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(54) **Tool connector**

(57) A tool connector for downhole use in oil and gas fields. The tool connector includes a stinger (10) and a stinger receptacle (100), the stinger (10) being adapted to be stabbed into the stinger receptacle (100). A loaded engaging member (108) is provided, which is movable between a running position before the stinger (10) is stabbed into the stinger receptacle (100) and a latched position when the stinger (10) is stabbed into the stinger receptacle to latch the stinger and the stinger receptacle together. A release member (170) retains the loaded engaging member in the running position. When the stinger (10) is stabbed into the stinger receptacle (100) and a set force is applied to the stinger (10) and stinger receptacle (100), the release member releases the loaded engaging member to move to the latched position and latch the stinger (10) and the stinger receptacle (100) together. The tool connector is particularly suitable for connecting perforating guns.

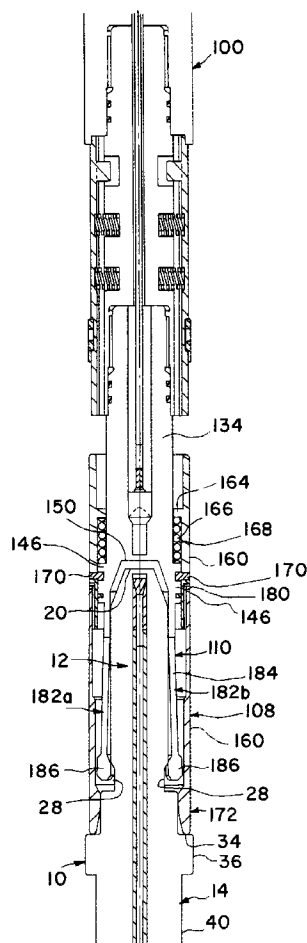


FIG. 7

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Description

This invention relates to a tool connector, and also relates to a method of connecting a first tool section to a second tool section. More particularly, the invention relates to assemblies and methods for connecting and releasing tubing sections and other tool sections for downhole use in oil and gas fields. More particularly, this invention relates to assemblies and methods for connecting and releasing tubing sections and other tool sections that do not require rotating the tool to latch and release the connector. The invention is especially suitable for connecting tool sections in the form of perforating guns.

For the purposes of this description, the term "tool section" refers to any tubular member or section intended for downhole use, including, for example, standard pipe joint sections, well packers, and other downhole tools for use in oil and gas wells, especially perforating guns.

In the past, conventional threaded pin-and-bell connectors have been used to connect tool sections for various downhole applications, including perforating gun sections. For example, after a tool section is positioned and set in a slip assembly at the rig floor of a well (usually with a threaded pin connector at the upper end thereof), a second tool section is picked up and brought into position over the first tool section. As the second tool section (usually with a threaded bell connector at the lower end thereof) is swinging in the blocks of the rig, it must be carefully axially aligned with the first tool section so that it can be set on the pin connector of the first tool section. The second tool section is then rotated to make up the threaded connection.

There are several problems of using threaded pin-and-bell connections. For example, the process of carefully aligning and threading one elongated tool section to the next is time consuming. Skilled oil-field hands need about one to two minutes to make up or break apart typical tool sections using threaded pin-and-bell connectors, which are often about thirty feet long. The step of aligning the second tool section can be particularly difficult in windy conditions, which cause the thirty-foot section to swing in the blocks. If the second tool section is not properly aligned, the threads of the pin-and-bell connectors are likely to gall and bind.

As an alternative to conventional threaded pin-and-bell connectors, some tool connectors are activated or released by certain types of rotational movements other than threading. However, it is becoming increasingly common to use tool sections with coil tubing. Coil tubing may be hundreds or thousands of feet long, such that it is extremely difficult or completely impractical to attempt to rotate the coil tubing to operate a latch or release connection. Thus, it would be desirable to provide a latch and release connector for use with tool sections that does not have to be rotated.

