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(54) **Upper tool and upper tool assembly comprising said upper tool for use with a press brake and method of clamping said upper tool**

Oberwerkzeug und Oberwerkzeugträger mit einem solchen Oberwerkzeug zur Verwendung mit einer Biegepresse und Verfahren zur Befestigung des Oberwerkzeuges

Outil supérieur et ensemble de support comportant un tel outil supérieur pour une presse plieuse et méthode de fixation de l'outil supérieur

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(73) Proprietor: **AMADA COMPANY LIMITED**
Kanagawa-ken (JP)

(72) Inventors:
• **Kawano, Susumu**
Atsugi-shi, Kanagawa (JP)
• **Sugimoto, Mamoru**
Ayase-shi, Kanagawa (JP)

• **Kawano, Toshiro**
Odawara-shi, Kanagawa (JP)

(74) Representative: **Hasenrader, Hubert et al**
Cabinet Beau de Loménie
158, rue de l'Université
75340 Paris Cédex 07 (FR)

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Description

BACKGROUND OF THE INVENTION

Field of the Invention

[0001] The present invention relates to an upper tool for a press brake, an upper tool holder assembly for a press brake, and a method of holding the upper tool by the upper tool holder assembly for a press brake.

[0002] An upper tool according to the pre-characterising part of claim 1 is known from EP-A-0 569 880.

Description of Related Art

[0003] A press brake is provided with an upper table (referred to as an upper apron, sometimes) and a lower table (referred to as a lower apron, sometimes) in mutually opposing positional relationship with respect to each other. Further, any one of the upper and lower tables is driven in the vertical direction as a ram.

[0004] Further, in the press brake, an upper tool (referred to as a punch) is attached to the upper table and further a lower tool (referred to as a die) is attached to the lower table in order to bend plate-shaped work to various desired shapes.

[0005] In the above-mentioned construction of the press brake, when the movable-side table is moved up and down to engage the upper tool with the die, the work positioned between the upper tool and the die can be bent.

[0006] In the press brake, in general, since the upper tools must be exchanged according to the bending shapes of work, a number of upper tool holders are arranged on the lower portion of the upper table. That is, a plurality of upper tools are supported by a plurality of upper tool holders so as to be exchanged with respect to each other.

[0007] In the conventional upper tool holder apparatus, an upper tool clamp member is attached to a holder body mounted on the lower portion of the upper table, and further fastened by use of fastening bolts, so that the upper portion of the upper tool can be strongly clamped between the holder body and the upper tool clamp member.

[0008] In the conventional upper tool holder, therefore, in order to exchange a plurality of upper tools attached to the upper table, a plurality of fastening bolts of the upper tool holder members must be rotated, thus causing a problem in that the upper tool exchange work is troublesome.

[0009] To overcome this problem, an upper tool holder apparatus such that the upper tools can be fastened or unfastened by use of an air cylinder mounted on the upper tool holder apparatus has been developed.

[0010] In the upper tool holder apparatus of this type, however, since an air cylinders must be provided for a plurality of upper tool holder apparatus, respectively and

since an air source is additionally necessary, there exists another problem in that the construction of the upper tool holder apparatus is complicated and thereby the cost thereof is relatively high.

5 **[0011]** Further, in the conventional upper tool holder apparatus, there exists such a danger that the upper tool drops when the upper tool is released from of an upper tool clamping force of the upper tool holder apparatus.

10 **[0012]** Further, in the conventional upper tool holder apparatus, the upper tool clamp member must be half fastened to such an extent that the upper tool does not drop for alignment with the die, and after that the upper tool clamp member must be full fastened tightly after the upper tool and die have been aligned with each other.

15 As a result, there exists another problem in that the tool setting work is troublesome.

[0013] As prior art examples related to the present invention, there are EP 0 387 121 A1 (the first prior art) and Japanese Published Unexamined (Kokai) Patent Application No. 6-23436 (the second prior art).

20 **[0014]** In the first prior art press brake, although the upper tool can be clamped between the upper clamp member pivotally supported by the upper table and the holder body, since the upper tool must be attached to or removed from the upper table by pivoting the upper tool so as not to interfere with a pivotal axle of the upper tool clamp member; that is, since the upper tool must be pivoted by supporting the upper tool as heavy as 20 to 30 kg manually on the front side of the press brake, there exists a problem in that this work is dangerous for the worker (in particular for the worker's hand and waist).

25 **[0015]** In addition, when a plurality of upper tools are clamped by connecting the upper tools in the longitudinal (horizontal) direction thereof, there exists a problem in that it is impossible to insert and connect an end of another upper tool between and with the already clamped upper tools so that a plurality of upper tools can be arranged in a longitudinal direction thereof.

30 **[0016]** Further, in the first prior art press brake, since the upper tool is clamped between the upper tool clamp member (pivotally supported by the upper table) and the holder body, the upper tool must be exchanged by pivoting the upper tool so as not to interfere with a pivotal axle of the upper tool clamp member, so that there exists another problem in that the shape of the upper tool is not simple, that is, complicated and thereby the upper tool cannot be processed easily.

35 **[0017]** In the second prior art press brake, the upper tool can be attached to or removed from the upper tool holder apparatus by pivoting an operation lever provided on the upper tool holder apparatus from a clamp position to an unclamp position or vice versa to facilitate the upper tool exchange. However, the upper tool is exchanged by the upper tool holder apparatus by shifting the upper tool in the longitudinal (the right and left) direction, when some upper tool elements of a series of the split type upper tools of different lengths are required to be exchanged at the middle portion thereof, there ex-

ists a problem in that the upper tool exchange work is rather troublesome.

[0018] Here, the conventional upper tool holder apparatus (not the prior art) will be explained in further detail with reference to Figs. 25A and 25B. In the drawings, an upper tool clamp member 403 is attached to an upper holder body 401 fixed to a lower portion of an upper table (not shown). Therefore, an upper tool 407 can be fixed or clamped between the clamp member 403 and the upper tool holder body 401 when the clamp member 403 is fastened by use of a fastening bolt 405. Therefore, whenever the upper tools 407 are required to be exchanged with other upper tools, a great number of fastening bolts 405 must be rotated to unfasten and fasten the upper tool clamp members 403, with the result that the upper tool exchange work is troublesome.

[0019] Further, in the upper tool 407, it is desirable that an angle θ of a lower end surface 409E of an upper support portion 409 of the upper holder body 401 and a sliding surface 407F slidable in contact with a vertical surface 409F of the upper tool support portion 409 is formed accurately at 90 degrees in correspondence to the angle between the lower surface 409E and the vertical surface 409F of the upper support body 409. In general, however, this angle is processed as $\theta < 90$ degrees to prevent $\theta > 90$ degrees under consideration of the processing error. The reason is as follows: if $\theta > 90$ degrees, as shown in Fig. 25A exaggeratively, a gap is produced between the lower end surface 409E of the upper tool support portion 409 and the shoulder portion 407S. Therefore, when work is bent, the upper tool 407 is deformed in a direction that this gap is reduced. As a result, the lower end portion 407E of the upper tool 407 is deformed relatively large (due to the punching pressure) in accompany with this deformation due to the pressure of this gap, thus degrading the bending precision.

[0020] On the other hand, if $\theta < 90$ degrees, as shown in Fig. 25B exaggeratively, a gap is produced between the vertical surface 409F of the upper tool support portion 409 and the sliding surface 407F of the upper tool 407. Therefore, when the upper tool clamp member 403 is fastened strongly by the fastening bolt 405, since the upper tool 407 is deformed in such a way that the gap is reduced, there exists the case where the lower end portion 407E of the upper tool 407 is dislocated slightly.

[0021] Here, since the fastening force applied to the fastening bolt 405 differs according to the worker, the deformation at the lower end portion 407E of the upper tool 407 differs according to the worker and/or worker's fatigued conditions.

[0022] Therefore, in the work bending processing by exchanging the upper tools, even if the above-mentioned angle is formed so precise as to lie within an allowable range, there exists a problem in that a high precise bending processing cannot be achieved, when the upper tools cannot be exchanged at a high upper tool exchange reproducibility.

SUMMARY OF THE INVENTION

[0023] With these problems in mind, therefore, it is the object of the present invention to provide an upper tool, which can be exchanged easily at a high upper tool exchange reproducibility by use of the upper tool holder apparatus, and further which will not be dropped ever if released from the upper tool clamping force.

[0024] Another object of the present invention is to provide an upper tool holder apparatus, which can clamp the upper tool under a constant clamping force and under excellent upper tool exchange reproducibility, without modifying the construction of the conventional structure largely.

[0025] Further, the other object of the present invention is to provide a method of holding the upper tool by the upper tool holder apparatus under a constant upper tool clamping force whenever the upper tools are exchanged with other ones.

[0026] To achieve the above-mentioned objects, the present invention provides an upper tool for a press brake, as defined in claim 1.

[0027] Preferred embodiments of the upper tool are defined in claims 1 to 9.

[0028] Further, the present invention provides an upper tool for a press brake in combination with a holding device as defined in any of claims 9 to 13.

[0029] Further, the present invention provides an upper tool assembly for a press brake, comprising an upper tool and an upper tool holder apparatus, as defined in any of claims 14 to 28 and a method of clamping an upper tool as defined in claim 29.

[0030] In the upper tool when the upper tool is moved upward relative to the support plate of the upper tool holder apparatus during the engagement thereof with the die, since the upper tool clamping force can be increased gradually, it is possible to facilitate the exchange work of the upper tool by the use of the upper tool holder apparatus.

[0031] Further, when the upper tool is clamped by the upper tool holder apparatus, since an appropriate clamping force can be always obtained, it is possible to prevent the upper tool from being dropped due to lack of sufficient clamping force, while facilitating the upper tool exchange work.

[0032] Further, whenever the upper tool is exchanged, since the upper tool can be supported by the upper tool clamp member, the upper tool exchange work can be facilitated in safety. Further, since the vertical surface of the upper tool is used as the clamping surface, the upper tool according to the present invention can be clamped to the upper table by use of the ordinary upper tool clamp member.

[0033] Further, the upper tool holder apparatus according to the present invention, since the clamping force control means (the pusher screw) is thread-engaged with the upper tool clamp member and further since the clamp operation means (the operation lever)

fixed to the pusher screw is selectively pivoted to the clamp position A for firmly clamping the upper tool, an unclamp position B for shifting the upper tool horizontally for upper tool adjustment and alignment, and further to the exchange position C for upper tool exchange, it is possible to adjust and exchange the upper tool easily and safely.

[0034] Further, since the upper tool clamp member urging means (the coil springs) are provided, when the operation lever is pivoted to the unclamp position B, the upper tool clamp member can be urged slightly in the clamp direction, so that it is possible to prevent the upper tool from being dropped during the upper tool adjustment or alignment.

[0035] Further, since the clamp release holding means (the engage recess and the ball plunger) are provided, when the operation lever is pivoted to the exchange position C, the upper tool clamp member can be kept away from the upper tool, so that it is possible to exchange the upper tool with a new one in the vertical direction easily.

[0036] Further, since the stopper members are provided for determining the clamp position A and the unclamp position B of the operation lever, it is possible to easily pivot the operation lever to the respective positions.

[0037] Further, since the wedge piece is provided on the engage projection of the upper tool so as to be engaged with the drop prevention groove of the upper tool, it is possible to reduce the gap between the engage projection of the upper tool clamp member and the drop prevention groove of the upper tool for improvement of the upper tool drop.

[0038] Further, the upper tool holder apparatus according to the present invention, the upper tool can be exchanged easily by use of the upper tool holder apparatus. Further, whenever the upper tools of the same upper wedge dimension are exchanged, since the clamping force of the upper tool can be kept at a constant value, it is possible to obtain an excellent reproducibility of the alignment between the upper tool and the die, ever after the upper tools are exchanged for various bending process. As a result, the upper tool exchange work can be facilitated and further the bending precision can be improved.

BRIEF DESCRIPTION OF THE DRAWINGS

[0039]

Fig. 1 is a front view showing a first embodiment of the upper tool holder apparatus according to the present invention;

Fig. 2 is a cross-sectional view, taken along the line 2S - 2S shown in Fig. 1;

Fig. 3 is a cross-sectional view, taken along the line

3S - 3S shown in Fig. 1;

Fig. 4 is a partial side view showing the shape of the upper tool according to the embodiment of the present invention;

Fig. 5 is an illustration for assistance in explaining the relationship between the upper tool and the upper tool holder apparatus;

Fig. 6 is a side view showing a modification of the upper tool according to a further embodiment of the present invention;

Fig. 7 is a side view showing a modification of the upper tool holder apparatus;

Fig. 8 is a side view showing another modification of the upper tool holder apparatus;

Fig. 9 is a side view showing still another modification of the upper tool holder apparatus;

Fig. 10 is a side view showing the other modification of the upper tool holder apparatus;

Fig. 11 is a front view showing a second embodiment of the upper tool holder apparatus;

Fig. 12 is a cross-sectional view, taken along the line 12S - 12S shown in Fig. 11;

Fig. 13 is a cross-sectional view, taken along the line 13S - 13S shown in Fig. 11;

Figs. 14A and 14B are illustrations for assistance in explaining the function when the operation lever is pivoted to the exchange position C for upper tool exchange;

Figs. 15A and 15B are illustrations for assistance in explaining the function when the operation lever is pivoted to the unclamp position B for upper tool adjustment;

Figs. 16A and 16B are illustrations for assistance in explaining the function when the operation lever is pivoted to the clamp position A for upper tool clamp;

Figs. 17A and 17B are illustrations for assistance in explaining the other function when the operation lever is pivoted to the clamp position A for alignment with the die;

Figs. 18A and 18B are illustrations for assistance in explaining the other function when the operation lever is pivoted to the exchange position C for upper tool exchange and further the upper tool clamp

member is pivoted away from the upper tool to provide an exchange space;

Fig. 19 is a front view showing an upper tool holder apparatus including an upper tool which is not according to the present invention;

Fig. 20 is a cross-sectional view, taken along the line 20S - 20S shown in Fig. 19;

Fig. 21 is a side view showing a modification of the fastening means;

Fig. 22 is a side view showing another modification of the fastening member;

Fig. 23 is a side view showing still another modification of the fastening member;

Fig. 24 is a side view showing the other modification of the fastening member; and

Figs. 25A and 25B are partial side views showing a conventional upper tool.

DETAILED DESCRIPTION OF THE EMBODIMENTS

[0040] First, the whole construction of the upper tool (punch) holder apparatus 1 will be described hereinbelow with reference to Figs. 1 to 3.

[0041] The upper tool holder apparatus 1 of the embodiment is removably attached to the lower portion of an upper table 3 of a press brake (not shown).

[0042] The upper tool holder apparatus 1 is composed of a holder body 5 removably attached to the upper table 3 and having a support plate 7, an upper tool clamp member 11 pivotally supported by the holder body 5 to clamp the upper portion 9U of an upper tool (punch) 9 in cooperation with the support plate 7 of the holder body 5, a clamping force adjusting device 13 for adjusting the clamping force applied to the upper tool clamp member 11, and a clamping force releasing device 15 for releasing the clamping force applied from the upper tool clamp member 11 to the upper tool 9. Further, a small pivotal contact member 17 is attached to end portion of the upper tool clamp member 11 in such a way that a contact surface 17S thereof is brought into slidable contact with an inclined surface 9S of the upper tool 9, as shown in Fig. 2.

[0043] In more detail, the holder body 5 is composed of a thick-wall upper block portion 5B and a thin-wall support plate 7 formed integral with each other (when the thickness is seen from the right and left direction in Figs. 2 and 3). Further, a mounting plate 21 is fixed to the front surface (the left side in Fig. 2) of the upper block 5B of the holder body 5 with a plurality of bolts 19 (See Fig. 1).

[0044] Therefore, when the mounting plate 21 is

brought into contact with the lower front surface of the upper table 3 and after that a clamp jaw 25 is fastened with a fastening bolt 23 to the upper table 3, the mounting plate 21 can be urged against the upper table 3, so that the holder body 5 can be fixed to the upper table 3.

[0045] To adjust the vertical position of the holder body 5, a wedge member 27 is interposed between the upper surface of the holder body 5 and the lower surface of the upper table 3. A fixing bolt 31 passed through a slot 29 (See Fig. 1) formed in the front mounting plate 21 so as to extend in the right and left direction is thread-engaged with the wedge member 27 (See Fig. 3).

[0046] In the above-mentioned construction, under the conditions that the clamp jaw 25 is slightly fixed to such an extent that the holder body 5 will not drop and further the fixing bolt 31 is slightly loosened, when the wedge member 27 is adjusted in the right and left direction in Fig. 1, it is possible to finely adjust the vertical position of the holder body 5 relative to the upper table 3.

[0047] The upper tool clamp member 11 is a plate member having roughly the same width as that of the holder body 5 (in Fig. 1), and pivotally supported by the holder body 5 via a plurality of mounting bolts 33 (when seen in Fig. 2). Therefore, the upper portion 9U of the upper tool 9 is clamped between the lower end of the upper tool clamp member 11 and the support plate 7 of the upper tool holder body 5.

[0048] In more detail, the upper tool clamp member 11 is supported by a plurality of mounting bolts 33 passed through a plurality of through holes 11H formed at a vertically middle portion of the upper tool clamp member 11 and thread-engaged with the support plate 7 in the horizontal direction so as to be pivotal in the front and rear direction (the right and left direction in Fig. 2). To facilitate the pivotal motion of the upper tool clamp member 11, the contact surfaces between the head of the respective mounting bolts 33 and the upper tool clamp member 11 are formed into a spherical shape, respectively. Further, an elastic body 37 (e.g., coil spring, rubber body, etc.) is interposed between the upper tool clamp member 11 and the support plate 7 to keep these two members away from each other.

[0049] Further, at the lower portion of the upper tool clamp member 11, an inclined surface 11S is formed so as to approach the support plate 7 at the upper portion thereof. Further, a contact member 17 is attached to the inclined surface 11S so as to be pivotal slightly.

[0050] In more detail, two through holes are formed on both sides of the lower portion of the upper tool clamp member 11 (See Fig. 1). A mounting bolt 39 is passed through each of the through holes and further thread-engaged with the contact member 17. A clearance is formed between each through hole and the mounting bolt 39, so that the contact member 17 can be pivotal slightly relative to the upper tool clamp member 11.

[0051] Further, an engage projection 11K is formed at the lower portion of the upper tool clamp member 11 (below the contact member 17) so as to be engaged with

an engage groove 9G (extending in the right and left direction in Fig. 1) formed in the upper tool 9.

[0052] The upper tool clamp member 11 serves to give a force for clamping the upper tool 9 in cooperation with the support plate 7. Further, a clamping force adjusting device 13 (See Fig. 3) for adjusting the upper tool clamping force is fitted to a horizontal hole 5H formed in the upper block portion 5B of the holder body 5.

[0053] In more detail, as shown in Fig. 3, the clamping force adjusting device 13 is composed of an adjust screw 41, a nut member 45 thread-engaged with the adjust screw 41 to adjust the position of a ring member 43 movably engaged with the adjust screw 41, and an elastic member (e.g., spring) 47 interposed between a head portion 41H of the adjust screw 41 and the ring member 43.

[0054] In the above-mentioned construction, when the adjust screw 41 is adjustably rotated, since the engage position between the adjust screw 41 and the nut member 45 can be shifted, the compression force or the clamping force of the elastic member 47 can be adjusted. Here, the head portion 41H of the adjust screw 41 of the clamping force adjusting device 13 is in contact with the bottom wall portion of the hole 5H, and further a cylindrical push member 49 enclosing the nut member 45 is in contact with the ring member 43, as shown in Fig. 3.

[0055] Further, an end portion of a fastening screw 51 of the clamping force releasing device 15 (provided at the upper portion of the upper tool clamp member 11) is in contact with the push member 49. In more detail, the clamping force releasing device 15 is a fastening screw 51 passed through and thread-engaged with the upper portion of the upper tool clamp member 11. Further, a lever 53 is fixed to the fastening screw 51.

[0056] Accordingly, when the lever 53 is pivoted, the fastening screw 51 is fastened toward or unfastened away from the push member 49 to apply or release the upper tool clamping force of the clamping force adjusting device 13.

[0057] Further, to limit the pivotal range of the lever 53, two stopper pins 55 are implanted in the mounting plate 21, as shown in Fig. 1.

[0058] In the above-mentioned construction, under the condition that the upper portion 9U of the upper tool 9 is interposed between the upper tool clamp member 11 and the support plate 7 of the holder body 5, when the lever 53 of the clamping force releasing device 15 is pivoted to the right side R in Fig. 1 to fasten the fastening screw 51, since the elastic member 47 of the clamping force adjusting device 13 is further compressed, the elastic force of the elastic member 47 increases, so that the upper tool 9 can be clamped by the upper tool clamp member 11 by a stronger force due to a reactive elastic force thereof.

[0059] In contrast with this, when the lever 53 of the clamping force releasing device 15 is pivoted to the left side L in Fig. 1 to unfasten the fastening screw 51, since

the elastic member 47 of the clamping force adjusting device 13 is released from compression, the elastic force of the elastic member 47 decreases, so that the upper tool 9 can be unclamped from the upper tool clamp member 11.

[0060] As described above, when the lever 53 is pivoted in the right and left direction in Fig. 1, the upper tool clamp member 11 is pivoted in the right and left direction in Fig. 2 to clamp or unclamp the upper tool 9 in cooperation of the support plate 7.

[0061] Further, a clamp opening springs 59 is interposed between each of the two bolts 57 (passed through the upper portion of the upper tool clamp member 11 and thread-engaged with the holder body 5 as shown in Fig. 2) and the holder body 5, so that the lower portion of the upper tool clamp member 11 can be opened, whenever the upper tool 9 is unclamped.

[0062] As described above, the upper tool clamp member 11 can clamp and unclamp the upper tool 9 at the fixed position.

[0063] The upper tool 9 removably attached to the upper tool holder apparatus 1 constructed as described above will be explained hereinbelow with reference to Fig. 4.

[0064] The upper tool 9 is formed with a contact surface 9F brought into contact with the lower end surface 7E of the support plate 7. Further, a vertical sliding surface 9V brought into slidably contact with the front 7F or rear surface of the support plate 7 is formed in the upper portion 9U projecting from the contact surface 9F of the upper tool 9. An engage groove 9G is formed on a surface opposite to the sliding surface 9V of the upper tool 9. In addition, a short vertical surface 9SV is formed between the inclined surface 9S formed above the engage groove 9G and the engage groove 9G.

