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London WC2A 2EB(GB)(54) **Rotary cutting apparatus.**

(57) A method and apparatus particularly suited for cutting continuously moving thermoplastic film such as linear low density polyethylene includes a stationary anvil, preferably a roller of tool steel and a tungsten carbide die mounted for rotation along a circular arc intersecting the anvil. In an embodiment for punching holes in the film, the apparatus includes a saddle shaped hollow tungsten carbide die projecting slightly above the cylindrical surface of a roll passing under a spring biased tool steel roller for cutting a semicircular hole in a continuously moving continuous plastic film web for the manufacturer draw tape bags. A vacuum is supplied to the center of the die for removing the segment of plastic film cut out by the apparatus. In an alternate embodiment, the rotary cutting apparatus includes a tungsten carbide plate member having a straight cutting edge is rotated along a circular arc at the end of support mounted to a shaft to intersect a spring biased roller supported along the arc to cut a perforation in a continuous plastic film passed between the straight cutting edge and roller. The alternate embodiment is suited for employing a plurality of the rotary cutting devices at spaced stations for cutting perforations across a moving film web.

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ROTARY CUTTING APPARATUS

This invention relates to rotary cutting apparatus and, in particular, an apparatus suitable for high speed cutting of plastic film webs, linear low-density polyethylene ("LLDPE"), film which is difficult to cut.

It is quite common in manufacturing processes, such as plastic bag manufacture, to punch holes in or otherwise perforate a plastic film web, particularly while the web is moving, to manufacture plastic bags from the web. Manufacturing productivity is generally increased the faster the plastic web can be moved through the manufacturing line. However, increased speed may affect the quality of the cuts being made in the web. In the case of mechanically contacting cutters, wear and downtime for replacing or refinishing of the cutting surfaces also may be significant. A number of devices suitable for automatic mechanical cutting of a moving plastic film are known and have been considered.

A reciprocating punch can be employed to make cuts at the edge of a continuously moving plastic film web. This method has previously been employed in punching semi-circular cutouts in the hem edge of a continuous moving plastic web during the manufacture of draw tape strip bags. See, for example, U.S. application Serial No. 652,254, filed September 20, 1984, assigned to the assignee of the present invention and incorporated by reference herein.

Air cylinder actuated reciprocating punches were used on the draw tape bag manufacturing line disclosed in the aforesaid U.S. application Serial No. 652,254. Initially, a hemispherical punch reciprocating against a smooth circular die was employed. However, the LLDPE material was so stretchable, that the slightest gap between the punch and die would stretch rather than cut the material. The larger the hole desired, the more likely that incomplete cutting would occur.

A serrated edge circular die was substituted which does not require a close fit with the punch thus avoiding metal contact. The serrated edge die was found adequate for existing cycling speeds (approximately 100 cycles per minute for 24 inch wide bags). At higher speeds, punching the plastic web on the fly produced tension variations within the web which were sufficiently great as to disrupt other web processing operations. Moreover, cutting at higher speeds leaves a more jagged edge. It would be extremely desirable to provide a punch for punching holes in a continuously moving LLDPE plastic film web at cycling rates of at least 120 cycles per minute and web speed in excess of 240 feet per minute. A commercially available, so-

lenoid powered reciprocating punch having a conical punch head penetrating a circular die was also examined. The punch was found to have a life of about one million cycles through four thicknesses of approximately 1.5 mils LLDPE film. However, the punch was noisy and was found to stick or "block" the edges of the layers together making subsequent insertion of a draw tape strip between the layers difficult.

Also examined was a commercially available rotary punch and die formed by a pair of complementary contacting cutting surfaces in a pair of adjoining rolls. Several thousand pounds of compression on the rolls was required to obtain a cut and quickly dulled the cutting surfaces. The system was found to have a life of only about 5,000 holes or cycles.

It is an object of the invention to provide a plastic film cutting apparatus capable of a high cycling rate and variable speed operation.

It is yet another object of the invention to provide a plastic film cutting apparatus which provides clean, undistorted cuts in the film at predetermined locations even at relatively high cycling rates and film speeds.

It is yet another object of the invention to provide a plastic film cutting apparatus which is relatively quiet in operation.

These objects and others are accomplished according to the present invention by providing a cutting die having a cutting surface mounted a rotary support for rotating the cutting surface along a circular arc and positioning a roller adjacent the rotary support at a stationary location along the circular arc so as to cyclically contact and roll across the cutting surface of the die when the die is rotated on the rotary support past the roller whereby the cutting surface and the roller cooperate for cutting a plastic film web passing therebetween at predetermined locations in the web.

According to another important aspect of the invention, the roller is resiliently biased in a controlled manner toward the contacted cutting surface so as to generate a minimum, substantially uniform compressive force against the cutting surface for a smooth, uniform cut.

It is yet another object of the invention to provide an apparatus for mechanically cutting plastic film and the like having long life.

It is yet another object of the invention to provide a mechanical cutting device providing superior performance on LLDPE film.

According to the invention, this and other objects are accomplished by forming the cutting surface of the die from tungsten carbide and the roller

from tool steel.

It is yet another object of the invention to provide an improved apparatus for cutting holes and moving plastic film web.

According to this object of the invention, a cutting die having a closed curve cutting surface, conforming to the outline of the cut desired, is employed. The cutting die is hollow within the closed curve cutting surface and is coupled with a hollow interior of the rotary support. A vacuum blower coupled with the hollow interior of the rotary support creates a vacuum at the die cutting surface for removing a plug of cut material. A hollow roller may be conveniently employed as the rotary support for the closed cutting surface die.

It is yet another object of the invention to provide a rotary cutting apparatus for making parallel perforation cuts along a continuously moving web.

According to this object of the invention, a cutting die having a straight tungsten carbide cutting edge is mounted on a rotary support for rotating the cutting edge along an arc so as to contact during each rotation a stationarily positioned tool steel roller.

The various objects and advantages of the invention will be better understood by reference to the accompanying drawings and detailed description of the embodiments.

Fig. 1 is a side elevation view of a rotary cutting apparatus of the present invention for cutting a hole in a moving plastic film web;

Fig. 2 is an elevation view of the apparatus of Fig. 1 taken along the lines 2-2 of Fig. 1;

Fig. 3 is a sectional view of a portion of the apparatus of Fig. 1 along the lines 3-3 of Fig. 1;

Fig. 4 is an enlarged sectional view of the roller support arm and cooperating rotary supported cutting die of the apparatus of Fig. 1 taken along the lines 4-4 in Fig. 3;

Fig. 5 is a sectional view taken along the lines 5-5 of Fig. 4;

Fig. 6 is a sectional view taken along the lines 6-6 in Fig. 4;

Fig. 7 is a side elevation of another embodiment of the invention incorporated in a plastic film perforator;

Fig. 8 is a partially sectioned plan view of the apparatus of Fig. 7 taken along the lines 8-8 in Fig. 7; and

Fig. 9 is a fractional sectional view of the apparatus of Fig. 7 taken along the lines 9-9 of Fig. 8.

