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(54) **MECHANISM FOR CONTROLLING THE FIRING RATE OF AN AUTOMATIC WEAPON**

VORRICHTUNG ZUR REGELUNG DER FEUERGESCHWINDIGKEIT AUTOMATISCHER
FEUERWAFFEN

MECANISME POUR COMMANDER LA VITESSE DE TIR D'UNE ARME AUTOMATIQUE

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BE-A- 902 696 **CH-A- 536 995**
DE-A- 1 428 774 **DE-C- 635 896**
US-A- 2 448 024

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Description

This invention relates to a mechanism for controlling the rate of fire of a sub-machine gun.

It is well-known that sub-machine guns have a tendency to 'climb' during automatic fire. As each round is fired, the weapon recoils upwardly and the muzzle tilts to some degree and thereafter returns to the approximate original point of aim. If the rate of fire in fully-automatic firing mode is too high, each successive round will be fired before the muzzle returns on target following the discharge of the preceding round and will therefore be high off target. Each successive round will therefore be aimed higher than its predecessor. The end result is what is known as 'climb' - that is, a burst of rounds which start on target and thereafter climb upwards off target. As the rate of fire increases in relation to a specific weapon so will there be a tendency for the rate of 'climb' to increase.

It has been found that a rate of fire in the order of 400 to 600 rounds per minute can be reasonably controlled by a trained user. At least to some extent this controllability is dependent upon the design and weight of the sub-machine gun; and also upon the training of the user. Important factors contributing to the rate of fire are the weight of the bolt and the distance the bolt travels.

Previous attempts to reduce the size and weight of sub-machine guns have usually necessitated a reduction in the size and weight of the bolt and a reduction in the distance the bolt is allowed to travel. Consequently, this has resulted in a bolt with a faster action giving weapons having rates of fire beyond the desirable maximum rate of about 600 rounds per minute. These efforts in size reduction have therefore resulted in weapons with varying degrees of climb and hence lack of control.

A number of efforts to control the firing rate of a sub-machine gun have involved increasing the mass of the bolt and/or regulating the movement of the bolt. In the first case the physical size of the weapon is reduced but the actual weight of the bolt is increased by use of 'dense material' such as alloys of tungsten, or infilling the bolt with lead. This method is self-defeating in that the overall weight of the gun is increased. Moreover, even the use of the densest materials available gives only a minimal firing rate reduction. Reducing a firing rate from 1,800 RPM to 1,500 RPM falls well short of a reduction to a controllable 400-600 RPM. Regulating the movement of the bolt by 'lock-up' or inertia activated mechanisms have involved utilizing the force generated by a discharging cartridge and have the disadvantage of disabling automatic firing of the gun if the mechanism fails, and also have in practice not removed the rate sufficiently to eliminate climb.

The mechanism of the present invention is incorporated in a sub-machine gun to give a weapon of handgun size which has a controllable rate of fire in the range of between 400 to 600 RPM. Furthermore, the rate of fire is not predetermined by the inherent physical prop-

erties of e.g. bolt weight or bolt travel distance. The rate of fire is controlled by means incorporated within the gun but independent of the force generated by a discharging cartridge and is determined by test to suit each weapon in which a mechanism according to the invention is incorporated. Therefore, if a specific model of sub-machine gun firing in fully automatic firing mode returns to its point of aim after each successive round is fired, say in one-tenth of a second, then that particular gun will be adjusted to fire ten times per second or 600 RPM. If another gun has, say, a longer barrel or lighter frame, or has any other feature which changes the firing characteristics of that gun, then that weapon's rate of fire will be adjusted to a rate determined as best suited for achieving its maximum accuracy of fire.

BE 902696 discloses a firing rate control system according to the preamble of claim 1.

US-A- 2 448 024 discloses an externally disposed trigger mechanism for a big antiaircraft machine gun. The trigger mechanism includes a rotating cam wheel to control the rate of fire.

The present invention relates to a trigger mechanism which comprises a timing device wherein the disadvantages of the prior art are overcome or at least mitigated. The invention accordingly provides a trigger mechanism for a gun capable of fully automatic firing, comprising a timing device operable to determine the rate of fire of the gun, characterised in that the trigger mechanism is in a sub-machine gun and that the device, which is disposed in the butt of the gun, cyclically interrupts means for discharging the ammunition to be discharged from the gun when a physically displaceable trigger of the mechanism is moved to a firing position and that the timing device comprises a rotatable cam which in fully automatic firing mode rotates to enable release of the discharge means once per cyclic revolution of the cam.

According to a preferred feature of this aspect of the invention, said ammunition discharging means may comprise a reciprocal bolt, said bolt being periodically arrested to reduce its natural cyclic rate of movement during automatic firing. Preferably, the device is an electrically powered timing device. Preferably, the timing device serves to actuate main sear lever for controlling movement of the bolt.

According to a preferred feature of the invention, the timing device may be actuated by the trigger of the trigger mechanism of the gun. In constructions where the timing device is trigger actuated, the trigger mechanism may comprise a trigger operatively connected to a pair of sears each of which sears has a sear face for cooperation with a bent, said bent being carried by means for actuating a main sear lever for controlling movement of the bolt. Preferably, said pair of sears both cooperate with said bent when the gun is set for semi-automatic firing, said bent being released from the sear face of one sear when the trigger is retracted to release the bolt from a cocked disposition and being arrested by engagement

with the sear face of the other sear after discharge of the gun but whilst the trigger remains retracted thereby to arrest movement of the bolt, said bent thereafter being transferred into engagement with the sear face of said one sear when the trigger is released to put the gun once again into a cocked condition.

One sear may actuate the timing device and maintain the device operative during automatic firing of the gun, and the device may regulate movement of the said other sear to cause a reduction in the natural cyclic rate of movement of the bolt. Preferably, the rotatable cam is arranged for cyclic engagement with a cam follower carried by said other sear to regulate movement of that sear.

The bolt and the main sear lever may have cooperating cam surfaces which are engaged when the bolt is retracted, the main sear lever being pivotally mounted to release and arrest said bolt and being controlled by a lever which carries said bent.

Preferably, an activator switch is brought into cooperation with said one sear to actuate the timing device during fully automatic firing of the gun and is taken out of cooperation with said one sear to deactivate said device during semi-automatic firing of the gun.

Another aspect of the invention provides a gun incorporating a trigger mechanism of the invention.

It is important to emphasise that a gun incorporating a trigger mechanism according to the present invention does not mean that the gun is an electrically fired weapon. The firing rate of the gun is controllable by electrical or electronic means in preferable embodiments but should such means fail to function then the gun is nevertheless fully operable in semi-automatic or fully automatic firing mode but, in such a case, at the weapons natural cyclic rate of fire.

In another aspect, the invention provides a trigger mechanism adapted for a sub-machine gun capable of fully automatic firing, comprising a bolt for discharging ammunition when a physically displaceable trigger of the mechanism is moved to a firing position and a timing device operable to determine the rate of fire of the gun, wherein the timing device cyclically interrupts operation of the bolt and comprises a rotatable cam which, when the mechanism is in fully automatic firing mode, rotates to enable release of the bolt once per revolution of the cam, in that the trigger is operatively coupled to a pair of sears each of which sears has a sear face for cooperation with a bent, the bent being carried by means for actuating a main sear lever for controlling movement of the bolt, and in that the rotatable cam is arranged for cyclic engagement with a cam follower carried by one of the sears to regulate movement of said one sear.

An embodiment of the invention will now be described, by way of example, with reference to the accompanying drawings, in which:

FIGURE 1 is a schematic side view partly shown in section, of a sub-machine gun incorporating a trig-

ger mechanism and timing device according to the invention;

Figures 2 to 7 are enlarged schematic and operationally sequential views of the trigger mechanism and timing device; and

Figure 8 is a plan view of a cam wheel of the timing device and upper sear lever.

