50) of claim 11, wherein said outlet channel (56) has a length from said tapered channel (55) to said outlet end (52) that is about forty percent (40%) of the length of said tapered channel (55) with a substantially constant diameter equal to said outlet diameter.

14. The fluid flow nozzle (50) of claim 1, wherein said plurality of vanes (58) includes four (4) vanes (58).

15. The fluid flow nozzle (50) of claim 1, further comprising an attachment (120) adapted to be mounted to said elongated body, said attachment (120) including:

a plurality of orifices (122a-122h) having differently configured discharge configurations; and
a mating face (123) at each orifice (122a-122h) adapted to be selectively aligned with said outlet channel (56), each orifice (122a-122h) having a diameter at said mating face (123) that is substantially equal to said outlet diameter.

Patentansprüche

1. Fluidströmungsdüse (50), aufweisend:

einen länglichen Körper, umfassend ein Einlassende (51) und ein Auslassende (52), wobei das Einlassende (51) für einen Eingriff mit einer Fluidversorgung konfiguriert ist, wobei der längliche Körper einen Kanal definiert, der sich da-

durch vom Einlassende (51) bis zum Auslassende (52) erstreckt;
wobei der Kanal einen zum Einlassende (51) benachbarten Einlasskanal und einen zum Auslassende (52) benachbarten Auslasskanal (56) aufweist, wobei der Einlasskanal bei einem Einlassdurchmesser definiert ist und der Auslasskanal (56) bei einem Auslassdurchmesser definiert ist, der kleiner als der Einlassdurchmesser ist;
wobei der Kanal ferner einen sich verjüngenden Kanal (55) definiert, der sich vom Einlasskanal bis zum Auslasskanal (56) erstreckt und eine Länge zwischen dem Einlass- und dem Auslasskanal aufweist; und
wobei der längliche Körper ferner mehrere Flügel (58) definiert, die in Umfangsrichtung um den sich verjüngenden Kanal (55) herum beabstandet sind und sich entlang mindestens eines Abschnitts der Länge des sich verjüngenden Kanals (55) erstrecken,
wobei die Fluidströmungsdüse **dadurch gekennzeichnet ist, dass** die mehreren Flügel (58) in Umfangsrichtung um den sich verjüngenden Kanal (55) herum beabstandet sind in einer im Wesentlichen spiralförmigen Form, die einem Radius folgt, der ungefähr gleich der Länge zwischen den Einlass- und Auslasskanälen ist, um ein durch den Kanal strömendes Fluid in ein Rotationsmoment zu versetzen.

2. Fluidströmungsdüse (50) nach Anspruch 1, wobei der sich verjüngende Kanal einen zum Einlasskanal benachbarten ersten sich verjüngenden Kanal und einen zum Auslasskanal benachbarten zweiten sich verjüngenden Kanal aufweist, wobei die mehreren Flügel (58) nur im ersten sich verjüngenden Kanal definiert sind.

3. Fluidströmungsdüse (50) nach Anspruch 2, wobei der erste und der zweite sich verjüngende Kanal bei dem gleichen Verjüngungswinkel definiert sind.

4. Fluidströmungsdüse (50) nach Anspruch 3, wobei der Verjüngungswinkel etwa dreizehn Grad (13°) beträgt.

5. Fluidströmungsdüse (50) nach Anspruch 2, wobei der erste sich verjüngende Kanal sich entlang etwa zwei Dritteln (2/3) der Länge des sich verjüngenden Kanals erstreckt.

6. Fluidströmungsdüse (50) nach Anspruch 2, wobei sich die mehreren Flügel (58) von einer maximalen Höhe neben dem Einlasskanal zu einer Höhe von im Wesentlichen null neben dem zweiten sich verjüngenden Kanal verjüngen.

