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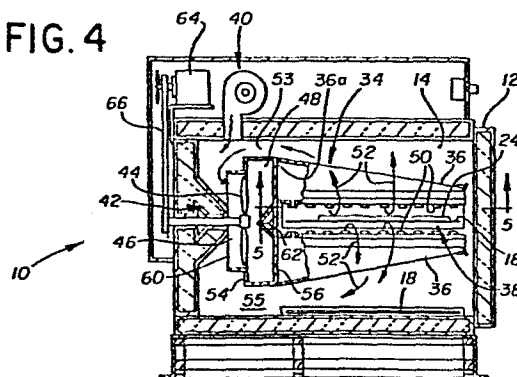
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(54) Fan-plenum configuration.

(57) An impingement device (10) for preparing food product comprising an enclosure (12) having a conveyor (18) received therethrough for conveying food product through the enclosure (12). Positioned within the enclosure (12) and about the conveyor (18) portion therein are ducts (36) for impinging gas against the food product, a plenum (48) connected to the ducts (36) for supplying gas to the ducts (36), and an impeller assembly (44) for recirculating the gas within the enclosure (12). A heating source (40) or cooling source (340) is provided within a chamber (14) on the opposite side of the plenum (48) from the ducts and controls (26) are provided for regulating the temperature of the heat or cooling source. The plenum (48) is spaced inwardly from the walls of the enclosure (12) to provide return flow around the plenum.

FIG. 4

FAN-PLENUM CONFIGURATION

Disclosed is impingement food heat and cooling devices and fan-plenum air flow configurations therefor.

5 This application is related to European Patent Application No. filed naming Michael C. Henke, Gordon D. Bell, Don P. Smith and Virgil L. Archer as the inventors and entitled "Impingement Oven".¹

10 In impingement heating and cooling devices for processing foods a jet of either heated or cooled gas impinges upon the surface of the food to sweep away the boundary layer and increase the rate of heat transfer between the gas and the food product. Cooking and cooling devices of this type have proven quite satisfactory and have been combined with other types of
15 cooking apparatus such as microwaves to produce unique combinations of cooking devices. Exemplary prior art impingement devices are found in prior art United States Patents Nos. 3,884,213 and 4,154,861.

20 In impingement devices used to process foods, the control of the flow of the gases presents problems which are not found in any other type of heating or cooling device. It is apparent that the volume and velocity of the gases contacting the food being processed determine the cooking or cooling rate. It
25 is therefore critical to assure that the gas flow is within the specifications necessary to properly cook or cool the food to insure that the food product is either completely cooked or cooled as the case may be. In addition to the problems of controlling the rate and
30 volume of the flow of the impinging gases in devices of this type, other unique problems are present relating to the flow of these gases which are not present in any other type of heating or cooling device. These problems relate to the distribution of the heating or cooling
35 gas jets across the product to insure that uniform cooking or cooling occurs throughout the entire product. The critical requirement of food processing impingement devices requires that the various jets contacting a

food product at a given time operate in a fixed relationship to each other even though variations in the supply flow rate occur. In addition, it has been found that the return flow path of the gases from the impingement jets affects the performance of the oven. In the past, it was believed that the most efficient configuration for removing discharged impingement gases was to draft the gases in a direction transverse to the direction of the flow of the gases in impingement jets, at least, in the area of the food product. Thus, impingement gases would flow from the nozzle and impinge upon the product and after contacting the product would move across the product in a direction transverse the direction of flow of the impingement jets. Although this configuration performed satisfactorily it was found, contrary to what would be expected, that the efficiency and performance of the system could be improved by discharging the impingement gases in a direction reverse of the direction of the impingement jets. Thus, it was found by removing as much as possible of the transverse draft of the impingement gases and by causing a discharge in a direction opposite to the direction of the flow of the impingement jets that surprising improvement in the operating characteristics occurred. It is believed that this improved configuration reduced turbulence and increased the uniformity of heating or cooling throughout the device.

The present invention provides an improved gas flow jet type impingement food processing device. The invention has application in both an impingement heating device such as ovens and an impingement cooling device such as cryofreezing units. The invention also has application in an impingement device combined with other types of processing apparatus such as microwave, convection and radiant ovens. One aspect of the invention describes a fan-plenum configuration which produces an improved distribution and supply of gases to the nozzles forming the impingement jets and thus improving

the performance of the device.

In another aspect of the invention, the return flow path is provided for the impingement gases. This path allows the gases to move in a direction opposite to the flow in the impingement jets and away from the food product. These and other aspects of the present invention will become better understood by considering the drawings and detailed description contained in this application.

10 The following drawings are included to illustrate the various aspects of the present invention:

Figure 1 is a perspective view of a commercial impingement oven incorporating the present invention;

Figure 2 is a plan view partly in section of the oven illustrated in Figure 1;

Figure 3 is a section taken on line 3-3 of Figure 2 looking in the direction of the arrows;

Figure 4 is a section taken along line 4-4 looking in the direction of the arrows;

20 Figure 5 is a partial sectional view taken along line 5-5 of Figure 4 and looking in the direction of the arrows;

Figure 6 is a view similar to Figure 4 showing an alternate configuration of the fan-plenum arrangement;

25 Figure 7 is a perspective view of the fan illustrated in Figures 6 and 8;

Figure 8 is a view similar to Figure 6 showing alternate configuration of the fan-plenum arrangement;

30 Figure 9 is a front elevation of a cooling impingement device of the present invention;

Figure 10 is a horizontal section of the device taken on line 10-10 of Figure 9 looking in the direction of the arrows; and

35 Figure 11 is a vertical sectional of the device taken on line 11-11 of Figure 9 looking in the direction of the arrows.

The drawings and the accompanying description when considered together and read fairly provide a sufficient description for a person of ordinary skill in the art to make and use the present invention.

5 Referring now to the drawings wherein like reference characters are used throughout the various views to indicate like or corresponding parts, there is shown in Figures 1-5 an impingement device for food products. For purposes of description, the device is designated
10 by reference numeral 10. The embodiment illustrated in Figures 1-5 is a commercial type and utilizes an endless conveyor to move food products therethrough. The device contains a plurality of upper and lower
15 impingement jets directed toward the top and the bottom of the food product as it moves through the oven. Preferably, the conveyor is of a chain mesh material to allow impingement jets to pass therethrough. The present invention is illustrated as a commercial oven but also has application to other types of ovens and
20 in food cooling devices. The description of the present invention with regard to the particular type of ovens illustrated in the Figures is by way of example and is not intended by implication to limit the invention to use in this particular type of oven. It should be
25 appreciated that the present invention, for example, has application to smaller ovens such as those used in the home wherein an endless conveyor does not extend through the oven.