Referring first to Figure 1 of the drawings, a sub-machine gun 10 has a breech 12 in which a bolt 14 (Fig 2 to 7) is reciprocable and slideable on a wear plate surface of the breech which, in Figures 2 to 7, is shown represented as a horizontal line 13.

Movement of the bolt indirectly is controlled by a main bolt sear lever 16 pivotally mounted at a leading end thereof to the frame 'F' of the gun at pivot 20 disposed below the wear plate surface 13. At its opposite trailing end lever 16 is pivotally mounted at pivot 22 to a further lever 24 which carries a bent 25 for co-operation with a pair of let-off sears described in more detail below. Lever 24 is pivotally mounted to frame 'F' intermediate its upper and lower ends at pivot 26. A trigger 28 is pivoted to the gun frame at pivot 30, and is operatively connected to a lower sear 32 of the pair of let-off sears by connecting link 34. Link 34 has one of its ends pivotally connected to trigger 28 at pivot 36 and has its opposite end hinged to the lower sear 32 at pivot 38. An upper sear 40 which co-operates with lower sear 32 to control movement of the bent 25 is pivotally mounted together with the lower sear to the frame at pivot 42. The upper sear carries a cam follower 44 which co-operates with an actuating cam of a timing unit 46 disposed in the butt of the gun.

In Figures 2 to 7 of the drawings, the large arrows indicate in each case the direction in which a component of the trigger mechanism or firing rate control device is urged to move either by spring force or, in the case of the latter, by electrical propulsive force or other actuating means.

Referring to Figure 2, the mechanism is shown in a condition in which the gun is cocked and ready to fire. In this condition, the bolt 14 is held retracted by the main sear lever 16. The main sear lever 16 cannot however, on its own, retain the bolt 14 in a retracted position because the sear face 17 and bent 15 of the bolt each provide cooperating 45 degree cam surfaces and because the main spring force indicated by the arrow on bolt 14 is greater than the main sear return spring force, shown by the arrow on lever 16, bolt 14 is urged forwards so that bent 15 is forced into abutment with the sear face 17 of sear lever 16. However, because the lever 16 is directly linked to lever 24, if bent 25 of lever 24 is locked against either the sear face 33 of sear 32 or sear face 41 of sear 40, the main sear lever 16 cannot move, and therefore retains the bolt 14 in its retracted position. In Figure 2, bent 25 is shown held by the lower sear 40 at

sear face 41.

As previously stated the trigger 28 is directly linked to the lower sear 40 by link arm 34. As can be seen in Figure 2, if the trigger is pulled back (retracted to the right in Figure 2) then the lower sear 40 will be caused to rotate clockwise (downwards) and thereby disengage sear face 41 from bent 25 thereby allowing lever 24 to pivot. Consequently, the bolt is then free to override the main bolt sear lever 16 and move forwards (left in Figure 2) to discharge the gun.

The upper let-off sear 32 has limited movement in a clockwise direction. Upper sear 32 can move only until face 33a of sear 32 meets face 41a of sear 41. Indeed, sear 32 and sear 40 are biased against each other at faces 33a and 41a by spring force between those sears in the manner of a common sprung clothes peg as indicated by the juxtaposed arrows. Pivot pin 38 protrudes axially and acts upon one of three stepped faces S, F and A respectively of a firing mode selector switch 39 shown in Figures 2-7 superimposed upon the lever 24 and sear 32. When the firing mode selector switch is in its lowermost position, face 'S' is aligned with the path of movement of pivot pin 38 (Figure 2) which prevents pivotal movement of lower sear 40. However, when firing, mode selector switch 39 is put into its intermediated position, face 'F' is aligned with the path of movement of pivot pin 38 so that limited pivoting movement of lower sear 40 can take place. Thus, as the lower sear 40 is rotated by retraction of the trigger 28 via link 34, the upper sear 32 will also have a tendency to rotate in the same direction as the lower sear 40. Sear face 33a can move away from sear face 41a a distance sufficient to follow bent 25 to pass between the faces 33a and 41a.

It will be seen in Figure 2 that when the bent 25 is engaged with the lower let-off sear face 41, bent 25 is beneath sear face 33 of the upper sear 32.

A firing mode selector switch 39 shown in Figure 3 is set to a semi-automatic firing mode, so that the switch 39 in this firing mode will allow movement of sear 40 until pivot pin 38 abuts face 'F' of firing mode selector switch 39 but will not allow sufficient movement of sear 40 for it to engage timing activator switch 54. Face 'F' stops sear 40 just before it engages switch 54 but just after sear 40 disengages from bent 25.

In Figure 3, the trigger is shown as having been pulled, thereby causing the sear face 41 of let-off sear 40 to disengage from bent 25 of the lever 24. Figure 3 shows events at the moment of disengagement. It can also be seen that the upper sear 32 does not move downwards, because it is held back in position by bent 25.

Figure 4 shows the relative positions of the components of the trigger mechanism immediately after the positions shown in Figure 3. Lever 24 is no longer retained at bent 25 by let-off sear 32 and consequently bolt 14 is able to over-ride the main reduction sear 16, thereby pushing the main sear lever 16 downward and causing anti-clockwise rotation of lever 24 away from engage-

ment with either let-off sears 32 or 40.

The main sear lever 16 must, of course, move below the surface 13 of the wear plate to release the bolt 14.

Referring now to Figure 5, the trigger is shown still in its pulled (retracted) position and the relative positions of the trigger components are shown in positions immediately after those shown in Figure 4.

The bolt 14 has now discharged a cartridge and has been returned in reaction to discharge of the cartridge along the breech allowing the main sear lever 16 to snap upwards under spring bias into engagement with the bolt at faces 15 and 17 to arrest forward travel of the bolt. As the main sear 16 snaps upwards, it simultaneously causes the lever 24 to rotate clockwise and causes the bent 25 of lever 24 to slide under and beyond the sear face 33 of the upper sear 32. Sear 32 becomes free to return to its neutral position against the lower sear 40 in which faces 33a and 41a are in abutment. In so doing, the upper sear 32 engages the bent 25 at sear face 33 and arrests return movement of lever 24 before the bolt 14 can over-ride cam face engagement at faces 15 and 17. The bolt is thereby locked in a retracted position.

In order for the gun to be fired again in semi-automatic firing mode, the trigger must be released to reset the mechanism. Thus, referring again to Figure 2, as the trigger 28 is released, it causes the lower sear 40 to rotate anti-clockwise (upwards). This action causes the upper sear 32 also to move upwardly because of the engagement between the sears at faces 32a and 41a. As the upper sear 32 disengages from bent 25 at face 33, face 41 of the lower sear 40 moves into position so that bent 25 is transferred into engagement with face 41 of the lower sear. The weapon is then fully cocked and the trigger is again ready to be pulled.

It will be appreciated that if the upper sear 32 were not present to engage bent 25 after the sear 40 had disengaged, the bolt would be free to continue moving back and forth after an initial round of ammunition had been discharged from the gun. In full unregulated automatic firing mode the weapon would continue to fire until the supply of ammunition was exhausted or until the trigger was released thereby re-engaging the lower sear 40 with the bent 25. In order to fire the gun in regulated fully automatic firing mode, the selector switch 39 is brought into engagement so that the trigger mechanism activates timing mechanism 46 to control movement of the bolt.