[0065] The vertical surface 9SV can be used when the upper tool 9 is clamped by the ordinary upper tool clamp member.

[0066] Further, the upper tool 9 is formed with a work processing portion 9M at the lower end portion thereof to bend work W in cooperation with a die (lower tool) 63 attached to the lower table 61 as shown in Fig. 2.

[0067] As understood by Fig. 4, the lower surface 9GL of the engage groove 9G of the upper tool 9 is formed at a position higher than the contact surface 9F thereof. Further, the upper surface 9GF of the engage groove 9G is formed so as to be slightly inclined downward toward the groove bottom in correspondence to the upper surface of the engage projection 11K of the upper tool clamp member 11.

[0068] In the above-mentioned construction, since the lower end portion of the upper tool clamp member 11 can be determined relatively short at a position higher than the lower end surface 7E of the support plate 7, it is possible to prevent the upper tool clamp member 11 from interfering with work W bent at an acute angle, for instance.

[0069] Further, when the upper tool 9 is engaged with

the upper tool clamp member 11 under the condition that the engage projection portion 11K of the upper tool clamp member 11 is in contact with the upper surface 9GF of the engage groove 9G of the upper tool 9, since the contact surface of the engage projection portion 11K is slightly inclined downward, it is possible to always urge the vertical sliding surface 9V of the upper tool 9 against the support plate 7 due to the weight of the upper tool 9.

[0070] Further, since the engage groove 9G of the upper tool 9 is located at a position higher than the contact surface 9F thereof, the width of the upper portion 9U of the upper tool 9 from a corner (at an intersection between the contact surface 9F and the sliding surface 9V) can be increased, so that the strength of the upper tool 9 can be increased.

[0071] In the upper tool 9 as described above, the upper tool 9 can be attached to the upper tool holder apparatus 1 in the procedure as follows:

[0072] First, the lever 53 of the clamping force releasing device 15 is pivoted to the position R to fasten the fastening screw 51. Even under the condition that the fastening screw 51 is fastened, a gap can be formed between the lower portion of the upper tool clamp member 11 and the support plate 7.

[0073] Therefore, the upper portion 9U of the upper tool 9 can be inserted between the upper tool clamp member 11 and the support plate 7 horizontally (in the right and left direction in Fig. 1) in such a way that the engage groove 9G formed in the upper tool 9 is engaged with the engage projection 11K of the lower portion of the upper tool clamp member 11 as shown in Fig. 2 and further the inclined surface 9S of the upper tool 9 is in contact with the contact surface 17S of the contact member 17.

[0074] After that, the movable table (any one of the upper table 3 and the lower table 61) of the press brake is moved in the vertical direction to engage the upper tool 9 with the die 63. Therefore, the upper tool 9 is moved upward relative to the holder body 5.

[0075] When the upper tool 9 is moved upward gradually toward the holder body 5, the inclined surface 9S of the upper tool 9 urges the lower portion of the upper tool clamp member 11 in the leftward direction via the contact member 17 in Figs. 2 and 4, so that the upper tool clamp member 11 is pivoted clockwise in Fig. 2 to gradually compress the elastic member 47 of the clamping force adjusting device 13.

[0076] Accordingly, when the upper tool 9 is moved upward and thereby the contact surface 9F is brought into contact with the lower end surface 7E of the support plate 7, the upper tool 9 can be clamped strongly by the upper tool clamp member 11 on the basis of an elastic force of the elastic member 47. Therefore, the upper tool 9 can be easily clamped by the upper tool holder apparatus 1.

[0077] As described above, after the upper tool 9 has been clamped by the upper tool holder apparatus 1, the

upper tool 9 can be removed from the upper tool holder apparatus 1 as follows: First, the lever 53 is pivoted to the left side L to loosen the fastening screw 51. Then, the upper tool 9 is released from the upper tool clamp member 11.

[0078] When the upper tool 9 has been released from the upper tool clamp member 11, the upper tool 9 drops to the original lower position (as shown in Fig. 2) by the weight of itself. In this case, since the upper surface 9GF of the engage groove 9G of the upper tool 9 is engaged with the engage projection 11K of the upper tool clamp member 11, it is possible to prevent the upper tool 9 from being dropped, so that the upper tool removal operation is safe.

[0079] After the upper tool 9 has been released from the upper tool clamp member 11 as described above, the upper tool 9 can be removed from the upper tool holder apparatus 1 by shifting the upper tool 9 horizontally in the right and left direction in Fig. 1.

[0080] As explained above, in the upper tool 9 and the upper tool holder apparatus 1, it is possible to easily clamp or unclamp the upper tool 9 by use of the upper tool holder apparatus 1, without use of any tools, in spite of the simple construction.

[0081] Further, as already understood, when the upper tool 9 is engaged with the die 63, since the lower portion of the upper tool clamp member 11 is urged upward or clockwise by the inclined surface 9S of the upper tool 9, the clamping force of the upper tool 9 between the upper tool clamp member 11 and the support plate 7 can be increased gradually as the upper tool 9 is moved upward. Therefore, it is possible to consider that the upper tool clamp member 11, the clamping force adjusting device 13, etc. constitutes a kind of clamping force increasing mechanism such that when the upper tool 9 is moved upward relative to the support plate 7, the clamping force can be increased gradually.

[0082] In the above-mentioned construction, although the clamping force adjusting device 13 having the elastic member 47 is used as a part of the clamping force increasing mechanism, it is also possible to adopt a hydraulic (e.g., gas) cylinder including a compressive fluid (gas) instead of the clamping force adjusting device 13. Further, it is also possible to replace the upper tool clamp member 11 with a leaf spring to use the elastic deformation of the upper tool clamp member itself, without use of the clamping force adjusting device 13. In other words, various constructions can be adopted as the clamping force increasing mechanism of the upper tool holder apparatus 1.

[0083] Further, in the upper tool holder apparatus 1, the upper tool 9 can be attached to the rear surface of the support plate 7 by inverting the front and rear sides of the upper tool 9, as shown in Fig. 3. In other words, a rear side upper tool clamp member 65 is pivotally provided on the rear surface of the support plate 7 in such a way that the upper tool 9 can be clamped between the rear side upper tool clamp member 65 and the rear sur-

face side of the support plate 7.

[0084] In more detail, as shown in Fig. 3, a stud 67 having a semi-spherical head 67H is attached horizontally to the rear surface of the support plate 7 of the holder body 5 with the use of a mounting bolt 69. Further, the upper tool clamp member 65 is pivotally supported by the head 67H of this stud 67. To rotate the stud 67, a tool hole 11T is formed in the front-side upper tool clamp member 11. Further, a stop pin 71 engaged with a groove 65G formed in the upper tool clamp member 65 is attached to the head 67H of the stud 67. Therefore, even when the mounting bolt 69 is rotated, the stud 67 will not be rotated together with the mounting bolt 69.

[0085] To use the clamping force adjusting device 13 in common for both the front-side upper tool clamp member 11 and the rear-side upper tool clamp member 65, a small diameter hole 73 is formed in the bottom wall portion of the hole 5H of the holder body 5. Further, a contact member 75 thread-engaged with the upper portion of the rear-side upper tool clamp member 65 is passed through the small diameter hole 73 and further brought into contact with the head 41H of the adjust screw 41.

[0086] Further, on the front side of the hole 5H, a ring nut 77 for stop the movement of the push member 49 of the clamping force adjusting device 13 is thread-engaged with the holder body 5.

[0087] In the construction as described above, when the mounting bolt 69 is fastened, the upper tool 9 can be clamped between the rear-side upper tool clamp member 65 and the rear surface of the support plate 7. In contrast with this, when the mounting bolt 69 is loosened, the upper tool 9 can be released from the rear-side upper tool clamp member 65.

[0088] Further, when an appropriate plate member is pinched between the front-side upper tool clamp member 11 and the support plate 7 to keep the front-side upper tool clamp member 11 unmoved, it is possible to clamp and unclamp the upper tool 9 between the support plate 7 and the rear-side upper tool clamp member 65 by pivoting the lever 53.

[0089] As described above, the upper tool 9 can be selectively clamped between any one of the upper tool clamp members 11 and 65 and any one of the front and rear surfaces of the support plate 7 according to the bending shape of work W. In addition, the upper tool 9 can be exchanged easily by use of the upper tool holder apparatus 1.

[0090] As described above, in the upper tool holder apparatus, the upper tool 9 can be easily exchanged by use of the upper tool holder apparatus 1. In this exchange work, even if the upper tool 9 is released from the clamping force of the upper tool holder apparatus 1, since the upper tool 9 will not drop, it is possible to improve the safety of the upper tool holder apparatus 1.

[0091] As already explained, when the upper tool 9 is moved upward relative to the support plate 7 of the upper tool holder apparatus 1 and thereby the inclined sur-

face 9S of the upper tool 9 urges the lower portion of the upper tool clamp member 11, the clamping force of the upper tool clamp member 11 increases gradually. Therefore, it is necessary to obtain a sufficiently large clamping force when the contact surface 9F of the upper tool 9 is brought into contact with the lower end surface 7E of the support plate 7.

[0092] Therefore, when the upper surface 9GF of the engage groove 9G of the upper tool 9 is in contact with the upper surface of the engage projection 11K of the upper tool clamp member 11, as shown in Fig. 5, it is preferable that the distance H between the contact surface 9F of the upper tool 9 and the lower end surface 7E of the support plate 7 satisfies the following relationship:

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

where A denotes the distance between a pivotal center of the upper tool clamp member 11 and the elastic means 47 of the clamping force adjusting device 13 for urging the upper tool clamp member 11; B denotes an average distance between a pivotal center of the upper tool clamp member 11 and a clamping force application point (at which the upper tool clamp member 11 pushes the upper tool 9 against the support plate 7 via the contact member 17); P denotes the clamping force for pushing the upper tool 9 against the support plate 7; K denotes the elastic coefficient (modulus of elasticity) of the elastic member 47; and the inclined surface 9S from the vertical line.

[0093] Here, when the upper tool 9 is moved upward relative to the support plate 7, as shown in Fig. 5, the lower portion of the upper tool clamp member 11 is displaced leftward by ΔL and the upper portion of the upper tool clamp member 11 is displaced rightward by ΔX , so that the elastic member 47 is compressed gradually to increase the clamping force gradually.

[0094] Here, it is not desirable that the distance H is smaller than that expressed by the above formula. This is because when the contact surface 9F of the upper tool 9 is brought into contact with the lower end surface 7E of the support plate 7 by a slight upward movement of the upper tool 9 relative to the support plate 7, a sufficient clamping force cannot be obtained. As a result, there exists a danger that the upper tool 9 drops.

[0095] In contrast with this, when the distance H is larger than that expressed by the above formula, when the contact surface 9F of the upper tool 9 is in contact with the lower end surface 7E of the support plate 7, since the displacement rate of the elastic member 47 becomes larger than ΔX , a large clamping force can be obtained. In this case, however, there exists such a case that when the upper tool 9 is required to be released by pivoting the lever 53, the upper tool 9 cannot be released sufficiently due to an excessive clamping force, so that it becomes difficult to exchange the upper tool 9 by use

of the upper tool holder apparatus 1. Therefore, this is also not desirable.

[0096] Accordingly, it is desirable that the inclination angle θ of the inclined surface 9S of the upper tool 9 lies between 5 and 20 degrees. That is, when the inclination angle is less than 5 degrees, the distance H must be increased to deform the elastic member 47 sufficiently. Further, when more than 20 degrees, although the distance H can be reduced, this is not desirable because the downward component force becomes large.

[0097] On the other hand, in order to hold the upper tool 9 so as not to be dropped by its weight, it is necessary to increase the frictional force between the inclined surface 9S of the upper tool 9 and the contact surface 17S of the contact member 17 beyond the weight of the upper tool 9. The frictional force (the maximum static friction force) can be expressed as (static friction coefficient $\times$ contact force), and the friction coefficient can be expressed as $\tan \theta$ (where θ is a friction angle).

[0098] The friction coefficient changes according to the surface conditions of the inclined surface 9S and the contact surface 17S (i.e., surface roughness, material, presence or absence of lubricant, etc.). However, since the friction coefficient between two metals usually lies between 0.15 and 0.20, it is desirable that the inclination angle θ of the inclined surface 7S lies between 7 and 11 degrees.

[0099] Further, the upper tool holder apparatus according to the present invention can be modified as follows:

[0100] As shown in Fig. 6, it is also preferable to form a high hardness portion 9H at the inclined surface 9S and the sliding surface 9H of the upper tool 9, in order to improve the abrasion resistance. To form the high hardness portion 9H, a hard alloy chip is attached onto the surfaces, or the surfaces are coated with a hard material or quenched.

[0101] With reference to Figs. 7 to 10, other upper tool holder apparatus for holding the upper tool 9 according to the present invention will be explained.

[0102] In the upper tool holder apparatus 100 shown in Fig. 7, an upper tool clamp member 101 is fastened to the support plate 7 of the holder body 5 by use of a bolt 103, and further a clamp piece 105 is provided at the lower portion of the upper tool clamp member 101.

[0103] The clamp piece 105 is formed into a semi-spherical or semi-cylindrical shape, and pivotally supported by a slider 107 slidably fitted to a hole 101H formed in the lower portion of the upper tool clamp member 101. Further, an elastic member 109 (such as a dish spring, urethane rubber, etc.) is interposed between the slider 107 and the bottom portion of the hole 101H of the upper tool clamp member 101.

[0104] In this embodiment, when the bolt 103 is fastened, the space between the support plate 7 of the holder body 5 and the upper tool clamp member 101 can be kept constant, so that the space between the support plate 7 and the clamp piece 105 can be also

kept constant.

[0105] To clamp the upper tool 9 by the upper tool holder apparatus 100, the upper portion 9U of the upper tool 9 is inserted between the support plate 7 and the clamp piece 105 horizontally in the right and left direction (perpendicular to the paper). After that, when the upper tool 9 is moved upward relative to the holder body 5 due to the engagement with a die (not shown), since the elastic member 109 is compressed, the upper tool 9 can be urged against the support plate 7 by the elastic force of the elastic member 109. Further, the upper tool 9 can be released when the bolt 103 is loosened.

[0106] In the case of an upper tool holder apparatus shown in Fig. 8, a wedge-shaped spring washer 111 is disposed at the bottom portion of the hole 101H formed in an upper tool clamp member 101 in such a way as to be adjusted by another wedge member 115 moved by an adjust screw 113. The structure other than the above is the same as that shown in Fig. 7.

[0107] In this modification, when the wedge 115 is moved by the adjust screw 113, the elastic force of the elastic member 109 can be adjusted via a spring washer 111.

[0108] In the case of an upper tool holder apparatus shown in Fig. 9, a thin-wall portion 101A is formed at a part of the upper tool clamp member 101 in such a way as to be deformed elastically. In other words, the upper tool clamp member 101 itself is provided with the elastic member.

[0109] In the case of the upper tool holder apparatus shown in Fig. 10, a bolt 103 is passed through the support plate 7, and an elastic member 119 is interposed between a nut member 117 thread-engaged with the end of the bolt 103 and the support member 7.

[0110] As described above, in the upper tool clamped by the upper tool holder apparatus, when the upper tool is moved upward relative to the support plate of the upper tool holder apparatus during the engagement thereof with the die, since the upper tool clamping force can be increased gradually, it is possible to facilitate the exchange work of the upper tool by the use of the upper tool holder apparatus.

[0111] Further, when the upper tool is clamped by the upper tool holder apparatus, since an appropriate clamping force can be always obtained, it is possible to prevent the upper tool from being dropped due to lack of sufficient clamping force, while facilitating the upper tool exchange work.

[0112] Further, whenever the upper tool is exchanged, since the upper tool can be supported by the upper tool clamp member, the upper tool exchange work can be facilitated in safety. Further, since the vertical surface of the upper tool is used as the clamping surface, the upper tool according to the present invention can be clamped to the upper table by use of the ordinary upper tool clamp member.

[0113] A second embodiment of the upper tool holder apparatus according to the present invention will be de-

scribed hereinbelow with reference to Figs. 11 and 12.

[0114] The upper tool holder apparatus 201 of the present embodiment is removably attached to the lower portion of an upper table 203 of a press brake (not shown). Further, a plurality of the upper tool holder apparatus 201 are mounted being arranged at appropriate intervals horizontally in the right and left direction of the lower portion of the upper table 203 in Fig. 11. In other words, since various types of the upper tools 205 such as a single long upper tool (extending horizontally in the right and left direction perpendicular to Fig. 12), a plurality of split-type upper tools of different short lengths, etc. are used for a press brake, a plurality of the upper tool holder apparatus 29 are arranged at intervals on the upper table 203.

[0115] As shown in Fig. 12, the upper tool holder apparatus 201 is composed of a holder body 207 removably attached to the upper table 203 and having an upper tool support portion 209 at the lower portion thereof, an upper tool clamp member 211 for clamping an upper portion 205U of an upper tool 205 in cooperation with the upper tool support portion 209 of the holder body 207, clamping force generating means (e.g., a spring) 213 for applying a clamping force to the upper tool clamp member 211, clamp operating means (e.g., a lever) 215 for controllably transmitting a clamping force of the clamping force generating means 213 to the upper tool clamp member 211, upper tool clamp member urging means (springs) 217 (See Fig. 11) for slightly urging the upper tool clamp member 211 in the upper tool clamp direction, and clamp release holding means 219 for keeping the upper tool clamp member 211 in the clamp release state (for upper tool exchange) against the urging force of the upper tool clamp member urging means 217.

[0116] In more detail, the holder body 207 is formed with a thick-wall upper block portion 207B and a thin-wall lower support plate 209 both integral with each other (when seen from the right and left direction in Fig. 12). Further, a mounting plate 225 projecting upward is fixed to the front surface (the left side in Fig. 12) of the upper block portion 207B of the holder body 207 with a plurality of bolts 221 and pins 223 (both shown in Fig. 11).

[0117] As shown in Fig. 11, the mounting plate 225 is formed with a cutout portion 255C widened downward at the lower middle portion thereof. Further, as shown in Fig. 12, the upward projecting portion of the mounting plate 225 is brought into contact with the front lower portion of the upper table 203. When a clamp jaw 229 is fastened with a fastening bolt 227 thread-engaged with the upper table 203, the upper projecting portion of the mounting plate 225 is fixed to the upper table 203, so that the holder body 207 can be fixed to the upper table 203.

[0118] Further, in the above-mentioned embodiment, although the holder body 207 and the mounting plate 225 are provided separately and fixed to each other, it is also possible to form the holder body 207 and the

mounting plate 225 integral with each other. In this case, the mounting plate 225 is regarded as a part of the holder body 207.

[0119] To adjust the vertical position of the holder body 207, a wedge member 231 is adjustably interposed between the upper surface of the holder body 207 and the lower surface of the upper table 203 so as to be movable horizontally in the right and left direction (in Fig. 11). A fixing bolt 235 passed through a slot 233 (extending in the right and left direction in Fig. 11) formed in the mounting plate 225 is thread-engaged with the wedge member 231.

[0120] In the above-mentioned structure, under the conditions that the clamp jaw 229 is slightly fastened to such an extent that the holder body 207 will not drop and further the fixing bolt 235 is loosened, when the wedge member 231 is adjustably moved horizontally in the right and left direction in Fig. 11, the vertical position of the holder body 207 can be adjusted finely relative to the upper table 203.

[0121] The upper tool clamp member 211 is a plate member having a width roughly the same as that of the holder body 207. Further, the upper tool clamp member 211 is formed with a projection fitted to the downward-widened cutout portion 255C formed in the mounting plate 225. The upper tool clamp member 221 is pivotally supported by the holder body 207 to clamp and unclamp the upper portion 205U of the upper tool 205.

[0122] In more detail, a plurality of through holes 211H are formed at the vertically middle portion of the upper tool clamp member 211. A plurality of mounting bolts 239 are passed through these through holes 211H, thread-engaged with the upper tool support portion 209 horizontally, and further fixed by nuts 237, respectively. Therefore, the upper tool clamp member 211 can be pivotally supported by head portions of the mounting bolts 239. Further, a coil springs 241 is elastically interposed between the upper tool support portion 209 and the upper tool clamp member 211, respectively.

[0123] Further, the contact surfaces of the head portion of the mounting bolt 239 and the through hole 211H are both formed into a spherical surface, respectively to allow the upper tool clamp member 211 to be pivoted smoothly.

[0124] At the lower portion of the upper tool clamp member 211, an engage projection 211P engaged with a drop prevention groove 205G formed at the upper portion of the upper tool 205 (extending in the right and left direction in Fig. 11) is formed so as to project toward the upper tool support member 209. Further, a push contact member 243 brought into contact with the inclined surface 205S of the upper portion 205U of the upper tool 205 to urge the upper tool 205 against the upper tool support portion 209 is provided slightly above the engage projection 211P.

[0125] The push contact member 243 is formed by cutting parts of the circumferential surface of a cylindrical body into flat surfaces, and mounted at the lower por-

tion of the upper tool clamp member 211 via a plurality of screws 245 so as to be pivoted slightly when seen in Fig. 12.

[0126] Further, a wedge member 247 engaged with the drop prevention groove 205G of the upper tool 205 is provided between the engage projection 211P of the upper tool clamp member 211 and the push contact member 243. This wedge member 247 is always urged by an elastic member 249 such as a coil spring interposed between the upper tool clamp member 211 and the wedge member 247 in the direction as to be engaged with the drop prevention groove 205G of the upper tool 205.

[0127] However, the horizontal movement of the wedge member 247 by the elastic member 249 is restricted when brought into contact with a part of the push contact member 243. Further, the upper rear-side (right side in Fig. 12) end surface of the wedge member 247 is chamfered (247S) to facilitate engagement with and disengagement from the drop prevention groove 205G of the upper tool 205.

[0128] The clamping force generating means 213 is provided in a hole 207H formed in the block portion 207B of the holder body 207. The clamping force generating means 213 applies a clamping force for urging the upper tool 205 against the upper tool support portion 209 of the holder body 207, via the push contact member 243 provided at the lower portion of the upper tool clamp member 211.

