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Machine for binding books and punching paper.

Apparatus for binding together apertured sheets (21) uses a first strip (24), a plurality of thermoplastic studs (23) projecting from and spaced longitudinally along the first strip, and a second strip (26) formed with holes spaced longitudinally of the second strip at intervals complementary to the studs. The apparatus has a frame having a pair of spaced sides (33) and a platen (86) to support the sheets with their apertures aligned with the holes of the second strip and with the first strip above the sheets with the studs extending through the apertures in the sheets and the holes in the second strip and protruding below the second strip, a pressure bar (106), cooperating means (108, 109) on the sides and bar for guiding movement of the bar from a first position above the platen to a second position adjacent the platen and resting on top of the first strip, actuating means for moving the pressure bar, a motor (116) for driving the actuating means, and a switch (113) for energising the motor. The characterising feature of the apparatus is that the switch is mounted on the bottom edge of the pressure bar and is positioned and arranged to be closed when the pressure bar rests on the first strip so to energise the motor and begin the binding cycle.

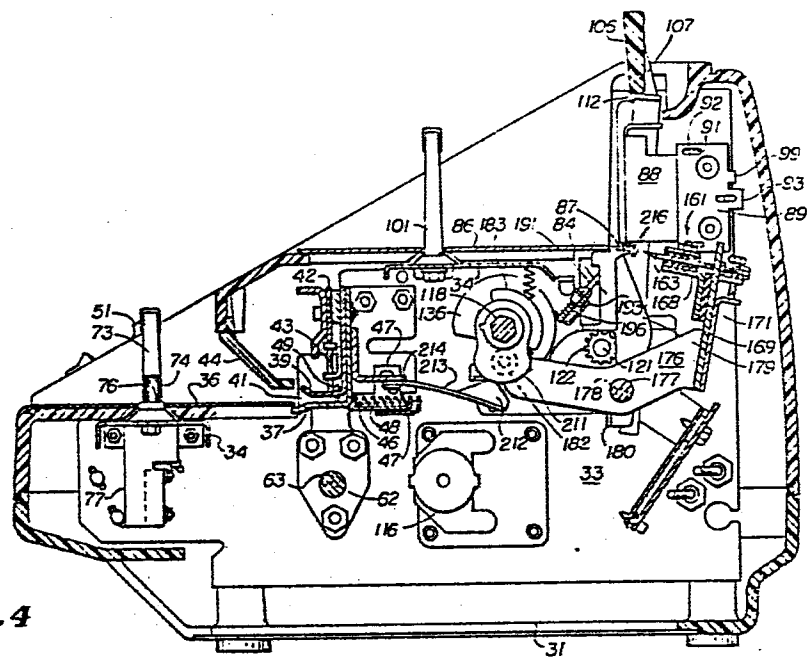


Fig. 4

MACHINE FOR BINDING BOOKS AND PUNCHING PAPER

This invention relates to a new and improved punching
and binding machine which comprises improvements on
5 commercially available machines, such as the machine
to which the applicant's earlier US Patent No. 3811146
relates, being apparatus for binding apertured sheets
together using a first strip, a plurality of thermoplastic
studs projecting from and spaced longitudinally along
10 the first strip, and a second strip formed with holes
spaced longitudinally of the second strip at intervals
complementary to the studs, the apparatus having a
frame having a pair of spaced sides and a platen to
support the sheets with their apertures aligned with
15 the holes of the second strip and the first strip above
the sheets with the studs extending through the apertures
in the sheets and the holes in the second strip and
protruding below the second strip, a pressure bar,
cooperating means on the sides and bar for guiding
20 movement of the bar from a first position above the

platen to a second position adjacent the platen and resting on top of the first strip, actuating means for moving the pressure bar, a motor for driving the actuating means, and a switch for energising the motor.

5 The object of the present invention is to render the mode of operation of such apparatus simpler and quicker, and for achieving this object the invention is characterised in that the switch is mounted on the bottom edge of the pressure bar and is positioned
10 and arranged to be closed when the pressure bar rests on the first strip so to energise the motor and begin the binding cycle.

 This application is a divisional of Application No. 81302180.5 which relates to apparatus for binding
15 apertured sheets together using a first strip, a plurality of thermoplastic studs projecting from and spaced longitudinally along said first strip and a second strip formed with holes spaced longitudinally of said second strip at intervals complementary to
20 said studs, said apparatus being of the type having a frame having a pair of spaced sides and a platen to support said sheets with their apertures aligned with said holes with said first strip above said sheets with said studs extending through said apertures and said

holes and protruding below said second strip, a
pressure bar, cooperating means on said sides and bar
for guiding movement of said bar from a first position
above the platen to a second position adjacent said
5 platen and resting on top of said first strip, actuating
means for moving said pressure bar having a shaft
transverse to said machine and a motor driving said
actuating means characterised by a first segmental
gear fixed to said shaft, a first link connected to
10 said pressure bar and a crank oscillatable on said
frame, a second segmental gear fixed to said crank and
engaging and being oscillatable by said first gear
for only a short arc of the rotation of said first
gear, a linkage including a second link pivotted to
15 said frame; said crank having means to engage
said second link when said second gear is turned
to pivot said second link and thereby lift said
first link and lift said pressure bar to said
first position.

Other objects of the present invention will become apparent upon reading the following specification and referring to the accompanying drawings in which similar characters of reference represent corresponding parts
5 in each of the several views.

In the Drawings:

Fig. 1 is a top plan view of the apparatus with the upper exterior casing and certain parts removed for clarity purposes.

10 Fig. 2,3,4 and 5 are sectional views taken substantially along the lines 2--2, 3--3, 4--4 and 5--5, respectively of Fig. 1.

Fig. 6 is an enlarged view of a portion of Fig. 3 showing the pressure bar return mechanism in various
15 positions of its cycle of operation.

Fig. 7 is a fragmentary perspective view showing a portion of the bind backstop and associated mechanism where a short stack of papers is to be bound.

Fig. 7A is a view similar to Fig. 7 showing a taller
20 backstop.

Fig. 8 is a fragmentary sectional view taken substantially along the line 8--8 of Fig. 7.

Fig. 9 is a fragmentary plan view showing the knife structure in retracted position and the bridge.

25 Fig. 9A is a fragmentary plan view of a heater

element for the knife structure with portions broken away to reveal internal construction.

Fig. 9B is a fragmentary side elevational view of a portion of the structure of Fig. 9A.

5 Fig. 10 is an enlarged sectional view taken substantially along the lines 10--10 of Fig. 9.

Fig. 11 is an enlarged sectional view taken substantially along the line 11-11 of Fig. 9.

10 Fig. 12 is a fragmentary enlarged sectional view taken substantially along the line 12--12 of Fig. 11.

Fig. 12A is a fragmentary sectional view taken substantially along the line 12A--12A of Fig. 11.

15 Fig. 13 is a fragmentary sectional view taken substantially along the line 13--13 of Fig. 9 showing a normal size binding strip.

Fig. 14 is a view similar to Fig. 13 showing use of a wider than standard size binding strip used when paper sheets contain apertures farther distances from the edges.

20 Fig. 15 is a view taken along lines 15--15 of Fig. 1.

Fig. 16 is a schematic simplified wiring diagram of the machine.

25 Fig. 17 is a fragmentary sectional view similar to a portion of Fig. 4 showing modification of the structure for wrap-around cover fabrication, the knife

structure being in retracted position.

Fig. 18 is a view similar to Fig. 17 showing the knife structure in forward and raised position and showing the wrap-around cover support elements in a
5 somewhat different position from those shown in Fig. 17.

Fig. 91 is a fragmentary view of a cooling finger and its environment.

General Description of Machine

As is set forth in prior Patent 3,811,146 among
10 others, the present invention is intended to punch and bind paper 21 formed with spaced apertures 22 along one spine edge thereof. In accordance with the present invention, provision is made to locate the apertures 22 at either of two distances from the spine edge of
15 the sheets of paper 21. If the sheets 21 are standard computer print-out sheets, the apertures 22 are spaced a greater distance from the spine edge of the paper than for ordinary or standard document binding. As appears in the discussion of the punching mechanism
20 hereinafter set forth, provision is made for spacing the apertures at either of two distances.

In accordance with the present invention, a first strip 24 of plastic having studs 23 spaced at the same intervals as the apertures 22 is provided. The
25 strips 24 may be of two widths depending upon the

spacing of the apertures 22 from the edges of the sheets 21. There is also used in accordance with this system of binding a second strip 26 formed with holes 22 and counterbores or countersinks 28. Typical
5 specifications for the material, sizes, etc., of the strips 24 and 26 are set forth in Patent 3,811,146.