Referring now to Figure 6, the selector switch 39 is shown in its uppermost position in which face 'A' is aligned with the path of movement of pivot pin 38. When selector switch 39 is in this position lower sear 40 can pivot sufficiently clockwise as to engage and depress timing activator switch 54 when the lower sear 40 is activated via the trigger 28, thus activating the timing unit 46. The timing unit comprises a motor (not shown) which, when active, causes a cam wheel 56 to be rotated in a clockwise direction as seen from above (see Figure 8).

Other than the intervention of the selector switch 39 the various components of the mechanism are in the relative positions as shown in Figure 5 for semi-automatic firing. Now, however, after the trigger is retracted and is maintained in pulled back position, and after an initial round of ammunition has been discharged, because the lower sear 40 has been rotated out of engagement with the bent 25, and the upper sear 32 has re-engaged the bent 25, the lower sear 40 serves only to hold switch 54 in a depressed position.

In Figure 8, the cam wheel 56 is shown in plan view. Once per revolution of the cam wheel, a cam lug 58 carried by the cam wheel will engage cam face 44 of the upper sear 32 and cause the upper sear to rotate anti-clockwise (upwardly) thereby disengaging from bent 25 at sear face 33. Each time the cam 58 causes the upper sear 32 to disengage from bent 25, the gun will be fired and thereafter the upper sear 32 will re-engage bent 25. This will happen only in a cycle in which the period of the cam is less than the natural firing period of the weapon itself. As a safety feature, when the gun is cocked, the cam lug cannot engage the cam face of the upper sear until the trigger is fully retracted. Therefore the movement of the components of the mechanism can be arranged so that the gun can be fired in semi-automatic mode with a less than full trigger pull even with the automatic firing selector switch engaged.

The upper sear 32 must have time to re-engage bent 10. The firing rate can be decreased infinitely by different gear trains or voltage changes etc, but the firing rate cannot be increased beyond the gun's natural unregulated firing rate.

Figure 6 shows the mechanism in a cocked condition, and Figure 7 shows the cam 56 positioned to dislodge the upper sear 32 out of engagement with bent 25 of lever 24. This cycle of timed automatic firing will continue until the supply of ammunition is exhausted or until the trigger is released. Upon release, the trigger allows the lower sear 40 to engage bent 25 of lever 24 and allows lower sear 40 to disengage from switch 54, thereby de-activating the timing motor and cam wheel 56. The gun is then ready to be reloaded, if empty, or fired again.

The trigger mechanism is constructed and arranged to reduce the let-off force at bent 25. If face 15 and 17 were perpendicular to the direction of bolt movement as is normal in known weapons, a main spring force of e.g. eighteen pounds would generate a sear let-off force of around six pounds. However, because the bent and sear faces of the bolt and lever 16 respectively must at a 45 degree angle to the perpendicular an e.g. eighteen pound mainspring force exerts only a nine pound downward force on the main sear lever 16 and therefore that force is transmitted to lever 24 at connecting pivot 22. The bent 25 of lever 24 is twice the distance away from pivot 26 as is pivot 22 so that the force transmitted to bent 25 of lever 24 is one half of that exerted on pivot 22 of lever 24. Therefore, a nine pound force is reduced

to a four and a half pound force at bent 25 against sear face 33 or sear face 41. The co-efficient of friction of a four and a half pound force (depending upon the finish of the mating surfaces) of bent 25 and sear faces 33 and 41 can be as low as 0.5. The upper sear 32 is the active let-off sear when the gun is set in regulated fully-automatic firing mode, so notwithstanding the return spring force of sear 32, the actual let-off force the timing unit must overcome with may be one pound or less.

In both semi-automatic and fully-automatic firing modes, the lower sear 40 holds the bolt in a cocked condition. Since the lower sear 40 is manually controlled via trigger 28, the lower sear can have a much heavier return spring.

It is envisaged that should one require an extremely light trigger pull for semi-automatic firing then, for extra accuracy, a system could be employed which temporarily disengages the heavy trigger return spring (for one shot), and then re-engages after the trigger is released.

The forces holding let-off sears 32 and 40 can be further reduced by substituting the return spring acting on the main bolt sear 16 and the lever 24 with one which exerts a greater force. The above example in which an eighteen pound main spring force transmits a nine pound downward force against pivot pin 22 occurs only if there is no spring force acting on sear 16 to urge it upwardly. If the return spring acting upwardly on sear 16 has a force of e.g., four pounds, then the actual force transmitted downwardly onto sear 16 and pivot pin 22 is five pounds so that only a two and a half pound force is exerted at bent 25. A main sear return spring force of about nine pounds would negate a nine pound downward force transmitted by the bolt main spring, and the bolt would not then move when the trigger was pulled.

Lever 24 has a portion above pivot pin 26 whose mass is equal to the mass of the lever below the pivot pin 26. Lever 24 is a elongate component positioned perpendicular to the line of motion during recoil. The presence of the portion above the pivot pin 26 cancels out any oscillation in lever 24 which could adversely affect the lock-up of bent 25 with let-off sear 32 or 40.

The above embodiment employs a cam wheel to trip the upper sear 32. However, any device which delivers the necessary force and can reliably and repeatedly trip the upper sear 32 with the desired and consistent period, could be utilized. The motion of such a device could be reciprocal as opposed to rotary, such as that produced by a solenoid.

The power source for activating the trip device could be other than electrical, such as mechanical, clockwork or gas pressure. However the present embodiment has the advantage that the timing unit is a self-contained unit, sealed against the elements. It has a life expectancy of up to and in excess of the life of the gun (i.e. 50,000 rounds) depending upon battery size. The timing unit may incorporate warning lights to indicate battery condition.

The mechanism of the present invention ideally is

adapted for incorporation in an open-bolt 9mm sub-machine gun although a closed bolt may be utilized simply by the replacement of one bolt type for the other. This feature allows the gun to be used with a closed bolt mainly in semi-automatic firing mode when accuracy is of primary importance but where the option of a fully automatic firing mode is available, perhaps in an emergency, even with a closed bolt.

Should the timing unit fail then the gun normally can be fired as a semi-automatic weapon. Alternatively it is envisaged that the gun may be furnished with a selector device (not shown) which engages the upper sear 32 and holds it up out of the path of travel of bent 25. This facility allows the gun to be fired in unregulated fully automatic firing mode.

It is also envisaged that means such as an electric spark may be utilized to discharge the ammunition rather than a mechanical bolt in which case the timing unit is adapted to control the emission of the detonating spark.