[0129] In more detail, as shown in Fig. 12, the clamping force generating means 213 is composed of an adjust screw 251, a ring member 253 slidably fitted to the adjust screw 251, a nut member 255 thread-engaged with the adjust screw 251, and an elastic member 257 such as a dish spring interposed between the head 251H of the adjust screw 251 and the ring member 253.

[0130] The head portion 251H of the adjust screw 251 is in contact with the bottom wall of the hole 207H. Further, one end of the cylindrical push member 259 enclosing the nut member 255 and slidably fitted into the hole 207H is in contact with the ring member 253. The other end of the push member 259 is in contact with a ring nut 261 adjustably thread-engaged with the hole 207H of the holder body 207. Further, a pin-shaped pusher member 259P provided at the other end of the cylindrical push member 259 projects from the ring nut 261 to the left side in Fig. 12.

[0131] The clamp operating means 215 for transmitting the clamping force of the clamping force generating means 213 to the upper tool clamp member 211 is provided above the upper tool clamp member 211.

[0132] In more detail, the clamp operating means 215 has a clamping force control member (a pusher screw) 263 thread-engaged with the upper portion of the upper tool clamp member 211. Therefore, the pusher screw 263 can be pivoted together with the upper tool clamp member 211. The clamping force control member (pusher screw) 263 is brought into contact or away from the

pin-shaped pusher member 259P.

[0133] In this embodiment, therefore, when the pusher screw 263 is loosened relative to the upper tool clamp member 211, the pusher screw 263 is moved deep into the upper tool clamp member 211 away from the pin-shaped pusher member 259P. In contrast with this, when the pusher screw 263 is fastened, the pusher screw 263 is slightly brought into contact with the pin-shaped pusher member 259P. When further fastened, since the pin-shaped pusher member 259P strongly pushes the cylindrical push member 259 and thereby the elastic member 257 is compressed, a strong clamping force can be obtained as a reaction force of the elastic member 257.

[0134] To rotate the clamping force control means (pusher screw) 263, an operation lever 265 is removably attached to the pusher screw 263. That is, an engage hole 263H is formed in the pusher screw 263, and an engage block 267 of the operation lever 265 is engaged with the engage hole 263H. Further, a ball plunger 269 is attached to the engage block 267 in such a way as to be engaged with an groove (not shown) formed in the engage hole 263H. This ball plunger 269 prevents the engage block 267 from being rotated relative to the pusher screw 263.

[0135] When the engage hole 263H is formed into an oval shape or gear shape, since the engage block 267 will not be rotated relative to the pusher screw 263, it is possible to omit the ball plunger 269.

[0136] In the structure as described above, when the operation lever 265 is pivoted, it is possible to pivot the pusher screw 263 as to be moved to or away from the pin-shaped pusher member 259P to control the clamping force of the clamping force generating means 213.

[0137] In more detail, the operation lever 265 can be pivoted to a clamp position A (See Fig. 11) at which the upper tool clamp member 211 clamps the upper tool 205 in cooperation with the upper tool support portion 209, to an unclamp position B at which the upper tool clamp member 211 unclamps the upper tool 205 (so that the upper tool 205 can be shifted horizontally in the right and left direction in Fig. 11) for upper tool horizontal position adjustment, and to an exchange position C at which the upper tool 205 can be removed vertically from the upper tool clamp member 211 for upper tool exchange. Further, two stoppers 271 (See Fig. 11) are attached to the mounting plate 225 to locate the operation lever 265 at the clamp and unclamp positions A and B, respectively.

[0138] In more detail, as shown in Fig. 11, two slits 273 are formed in the mounting plate 225 at such positions as to correspond to the clamp and unclamp positions A and B, respectively. As shown in Fig. 13, a stopper member 271 is pivotally supported via a pin 275 in such a way as to project from or retract from the front surface of the mounting plate 225. Further, a recessed portion 225H communicating with the slit 273 is additionally formed in the mounting plate 225 so that the

stopper member 271 can be easily pivoted upward and downward relative to the surface of the mounting plate 225. Further, a ball plunger 277 engaged with a recessed portion formed in the outer circular-arc shaped portion of the stopper member 271 is attached to the mounting plate 225 to keep the raised stopper member 271 and retracted stopper member 271 at its position, respectively.

[0139] The upper tool clamp member urging means 217 (See Fig. 11) push and urge the upper tool clamp member 211 so that the upper tool 205 can be urged slightly against the upper tool support portion 209. The upper tool clamp member urging means 217 are elastic members 217S such as coil springs interposed between the upper portion of the upper tool clamp member 211 and the block portion 207B of the holder body 207, as shown in Fig. 11.

[0140] The clamp release holding means 219 holds the upper tool clamp member 211 at such a position that the upper tool 205 is perfectly released from the upper tool clamp member 211, against an elastic force of the upper tool clamp member urging means 217. The clamp release holding means 219 is a ball plunger 281 attached on the mounting plate 225 and engaged with an engage recess 279 (e.g., groove, hole, etc.) formed in the upper portion of the upper tool clamp member 211. Further, it is also possible to form the recessed portion 279 in the mounting plate 225 and to attach the ball plunger 281 to the upper tool clamp member 211, in the opposite way thereto.

[0141] The operation of the upper tool holder apparatus according to the present invention will be described hereinbelow with reference to Figs. 11 and 12.

[0142] Under the condition that the upper tool 205 is clamped between the upper support portion 209 of the holder body 207 and the upper tool clamp member 211, as shown by dot-dot-dashed lines in Fig. 12, when the operation lever 265 is pivoted to the clamp position A, since the clamping force control member (the pusher screw) 263 is fastened inward, the pin-shaped pusher member 259P pushes the cylindrical pusher member 259, so that the elastic member 257 of the clamping force generating means 213 is further compressed. Therefore, since the elastic force of the elastic member 257 further increases, the upper tool clamp member 211 is urged counterclockwise by a reaction force thereof in Fig. 12, so that the upper tool 205 can be firmly clamped further strongly between the upper tool clamp member 211 and the upper tool support portion 209.

[0143] In contrast with this, when the operation lever 265 is pivoted to the unclamp position B, since the clamping force control member (the pusher screw) 263 is loosened outward away from the pusher member 259P, the pin-shaped pusher member 259P is in slight contact with the cylindrical pusher member 259, so that the upper tool clamp member 211 is released from the clamping force into the unclamp condition. Under these conditions, however, since the engage projection 211P

of the upper tool clamp member 211 is engaged with the drop prevention groove 205G of the upper tool 205, the upper tool 205 can be shifted horizontally for adjustment or alignment in the right and left direction in Fig. 11.

[0144] Here, when the stopper member 271 provided at the unclamp position B is pushed down deep into the slit 273 formed in the mounting plate 225 and further the operation lever 265 is pivoted counterclockwise to the exchange position C, since the pusher screw 263 is shifted deep into the upper tool clamp member 211 being separated in the left direction in Fig. 12, the pusher screw 263 is kept far away from the pin-shaped pusher member 253P of the clamping force generating means 213. That is, the upper tool clamp member 211 is perfectly released from the clamping force generating means 213. Here, the upper tool clamp member 211 is pivoted counterclockwise for safety by a weak elastic force of the elastic members 217S of the upper tool clamp member urging means 217 (shown in Fig. 11). Under these conditions, when the worker pivots the upper tool clamp member 211 clockwise manually, since the engage recessed portion 279 formed at the upper portion of the upper tool clamp member 211 is engaged with the ball plunger 281, the upper tool clamp member 211 is kept away from the upper tool support portion 209 for upper tool exchange. Here, it should be noted that since the pin-shaped pusher screw 263 is sufficiently shifted leftward relative to the upper tool clamp member 211, the cylindrical pusher member 259P will not interfere with the pusher member 263, when the recessed portion 279 is engaged with the ball plunger 281. That is, under the condition that the recessed portion 279 is engaged with the ball plunger 281, since the lower portion of the upper tool clamp member 211 is kept opened away from the upper tool support portion 209, the upper tool 205 can be exchanged by moving the upper tool 205 in the vertical direction relative to the upper tool clamp member 211.

[0145] Further, under the condition that the engage portion 279 of the upper tool clamp member 211 is engaged with the ball plunger 281, when the operation lever 265 is returned clockwise from the exchange position C to the unclamp position B, since the pusher screw 263 projects from the upper tool clamp member 211 into contact with the pin-shaped pusher member 259P (the clamping force generating means 213), the upper tool clamp member 211 is pivoted counterclockwise again in Fig. 12, so that the ball plunger 281 is disengaged from the recessed portion 279 of the upper tool clamp member 211.

[0146] The upper tool exchange procedure of the upper tool holder 201 apparatus will be explained in further detail with reference to Figs. 14 to 18.

[0147] When the upper tool 205 is required to be clamped by the upper tool holder apparatus 201 from the lower side, the operation lever 265 is shifted counterclockwise to the vertical exchange position C, as shown in Fig. 14A. Under these conditions, since the

lower portion of the upper tool clamp member 211 can be opened against an elastic force of the upper tool clamp member urging means 217, it is possible to insert the upper portion 205U of the upper tool 205 into between the upper tool support portion 209 of the holder body 207 and the upper tool clamp member 211 from below, as shown in Fig. 14B.

[0148] After the upper portion 205U of the upper tool 205 has been inserted between the upper tool support portion 209 of the holder body 207 and the upper tool clamp member 211, since the upper tool clamp member 211 is urged weakly in the clamping direction by the upper tool clamp member urging means 217, the engage projection 211P formed in the lower portion of the upper tool clamp member 211 is engaged with the drop prevention groove 205G formed in the upper tool 205.

[0149] Under the condition that the engage projection 211P of the upper tool clamp member 211 is engaged with the drop prevention groove 205G of the upper tool 205, when the operation lever 265 is pivoted to the unclamp position B, as shown in Fig. 15A, since the pusher screw 263 is brought into slight contact with the pusher member 259P (the clamping force generating means 213), a slight clamping force is applied to the upper tool 205 and thereby the upper tool 205 is prevented from being dropped. Under these conditions, the upper tool 205 can be shifted in the horizontal (right and left) direction in Fig. 11, as shown in Fig. 15B. That is, under the condition that the operation lever 265 is pivoted to the unclamp position B, the upper tool 205 can be adjustably moved in the horizontal direction to locate the upper tool 205 relative to the upper tool holder apparatus 201. In this case, if the upper tool 205 is of split type, a plurality of split upper tools are arranged and located so as to be brought into contact with each other.

[0150] Under these conditions, when the operation lever 265 is pivoted to the clamp position A as shown in Fig. 16A, the upper tool 205 can be firmly clamped between the upper tool support portion 209 of the holder body 207 and the upper tool clamp member 211 of the upper tool holder apparatus 201. That is, since the pusher screw 263 pushes strongly the pusher member 259P of the clamping force generating means 213, the upper tool clamp member 211 firmly clamps the upper tool 205 in cooperation with the upper tool support portion 209 by a reaction force of the clamping force generating means 213 (the elastic member 257), as shown in Fig. 16B.

[0151] After that, when the movable-side table of the press brake is moved up and down to engage the upper tool 205 with the die (not shown), the upper tool 205 is inserted further upward between the upper tool support portion 209 and the upper tool clamp member 211, as shown in Fig. 17B.

[0152] In this case, since the inclined surface 205S of the upper tool 205 pushes the push contact member 243, when the upper tool 205 is moved upward relative to the upper tool clamp member 211, the upper tool

clamp member 211 is pivoted clockwise in Fig. 12, so that the pusher screw 263 pushes the pusher member 259P of the clamping force generating means 213 gradually, with the result that the elastic force of the elastic member 257 further increases, until the shoulder portion 205F of the upper tool is brought into contact with the upper tool support portion 209. Therefore, it is possible to clamp the upper tool 205 by the upper tool clamp member 211 gradually more strongly.

[0153] Further, when the shoulder portion of the upper tool 205 is brought into contact with the lower end surface of the upper tool support portion 209, the upper tool 205 is clamped firmly between the upper tool clamp member 211 and the upper tool support portion 209, as shown in Fig. 17B. Further, as shown in Figs. 12 and 17B, since the wedge piece 247 is engaged with the drop prevention groove 205G of the upper tool 205, a gap between the engage projection 211P and the drop prevention groove 205G can be reduced.

[0154] As described above, under the conditions that the wedge piece 247 is engaged with the drop prevention groove 205G of the upper tool 205, since a gap between the two is reduced, even if the upper tool clamp member 211 is released, the upper tool 205 will not drop by the weight thereof.

[0155] Further, after the upper tool 205 has been clamped between the upper tool clamp member 211 and the upper tool support portion 209 and the work is bent in cooperation with the die, the upper tool 205 is required to be removed from the upper tool holder apparatus 201, the operation lever 256 is pivoted counterclockwise to the vertical exchange position C, as shown in Fig. 18A.

[0156] Under these conditions, since the upper tool clamp member 211 is released from the clamping force, the upper tool clamp member 211 can be pivoted clockwise (the arrow direction R) in Fig. 18B manually to engage the ball plunger 281 with the recess 279 of the upper tool clamp member 211, against an elastic force of the upper tool clamp member urging means 217, so that the lower portion of the upper tool clamp member 211 can be kept opened. Therefore, the upper tool 205 can be moved vertically downward, so that it is possible to remove the upper tool 205 from the upper tool clamp member 211 easily in the downward direction.

[0157] After the upper tool 205 has been removed in the downward direction, the operation lever 265 is pivoted clockwise to the unclamp position B once to disengage the engage recessed portion 279 from the ball plunger 281. After that, the operation lever 265 is pivoted counterclockwise again to the vertical exchange position C, as shown in Fig. 14A. Under these conditions, the upper tool 205 can be attached to the upper tool holder apparatus 211 again from below.

[0158] With reference to Fig. 12 again, a rear upper tool clamp member 283 can be pivotally attached on the rear surface (on the right side surface in Fig. 12) of the holder body 207 so that the upper tool 205 can be clamped by turning over the upper tool 205 from the front

side to the rear side.

[0159] In more detail, on the rear surface of the upper tool support portion 207, a support pin 285 (which corresponds to the mounting bolt 239) is mounted horizontally with a fixing bolt 289 passed through a pipe-shaped spacer 287. Further, the rear upper tool clamp member 283 is pivotally supported by the support pin 285. Further, the head portion 291 of a bolt 291 mounted on the upper portion of the rear upper tool clamp member 283 is in contact with the head portion 251H of the adjust screw 251 of the clamping force generating means 213.

[0160] Further, an engage pin 293 fixed to the support pin 285 is engaged with a groove 283G formed in the rear upper tool clamp member 283, to prevent the support pin 285 from being pivoted relative to the rear upper tool clamp member 283.

[0161] The construction other than the above is the same as that of the upper tool clamp member 211, so that the same reference numerals have been retained for the similar elements which have the same functions as with the case of the front-side upper tool clamp member 211, without repeating the same description.

[0162] The upper tool 205 can be unclamped from the rear upper tool clamp member 283 by rotating the fixing bolt 289. Further, the upper tool 205 can be clamped by the rear upper tool clamp member 283 in the same way as with the case of the second prior art example, so that any detailed description thereof is omitted herein.

[0163] Further, without being limited to only the above-mentioned structure, various modifications can be made. For instance, a gaseous spring can be used as the clamping force generating means 213. Further, instead of the pusher screw (clamping force control member) 263, a cylindrical cam formed with three-stage cams brought into contact with the pusher member 259P can be adopted.

[0164] As described above, in the second embodiment of the upper tool holder apparatus according to the present invention, since the clamping force control means (the pusher screw) is thread-engaged with the upper tool clamp member and further since the clamp operation means (the operation lever) fixed to the pusher screw is selectively pivoted to the clamp position A for firmly clamping the upper tool, an unclamp position B for shifting the upper tool horizontally for upper tool adjustment and alignment, and further to the exchange position C for upper tool exchange, it is possible to adjust and exchange the upper tool easily and safely.

[0165] Further, since the upper tool clamp member urging means (the coil springs) are provided, when the operation lever is pivoted to the unclamp position B, the upper tool clamp member can be urged slightly in the clamp direction, so that it is possible to prevent the upper tool from being dropped during the upper tool adjustment or alignment.

[0166] Further, since the clamp release holding means (the engage recess and the ball plunger) are provided, when the operation lever is pivoted to the ex-

change position C, the upper tool clamp member can be kept away from the upper tool, so that it is possible to exchange the upper tool with a new one in the vertical direction easily.

[0167] Further, since the stopper members are provided for determining the clamp position A and the unclamp position B of the operation lever, it is possible to easily pivot the operation lever to the respective positions.

[0168] Further, since the wedge piece is provided on the engage projection of the upper tool so as to be engaged with the drop prevention groove of the upper tool, it is possible to reduce the gap between the engage projection of the upper tool clamp member and the drop prevention groove of the upper tool for improvement of the upper tool drop.

[0169] An upper tool holder apparatus which is not according to the present invention will be described hereinafter with reference to Figs. 19 and 20. In the drawings, an upper tool holder 301 is removably attached to the lower portion of an upper table 303 of a press brake (not shown). Although only one upper tool holder apparatus 301 is shown, in practice, however, a plurality of upper tool holder apparatus 301 are attached at appropriate intervals to the lower portion of the upper table 303.

[0170] The upper tool holder apparatus 301 is composed of a holder body 307 for supporting the upper tool 305, and an upper tool clamp member 309 for clamping the upper tool 305 in cooperation of the upper tool holder body 307.

[0171] In more detail, the upper tool holder body 307 is formed with a thick-wall upper block portion 311 and a thin-wall upper tool support portion 313 integral with each other. Further, a mounting plate 317 is fixed to the front surface (the right side in Fig. 20) of the upper block portion 311 of the upper tool holder body 307 with a plurality of bolts 315 (See Fig. 19).

[0172] Therefore, when the mounting plate 317 is brought into contact with the lower front surface of the upper table 303 and after that a clamp jaw 321 is fastened by a fastening bolt 319 thread-engaged with the upper table 303, the mounting plate 317 can be fixed to the upper table 303, so that the upper tool holder body 307 can be mounted to the upper table 303.

[0173] To adjust the vertical position of the upper tool holder body 307, a wedge member 323 extending horizontally in the right and left direction (in Fig. 19) is interposed between the upper surface of the upper block portion 311 and the lower surface of the upper table 303. A fixing bolt 325 passed through a slot 317H (See Fig. 20) formed in the mounting plate 317 so as to extend in the right and left direction is thread-engaged with the wedge member 323.

[0174] In the above-mentioned construction, under the conditions that the fastening bolt 319 is slightly fastened to such an extent that the upper tool holder 301 does not drop, when the wedge member 27 is adjusted

in the right and left direction in Fig. 19, it is possible to finely adjust the vertical position of the upper tool holder body 307 relative to the upper table 303.

[0175] The upper tool clamp member 309 is formed by a plate member, and is pivotally (when seen in Fig. 20) supported by the upper tool holder body 307 via a bolt 327 (fastening means) passed through a through hole 309H formed in the upper tool clamp member 309.

[0176] A contact portion 309T is formed in the upper and inner surface of the upper tool clamp member 309 so as to be brought into contact with the front surface of the upper block portion 311 of the upper tool holder body 307, and an engage projection 309P engaged with an engage groove 5G of the upper tool 305 is formed at the lower portion of the upper tool clamp member 309 so as to project inward.

[0177] Further, an inclined surface 309F widened downward and brought into contact with an inclined surface 305F formed at the upper wedge portion 305W of the upper tool 305 is formed in the lower inner surface of the upper tool clamp member 309. As shown in Fig. 20, the inclined surface 309F is so inclined that the space between the upper tool support portion 313 of the upper tool holder body 307 and the upper tool clamp member 309 can be narrowed upward. Further, both the end edges of the inclined surface 309F and the upper surface of the engage projection 309P of the upper tool clamp member 309 are chamfered (309C) so that the upper tool 305 can be inserted smoothly between the upper tool support portion 313 and the upper tool clamp member 309 horizontally in the right and left direction in Fig. 19.

[0178] Further, the bolt 327 (fastening means) is formed with a small-diameter threaded portion 327S thread-engaged with the upper tool support portion 313 and a large-diameter body portion 327B, so that the upper tool clamp member 309 can be fastened by the bolt 327 always at a constant fastening position relative to the upper tool support portion 313.

[0179] Between the head portion 327H of the bolt 327 and the upper tool clamp member 309, an elastic member 329 (e.g., dish spring, urethane rubber, etc.) having a large coefficient of elasticity is interposed. Further, between the upper tool clamp member 309 and the upper tool support portion 313 of the upper tool holder body 307, a weak coil spring 331 is interposed to widen the space between the upper tool clamp member 309 and the upper tool support portion 313.

[0180] In the above-mentioned construction, to clamp the upper tool 305 by the upper tool holder apparatus 301, first the bolt 327 is previously fastened tightly to the upper tool support portion 313 by use of a fastening tool 333 (e.g., wrench). In this case, when the body portion 327B of the bolt 327 is brought into contact with the upper tool support portion 313, since the bolt 327 cannot be further fastened, it is possible to determine the distance between the upper tool support portion 313 and the head of the bolt 327 always at a constant value.

[0181] When the bolt 327 is fastened as described above, since the spring 331 is slightly compressed via the elastic member 329 and the upper tool clamp member 309, the space between the upper tool support portion 313 and the upper tool clamp member 309 can be kept opened at a previously determined constant value by an elastic force of the spring 331.

[0182] Under these conditions, the wedge portion 305W of the upper tool 305 is inserted into the above-mentioned space horizontally in the right and left direction in Fig. 19. In this case, since the chamfered surfaces 309C are formed in the upper tool clamp member 309, the wedge portion 305W of the upper tool 305 can be inserted smoothly between the upper tool support portion 313 and the upper tool clamp member 309.

[0183] Under these conditions, since the engage projection 309P of the upper tool clamp member 309 is engaged with the engage groove 305G of the upper tool 305, it is possible to prevent the upper tool 305 from dropping by the weight itself.