The machine of the present invention comprises a base 31 on which fits an ornamental molded casing 32 which houses the mechanisms hereinafter described.
10 Within the sides of the casing 32 are vertical side plates 33 which are the main supports for most of the mechanisms hereinafter set forth. The side plates 33 are interconnected by various transverse, horizontal cross members 34.

15 The machine heretofore described performs two functions. The first is to punch apertures 22 in the paper 21. The second is to bind the paper 21 between the strips 24,26. In a preferred embodiment of the invention, both the punching and binding functions
20 are located within the same casing 32, but it will be understood that the two functions of the machine may be installed in separate machines.

Punching Mechanism

Extending horizontally, transversely near the
25 proximal end of the machine and at a relatively low

elevation, is a punch platen 36 on which the papers 21 are supported and positioned relative to the punch throat 41. Behind the inner edge of the platen 35 is a die plate 37 formed with holes corresponding to the spacing of the apertures 22. Plate 37 is located immediately rearwardly of the rear edge of the platen 36 and is horizontally aligned therewith. Vertical member 38 is an integral portion of the punch mechanism which also functions as a transverse, horizontal cross member 34 and is fixed to the side plate 33, and this carried a punch guide plate 39 which extends horizontally forwardly and then slants upwardly forwardly providing an entrance chute and provides a throat 41 between the plate 39 and the die plate 37. Punch plate 42 is vertically reciprocal relative to member 38 as hereinafter explained. A plurality of pins 43 corresponding in number to the holes in the die plate 37 are retained in punch plate 42 as is described in Patent 4,079,647. Forwardly of plate 42 is a downwardly-rearwardly slanted paper deflector 44 which guides and deflects papers as they are inserted into the machine toward the throat 41. There is a vertical gap between the lower edge of plate 39 and 37 and horizontally reciprocal therein is a slide 46 which rests upon a slide support 47 fixed to side plates 33. Slide 46 is retained

and guided by studs 29 fixed to die plate 37. Spring 48 biases the slide 46 rearwardly. Actuators 72 of switches 71 on slide 46 extend into throat 41 and trigger the electrical circuitry to actuate the punching mechanism when it is moved by the inward insertion of sheets 21. The stop means 49 is located on slide 46. By means hereinafter described, the slide 46 may be moved forwardly and rearwardly to control the spacing from the spine edge of the paper 21 to the apertures 22.

10 Extending through a suitable window in the casing 32 is a selector slide 51 which may be manually moved between two positions. The slide 51 is connected to a flexible cable 52 which extends to a lever 53 which is pivoted to a pin 54 on die plate 37. As the slide 51

15 is moved forward and rearwardly, the cable 52 causes a pivoting of lever 53. A forward extension 56 of slide 46 has a down-turned forward terminus in which is caught one of the arms of lever 53. Thus, the slide is moved forwardly and rearwardly to control the spacing

20 of the apertures 22 as hereinafter explained.

 The punch may be manually actuated by depressing control switch 238 or set by switch 237 to automatically trigger electrical circuitry when actuator 72, of switch 71, is moved by sheets of paper. The punch mechanism is

25 motor driven by punch motor 61 which has extended shafts

62 having eccentric crank 63 on both ends. A vertical lever 64 has an aperture in its end which receives the eccentric crank 63. At the upper end of lever 64 is an elongated slot 66. An outward directed ear 67 is formed in punch plate 42 and is received within slot 66. Hence, as the motor 61 turns, the eccentric crank 63 causes the vertical lever 64 to reciprocate and thus to reciprocate the punch plate 42 and the pins 43 which are aligned with the holes in the die plate 37. Hence, paper in the throat 41 having its spine edge against the stop 49 is properly punched. Member 50 retains lever 64 in position.

In one form of the invention, the punch motor 61 is manually controlled by a switch. However, preferably, the punch may be made to perform automatically.

Thus, slide 46 carries two switches 71 having forwardly extending switch actuators 72 which extend into the throat 41 at its rearward end. The positions of the switches 71 and actuators 72 is dependent upon the position of slide 46 under the control of selector slide 51.

An edge guide is located on the left side of the machine (although, of course, it might be located on the right side). This guide 73 is vertically upstanding and may be moved inwardly and outwardly relative to the

center line of the machine to adjust for different lengths of sheets. The details of this adjustment are not herein illustrated or described since they are old in the art. The inward-facing edge of guide 73 is formed with an opening 74. Within the opening 74 is a third switch actuator 76. As shown in Fig. 4, actuator 76 is sloped downward at a 45° angle forming a sharp point, while also being twisted at a 45° angle on its top edge. Thus, if paper is fed from the top, it will not jam or hang up on the edge of actuator 76. The tip of the actuator will be moved inside the opening 74. The actuator 76 is twisted 45° so that if paper is inserted from the front, it will not jam against the actuator but in both instances will move the actuator inward to actuate switch 77. If both switches 71 and switch 77 are closed, it is insured that the paper 21 is fully inserted in the throat 41 against the stops 49 and that the edge of the paper is located properly against the edge guide 73. Closing of all three switches assures paper length is parallel to pins 43 and perpendicular to other edges of paper and energizes the motor 61 and causes reciprocation of the lever 64 and a punching cycle.

Preferably, there is a cycle cam 78 on an eccentric crank 63 on one side of the machine which controls

switch 79 and turns off the motor 61 after one cycle of one revolution of the motor shaft.

Binding Mechanism

As has previously been stated, the binding mechanism
5 of the present machine resembles, in many respects, that of Patent No. 3,811,146, and where the elements of the present machine are substantially the same as those of the preceding machine, they are not herein described in detail.

10 Extending horizontally across the widths of the machine and supported by cross-members 34 is a binding platen 86, having at its rearward edge a depression or recess 87, at the level of the top of transverse bridge 84 which is secured at opposite ends to the side-plates
15 33. This depression 87 is shaped to receive strip 26. Rearward of depression 87 is a stop 88 which limits inward movement of sheets 21 supported by platen 86. As has been stated, the present machine may be provided with parts which are interchangeable to accommodate
20 different heights of stacks of paper, such as, for example, a two-inch maximum stack and a three-inch maximum stack. Therefore, the stop 88 is a changeable part of the machine.

Stop 88 has a rearward and outward extension 89 formed
25 with horizontal slots 91. Pins 92 fit through the slots

91 and are anchored in side plates 33. An ear 93 on extension 89 extends behind the rear edge of plate 33 and receives a spring 94 which is connected at its opposite end to spring anchor 96 on the side of plate 33 opposite extension 89. The function of spring 94 is to bias the stop 88 forward so that pressure is applied to strip 26 when it is placed in depression 87 and so that the tabs 99 (an extension of 88) contact side plate 33 to limit its forward movement. This is the normal position of the stop 88. As is described in the Punching Mechanism heretofore set forth, for certain types of sheets, the apertures 22 are punched at a greater distance from the spine edge of the sheets. When the latter sheets are used, a wider strip 26 is used than normal. Placing such a wider strip in the depression 87 forces the stop 88 rearwardly against the force of the springs 94. Hence, when the sheets 21 are punched rearwardly to contact the stop 88, the holes therein are in proper relationship to the holes in the strip 26.

For purposes hereinafter described in detail, it will be noted that there is formed a depression 98 on top surface of 88 to provide clearance for the switch plunger, so it will not accidentally actuate the electrical circuitry to start the binding cycle.

Edge guide 101 on platen 86 is transversly moveable to accommodate different widths of sheets 21.

The first step in the operation of the binding mechanism, therefore, is to place a strip 26 of proper width, depending upon the positioning of the apertures 22 for the sheets 21 with which it is to be used, into the depression 87, thereby positioning backstop 88 correctly. Locating pin 102 (see Fig. 14) fits into a hole (not shown) in strip 26 to locate strip 26 laterally in depression 87. A stack of sheets 21 is then placed on the platen 86 against the edge guide 101 and against stop 88, thereby aligning the apertures 22 with the holes 27 in strip 26. Second strip 24 is then installed by inserting studs 23 through apertures 22 and 27.