In some applications, tool sections and connector

assemblies must be able to pass through reduced diameter tubing or other downhole restrictions to reach the location in the casing where the perforation is to be performed. In these applications, the axial cross-section profile of the tool string (i.e. a perforating gun string, when the tool sections are perforating guns) is particularly important. For example, in the perforation of a five-inch (12.7cm) casing, passing through a small bore may be necessary for the tool assemblies, such as two-and-one-half inch (6.4cm) or one-and-eleven-sixteenth inch (4.3cm) tubing or other passageway. These through-tubing tool assemblies can be characterized as low-profile assemblies because of the restricted passageways through which they must pass to reach the desired downhole perforation location. These low-profile tool assemblies do not have the luxury of design spacing which is present in tool assemblies whose maximum outside dimensions approximate that of the casing that is to be perforated. These small profile or through-tubing tool assemblies present particular problems that are not present in their larger profiled cousins.

Additional problems are encountered in using downhole tool sections through a blowout preventer. The typical drilling well is provided with a blowout preventer ("BOP") at the well head, which is intended to maintain any pressure within the well head and prevent a blowout of the well. A blowout preventer is also used for safety to recompleat an existing well. A blowout can be an extremely hazardous situation if the oil or gas explodes or catches fire. Furthermore, even if the oil or gas does not ignite, allowing such uncontrolled escape is extremely wasteful of a valuable resource and harmful to the environment. In some countries such as the United States, an uncontrolled escape can subject the producer to substantial government fines for the environmental pollution and the costs of environmental clean up. Blowout preventers are well known in the art, and represented, for example, by U.S. Patent No. 4,416,441 entitled "Blowout Preventer" issued to Denzal W. Van Winkle on November 22, 1983 and by U.S. Patent No. 4,943,031 entitled "Blowout Preventer" issued to Denzal W. Van Winkle on July 24, 1990.

According to the art, two or more blowout preventers are typically used in a stack at the well head. For example, the rams of a lower blowout preventer are employed as slip rams, which have serrated metal teeth for gripping and holding a section of downhole tubing or other tool. The slip rams are useful as a type of slip assembly for holding a section of downhole tubing or tool section, which can have many additional sections connected to and suspended from the lower end thereof. The rams of a second blowout preventer above the first are employed as sealing rams, having rubber seals adapted to be compressed against the downhole tubing or other tool to form a pressure-tight seal around the tubing or tool.

Having additional blowout preventers in the stack is common. For example, the rams of a third blowout pre-

venter above the sealing BOP can be equipped with shearing blades for cutting a piece of tubing for which the threads have seized onto the next tubing and cannot be normally unthreaded. The rams of a fourth blowout preventer above the rest can be employed as a blind seal, such that the well head can be completely sealed. Thus, a production well usually has at least two blowout preventers at the well head used for controlling the well.

Unfortunately, the use of conventional threaded pin-and-bell connectors through a lubricator above a blowout preventer stack is particularly time consuming. For example, it typically requires about five minutes for skilled oil-field hands to make up tool sections together through a lubricator above a blowout preventer stack. There has been a particular long-felt need for an apparatus and method that would permit much faster connection and release of tool sections through a lubricator and blowout preventer stack. The cost of oil field hands and recovered production time involved in stringing several tool sections together has driven the search for faster apparatus and methods. Nevertheless, there is still a great need for additional improvements and methods.

Of all the downhole tool sections employed in a well, perforating gun sections present some of the most serious difficulties and challenges. Conventional perforating gun sections used in perforating well casings typically include charge carriers designed to support several separate perforating charges within the desired longitudinal spacing and sometimes a desired radial orientation. Examples of various convention perforating gun sections are illustrated in U.S. Patent No. 5,095,999 issued to Daniel C. Markel on March 17, 1992. In particular, the Markel patent illustrates a conventional enclosed perforating gun section having a plurality of perforating charges mounted on a carrier strip and enclosed and protected within a carrier tube. (See U.S. Patent No. 5,095,999, Column 5, lines 20-39 and Figure 5.)