[0184] After the wedge portion 305W of the upper tool 305 has been inserted and located in position between the upper tool support portion 313 of the upper tool holder body 307 and the upper tool clamp member 309, the upper table 303 or the lower table 335 of the press brake is moved up and down relative to each other to engage the lower end portion 305E of the upper tool 305 with a die 337 mounted on the lower table 335. In this case, the upper tool 305 is moved upward relative to the upper tool holder body 307. When the shoulder portion 305S of the upper tool 305 is brought into contact with the lower end surface 313E of the upper tool support portion 313, the upper tool 305 stops moving upward and thereby the upper tool 305 can be located (setup) relative to the upper tool holder body 307.

[0185] As described above, when the upper tool 305 is moved upward relative to the upper tool holder body 307, since the inclined surface 305F of the wedge portion 305W of the upper tool 305 urges the inclined surface 309F of the upper tool clamp member 309, the space between the upper tool support portion 313 and the upper tool clamp member 309 is opened, so that the elastic member 329 is compressed.

[0186] In this case, since the space between the lower end surface 313E of the upper tool support portion 313 and the shoulder portion 305S of the upper tool 305 is always kept constant when the wedge portion 305W of the upper tool 305 is inserted between the upper tool support portion 313 and the upper tool clamp member 309, the compression rate of the elastic member 329 is always kept constant, as far as the upper tool 305 of the same dimension is inserted.

[0187] In other words, since the upper tool clamp member 309 is urged by an elastic force (the reaction force) of the elastic member 329, the wedge portion 305W of the upper tool 305 can be clamped between the upper tool clamp member 309 and the upper tool support portion 313. In this case, since being decided

by the elastic force of only the elastic member 329, the clamping force is kept constant, as far as the upper tool 305 of the same wedge portion in dimension are clamped. Accordingly, it is possible to prevent the lower end portion 305E of the upper tool 305 from being dis-

located slightly due to a difference in the clamping force of the upper tool clamp member 309, so that the lower end portion 305E of the upper tool 305 can be held always at a constant position.

[0188] In other words, even if the upper tools 305 are exchanged, as far as the same upper tools are used, it is possible to locate the lower end portion 305E of the upper tool 305 always at a constant position, so that it is possible to reproduce the accurate alignment of the upper tool with the die.

[0189] Therefore, work can be bent precisely in a plu-

rality of steps by changing the upper tools. In the upper tool holder apparatus according to the present inven-

tion, since the reproducibility of the exchanged upper tools of the same dimensions in the wedge portion is excellent, as compared with the conventional upper tool holder apparatus, it is possible to improve the bending precision.

[0190] Further, when the upper tool 305 is required to be removed from the upper tool holder apparatus 301, the bolt 327 is slightly loosened by use of the tool 333. In this case, the loosening rate of the bolt 327 is small, because the elastic force of the elastic member 329 is reduced to such an extent that the upper tool 305 can be moved. Therefore, it is possible to exchange the upper tool 305 easily by use of the upper tool holder apparatus 301.

[0191] Further, when the fastening stroke of the bolt 327 is required to adjust, as shown in Fig. 21, an annular spacer 339A is interposed between the upper tool support portion 313 and the body portion 237B of the bolt 327.

[0192] Further, when the elastic force of the elastic member 329 is required to be adjusted, another spacer 339B is interposed between the upper tool clamp member 309 and the elastic member 329 or between the elastic member 329 and the head portion 327H of the bolt 327.

[0193] Fig. 22 shows a modification of the apparatus of Figures 19 and 20. In this modification, the head portion of the bolt 327 shown in Fig. 20 is replaced with a push member 341, a lever 347 and a cam member 345. In more detail, the annular push member 341 is fitted to the body portion 327B of the bolt 327 so as to be brought into contact with the elastic member 329. The cam member 345 is pivotally supported by the bolt 327 via a pin 343. The lever 347 is attached to the cam member 345.

[0194] Therefore, after the bolt 327 has been fastened to the upper tool support portion 313, when the lever 347 attached to the cam member 345 is pivoted vertically, for instance as shown in Fig. 22, the push member 341 can be shifted by the cam member 345 by a predetermined distance, so that a space between the upper

tool clamp member 309 and the annular push member 341 can be determined at a previously determined constant value to obtain a constant clamping force of the elastic means 329.

[0195] When the upper tool 305 is required to be released from the clamping force, the lever 347 is pivoted horizontally, for instance. Then, since the annular push member 341 is released via the cam member 345, the clamping force of the elastic member 239 can be released.

[0196] In this modification, the effect is the same as that of the third embodiment. Further, the clamp and unclamp conditions of the upper tool holder apparatus 301 can be confirmed easily by seeing whether the lever 347 is pivoted vertically or horizontally.

[0197] Fig. 23 shows still another modification of the apparatus of Figures 19 and 20. In this modification, the bolt 327 is passed through the upper tool support portion 313, and the elastic member 329 is interposed between a nut member (fastening member) 349 thread-engaged with the end portion of the bolt 327 and the upper tool support portion 313, without use of the elastic member interposed between the head portion 327H of the bolt 327 and the upper tool clamp member 309. In this modification, the upper tool 305 can be clamped at a constant clamping force, and thereby the same effect as described above can be obtained.

[0198] In the modification shown in Fig. 24, an upper tool clamp member 351 is formed with a hole 351H, and a semi-spherical clamp piece 353 fixed to a piston slider 355 is fitted to the hole 351. Further, a nitrogen gas is introduced into the hole 351H. The clamping force of the elastic means is adjusted by the pressure of the nitrogen gas introduced into the hole 351H. In this modification, the same effect can be obtained.

[0199] As described above, the upper tool can be exchanged easily by use of the upper tool holder apparatus. Further, whenever the upper tools of the same upper wedge dimension are exchanged, since the clamping force of the upper tool can be kept at a constant value, it is possible to obtain an excellent reproducibility of the alignment between the upper tool and the die, even after the upper tools are exchanged for various bending process. As a result, the upper tool exchange work can be facilitated and further the bending precision can be improved.

Claims

1. An upper tool for use with a press brake provided with an holding device (1), preferably such as the upper tool holder apparatus defined in claim 14 or 24, said holding device (1) including an upper tool holder body (5) provided with a support plate (7) and an upper tool clamp member (11), said upper tool (9) being clamped to said holder body (5) mounted on an upper table (3) by a clamping force of said

upper tool clamp member (11), said upper tool (9) comprising

- an inclined surface (9S), appropriate to co-operate, in use, with said upper tool clamp member (11) so that said upper tool (9) may execute relative movements with respect to said upper tool clamp member (11) thereby generating a gradually increasing clamping force; 5
- a vertical sliding surface (9V) appropriate for sliding, in use, on a front or a rear surface of said support plate (7); 10
- an upper tool portion (9U) having a contact surface (9F) which is capable of contacting, in use, with a lower end surface (7E) of said support plate (7); and 15
- a work processing portion (9M) for processing a work piece (W) in co-operation with a lower tool (63), 20

characterised in that said upper tool (9) further comprises an engage groove (9G) on the same side and beneath said inclined surface (9S), which is appropriate to be engaged, in use, with an engage projection (11K) formed in said upper tool clamp member (11), said contact surface (9F) being arranged to be lower than a lower surface of said engage groove (9G). 25 30

2. An upper tool for a press brake according to claim 1, **characterised in that** said inclined surface (9S) is inclined with an inclination angle of 5~20° to a vertical direction. 35
3. An upper tool for a press brake according to claim 1, **characterised in that** said inclined surface is inclined with an inclination angle of 7~11° to a vertical direction. 40
4. An upper tool for a press brake according to claim 1, **characterised in that** an upper surface of said engage groove (9G) is formed to be inclined to a direction perpendicular to said vertical direction. 45
5. An upper tool for a press brake according to claim 1, **characterised in that** a vertical surface (9SV) is formed between said inclined surface (9S) and said engage groove (9G). 50
6. An upper tool for a press brake according to claim 1, **characterised in that** an upper surface (9GF) of the engage groove (9G) is slightly inclined downward from a horizontal line inward to urge the upper tool against the support plate (7). 55

7. An upper tool for a press brake according to claim 1, **characterised in that** said inclined surface (9S) of the upper tool is hardened by attaching a hard metal on the surface thereof.

8. An upper tool for a press brake according to claim 1, **characterised in that** said inclined surface (9S) of the upper tool is hardened by coating a hard metal on the surface thereof.

9. An upper tool for a press brake according to claim 1 in combination with said holding device (1), **characterised in that** a dimension of said inclined surface (9S) in a vertical direction when an upper projection portion of said upper tool engages said upper tool clamp member (11) is more than a gap (H) between a lower end surface (7E) of said holder body (5) and a shoulder portion (9F) of said upper tool (9).

10. An upper tool for a press brake according to claim 1, in combination with said holding device (1) **characterised in that** said upper tool (9) is detachably mounted between said support plate (7) and said upper tool clamp member (11), said support plate (7) being arranged on a lower part of a holder body (5) in an upper tool holder (1) attached on an underside of said upper table (3) of said press brake, said upper tool clamp member (11) being pivotally mounted to said holder body (5) and being constructed to be capable of pressing said upper tool (9) against said support plate (7),

said inclined surface (9S) making said upper tool clamping member (11) pivoting to increase a clamp force of said upper tool (9) due to said upper tool clamping member (11) in case of rising said upper tool (9) relatively with respect to said support plate (7) to contact said contact surface (9F) with said lower end surface of said support plate (107) wherein, in a condition under which an upper surface of said engaging groove (9G) provided in said upper tool (9) is supported by an engaging projection (11K) provided in said upper-tool clamping member (11), a distance H between said lower end surface (7E) of said support plate (7) and said contact surface (9F) of said upper tool (9) can be calculated by the following expression :

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

where A denotes a distance between a pivotal centre of said upper tool clamp member (11) and elastic means for generating a clamping force to said upper tool clamp member ;

B denotes an average distance between said pivotal centre of said upper tool clamp member

and a clamping force application point ;

P denotes said clamping force applied to said upper tool clamp member ;

K denotes an elastic coefficient of said elastic means ;

and θ denotes the inclination angle of the inclined surface (9S) of said upper tool from said vertical line.

11. An upper tool for a press brake according to claim 1 in combination with said holding device (1), said upper tool (9) is elastically clamped between said support plate (7) of said holder body (5) attached to a lower portion of said upper table (3) and said upper tool clamp member (11), **characterised in that** said inclined surface (9S) is brought into contact with an inclined surface of said upper tool clamp member (11) when said upper tool is clamped between said upper tool holder body (5) and said upper tool clamp member (11) in such a way that clamping force can be further increased when said upper tool is engaged with a die (63).

12. An upper tool and holding device (1) according to claim 11, **characterised in that** a lower surface (9GL) of said engage groove (9G) of said upper tool is formed at a position vertically higher than a contact surface (9F) brought into contact with a lower end surface (7E) of said support plate (7) of said holder body (5).

13. An upper tool and holding device (1) according to claim 11, **characterised in that** a distance H between said lower surface (7E) of said support plate (7) and an upper surface (7E) of said support plate (7) and an upper surface (9F) of said upper tool (9) obtained when said upper surface of said engage groove (9G) of said upper tool is in contact with said engage projection (11K) of said upper tool clamp member (11) is determined as follows :

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

where A denotes a distance between a pivotal centre of said upper tool clamp member (11) and elastic means for generating a clamping force to said upper tool clamp member ;

B denotes an average distance between said pivotal centre of said upper tool clamp member and a clamping force application point ;

P denotes said clamping force applied to said upper tool clamp member ;

K denotes an elastic coefficient of said elastic means ;

and θ denotes the inclination angle of the inclined surface (9S) of said upper tool from said vertical line.

14. An upper tool assembly for a press brake, comprising an upper tool according to any one of claims 1 to 8 and an upper tool holder apparatus, for clamping an upper tool(205) between an upper tool support portion (209) of an upper tool holder body (207) attached to a lower portion of an upper table (203) and a pivotal upper tool clamp member (211) by a clamping force generating means (213), **characterised in that**:

said pivotal upper tool clamp member (211) is formed with an engage projection (211P) engaged with a drop prevention groove (205G) formed in said upper tool (205) ; and

said upper tool holder apparatus further comprises clamp operating means (215) for selectively move to a clamp position A at which said clamping force can be applied to said upper tool clamp member (211) from said clamping force generating means (213) to clamp said upper tool tightly, an unclamp position B at which said clamping force can be reduced to allow said upper tool to be adjustably shifted horizontally, and an exchange position C at which said upper tool clamp member (211) can be moved away from said upper tool to allow said upper tool to be exchanged vertically with another one.

15. An upper tool assembly for a press brake according to claim 14, **characterised in that** said clamp operating means (215) comprises :

a pusher member (259P) for pushing said clamping force generating means (213) to generate said clamping force of said clamping force generating means (213) ;

an operation lever (265) ; and

a clamping force control means (263) attached to said operation lever, controllably brought into contact with and separated from said pusher member (259P) to control said clamping force applied from said clamping force generating means (213) to said upper tool.

16. An upper tool assembly for a press brake according to claim 14, **characterised in that** said upper tool holder apparatus further comprises upper tool clamp member urging means (217) for urging said

upper tool clamp member (211) slightly in clamp direction when said clamp operating means (215) is moved to said exchange position C.

17. An upper tool assembly for a press brake according to claim 16, **characterised in that** said upper tool clamp member urging means (217) is at least one coil spring disposed between said upper tool clamp member (211) and said upper tool holder body (207).

18. An upper tool assembly for a press brake according to claim 15, **characterised in that** said clamping force control means attached to said operation lever (265) is a pusher screw (263) fixed to said operation lever and thread-engaged with said upper tool clamp member (211), said pusher screw being shifted relative to said upper tool clamp member (211) when rotated by said operation lever.

19. An upper tool assembly for a press brake according to claim 14, **characterised in that** said upper tool holder apparatus further comprises clamp release holding means (219) for holding said upper tool clamp member (211) in a clamping force released state, to form an upper tool exchange space between said upper tool support portion (209), under condition that said clamp operating means (215) is at said exchange position C.

20. An upper tool assembly for a press brake according to claim 19, **characterised in that** said clamp release holding means (219) comprises :

an engage recess (279) formed in said upper tool clamp member (211) ; and

a plunger (218) attached to said holder body (207) so as to be engaged with said engage recess, when said upper tool clamp member (211) is pivoted away from said upper tool support portion (209).

21. An upper tool assembly for a press brake according to claim 15, **characterised in that** said upper tool holder apparatus further comprises two clamp operation lever stopping means (271) for stopping said moved clamp operation lever (265) at said unclamp position B and said exchange position C, respectively.

22. An upper tool assembly for a press brake according to claim 21, **characterised in that** each of said clamp operation lever stopping means (217) comprises :

a pin (275) attached to a side surface of a slot (273) formed in a mounting plate (225) ;

a stopper member (271) pivotally supported by said pin so as to be projected or retracted from a surface of said mounting plate ; and

a plunger (277) attached to said mounting plate to hold said stopper at said projected or retracted position, respectively.

23. An upper tool assembly according to claim 14, **characterised in that** said upper tool holder apparatus further comprises :

a wedge piece (247) attached above said engage projection (211P) formed in said upper tool clamp member (211) so as to be engaged with said drop prevention groove (205G) formed in said upper tool (205) ; and

a coil spring (249) for urging said wedge piece in a direction that said clamped upper tool can be urged in clamping direction.

24. An upper tool assembly for a press brake, comprising an upper tool according to any one of claims 1 to 8 and an upper tool holder apparatus for clamping said upper tool (305) between an upper tool support portion (313) of an upper tool body (301) attached to a lower portion of an upper table (303) and an upper tool clamp member (309) by a fastening member (327), **characterised in that** the upper tool (305) is clamped by a fastening member (327) co-operating with an elastic member (329) for elastically clamping said upper tool between said upper tool support portion (313) and said upper tool clamp member (309), said elastic member (329) being interposed between said upper tool clamp member (309) and said fastening member (327).

25. An upper tool assembly for a press brake according to claim 24, **characterised in that** said upper tool clamp member (309) is formed with an inclined surface (309F) widened downward at an upper portion thereof and brought into contact with an inclined surface (305F) of an upper tool (305) when said upper tool is clamped between said upper tool support portion (313) and said upper tool clamp member (309) in such a way that said clamping force can be further increased when said upper tool is engaged with a die (63)

26. An upper tool assembly for a press brake according to claim 24, **characterised in that**, said fastening member (327) is composed of a threaded portion (327S) thread-engaged with said upper tool holder body (313) and a cylindrical body portion (327B) for determining a constant space for disposing said upper tool clamp member (309) and said elastic member (329).

27. An upper tool assembly for a press brake according to claim 24, **characterised in that** said upper tool clamp member (309) is formed with an engage projection (309P) engaged with an engaged groove (305F) formed in said upper tool (305).

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28. An upper tool assembly for a press brake according to claim 24, **characterised in that** right and left side edges (309C) of said upper tool clamp member (309) are chamfered to facilitate the insertion of said upper tool between said upper tool support portion (313) and said upper tool clamp member (309).

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29. A method of clamping an upper tool (305) with an upper tool assembly according to claim 24, between an upper tool holder body (307) attached to an upper table (303) of a press brake and an upper tool clamp member (309) by an elastic clamping force of an elastic member, which comprises the step of :

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fastening a fastening member (327) for urging said upper tool clamp member (309) formed with an inclined surface (309F) toward said upper tool holder body (307) at a constant fastening position relative to said upper tool holder body (307);

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inserting said upper tool (305) formed with an inclined surface (305F) into the predetermined space from below in such a way that the upper tool inclined surface (305F) is brought into contact with said clamp member inclined surface (309F) ; and

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engaging said upper tool (305) with a die (337) by moving any one of said upper tool and die to move said upper tool upward relative to said upper tool holder body (307) and thereby to compress an elastic member (329) interposed between said fastening member (327) and said upper tool clamp member (309) always at constant compression rate due to a wedge effect between said two inclined surfaces of said upper tool (305) and said upper tool clamp member (309), whereby said upper tool (305) can be clamped between said upper tool holder body (307) and said upper tool clamp member (309) always by a constant clamping force.

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Patentansprüche

1. Oberwerkzeug zum Einsatz mit einer Biegepresse, die mit einer Haltevorrichtung (1), vorzugsweise wie die Oberwerkzeug-Haltevorrichtung, die in Anspruch 14 oder 24 definiert ist, versehen ist, wobei die Haltevorrichtung (1) einen Oberwerkzeug-Halterkörper (5), der mit einer Stützplatte (7) versehen

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ist, und ein Oberwerkzeug-Klemmelement (11) enthält, wobei das Oberwerkzeug an dem Halterkörper (5), der an einem Obertisch (3) angebracht ist, durch eine Klemmkraft des Oberwerkzeug-Klemmelementes (11) festgeklemmt wird, wobei das Oberwerkzeug (9) umfasst:

- eine geneigte Fläche (9S), die dazu geeignet ist, in Funktion mit dem Oberwerkzeug-Klemmelement (11) so zusammenzuwirken, dass das Oberwerkzeug (9) relative Bewegungen in Bezug auf das Oberwerkzeug-Klemmelement (11) ausführen kann, um so eine allmählich zunehmende Klemmkraft zu erzeugen;
- eine vertikale Gleitfläche (9V), die geeignet ist, um in Funktion an einer vorderen oder einer hinteren Fläche der Stützplatte (7) zu gleiten;
- einen Oberwerkzeug-Abschnitt (9U), der eine Kontaktfläche (9F) aufweist, die in Funktion mit einer unteren Endfläche (7E) der Stützplatte (7) in Kontakt kommen kann; und
- einen Erzeugnisbearbeitungs-Abschnitt (9M) zum Bearbeiten eines Werkstücks (W) im Zusammenwirken mit einem Unterwerkzeug (63),

dadurch gekennzeichnet, dass das Oberwerkzeug (9) des Weiteren eine Eingriffsnut (9G) auf der gleichen Seite wie die geneigte Fläche (9S) und unter ihr umfasst, die dazu geeignet ist, in Funktion mit einem Eingriffsvorsprung (11K) in Eingriff zu kommen, der an dem Oberwerkzeug-Klemmelement (11) ausgebildet ist, wobei die Kontaktfläche (9F) so angeordnet ist, dass sie niedriger ist als eine untere Fläche der Eingriffsnut (9G).

2. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** die geneigte Fläche (9S) mit einem Neigungswinkel von 5-20° zu einer vertikalen Richtung geneigt ist.

3. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** die geneigte Fläche mit einem Neigungswinkel von 7-11° zu einer vertikalen Richtung geneigt ist.

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4. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** eine obere Fläche der Eingriffsnut (9G) so ausgebildet ist, dass sie in einer Richtung senkrecht zu der vertikalen Richtung geneigt ist.

5. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** eine vertikale Fläche (9SV) zwischen der geneigten Fläche (9S) und der Eingriffsnut (9G) ausgebildet ist.

6. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** eine obere Fläche (9GF) der Eingriffsnut (9G) leicht nach unten von einer horizontalen Linie nach innen geneigt ist, um das Oberwerkzeug an die Stützplatte (7) zu drücken. 5
7. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** die geneigte Fläche (9S) des Oberwerkzeugs gehärtet wird, indem ein Hartmetall an der Oberfläche derselben angebracht wird. 10
8. Oberwerkzeug für eine Biegepresse nach Anspruch 1, **dadurch gekennzeichnet, dass** die geneigte Fläche (9S) des Oberwerkzeugs gehärtet wird, indem ein Hartmetall auf die Oberfläche derselben aufgetragen wird. 15
9. Oberwerkzeug für eine Biegepresse nach Anspruch 1 in Kombination mit der Haltevorrichtung (1), **dadurch gekennzeichnet, dass** eine Abmessung der geneigten Fläche (9S) in einer vertikalen Richtung, wenn ein oberer Vorsprungsabschnitt des Oberwerkzeugs mit dem Oberwerkzeug-Klemmelement (11) in Eingriff kommt, größer ist als ein Zwischenraum (H) zwischen einer unteren Endfläche (7E) des Halterkörpers (5) und einem Schulterabschnitt (9F) des Oberwerkzeugs (9). 20 25
10. Oberwerkzeug für eine Biegepresse nach Anspruch 1 in Kombination mit der Haltevorrichtung (1), **dadurch gekennzeichnet, dass** das Oberwerkzeug (9) abnehmbar zwischen der Stützplatte (7) und dem Oberwerkzeug-Klemmelement (11) angebracht ist, wobei die Stützplatte (7) an einem unteren Teil eines Halterkörpers (5) in einem Oberwerkzeug-Halter (1) angeordnet ist, der an einer Unterseite des Obertischs (3) der Biegepresse angebracht ist, wobei das Oberwerkzeug-Klemmelement (11) schwenkbar an dem Halterkörper (5) angebracht und so aufgebaut ist, dass es in der Lage ist, das Oberwerkzeug (9) an die Stützplatte (7) zu pressen, 30 35 40 45 50 55
- die geneigte Fläche (9S) bewirkt, dass das Oberwerkzeug-Klemmelement (11) geschwenkt wird, um eine Klemmkraft des Oberwerkzeugs (9) aufgrund des Oberwerkzeug-Klemmelementes (11) zu vergrößern, wenn das Oberwerkzeug (9) in Bezug auf die Klemmplatte (7) angehoben wird, um die Kontaktfläche (9F) mit der unteren Endfläche der Stützplatte (107) in Kontakt zu bringen, wobei in einem Zustand, in dem eine obere Fläche der Eingriffsnut (9G), die in dem Oberwerkzeug (9) vorhanden ist, von einem Eingriffsvorsprung (11K) getragen wird, der in dem Oberwerkzeug-Klemmelement (11) vorhanden ist, ein Abstand H zwischen der unteren Endfläche (7E) der Trägerplatte (7) und der Kontaktfläche (9F) des Oberwerkzeugs (9) mit dem folgenden Ausdruck errechnet werden kann:
- $$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$
- wobei A einen Abstand zwischen einem Schwenkmittelpunkt des Oberwerkzeug-Klemmelementes (11) und einer elastischen Einrichtung zum Ausüben einer Klemmkraft auf das Oberwerkzeug-Klemmelement bezeichnet;
- B einen durchschnittlichen Abstand zwischen dem Schwenkmittelpunkt des Oberwerkzeug-Klemmelementes und einem Klemmkraft-Wirkpunkt bezeichnet;
- P die auf das Oberwerkzeug-Klemmelement wirkende Klemmkraft bezeichnet;
- K einen Elastizitätsmodul der elastischen Einrichtung bezeichnet;
- und θ den Neigungswinkel der geneigten Fläche (9S) des Oberwerkzeugs gegenüber der vertikalen Linie bezeichnet.
11. Oberwerkzeug für eine Biegepresse nach Anspruch 1 in Kombination mit der Haltevorrichtung (1), wobei das Oberwerkzeug (9) elastisch zwischen der Stützplatte (7) des Halterkörpers (5), der an einem unteren Abschnitts des Obertischs (3) angebracht ist, und dem Oberwerkzeug-Klemmelement (11) eingeklemmt wird, **dadurch gekennzeichnet, dass** die geneigte Fläche (9S) in Kontakt mit einer geneigten Fläche des Oberwerkzeug-Klemmelementes (11) gebracht wird, wenn das Oberwerkzeug zwischen dem Oberwerkzeug-Halterkörper (5) und dem Oberwerkzeug-Klemmelement (11) so eingeklemmt wird, dass die Klemmkraft weiter vergrößert werden kann, wenn das Oberwerkzeug mit einem Gesenk (63) in Eingriff gebracht wird. 30 35 40 45 50 55
12. Oberwerkzeug und Haltevorrichtung (1) nach Anspruch 11, **dadurch gekennzeichnet, dass** eine untere Fläche (9GL) der Eingriffsnut (9G) des Oberwerkzeugs an einer Position ausgebildet ist, die vertikal höher liegt als eine Kontaktfläche (9F), die mit einer unteren Endfläche (7E) der Stützplatte (7) des Halterkörpers (5) in Kontakt gebracht wird.
13. Oberwerkzeug und Haltevorrichtung (1) nach Anspruch 11, **dadurch gekennzeichnet, dass** ein Abstand (H) zwischen der unteren Fläche (7E) der Stützplatte (7) und einer oberen Fläche (7E) der Stützplatte (7) sowie einer oberen Fläche (9F) des

Oberwerkzeugs (9), der entsteht, wenn die obere Fläche der Eingriffsnut (9G) des Oberwerkzeugs mit dem Eingriffsvorsprung (11K) des Oberwerkzeug-Klemmelementes (11) in Kontakt ist, wie folgt bestimmt wird:

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

wobei A einen Abstand zwischen einem Schwenkmittelpunkt des Oberwerkzeug-Klemmelementes (11) und einer elastischen Einrichtung zum Ausüben einer Klemmkraft auf das Oberwerkzeug-Klemmelement bezeichnet;

B einen durchschnittlichen Abstand zwischen dem Schwenkmittelpunkt des Oberwerkzeug-Klemmelementes und einem Klemmkraft-Wirkpunkt bezeichnet;

P die auf das Oberwerkzeug-Klemmelement wirkende Klemmkraft bezeichnet;

K ein Elastizitätsmodul der elastischen Einrichtung bezeichnet;

und θ den Neigungswinkel der geneigten Flächen (9S) des Oberwerkzeugs gegenüber der vertikalen Linie bezeichnet.

14. Oberwerkzeug-Anordnung für eine Biegepresse, die ein Oberwerkzeug nach einem der Ansprüche 1 bis 8 und eine Oberwerkzeug-Haltervorrichtung zum Festklemmen eines Oberwerkzeugs (205) zwischen einem Oberwerkzeug-Stützabschnitt (209) eines Oberwerkzeug-Halterkörpers (207), der an einem unteren Abschnitt eines Obertisches (203) angebracht ist, und einem schwenkbaren Oberwerkzeug-Klemmelement (211) mit einer Klemmkraft-Erzeugungseinrichtung (213) umfasst, **dadurch gekennzeichnet, dass:**

das schwenkbare Oberwerkzeug-Klemmelement (211) mit einem Eingriffsvorsprung (211P) versehen ist, der mit einer Fallverhinderungs-Nut (205G) in Eingriff ist, die in dem Oberwerkzeug (205) ausgebildet ist; und

die Oberwerkzeug-Haltervorrichtung des Weiteren eine Klemmbetätigungs-Einrichtung (215) zum wahlweisen Bewegen an eine Klemmposition A, in der die Klemmkraft von der Klemmkraft-Erzeugungseinrichtung (213) auf das Oberwerkzeug-Klemmelement (211) ausgeübt werden kann, um das Oberwerkzeug festzuklemmen, eine Losklemm-Position B, in der die Klemmkraft verringert werden kann, um das Oberwerkzeug verstellbar horizontal zu

verschieben und eine Austauschposition C in der das Oberwerkzeug-Klemmelement (211) von dem Oberwerkzeug weg bewegt werden kann, so dass das Oberwerkzeug vertikal gegen ein anderes ausgetauscht werden kann, umfasst.

15. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 14, **dadurch gekennzeichnet, dass** die Klemmbetätigungs-Einrichtung (215) umfasst:

ein Schiebeelement (259P), das die Klemmkraft-Erzeugungseinrichtung (213) schiebt, um die Klemmkraft der Klemmkraft-Erzeugungseinrichtung (213) zu erzeugen;

einen Betätigungshebel (265); und

eine Klemmkraft-Steuereinrichtung (263), die an dem Betätigungshebel angebracht ist und gesteuert mit dem Schiebeelement (259P) in Kontakt gebracht und von ihm getrennt wird, um die von der Klemmkraft-Erzeugungseinrichtung (213) auf das Oberwerkzeug ausgeübte Klemmkraft zu steuern.

16. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 14, **dadurch gekennzeichnet, dass** die Oberwerkzeug-Haltervorrichtung des Weiteren eine Oberwerkzeug-Klemmelement-Drückeinrichtung (217) umfasst, die das Oberwerkzeug-Klemmelement (211) leicht in Klemmrichtung drückt, wenn die Klemmbetätigungs-Einrichtung (215) an die Austauschposition C bewegt wird.

17. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 16, **dadurch gekennzeichnet, dass** die Oberwerkzeug-Klemmelement-Drückeinrichtung (217) wenigstens eine Schraubenfeder ist, die zwischen dem Oberwerkzeug-Klemmelement (211) und dem Oberwerkzeug-Halterkörper (207) angeordnet ist.

18. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 15, **dadurch gekennzeichnet, dass** die Klemmkraft-Steuereinrichtung, die an dem Betätigungshebel (265) angebracht ist, eine Druckschraube (263) ist, die an dem Betätigungshebel befestigt und in Gewindeeingriff mit dem Oberwerkzeug-Klemmelement (211) ist, wobei die Druckschraube in Bezug auf das Oberwerkzeug-Klemmelement (211) verschoben wird, wenn sie durch den Betätigungshebel gedreht wird.

19. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 14, **dadurch gekennzeichnet, dass** die Oberwerkzeug-Haltervorrichtung des Wei-

teren eine Klemmlöse-Halteeinrichtung (219) umfasst, die das Oberwerkzeug-Klemmelement (211) in einem Klemmkraft-Lösezustand hält, um einen Oberwerkzeug-Austauschraum zwischen dem Oberwerkzeug-Stützabschnitt (209) in dem Zustand auszubilden, in dem sich die Klemmbetätigungs-Einrichtung (215) in der Austauschposition C befindet.

20. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 19, **dadurch gekennzeichnet, dass** die Klemmlöse-Halteeinrichtung (219) umfasst:

eine Eingriffsaussparung (279), die in dem Oberwerkzeug-Klemmelement (211) ausgebildet ist; und

einen Kolben (218), der an dem Halterkörper (207) angebracht ist und mit der Eingriffsaussparung in Eingriff kommt, wenn das Oberwerkzeug-Klemmelement (211) von dem Oberwerkzeug-Stützabschnitt (209) weg geschwenkt wird.

21. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 15, **dadurch gekennzeichnet, dass** die Oberwerkzeug-Haltervorrichtung des Weiteren zwei Klemmbetätigungs-Hebel-Stoppeinrichtungen (271) umfasst, die den bewegten Klemmbetätigungs-Hebel (265) in der Losklemm-Position B bzw. der Austauschposition C stoppen.

22. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 21, **dadurch gekennzeichnet, dass** jede der Klemmbetätigungs-Hebel-Stoppeinrichtungen (217) umfasst:

einen Zapfen (275), der an einer Seitenfläche eines Schlitzes (273) angebracht ist, der in einer Anbringungsplatte (225) ausgebildet ist;

ein Anschlagelement (271), das schwenkbar von dem Zapfen so getragen wird, dass es von einer Fläche der Anbringungsplatte aus vorsteht oder eingezogen ist; und

einen Kolben (277), der an der Anbringungsplatte angebracht ist, um den Anschlag in der vorstehenden bzw. eingezogenen Position zu halten.

23. Oberwerkzeug-Anordnung nach Anspruch 14, **dadurch gekennzeichnet, dass** die Oberwerkzeug-Haltervorrichtung des Weiteren umfasst:

ein Keilstück (247), das über dem Eingriffsvorsprung (211P) angebracht ist, der in dem Ober-

werkzeug-Klemmelement (211) ausgebildet ist, um mit der Fallverhinderungs-Nut (205G) in Eingriff zu kommen, die in dem Oberwerkzeug (205) ausgebildet ist; und

eine Schraubenfeder (249), die das Keilstück in einer Richtung drückt, in der das festgeklemmte Oberwerkzeug in einer Klemmrichtung gedrückt werden kann.

24. Oberwerkzeug-Anordnung für eine Biegepresse, die ein Oberwerkzeug nach einem der Ansprüche 1 bis 8 und eine Oberwerkzeug-Haltervorrichtung zum Einklemmen des Oberwerkzeugs (305) zwischen einem Oberwerkzeug-Stützabschnitt (313) eines Oberwerkzeug-Körpers (301), der an einem unteren Abschnitt eines Obertisches (303) angebracht ist, und einem Oberwerkzeug-Klemmelement (309) mit einem Befestigungselement (327) umfasst, **dadurch gekennzeichnet, dass** das Oberwerkzeug (305) mit einem Befestigungselement (327) eingeklemmt wird, das mit einem elastischen Element (329) zusammenwirkt, um das Oberwerkzeug zwischen dem Oberwerkzeug-Stützabschnitt (313) und dem Oberwerkzeug-Klemmelement (309) elastisch einzuklemmen, wobei das elastische Element (329) zwischen dem Oberwerkzeug-Klemmelement (309) und dem Befestigungselement (327) angeordnet ist.

25. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 24, **dadurch gekennzeichnet, dass** das Oberwerkzeug-Klemmelement (309) mit einer geneigten Fläche (309F) versehen ist, die an einem oberen Abschnitt derselben nach unten verbreitert ist und mit einer geneigten Fläche (305F) eines Oberwerkzeugs (305) in Kontakt gebracht wird, wenn das Oberwerkzeug zwischen dem Oberwerkzeug-Stützabschnitt (313) und dem Oberwerkzeug-Klemmelement (309) so eingeklemmt wird, dass die Klemmkraft weiter vergrößert werden kann, wenn das Oberwerkzeug mit einem Gesenk (363) in Eingriff gebracht wird.

26. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 24, **dadurch gekennzeichnet, dass** die Befestigungseinrichtung (327) aus einem Gewindeabschnitt (327S), der in Gewindeeingriff mit dem Oberwerkzeug-Halterkörper (313) ist, und einem zylindrischen Körperabschnitt (327B) besteht, der einen konstanten Raum zum Anordnen des Oberwerkzeug-Klemmelementes (309) und des elastischen Elementes (329) bestimmt.

27. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 24, **dadurch gekennzeichnet, dass** das Oberwerkzeug-Klemmelement (309) mit einem Eingriffsvorsprung (309P) versehen ist, der

mit einer Eingriffsnut (305F) in Eingriff kommt, die in dem Oberwerkzeug (305) ausgebildet ist.

28. Oberwerkzeug-Anordnung für eine Biegepresse nach Anspruch 24, **dadurch gekennzeichnet, dass** die rechte und die linke Seitenkante (309C) des Oberwerkzeug-Klemmelementes (309) abge- 5
schrägt sind, um das Einführen des Oberwerkzeugs zwischen den Oberwerkzeug-Stützabschnitt (313) und das Oberwerkzeug-Klemmelement (309) zu er- 10
leichtern.

29. Verfahren zum Festklemmen eines Oberwerkzeugs (305) mit einer Oberwerkzeug-Anordnung nach An- 15
spruch 24 zwischen einem Oberwerkzeug-Halter-
körper (307), der an einem Obertisch (303) einer
Biegepresse angebracht ist, und einem Oberwerk-
zeug-Klemmelement (309) durch eine elastische
Klemmkraft eines elastischen Elementes, das die 20
folgenden Schritte umfasst:

Befestigen eines Befestigungselementes (327), um das Oberwerkzeug-Klemmelement (309), das mit einer geneigten Fläche (309F) versehen ist, an einer konstanten Befesti- 25
gungsposition in Bezug auf den Oberwerk-
zeug-Halterkörper (307) auf den Oberwerk-
zeug-Halterkörper (307) zu drücken;

Einführen des Oberwerkzeugs (305), das mit einer geneigten Fläche (305F) versehen ist, in den vorgegebenen Raum von unten, und zwar 30
so, dass die geneigte Fläche (305F) des Ober-
werkzeugs mit der geneigten Fläche (309F)
des Klemmelementes in Kontakt gebracht wird; 35
und

Herstellen von Eingriff des Oberwerkzeugs (305) mit einem Gesenk (327), indem entweder das Oberwerkzeug oder das Gesenk so be- 40
wegt werden, dass das Oberwerkzeug in Be-
zug auf den Oberwerkzeug-Halterkörper (307)
nach oben bewegt wird, um so ein elastisches
Element (329), das sich zwischen dem Befesti-
gungselement (327) und dem Oberwerkzeug- 45
Klemmelement (309) befindet, aufgrund einer
Keilwirkung zwischen den zwei geneigten Flä-
chen des Oberwerkzeugs (305) und des Ober-
werkzeug-Klemmelementes (309) stets mit ei-
nem konstanten Zusammendrückverhältnis 50
zusammenzudrücken, so dass das Oberwerk-
zeug (305) zwischen dem Oberwerkzeug-Hal-
terkörper (307) und dem Oberwerkzeug-Klem-
melement (309) stets mit einer konstanten
Klemmkraft eingeklemmt werden kann. 55

Revendications

1. Outil supérieur pour utilisation avec une presse plieuse pourvue d'un dispositif de fixation (1), de 5
préférence tel que l'appareil de support d'outil su-
périeur selon la revendication 14 ou 24, ledit dispo-
sitif de fixation (1) comprenant un corps de support
d'outil supérieur (5) pourvu d'une plaque de support
(7) et d'un élément de serrage d'outil supérieur (11),
ledit outil supérieur (9) étant serré sur ledit corps de
support (5) monté sur une table supérieure (3) par
une force de serrage dudit élément de serrage
d'outil supérieur (11), ledit outil supérieur (9) com-
portant

- une surface inclinée (9S), prévue pour coopé-
rer, en service, avec ledit élément de serrage
d'outil supérieur (11) de telle sorte que ledit outil
supérieur (9) peut exécuter des mouvements
relatifs par rapport audit élément de serrage
d'outil supérieur (11) en générant ainsi une for-
ce de serrage progressivement croissante;
- une surface de glissement verticale (9V) pré-
vue pou glisser, en service, sur une surface
avant ou arrière de ladite plaque de support (7);
- une partie d'outil supérieur (9U) ayant une sur-
face de contact (9F) qui est capable d'entrer en
contact, en service, avec une surface d'extré-
mité inférieure (7E) de ladite plaque de support
(7); et
- une partie de traitement de pièce (9M) destinée
à traiter une pièce (W) en coopération avec un
outil inférieur (63),

caractérisé en ce que ledit outil supérieur (9) com-
porte en outre une rainure d'engagement (9G) du
même côté et sous ladite surface inclinée (9S), qui
est prévue pour être engagée, en service, avec une
saillie d'engagement (11K) formée dans ledit élé-
ment de serrage d'outil supérieur (11), ladite sur-
face de contact (9F) étant prévue pour être plus basse
qu'une surface inférieure de ladite rainure d'enga-
gement (9G).

2. Outil supérieur pour une presse plieuse selon la re-
vendication 1, **caractérisé en ce que** ladite surface
inclinée (9S) est inclinée avec un angle d'inclinaison
de 5 à 20° par rapport à une direction verticale
3. Outil supérieur pour une presse plieuse selon la re-
vendication 1, **caractérisé en ce que** ladite surface
inclinée est inclinée avec un angle d'inclinaison de
7 à 11° par rapport à une direction verticale.
4. Outil supérieur pour une presse plieuse selon la re-
vendication 1, **caractérisé en ce qu'une** surface
supérieure de ladite rainure d'engagement (9G) est
formée afin d'être inclinée par rapport à une direc-

tion perpendiculaire à ladite direction verticale.

5. Outil supérieur pour une presse plieuse selon la revendication 1, **caractérisé en ce qu'**une surface verticale (9SV) est formée entre ladite surface inclinée (9S) et ladite rainure d'engagement (9G). 5
6. Outil supérieur pour une presse plieuse selon la revendication 1, **caractérisé en ce qu'**une surface supérieure (9GF) de la rainure d'engagement (9G) est légèrement inclinée vers le bas depuis une ligne horizontale vers l'intérieur afin de pousser l'outil supérieur contre la plaque de support (7). 10
7. Outil supérieur pour une presse plieuse selon la revendication 1, **caractérisé en ce que** ladite surface inclinée (9S) de l'outil supérieur est durcie en fixant un métal dur sur sa surface. 15
8. Outil supérieur pour une presse plieuse selon la revendication 1, **caractérisé en ce que** ladite surface inclinée (9S) de l'outil supérieur est durcie en déposant un métal dur sur sa surface. 20
9. Outil supérieur pour une presse plieuse selon la revendication 1 en combinaison avec ledit dispositif de support (1), **caractérisé en ce qu'**une dimension de ladite surface inclinée (9S) dans une direction verticale quand une partie de saillie supérieure dudit outil supérieur engage ledit élément de serrage d'outil supérieur (11) est supérieure à un espace (H) entre une surface d'extrémité inférieure (7E) dudit corps de support (5) et d'une partie d'épaulement (9F) dudit outil supérieur (9). 25 30
10. Outil supérieur pour une presse plieuse selon la revendication 1 en combinaison avec ledit dispositif de support (1), **caractérisé en ce que** ledit outil supérieur (9) est monté de manière démontable entre ladite plaque de support (7) et ledit élément de serrage d'outil supérieur (11), ladite plaque de support (7) étant prévue sur une partie inférieure d'un corps de support (5) dans un support d'outil supérieur (1) fixé sur un côté inférieur de ladite table supérieure (3) de ladite presse plieuse, ledit élément de serrage d'outil supérieur (11) étant monté de façon pivotante sur ledit corps de support (5) et étant construit afin d'être capable d'appuyer ledit outil supérieur (9) contre ladite plaque de support (7), 35 40 45
ladite surface inclinée (9S) amenant ledit élément de serrage d'outil supérieur (11) à pivoter afin d'augmenter une force de serrage dudit outil supérieur (9) due audit élément de serrage d'outil supérieur (11) en cas de soulèvement dudit outil supérieur (9) par rapport à ladite plaque de support (7) afin de faire entrer en contact ladite surface de contact (9F) avec ladite surface d'extrémité inférieure de ladite plaque de support (107), 50 55

où, dans une condition dans laquelle une surface supérieure de ladite rainure d'engagement (9G) prévue dans ledit outil supérieur (9) est supportée par une saillie d'engagement (11K) prévue dans ledit élément de serrage d'outil supérieur (11), une distance H entre ladite surface d'extrémité inférieure (7E) de ladite plaque de support (7) et ladite surface de contact (9F) dudit outil supérieur (9) peut être calculée par l'expression suivante :

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

où A désigne une distance entre un centre de pivotement dudit élément de serrage d'outil supérieur (11) et des moyens élastiques destinés à générer une force de serrage sur ledit élément de serrage d'outil supérieur;

B désigne une distance moyenne entre ledit centre de pivotement dudit élément de serrage d'outil supérieur et un point d'application de force de serrage;

P désigne ladite force de serrage appliquée sur ledit élément de serrage d'outil supérieur;

K désigne un coefficient élastique desdits moyens élastiques;

et θ désigne l'angle d'inclinaison de la surface inclinée (9S) dudit outil supérieur par rapport à ladite ligne verticale.