Claims

1. A trigger mechanism for a gun (10) capable of fully automatic firing, comprising a timing device operable to determine the rate of fire of the gun, characterised in that the trigger mechanism is in a sub-machine gun and that the device, which is disposed in the butt of the gun, cyclically interrupts means (14) for discharging the ammunition to be discharged from the gun when a physically displaceable trigger (28) of the mechanism is moved to a firing position and that the timing device comprises a rotatable cam (56) which in fully automatic firing mode rotates to enable release of the discharge means (14) once per cyclic revolution of the cam.
2. A trigger mechanism as claimed in claim 1 wherein the ammunition discharge rate is substantially constant.
3. A trigger mechanism as claimed in claims 1 or 2 wherein the ammunition discharge rate is determined by the timing device and is independent of any other weapon characteristic.
4. A trigger mechanism as claimed in any of the preceding claims wherein the timing device is electrically powered.
5. A trigger mechanism as claimed in any of the preceding claims wherein the means (14) for discharging ammunition comprises a bolt and the trigger (28) is operatively connected to a pair of sears (32,40) each of which sears has a sear face (33,41) for cooperation with a bent (25), the bent (25) being carried by means (24) for actuating a main sear lever (16) for controlling movement of the bolt (14).
6. A trigger mechanism according to claim 5 wherein said pair of sears (32,40) are both arranged to cooperate with said bent (25) when the gun is set for semi-automatic firing, said bent (25) being released from the sear face (41) of one sear (40) when the trigger (28) is retracted to activate the trigger mechanism and being arrested by engagement with the sear face (33) of the other sear (32) after discharge of the gun but whilst the trigger (28) is in said position to activate the trigger mechanism thereby to arrest movement of the bolt (14), said bent (25) thereafter being transferred into engagement with the sear face (41) of said one sear (40) when the trigger (28) is released to again put the gun in a cocked condition.
7. A trigger mechanism according to claim 6 wherein said one sear (40) is arranged to actuate the timing device and maintain the timing device in operation during automatic firing of the gun, the timing device in use regulating movement of the said other sear (32) to cause a reduction in the natural cyclic rate of movement of the bolt (14).
8. A trigger mechanism according to claim 7 wherein the rotatable cam (56) is arranged for cyclic engagement with a cam follower (44) carried by said other sear (32) to regulate movement of that sear (32).
9. A trigger mechanism according to any one of claims 5 to 8 wherein the bolt (14) and the main sear lever (16) have cooperating cam surfaces (15,17) which are engaged when the bolt is retracted, the main sear lever (16) being pivotally mounted to release and arrest the bolt (14) and being controlled by a lever (24) which carries the bent (25).
10. A trigger mechanism according to claim 7, 8 or 9 wherein an activator switch (54) is arranged to be brought into cooperation with said one sear (40) to actuate the timing device during fully automatic firing of the gun and to be taken out of cooperation with said one sear (40) to deactivate the timing device during semi-automatic firing of the gun.
11. A trigger mechanism adapted for a sub-machine gun (10) capable of fully automatic firing, comprising a bolt (14) for discharging ammunition when a physically displaceable trigger (28) of the mechanism is moved to a firing position and a timing device operable to determine the rate of fire of the gun, wherein the timing device cyclically interrupts operation of the bolt (14) and comprises a rotatable cam (56) which, when the mechanism is in fully automatic firing mode, rotates to enable release of the bolt

(14) once per revolution of the cam, in that the trigger (28) is operatively coupled to a pair of sears (32,40) each of which sears has a sear face (33,41) for cooperation with a bent (25), the bent (25) being carried by means (24) for actuating a main sear lever (16) for controlling movement of the bolt (14), and in that the rotatable cam (56) is arranged for cyclic engagement with a cam follower (44) carried by one (32) of the sears (32,40) to regulate movement of said one sear (32).

12. A sub-machine gun (10) incorporating a trigger mechanism according to any of the preceding claims.

13. The use of a trigger mechanism as claimed in any one of claims 1 to 11 for controlling the climb of a sub-machine gun.

Patentansprüche

1. Ein Triggermechanismus für eine zur vollautomatischen Feuerung fähigen Feuerwaffe (10), umfassend eine Zeitsteuereinrichtung zum Bestimmen der Feuergeschwindigkeit der Feuerwaffe, dadurch gekennzeichnet, daß der Triggermechanismus sich in einer Maschinenpistole befindet und daß die Vorrichtung, welche am Kolben der Feuerwaffe angeordnet ist, zyklisch Einrichtungen (14) zum Auswerfen der aus der Feuerwaffe auszuwerfenden Munition unterbricht, wenn ein physikalisch versetzbarer Trigger (28) des Mechanismus in eine Feuerposition versetzt wird, und daß die Zeitsteuereinrichtung eine drehbare Nocke (56) umfaßt, welche im vollautomatischen Feuermodus rotiert, um die Auswerfeinrichtung (14) einmal pro Umdrehungszyklus der Nocke freizugeben.

2. Triggermechanismus nach Anspruch 1, worin die Munitionsauswerfgeschwindigkeit im wesentlichen konstant ist.

3. Triggermechanismus nach Anspruch 1 oder 2, worin die Munitionsauswerfgeschwindigkeit durch die Zeitsteuereinrichtung bestimmt und unabhängig von irgendwelchen anderen Waffeneigenschaften ist.

4. Triggermechanismus nach jedem der vorhergehenden Ansprüche, worin die Zeitsteuereinrichtung elektrisch gesteuert wird.

5. Triggermechanismus nach irgendeinem der vorhergehenden Ansprüche, worin das Mittel (14) zum Austragen von Munition einen Bolzen umfaßt und der Trigger (28) operativ mit einem Paar von Abzugsstollen (32, 40) verbunden ist, wobei jeder der

Abzugsstollen eine Abzugsstollenoberfläche (33, 41) zum Zusammenwirken mit einem Bogen (25) aufweist, und der Bogen (25) durch ein Mittel (24) zum Betätigen eines Hauptabzugsstollenhebels (16) zur Regelung der Bewegung des Bolzens (14) trägt.

6. Triggermechanismus nach Anspruch 5, worin das Paar von Abzugsstollen (32, 40) beide zum Zusammenwirken mit dem Bogen (25) angeordnet sind, wenn die Feuerwaffe auf halbautomatische Feuerung eingestellt ist, wobei der Bogen (25) von der Abzugsstollenoberfläche (41) eines Abzugsstollens (40) freigegeben wird, wenn der Trigger (28) zurückgezogen wird, um den Triggermechanismus zu aktivieren und durch in Eingriffnahme mit der Abzugsstollenoberfläche (33) des anderen Abzugsstollens (32) nach dem Auswerfen der Feuerwaffe arretiert wird, aber wenn der Trigger (28) in der Stellung zur Aktivierung des Triggermechanismus ist, um dabei die Bewegung des Bolzens (14) zu arretieren, der Bogen (25) danach zur Ineingriffnahme mit der Abzugsstollenoberfläche (41) des einen Abzugsstollens (40) gebracht wird, wenn der Trigger (28) freigegeben wird, um erneut die Feuerwaffe in einen gespannten Zustand zu bringen.

7. Triggermechanismus nach Anspruch 6, worin der eine Abzugsstollen (40) zur Betätigung der Zeitsteuereinrichtung und Aufrechterhalten der Zeitsteuereinrichtung im Betrieb während einer automatischen Feuerung der Feuerwaffe angeordnet ist, wobei die Zeitsteuereinrichtung die Bewegung des anderen Abzugsstollens (32) reguliert, um eine Verminderung der natürlichen Umdrehungszahl des Bolzens (14) zu verursachen.

8. Triggermechanismus nach Anspruch 7, worin die Drehnocke (56) zur zyklischen Ineingriffnahme mit einem Nockenstößel (44), der durch den anderen Abzugsstollen (32) getragen ist, angeordnet ist, um die Bewegung des Abzugsstollens (32) zu steuern.

9. Triggermechanismus nach jedem der Ansprüche 5 bis 8, worin der Bolzen (14) und der Hauptabzugsstollenhebel (16) zusammenwirkende Nockenoberflächen (15, 17) aufweisen, welche ineinander eingreifen, wenn der Bolzen zurückgezogen wird, wobei der Hauptabzugsstollenhebel (16) schwenkbar angeordnet ist, um den Bolzen (14) freizugeben und festzustellen und durch einen Hebel (24), der den Bogen (25) trägt, gesteuert wird.

10. Triggermechanismus nach Anspruch 7, 8 oder 9, worin ein Aktivierungsschalter (54) zum Zusammenwirken mit einem der Abzugsstollen (40) angeordnet ist, um die Zeitsteuereinrichtung während einer vollautomatischen Feuerung der Feuerwaffe zu

betätigen und außer Eingriff mit dem einen Abzugsstollen (40) gebracht wird, um die Zeitsteuereinrichtung während einer halbautomatischen Feuerung der Feuerwaffe zu inaktivieren.

