

12

EUROPEAN PATENT APPLICATION

21 Application number: 88115214.4

51 Int. Cl. 4: **F41D 7/00**

22 Date of filing: 17.10.86

30 Priority: 30.10.85 US 793013
18.12.85 US 878470

43 Date of publication of application:
08.03.89 Bulletin 89/10

60 Publication number of the earlier application in
accordance with Art.76 EPC: 0 220 630

84 Designated Contracting States:
CH DE FR GB LI SE

71 Applicant: **GENERAL ELECTRIC COMPANY**
1 River Road
Schenectady New York 12305(US)

72 Inventor: Pettinga, Robert
Box 2397 R.D. 2
Shelburne Vermont 05482(US)
Inventor: Dix, Joseph
Box 486 R.R.
Williston Vermont 05495(US)
Inventor: Forrester, Victor James
179 Staniford Road
Burlington Vermont 05401(US)
Inventor: McHugh, Thomas Owen
148 Staniford Road
Burlington Vermont 05401(US)
Inventor: Sibley, Lincoln Lewis, Jr.
80 Austin Drive No. 174
Burlington Vermont 05401(US)
Inventor: Ziemba, Richard Thomas
99 South Cove Road
Burlington Vermont 05401(US)

74 Representative: Catherine, Alain
General Electric France Service de Propriété
Industrielle 18 Rue Horace Vernet B.P. 76
F-92134 Issy-les-Moulineaux Cédex(FR)

54 External drive for a machine gun.

EP 0 306 061 A2
 57 An external drive for a gun having an operating
 mechanism including a rocking lever journaled for
 oscillation about a central axis, comprising a motor
 having an output shaft which is coupled through an
 intermittent drive mechanism to the rocking lever.

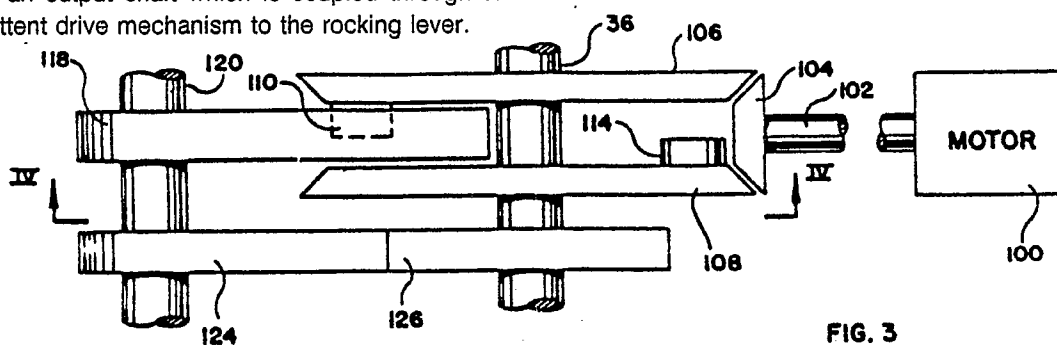


FIG. 3

EXTERNAL DRIVE FOR A MACHINE GUN

This invention relates to an external drive particularly suitable for a machine gun having two gun barrels and a linkage which alternatively drives each breech bolt via a cam operated accelerator.

A two barreled machine gun having an operating mechanism driven by an external source is shown in US-A-430206. In this gun, a hand crank or a motor rotates a shaft to rotate two 180° out of phase cams, which cams respectively oscillate two levers, which levers reciprocate respective loading and firing bolts.

It is an object of this invention to provide an external drive that oscillates a rocking lever in the drive linkage of a machine gun.

According to the present invention there is provided an external drive for a gun having an operating mechanism including a rocking lever journaled for oscillation about a central axis, characterized in that it comprises a motor having an output shaft which is coupled through an intermittent drive mechanism to said rocking lever.

These and other objects, features and advantages of the invention will become apparent from the following specification thereof taken in conjunction with the accompanying drawings in which:

FIG.1 is a perspective view of a gun embodying this invention;

FIG.2 is a detail of FIG.1 showing the operating linkages contained in the receiver assembly of the gun of FIG.1;

FIG.3 and 4 are a top view and a side view respectively in elevation of an external drive, and having a rotary input coupled to the operating mechanism; and

FIG.5 is a perspective view, similar to FIG.2, showing an external drive having a rectilinear, reciprocating input coupled to the operating mechanism.