7. Fluidströmungsdüse (50) nach Anspruch 1, wobei der Einlasskanal einen im Wesentlichen konstanten Durchmesser aufweist, der gleich dem Einlassdurchmesser ist, und der Auslasskanal einen im Wesentlichen konstanten Durchmesser aufweist, der gleich dem Auslassdurchmesser ist.

8. Fluidströmungsdüse (50) nach Anspruch 7, wobei der Einlassdurchmesser ungefähr vier (4) Mal größer als der Auslassdurchmesser ist.

9. Fluidströmungsdüse (50) nach Anspruch 7, wobei der Auslasskanal eine Länge vom zweiten sich verjüngenden Kanal bis zum Auslassende (52) aufweist, die ungefähr vierzig Prozent (40%) der Länge des sich verjüngenden Kanals beträgt.

10. Fluidströmungsdüse (50) nach Anspruch 1, wobei die Außenfläche des länglichen Körpers vom Einlassende (51) zum Auslassende (52) verjüngt ist und der Körper ferner Verstärkungsrippen (25) definiert, die sich entlang der Außenfläche vom Einlassende (51) zum Auslassende (52) erstrecken.

11. Fluidströmungsdüse (50) nach Anspruch 1, wobei sich die mehreren Flügel entlang des sich verjüngenden Kanals (55) vom Einlassende (51) zum Auslassende (52) erstrecken.

12. Fluidströmungsdüse (50) nach Anspruch 11, wobei die mehreren Flügel (58) ein erstes Ende (60) neben dem Einlassende (51) der Düse (50) und ein zweites Ende (61) neben dem Auslassende (52) der Düse (50) aufweisen, wobei das erste Ende (60) und das zweite Ende (61) im Wesentlichen in der gleichen Winkelposition um den Umfang des sich verjüngenden Kanals (55) angeordnet sind.

13. Fluidströmungsdüse (50) nach Anspruch 11, wobei der Auslasskanal (56) eine Länge ab dem sich verjüngenden Kanal (55) bis zum Auslassende (52) aufweist, die ungefähr vierzig Prozent (40%) der Länge des sich verjüngenden Kanals (55) mit einem im Wesentlichen konstanten Durchmesser gleich dem Auslassdurchmesser beträgt.

14. Fluidströmungsdüse (50) nach Anspruch 1, wobei die mehreren Flügel (58) vier (4) Flügel (58) umfassen.

15. Fluidströmungsdüse (50) nach Anspruch 1, ferner aufweisend einen Aufsatz (120), der für einen Anbau am länglichen Körper angepasst ist, wobei der Aufsatz (120) aufweist:

mehrere Öffnungen (122a-122h) mit unterschiedlich konfigurierten Ausstoßkonfigurationen; und eine Passfläche (123) an jeder Öffnung (122a-122h), die angepasst ist, um selektiv mit dem Auslasskanal (56) ausgerichtet zu werden, wobei jede Öffnung (122a-122h) an der Passfläche (123) einen Durchmesser aufweist, der im Wesentlichen gleich dem Auslassdurchmesser ist.

Revendications

1. Buse d'écoulement de fluide (50) comprenant :

un corps allongé ayant une extrémité d'entrée (51) et une extrémité de sortie (52), l'extrémité d'entrée (51) étant configurée pour la mise en prise avec une alimentation en fluide, le corps allongé définissant un canal s'étendant à travers ce dernier, de ladite extrémité d'entrée (51) à ladite extrémité de sortie (52) ;
ledit canal comprenant un canal d'entrée adjacent à ladite extrémité d'entrée (51) et un canal de sortie (56) adjacent à ladite extrémité de sortie (52), ledit canal d'entrée étant défini à un diamètre d'entrée et ledit canal de sortie (56) étant défini au niveau d'un diamètre de sortie qui est inférieur audit diamètre d'entrée ;
ledit canal définissant en outre un canal progressivement rétréci (55) s'étendant à partir dudit canal d'entrée jusqu'audit canal de sortie (56) et ayant une longueur entre lesdits canaux d'entrée et de sortie ; et
ledit corps allongé définissant en outre une pluralité de pales (58) circonférentiellement espacées autour dudit canal progressivement rétréci (55) et s'étendant le long d'au moins une partie de la longueur dudit canal progressivement rétréci (55),
la buse d'écoulement de fluide étant **caractérisée en ce que** la pluralité de pales (58) sont circonférentiellement espacées autour dudit canal progressivement rétréci (55) en une forme sensiblement en spirale suivant un rayon qui est approximativement égal à la longueur entre lesdits canaux d'entrée et de sortie afin de communiquer un moment de rotation au fluide s'écoulant à travers le canal.