Fig. 1 illustrates a portion of a thermoplastic draw tape bag manufacturing line of the type disclosed in U.S. application Serial No. 652,254 referred to above. A thermoplastic film web 10 in the form of a once folded thickness of linear low-

density polyethylene (LLDPE) or other plastic film is drawn from a supply roll (not shown) by draw rolls 11 and 12 over a hem folder 14 mounted on a vertical member 16 of the frame F. Idler rolls 19 and 20, also supported on vertical member 16 below and to either side of the hem folder 14, cause the web 10 to pass over the upper surface of the hem folder 14 where the slit edges of the two film layers are folded over to form hems for receiving draw tapes. From the idler roll 20, the web with the hems formed therein is passed over the roller 22 of the rotary cutter or punch of the present invention, referred to generally by the reference numeral 24. Idler rolls 26 and 28 mounted on the frame F downstream from the rotary cutting apparatus 24 raise the film path for passage of the web 10 through a draw tape strip insertion device and a hot air hem sealer, both normally positioned between the roller 28 and draw rolls 11 and 12. The tape inserter and hem sealer are not a part of the present invention and are omitted from the figures for clarity.

The roll 22 is rotated by means of a motor 32 driving a pulley 34 through a reduction gear assembly. A belt 38 couples the pulley 34 with a pulley 40 coaxial with and mounted to an end of the roll 22. A pulley 42 is provided for tensioning the belt 38. A vacuum line 44 extends from the center of the roll 22 and pulley 40 to a blower motor 46 supporting a blower 48 on an opposing side of the machine, Figs. 1 and 2.

The rotary cutting apparatus 24 further includes a stationary support member 50 coupled to the frame F by another stationary support member 51, Fig. 2. The support member 50 in turn supports a pivot shaft 52, Fig. 1, rotatably supporting an arm 54 which carries a roller 53, Figs. 2 and 3 (omitted from Fig. 1 for clarity) which runs against a cutting die 134 which projects above the surface of roll 22, Figs. 4 and 5, as hereinafter described. A small gap is maintained between the roller 53 and the surface of roll 22 so that the roller 53 does not ride on the web between cutting operations. Retraction and advancement of the arm 54 is controlled by an air cylinder 56, Fig. 1, supported by a bracket 58 mounted to the support member 50. A clevis 60 at the end of piston rod 62 of the cylinder 56 is rotatably coupled to an end of the arm 54. An adjusting screw 64 passing through a portion of the arm 54 contacts a stop 66 mounted to the support member 50 to set the pressure on a leaf spring member 142, Figs. 3 and limits the downward motion of the arm 54 by the cylinder 56. A plate 67 attached to member 50 supports one side of the idler rolls 26, 28. A transverse hand adjustment screw 68 threaded through a member 69, Fig. 2, stationarily mounted to the floor adjusts the position of the frame F into and out of the plane of Fig.

1 for aligning the frame F with a downstream portion of the bag manufacturing line on a separate frame member.

Referring to Figs. 2 and 3, mounting block 51 supports the stationary support member 50 on its upper surface and is supported on horizontal frame member 17. The support block 51 includes a pair of journaled bearing 70 and 72, Fig. 3, rotatably supporting a hollow shaft 74 fixed to the left hand end of roll 22 and to support the pulley 40. A collar 76 holds a projecting hollow end portion 74a of the shaft 74 against a split bearing 78 mounted in an end of a hollow adaptor 80 which connects the vacuum line 44. The adaptor 80 is connected to the horizontal frame member 17 by a bracket 82 and pin 84 to prevent the the vacuum line 44 from turning with the roll or drum 22. The vacuum tube 44 is fit over the remaining open end of the adaptor 80. The right hand end of the hollow shaft 74 supports for rotation therewith a radially extending die assembly 90 which forms part of the rotary cutting apparatus 24 of the present invention. An annular plate member 92 is mounted within the roll 22 recessed from the left hand edge of roll 22 to permit insertion of the assembly 90 within the roll 22. An inner circular opening of the annular plate 92 receives a plugged end 74b of the hollow shaft 74. An O-ring 94 is provided between a shoulder on the shaft 74 adjacent the end 74b and plate 92 and is held in place by the annular compression nut 93 threadedly engaging the opposite end of shaft 74. The extreme radial end of die assembly 90 is exposed through an opening 94 at the left hand axial edge of the cylindrical outer surface of the roll 22. Another annular plate 96 is mounted within the roll 22 at the right hand end and receives a cylindrical plug assembly 98 supported on a stub shaft 100 coaxial with the roll 22 and hollow shaft 74. The assembly 98 includes a cylindrical housing having therein a pair of journal bearings 104, 106 rotatably supporting the roll 22 on the shaft 100. The housing 102 is inserted into a central circular opening in the annular plate 96 and is frictionally coupled thereto for rotation therewith about the shaft 100. The shaft 100 is slideably supported in a pair of flange members 110 and 112 coupled through a plurality of frame member 114, 116 and 118 to a horizontal tubular member 17' symmetric with the horizontal tubular frame member 17. A handle 120 connected with the shaft 100 allows the shaft 100 to be moved axially with respect to the cylinder 22 to insert or withdraw the plug assembly 98 from the annular plate member 96. The roll 22 is adapted to be replaced for different widths of bags. For example the same roll 22 may be used for bag repeat lengths of 20" to 24". Another roll may be used for repeat lengths of 30" to 33". This mounting arrangement provides for ease of re-

moval of the roll 22 for change in repeat lengths between bags. The roll 22 may be temporarily supported by any conventional means such as wedges beneath the roll when plug 98 is withdrawn.

The die assembly 90 is mounted by screws or other suitable means (not shown), to a tubular collar 74c, Fig. 3, extending radially from the hollow shaft 74 and having its hollow interior coupled with the hollow interior of the hollow shaft 74. The assembly 90 includes a hollow, tubular spacer 130 positioned in the end of collar 74c and adapted to receive a die and baffle sub-assembly 132. The spacer 130 is used when a roll size is changed for different bag sizes. The components of sub-assembly 132 are best seen in Figs. 4-6 and include a tungsten carbide die 134 mounted by suitable means such as braising to a hollow metal holder 136 which is mounted by screws 137, Fig. 6, to the underlying tubular spacer or collar 130. A sheet metal baffle plate 138 is positioned within the die 134 and held in place by means of an external annular flange 139 held between the holder 136 and the collar 130. A quarter circle opening 138' is provided in the baffle plate 138 to apply a vacuum to the leading end of the portion of the plastic film web being cut from the web by the die 134. By applying vacuum to the leading end of the cut out portion, the film is stretched during the cutting thereby preventing wrinkling and a jagged cut.

Referring to Figs. 3-5, the cutting action of the rotary cutting apparatus 24 is provided by the cutting die 134 in cooperation with the roller 53. The roller 53 is mounted to "L" shaped support member 55 extending from a side of support arm 54, by an assembly which includes a bracket 140 mounting a shaft 141 supporting the roller 53 and attached to one end of a leaf spring member 142, the opposite end of which is mounted by means x of a bracket 144 and screws 146 to the "L" shaped member 55. It is desirable to minimize the movement of the roller 53 with respect to the die 134 to cut down on the bounce effect. An adjustment screw 148 extending through a threaded bore in the end of support member 55 contacts the leaf spring member 142 to preload the spring to minimize the inertia forces and adjust the height of the roller 53 with respect to the cutting surface or edge 134' of the die 134. As seen in Fig. 5, the roller 53 is supported on bearings 154 and 156 on shaft 141. The roller 53 is desirably made from a material softer than tungsten carbide, preferably A7 tool steel.