As seen in FIG.1, the gun includes a left hand gun barrel 10L and a right hand gun barrel 10R which are secured to a breech housing 12 of a breech assembly 14. The barrels are also constrained by a muzzle clamp assembly 16, a mid-barrel clamp assembly 18, and an aft clamp assembly 22. The breech housing 12 is fixed to a receiver housing 26 in a receiver assembly 28. A hammer assembly 30 is also fixed to the breech housing 12. The breech and receiver housings may be considered together to be the classical "receiver".

As seen in FIG.2, the operating mechanism 32 is mainly disposed within the receiver housing 26 and is included in the receiver assembly 28. This operating mechanism includes a rocking lever 34 which is fixed to a central post 36, and which post

is journaled for oscillation to the receiver housing 26. The rocking lever 34 has a left arm 34L to which is fixed a left post 38L which is journaled to the forward end of a left link 40L, and has a right arm (not visible) to which is fixed a right post (not visible) which is journaled to the forward end of a right link 40R. The aft end of the left link 40L is journaled to a left post 42L which is fixed to a left slider 44L. The aft end of the right link 40R is journaled to a right post (not visible) which is fixed to a right slider 44R which is identical to the left slider 44L. The aft end of each link 40L/R has a projection 40P which is captured in a respective notch in the respective slider 44L/R to capture the link to its respective post 42L/R. An additional post 46 is fixed to the rocking lever 34 and is journaled to the forward end of a feeder arm 48. The aft end of the feeder arm 48 has a post (not visible) to which is pivotally journaled a tail slider 52. The aft end of the slider has a cross-piece 54 which rides in a channel 56 in the receiver housing 26. The forward end of the slider 52 has a cam driver 58 with two mutually spaced apart upstanding arms which terminate in mutually opposed cam elements 60L and 60R. The cam driver 58 also has a pair of slots 62L and 62R which ride on a pair of tracks (not visible) in the receiver housing. The cam driver is thereby constrained to rectilinear reciprocating movement when driven by the feeder arm 48 whose forward end is free to oscillate laterally while reciprocated fore and aft by the rocking lever 34. The cam driver drives (not shown here) a drum cam which in turn drives a feed sprocket to advance a belt of linked ammunition transversely through the gun. Each full cycle of oscillation of the rocking lever 34 results in the sprocket advancing the belt of ammunition by two rounds. The aft ends of the two sliders 44L/R are respectively coupled to two accelerator arms which in turn are respectively coupled to two bolt assemblies. Each full cycle of oscillation of the rocking lever 34 results in the fore and aft reciprocation of both bolt assemblies to chamber and fire of two rounds of ammunition. Each of the two sliders also cooperates (not shown here) with a respective set of front and rear strippers which serve to strip a round of ammunition from the belt and to place the round on the face of the bolt assembly, and to lock the bolt assembly when the round is chambered.

As seen in FIGS 3 and 4, the external drive assembly includes an electric motor 100, fixed to the receiver housing, whose output shaft 102 is fixed to a bevel pinion gear 104. The pinion gear 104 is meshed with two mutually opposed bevel gears 106, 108 respectively journaled for rotation

about a common axis, in opposite direction. A lower cam roller 114 is journaled for rotation on a stub shaft 116 which is carried by the lower gear 108. An upper cam roller 110 is journaled for rotation on a stub shaft 112 which is carried by the upper gear 106. A shaft 120 is splined to the central post 36 shown in FIG.2, which post is fixed to the rocking lever 34. A bifurcated cam follower 118 is fixed to the shaft 120 and its bifurcation projects between the gears 106 and 108 so that the rollers 110 and 114 may ride into and out of the cleft cam following surface 122 of the bifurcation. The shafts 102, 120, and post 36 are each respectively supported by stationary journals, (not shown). Shaft 120 is fixed to a first gear 124 which is meshed with a second gear 126 which is journaled for oscillation about said central axis of said rocking lever and connected to said rocking lever 34, such as by a common shaft 36 fixed to both. Second gear 126 may be integrally with said rocking lever 34.