2. Buse d'écoulement de fluide (50) selon la revendication 1, dans laquelle ledit canal progressivement rétréci comprend un premier canal progressivement rétréci adjacent audit canal d'entrée et le second canal progressivement rétréci adjacent audit canal de sortie, ladite pluralité de pales (58) étant définie uniquement dans ledit premier canal progressivement rétréci.

3. Buse d'écoulement de fluide (50) selon la revendication 2, dans laquelle lesdits premier et second canaux progressivement rétrécis sont définis au même angle de conicité.

4. Buse d'écoulement de fluide (50) selon la revendication 3, dans laquelle ledit angle de conicité est d'environ 13 degrés (13°).

5. Buse d'écoulement de fluide (50) selon la revendication 2, dans laquelle ledit premier canal progressivement rétréci s'étendant le long d'environ deux tiers (2/3) de la longueur dudit canal progressivement rétréci.

6. Buse d'écoulement de fluide (50) selon la revendication 2, dans laquelle ladite pluralité de pales (58) sont progressivement rétrécies à partir d'une hauteur maximum adjacente audit canal d'entrée jusqu'à une hauteur sensiblement nulle adjacente audit second canal progressivement rétréci.

7. Buse d'écoulement de fluide (50) selon la revendication 1, dans laquelle ledit canal d'entrée a un diamètre sensiblement constant égal audit diamètre d'entrée et ledit diamètre de sortie a un diamètre sensiblement constant égal audit diamètre de sortie.

8. Buse d'écoulement de fluide (50) selon la revendication 7, dans laquelle ledit diamètre d'entrée est environ quatre (4) fois supérieur audit diamètre de sortie.

9. Buse d'écoulement de fluide (50) selon la revendication 7, dans laquelle le canal de sortie a une longueur allant dudit second canal progressivement rétréci à ladite extrémité de sortie (52) qui représente environ quarante pour cent (40 %) de la longueur dudit canal progressivement rétréci.

10. Buse d'écoulement de fluide (50) selon la revendication 1, dans laquelle la surface externe dudit corps allongé est progressivement rétrécie de ladite extrémité d'entrée (51) à ladite extrémité de sortie (52) et ledit corps définit en outre des nervures de renforcement (25) s'étendant le long de ladite surface externe de ladite extrémité d'entrée (51) à ladite extrémité de sortie (52).

11. Buse d'écoulement de fluide (50) selon la revendication 1, dans laquelle ladite pluralité de pales s'étendent le long dudit canal progressivement rétréci (55), de ladite extrémité d'entrée (51) à ladite extrémité de sortie (52) .

12. Buse d'écoulement de fluide (50) selon la revendication 11, dans laquelle ladite pluralité de pales (58) ont une première extrémité (60) adjacente à ladite extrémité d'entrée (51) de ladite buse (50) et une seconde extrémité (61) adjacente à ladite extrémité de sortie (52) de ladite buse (50), ladite première extrémité (60) et ladite seconde extrémité (61) étant agencées sensiblement dans la même position angulaire autour de la circonférence dudit canal progressivement rétréci (55).

13. Buse d'écoulement de fluide (50) selon la revendication 11, dans laquelle ledit canal de sortie (56) a une longueur dudit canal progressivement rétréci (55) à ladite extrémité de sortie (52) qui représente environ quarante pour cent (40 %) de la longueur dudit canal progressivement rétréci (55) avec un diamètre sensiblement constant égal audit diamètre de sortie.

