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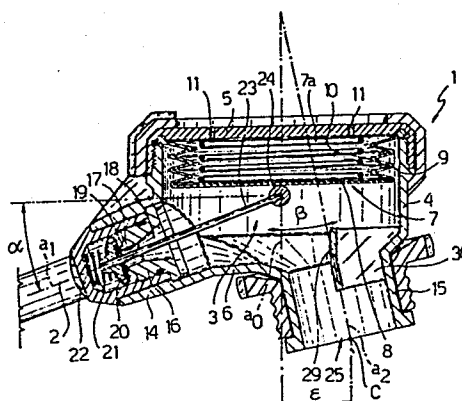
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54 Demand-valve for compressed-air breathing apparatus.

57 What shall be described is a demand-valve (1) for delivering air into a compressed-air breathing apparatus that comprises a substantially cylindrical chamber (3), a first entry-gate (14) for admitting air into the chamber, that presents a passage-opening - controlled by intercepting-means which comprise a blocking-element (20) that is mechanically connected to a deformable membrane (7), and a second entrygate (15) predisposed for feeding-in said air. The device is characterized by the fact that, the axis (a_1) of said first entry-gate forms a first angle (α) with the bottom-surface (8) of said chamber, and that the axis (a_2) of said second exit-gate forms a second angle (β) with the lateral-surface (4) of the chamber; said two axes lying substantially on the same diametrical plane of said chamber.



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1 DEMAND-VALVE FOR COMPRESSED-AIR BREATHING APPARATUS

Specification

5 The present invention refers to an air demand-valve device of
a compressed-air breathing apparatus (for open circuit
compressed-air), that is apt for supplying air, at a prefixed
pressure - slightly higher than atmospheric pressure, into an
ambient (normally comprising an inner cavity with an anti-gas
10 mask) of said breathing apparatus, which must result as being
structurally very simple, reliable, and which must allow for
a very high delivery-flow of air into the above-said ambient,
and with moreover, maintaining substantially constant the
ambiental-pressure itself.

15 A demand-valve device of the type to which the device of the
invention belongs, usually comprises a chamber that
communicates with a source-of-air having a mean pressure -
i.e. comprised between 2 and 10 Bar., through a first-entry-
20 gate, and with the above-said ambient, through a second-
exit-gate. The passage-opening, through the first-entry-gate
is controlled by intercepting-means comprising a blocking-
element mechanically connected to a deformable membrane apt
for defining a surface of said chamber, in such a way that -
25 following the shifting of the membrane, the blocking-element
is also shifted, for thus controlling the passage-opening.
With said constructive disposition, whenever - following a
pressure decrease that is verified in said ambient due to the
effects of inhalation on the part of the user of the
30 breathing apparatus, the above-said deformable membrane
shifts - and the opening is effectuated of the above-
mentioned blocking-element, in such a way as to allow a
sufficient admission of the air-flow into said chamber for

1 restoring the original pressure conditions once again.

The complex - comprised by the above-said membrane and by the kinematic chain that connects it to the above-mentioned
5 blocking-element, substantially constitutes a transducer which is apt for measuring the pressure existing inside the chamber, and for controlling - as a function of said pressure, the shifting of the blocking-element, for feeding appropriate quantities of air into the chamber.

10

Under the conditions described till now, the pressure existing downstream of the blocking-element is, in static terms, equal to atmospheric-pressure, and it decreases slightly during the act of inhalation - with putting the
15 interior of the mask under vacuum. The addition of a spring - having suitable mechanical characteristics, that presses against the membrane from the outside, or else, some other device that performs the same task - allows for obtaining, in static terms, a slight overpressure, with respect to the
20 surrounding atmosphere, downstream of the blocking-element - which, during any actual inhalation, decreases and can also change signs.

However, the air demand-valve devices of the type described,
25 present certain drawbacks.

First and foremost, when particularly high air delivery-flows are required - as a consequence of increased and repeated inhalation on the part of the user, the air pressure existing
30 inside said chamber - and hence, also in the ambient that communicates with the chamber itself, decreases sensibly with respect to the initial pressure. In such a situation, the respiratory efforts of the user himself, also increase -

1 owing to the greater difficulty had in inhaling air.
Moreover, the pressure inside the mask decreases sensibly,
with assuming - even in instances of the apparatus also being
endowed with an overpressure spring, lower values as compared
5 to those of atmospheric-pressure.