11. Triggermechanismus, angepaßt für eine Maschinenpistole (10), die zur vollautomatischen Feuerung fähig ist, umfassend einen Bolzen (14), zum Auswerfen von Munition, wenn ein physikalisch verschiebbarer Trigger (28) des Mechanismus in eine Feuerstellung bewegt wird, und eine Zeitsteuereinrichtung, die Feuergeschwindigkeit der Feuerwaffe bestimmen kann, worin die Zeitsteuereinrichtung zyklisch die Arbeit des Bolzens (14) unterbricht und eine Drehnocke (56) umfaßt, welche, wenn sich der Mechanismus im vollautomatischen Feuermodus befindet, sich dreht, um die Freigabe des Bolzens (14) einmal pro Umdrehung der Nocke zu ermöglichen, der Trigger (28) mit einem Paar von Abzugsstollen (32, 40) wirksam verbunden ist, wovon jeder Abzugsstollen eine Abzugsstollenoberfläche (33, 41) zum Zusammenwirken mit einem Bogen (25) aufweist, wobei der Bogen (25) durch ein Mittel (24) zum Betätigen eines Hauptabzugsstollenhebels (16) zur Steuerung der Bewegung des Bolzens (14) trägt und worin die Drehnocke (56) zur zyklischen Ineingriffnahme mit einem Nockenstößel (44) angeordnet ist, der durch einen (32) der Abzugsstollen (32, 40) getragen ist, um die Bewegung des einen Abzugsstollens (32) zu steuern.
12. Eine Maschinenpistole (10), welche einen Triggermechanismus nach irgendeinem der vorhergehenden Ansprüche beinhaltet.
13. Verwendung des Triggermechanismus nach irgendeinem der Ansprüche 1 bis 11, zur Steuerung des Fortschreitens des Maschinengewehres.

Revendications

1. Mécanisme déclencheur pour un pistolet (10) capable d'un tir complètement automatique, comprenant un dispositif de minutage apte à être actionné pour déterminer la vitesse de tir du pistolet, caractérisé en ce que le mécanisme déclencheur est un pistolet-mitrailleur et en ce que le dispositif, qui est disposé dans la crosse du pistolet, interrompt cycliquement un moyen (14) de décharge des munitions devant être déchargées par le pistolet, lorsqu'un déclencheur (28) déplaçable physiquement du mécanisme est déplacé vers une position de tir, et en ce que le dispositif de minutage comprend une came apte à tourner (56) qui, dans un mode de tir complètement automatique, tourne afin de permettre la libération du moyen de décharge (14), une fois par rotation cyclique de la came.

2. Mécanisme déclencheur selon la revendication 1, dans lequel la vitesse de décharge des munitions est sensiblement constante.

3. Mécanisme déclencheur selon la revendication 1 ou 2, dans lequel la vitesse de décharge des munitions est déterminée par le dispositif de minutage et est indépendante de toute autre caractéristique de l'arme.

4. Mécanisme déclencheur selon l'une quelconque des revendications précédentes, dans lequel le dispositif de minutage est alimenté électriquement.

5. Mécanisme déclencheur selon l'une quelconque des revendications précédentes, dans lequel le moyen (14) de décharge des munitions comprend un boulon et le déclencheur (28) est relié de façon opérationnelle à un couple de gâchettes (32; 40), chacune des gâchettes ayant une face de gâchette (33; 41) pour la coopération avec un coude (25), le coude (25) étant supporté par un moyen (24) pour actionner un levier de gâchette (16) principal pour commander le mouvement du boulon (14).

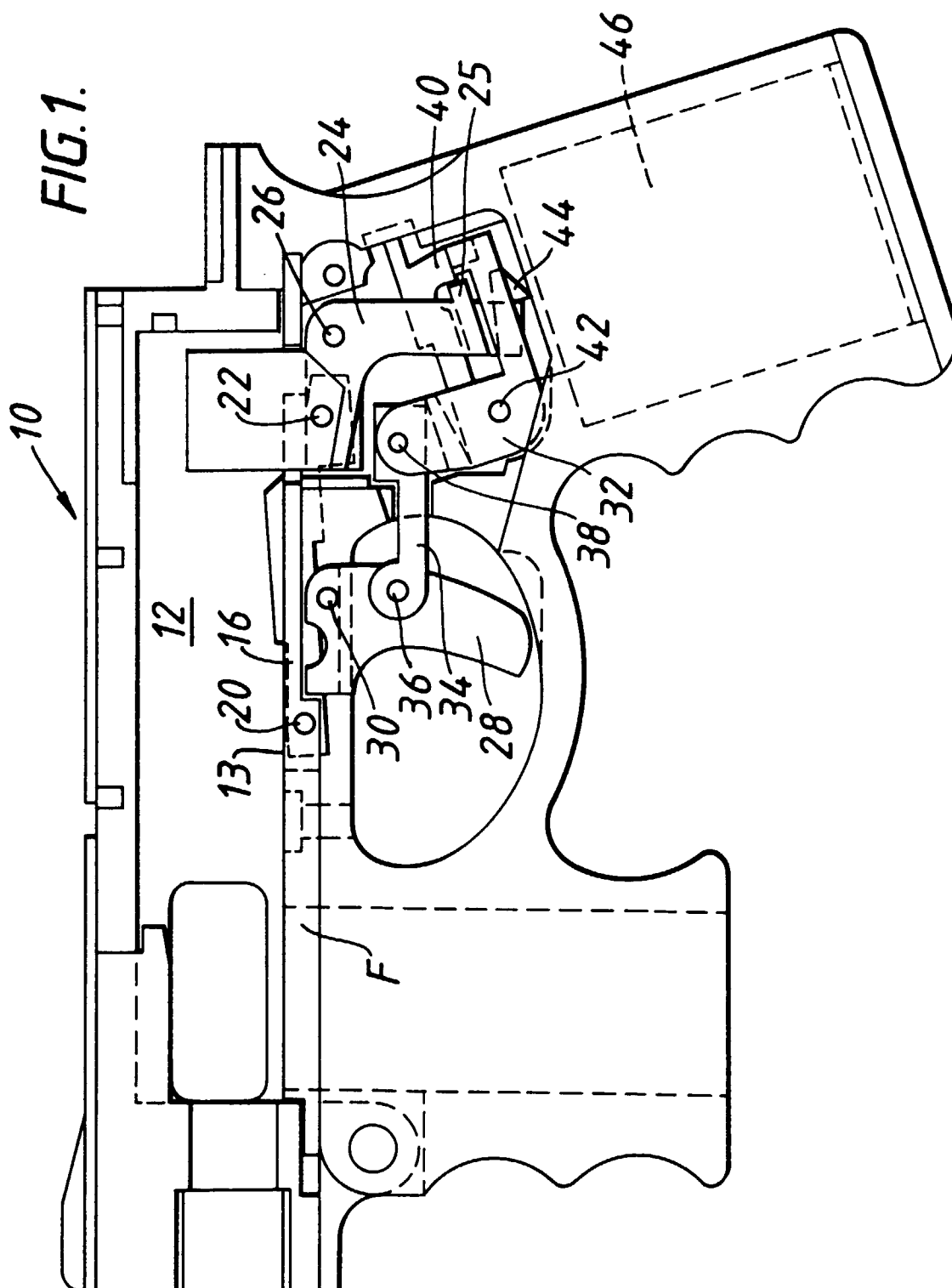
6. Mécanisme déclencheur selon la revendication 5, dans lequel les deux gâchettes (32; 40) sont toutes deux agencées de façon à coopérer avec ledit coude (25) lorsque le pistolet est réglé en vue d'un tir semi-automatique, ledit coude (25) étant libéré de la face de gâchette (41) d'une gâchette (40), lorsque le déclencheur (28) est rétracté, afin d'actionner le mécanisme déclencheur et étant arrêté par un contact avec la face de gâchette (33) de l'autre gâchette (32), après une décharge du pistolet, mais tandis que le déclencheur (28) se trouve dans ladite position d'actionnement du mécanisme déclencheur, arrêtant ainsi le mouvement du boulon (14), ledit coude (25) est ensuite transféré afin de venir au contact de la face de gâchette (41) de ladite gâchette (40), lorsque le déclencheur (28) est libéré afin de placer de nouveau le pistolet dans un état armé.

