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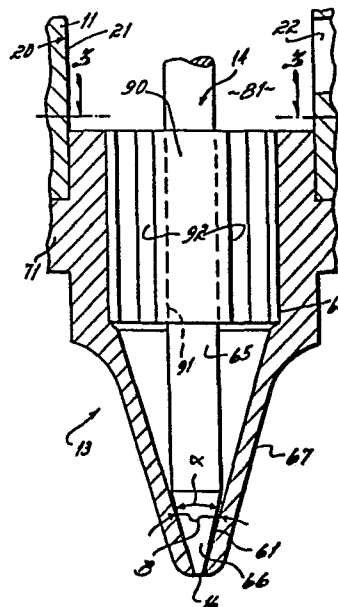
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54 Liquid dispensing device.

57 A liquid dispensing device in which there is a movable needle valve (14) operable to open and close a valve seat located within the nozzle of the dispensing device. The needle valve (14) is guided by a guide bushing (90) contained within the nozzle bore. A very small radiused semi-spherical end surface on the nozzle has been found to eliminate stringing of viscous liquid materials dispensed from the nozzle, particularly when the end surface is combined with a needle valve which is maintained concentric with the nozzle orifice by the guide bushing contained within the nozzle bore.



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This invention relates to liquid dispensing devices and particularly to devices adapted for dispensing relatively viscous sticky substances such as adhesives, hot melts, sealing compounds, etc.

5 There are many applications for dispensers in which it is desirable or necessary to sharply cut off the flow of liquid from the nozzle of the device without any drooling, dripping, or stringing of the liquid from the nozzle after closing of the nozzle.

10 To that end there has been a long standing need for a dispensing device which would sharply cut off the flow of liquid from the nozzle when the valve of the nozzle is closed.

In a dispenser in accordance with the invention

15 sharp cut off of flow without stringing from the nozzle of the dispensing device is achieved by providing a small orifice in a very small radius semi-spherical end surface of the nozzle.

The radius of the semi-spherical end may for

20 example be between 0.05 inches and 0.10 inches. Further advantages are obtained if the needle valve is mounted wholly inside the nozzle with the valve tip located at the nozzle tip so that on closure of the valve there is no cavity from which the liquid dispensed from the nozzle

25 can drip.

In a preferred embodiment, the nozzle orifice is only .020 inch in diameter and the nozzle tip is semi-spherical in shape and only .084 inch in radius. A nozzle thus configured and having a frustoconical

5 valve seat located immediately adjacent the nozzle orifice has been found to result in a nozzle which materially reduces the amount of stringing of material between the nozzle and substrate upon which the material is being deposited.

10 The improved seating of the needle valve on the valve seat of a dispenser of the invention with the consequent reduction of dripping or drooling or stringing from the nozzle may be achieved by providing a needle valve guide surface within a bushing contained
15 in the nozzle and located immediately adjacent the nozzle valve seat. To that end, the nozzle may include a guide bushing press fit into the nozzle and having peripheral channels or grooves through which liquid may flow from the interior of the device to the valve seat.

20 The primary advantage of a dispenser of the invention is that it provides very sharp cut off of liquid flow from the nozzle whenever the valve of the dispensing device is closed. Consequently, there is no dripping of liquid from the device after valve closing.
25 If the device is used to dispense very viscous substances

such as adhesive or sealing compounds, this unique nozzle and needle valve guide structure materially reduces stringing of the viscous substance between the nozzle orifice and the substrate on which the viscous substance is deposited.

The invention will now be described by way of example with reference to the accompanying drawings in which:

Figure 1 is a cross sectional view of one embodiment of a dispensing device incorporating the invention of this application.

Figure 2 is an enlarged cross sectional view of the nozzle and nozzle orifice of the dispensing device illustrated in Figure 1.

Figure 3 is a cross sectional view taken on line 3-3 of Figure 2.

Referring to Figures 1 and 2, a dispensing gun 10 has a generally cylindrical body 11, an end cap 12, and a nozzle 13. The end cap 12, body 11, and nozzle 13 all have a longitudinal bore extending there-through within which there is located an axially movable needle valve 14 for controlling flow of liquid from the orifice 16 of the nozzle 13.

The valve body 11 has a stepped axial bore 20, the larger diameter section 21 of which is located at the

forward end of the body. This larger diameter section 21 is intersected by a transverse passage 22 through which liquid is supplied to the gun. Additionally, a vent port 23 intersects the smaller diameter section 5 24 of the bore 20. There is also an air inlet passage 25 which connects the rear end of the valve body with an inlet air port 26.