As a consequence of this, with the air demand-valve devices
of the type described, the user is required to make non-
negligible breathing efforts - and particularly, whenever
10 greater air delivery-flows are necessary. Therefore the
specified conditions of safety cannot be realized - because,
in the proximity of the sealing-surface, inside the mask, the
pressure can notably decrease to below that of the outside
atmosphere, with thus involving risks of gases and/or
15 vapours entering-in from the polluted surrounding ambient
where the user happens to be.

The above-mentioned drawbacks depend substantially upon the
fact that - apart from the atmospheric-pressure and the
20 pressure existing inside the above-mentioned chamber, there
also act, on the above-said membrane, the fluodynamic effects
of the air-jet that enters into said chamber, and which -
owing to the repeated surface-reflections taking place, can
affect the membrane itself.

25 Said fluodynamic actions can have either a positive or a
negative sign - and can vary considerably, besides
irregularly, as a function of the air-flow delivered by the
device, with also changing signs, and with resulting as being
30 strongly resistant to static action in instances of any
greater air-flows. Therefore, for obtaining these actions, a
vacuum is often required - and hence, an effort on user's
part, that proves to be much greater than what is foreseen by

1 a calculation of static-balance. It results from this that,
to-date, there are many constructors who have conceived
various, more or less complex, devices for eliminating any
direct contact of the dynamic air-flow upon the surface of
5 the membrane itself - for suppressing in this way, the above-
described fluodynamic effect.

The aim of the present invention is to realize an air demand-
valve device for a compressed-air breathing apparatus of the
10 previously indicated type, but which will be in a condition
to eliminate the above-mentioned drawbacks, and which, in
particular, will be apt for supplying air - that is sensibly
constant and of a better quality than the air of the
surrounding atmosphere, into the ambient of the breathing
15 apparatus, and even in those instances when the air-delivery
flows may be considerably increased.

On the basis of the present invention, there has been
realized an air demand-valve device for a compressed-air
20 breathing apparatus, that is apt for supplying air, at a pre-
fixed pressure, into an ambient of said breathing apparatus,
said device comprising a chamber that is substantially
cylindrical and a first air-entry-gate into the chamber,
which substantially opens out onto the lateral surface of
25 said chamber, and which communicates with a source of
compressed-air, and which presents a passage-opening that is
controlled by intercepting-means - comprising a blocking-
element mechanically connected to a deformable membrane
which is apt for defining a base-surface of said chamber, in
30 such a way that - following the shifting of said membrane,
the said blocking-element is also shifted for controlling
said opening, the said device comprising moreover, a second
exit-gate disposed on the other base-surface of said chamber,

1 and predisposed for feeding said air into said ambient, and
being characterized by the fact that the axis of said first-
entry-gate forms a second angle with the axis of said lateral
surface, said two axes lying substantially on the same
5 diametrical plane of said chamber.

Said first angle is opportunely comprised between 15° and
 25° , and said second angle is comprised between 5° and 15° .
Moreover, for convenience sake, the distance between said
10 axis of said lateral surface, and the point-of-intersection
of said axis of the second gate with the plane that frontally
delimits the gate itself, is comprised between 20% and 35% of
the diameter of said chamber.

15 For having a better comprehension of the device of the
present invention, there will now be given, solely by way of
example, the description of one particular form of
realization with making reference to the attached drawings,
whereby:

20

- FIGURE 1 represents a vertical section of the air
demand-valve device of the invention;

25

- FIGURE 2 represents a plane view, partially in section,
of the device of FIG. 1; FIGURES 3 and 4 represent
sections - of a part of the air demand-valve device of
Fig. 1, in two different operative positions;

30

- FIGURE 5 represents certain diagrams apt for
illustrating the curve of the air-pressure delivered, as
a function of the air-flow capacity itself, that is
encountered in the device of the invention, as well as
in the devices of the prior art of the same type.

1 First and foremost, referring to FIG. 1, the air demand-valve
device of the invention - indicated in its complex whole with
the reference numeral 1, is apt for being interposed between
a compressed-air feeding breathing-tube 2 and an ambient (not
5 shown) comprised, for example, by the space inside an anti-
gas mask, into which breathable air has to be fed.

The device comprises a substantially cylindrical chamber 3,
delimited by a lateral-wall 4 and a pair of substantially
10 flat base-walls 5 and 6. Inside the chamber 3, there is
disposed a membrane 7 which comprises a substantially flat
base-wall 8 and a lateral-wall 9, that presents a plurality
of corrugations apt for rendering said membrane substantially
deformable in the direction of the axis of the chamber 3.
15 With the wall 5, said membrane defines another chamber 10,
which communicates with the atmosphere through one or several
holes 11, made in the wall 5. For convenience sake, the
device can comprise a spring 7a, disposed inside the membrane
7 and predisposed for exercising a prefixed axial-force on
20 the flat wall 8. The device comprises moreover, a first
entry-gate 14 - that is apt for putting the tube 2 into
communication with the chamber 3, and which is made
substantially in the lateral-wall 4 which delimits said
chamber, and a second exit-gate 15 - for putting the latter
25 into communication with the internal ambient of the mask, and
which is made in the base-wall 6 that also delimits the
chamber itself.