 In the embodiment of Figures 1-5, the device 10
30 comprises an insulated enclosure 12. Enclosure 12 can be constructed in any suitable manner well known in the food processing industry to produce an enclosure which forms an insulated chamber. Enclosure 12 is an elongated structure with openings 16 in the ends
35 thereof. An endless conveyor 18 extends through these openings 16. The conveyor 18 is conventionally supported and driven by a motor (not shown) such that the food supporting surface 20 of the conveyor is moved in the

direction of arrow 22. In operation, food products 24 can be continuously moved through the enclosure 12 in the direction of arrow 22. As will be described in more detail, food products 24 are carried by conveyor 18 past a plurality of impingement jets to properly cook the food products. A control panel 26 is provided with control means for regulating the speed and operation of conveyor 18 and the impingement jets within enclosure 12.

10 As can be seen in Figure 3, the products 24 are moved by the conveyor 18 through the enclosure 12 and pass through successive impingement apparatus assemblies 28,30,32 and 34, respectively. Each of the impingement assemblies has upper and lower impingement ducts 36.

15 Ducts 36 are positioned in close proximity to the conveyor 18 and define a cooking area or zone 38 between the upper and lower ducts 36. By varying the location and number of these impingement ducts 36 in each of the cooking areas or zones 38 the cooking rate can be varied.

20 The basic construction of each of the impingement assemblies 28-34 is similar in overall construction and configuration. The impingement assemblies 28-34 differ in the position in spacing of the respective impingement ducts 36. In Figure 4, impingement assembly 25 34 is shown in section. It should be understood, of course, that the description of impingement assembly 34 is typical for the other impingement assemblies except as noted herein.

Impingement assembly 34 comprises the following
30 basic elements which co-operate together to supply an even and continuous supply of heated air to the product 24 as the product is moved through the cooking area 38 by conveyor 18: a heater assembly 40, a tapering shroud 42, a driven impeller 44, a collar 46, a plenum
35 assembly 48, and a plurality of tapered impingement ducts 36 defining a cooking area 38 therebetween. In operation, tapering shroud 42 co-operates with collar 46 to direct return air into the impeller 44.

Impeller 44 in turn moves the return air into the plenum assembly 48. Plenum 48 acts as a manifold to properly distribute air into the various impingement ducts 36. Air in the impingement ducts exits through nozzles 59
5 and is directed against the upper and lower surfaces of the products 24.

One embodiment of an impingement duct 36 is shown in Figures 4 and 5. As is illustrated, the duct 36 is greater in cross sectional area at the end 36A adjacent
10 to the plenum 48. The cross sectional area of the duct 36 decreases along the length of the duct as the duct extends away from the end 36A. This configuration provides an even pressure distribution of the air along the length of the duct. Duct 36 has a plurality
15 of nozzles or perforations 50 which are arranged in the pattern illustrated in Figure 5. The gas supplied to the interior of the duct 36 exits the perforations 50 and is directed into the cooking area 38 in streams or jets as illustrated by arrows 52. These ducts 36
20 can be embodied as shown and described in United States Patent 4,154,861 and operate as described therein to provide a plurality of spaced discrete streams of gas or impingement jets that contact and cook the food. These jets each contact a discrete area on
25 the food product and will not be substantially diffused prior to impingement against the surface of the food product.

As is shown in Figure 4, impingement ducts 36 are mounted on and supported by the plenum assembly 48.
30 Details of the configuration of the plenum assembly is shown in Figures 2,3 and 4. The plenum assembly is constructed from a suitable sheet of material, such as stainless steel, welded, riveted or otherwise fastened together to form a plenum chamber. Plenum assembly 48
35 is positioned to extend above and below said conveyor 18. Plenum assembly 48 is spaced inwardly from the walls of said enclosure to provide a return flow path for air through spaced 53 and 55 located respectively

above and below the plenum. Backwall 54 tapers as shown in Figure 2 to direct air outward to the impingement ducts 36 positioned across the width of the plenum. The frontwall 56 has a plurality of rectangular openings 58 through which air moves into the impingement ducts 36. In the embodiment shown in Figures 1-5, a collar 46 surrounds an opening 60 in the backwall 54 of the plenum 48. This collar is of a size to closely surround the path of impeller 44 without interfering with its movement. The collar 46 may be shaped to reduce turbulence and is small enough to co-operate with the impeller 44 to increase the efficiency and distribution of air into the plenum 48. This deflector 62 co-operates with the impeller 44 to direct air in a direction above and below the deflector 62 to improve the distribution characteristics of the plenum, reduce turbulence, and supply air to the individual impingement ducts 36 through openings 58.

As can be seen in Figure 4, tapering shroud 42 is positioned at the rear of impeller 44. This tapering shroud can be either frusto, conical or pyramid shaped to guide the return air into the collar 46. In addition, because the return air must change direction as it enters the collar 46 and impeller 44, tapering shroud 42 co-operates to eliminate dead spaces behind the impeller and forms an annular opening to direct the gas into the impeller.

Impeller 44 is shown suitably connected and driven by motor 64 through an endless belt 66. The motor 64 of course could be axial with the shaft of impeller 44. The shaft of impeller 44 extends through the center of the tapering shroud 42.

In Figure 4, heater assembly 40 is shown communicating with chamber 14. As is conventional in ovens, this heater assembly 40 is controlled by a thermostat and control circuitry not shown and operates as required to add heat to the return air before it enters the collar impeller area of the enclosure 12. Thus the

temperature of the air in the plenum can be controlled by mixing return air with heated air from the heater 40. The heater assembly 40 can be conventional gas or electric type. It should be understood that for purposes of description the improved air flow mechanism of the present invention is disclosed in regard to oven. However, the present invention has application and utility in devices which are used to cool food products. In such instance, the heater 40 would be replaced with a suitable cooling device such as a refrigeration coil or the like to cool the air as it flows through the system. Such devices could be utilized in freezing or cooling food products. It should therefore be understood that the improved flow mechanism of the present invention would likewise have application in the freezing or cooling of foods in an apparatus of this device.