The apparatus as described above is used to cut semicircular openings along the hem edge of the web 10 to form hand holes for draw tape bags. As may be seen in Fig. 6, the hem edge of the web 10 passes over only one-half of the cutting die

134. Accordingly, the roller 53 which cooperates with the cutting die 134 to cut the web 10 spans only one-half the circular die 134, Figs. 3 and 5.

As best seen in Fig. 6, the die 134 is circular in plan view. However, as best seen in Figs. 3-5, the contour of the cutting surface or edge 134' is saddle shaped to conform to the cylindrical geometry of the roll 22 and to project slightly and uniformly beyond the surface of the roll 22. Referring to Fig. 3, the radial distance between each point of the cutting edge 134' and the central axis 22' of the roll 22 (i.e. axis of rotation) is uniform. In a specific application of the invention, the height of the spring 142 was adjusted to apply approximately 30 lbs. of force through the roller 53 when used to cut through four thicknesses of 1.4 mil linear low density polyethylene film forming draw tape bags of the type described in the aforesaid related patent application Serial No. 652,254. At speeds of up to 135 cycles per minute, a rotary cutting apparatus according to the present invention was capable of providing a service life of about 7,000,000 (holes) cycles when a tool steel roller is used in combination with a tungsten carbide die. This compares with a service life of merely thousands of cycles for contacting roll type cutters.

When a tool steel planar wiper blade was mounted to the leaf spring member 142 in place of the roller 53 for contacting the cutting surface 134' of tungsten carbide die 134 with an interference between the blade and the die of about .003-.004 inch, service life of this combination was reduced to approximately 500,000 cycles.

While the invention has been described thus far in connection with cutting holes of substantial size in a thermoplastic film web at high speeds, the invention is also applicable to cutting a line of perforations across a plastic film web moving at high speeds. An embodiment of the invention for cutting perforations in thermoplastic film is illustrated in Figs. 7-9. The web perforator 200 illustrated in Figs. 7-9 utilizes the rotary cutting concept of the present invention and is applicable for making perforations for bags on a roll of consumer bags and other perforation applications. A thermoplastic film web 10 is fed into the web perforator 200 by draw rolls 201 and 202 and onto a series of horizontal belts 204 which also pass over the input drive roll 201 and are driven therefrom. The belts 204 provide a support for the film web 10 and while three have been illustrated in Fig. 8 it is to be understood that additional belts can be utilized. In one embodiment of the invention six belts were used in connection with six cutting apparatus. The belts 204 pass over and drive a roll 205 Figs 7 and 9, which in turn drives a corresponding plurality of horizontal belts 206. The belts 206 also provide a support for web 10 after it leaves the belts 204.

The belts 206 pass over a roll 207 at the right hand end of the web perforator 200 as shown in Figs. 7 and 8. As may be seen in Fig. 8 the belts 204 and 206 are staggered with respect to each other. The web perforator 200 has been illustrated with two cutting stations I and II spaced horizontally with an adjustable on the fly phase angle between them so that the cuts or perforations will occur at substantially the same point, i.e. along approximately the same line on the film web 10. It is to be understood that the perforations made at the two stations I and II are offset approximately 1/32" so as not to completely sever the web as it continues on through the bag line beyond station II. Each of the cutting stations I and II is provided with a plurality of spaced and staggered rotary cutting apparatus embodying the present invention, for example six at each station referred to generally by the reference numeral 24'. The two cutting or perforating stations I and II are spaced apart to provide the necessary space to support the plurality of the small cutting rollers 53 of the rotary cutting apparatus 24' and provide the additional advantage of being able to support the web 10 as it moves through the perforating stations on the conveyor belts. For convenience the corresponding parts in this embodiment have been provided with corresponding reference characters.

Each rotary cutting apparatus 24' is the same and comprises a tool steel roller 53 rotatably supported on the shaft 141, Fig. 9, of a bracket 140 fixed to a leaf spring 142 by suitable means such as screw 143. The spring 142 is also mounted by means of a bracket 144 and screw 146 to an end of a support arm 155. Cooperating with the roller 53 is a tungsten carbide die or cutter 231 in the form of a flat, substantially rectangular plate member held by means of a clamp 232 and screw 233 to a die holder block 234. The holder block 234 is in turn mounted to an elongated rectangular rotary shaft 237, the cylindrical ends 238 of which are supported in bearings for rotation about the shaft axis 239. The spacing of the cutter die 231 from the rotary shaft 237 is adjustable and is accomplished by means a lock screw 241, received in a bore in the head portion 234a of the holder block 234 remote from the rotary shaft 237 and a hollow, coaxial adjustment screw 236. As shown in Fig. 7, another cutter die is adapted to be supported on a second die holder block 234 extending radially in opposite direction from the opposing side of the rotary shaft 237 to permit two cuts to be made in the web 10 during each rotation of shaft 237. The shaft 237 is rotated clockwise in the direction of the arrow along a circular arc to move the die 231 in the same direction as the web 10. The cutter die 231 moves at the same speed as the film web 10 to minimize the chance of the cutting action break-

ing a fragile perforation. As may be seen in Fig. 9 the cutter roll 53 cooperates with the cutting edge 231a of the cutter die 231 in perforating the web 10. The axis of the cutter rolls 53, as shown in Fig. 8 preferably is skewed or inclined 15 relative to the cutting edge 231a such that the cutting of the web is always at a point contact rather than a line contact. When the die edge 231a engages the roller 53, Fig. 8, it tends to straighten the roller and force it to one side. Accordingly, a compression spring 153 is provided at one end of the roller 53 to push the roller back in position. It has been found that approximately 30 lbs. of pressure is required between the roll 53 and the cutting edge 231a to cut 6 mils of film.

As previously described and as illustrated in Figs. 7 and 8, the web perforator 200 is provided with a plurality of spaced and staggered cutting apparatus 24' at each of the two cutting stations I and II. The cutter dies 231 of the rotary cutting apparatus 24' at the first cutting station I are driven by the shaft 237. The cutter dies 231 of the rotary cutting apparatus 24' at the second cutting station II are illustrated at the right hand side of Figs. 7 and 8. These cutter dies 231 at the second cutting station are driven from the rotary shaft 237'. The two shafts 237 and 237' are synchronized by means of a common gear belt 242 which passes around the pulleys 243 and 244 on the respective shafts 237 and 237' and the intermediate or idler pulleys 246-251.

Since the rotary cutting devices 24' at the first cutting station I are spaced from each other transversely of the web this will provide a series of cuts or perforations in the web which are likewise spaced transversely from each other but on the same line. A similar series of staggered spaced cuts or perforations will be made by the rotary cutting devices 24' at station II when the web 10 passes the second cutting station II. By adjusting the phase angle between the cutter dies 231 on shaft 237 at the first cutting station I and the cutter dies 231 on the shaft 237' at the second cutting station II, it can be assured that the staggered cuts or perforations in the web will occur at the required point on the web 10 and with the desired offset as previously described. In one embodiment the web perforator was designed for a width capability of 12" and with suitable adjustments to handle a repeat distance between perforations of 18" to 30". This repeat flexibility is accomplished by providing the rotating cutting edges 231a of the cutter dies 231 with an adjustable radius as previously described.