Inside the entry-gate 14, there is disposed a bush 16,
30 provided with a pair of axial holes 17 and 18 - of which the
first is cylindrical and the second conical, as well as with
a seat 19 having a substantially annular form, apt for

housing the blocking-element 20 that is normally held against
said seat by means of a spring 21 - interposed in-between the
latter and another bush 22 that is screwed into a threaded-
hole made inside the entry-gate 14 - as is clearly visible in
FIG. 1.

The blocking-element 20 is made solidal with a rod 23 that
goes through the holes 17 and 18, and whose extremity is
provided with a stylus 24 apt for collaborating with the
surface of the wall 8 of the membrane 9.

On the basis of the invention, the axis of the first entry-
gate 14 - indicated with a a_1 in FIG. 1, forms a first angle
 α with the base-surfaces that delimit the chamber 23 and,
in particular, with the surface of the flat wall 8 of the
deformable membrane 9. Moreover, the axis of the second
exit-gate 15, indicated with a a_2 in Fig. 1, forms a second
angle β , with the axis a_0 of the chamber itself; the two
axes, a_1 and a_2 , lie substantially in the same diametrical
plane of the chamber 3 - which coincides with that in the
drawing of FIG. 1.

For the purpose of achieving the aims that shall be indicated
further on in the text, the angles α and β are
comprised respectively between 15° and 25° and between 5° and
 15° . Moreover, the distance - between the axis a_0 of the
chamber 13 and the intersection point C of the axis a_2 of
the exit-gate 15 with the plane that frontally delimits the
gate itself (and which has been indicated with E in FIG. 1),
is comprised between 20% and 35 % of the diameter of said
chamber. In this manner, the opening of the outlet 25 of the
exit-gate 15, results as being substantially eccentric with
respect to the axis a_0 of the chamber 3 - as clearly results

1 in FIG. 1.

5 Inside the chamber 3, and in the vicinity of the lead-in of the second gate 15, there is disposed a substantially flat wall 29. The plane of said wall is substantially parallel to that of the axis a_0 and perpendicular to the plane that contains the axes a_1 and a_2 respectively, of the gates 14 and 15. Moreover, the said wall comprises, for convenience sake, a rib 30, having the function of conveying the air-
10 flow, as well as that of acting as a stiffening.

Next, the half-opening of the conical surface - that delimits the second hole 18, downstream of the blocking-element 20, is comprised between 16° and 18° . The ratio, between the axial
15 length of the first and second hole 17 and 18, is comprised between 0.85 and 0.45; whereas, for convenience sake, the axial lengths of the holes themselves, are comprised between 4 and 6 mm and between 7 and 9 millimeters respectively.

20 To end with, the ratio between the diameter of the first hole 17 and its axial length, is comprised between 0.85 and 1.32, and - for convenience sake, the diameter of the hole itself, is comprised between 5.1 and 5.3 mm.

25 In practical usage tests, the air demand-valve device described, has - quite surprisingly, proved to function in a very satisfactory way. It has been endeavoured to give a rational explanation, with the following considerations, which must be taken as being qualitative - since the
30 complexity of the forms, and of the surface reflections in particular, that are for the most part curved, impede obtaining, with any precise mathematical calculus, an exact prevision of the behaviour of the device itself.

1 When it is inserted into the circuit of the self-breathing
apparatus, there exists inside of chamber 3 - between the
compressed-air feeding tube 2 and the mask of the compressed-
5 air breathing apparatus itself, and under normal functioning
conditions of the device, a pressure that is slightly higher
than atmospheric-pressure - for example: a pressure that has
from 30-40 mm of water-column height above said pressure. As
is already known, such a pressure inside the chamber 3 - and
hence, in the ambient of the compressed-air breathing-
10 apparatus into which breathable air must be introduced,
offers a greater guarantee of safety for the user - since,
under such conditions, there is impeded any infiltration of
gases or vapours from the polluted surrounding atmosphere
wherein the user happens to be.