Pressure Bar

A transverse horizontal, vertically moveable pressure bar 106 is provided. Again, to accomodate different maximum height stacks of sheets, the pressure bars 106 are interchangeable. Such interchangeability is preferably provided by using different brackets 107 which may be an integral part of pressure bar 106 and which are mounted to rack 110 on each side of the machine. A roller 108 fits into a slot 109 in side plate 33 and, at the top of the slot 109, there is a rearward offset

111. When the roller 108 is in the offset 111, the pressure bar 106 is held in upward position. However, when pressure bar 106 is lifted and moved forwardly so that the roller 108 slides down the slot 109, the bottom edge 112 of pressure bar 106 is brought into contact with the top of strip 24. Recessed into bottom edge 112 is a switch 133 which, when its actuator contacts strip 24, initiates the binding cycle of the machine. The function of depression 98 in the flange surface 97 is to prevent closing of switch 113 when the pressure bar 106 is in retracted position, with roller 108 in the offset 111.

Binding motor 116 is mounted on side plate 33 and, by means of a belt-drive 117, is connected to drive transverse horizontal hexagonal camshaft 118. Shaft 118 is supported between the plates 33 by bearings (not shown) and is held in place by snap ring retainers 119 at either edge.

Horizontal transverse floating shaft 121 carries pinions 122 which mesh with the teeth of the racks 110. Also on shaft 121 is ratchet wheel 123 which is engaged by pawl 124, biased into engagement with the ratchets 123 by springs 126. Pawl 124 is pivoted on pivot 127 to spring anchor 121. Shaft 121 is supported on either side of the machine by spring anchors 131, pivoted on pivots 132 which are mounted to the plates 33. On the forward ends

of spring anchors 131 are flat springs 133 which carry cam followers 134 which are engaged by pressure cams 136 on shaft 118.

As shaft 118 revolves, follower 135, which engages
5 pressure cam 136, is increasingly depressed, thus pulling rack 110 downward and causing pressure bar 106 to be forced against strip 24 until a predetermined pressure is reached, whereupon spring 133 flexes, and despite continued turning of cam 136, no greater
10 pressure is applied to the strip 24.

To prevent unacceptable variations in acoustical noise levels by applying varying loads to the D.C. bind motor 116 as each cam performs its function as they rotate through the binding cycle, there is provided on
15 shaft 118 a counter-pressure cam 211 which assures uniform loading onto motor 116. Cam follower 212 is mounted on spring 213 and fastened by attachment 214 to one of the cross-members 34. The follower 212 exerts a pressure on the cam 211 which counter-balances the forces imposed
20 on the system by the binding mechanisms.

One novel feature of the present invention is the provision of means which automatically returns the pressure bar 106 to upper position. Return arm 141 is angular and is pivoted at pivot 142 to side plate 33.
25 On its forward end there is a gear segment 143

consisting of approximately two teeth. Fixed for movement with shaft 118 is a meshing gear segment crank 144, also having approximately two teeth. Adjustably positioned on the lower end of return arm 5 141 is a roller 146. The rest position of arm 141 is determined by stop 147. Pivoted to plate 33 is a first pressure bar link 148 which is pivoted to second pressure bar link 149 by pivot 151. The upper end of link 149 is connected to the pressure bar by the same means as 10 connects roller 108. There is a stop 152 fixed on plate 33 which limits clockwise movement of link 149 as viewed in Fig. 3.

As the binding cycle is completed, rotation of shaft 118 causes the driver gear segment 144 to mesh with 15 gear segment 143, and this causes counter-clockwise rotation of return arm 141 about pivot 142. Roller 146 engages link 148 and oscillates it around the shaft of gear segment 143, and this causes link 149 to raise the pressure bar to upper position and drop the roller 108 20 into offset 111, causing pressure bar 106 to remain at rest positioned until the operator initiates the next binding cycle of the machine. Meanwhile, continued rotation of shaft 118 brings the gear segments 143 and 144 out of mesh with each other. Whereupon, return arm 25 141 drops to initial position with its lower end resting

on stop 147.

Carried on shaft 118 is a cam 153 which engages the actuator of switch 154 after the gear segments 143 and 144 have ceased to mesh. Switch 154 stops the rotation
5 of bind motor 116.

Cut-Off

After the sheets 21 have been clamped between the strips 24, 26, continued turning of shaft 118 causes cutting of the excess lengths of the studs 23 projecting
10 below the bottom strip 26. Mounted behind the depression 87 is a knife structure 161. Structure 161 comprises a cutter 162 having blades 163 projecting forwardly and spaced about the same distance as the studs 23. Above and below cutter 162 are heating elements 164 which are
15 resistance heating sandwiches which are elevated in temperature when current flows therethrough. Heater element 164 is composed of several layers sandwiched together. The central conductor is cut from a sheet approximately 0.007 inches thick metal such as "Inconel".
20 The pattern is composed of right angle bent, structures, at best shown in Fig. 9A, consisting of ends 155 parallel to the length of cutter 162 interconnected by longitudinal stretches 156 transverse thereto. To make provision for the bolts which bolt knife structure 161
25 together, holes 158 are formed in the heater element 164

and there are extended sections 157 to accommodate the space for the holes 158. On the top and bottom of the members 115-157 are sheets of asbestos cloth 159. On the outsides of the sheets 159 are sheets of mica 172 of a thickness of about 0.015 inches. As shown in Fig. 9B, on the side of the heater elements 155-157 closest to cutter 162 there is but one sheet 172 whereas on the opposite side there are several sheets 172. On each end of element 164 there is an outward extension 173 of nickel bonded to the outermost stretch 156. Extension 173 is surrounded by a glass fiber sleeve 174 which surrounds also the end of the conductor 175 and the connector between the conductor and element 173.

In a preferred heater element 164, at either end the widths of 155 and 156 are less than they are at the center. Thus at the end shown in Fig. 9 the stretches 156 are approximately 0.052 inches in width and the spacing between these widths is about 0.042 inches. At the center, however, the width of the member 156 is 0.054 inches whereas the spacing between the members 156 remains 0.042. The width change, although slight in dimension, alters the electrical resistance of the metal and consequently the heat or watt density. Hence more heat is generated at the ends of the sandwich but there is more heat loss at the ends. Thus, there is a balancing which

creates a uniform temperature across the entire length of the knife blades 163.

The cutter 162 temperature is sensed by means of a thermistor 165 that is inserted within the cutter and is retained under tension by spring wire 160. Above and below each heating element 164 is an insulator 166, and on top of top insulator 166 is a cover 167. Below the lower insulator 166 is a support bar 168 to which is attached a support bracket 169 insulated therefrom by a high temperature insulator 171. Each support bracket 169 is connected to pivot arm 176 which is pivoted about transverse horizontal eccentric shaft 177. The outer ends 178 of shaft 177 are received in brackets 179, dependent from bridge 84 and held in place by keeper plate 180. On the forward ends of pivot arm 176 are cam followers 182 which engage cutter cams 183 on camshaft 118. The cams 183 cause the knife support bar 168 to move about shaft 177 as a center and cut off the excess lengths of the studs 23.

To prevent build-up of plastic particles on the blade 163, a pair of vertically-spaced apart, transversely-extending wiper-wires 216 is mounted on holders 217 affixed to bridge 84. The blades 163 reciprocate between the wires 216, and the latter wipe off any accumulation of plastic particles. The wipers 216 are

supported and tensioned by coil springs on either end. The wipers are of flexible plastic tubes with non-sticking surface characteristics loosely encasing wires 216. This enables a relative rotating motion of the 5 tubes with the wires that assures a more positive wiping and cleaning action.

Head Forming

Extending forwardly from shaft 177 is lift arm 186 carrying at its forward end follower 187 which engages 10 lift cams 188 on shaft 118. The function of cam 188 is to cause the support bar 168 to be raised to deform the heated ends of the severed studs 23 and initiate the forming of heads thereon.

Cooling fingers 191 correspond in spacing transversely 15 of the machine to studs 23. Each finger 191 has a first non-circular cross-section lower end 202, a circular cross-section portion 205 above portion 202, a second non-circular cross-section portion 192 above portion 205, a reduced diameter circular section 203 above portion 202 20 and an enlarged cross-section upper portion 204 above portion 203. The surface at the upper end 205 is angular to the center line of finger 191.

Channel 193 reciprocates upwardly-rearwardly from the retracted position of Fig. 11 to a raised position with 25 angular surface at the upper end 205 immediately below

recess 87 in bridge 84 which is suitably apertured for passage first of blades 163 and then of fingers 191.

The lower flange 221 of channel 193 is formed with first non-circular holes 223 complementary to ends 202. The
5 upper flange 222 is formed with second holes 194 complementary to second portion 192. Springs 196 between flanges 221 and 222 surround fingers 191.