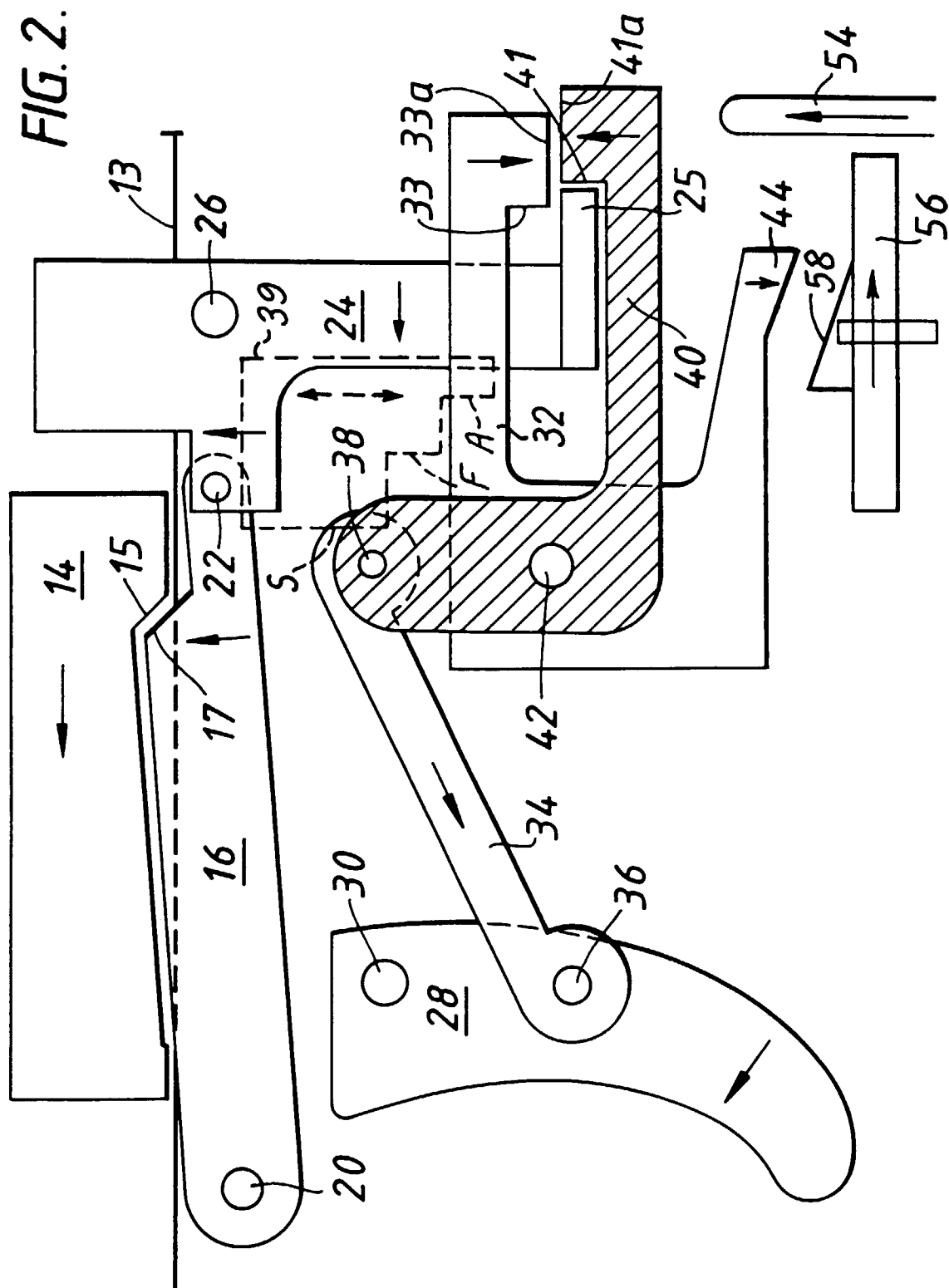
7. Mécanisme déclencheur selon la revendication 6, dans lequel ladite gâchette (40) est agencée de façon à actionner le dispositif de minutage et à maintenir le dispositif de minutage en fonctionnement durant un tir automatique du pistolet, le dispositif de minutage régulant en service le mouvement de ladite autre gâchette (32) afin de provoquer une réduction de la vitesse cyclique naturelle de déplacement du boulon (14).

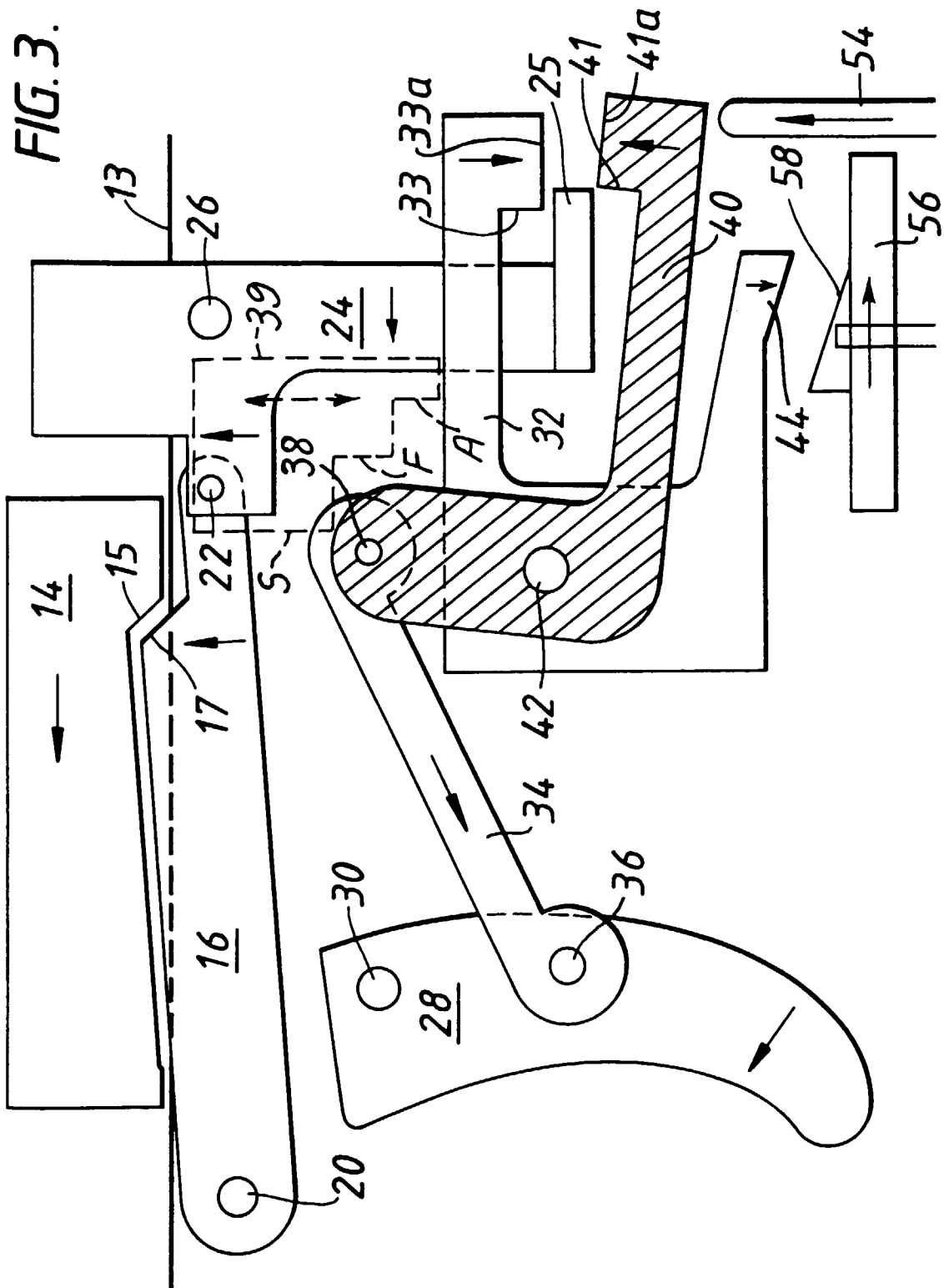
8. Mécanisme déclencheur selon la revendication 7, dans lequel la came apte à tourner (56) est agencée de façon à établir un contact cyclique avec un suiveur de came (44) supporté par ladite autre gâchette.