15 If it is supposed that the spring 7a is absent (a functioning
condition that can be defined as "negative pressure") the
membrane 9 becomes disposed in a balanced position, depending
upon the pressure value that exists inside the chamber 10 -
20 which acts on one of the surfaces of the wall 8, and the
above-said pressure that exists in the chamber 3 - which acts
upon the other surface of the wall itself. When the pressure
inside the chamber 3 is reduced, even to a modest extent -
following an inhalation, said 'balanced condition' becomes
25 changed and the wall 8 of said membrane, shifts towards the
stylus 24, determining the rotation of the rod 23
substantially in the plane of the drawing of FIG. 1, for
controlling the opening of the blocking-element 20 - which
substantially assumes the configuration illustrated in FIG.
30 3, whereby it leans just on a very restricted zone of the
housing 19, with leaving free a part having prefixed
dimensions, through which there is established an air-flow

1 entry into the chamber 3.

5 The addition of a spring such as 7a - having suitable mechanical characteristics, which presses against the wall 8 of the membrane 7 (or some other such device for carrying out the same task), allows for obtaining - under static conditions, a slight overpressure, with respect to the surrounding atmosphere, downstream of the blocking-element - and hence, inside of the mask; which said overpressure
10 becomes reduced during the act of inhalation and can also change its sign. This functioning condition can also be defined as being "positive pressures".

15 The above-said air-flow that, given the pressure conditions inside chamber 3 and inside tube 2, realizes a true and proper supersonic expansion - enters into said chamber through a plurality of axial deviations that are comparable, for simplicity sake, to reflections on the walls of the holes 17 and 14 - for example: according to the succession of
20 reflections indicated with the broken-line of FIG. 3. As can clearly be seen from said broken-line, the above-said flow follows a trajectory comprised by a succession of rectilinear tracts, as a consequence of these said reflections, which do not interfere with the flat wall 8 of the membrane 9. In
25 particular, the trajectory tract indicated with t_0 , by which the above-said flow passes through chamber 3 - substantially in the transversal direction, has a slightly inclined direction with respect to the plane of the wall, in such a sense as to draw away from it, so as to strike further
30 surfaces inside of the chamber 3 and to come out from the exit-opening 15, through the means of a series of further reflections.

1 A trajectory of this type, i.e. comprising a succession
trajectory-tracts - none of which strike the wall 8 of the
membrane 9, is the result of the form of the disposition of
the various parts of the device and, in particular, of the
5 relative position of the axes a_1 and a_2 of the gates 14 and
15, defined by the angles α and β , of the presence of the
eccentricity presented by the exit-gate 25 of the gate 15
with respect to the axis a_0 (eccentricity ϵ), as well as by
the form and the dimensions, defined previously, for the
10 holes 14 and 17, downstream of the blocking-element 20.

It has been verified that, whenever the previously mentioned
geometrical parameters are selected from the indicated ranges
of values, the favourable condition - for realizing
15 trajectories of air-flows from the entry-gate 14 to the exit-
gate 15, with trajectories that never interfere with the wall
8 of the membrane 9 - is obtained not only for a particular
opening angle of the blocking-element 20, but substantially
for opening angles which go from zero to a maximum angle - to
20 which correspond considerably high air-flow entries into the
chamber 3, that are necessary for feeding breathable air for
the user, with continuity, even under the most unfavourable
operative conditions. In FIGS. 3 and 4, there have been
indicated two different configurations of the membrane 9, to
25 which correspond two diverse opening-angles of the blocking-
element 20. There are also shown herein, the trajectories of
the air-flows which are obtained presumably, in these two
instances. As can be clearly seen, none of these comprise
flow-tracts that strike the membrane 8.

30

It has also been verified that the first wall 29, inserted
into the first position indicated inside of the chamber 3,
also contributes for conveying the air-flow towards the exit-

1 gate 15. Moreover, said wall can constitute the end-stop
limiter, for the shifting-movement of the wall 8 of the
membrane 9.

5 Therefore, it is evident that, with the air demand-valve
device of the invention, any negative fluodynamic actions on
the membrane 8 generated by the air-jet entering chamber 3,
is prevented, i.e. those actions which tend to take membrane
towards the at-rest conditions corresponding to the null flow
10 - and without even needing to have recourse to any costly and
complex mechanisms for removing the membrane from the effects
of said air-flow; but instead, it is had that, through the
effects of the air-flow divergence from the membrane, as can
be seen in FIGS. 3 and 4, the action of said membrane on the
15 blocking-element 20 is amplified in such a way as to achieve
a minimum lowering of the pressure (or of the overpressures)
existing inside the mask for the entire field of
instantaneous respiratory intensity, as necessitated by human
physiology (upper limit about 300 l/min). Under these
20 conditions, an air-flow having a very high capacity can be
controlled, without any particular respiratory efforts on the
user's part. It has been found that, under particular
functioning conditions of the device, involving trajectory
tracts that tend to draw away from the membrane 8 (as was
25 shown in the instance of FIGS. 3 and 4), there is even had a
suction action upon the membrane, which can favour the
opening of the blocking-element whenever very high air-flows
must be supplied.