11. Outil supérieur pour une presse plieuse selon la revendication 1 en combinaison avec ledit dispositif de support (1), ledit outil supérieur (9) est serré de manière élastique entre ladite plaque de support (7) dudit corps de support (5) fixé sur une partie inférieure de ladite table supérieure (3) et ledit élément de serrage d'outil supérieur (11), **caractérisé en ce que** ladite surface inclinée (9S) est amenée en contact avec une surface inclinée dudit élément de serrage d'outil supérieur (11) quand ledit outil supérieur est serré entre ledit corps de support d'outil supérieur (5) et ledit élément de serrage d'outil supérieur (11) d'une manière telle que la force de serrage peut encore être accrue quand ledit outil supérieur est engagé avec une matrice (63). 35 40 45
12. Outil supérieur et dispositif de support (1) selon la revendication 12, **caractérisé en ce qu'**une surface inférieure (9GL) de ladite rainure d'engagement (9G) dudit outil supérieur est formée dans une position verticalement plus haute qu'une surface de contact (9F) amenée en contact avec une surface d'extrémité inférieure (7E) de ladite plaque de support (7) dudit corps de support (5). 50
13. Outil supérieur et dispositif de support (1) selon la revendication 12, **caractérisée en ce qu'**une distance H entre ladite surface inférieure (7E) de ladite 55

plaque de support (7) et une surface supérieure (7E) de ladite plaque de support (7) et une surface supérieure (9F) dudit outil supérieur (9) obtenue quand ladite surface supérieure de ladite rainure d'engagement (9G) dudit outil supérieur est en contact avec ladite saillie d'engagement (11K) dudit élément de serrage d'outil supérieur (11) est déterminée comme suit :

$$H = (B^2 \cdot P) / (A^2 \cdot K \cdot \tan \theta)$$

où A désigne une distance entre un centre de pivotement dudit élément de serrage d'outil supérieur (11)

et des moyens élastiques destinés à générer une force de serrage sur ledit élément de serrage d'outil supérieur;

B désigne une distance moyenne entre ledit centre de pivotement dudit élément de serrage d'outil supérieur et un point d'application de force de serrage;

P désigne ladite force de serrage appliquée sur ledit élément de serrage d'outil supérieur;

K désigne un coefficient élastique desdits moyens élastiques;

et θ désigne l'angle d'inclinaison de la surface inclinée (9S) dudit outil supérieur par rapport à ladite ligne verticale.

14. Ensemble d'outil supérieur pour une presse plieuse, comportant un outil supérieur selon l'une quelconque des revendications 1 à 8 et un appareil de support d'outil supérieur, afin de serrer un outil supérieur (205) entre une partie de support d'outil supérieur (209) d'un corps de support d'outil supérieur (207) fixé sur une partie inférieure d'une table supérieure (203) et un élément de serrage d'outil supérieur pivotant (211) par des moyens de génération de force de serrage (213), **caractérisé en ce que**:

ledit élément de serrage d'outil supérieur pivotant (211) est formé avec une saillie d'engagement (211P) engagée avec une rainure de prévention de chute (205G) formée dans ledit outil supérieur (205); et

ledit appareil de support d'outil supérieur comporte en outre des moyens d'actionnement de serrage (215) afin de se déplacer de manière sélective vers une position de serrage A dans laquelle ladite force de serrage peut être appliquée sur ledit élément de serrage d'outil supérieur (211) par lesdits moyens de génération de force de serrage (213) afin de serrer légèrement ledit outil supérieur, une position de des-serrage B dans laquelle ladite force de serrage peut être réduite afin de permettre audit outil

supérieur d'être déplacé horizontalement de manière réglable, et une position d'échange C dans laquelle ledit élément de serrage d'outil supérieur (211) peut être déplacé à l'écart dudit outil supérieur afin de permettre audit outil supérieur d'être remplacé verticalement par un autre.

15. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 14, **caractérisé en ce que** lesdits moyens d'actionnement de serrage (215) comportent :

un élément de poussée (259P) destiné à pousser lesdits moyens de génération de force de serrage (213) de façon à générer ladite force de serrage desdits moyens de génération de force de serrage (213);

un levier d'actionnement (265); et

des moyens de commande de force de serrage (263) fixés sur ledit levier d'actionnement, amenés en contact avec et séparés de manière commandée dudit élément de poussée (259P) afin de commander ladite force de serrage appliquée par lesdits moyens de génération de force de serrage (213) sur ledit outil supérieur.

16. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 14, **caractérisé en ce que** ledit appareil de support d'outil supérieur comporte en outre des moyens de poussée d'élément de serrage d'outil supérieur (217) destinés à pousser légèrement ledit élément de serrage d'outil supérieur (211) dans la direction de serrage quand lesdits moyens d'actionnement de serrage (215) sont déplacés vers ladite position d'échange C.

17. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 16, **caractérisé en ce que** lesdits moyens de poussée d'élément de serrage d'outil supérieur (217) sont constitués par au moins un ressort hélicoïdal disposé entre ledit élément de serrage d'outil supérieur (211) et ledit corps de support d'outil supérieur (207).

18. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 15, **caractérisé en ce que** lesdits moyens de commande de force de serrage fixés sur ledit levier d'actionnement (265) sont constitués par une vis de poussée (263) fixée sur ledit levier d'actionnement et engagée de manière vissée avec ledit élément de serrage d'outil supérieur (211), ladite vis de poussée étant décalée par rapport audit élément de serrage d'outil supérieur (211) lorsqu'elle est tournée par ledit levier d'actionnement.

19. Ensemble d'outil supérieur pour une presse plieuse

selon la revendication 14, **caractérisé en ce que** ledit appareil de support d'outil supérieur comporte en outre des moyens de maintien de libération de serrage (219) destinés à retenir ledit élément de serrage d'outil supérieur (211) dans un état de relâchement de force de serrage, afin de former un espace d'échange d'outil supérieur entre ladite partie de support d'outil supérieur (209), sous la condition que lesdits moyens d'actionnement de serrage (215) sont dans ladite position d'échange C.

20. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 19, **caractérisé en ce que** lesdits moyens de maintien de libération de serrage (219) comportent :

un renforcement d'engagement (279) formé dans ledit élément de serrage d'outil supérieur (211); et
un plongeur (218) fixé sur ledit corps de support (207) de façon à être engagé dans ledit renforcement d'engagement, quand ledit élément de serrage d'outil supérieur (211) est amené à pivoter à l'écart de ladite partie de support d'outil supérieur (209).

21. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 15, **caractérisé en ce que** ledit appareil de support d'outil supérieur comporte en outre deux moyens d'arrêt de levier d'actionnement de serrage (271) afin d'arrêter ledit levier d'actionnement de serrage déplacé (265) dans ladite position de desserrage B et dans ladite position d'échange C, respectivement.

22. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 21, **caractérisé en ce que** chacun desdits moyens d'arrêt de levier d'actionnement de serrage (217) comporte :

un axe (275) fixé sur une surface latérale d'une fente (273) formée dans une plaque de support (225);
un élément de butée (271) supporté de façon pivotante par ledit axe de façon à dépasser ou être rétracté d'une surface de ladite plaque de support; et
un plongeur (277) fixé sur ladite plaque de support afin de retenir ladite butée dans ladite position de dépassement ou rétractée, respectivement.

23. Ensemble d'outil supérieur selon la revendication 14, **caractérisé en ce que** ledit appareil de support d'outil supérieur comporte en outre :

une pièce de cale (247) fixée au-dessus de ladite saillie d'engagement (211P) formée dans

ledit élément de serrage d'outil supérieur (211) de façon à être engagée avec ladite rainure de prévention de chute (205G) formée dans ledit outil supérieur (205); et
un ressort hélicoïdal (249) destiné à pousser ladite pièce de cale dans une direction telle que ledit outil supérieur serré peut être poussé dans une direction de serrage.

24. Ensemble d'outil supérieur pour une presse plieuse, comportant un outil supérieur selon l'une quelconque des revendications 1 à 8 et un appareil de support d'outil supérieur destiné à serrer ledit outil supérieur (305) entre une partie de support d'outil supérieur (313) d'un corps d'outil supérieur (301) fixé sur une partie inférieure d'une table supérieure (303) et un élément de serrage d'outil supérieur (309) grâce à un élément de fixation (327), **caractérisé en ce que** l'outil supérieur (305) est serré par un élément de fixation (327) qui coopère avec un élément élastique (329) afin de serrer de manière élastique ledit outil supérieur entre ladite partie de support d'outil supérieur (313) et ledit élément de serrage d'outil supérieur (309), ledit élément élastique (329) étant interposé entre ledit élément de serrage d'outil supérieur (309) et ledit élément de fixation (327).

25. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 24, **caractérisé en ce que** ledit élément de serrage d'outil supérieur (309) est formé avec une surface inclinée (309F) élargie vers le bas au niveau d'une partie supérieure et amenée en contact avec une surface inclinée (305F) d'un outil supérieur (305) quand ledit outil supérieur est serré entre ladite partie de support d'outil supérieur (313) et ledit élément de serrage d'outil supérieur (309) d'une manière telle que ladite force de serrage peut être encore accrue quand ledit outil supérieur est engagé avec une matrice (63).

26. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 24, **caractérisé en ce que** ledit élément de fixation (327) se compose d'une partie filetée (327S) engagée de manière vissée avec ledit corps de support d'outil supérieur (313) et d'une partie de corps cylindrique (327H) afin de déterminer un espace constant pour disposer ledit élément de serrage d'outil supérieur (309) et ledit élément élastique (329).

27. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 24, **caractérisé en ce que** ledit élément de serrage d'outil supérieur (309) est formé avec une saillie d'engagement (309P) engagée dans une rainure engagée (305F) formée dans ledit outil supérieur (305).

28. Ensemble d'outil supérieur pour une presse plieuse selon la revendication 24, **caractérisé en ce que** des bords latéraux droit et gauche (309C) dudit élément de serrage d'outil supérieur (309) sont chanfreinés afin de faciliter l'insertion dudit outil supérieur entre ladite partie de support d'outil supérieur (313) et ledit élément de serrage d'outil supérieur (309). 5
29. Procédé de serrage d'un outil supérieur (305) avec un ensemble d'outil supérieur selon la revendication 24, entre un corps de support d'outil supérieur (307) fixé sur une table supérieure (303) d'une presse plieuse et un élément de serrage d'outil supérieur (309) grâce à une force de serrage élastique d'un élément élastique, qui comporte l'étape consistant à : 10
- fixer un élément de fixation (327) destiné à pousser ledit élément de serrage d'outil supérieur (309) formé avec une surface inclinée (309F) vers ledit corps de support d'outil supérieur (307) dans une position de fixation constante par rapport audit corps de support d'outil supérieur (307); 20
- insérer ledit outil supérieur (305) formé avec une surface inclinée (305F) dans l'espace prédéterminé par en dessous d'une manière telle que la surface inclinée d'outil supérieur (305F) est amenée en contact avec ladite surface inclinée d'élément de serrage (309F); et 25
- engager ledit outil supérieur (305) avec une matrice (337) en déplaçant n'importe lequel dudit outil supérieur et de ladite matrice de façon à déplacer ledit outil supérieur vers le haut par rapport audit corps de support d'outil supérieur (307) et comprimer ainsi un élément élastique (329) interposé entre ledit élément de fixation (327) et ledit élément de serrage d'outil supérieur (309) en permanence à un taux constant de compression du fait d'un effet de coin entre les deux dites surfaces inclinées dudit outil supérieur (305) et dudit élément de serrage d'outil supérieur (309), de sorte que ledit outil supérieur (305) peut être serré entre ledit corps de support d'outil supérieur (307) et ledit élément de serrage d'outil supérieur (309) en permanence avec une force de serrage constante. 30
- 35
- 40
- 45
- 50
- 55