Claims

1. Apparatus for cutting a moving plastic film web at predetermined locations:

a die having a cutting surface;

means supporting said die for rotating said cutting surface along a circular arc; and

roller means positioned at a stationary location along said circular arc for cyclicly contacting and rolling across said cutting surface for cutting a plastic film web passing between said roller means and said cutting surface at the predetermined locations in the web.

2. The apparatus of claim 1 wherein said cutting surface of said die is made of tungsten carbide.

3. The apparatus of claim 2 wherein said roller means comprises a roller of tool steel.

4. The apparatus of claim 1 wherein said cutting surface of said die has a straight cutting edge and said roller means is skewed with respect to said cutting edge.

5. The apparatus of claim 1 wherein said die has a curved cutting edge for cutting a hole in the web.

6. Apparatus according to claim 1 comprising a first cutting station having a plurality of dies having cutting surfaces, said dies being mounted adjacent said means supporting said first-named die for rotating said cutting surfaces along parallel circular arcs; and a corresponding plurality of roller means positioned at said first cutting station for cyclicly contacting and rolling across the corresponding cutting surfaces of said dies, said dies and said roller means being spaced transversely of the film web at the first cutting station for concurrently making a plurality of spaced cuts across the plastic film web.

7. Apparatus according to claim 6 comprising a second cutting station spaced from said first cutting station along the path of movement of the film web, means for supporting and moving the film web from said first cutting station to said second cutting station, said second cutting station having a second plurality of dies having cutting surfaces, said second plurality of dies at said second cutting station being mounted on means supporting said dies for rotating said cutting surfaces thereof along parallel circular arcs, and a corresponding second plurality of roller means positioned at said second cutting station for cyclicly contacting and rolling across the corresponding cutting surfaces of said dies at said second cutting station, said dies and said roller means at said second cutting station being spaced transversely of the film web for concurrently making a second plurality of spaced cuts

across the plastic film web and in line with the first plurality of spaced cuts made at said first cutting station.

8. Apparatus according to claim 7 wherein said means for rotating said dies at said first and second cutting stations are synchronized for rotating said cutting surfaces of said dies at both of said cutting stations at the same speed along their respective circular arcs.

9. Apparatus according to claim 8 wherein said means for supporting and moving the film web from said first cutting station to said second cutting station is synchronized with said means for rotating said dies at said first and second cutting stations so that the speed of the film web is the same as the speed of said cutting surfaces of said dies.

10. Apparatus according to claim 6 wherein the axis of said plurality of roller means is skewed with respect to the cutting surfaces of said corresponding dies.

11. Apparatus according to claim 10 wherein said plurality of roller means are biased in the axial direction to maintain said roller means in position when said roller means cyclicly contact and roll across the corresponding cutting surfaces of said dies.

12. Apparatus according to claim 10 wherein the axis of said plurality of roller means at both of said cutting stations is skewed with respect to the cutting surfaces of said corresponding dies.

13. Apparatus according to claim 10 wherein said plurality of roller means at both of said cutting stations are biased in the axial direction to maintain said roller means in position when said roller means cyclicly contact and roll across the corresponding cutting surfaces of said dies.

14. Apparatus according to claim 7 wherein said means for supporting and moving the plastic film web from the first cutting station to the second cutting station comprises a plurality of belts for supporting the plastic film web and extending along the path of movement of the plastic film web, the belts being spaced transversely of the path of movement so as to permit cooperation of said cutting surfaces of said dies and the cooperating said roller means at both of said cutting stations.

15. A web cutting device comprising:

roll means mounted for rotation and having a cylindrical outer surface;

a die mounted within said roll means and having a cutting surface substantially conforming in curvature to said cylindrical outer surface and extending above said cylindrical outer surface for engaging one side of the web;

a roller positioned adjacent said cylindrical outer surface spanning at least a portion of the die cutting surface for engaging the other side of the

web; and

means for biasing said roller against said die cutting surface for cutting the web therebetween.

16. The apparatus of claim 15 wherein said roll means has a central axis of rotation and each point of said cutting surface is radially spaced the same distance from said central axis.

17. The apparatus of claim 16 wherein said cutting surface further forms a closed curve.

18. The apparatus of claim 17 wherein said die has an opening within said closed curve cutting surface and further comprising blower means coupled with said die for applying a vacuum through said opening in said die.

19. The apparatus of claim 18 including baffle positioned within said die covering a portion of said opening of said die within said closed curve cutting surface.

20. The apparatus of claim 15 wherein said cutting surface of said die is tungsten carbide.

21. The apparatus of claim 20 wherein said roller comprises a tool steel cylinder.

22. A method of cutting a moving plastic film web at spaced locations comprising the steps of:

rotatingly supporting a roller;

rotating a cutting edge about an axis of rotation in a circular arc intersecting the roller to cyclicly contact the roller with the cutting edge; and

passing a plastic film web between the roller and the rotating cutting edge for cutting the web at spaced location corresponding with the cyclical contact of the roller and the rotating cutting edge.

23. The method of claim 22 for cutting a portion of the plastic film web from the remainder of the web comprising the steps of applying a vacuum to the leading ends of the portion of the plastic film web being cut from the web, to stretch the film during the cutting operation thereby preventing wrinkling and a jagged cut.

24. A method according to claim 22 comprising the steps of:

providing at a first cutting station a plurality of dies having cutting edges positioned on one side of the plastic film web, providing a corresponding plurality of rollers at the first cutting station and positioned on the opposite side of the plastic film web, the dies and the rollers being spaced transversely of the plastic film web at the first cutting station, and rotating the cutting edges of the dies about a common axis of rotation in parallel circular arcs intersecting the rollers to cyclicly contact the rollers with the cutting edges for concurrently making a plurality of spaced cuts across the plastic film web.

25. The method of claim 24 further comprising the steps of:

providing a second cutting station spaced from the first cutting station along the path of movement

of the plastic film web, supporting and moving the plastic film web from the first cutting station to the second cutting station, providing at the second cutting station a plurality of dies having cutting edges positioned on one side of the plastic film web, providing a corresponding plurality of rollers at the second cutting station and positioned on the opposite side of the plastic film web, the dies and the rollers being spaced transversely of the plastic film web at the second cutting station, and rotating the cutting edges of the dies at the second cutting station about a common axis of rotating in parallel circular arcs intersecting the rollers at the second cutting station to cyclicly contact the rollers with the cutting edges for concurrently making a second plurality of spaced cuts across the plastic film web staggered and in line with the first plurality of spaced cuts previously made at the first cutting station.

26. A method according to claim 25 wherein the cutting edges of the dies at both of the cutting stations are rotated at the same speed along their respective circular arcs.

27. A method according to claim 26 wherein the movement of the plastic film web from the first cutting station to the second cutting station is synchronized with the rotation of the cutting edges of the dies at both of the cutting stations so that the speed of the plastic film web is the same as the speed of the cutting edges of the dies along their respective circular arcs.