By inserting end 202 first through hole 194, then through spring 196 and then through hole 224 (as section
10 192 fits through hole 194) until section 203 is even with second hole 194 and then twisting finger 191 through 90° , lower end 202 locks behind flange 221. Spring 196 bears against section 192 and biases finger 191 upwardly-rearwardly. If finger 191 encounters resistance,
15 spring 196 flexes so that finger 191 yields with a reciprocating sliding motion, rather than the finger remaining immobile and breaking. Spring 196 restores finger 191 to projected position when resistance to movement has been removed.

20 Extending forwardly from each end of channel 193 is bracket 197 which carries on its forward end cam followers 198 which engage cooling cams 199 on camshaft 118. As the cam 199 rotates, after the cutter blades 163 have been raised and lowered by reason of the shape of cams 188, the
25 channel 193 is raised, causing the angular surface at

the upper ends 205 of cooling fingers 191 to engage and set the heads at the ends of the studs and cool the same so that the strips are locked together a fixed distance apart. Bifurcations 201 extending downward
5 from brackets 197 straddle and slip into grooves in shaft 118 to maintain the positioning of fingers 191 within slots in bridge 84.

The individual cams 136, 153, 183, 188, 199, 211 are formed with hexagonal holes through their hubs to
10 receive shaft 118 and are located laterally by suitably-placed E-clips 120 fitting in grooves in shaft 118. By removing end clips 119 and the clips 120, the shaft 118 may be pulled axially outwardly, the cams dropping off. To facilitate proper replacement, each cam hub 136, 153,
15 183, 188, 199, 211 has either a projection or depression 215 and these are aligned as the shaft 118 is inserted, insuring proper cam positioning of their profiles.

Operation

At the commencement of operation, the operator pushes
20 the main switch 236 which is located on one of the control panels on the sides of the machine. Although per-punched paper or paper punched by another machine may be used in connection with the binding portion of the machine, it will be assumed, for the purpose of this
25 specification, that it is desired to punch appropriate

holes 22 in the sheets 21. One determination of the operator is the spacing of the apertures 22 from the spine edge of paper 21, depending upon whether regular paper or computer printout paper, etc., is being used.

5 This selection is made by means of selector slide 51 on the control panel, as has heretofore been described, and this moves the slide 46 and the switch-actuator 72 inward and outward to determine how far into the throat 41 the spine edges of papers 21 may be inserted.

10 Another decision to be made by the operator is whether punching is to be done automatically or under manual control. This selection is made by means of the punch-mode switch 237 on the control panel. Assuming that a manual mode has been selected, the operator then
15 inserts the sheets into the throat 41 resting on the platen 36 and preferably against the edge guide 73. Thereupon, the operator pushes the punch manual switch 238, and this energizes the punch motor 61 which reciprocates the punchplate 42 so that the pins 43 punch holes in the paper 21. At the end of one cycle, the
20 cam 78 actuates the switch 79 which de-energizes motor 61.

Assuming that the punch operation will be automatic, punch-mode switch 237 is properly set for such operation. The selector slide 51 is positioned for the proper
25 desired distance of the apertures from the spine of the

paper. The sheets 21 are then laid on the platen 36 and inserted into the throat 41 and against the edge guide 73. When both switches 71 and switch 77 are closed by reason of proper positioning of paper 21, the motor 61
5 is energized, and the punching cycle is performed, continuing until the cam 78 opens switch 79 and turns off motor 61.

The first binding choice of the operator involves the bind-mode switch 241 which is for standard or rapid
10 speed of binding. It is not desirable to use the rapid mode when the paper stack is high. Hence, paper height switches 243 are located with their actuators extending into the path of the pressure bar rollers 108. Unless the pressure bar has been lowered sufficiently
15 so that the rollers 108 clear the actuators for switches 243, the machine overrides the operator's selection of the bind-mode as per switch 241.

The operator inserts a proper strip 26 into the depression 87, depending upon the spacing of the
20 apertures 22 from the spine edge of the sheets 21. If the wider strip 26 is selected, the stop 88 is pushed rearward; if the narrower strips 26 are selected, spring 94 pulls the stop 88 forward. In any event, the spine edge of the paper 21 is pushed against the stop 88 and
25 also against the edge guide 101. It will be observed

from the foregoing discussion that different heights of stacks of paper may be accommodated. For example, the stop 88 may be at a height to accommodate either two-inch or three-inch stacks of paper. This selection
5 is made at the factory or by a field serviceman and is not ordinarily performed by the operator. It is assumed that an appropriate stop 88 and also an appropriate pressure bar bracket 107 have been selected and these are coordinated.

10 The operator observes the ready light 242, also on the control panel which turns on when the heaters 164 have sufficiently heated the cutter 162, as determined by a thermistor 165. Prior to the cutter 162 being heated, "wait light" 244 signals the operator that the
15 binding cycle is not operable.

Sheets 21 are placed on the platen 86 and pushed against the stop 88 and edge guide 101. The studs 23 of strip 24 are then inserted in holes 22 and 27 and strip 24 is pushed down. The operator then grips pressure
20 bar 106 and pulls it forwardly out of the offset 111 and then downwardly so that the roller 108 rides in the slot 109. When the bottom edge 112 of pressure bar 106 contacts the male strip, switch 113 is closed, and this energizes the bind motor 116 which continues to
25 cycle the camshaft 118 through one cycle.

The cycle of operation of the binding function of the machine has been heretofore described in detail in connection with the description of the various components thereof and is not here repeated. Completion of the
 5 cycle of revolution of the camshaft 118 returns the pressure bar to upper and retracted position through the operation of the elements 141-152.

Cam 153 actuates switch 154 to de-energize the bind motor 116 at the completion of the sequence of
 10 operations.

Referring to Fig. 16, the essential electrical components of the machine are illustrated schematically and in simplified form. For practical purposes the following are representative values of components
 15 illustrated:

	R 1	2.5K (max)	
	2,6	1 K	
	3	1.5K	
20	4	10 K	
	C 2	100 Micro f. (125 V)	
	5	10 "	(200 V)
	6	10 "	(25 V)
25	7	10 "	(50 V)

	Q 2	Triac
	CR 1	Bridge network
	7,8	IN 4004
5	K	Switching relays
	RT 1	Thermistor - 1 milliamp at 1 M ohm
10	A 1	CA 3059

Strips 164 - comparable to:

Cat. No. ST 1018 of Hotwatt, Inc., Danvers, Mass.

15 Cat. No. HF-1514 of Ogden, Inc., Arlington Heights,
Ill.

Cat. No. S-1405 of Chromalox, Inc., Pittsburgh, Pa.

Wherein the term "switch" is used herein, it will be
understood that various sensors may be substituted.

20 Modification

The preceding modification has as its principal
purpose the fabrication of a conventional VELO-BIND strip-
bound document. Fig. 17 shows modification of the
apparatus to accomodate a "wrap-around" cover type book.
25 As shown in Fig. 18, sheets 21a are provided front and

back with end leaves 281 which project rearwardly beyond the binding strips 24a, 26a, and are joined to front and rear covers 282. After the book is fabricated and removed from the binding machine, the covers 282
5 are folded forwardly, the connecting portions 283 which are rearward of the strips 24a, 26a, overlying the strips. If adhesive is applied to the end leaves 281, the insides of the covers 282 adhere thereto in the manner of conventional hard-bound books. Pressure
10 sensitive end leaves; such as those shown in U.S. Patent 3,749,423 may be used as the end leaves 281.

Alternatively, instead of hard covers 282, soft covers such as those made of plastic, paper, or other materials integral with the end leaves 281 may be employed.

15 Modifications of the previously described apparatus may be substituted. Directing attention first to the knife structure 161a, the angular support bar 168a is connected to pivot arm 176a in the manner shown in Fig. 17. Blades 163a are disposed at an angle relative to
20 cutter 162a. As the blades 163a move from the retracted position shown in Fig. 17 to the projected and raised position shown in Fig. 18, the cutter structure 161a is at all times below the plane of the lower strip 26a and thus does not contact or damage the lower cover 282.

35 An additional modification of the structure is that

the paper stop 88a is removable. Stop 88a is formed with an elongated vertical section 251, an upper horizontal rearward extending section 252 and a downward rear vertical section 253. Attached to each of the side plates 33a is a stop bracket 254 which has a vertical slot 256 into which rear section 253 slides and against the top of which the horizontal section 252 rests. For production of the books which have been described in connection with the previous modification, the stop 88a is located in place as shown in Fig. 17. However, to make the wrap-around cover modification, the stop 88a is removed by sliding upward so that section 253 slides out of the slot 256. To facilitate removal, the pressure bar 106a may be moved forwardly from its start or rest position.