14. Buse d'écoulement de fluide (50) selon la revendication 1, dans laquelle ladite pluralité de pales (58) comprend quatre (4) pales (58).

15. Buse d'écoulement de fluide (50) selon la revendication 1, comprenant en outre une fixation (120) adaptée pour être montée sur ledit corps allongé, ladite fixation (120) comprenant :

> une pluralité d'orifices (122a-122h) ayant des configurations de décharge différemment configurées ; et
> une face de couplage (123) à chaque orifice (122a-122h) adaptée pour être sélectivement alignée avec ledit canal de sortie (56), chaque orifice (122a-122h) ayant un diamètre au niveau de ladite face de couplage (123) qui est sensiblement égal audit diamètre de sortie.

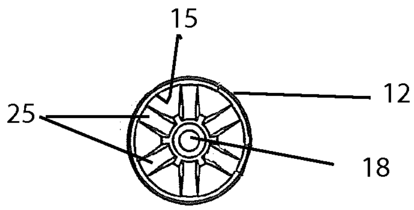


FIG. 2(b)

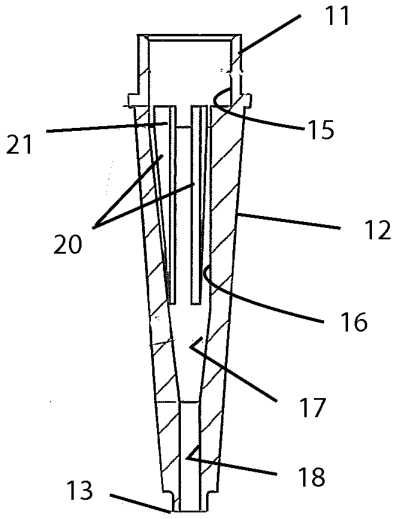


FIG. 2(d)

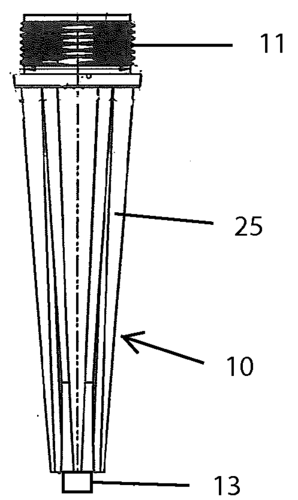


FIG. 2(a)

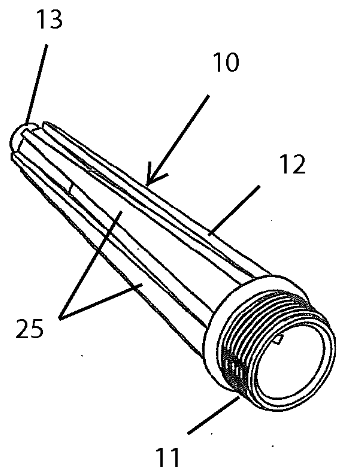


FIG. 1

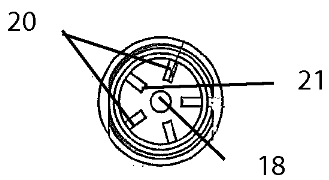


FIG. 2(c)