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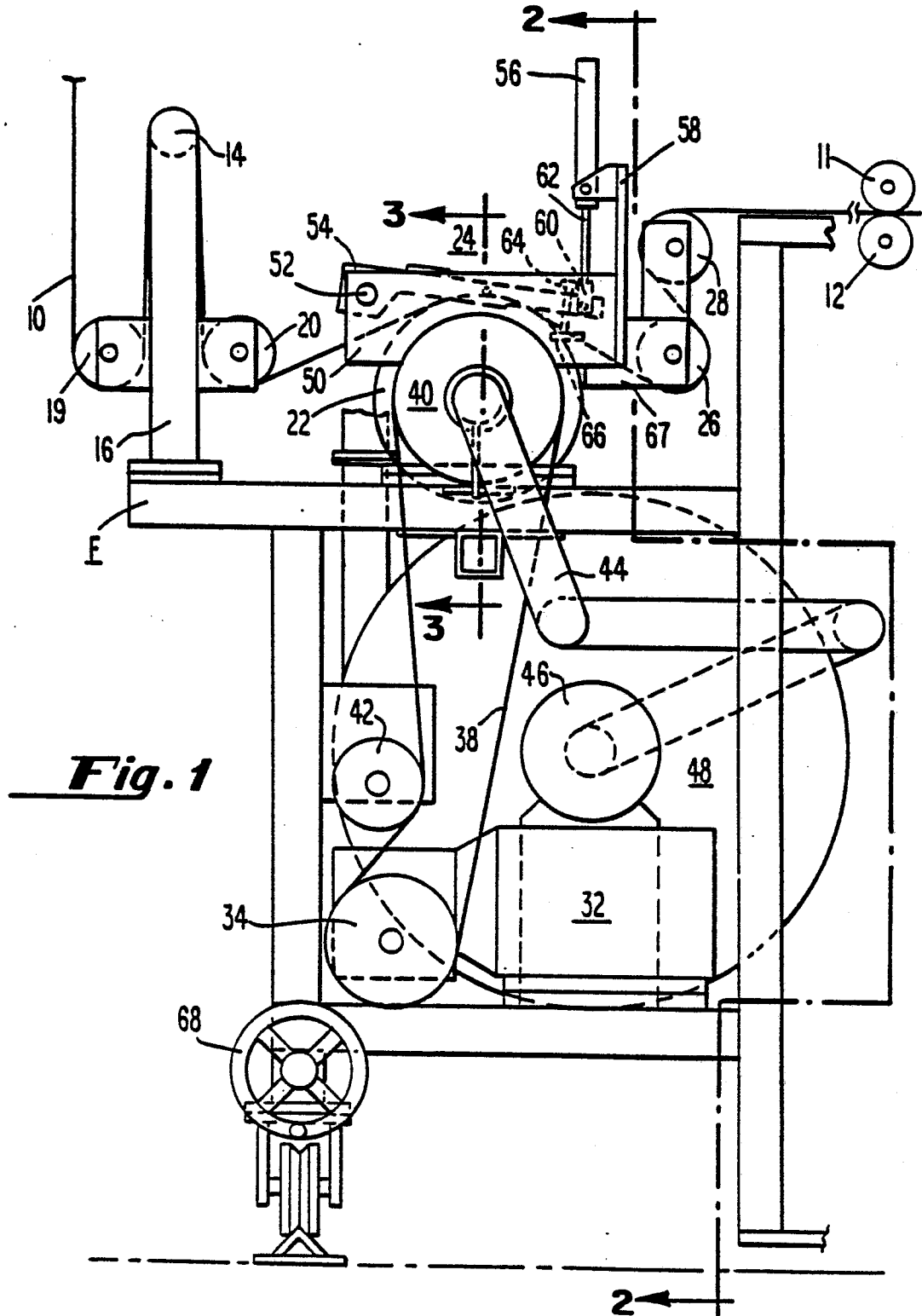
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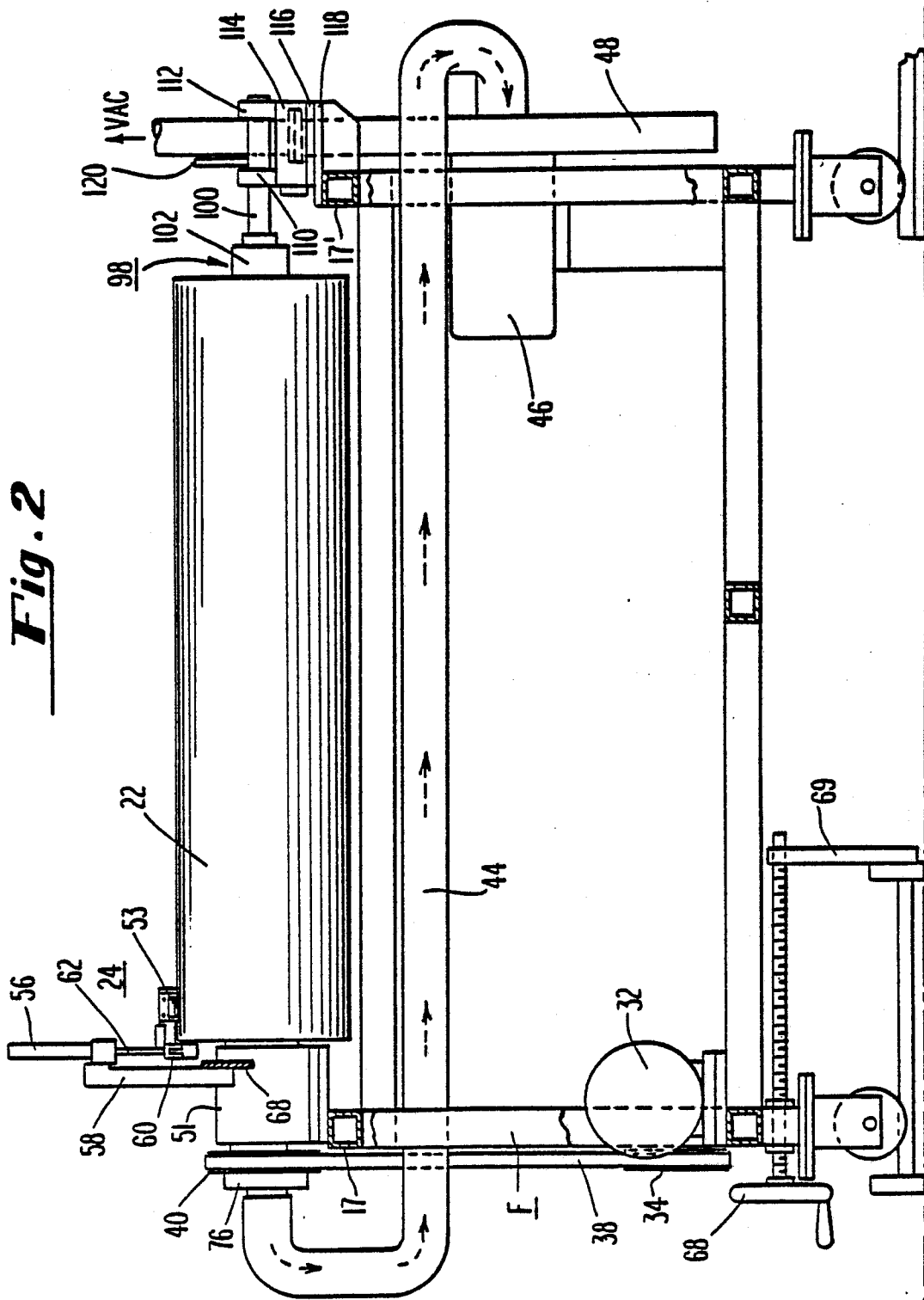
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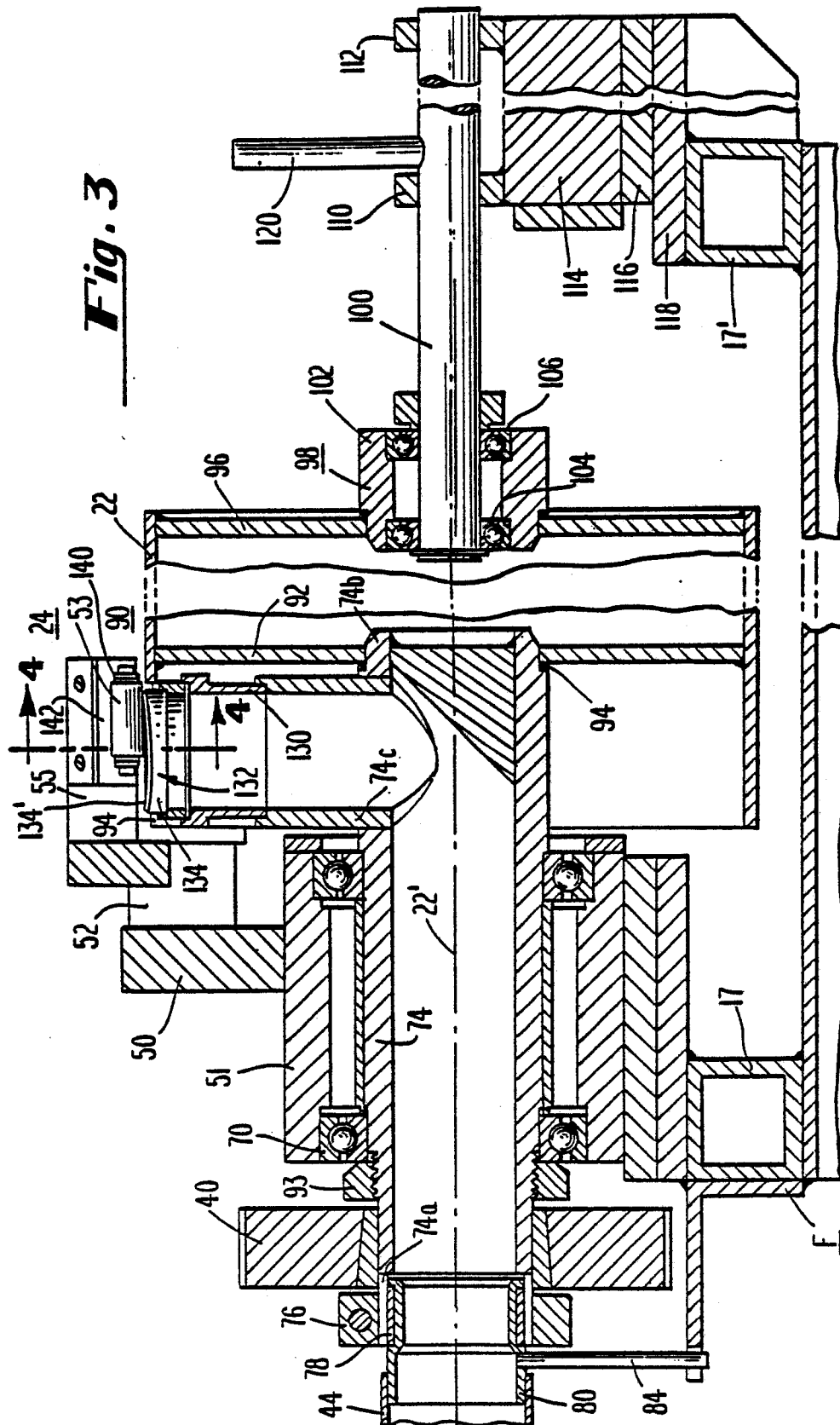
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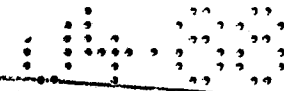