A third modification of the machine is the installation of a horizontal rear platen 258 behind bridge 84a and co-planar with the platen 86a. Brackets 259 attached to side plates 33a supports rear platen 258. Rearward and an integral part of platen 258 is a formed right angle flange 261. Spring 262 attached to flange 261 and bracket 259 biases the platen 258 which is being pulled by the spring forwardly of the machine. Hence, when an over-sized strip 26a is installed, the spring 262 flexes to permit the platen 258 to move rearwardly

a sufficient distance to accomodate such wider strip.

The rear of bracket 259 has a vertical downward slot 264 into which the shaft of roller 263 slips. Roller 263 rides on the top of lower cover 282 and holds it
5 down.

Still a third bracket 266 is attached adjacent the rear of the machine and above the level of brackets 259. Bracket 266 is formed with a slot 267 on its top edge dividing the upper ends 266 into bifurcations 268.
10 Screw 269 straddles slot 267.

Rack 271 is formed of wire or other suitable material for the upper cover 282. Rack 271 has forward-outward directed ends 272 which fit into slots 267. Friction holds rack 271 in any angle of adjustment, such as the
15 horizontal adjustment shown in Fig. 17 or the downward-rearward slanted adjustment shown in Fig. 18. The selected angle of rack 271 depends upon the thickness of the boom to be bound. Sides 273 parallel to plates 33a are inter-connected by cross bars 274 intermediate
20 sides 273 and at the rearward end thereof. As best shown in Fig. 18, the lower edge of upper cover 282 rests on one or the other of the cross bars 274.

In use of the device, the paper stop 88a is removed by lifting same until rear portions 253 clear slot 256.
25 A proper lower strip 26a is inserted in the recess 87a

and positioned by pin 102a fitting into a hole in the strip 26a, as well understood in the art. The forward edge of rear platen 258 is biased by spring 262 against the rearward edge of the strip 26a. If strips wider 5 than those shown in Fig. 18 are required, the platen 258 moves a corresponding distance rearward, stressing the spring 262.

Although it may be more convenient to preassemble the elements of the book before installing it in the 10 machine, the individual elements may be assembled on the machine. First, the cover 282 is pushed rearwardly, resting on the rear platen 258 and under the roller 263 and thus being held horizontal as shown in Fig. 18.

The end leaf portions 281 rest on platen 86a, and the 15 holes (not shown) in end leaf 281 are aligned with the holes (not shown) in strip 26a. The sheets 21a are then placed on top of the lower end leaf 281, the holes therein again being aligned with the holes in strip 26a or end leaf 281. Thereupon, the top cover is 20 installed with the top cover 282 resting upon one of the cross bars 274 of the rack 271 and the forward end leaf 281 resting on top of the sheets 21a. Again, the holes in the top end leaf 281 are aligned with the holes in the sheets 21a. The studs (not shown) of top strip 24a 25 are then passed through the holes in the top end leaf

281, and the lower strip 26a.

The machine is then energized through one cycle of rotation of the shaft 118a as in the preceding modification. As shown in Figs. 17 and 18, the knife structure 161a
5 pivots about shaft 177a from the retracted position of Fig. 17 to the forward and then to the forward-elevated position of Fig. 18, cutting off the excess lengths of the studs and softening the stub end and forming heads on the stub ends. Cooling fingers 191a then engage the
10 softened heads on the stud ends to cool and set the rivet heads, all as explained in connection with the preceding modification.

The book is then removed from the machine and the covers 282 folded forwardly, the connecting portions
15 283 wrapping around the strips 24a and 26a, and the covers 282 may be caused to adhere to the end leaves 281 as previously explained or such step may be omitted.

If it is desired to use the machine in fabricating books which do not have the wrap-around feature, the
20 backstop 88a may be reinstalled to the position of Fig. 17. When the backstop 88a is in place, rear platen 258 exerts forward pressure against vertical section 251, which exerts pressure on strip 26a.

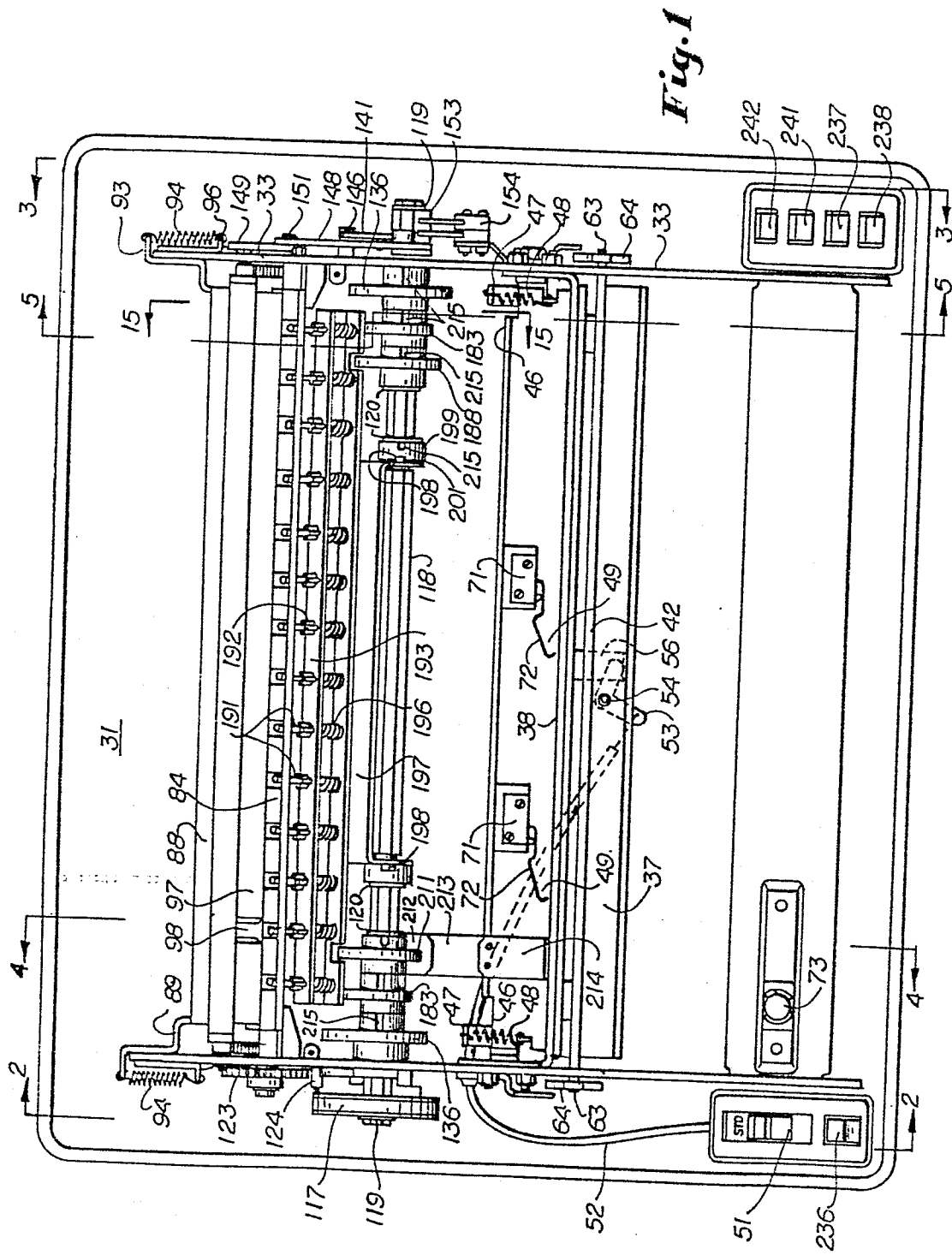
Many of the elements of the apparatus shown in Figs.
25 17 and 18 are simialr in function to those in the