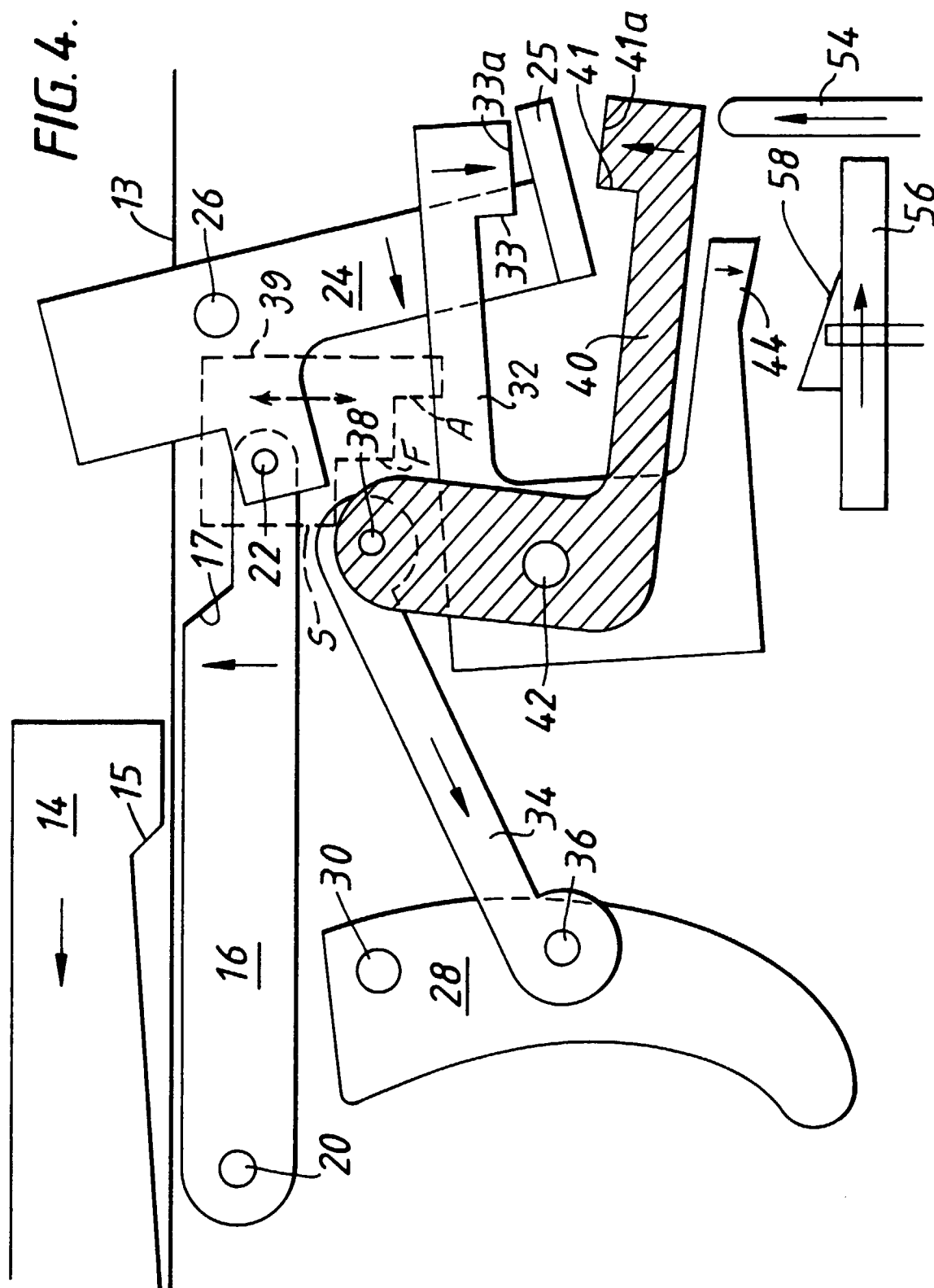
te (32), en vue de réguler le mouvement de cette gâchette (32).