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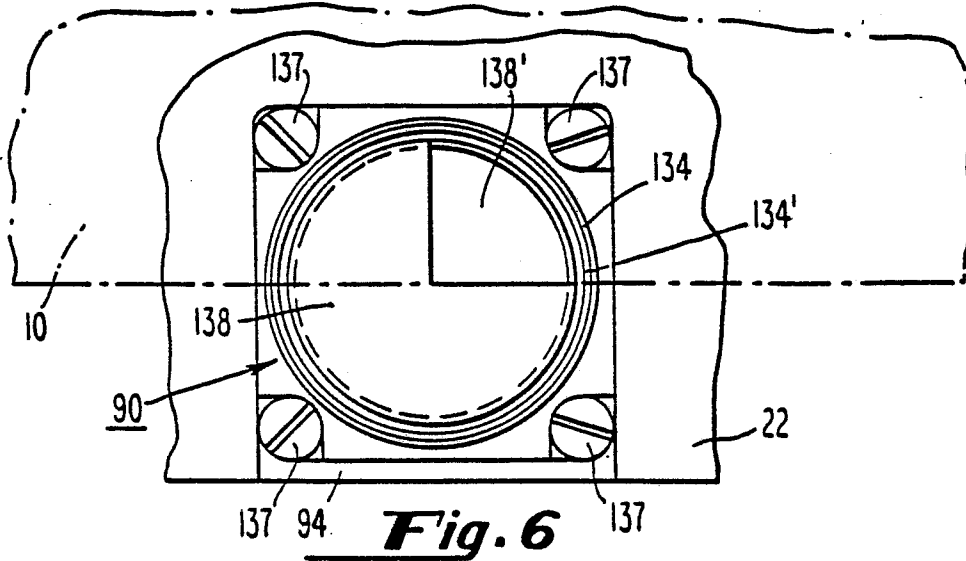
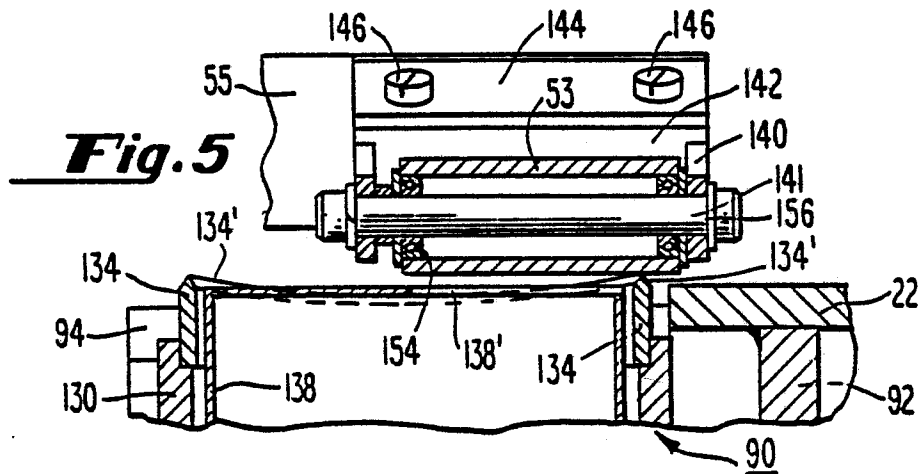
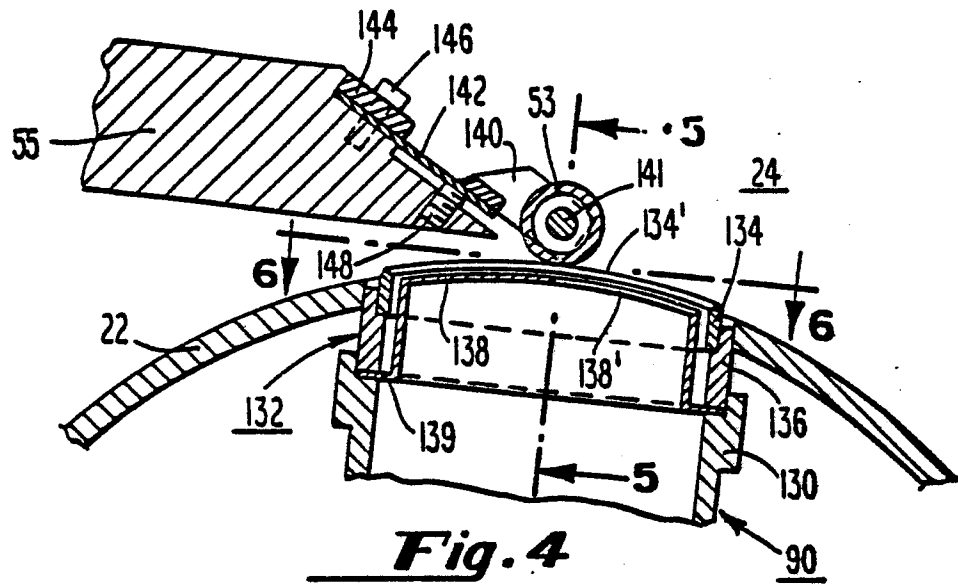
Fig. 2