As is well known in the industry, perforating gun sections use perforating shaped explosive charges designed to shape and direct the explosion with great precision along the focal axis. Typically, a perforating shaped charge will shape and direct a liner material to create a uniform circular jet that is highly focused and directed along the focal axis. The focused jet penetrates the casing that lines the well bore and the surrounding geological formation. The detonation of the perforating charges is intended to increase production of the well, which is hoped will result in a substantial increase in production pressure at the well head.

Usually, maximizing the perforations achievable in a single-shot downhole procedure is desirable. For example, it is sometimes desirable to perforate hundreds, even thousands, of linear feet (1 foot = 0.305m) of downhole casing to enhance well production. However, the length of the typical perforating gun section is about thirty feet (9.1m). Of course, it is possible to achieve increased perforation of the downhole well casing by re-

peating the procedure of lowering a perforating gun section to perforate the downhole well casing and retrieving the spent perforating gun section until the desired longitudinal portion of the downhole well casing has been perforated. However, the time and expense involved in repeating each such downhole procedure mitigate in favor of perforating the desired portion of the well bore in a single downhole procedure. Thus, if it is desirable to perforate such lengths of the downhole casing, as is frequently desirable, two or more perforating gun sections must be connected together. The assembled string of perforating gun sections is then lowered downhole to perforate the well in a single shot.

Furthermore, connecting perforating gun sections with the conventional threaded pin-and-bell connectors described above presents special problems and risks. For example, manually rotating the second perforating gun section with a hand wrench is more time consuming than the with the use of power tongs. With a hand wrench, however, the oil-field hands can feel the process of threading the connector and be more sensitive to whether the threads are properly aligned to prevent galling. But while the use of power tongs to rotate a perforating gun section to make up the threaded connection is faster, if it works, the threads of the connection are much more likely to gall because of the speed of rotation and the oil-field hands' inability to feel the threading and make any necessary adjustments in the alignment of the threads.

A galled threaded connector for perforating gun sections presents particular problems and dangers because of the explosives used in the sections. For example, if the threads gall and bind in a threaded pin-and-bell connector between two perforating gun sections, the transmission of the detonating signal between the two sections may not be reliable. Thus, it is usually desirable or necessary to separate the galled connection, and replace the connector and possibly both the perforating gun sections. However, unthreading the galled threads of the connector is sometimes difficult or impossible. Furthermore, cutting or shearing galled perforating gun sections, which contain high explosives, is counter indicated for obvious safety concerns. Thus, a galled threaded connection between perforating gun sections presents a serious problem. In the past, one of the only solutions to the problem of a seriously galled threaded connection has been to raise the two galled perforating gun sections and unthread the lower connection from the remainder of the perforating gun string, to then safely remove and handle the two improperly joined sections. However, this is wasteful of expensive perforating gun section equipment and extremely time consuming.

For these reasons, it can take several minutes to align, set, and manually make-up each threaded connection between the perforating gun sections, and a galled connection can seriously impede the process of perforating a well. Thus, there has been a long-felt need for a better, more reliable, and faster connector for per-

forating gun sections.

Furthermore, working with perforating gun sections through a stack of blowout preventers presents several additional problems and challenges. This is true even though the pressure at the well head is initially substantially balanced such that the well head can be opened for the insertion of a tool section. For example, after using the perforating gun section to perforate the downhole well bore, it hopefully increases the well production and the production pressure at the well head. Thus, a problem is then presented of how to withdraw the spent perforating gun section through the blowout preventer. The problem is particularly problematic because a spent perforating gun section has itself been thoroughly perforated by the detonation of the perforating shaped charges. For example, the sealing rams of the sealing blowout preventer may have difficulty fully sealing against the warped, twisted, and punctured metal of the perforating gun section. Furthermore, the open holes created in the spent perforating gun section provide multiple conduits for the pressurized fluid in the well beneath the blowout preventers to enter the spent perforating gun section. Thus, the spent perforating gun section provides an undesired conduit through the blowout preventer stack, leaking or spewing the pressurized production.