30 In FIG. 5 there is indicated, with the curve A, the
development of the pressure inside the chamber 3 as a
function of the air-flows supplied. As can be seen from said
curve, inside of the chamber 3 there exists a substantially

1 constant pressure (no matter what the flow of air supplied
may be) comprised within the range that concerns those
applications to which the air demand-valve device of the
invention is intended. Moreover, the above-said pressure
5 tends to become lower, only in correspondence of the upper
value limit of the above-said range of values - without
however, creating a depression inside the mask to which the
air-breathing device itself is connected.

10 In the same FIG. 5, there are represented, with the curves B
and C, the pressure-flow characteristics, for other known air
breathing-devices of the type described. As can be seen, not
only is a sensible reduction of pressure had - as soon as the
air-flow introduced by the device tends to increase, but said
15 pressure can become already negative because of the flow-
values comprised within the range that concerns the viewpoint
of practical applications.

In said FIGURE, the scales given on the left-hand side,
20 relate to a functioning-configuration with "positive
pressures" (obtained through the presence of the spring 7a);
whereas those given on the right-hand side, relate to a
functioning-configuration with "negative pressures" (obtained
without the spring 7a).

25 The air demand-valve device of the invention - apart from its
presenting the favourable properties described previously,
results as also being constructively very simple - because of
its being constituted of a only a few elements that can
30 easily be constructed and which do not require any restricted
working tolerances. Hence, this device results as also being
very reliable - seeing that it maintains, during the course
of its usage, the favourable characteristics that have been

1 previously described.

What results as being obvious, is that to the forms of
realization described in the present invention, there can be
5 brought about modifications and other alternative variations
- both, in the form as well as in the disposition of the
various parts, without this exiting in any way from the ambit
of the inventive idea itself.

1 WHAT IS CLAIMED IS:

5 1. Air demand-valve device (1) for a compressed-air breathing apparatus, apt for distributing air at a pre-fixed pressure into an ambient of said compressed-air breathing apparatus, said device - that comprises a substantially cylindrical chamber (3) and a first air entry-gate (14) into the chamber, which opens out substantially upon the lateral surface of said chamber, is in communication with a source of compressed-air and presents a passage-opening controlled by intercepting-means, comprising a blocking-element (20) fixed mechanically to a deformable membrane (7) apt for defining a substantially flat base-surface (8) in said chamber, in such a way that, following the shifting of said membrane, said blocking-element is also moved for controlling said opening, the said device comprising moreover, a second gate (15) disposed on the other base-surface of said chamber and predisposed for feeding said air into said ambient, characterized by the fact that the axis (a_1) of said first-gate forms a first angle (α) with said base-surfaces (8) of said chamber and the axis (a_2) of said second-gate (15) forms a second angle (β) with the axis (a_0) of said lateral surface, said two axes lying substantially on the same diametrical plane of said chamber (3).

25 2. Air demand-valve device, according to CLAIM 1, characterized by the fact that said first angle is comprised between 15° and 25° and said second angle is comprised between 5° and 15° .

30 3. Air demand-valve device, according to CLAIMS 1 or 2, characterized by the fact that the distance (ϵ), between said axis (a_0) of said lateral surface and the point-of-

1 intersection (C) of said second-gate's axis with the plane
that frontally delimits the gate itself, is comprised between
20% and 35% of the diameter of said chamber.

5 4. Air demand-valve device, according to one of the
previous Claims, characterized by the fact that it comprises
a diaphragm, disposed inside said chamber, comprising at
least a flat wall (29) disposed in the vicinity of the mouth
of said second-gate and on a plane that is substantially
10 parallel to said axis (a_0) of said lateral surface and
perpendicular to said plane that contains the axes of the
gates (a_1 , a_2).

15 5. Air demand-valve device, according to one of the
previous Claims, wherein, downstream of said blocking-element
(20), said gate comprises a first, substantially cylindrical
hole (17) and a second hole (18) delimited by a conical
surface, characterized by the fact that the half-opening of
said conical surface is comprised between 16° and 18° .

20 6. Air demand-valve device, according to CLAIM 5,
characterized by the fact that the ratio between the axial
length of said first and second holes, is comprised between
0.85 and 0.45.

25 7. Air demand-valve device, according to CLAIM 4 or 5,
characterized by the fact that the axial lengths, of said
first and said second hole, are comprised respectively,
between 4 and 6 mm. and between 7 and 9 mm.