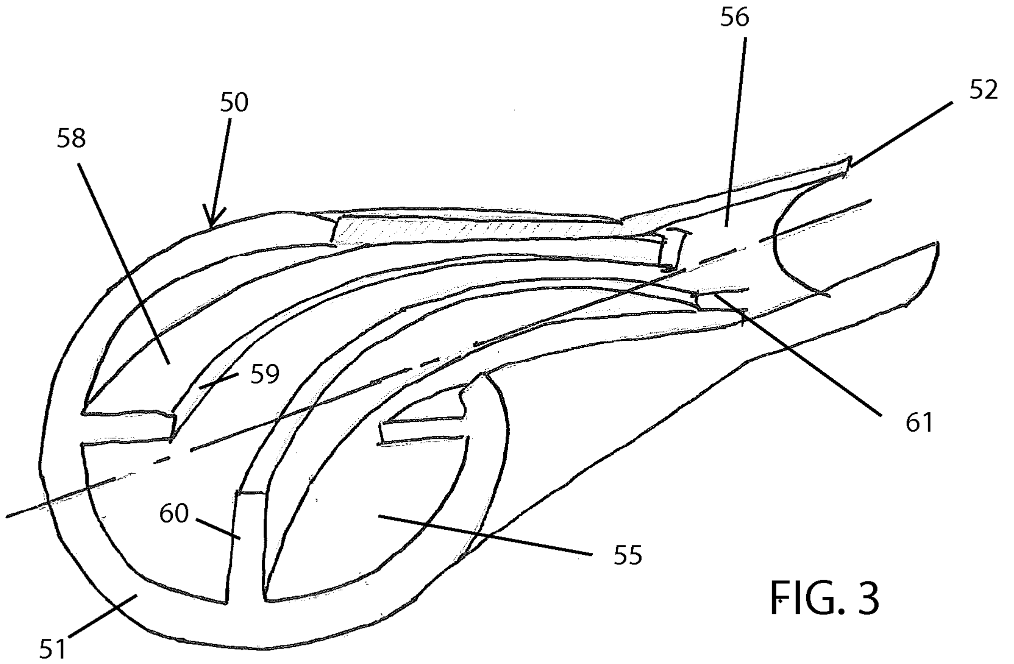
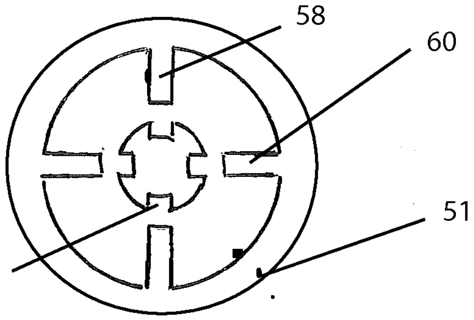