A bushing 27 is located within the bore 20 of the body 11. This bushing supports seal assemblies 10 29 and 30 within a bore 28 which extends longitudinally through the bushing.

To prevent fluid flow around the bushing 27, there is an annular groove in the surface of the bushing within which there is an O-ring seal 31. Additionally, 15 there is an O-ring seal 32 contained within an annular channel formed in the valve body 11. There is a piston assembly 35 mounted upon the needle valve 14 for controlling movement of the needle valve 14. This piston assembly comprises a nut 36 threaded onto a threaded 20 section 37 of the needle valve and a piston retainer ring 38 sealingly secured onto the periphery of the nut 36. This retainer ring carries a pair of piston rings 39, 40 between which there is sandwiched a resilient gasket 41. The outer edge of this gasket 41 contacts

the interior surface of a cylinder 42 formed on the interior of the end cap so as to form a pneumatic seal between the lower side of the piston and the surface of the cylinder 42.

5 The end cap 12 is bolted onto the upper end of the body 11 by bolts (not shown). Preferably, a resilient gasket seal 44 is located between the contacting surfaces of the end cap and the body.

Communicating with the cylinder 42 formed in
10 the end cap 12, there is a stepped axial bore 45 which extends through the end cap. The upper smaller diameter section 46 of this stepped bore is threaded and receives an adjustment stud 47. A lock nut 48 secures the stud 47 in a position of axial adjustment.

15 Between the bottom surface of the stud 47 and the top of the piston assembly 35, there is a compression spring 50. The upper end of this compression spring 50 contacts the bottom surface of the stud 47 and the lower end of the spring 50 contacts the top surface of a
20 spring retainer 51. This spring retainer is supported upon the top surface of the piston assembly nut 36. By adjusting the axial position of the stud 47 within the bore 46, the closing force for retaining the needle valve closed may be adjusted or varied.

With particular reference to Figures 2 and 3, it will be seen that the nozzle 13 contains a generally cylindrical axial bore 60 which extends forwardly from the rear of the nozzle until it intersects a frustocon-
5 ical valve seat 61 at the forward end of the nozzle. This frustoconical valve seat 61 in turn communicates with a small cylindrical nozzle orifice 16 through which liquid exits from the nozzle. In a preferred embodiment, this orifice is approximately .020 inch in diameter and
10 the valve seat 61 defines an included angle of 34° .

Press fit into the bore 60 there is a guide bushing 90. This bushing 90 has a central axial bore 91 through which a needle valve 14 passes. Additionally, it has four equidistantly spaced longitudinal channels
15 or grooves 92 in its peripheral surface through which liquid may flow from the interior of the gun body 11 to the nozzle orifice 16 as is explained more fully hereinafter.

At its forward end the needle valve 14 has a
20 cylindrical section 65 slideably received within the bore 91 of the guide bushing 90. This sliding fit requires that the bore 91 be slightly larger than the cylindrical section 65 of the needle, but that there be no more than approximately .002 inch clearance between
25 the bore and the needle valve. Preferably, the

clearance is approximately .001 inch.

At its forward end the needle valve 14 terminates in a frustoconical shaped valve section 66 engageable with the frustoconical shaped valve seat 61 in the nozzle 13. The frustoconical shaped end section 66 of the needle valve defines an included angle α of approximately 30° while the valve seat 61 defines an included angle B of approximately 34° . The exterior surface 67 of the nozzle tip is also tapered at approximately 34° . Consequently, there is some slight clearance between the frustoconical shaped section of the needle valve 65 and the valve seat 61 at the rearward end of the valve seat.

The nozzle tip is semi-spherical and has a very small radius of 0,084 inches. The nozzle 13 has a cylindrical hub section 70 which is tightly fitted within the large diameter section 21 of cylindrical bore 20 in the body 11. Forwardly of the hub section 70, there is a flange 71 which is bolted to the forward end of the body 11 by conventional threaded connectors 73. Preferably, there is an O-ring seal 74 contained within a channel 75 formed in the forward end of the dispenser body 11. This seal 74 prevents any leakage of liquid between the nozzle 13 and the body 11.

To prevent any movement of the bushing 27 within the bore 20, there is a spring 76 located between the rearward end of the nozzle 13 and the forward end of the bushing seal assembly 29. This spring 76
5 biases the bushing rearwardly and maintains a flange 77 of the bushing engaged with a shoulder 78 formed in the bore 20.