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Nouvellement déposé



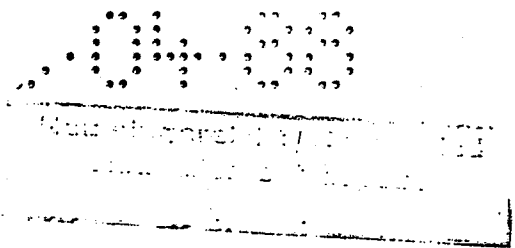
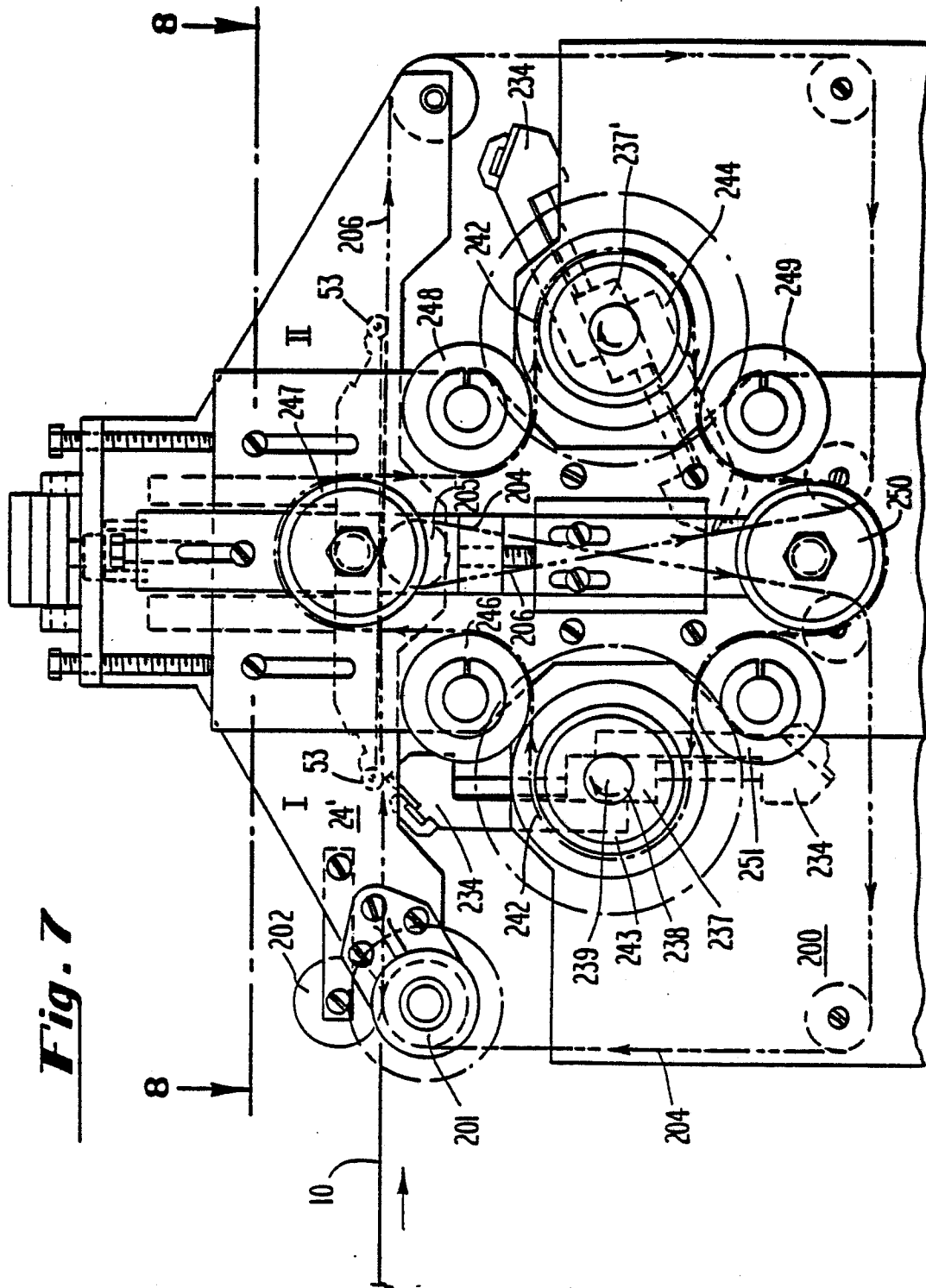