FIG. 3

FIG. 4



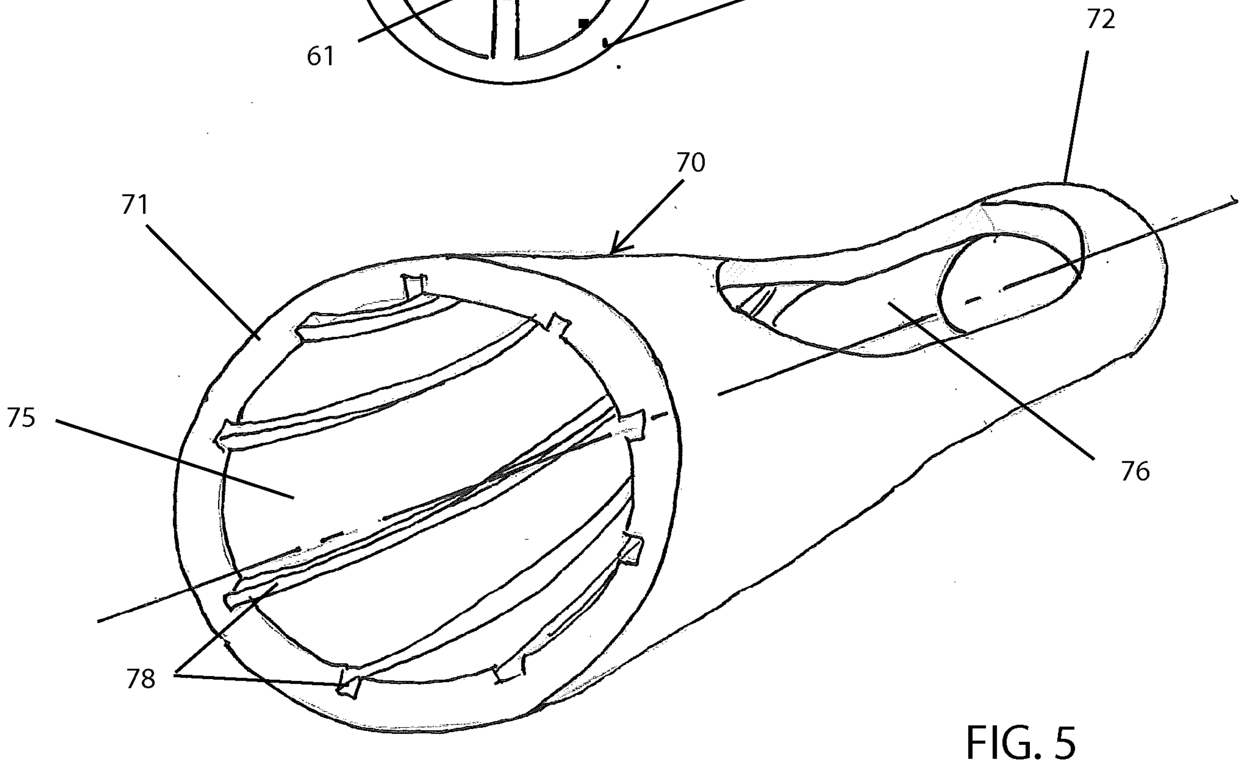


FIG. 5

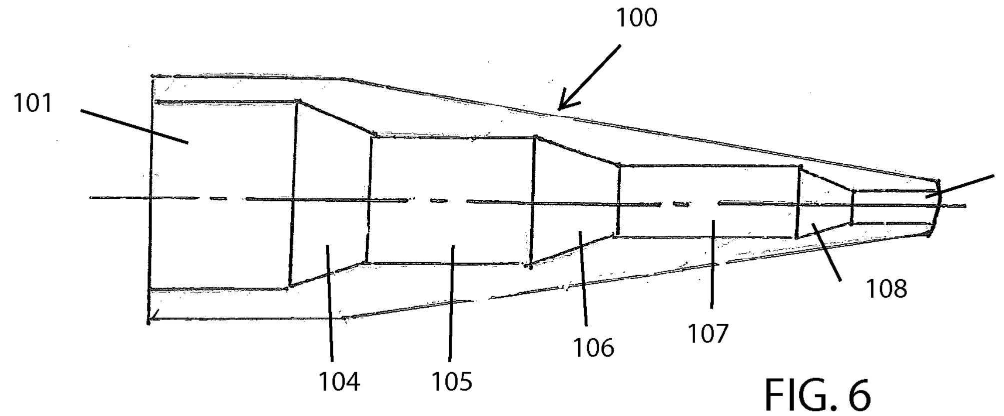
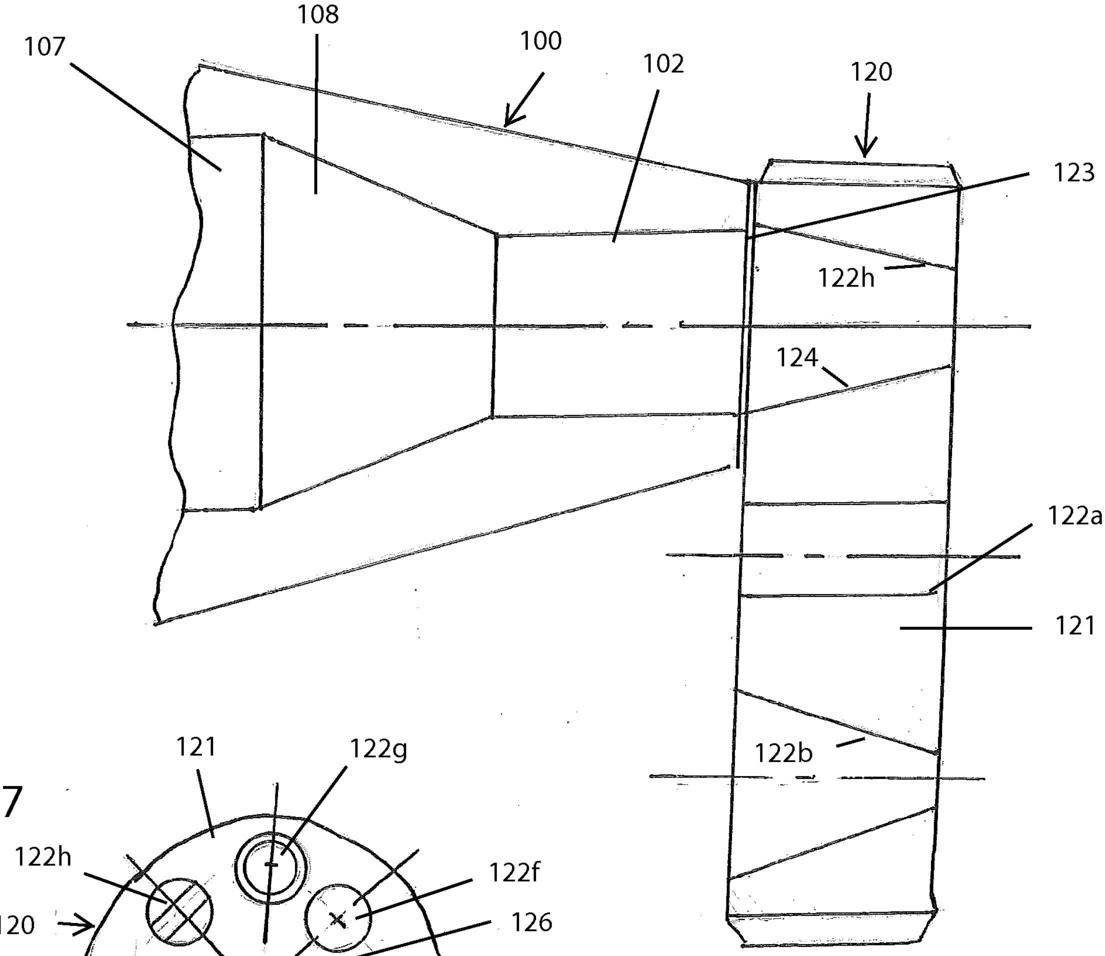


FIG. 6



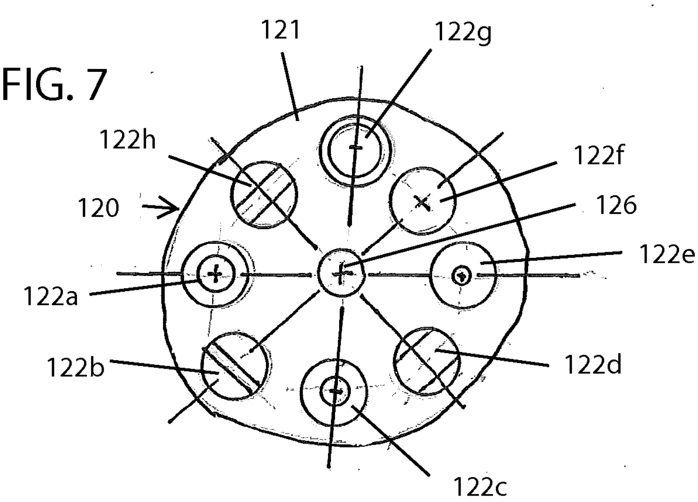


FIG. 7

FIG. 8

REFERENCES CITED IN THE DESCRIPTION

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Patent documents cited in the description

- WO 61816596 A **[0001]**
- WO 9902271 A **[0006]**