According to a particular feature of the present invention the air flow of the impingement jets and the return air is such that uniform cooking occurs of the product 24 without regard to location across the width of the conveyor 18. It is believed that this uniform cooking is a result of two primarily co-operating aspects of the apparatus of the present invention. The first aspect is a co-operation of the impeller 44, tapering shrouds 42, collar 46, deflector 62 and tapered impingement ducts. These elements of the invention co-operate to provide an even and adequate distribution of flow of return gases into the plenum assembly 48 thus insuring an adequate supply of gases to each of the impingement ducts 36 communicating with the plenum 48. An adequate and uniform distribution of return air to the various impingement ducts 36 insures a uniform and adequate supply of air for proper cooking throughout the cooking area 38. The adequate and even cooking by the impingement jets is further enhanced by the co-operation of the return path of the discharged gas. As is illustrated in Figures 4-5, the flow path of

the air from the impingement jets is schematically shown by arrows 52. In Figure 4, the impingement jets are shown contacting the product 24 to sweep away the boundary layer and cause rapid heat transfer between a product 24 and the air of the impingement jet. Air from the impingement jet moves in a reverse direction after contacting product 24. The return air moves up between the impingement ducts 36 as shown in Figure 3. By moving the return air in a direction opposite to the vector of the impingement jet, side or transverse drafts are minimized and a uniform cooking occurs across the width of the conveyor 18 in the cooking area 38. It is believed that transverse or side drafts cause variations in cooking rates which can make the heat transfer rate of the particular impingement jets vary with position across the width of product 24 creating undesirable variations in cooking. By providing subject space between each of the impingement ducts 36, the return path 52 of the discharge air can move up or down, respectively, between the impingement jets and out of the cooking area. The return of these gases in this manner minimizes the affect on the uniformity of cooking.

Return air can move above and below the cooking area, over the plenum assembly 48, past the heater assembly 40 and into the impeller 44 of the tapered shroud 42. Dividing of the return gases above and below the plenum assembly 48 minimizes undesirable transverse drafts in the cooking chamber. This improvement is surprising in that one would not expect that even cooking would result from the movement of the discharged air in a direction opposite to the vector of impingement flow. It would have been assumed that moving the discharge gases in a direction opposite to the direction of the impingement flow would reduce efficiency and interfere with the flow of the impingement jets. In addition, by moving the air over both the top and bottom of the plenum provides a more even distribution of the

air through the impeller 44. This increases the efficiency of the system by providing short length air return paths from the product to the impeller. Also if the conveyor surface is blocked solid, as when covered
5 cooking pans are used, or when a solid conveyor belt is used, the air from the top and the bottom returns directly to the impeller without passing around the closed conveyor surface systems which return the gases solely to either the top or the bottom of the oven.

10 It is to be understood, of course, that the illustration of the invention in Figures 1-5 is by way of example and that other configurations could be utilized to practice the present invention. For example, alternate embodiments of the present invention are shown in
15 Figures 6-11.

In Figure 6 an alternate embodiment of an impingement oven 110 is shown. This embodiment would also have the same general configuration of the oven shown in Figures 1-5 and the details of construction of the
20 impeller assembly 144 and plenum assembly 148. In Figure 6, the backwall 154 of the plenum 148 is provided with an internally directed flange 146 which forms a collar for the impeller 144. This internally turned flange 146 extends completely around the periphery
25 of the rear opening 160 in the plenum assembly 148. Flange 146 can be integrally formed with backwall 154 or can be formed by a separate piece fixed to the backwall 154 in a suitable manner. Flange 146 co-operates with the impeller 144 and tapering shroud 142 to improve
30 the air flow and distribution into the plenum 148.

The impeller assembly 144 is shown in detail in perspective view in Figure 7. The tapering shroud and the impeller 44 has a central dome shaped portion with impeller blades 145 on the periphery thereof.
35 Impeller assembly 144 is positioned with the concave side of the dome shaped portion 143 facing toward the plenum 148. The dome shaped portion co-operates with and fits around tapering shroud 42 to assist in directing

the flow of air into the plenum 148. The central dome shaped portion 143 has a cylindrical skirt 147 on the periphery thereof. The skirt 147 extends parallel to the flange 146 to define an annular space therebetween.

5 This annular space between the skirt 147 and flange 146 is the area through which the blades 145 rotate as they are driven by the motor 164. The blades 145 are each scoop shaped and provide the unique advantage of causing flow in both the axial and radial direction

10 as shown by arrows 149. This axial and radial flow distributes the air throughout the plenum and co-operates with the shroud 142, flange 146 and a deflector 162 to provide uniform distribution of the air and uniform heat transfer rates throughout the impingement ducts 136.

15 In Figure 8, an oven 210 is illustrated having a plenum assembly 248 with a tapering shroud 142 and impeller assembly 144 identical to the construction illustrated in Figure 7. In the embodiment of Figure 8, the plenum 148 has a backwall 154 (not shown) with

20 an opening 260 therein. Fixed around the periphery of the circular opening 260 is a length of tubing 261 positioned as shown in Figure 8. This tubing 261 is fixed around the edge of the opening 260 by welding, fasteners, or the like and operates to form a smooth

25 edge around which the returning air flows. This smooth edge around the opening 260 co-operates with the fan to improve the flow of air into the plenum assembly 248.

In Figures 9-11, the invention is shown in the embodiment of a cooling device. This cooling device

30 is identified generally by reference 310 and comprises a sealed insulated enclosure 312 forming a chamber 314 therein. Openings 316 provide clearance for an endless conveyor 318 to extend through the chamber 314. The conveyor 318 is suitably driven and controlled as

35 explained in regard to the embodiment of Figures 1-5 and is used to move food product through the chamber 314.

A single impingement assembly 328 is mounted within chamber 314. Impingement assembly 328 has a plurality of impingement ducts 336. Impingement assembly 328 and ducts 336 are similar in construction to those shown in Figures 1-5. A plurality of refrigeration coils 340 are provided within the chamber 314 in the flow path of the return to the plenum assembly 348. A conventional refrigeration assembly 341 is connected to the cooling coils 340 and is provided with suitable compressors, evaporators and the like to provide cooling for the return air passing across the coils 340. In this manner device 310 can be utilized to cool food products 324 carried by conveyor 318 as it moves through cooking area 338. It is also envisioned that additional cooling could be provided by apparatus not shown to directly inject cooling gases such as nitrogen to the rear of the plenum 348. Thus the improved flow characteristics of the oven of Figures 1-5 can be applied to a food cooling device such as shown in Figures 9-11. In the cooling device of Figures 9-11 the return air flow path can be identical to that shown and described in regard to Figures 1-5 even though the air is utilized for cooling rather than cooking.

The various aspects of the present invention have been described in regard to exemplary embodiments incorporating the present invention. In addition, the invention can, for example, be applied to cooking or cooling apparatus without the presence of an endless conveyor extending through a chamber. The improved flow characteristics disclosed in the present invention could have utility and advantage in devices having a sealed enclosure wherein the food product could be moved within the carrousel or oscillating support. It is also envisioned that the food product could be fixed and the impingement apparatus moved to cause a sweep over the surface of the product. In addition, a closed system which combines microwave or other type of food cooking or processing with the apparatus of

the present invention could be used.

Thus, having described the present invention it
is to be understood, of course, that numerous modifica-
tions, alterations and changes can be made in the device
5 by one of ordinary skill in the art without departing
from the spirit and scope of the invention as claimed
herein.