In operation, liquid under pressure is supplied to the liquid inlet port 80 of the device. This port
10 communicates via passage 22 with the bore 20 of the body 10 such that liquid supplied to the port 80 flows into a liquid storage chamber 81 contained within the device. This chamber 81 is in turn open to the longitudinal passages 92 within the guide bushing 90.

15 Whenever the device is to be actuated so as to permit liquid to flow from the storage chamber 81 through the passages 92 and past the valve seat 61 to the orifice 16, high pressure air is supplied to the port 26. This high pressure air overcomes the bias of the spring
20 50 and causes the piston assembly 35 to move upwardly, carrying with it needle valve 14. This upward movement of the needle valve results in the lifting of the conical section 66 of the valve off of the seat 61 and, results in flow of liquid from the storage chamber 81 through the

passages 92 via the valve to the orifice 16. This flow continues so long as the air pressure is maintained to the port 26. When that air pressure is relieved, as for example by actuation of a controlling pneumatic valve
5 (not shown) the spring 50 effects closing movement of the valve.

The most important advantage of this invention resides in the fact that when it is used to dispense high viscosity liquids such as adhesives or sealing
10 gasket material compounds, etc., it materially eliminates or reduces stringing of material from the nozzle orifice after valve closing. This advantage is apparently partially attributable to the needle valve guide surface being in close proximity to the nozzle valve seat
15 and partially attributable to the small radius semi-spherical tip on the nozzle. The needle valve guide surfaces on the guide bushing apparently maintains the needle valve concentric to the valve seat with the result that sharp cut off of flow and the reduction or
20 elimination of stringing is enhanced and apparently the small radius semi-spherical tip on the nozzle also contributes to this reduction.

CLAIMS:

1. A liquid dispensing device comprising a nozzle having an axial bore, the bore having a frustoconical valve seat formed therein, terminating at a nozzle outlet
5 orifice, an axially movable needle valve having a generally conically shaped section on the distal end thereof, engageable with the valve seat to close the valve, characterised in that a bushing (90) is located within the nozzle, and having a needle valve guide
10 surface (91) formed therein, engageable with the needle valve at a location spaced from the valve seat so as to guide movement of the needle valve along a longitudinal axis concentric with the valve seat, the bushing providing passageways for supplying liquid to the nozzle.
- 15 2. A liquid dispensing apparatus as claimed in Claim 1 wherein the passageways are formed by longitudinal grooves located in the surface of the bushing.
3. A liquid dispensing a device as claimed in Claim 1 or 2 wherein the needle valve guide surface extends
20 over a substantial portion of the axial length of the nozzle.
4. A liquid dispensing device as claimed in any of the preceding claims wherein a liquid storage chamber is defined, at least in part, by an axial bore in the

body of the device, the bore communicating with the bore of the nozzle.

5. A liquid dispensing device as claimed in any of the preceding claims wherein the nozzle has a tip which tapers inwardly toward the nozzle orifice, the tip terminating in a semi-spherical end surface having a radius of between 0.05 and 0.10 inches.

6. A liquid dispensing device as claimed in Claim 5 in which the radius of the semi-spherical end surface is 0.084 inches.

7. A liquid dispensing device as claimed in Claim 5 or 6 wherein the taper of the exterior surface of the nozzle tip defines an included angle of approximately 34° .

8. A liquid dispensing device as claimed in any of the preceding claims wherein fluid motor means are mounted in a bore of the body for effecting movement of the needle valve into and out of sealing engagement with the valve seat.

9. A liquid dispensing device as claimed in any of the preceding claims in which the needle valve is mounted within the nozzle, the valve tip being located at the nozzle tip so that there is no significant cavity from which liquid being dispensed from the nozzle, may drip.