FIG. 1

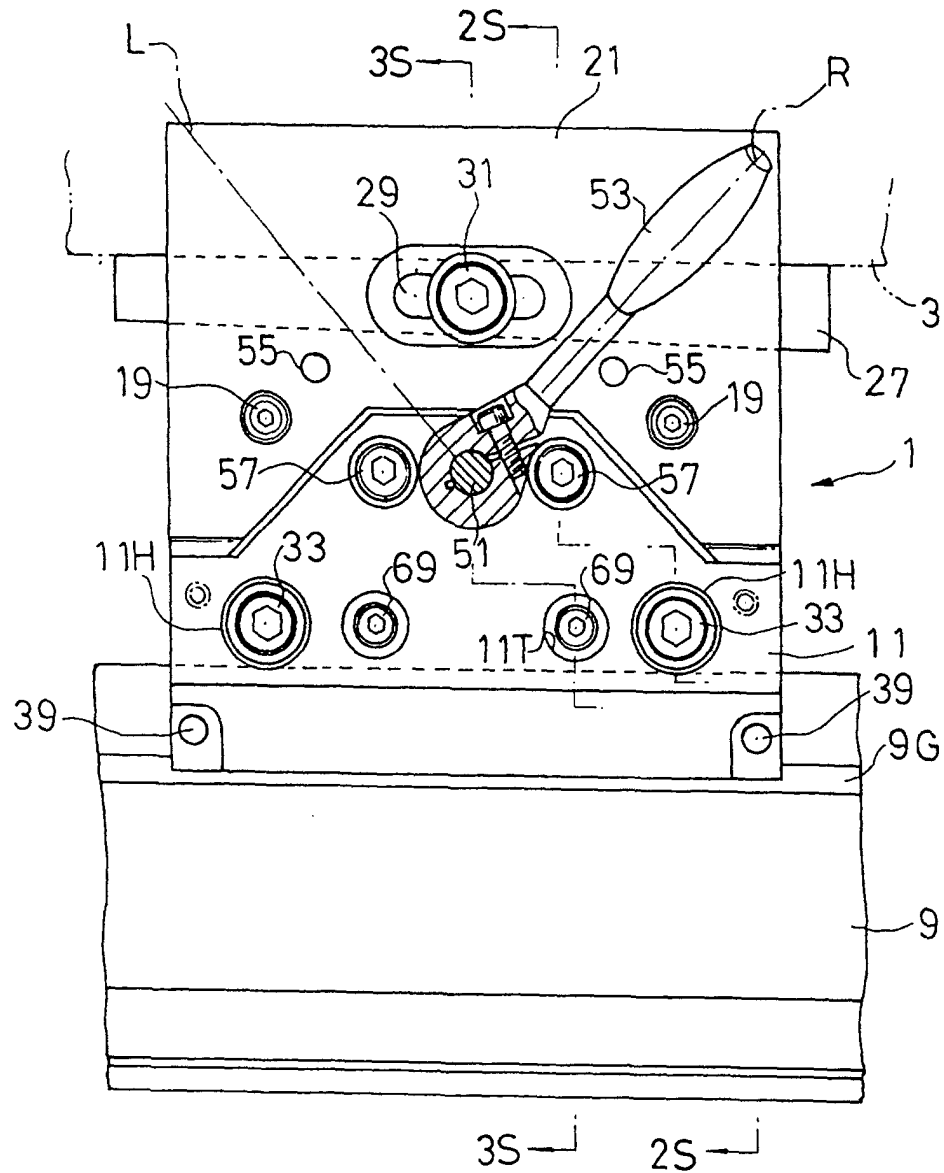


FIG. 2

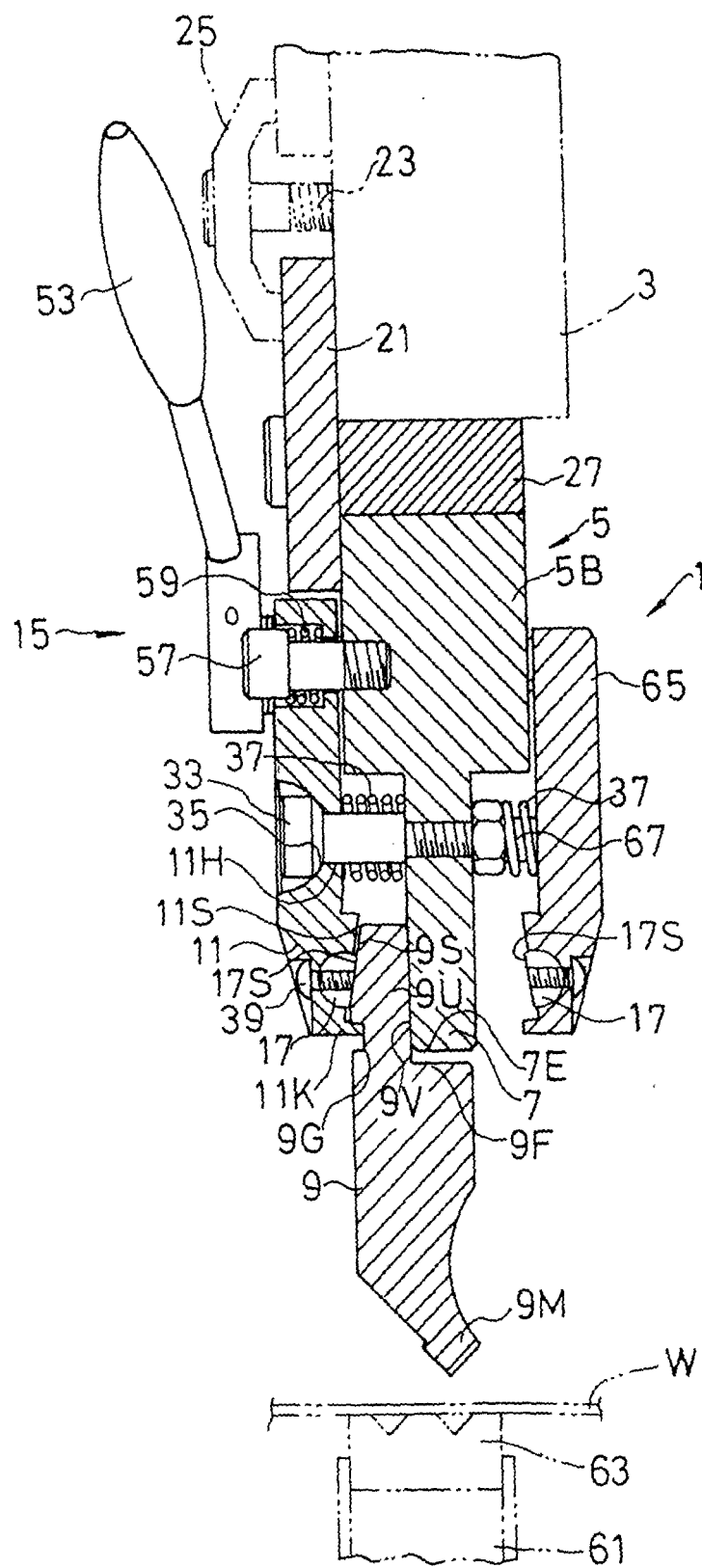


FIG.3

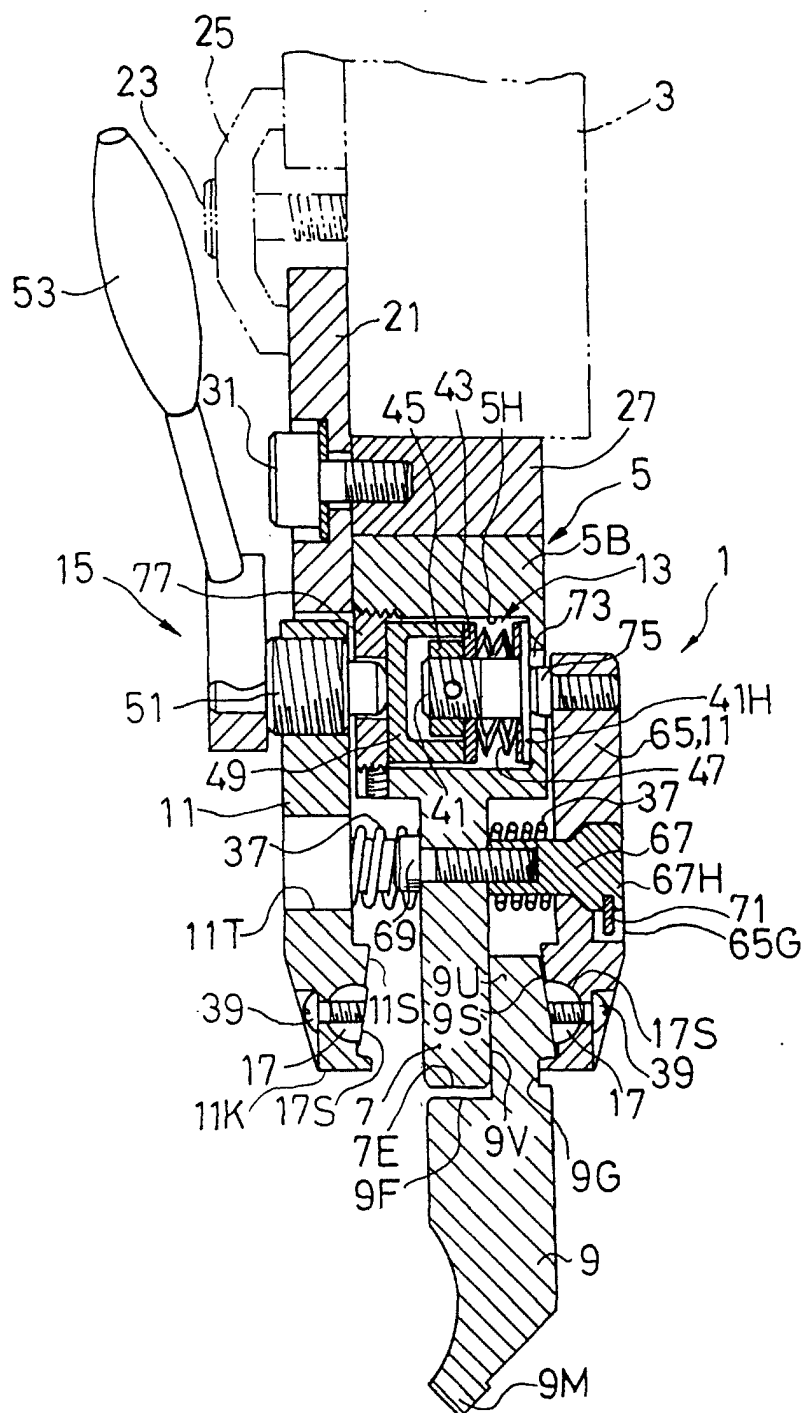


FIG.4

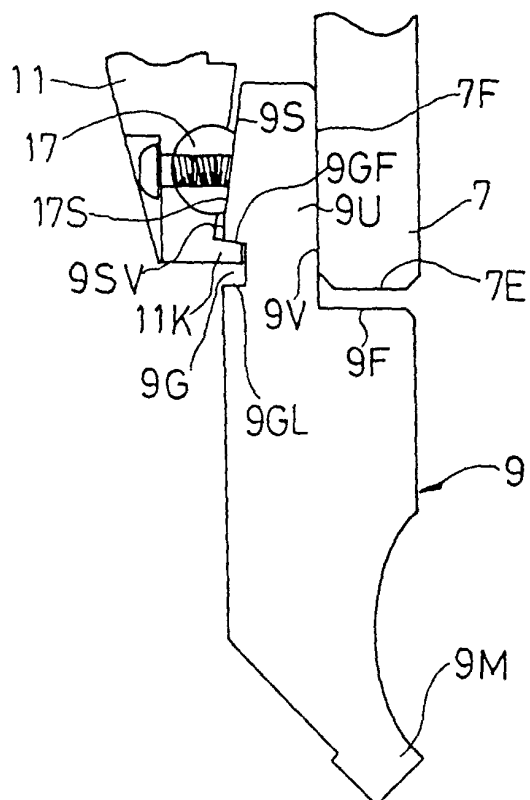


FIG.5

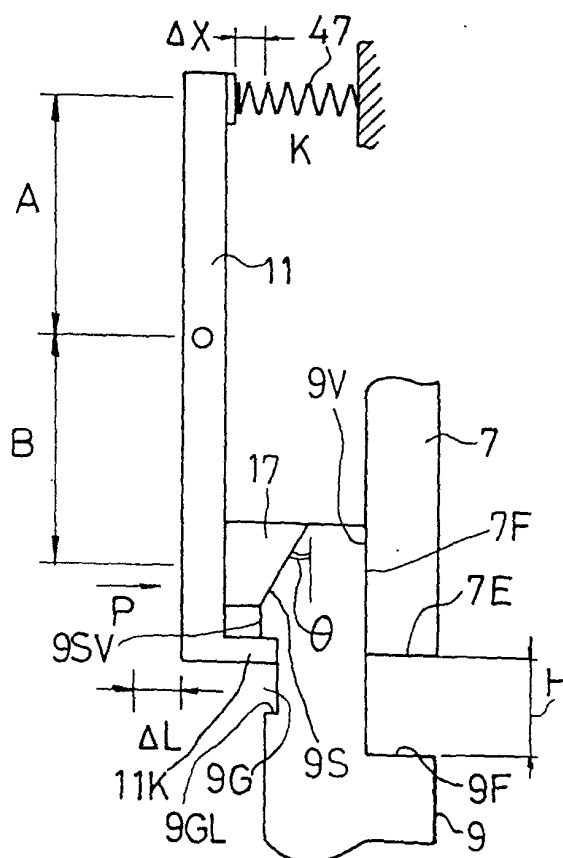


FIG.6

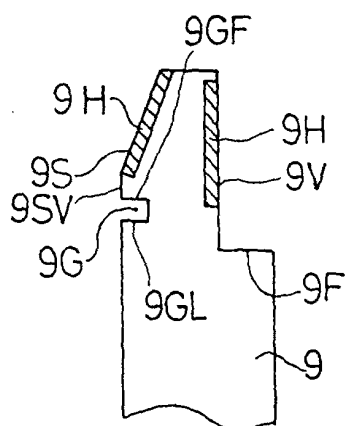


FIG.7

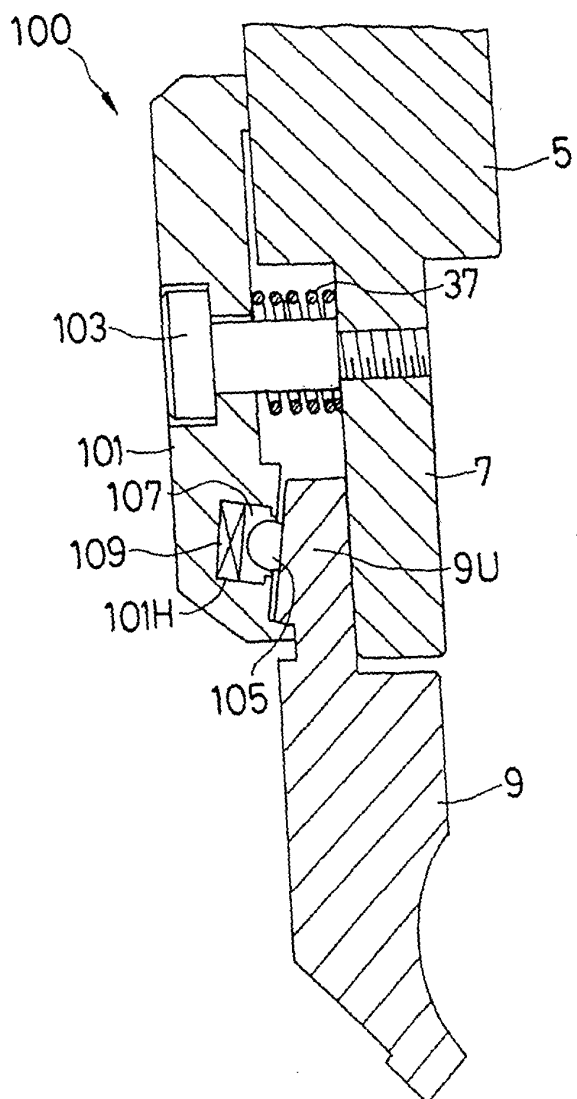


FIG.8

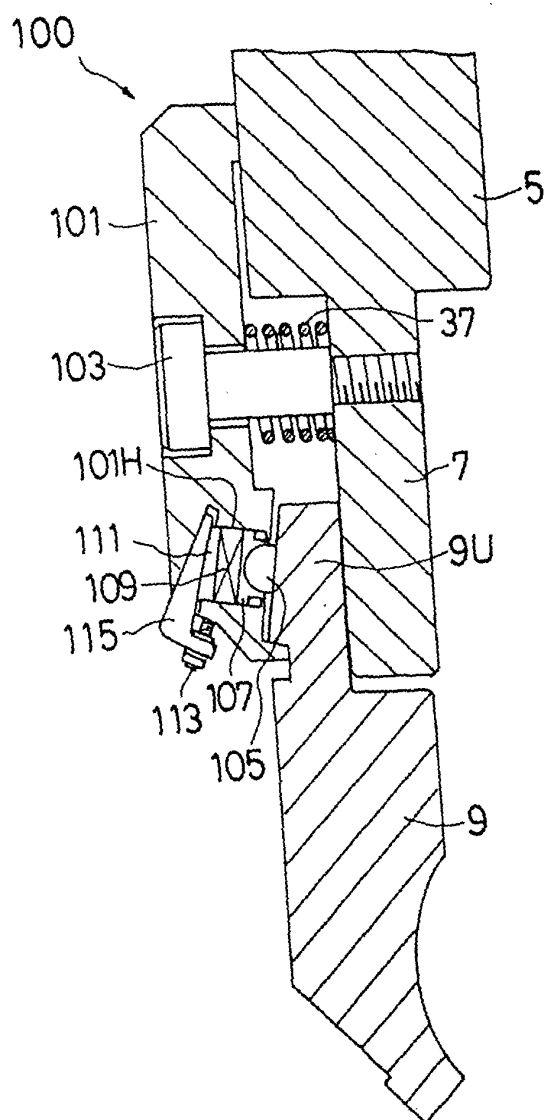


FIG.9

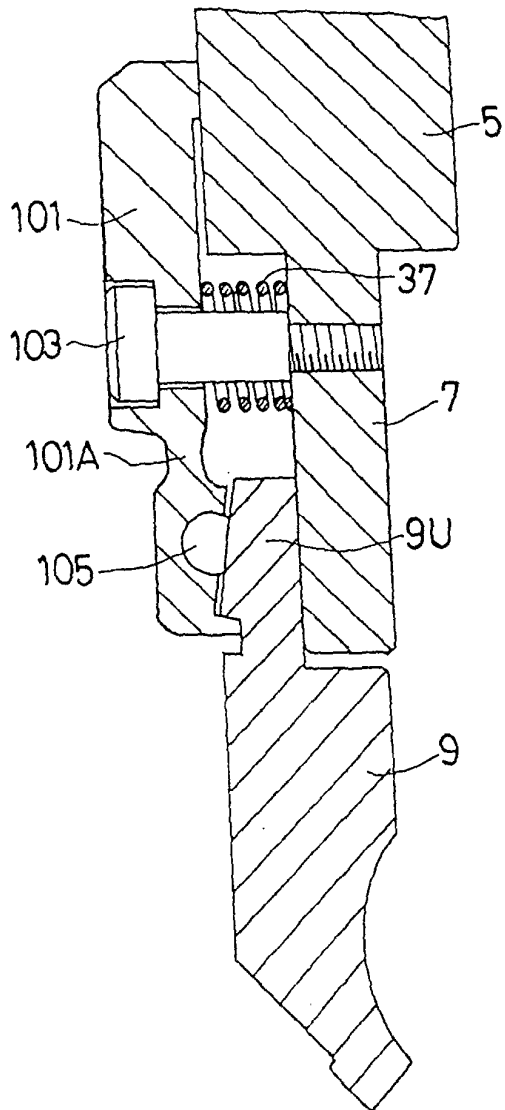


FIG.10

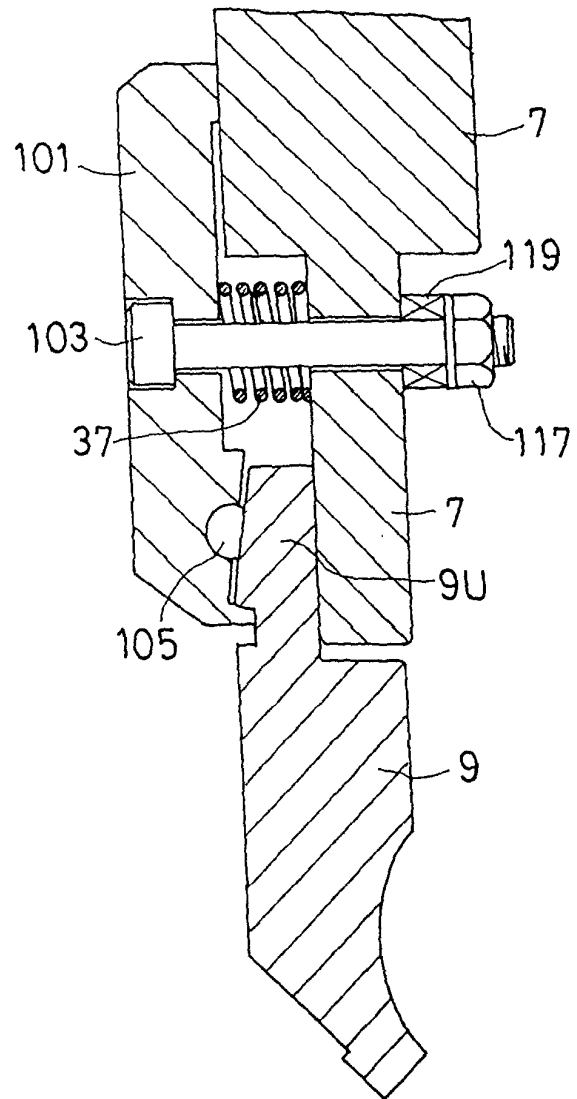


FIG. 11

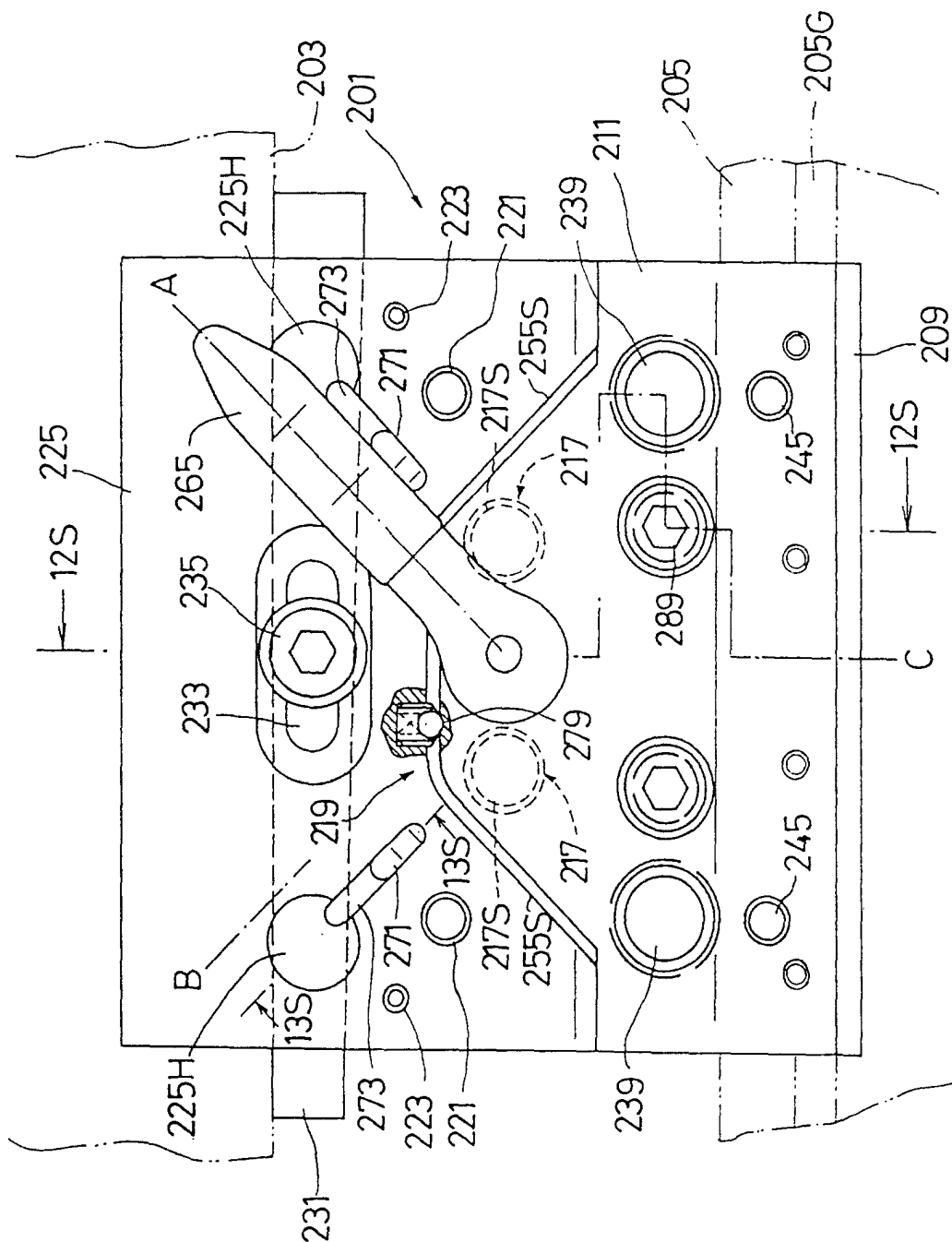


FIG. 12

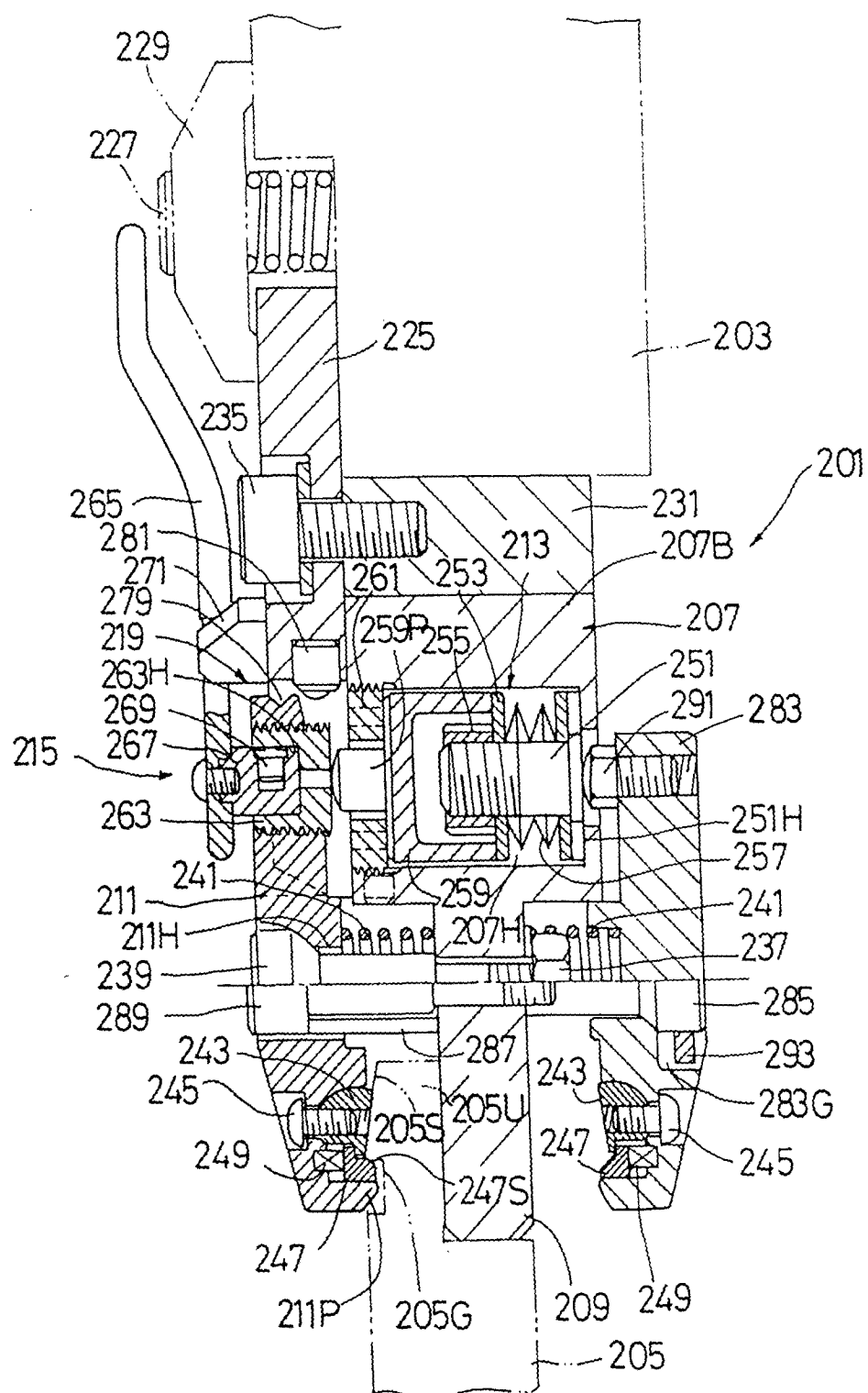


FIG. 13

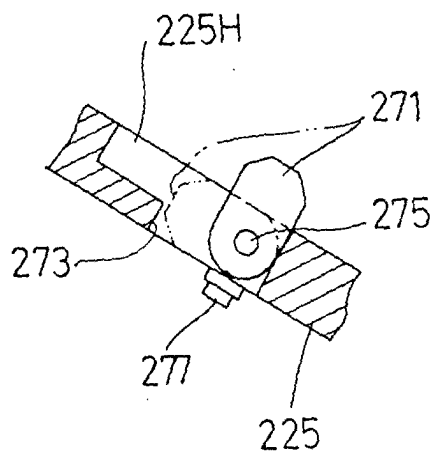


FIG. 14A

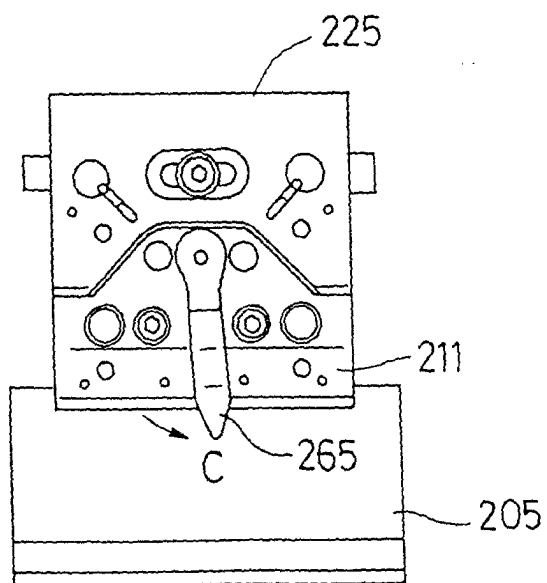


FIG. 14B

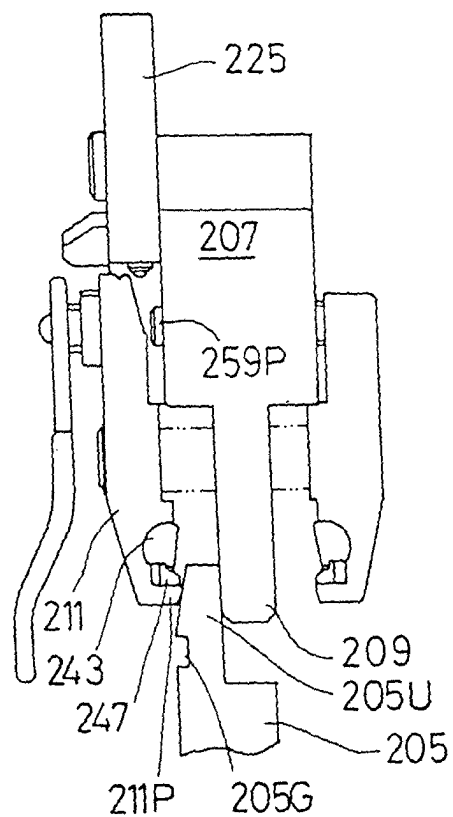


FIG.15A