CLAIMS

1. An impingement device for processing food products comprising:

- 5 an enclosure defining a chamber for receiving a food product therein,
- means for supporting a food product within the chamber,
- a plurality of duct means mounted in said chamber, nozzles in each of said duct means shaped and spaced to form a plurality of discrete streams of gas each stream impinging upon a discrete
- 10 area of said food product and diffusing upon contact with said food product, at least one duct means positioned above and one duct means positioned below said food product supporting means for directing a gas through said nozzles in said duct means to impinge upon the food product on said supporting means,
- 15 a plenum in said chamber for enclosing a reservoir of gas, said plenum being positioned adjacent an edge of said support means and having said duct means connected thereto, said plenum having one side with an outlet openings in communication with said duct means and having another side with an inlet opening therein,
- 20 said plenum being of a size and shape to extend above and below said support means and being spaced inwardly from said enclosure whereby a flow path for gas in said chamber is formed both above and below said plenum to allow gas to return to said inlet opening,
- impeller means positioned in said chamber at said inlet
- 25 opening for providing a supply of pressurized gas to said plenum means from said chamber whereby gas is caused to flow through said nozzles in said ducts to impinge upon said food product and once contacting said food product to return around said plenum and to said impeller means by paths which extend above and below
- 30 said plenum means, and means for moving said food product within said chamber relative to said duct means whereby the impingement jets formed by_____
-

the nozzles of said duct means are moved across the surface of said food product.

2. The food processing device of claim 1 wherein said support means comprises a conveyor means mounted
5 within said enclosure for moving food product within said enclosure whereby the impingement jets formed by said ducts move across the surface of said food product.

3. The food processing device of claim 1, wherein
10 said enclosure has front and rear openings therein and a generally horizontally disposed passageway there-through in alignment with said openings, and additionally comprising a conveyor means receivable through said passageway for conveying food products to be processed
15 through said enclosure.

4. The food processing device of claim 1, additionally comprising a generally annular collar on the plenum disposed around the inlet opening, and

a shroud having a surface in close proximity to
20 said collar for directing return gases into said collar.

5. The device of claim 4, wherein said collar has cylindrical portion which extends axially relative to the axis of rotation of said impeller means.

6. The food processing device of claim 5, wherein
25 said impeller means comprises a generally circular plate member, said plate member having a diameter less than the diameter of the inlet opening of the plenum, a plurality of blades annularly disposed on said plate member, and means for rotating said plate member.

30 7. The device of claim 6 additionally comprising an axially extending surface on said plate member.

8. The device of claim 7, wherein said ducts taper in cross section in a direction away from said plenum.

9. The device of claim 1 additionally comprising
35 a heating means for heating the gases within said enclosure.

10. The device of claim 1, additionally comprising
a cooling means for cooling the gases within said
chamber.

11. An impingement device for processing food products
5 comprising:

an enclosure,

duct means mounted within said enclosure and being
generally vertically aligned with and spaced apart
from said support means for directing therethrough

10 heated air for subsequent exiting through a plurality
of jets in said duct means and impingement on the
food product to be cooked supported on said support
conveyor means,

a plenum disposed within said enclosure and connected
15 to said duct means to provide heated air to said duct
means, said plenum having a front portion with a plurality
of openings therein in fluid communication with said
duct means and having a rear portion with a rear opening
therein,

20 a generally annular collar on said rear portion
of said plenum disposed around said rear opening,

impeller means positioned within said collar
for drawing heated air into said plenum through said
rear opening, said collar having an annular surface
25 which extends axially relative to said impeller means,

said plenum being spaced inwardly from said enclosure
so as to permit the air which has flowed out of said
jets to flow around said plenum and then through said
collar and into said plenum,

30 a shroud having a surface which is in close proximity
to and at least partially rearwardly of said collar
for directing the air into said collar, and

means for heating the air in said enclosure.

12. An impingement device for processing food products
35 comprising:

an enclosure having an accessible interior and
a generally horizontally disposed passageway therethrough,

conveyor means receivable through said passageway,

duct means mounted within said enclosure and being generally vertically aligned with and spaced apart from said conveyor means, said duct means and said

5 conveyor means defining therebetween a cooking chamber for receiving from a plurality of jets in said duct means heated air impinging on the food conveyed by said conveyor means,

said duct means and a front portion of said enclosure
10 defining therebetween a return chamber in communication with said cooking chamber for receiving return air therefrom,

a plenum attached to and in communication with said duct means for providing heated air to said duct
15 means, said plenum having a front portion with a plurality of openings therein in fluid communication with said duct means, said plenum having a rear portion with a rear opening therein,

said plenum and a back portion of said enclosure
20 defining therebetween a heating chamber in communication with said plenum through said collar,

means for heating air in said chamber,

a generally cylindrical collar disposed about said plenum rear opening,

25 impeller means being positioned between said heating chamber and said plenum and within said collar for rapidly drawing heated air from said heating chamber to said plenum, said collar having an annular surface extending axially relative to said impeller means,

30 a shroud having a surface in close proximity to and at least partially rearward of said collar for directing the air into said collar, and

control means for regulating the temperature of said heating means, whereby heated air is forced through
35 said jets into said cooking chamber for impingement on the food conveyed by said conveyor means, the air then being drawn through said return chamber to said heating chamber, and then drawn by said impeller means

through said collar and said plenum into said duct means.

13. An impingement device for processing food products comprising:

an enclosure defining a chamber for receiving a
5 food product therein,

means for supporting a food product within the chamber,

a plurality of duct means mounted in said chamber, nozzles in each of said duct means to form discrete
10 streams of gas which will not be diffused prior to impingement against the surface of the food product, at least one duct means positioned adjacent opposite sides of said food product supporting means for directing gas through said nozzles in said duct means to impinge
15 upon opposite sides of a food product on said supporting means,

means to move said support means relative to said nozzles to move a food product relative to the discrete streams to move points upon which the streams impinge
20 relative to the streams,

a plenum in said chamber for enclosing a reservoir of gas, said plenum being poistioned adjacent an edge of said support means and having said duct means connected thereto, said plenum having one side adjacent the food
25 supporting means with openings therein in communication with said duct means and having another side with an inlet opening therein, said plenum being spaced inwardly from said enclosure whereby a flow path for diffused gas in said chamber is formed around two sides of said
30 plenum to allow gas to return to said inlet opening, and

impeller means positioned in said chamber at said inlet opening for providing a supply of pressurized gas to said plenum means from said chamber whereby gas is caused to flow through said nozzles in said
35 ducts to impinge upon said food product and once contacting said food product to diffuse and return around said plenum and to said impeller means by paths which extend around opposite sides of said plenum means.

14. The impingement duct of claim 13 additionally comprising a chamber having an inlet passage at one end and a delivery passage at the opposite end; said means to move the support means including a conveyor to move
5 a food product through the inlet passage, through the chamber between the duct means, and through the delivery passage.