- 34 -

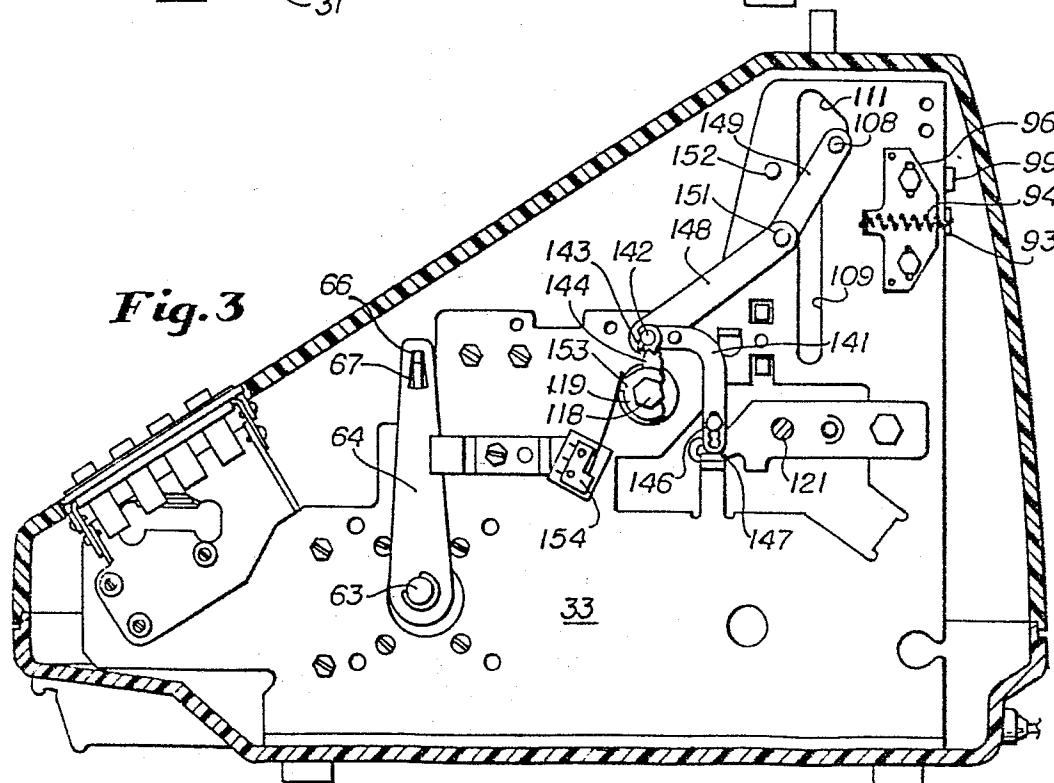
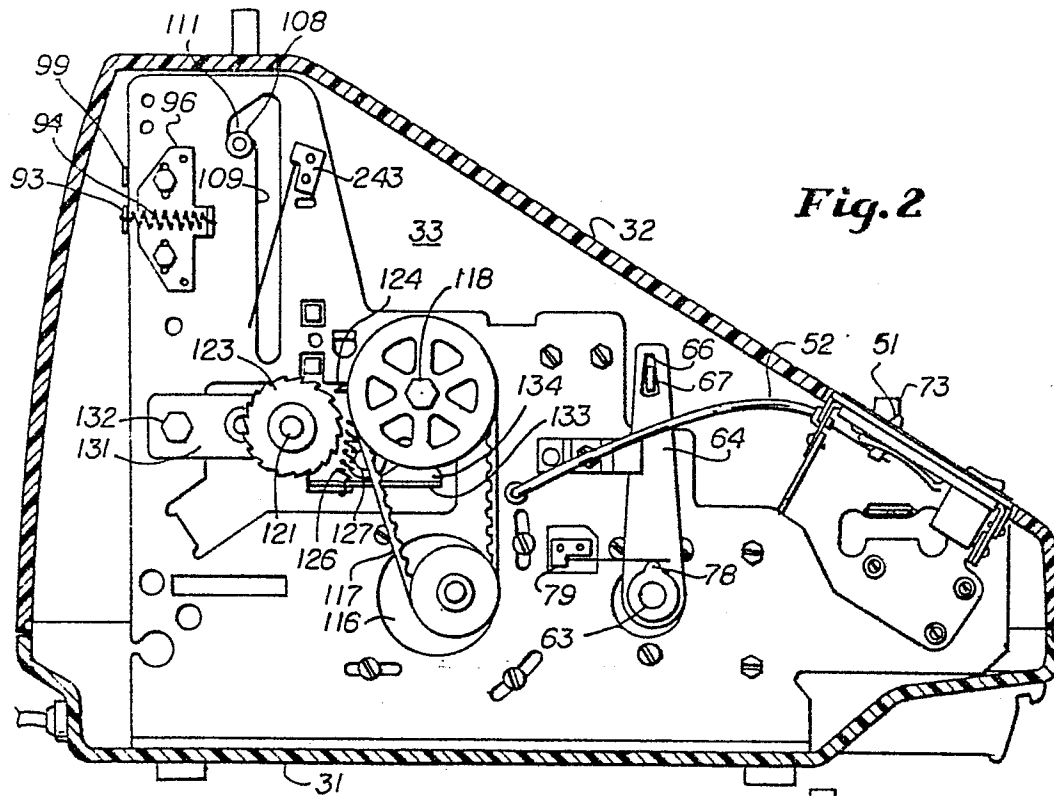
preceding modification, and the same reference numerals followed by the subscript a are used to designate corresponding parts.

CLAIMS:

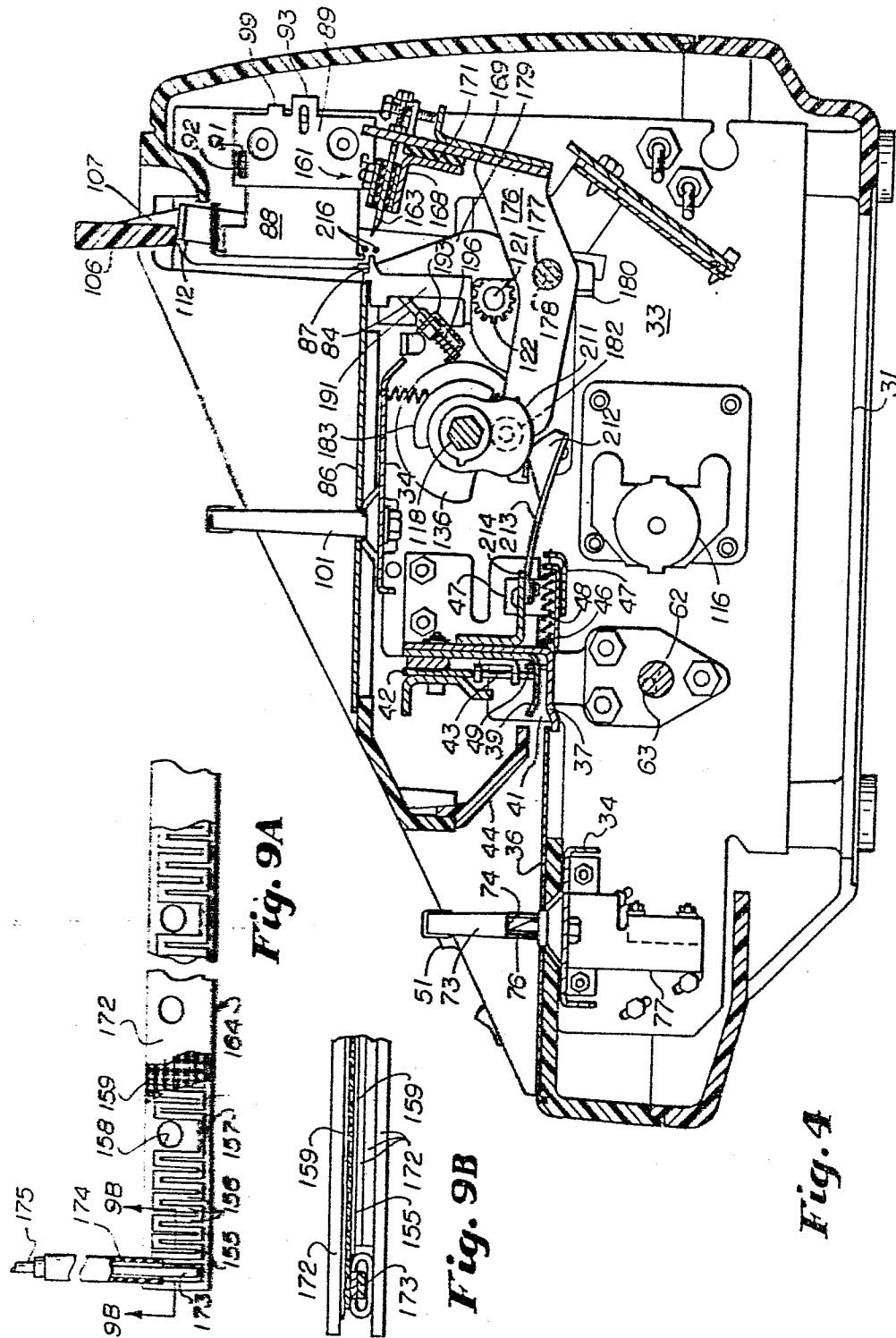
1. Apparatus for binding apertured sheets (21) together using a first strip (24), a plurality of thermoplastic studs (23) projecting from and spaced longitudinally along the first strip, and a second
5 strip (26) formed with holes spaced longitudinally of the second strip at intervals complementary to the studs, the apparatus having a frame having a pair of spaced sides (33) and a platen (86) to support the sheets with their apertures aligned with
10 the holes of the second strip and with the first strip above the sheets with the studs extending through the apertures in the sheets and the holes in the second strip and protruding below the second strip, a pressure bar (106), cooperating means (108, 109)
15 on the sides and bar for guiding movement of the bar from a first position above the platen to a second position adjacent the platen and resting on top of the first strip, actuating means for moving the pressure bar, a motor (116) for driving the actuating
20 means, and a switch (113) for energising the motor, characterised in that the switch is mounted on the bottom edge of the pressure bar and is positioned and arranged to be closed when the pressure bar rests on the first strip so to energize the motor
25 and begin the binding cycle.