Fig. 7



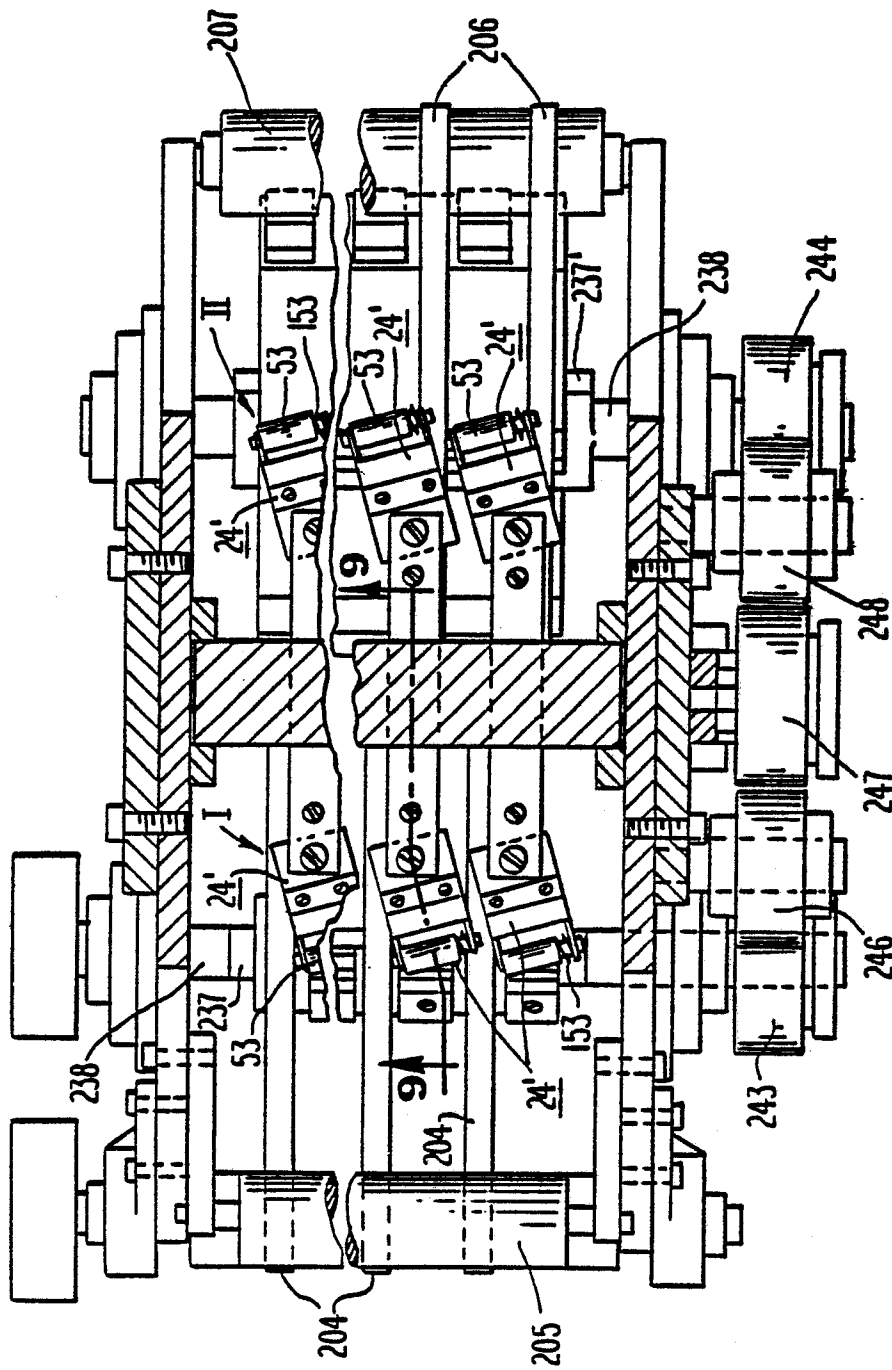
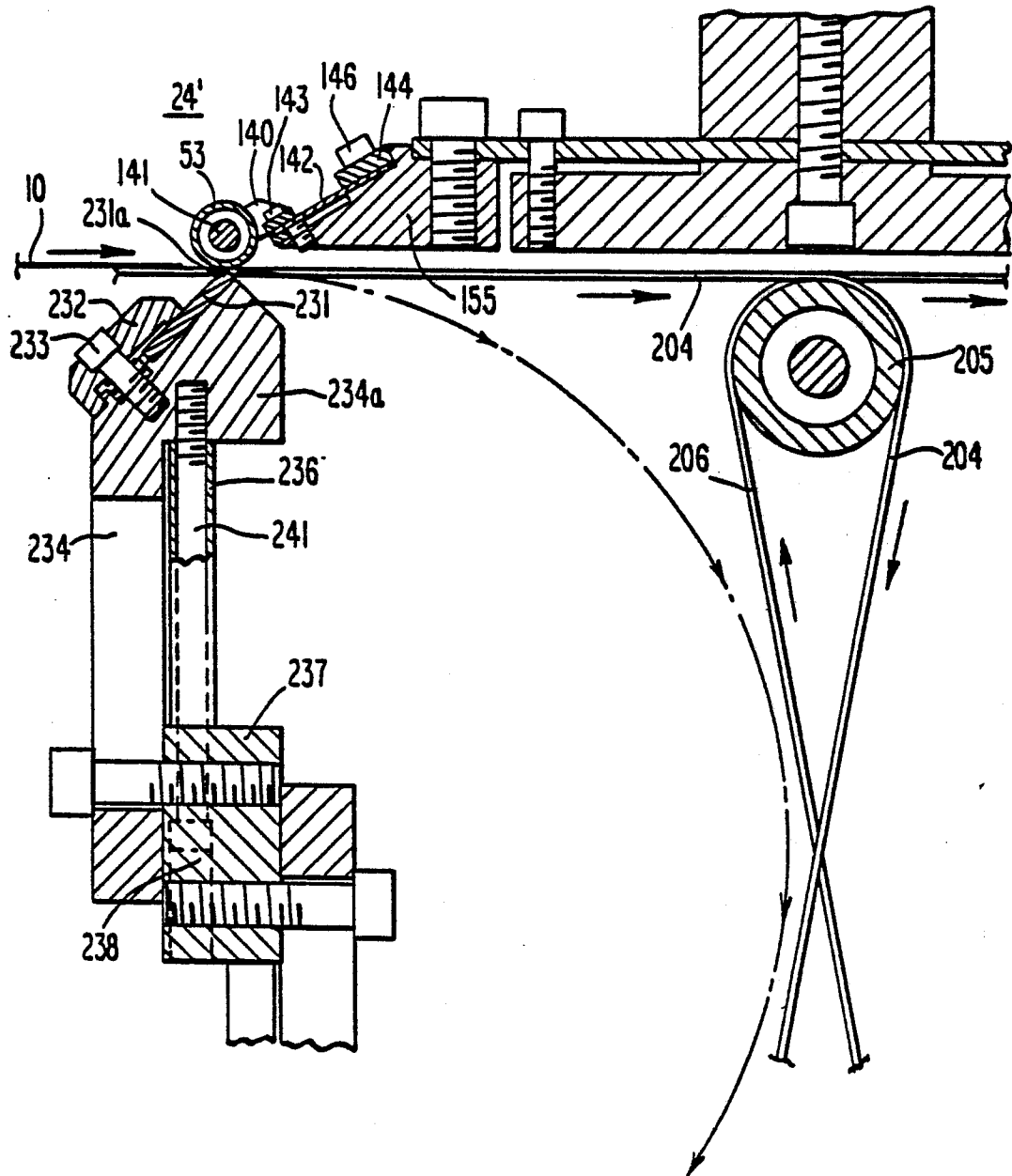


Fig. 8

**Fig. 9**



European Patent
Office

EUROPEAN SEARCH REPORT

Application number

EP 88303102.3

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl. 4)
X	DE - A1 - 3 302 038 (WINKLER) * Fig. 1 * --	1	B 26 F 1/22 B 31 B 1/24
X	US - A - 3 106 121 (NOWICK) * Totality * --	1	
X	DE - A1 - 2 722 972 (WINKLER) * Fig. 2,3 * --	1	
X	DE - A - 2 054 474 (IBM) . * Fig. 5 * -----	1	
			TECHNICAL FIELDS SEARCHED (Int. Cl. 4)
			B 26 F B 31 B B 29 C B 26 D
The present search report has been drawn up for all claims			
Place of search VIENNA		Date of completion of the search 02-08-1988	Examiner TRATTNER
CATEGORY OF CITED DOCUMENTS			
X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document		T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document	