As viewed in FIG.4, gear 108 is driven counter-clockwise so that the roller 114 is carried counter-clockwise and swings the bifurcated cam follower 118 with the shaft 120 and the rocking lever 34 clockwise until the roller 114 exits the cleft 112 and the swing of the follower 118 halts. Meanwhile, gear 104 drives the gear 106 clockwise until it enters the cleft 122 and swings the follower 118, and thereby, the rocking lever 34 counter-clockwise until the roller 110 exits the cleft 122 and the swing of the follower 118 halts until the roller 114 reenters the cleft, and so on.

This oscillation is communicated to gear 124 and thereby to gear 26 and the post 36 and rocking lever 34.

Alternatively, an external drive may be substituted as shown in FIG.5. This external drive assembly includes a double acting cylinder 500 having a piston 502 whose forward end is connected by a pivotal joint 504 to a journal rod 506 which is journaled on a post 508 which is fixed to and depends from the underside of the rocking lever 34 and is eccentric to the central post 36. Aftward movement of the piston 502 pulls the rocking lever through the disposition shown in FIG.5 to bring the left slider 44L into its aft disposition. Forward movement of the piston 502 pushes the rocking lever to bring the right slider 44R into its aft disposition.

Claims

1- An external drive for a gun having an operating mechanism including a rocking lever (34) journaled for oscillation about a central axis (36), characterized in that it comprises a motor (100) having

an output shaft (102) which is coupled through an intermittent drive mechanism to said rocking lever (34).

2- A drive according to claim 1 characterized in that it comprises;

- a pinion gear (104) driven by said motor (100);
- said pinion gear (104) being meshed with and driving two mutually opposed gears (108, 106), respectively journaled for rotation about a common axis (36), in opposite direction;

- a cam follower (118) having a cleft cam surface (122) disposed between said two opposed gears (108, 106) and journaled for oscillation about an axis to and between two angularly spaced apart dispositions;

- two cams (114, 110), each carried by a respective one of said two opposed gears (108, 106), and one of said cams being adapted to enter said cleft at one of said dispositions and to swing said follower (118) to the other of said dispositions and to exit said cleft, and the other of said cams being adapted to enter said cleft at said other of said dispositions and to swing said follower (118) to said one of said dispositions and to exit said cleft.

3- A drive according to claim 2 wherein said cam follower (118) is fixed to a shaft (120) which is fixed to a first gear (124) which is meshed with a second gear (126) which is journaled for oscillation about said central axis of said rocking lever (34) and connected to said rocking lever; having a mode operation that