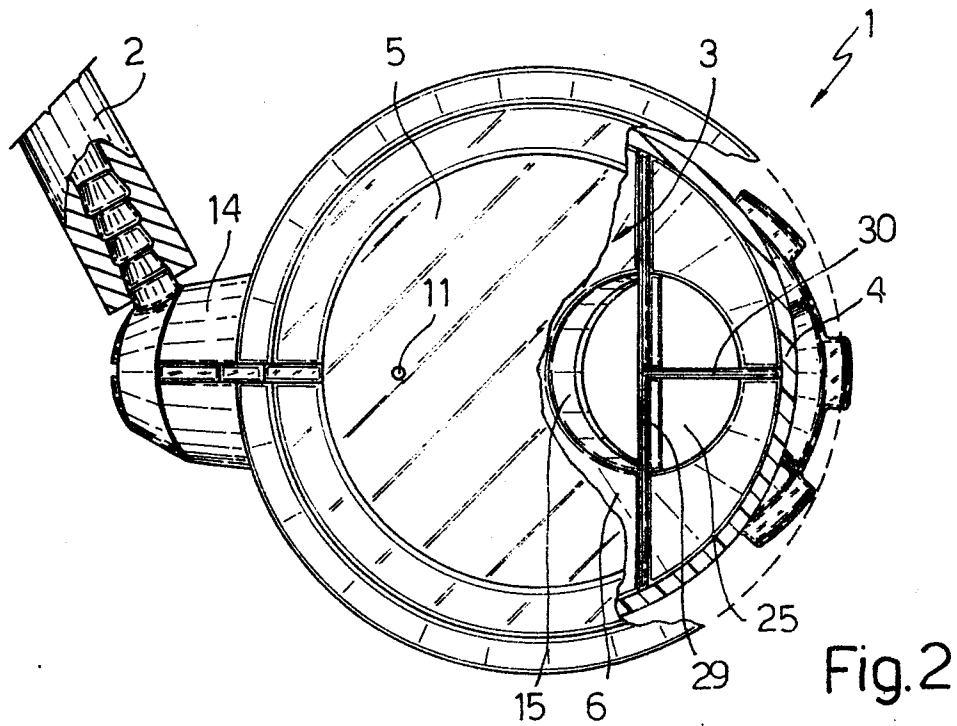
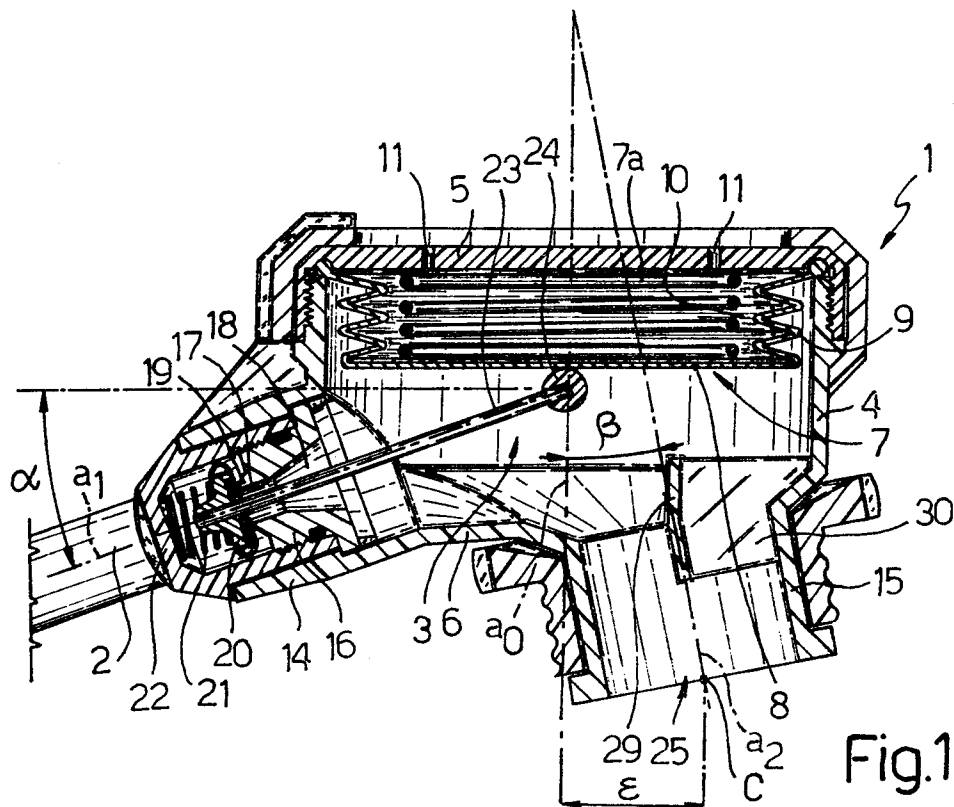
30 8. Air demand-valve device, according to one of the CLAIMS
from 5 to 7, characterized by the fact that the ratio -
between the diameter of said first hole and its axial length,

1 is comprised between 0.85 and 1.32.