15. The impingement duct of claim 14 wherein said impeller means comprises a fan to direct gas in a
10 generally axial direction and with the addition of means in said plenum to split a stream of gas delivered by said fan to direct flow to each of said ducts.

16. The impingement device of claim 15, wherein said duct means comprising a plurality of ducts on each
15 side of said food product support means.

17. The food processing device of claim 13 with the addition of a deflector in said plenum, said deflector being shaped and positioned relative to said impeller to direct gas to said duct means.

FIG. 1

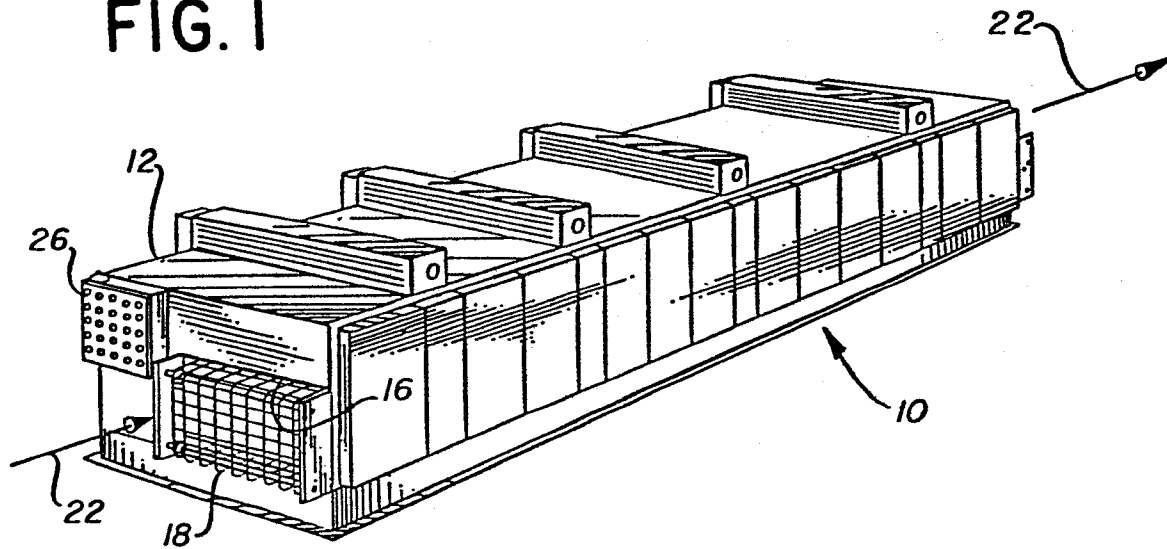


FIG. 4

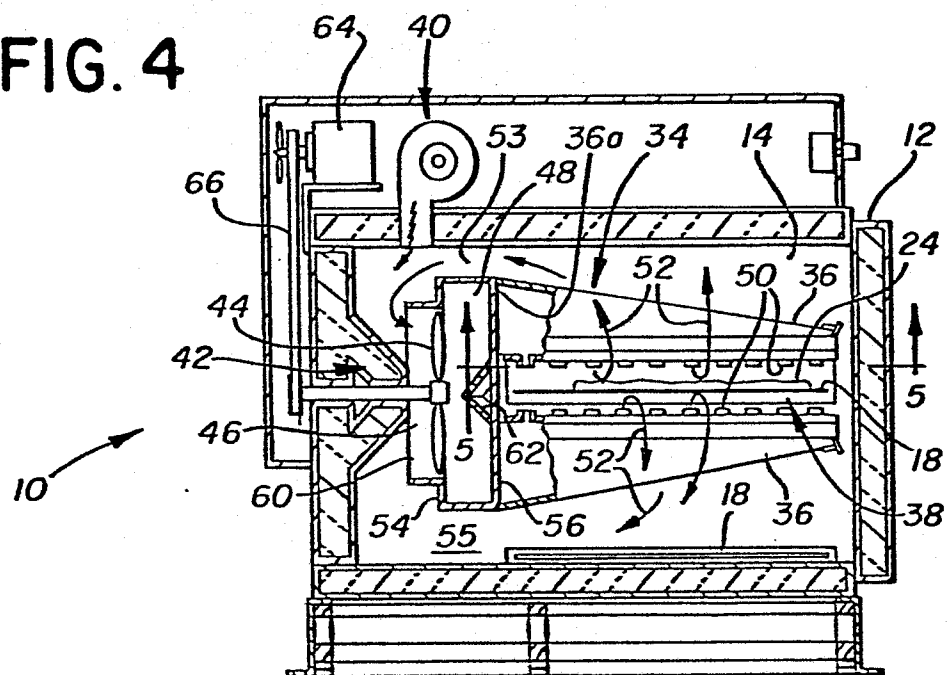
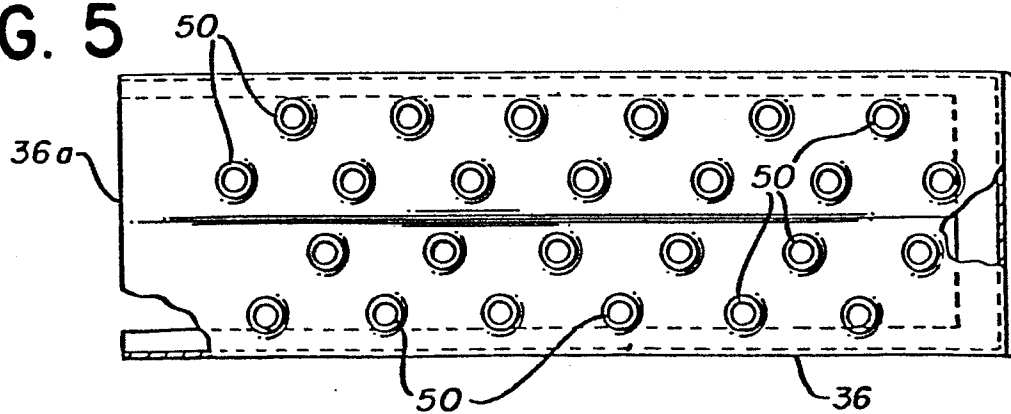
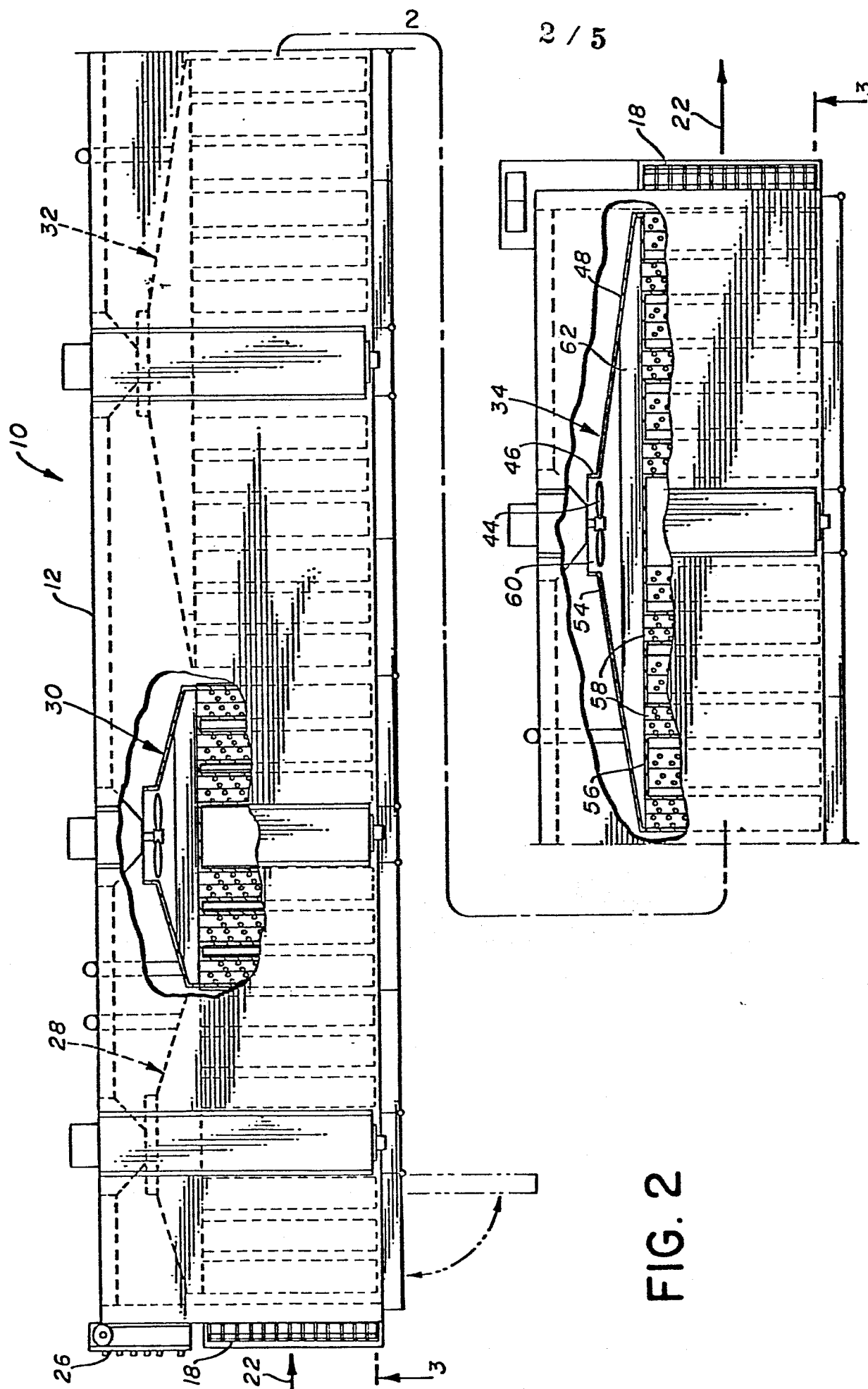