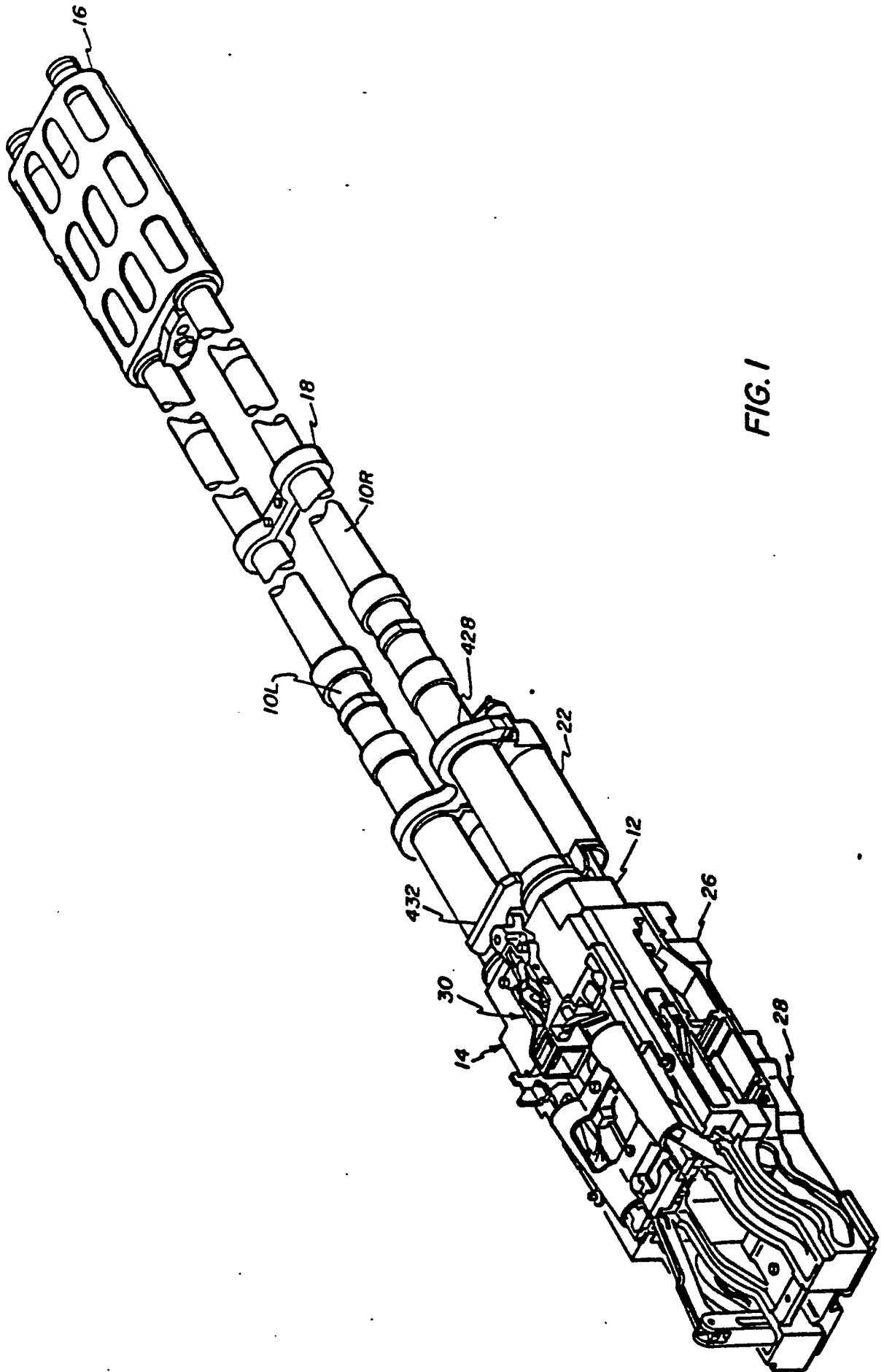
continuous rotation of said motor (100) provides rotation of said two cams (114, 110) in respective opposite directions, one of said cams swinging said follower (118) in one direction and the other of said cams swing said follower in the other direction and such oscillation of said follower (118) providing oscillation of said rocking lever.

4- A drive according to claim 2 wherein:

said second gear (126) is connected to said rocking lever (34) by a common shaft (36) fixed to both.

5- A drive according to claim 2 wherein:

said second gear (126) is integral with said rocking lever (34).