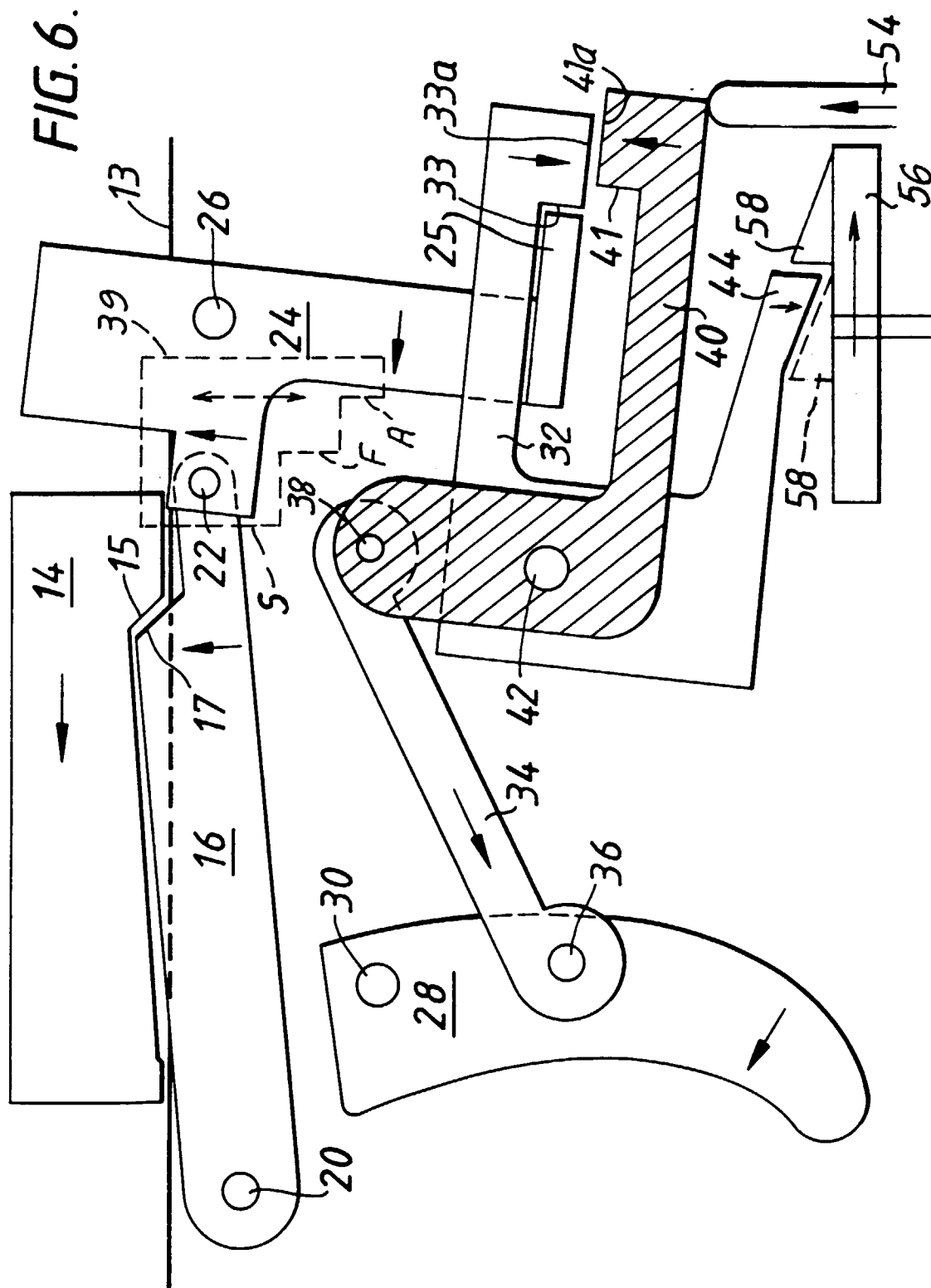
9. Mécanisme déclencheur selon l'une quelconque des revendications 5 à 8, dans lequel le boulon (14) et le levier de gâchette (16) principal présentent des surfaces de came (15; 16) coopérantes qui sont en prise lorsque le boulon est rétracté, le levier de gâchette (16) principal étant monté pivotant afin de libérer et d'arrêter le boulon (14) et étant commandé par un levier (24) qui supporte le coude (25). 5 10
10. Mécanisme déclencheur selon la revendication 7, 8 ou 9, dans lequel un commutateur d'actionnement (54) est agencé de façon à être amené en coopération avec ladite gâchette (40), en vue d'actionner le dispositif de minutage durant un tir complètement automatique du pistolet, et à sortir de la coopération avec ladite gâchette (40) afin de désactiver le dispositif de minutage durant un tir semi-automatique du pistolet. 15 20
11. Mécanisme déclencheur adapté à un pistolet-mitrailleur (10) capable d'un tir complètement automatique, comprenant un boulon (14) pour décharger des minutions lorsqu'un déclencheur (28) déplaçable physiquement du mécanisme est déplacé vers une position de tir et un dispositif de minutage pouvant déterminer la vitesse de tir du pistolet, dans lequel le dispositif de minutage interrompt cycliquement le fonctionnement du boulon (14) et comprend une came apte à tourner (56) qui, lorsque le mécanisme se trouve dans un mode de tir complètement automatique, tourne afin de permettre la libération du boulon (14) une fois par rotation de la came, dans lequel le déclencheur (28) est couplé de façon opérationnelle à un couple de gâchettes (32; 40), chacune des gâchettes présentant une face de gâchette (33; 41) pour la coopération avec un coude (25), le coude (25) étant supporté par un moyen (24) pour actionner un levier de gâchette (16) principal afin de commander le mouvement du boulon (14), et dans lequel la came apte à tourner (56) est agencée de façon à établir un contact cyclique avec un suiveur de came (44) supporté par l'une (32) des gâchettes (32; 40) afin de réguler le mouvement de ladite gâchette (32). 25 30 35 40 45
12. Pistolet-mitrailleur (10) comprenant un mécanisme déclencheur selon l'une quelconque des revendications précédentes. 50
13. Utilisation d'un mécanisme déclencheur selon l'une quelconque des revendications 1 à 11, afin de commander la montée d'un pistolet-mitrailleur. 55











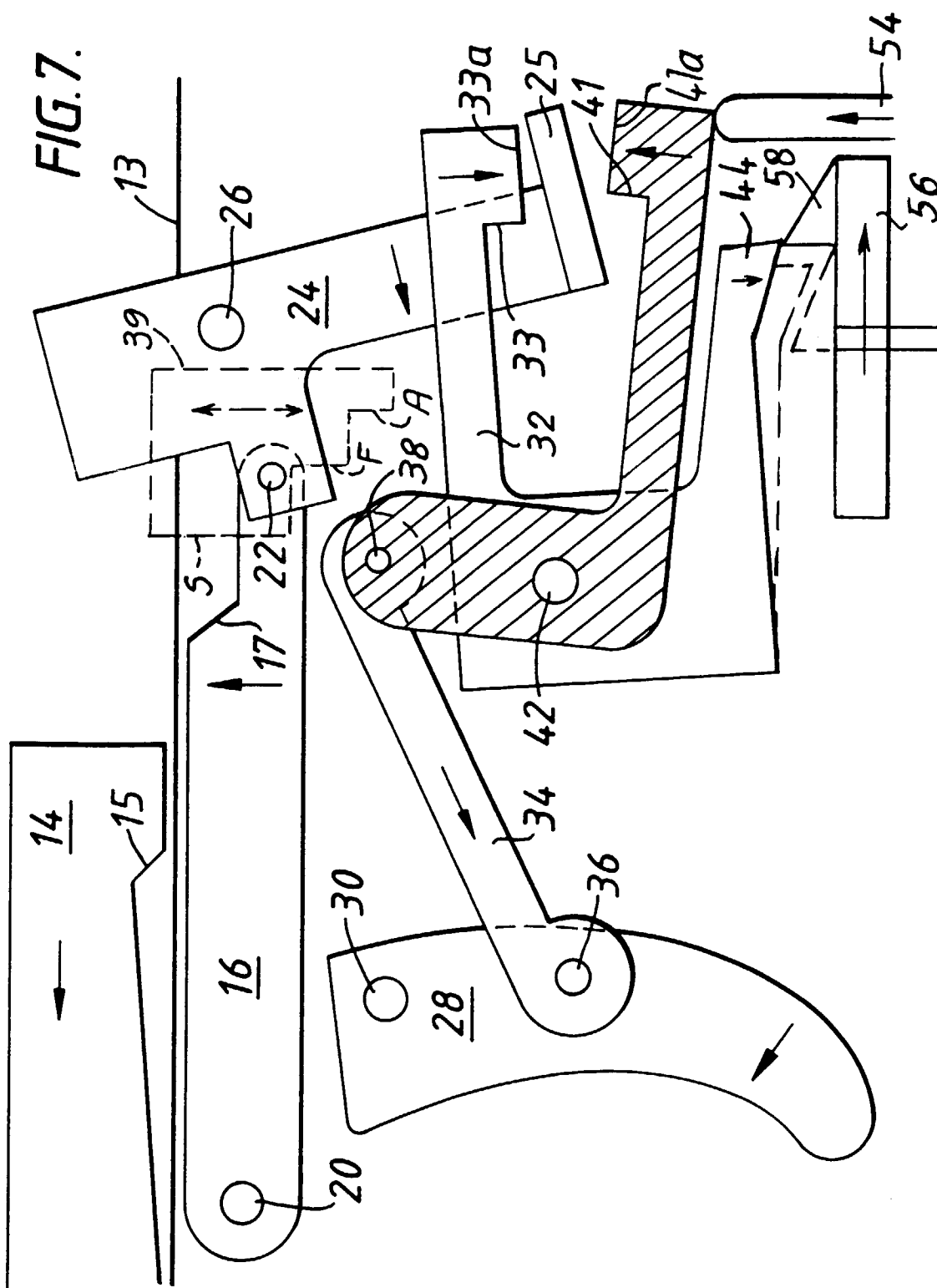


FIG. 8.