FIG. 5





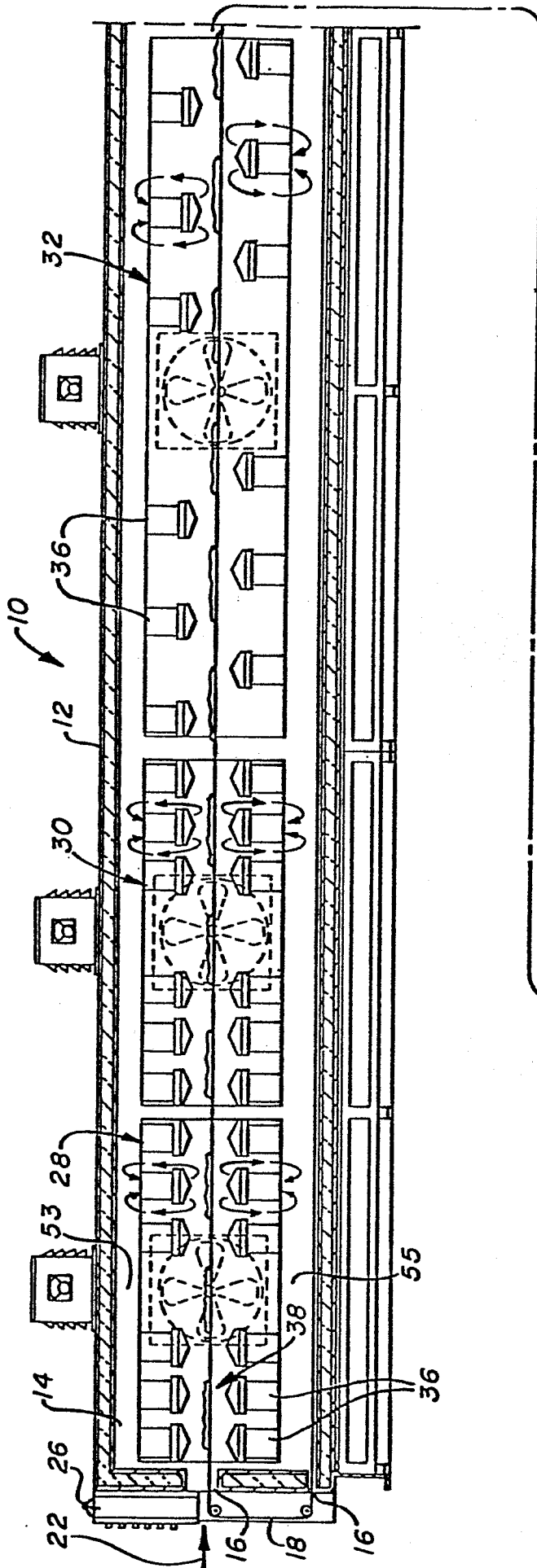


FIG. 3

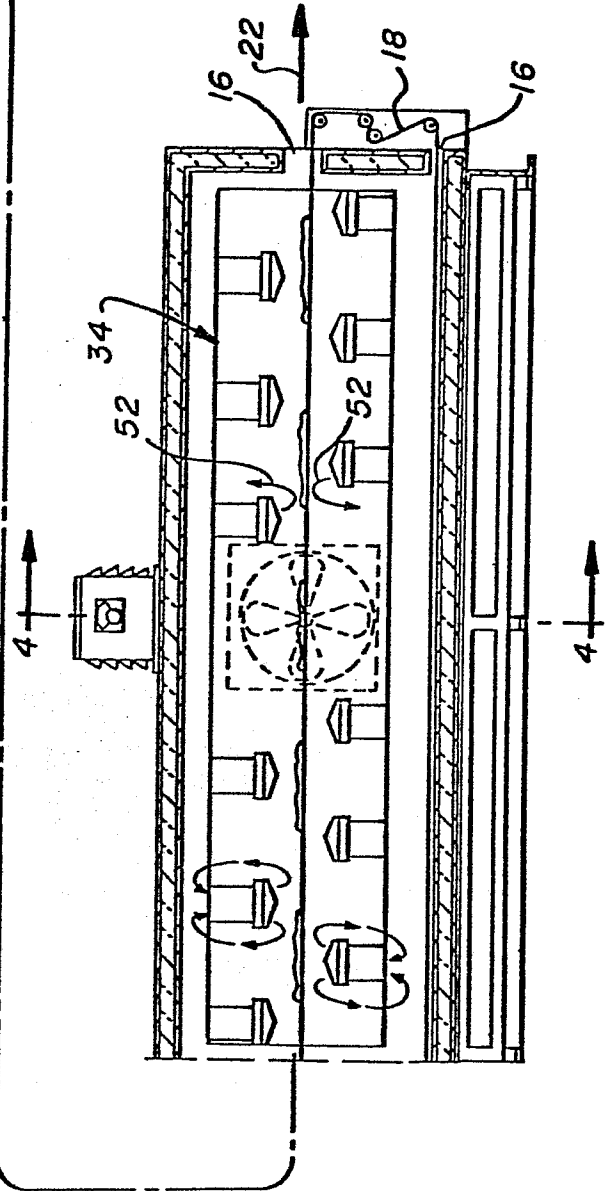


FIG. 6

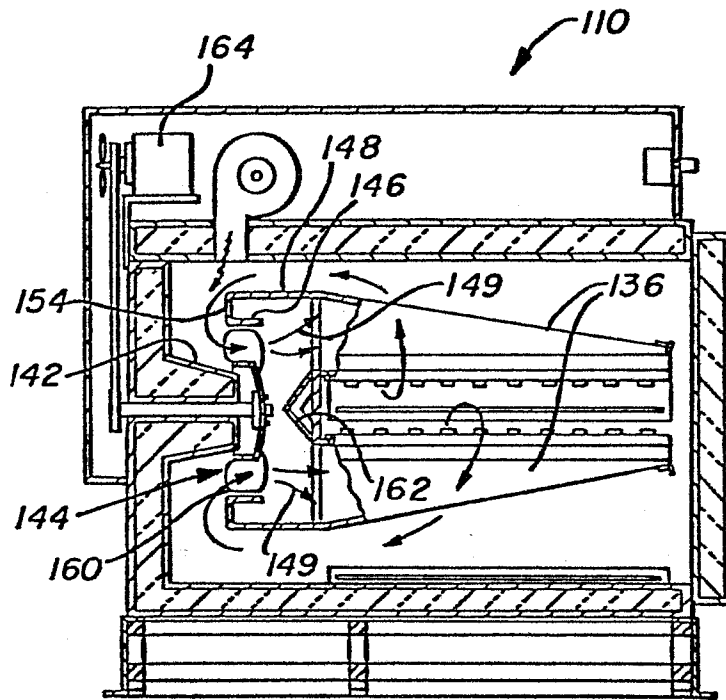


FIG. 7

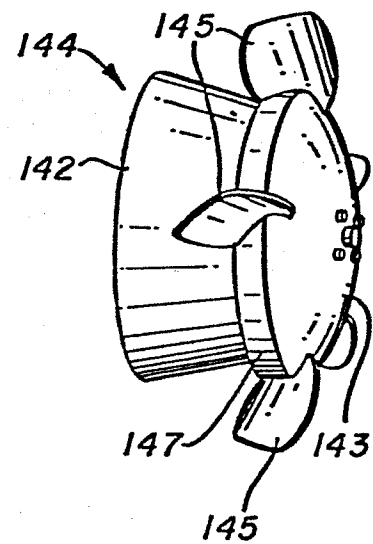


FIG. 8

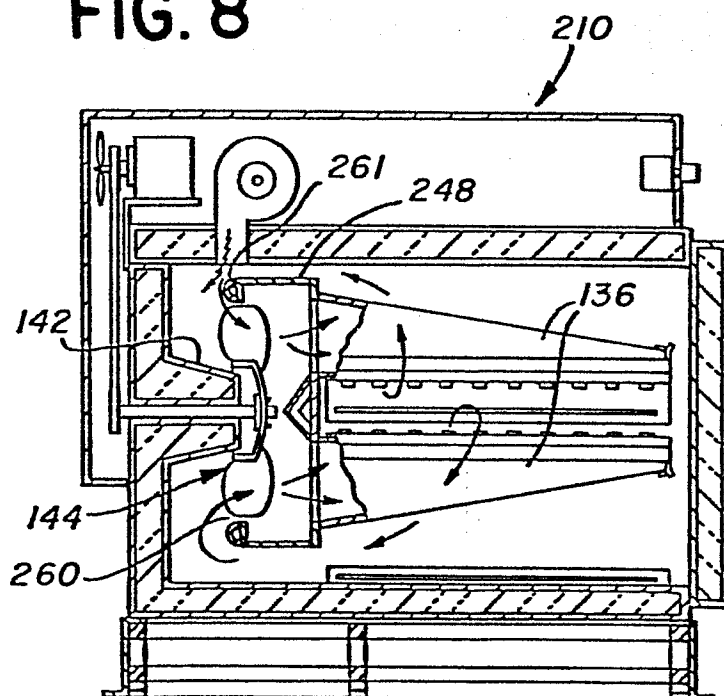


FIG. 9

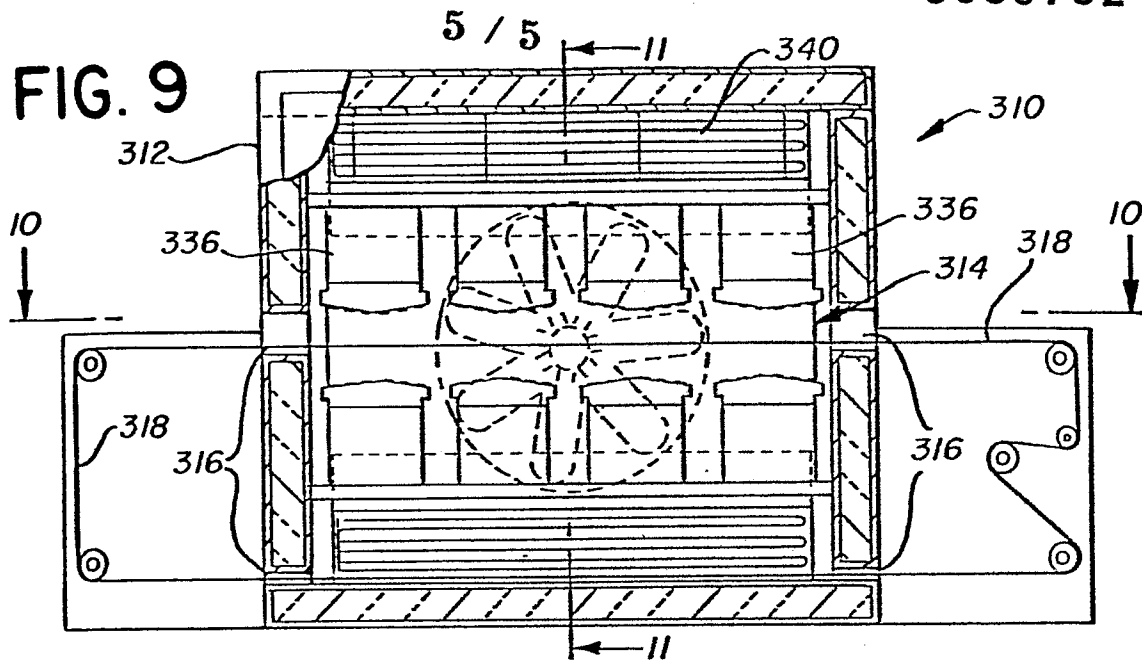


FIG. 10

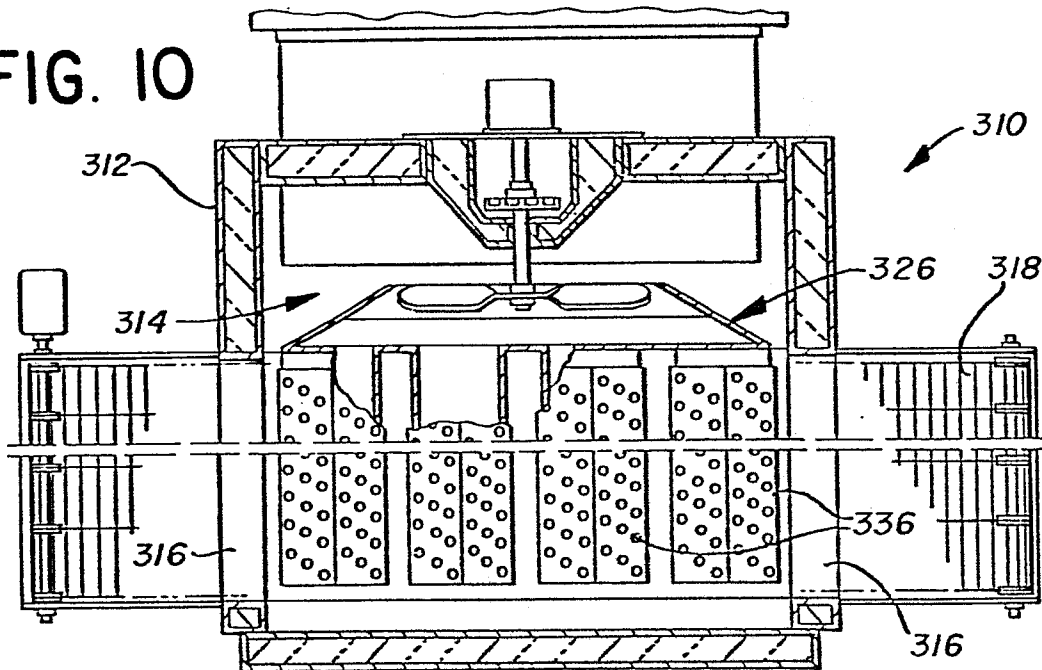