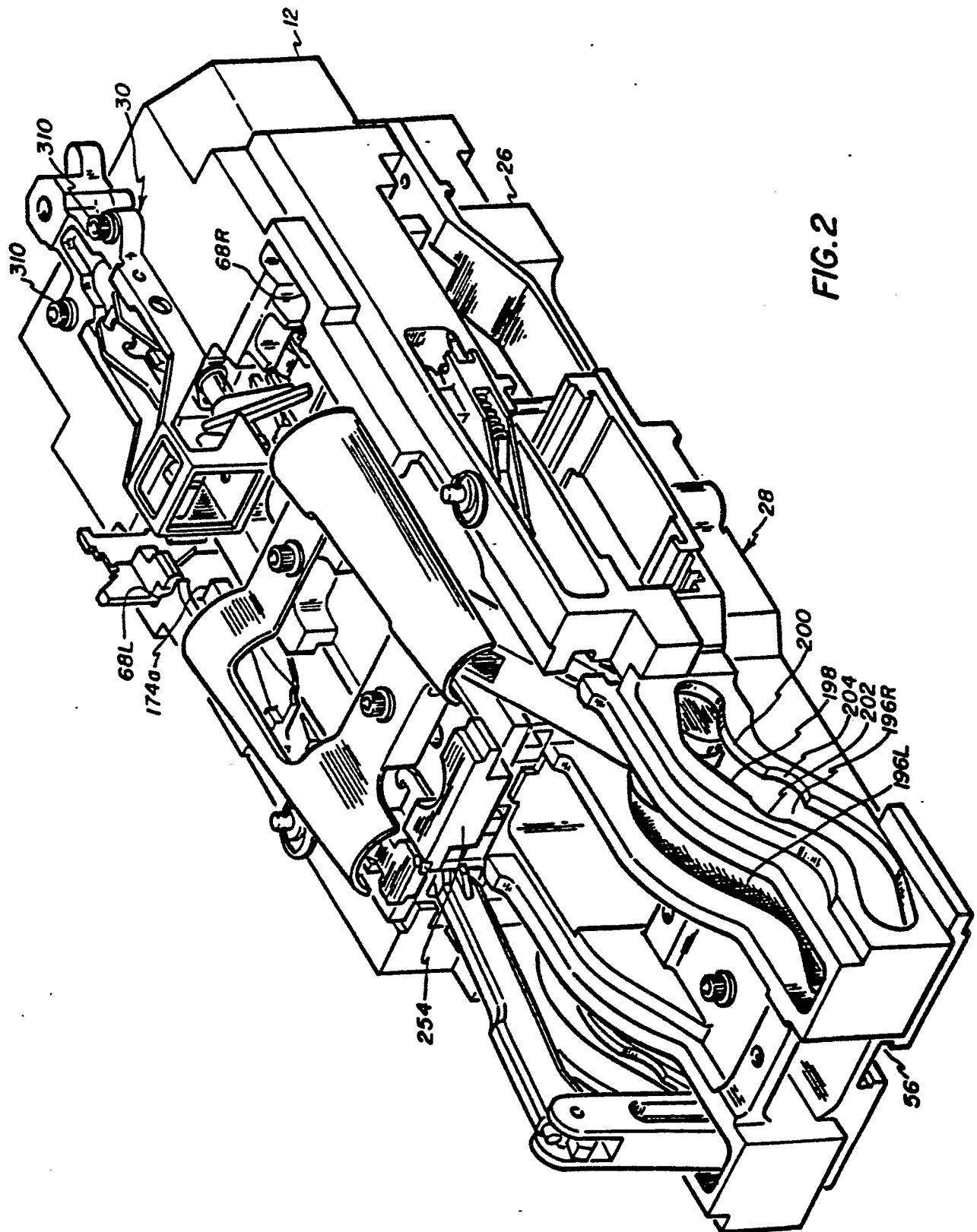


FIG. 2

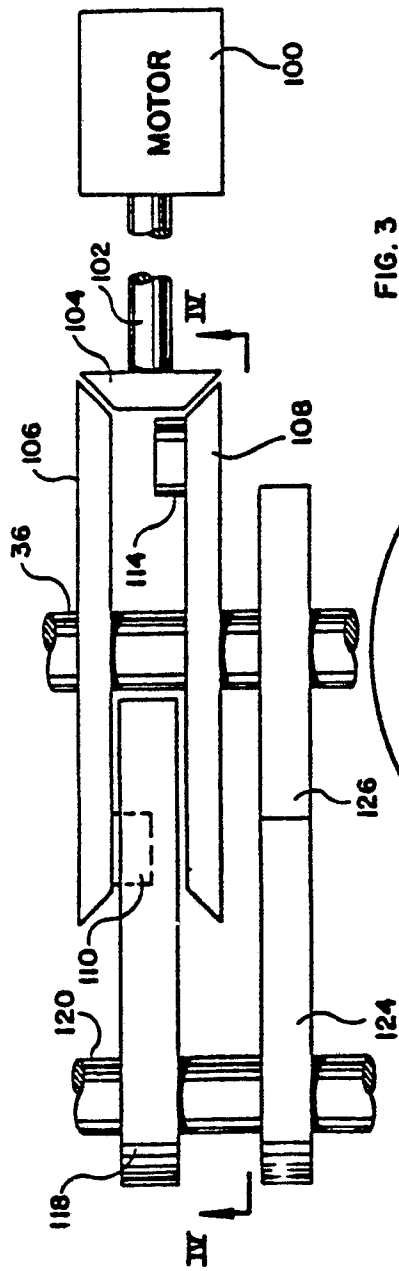


FIG. 3