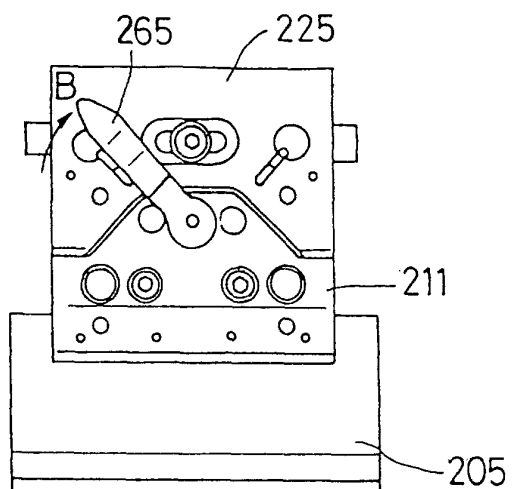


FIG.15B

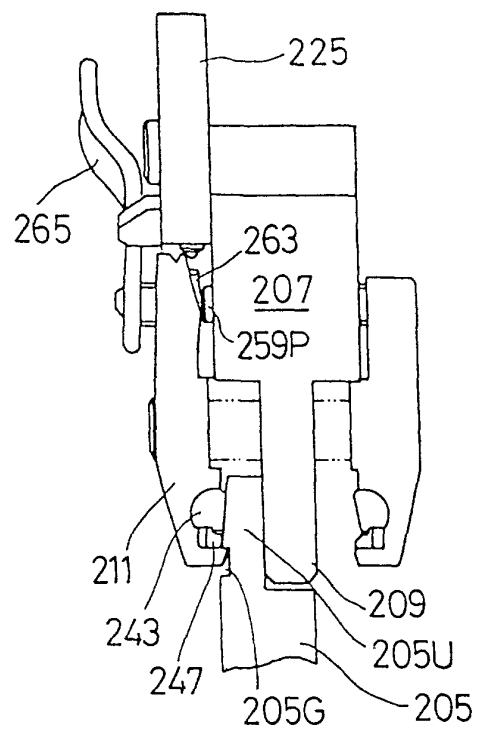


FIG.16A

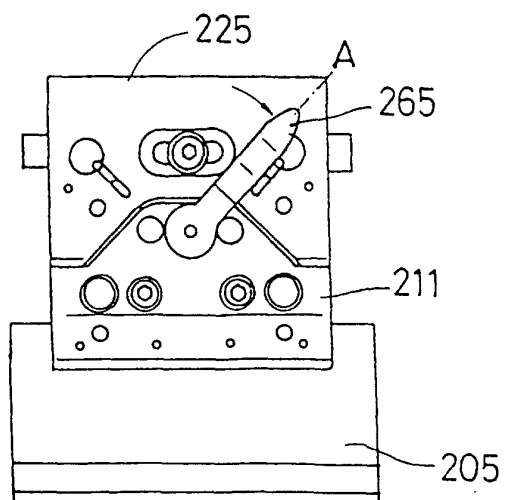


FIG.16B

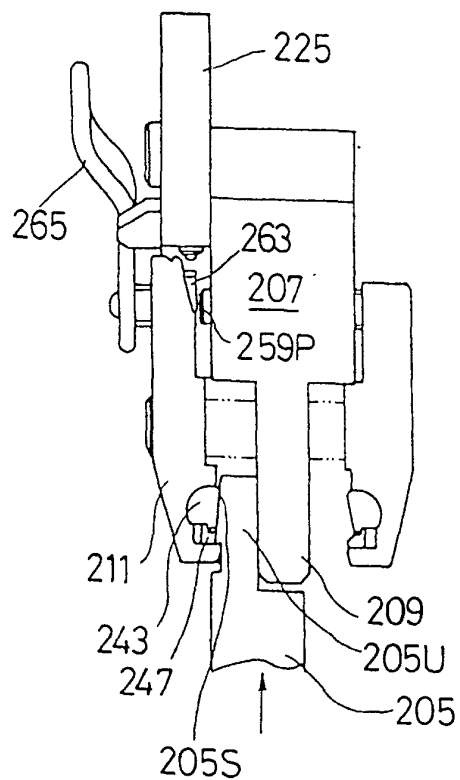


FIG.17A

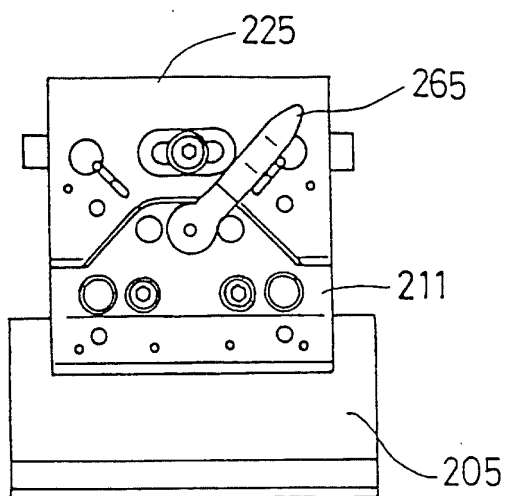


FIG.17B

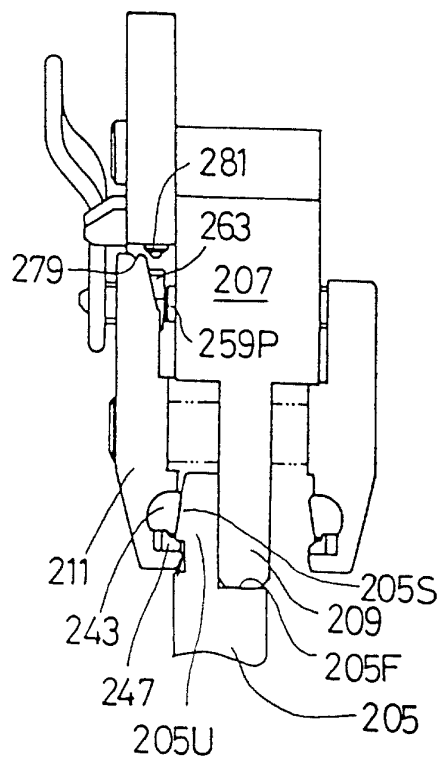


FIG.18A

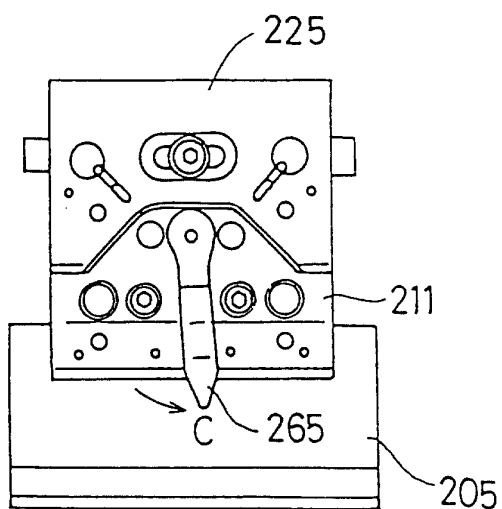


FIG.18B

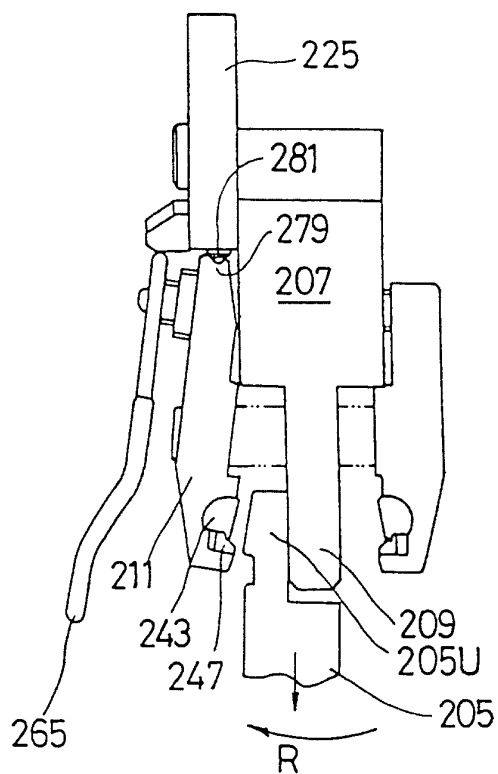


FIG. 19

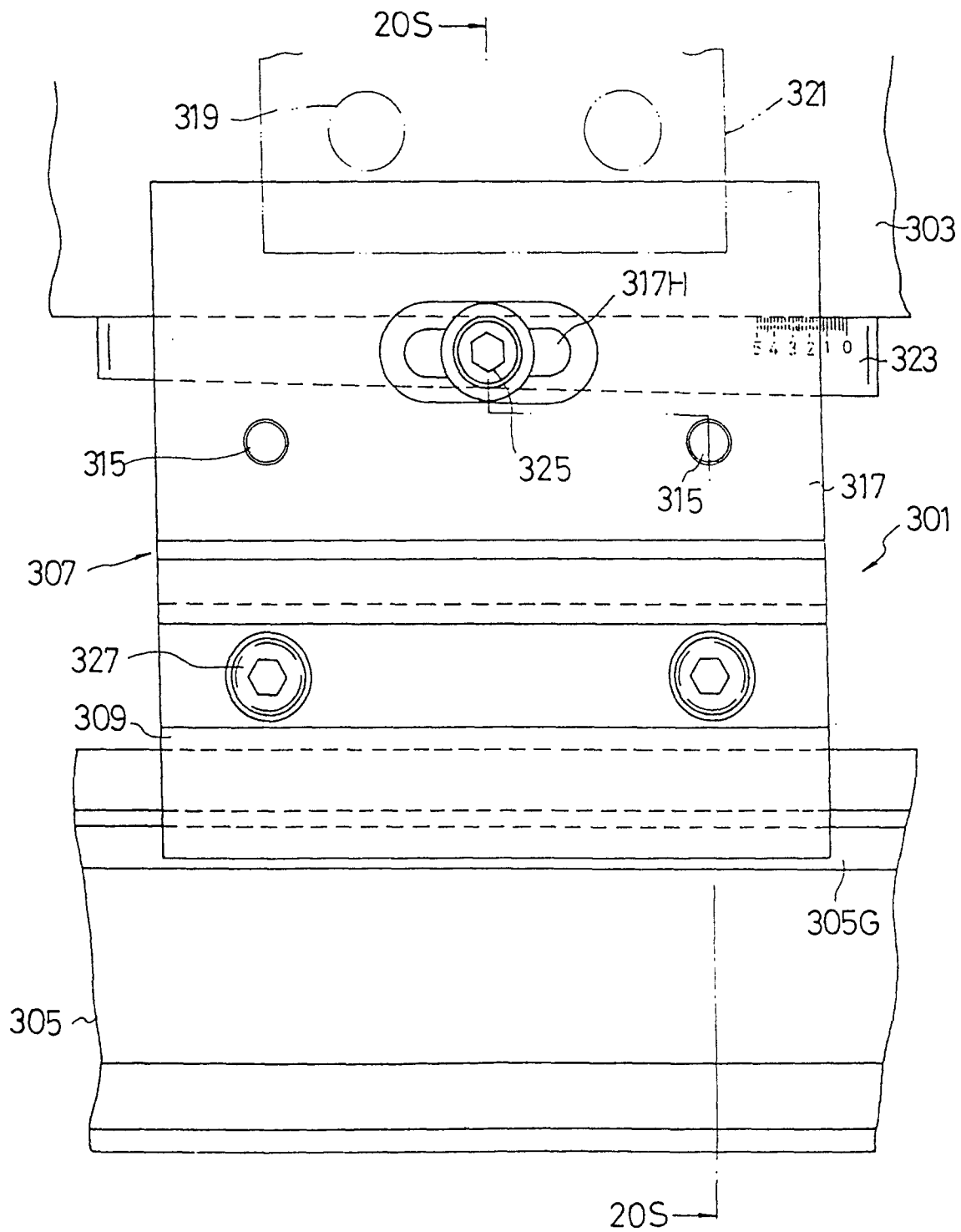


FIG. 20

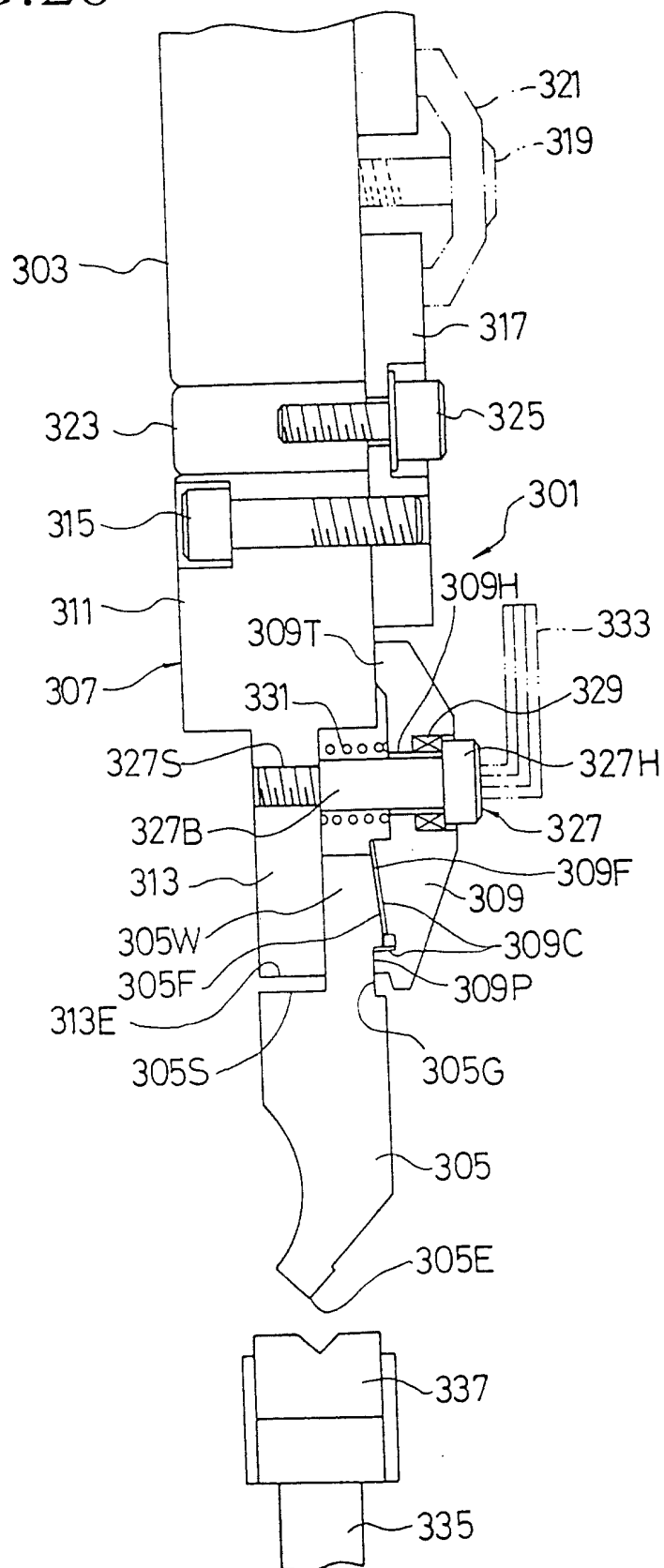


FIG.21

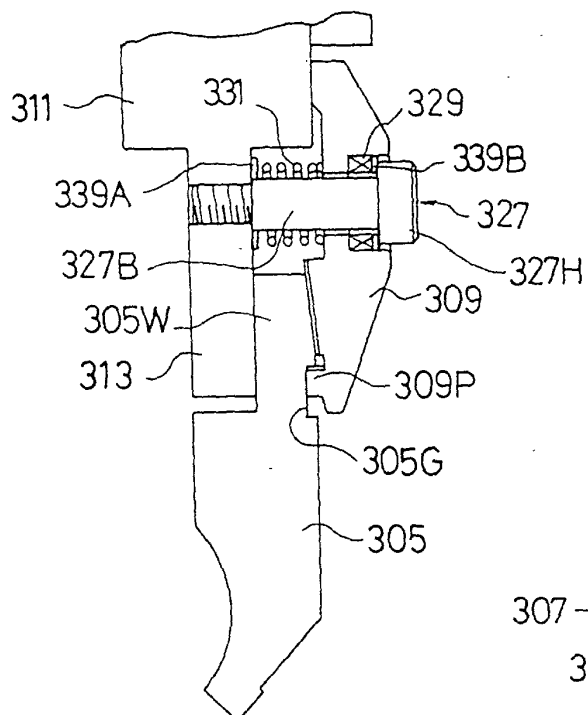


FIG.22

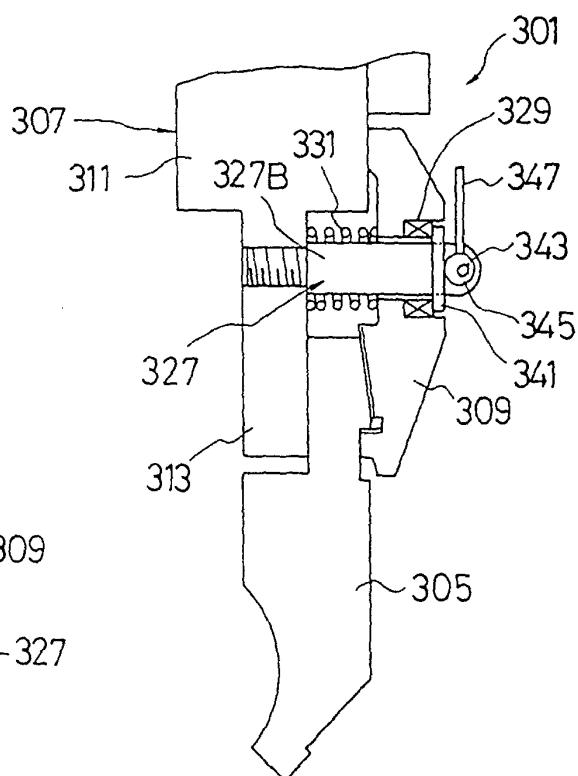


FIG.23

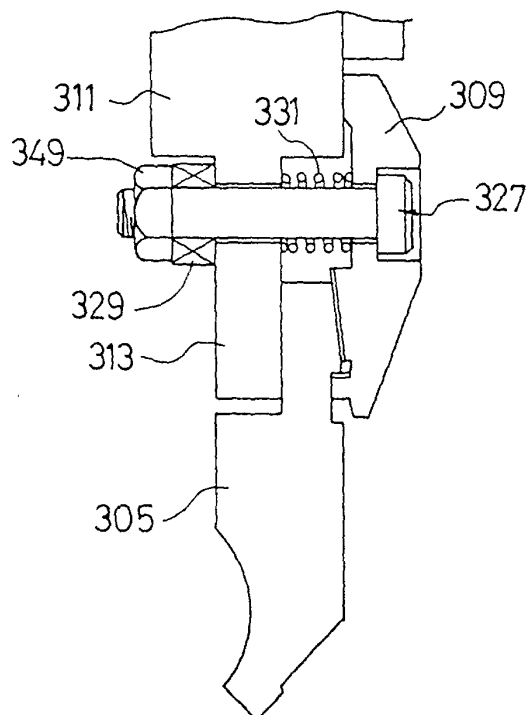


FIG.24

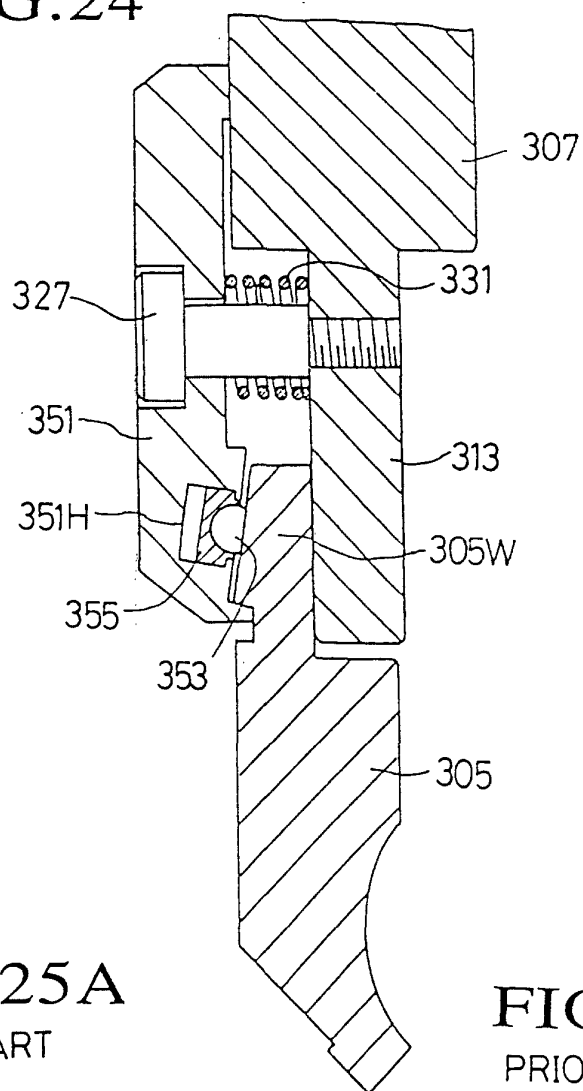


FIG.25A

PRIOR ART

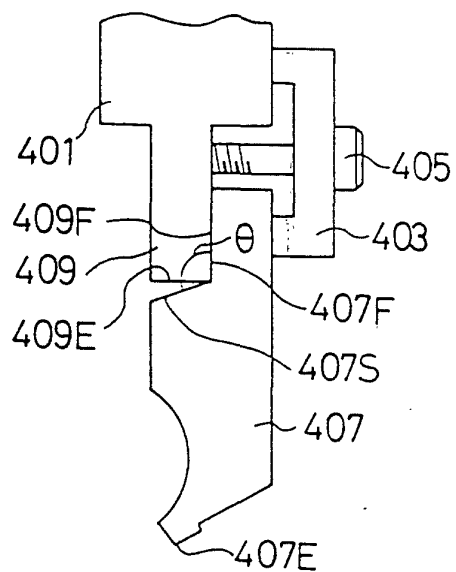


FIG.25B

PRIOR ART