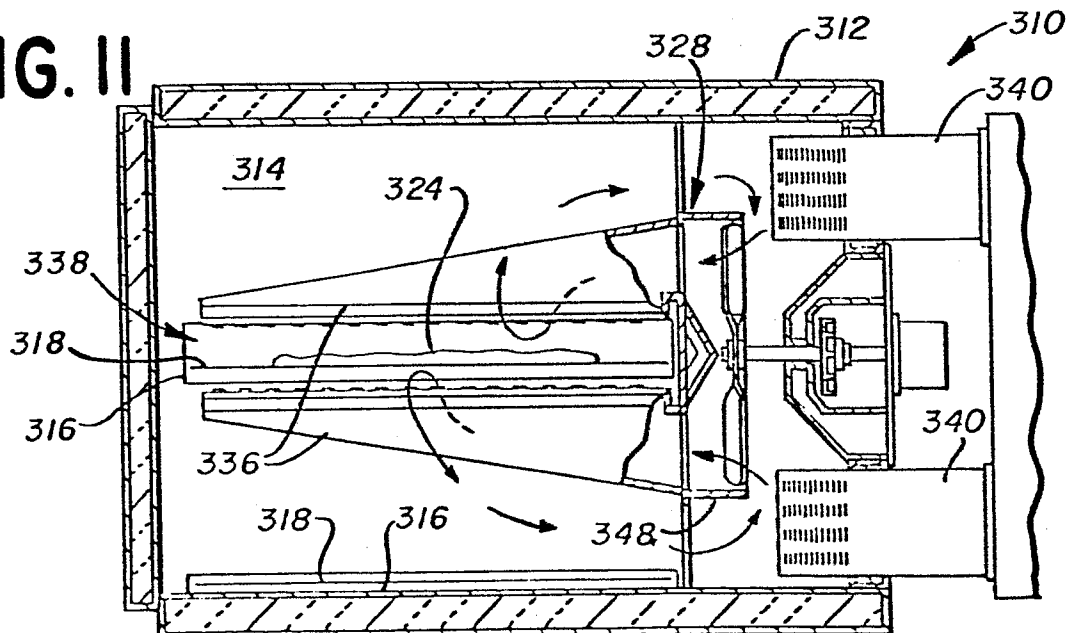


FIG. 11





DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl. 3)
D,A	US-A-3 884 213 (D.P. SMITH) * Abstract; figure 2 * ---		A 23 L 3/36 A 23 L 3/18
D,A	US-A-4 154 861 (D.P. SMITH) * Abstract; figure 2 * ---		
A	FR-A-2 201 044 (AIR PRODUCTS AND CHEMICALS INC.) * Figure 2; claims 1, 12, 13 * -----		
			TECHNICAL FIELDS SEARCHED (Int. Cl. 3)
			A 23 L 3/00 F 24 C 15/00
The present search report has been drawn up for all claims			
Place of search BERLIN		Date of completion of the search 26-05-1983	Examiner SCHULTZE D
CATEGORY OF CITED DOCUMENTS			
X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document		T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document	