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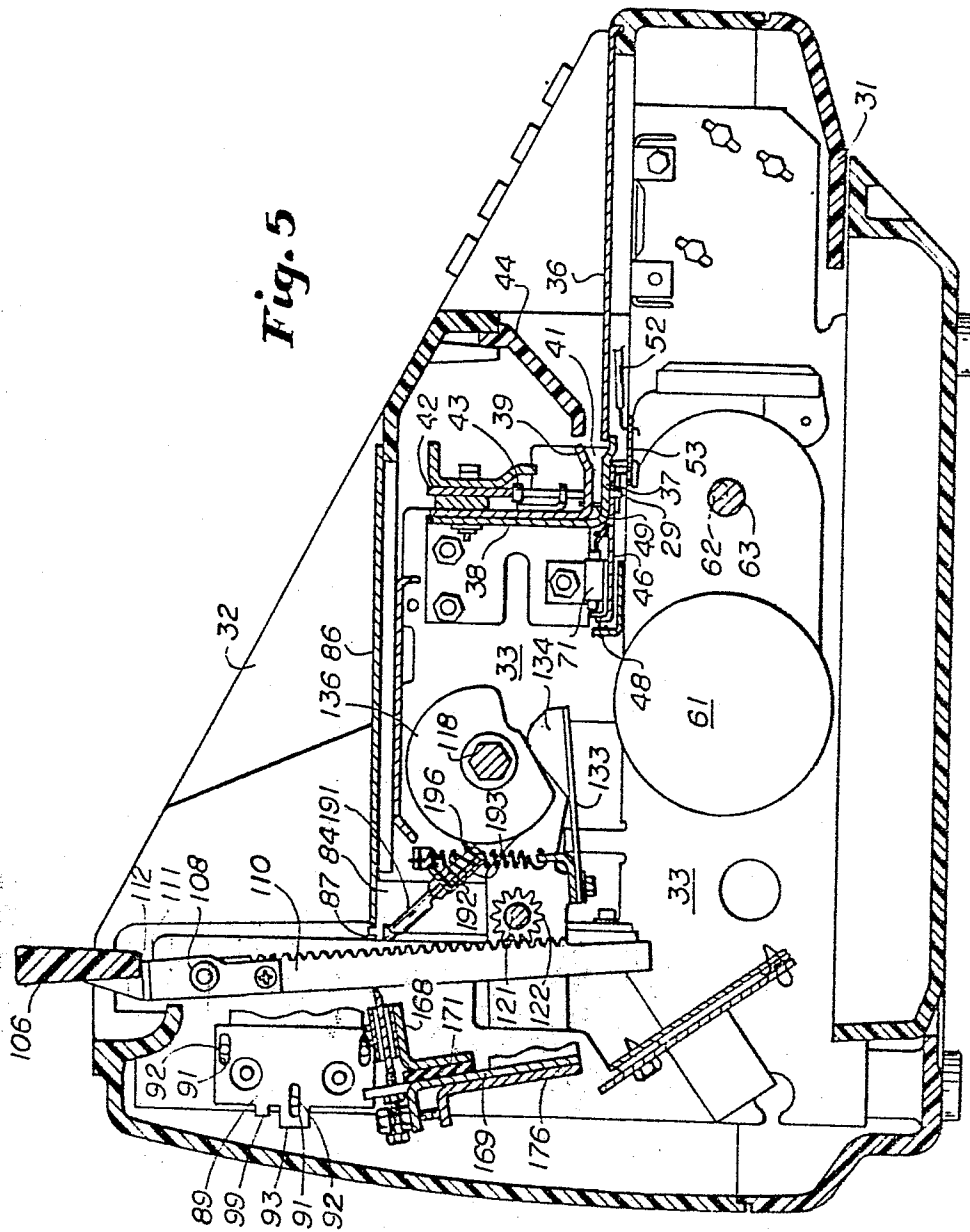


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Fig. 5



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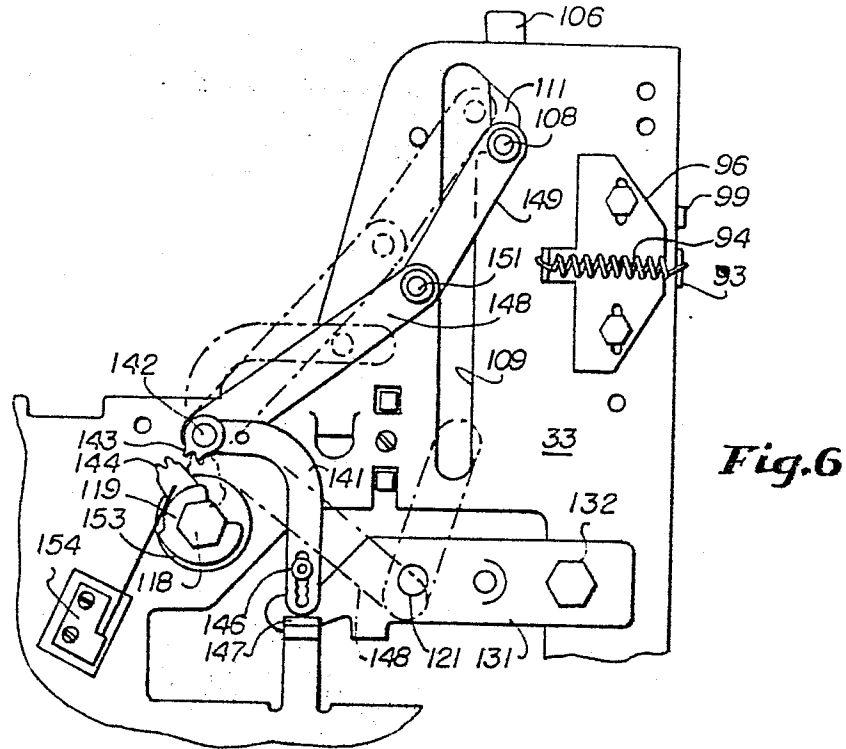