9. Air demand-valve device, according to CLAIM 8,
characterized by the fact that the diameter of said first
5 hole, is comprised between 5.1 and 5.3 mm.

10. Device, according to CLAIM 1, characterized by the fact
that said membrane (7) comprises a substantially flat and
indeformable base-surface (8), apt for defining said base-
10 surface of said chamber, and a deformable lateral surface (9)
provided with a plurality of corrugations.

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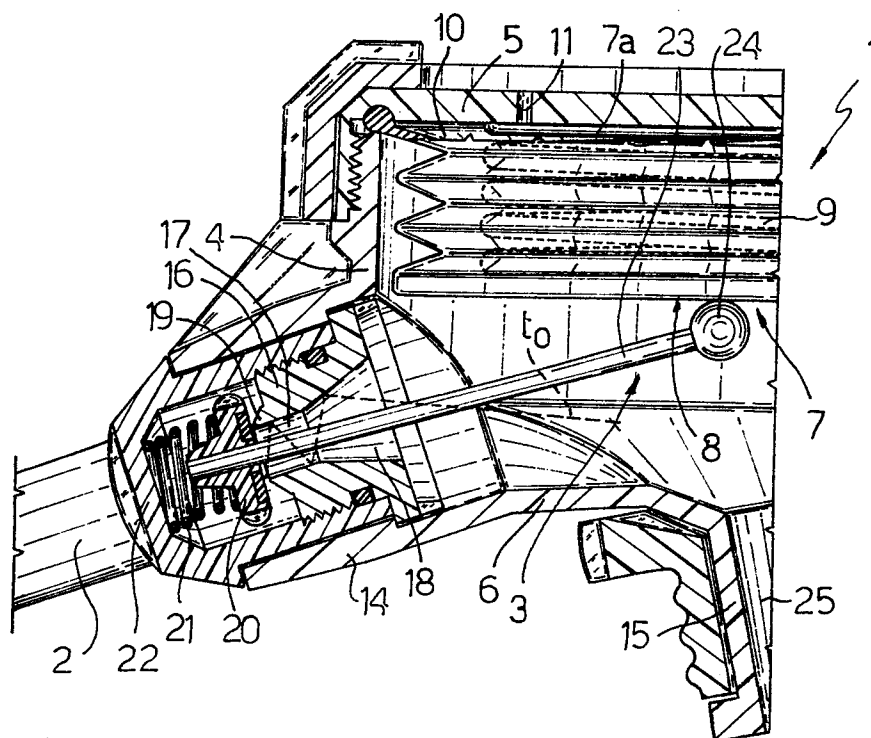