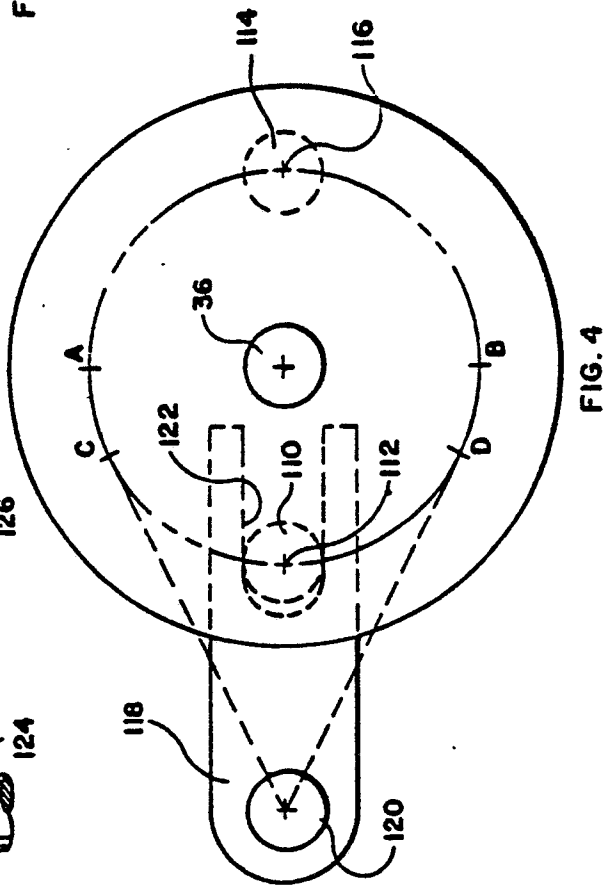


FIG. 4

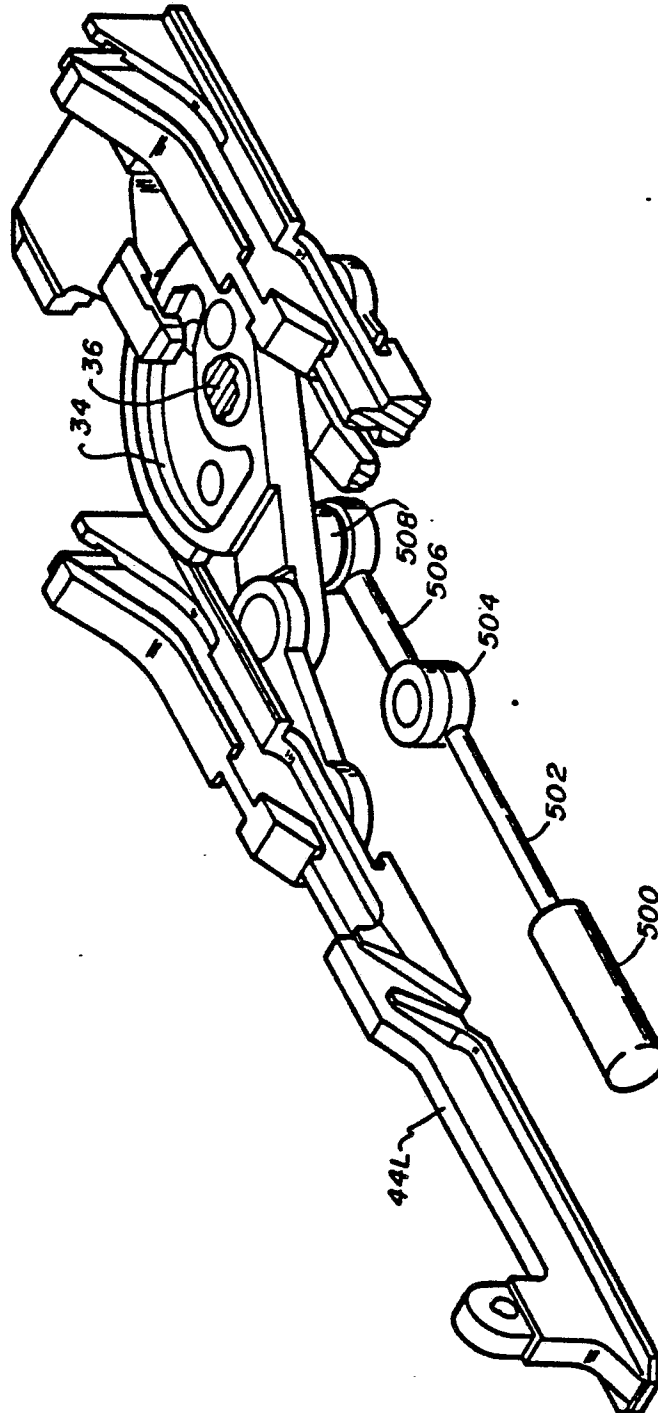


FIG. 5