Fig. 6

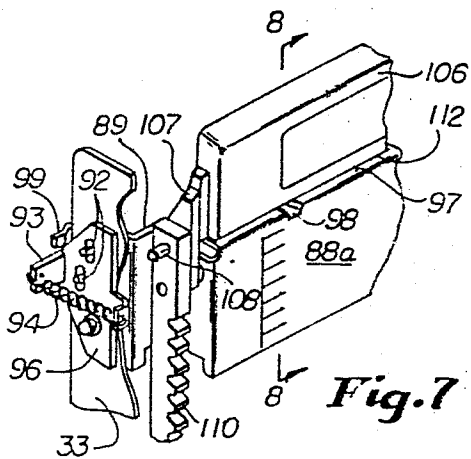


Fig. 7

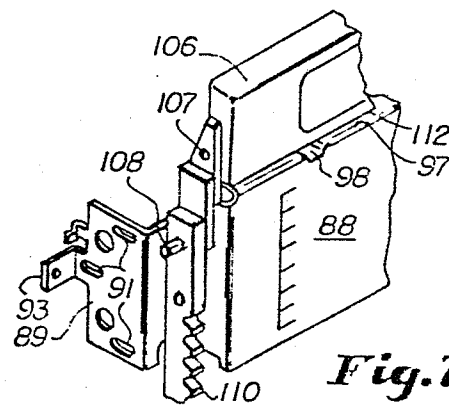


Fig. 7A

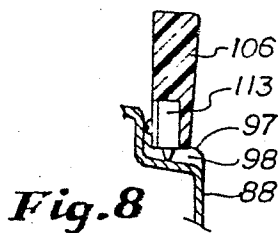


Fig. 8

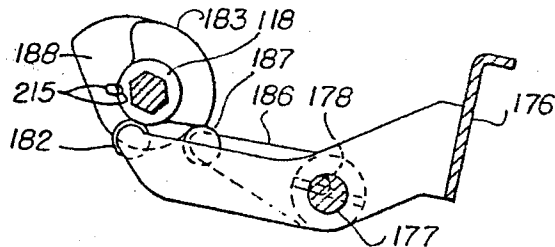
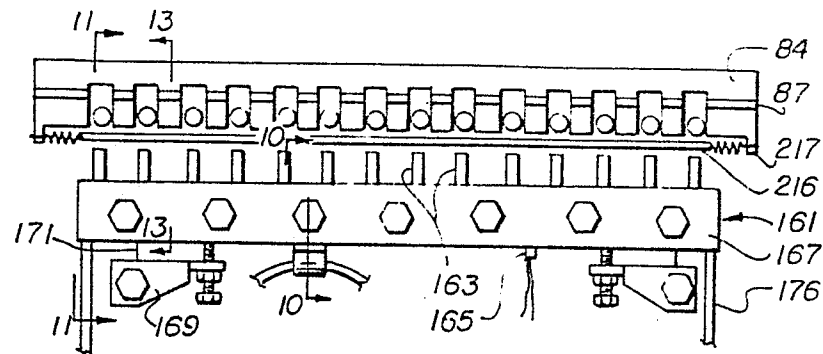
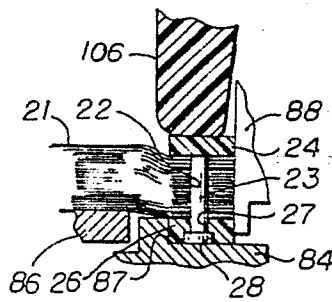
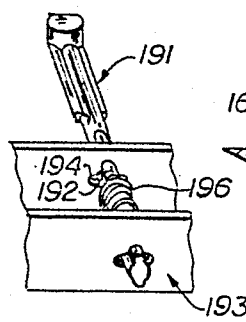
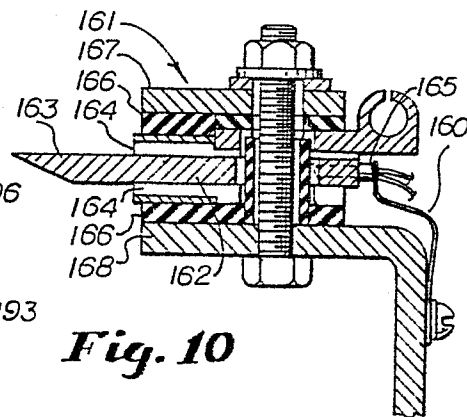
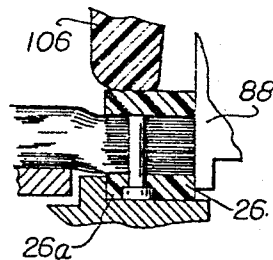
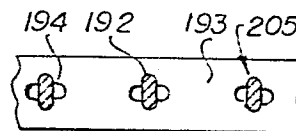
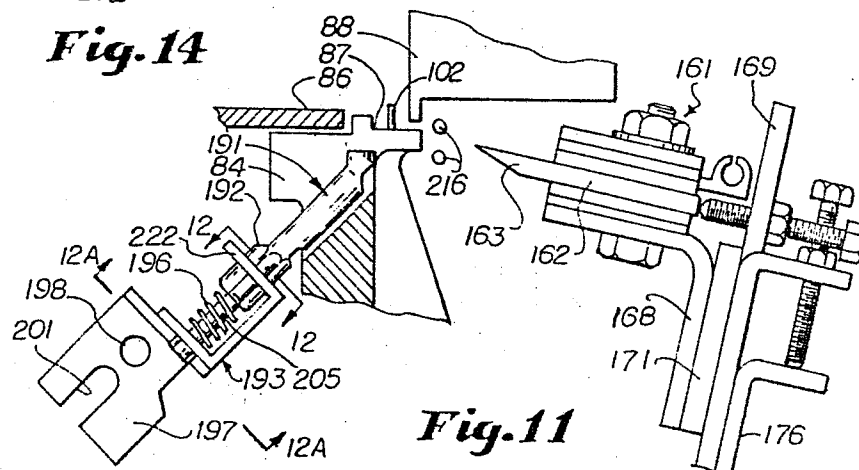


Fig. 15

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**Fig. 9****Fig. 13****Fig. 19****Fig. 10****Fig. 14****Fig. 12****Fig. 11**

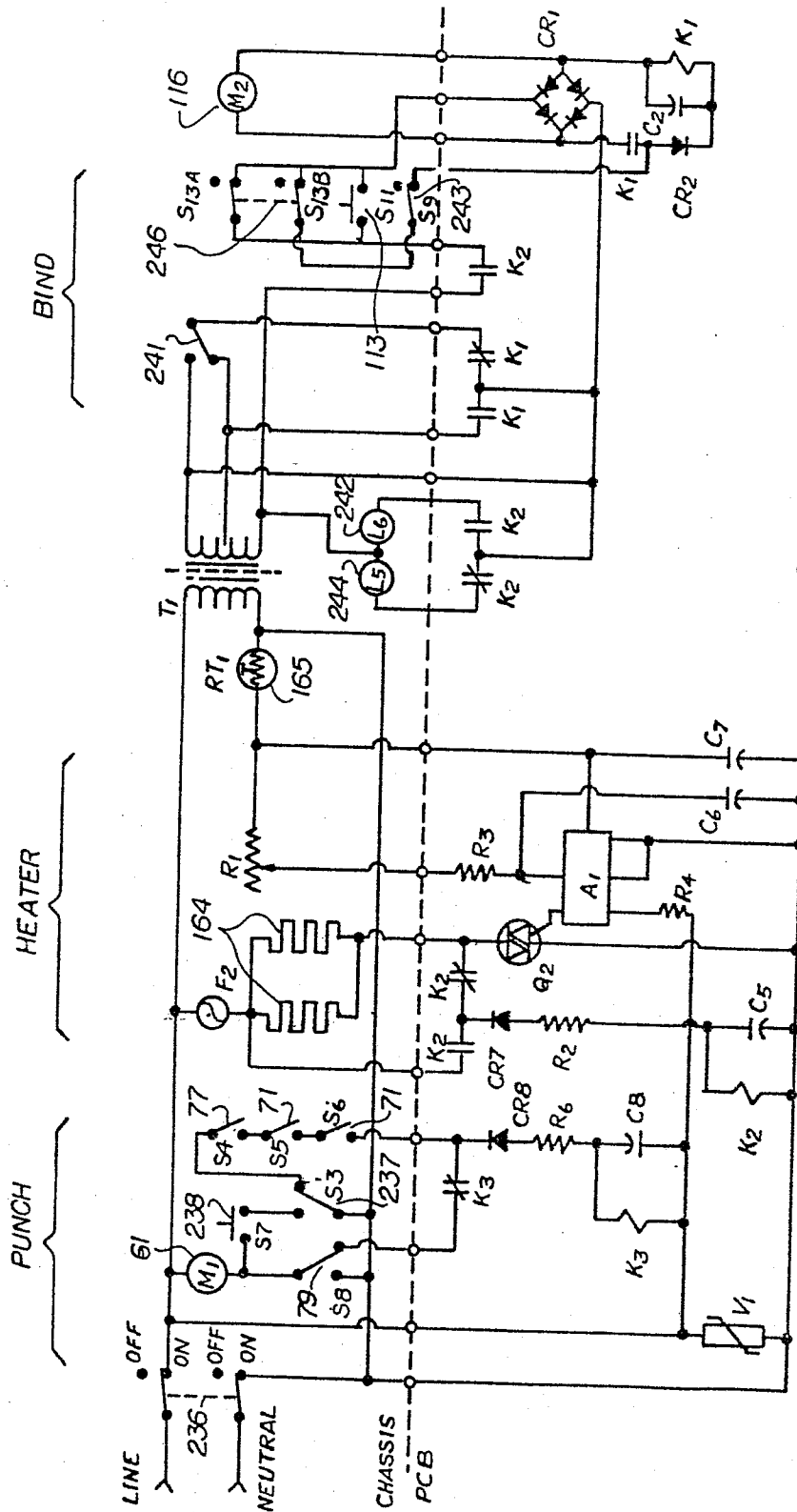


Fig. 16