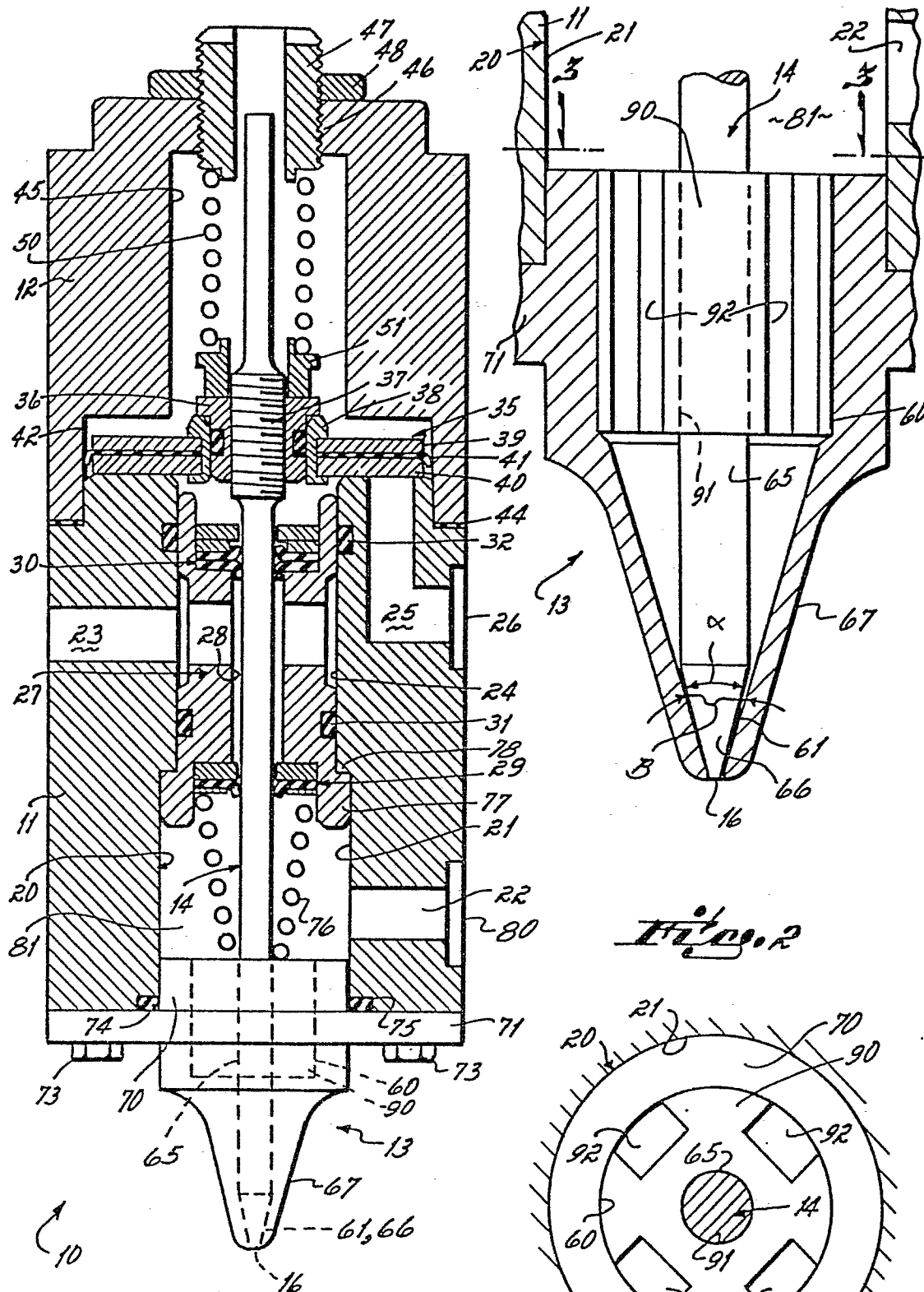


Fig. 1

Fig. 2

Fig. 3



European Patent
Office

EUROPEAN SEARCH REPORT

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Application number

EP 81 30 2056

DOCUMENTS CONSIDERED TO BE RELEVANT			CLASSIFICATION OF THE APPLICATION (Int. Cl.)
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	
	<u>US - A - 2 865 675</u> (V.D. ROOSA) * Column 2, lines 33-53; figures * ---	1,3	B 05 B 1/32 B 05 C 11/10 B 65 B 51/02
	<u>GB - A - 398 788</u> (O. TREICHEL) * The whole document * ---	1-3,8, 9	
	<u>FR - A - 1 463 455</u> (F.J. CONTE) * The whole document * ---	1,3,4, 8,9	
	<u>US - A - 2 670 241</u> (G.S. PYLES) * Figure 2 * ---	5,9	TECHNICAL FIELDS SEARCHED (Int. Cl.) B 05 B B 05 C B 65 B
	<u>DE - A - 2 055 127</u> (A. CAST) * Figure 2 * ---	4,5,9	
	<u>US - A - 4 066 188</u> (C.H. SCHOLL) * The whole document * ---	4,8	
	<u>US - A - 2 983 480</u> (D.G. GREENLIE) * The whole document * -----	4,8	CATEGORY OF CITED DOCUMENTS X: particularly relevant A: technological background O: non-written disclosure P: intermediate document T: theory or principle underlying the invention E: conflicting application D: document cited in the application L: citation for other reasons &: member of the same patent family, corresponding document
The present search report has been drawn up for all claims			
Place of search The Hague	Date of completion of the search 13.08.1981	Examiner COLPAERT	