Fig. 3

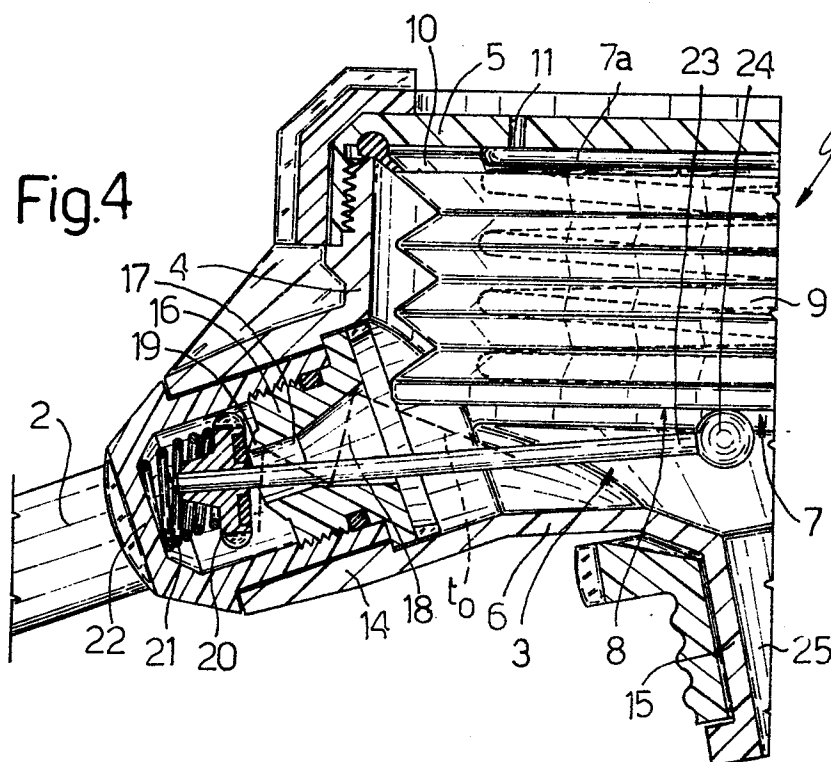


Fig. 4

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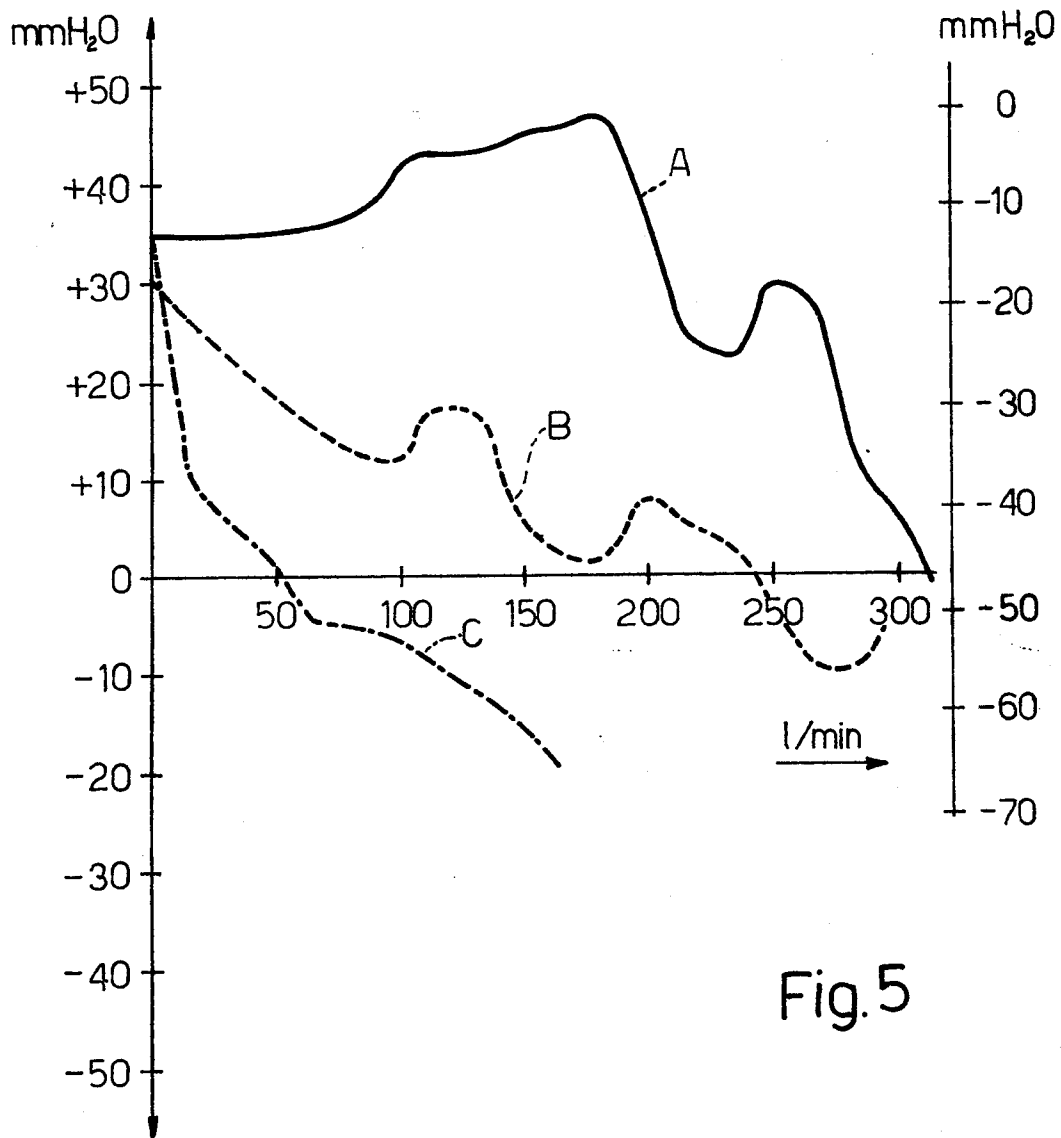


Fig. 5