A prior art method of addressing this problem of how to remove a spent perforating gun section has been to balance the pressure in the well. Balancing the pressure is normally accomplished by pumping the appropriate density of drilling mud into the well head to equalize the pressure below and above the well head. However, this balancing procedure is sometimes called "killing" the well because it inhibits the production and can create other pressure management and technical difficulties. There has been a long-felt need for an apparatus and method for withdrawing the spent perforating gun section through the stack of blowout preventers at the well head without having to even temporarily kill the enhanced well production.

Furthermore, enhancing the well production of a well that has some positive well pressure at the well head is often desirable. In such a case, perforating the downhole casing is still desirable. Of course, working through a blowout preventer stack with an intact perforating gun section before it has been detonated can be accomplished by employing a lubricator above the blowout preventer stack. The perforating gun sections can be made up with the lubricator according to techniques well known to those of skill in the art. However, the use of a lubricator above the blowout preventer further limits the length of the perforating gun sections that can be used to the practical length of the lubricator. A typical lubricator for such applications can accommodate perforating gun sections of up to about 35 feet (11 meters).

Thus, there has been a long-felt need for assemblies and methods capable of more quickly stringing two tools, such as perforating guns, together for firing in a

single downhole procedure, thereby reducing the time and expense involved in perforating a well. There has been a long-felt need for apparatus and methods of withdrawing and more quickly separating spent tool sections from a well. In addition, there has been a particular need for apparatus and methods for connecting and separating tool sections through a blowout preventer stack while maintaining the pressure below the blowout preventer stack.

Broadly, the invention provides assemblies and methods for connecting tool sections for downhole use. In one aspect, the invention provides a tool connector including a stinger and a stinger receptacle. The stinger is adapted to be stabbed into the stinger receptacle. A loaded engaging member is provided, which is movable between a running position before the stinger is stabbed into the stinger receptacle and a latched position when the stinger is stabbed into the stinger receptacle to latch the stinger and the stinger receptacle together. A release member retains the loaded engaging member in the running position. When the stinger is stabbed into the stinger receptacle and a set force is applied to the stinger and stinger receptacle, the release member releases the loaded engaging member to move to the latched position and latch the stinger and the stinger receptacle together. Neither the stinger nor the stinger receptacle have to be rotated to make up the connection between the perforating gun sections.

In another aspect of the invention, the tool connector is releasable. The tool connector further includes a releasable stop member to stop the engaging member in the latched position. When the stop member is released, the engaging member moves to a released position such that the stinger and stinger receptacle are separable. Thus, the tool sections can also be released without rotating.

In another aspect of the invention, a tool connector having particular application to perforating gun sections is provided. According to this aspect, the tool connector is provided with an internal explosive transfer system for transferring the detonation signal from one perforating gun, through the perforating gun connector, and to the next perforating gun. The internal explosive transfer system protects the booster charges to provide additional safety.

In accordance with one aspect of the invention, there is provided a tool connector comprising: a stinger; a stinger receptacle, the stinger being adapted to be stabbed into the stinger receptacle; a loaded engaging member movable between a running position before the stinger is stabbed into the stinger receptacle and a latched position when the stinger is stabbed into the stinger receptacle to latch the stinger and the stinger receptacle together; and a release member retaining the loaded engaging member in the running position, whereby when the stinger is stabbed into the stinger receptacle and a set force is applied to the stinger and stinger receptacle, the release member releases the

loaded engaging member to move to the latched position and latch the stinger and the stinger receptacle together.

In accordance with another aspect of the invention, there is provided a perforating gun connector comprising: a stinger; a stinger receptacle, the stinger being adapted to be stabbed into the stinger receptacle; a loaded engaging member movable between a running position before the stinger is stabbed into the stinger receptacle and a latched position when the stinger is stabbed into the stinger receptacle to latch the stinger and the stinger receptacle together; and a release member retaining the loaded engaging member in the running position, whereby when the stinger is stabbed into the stinger receptacle and a set force is applied to the stinger and stinger receptacle, the release member releases the loaded engaging member to move to the latched position and latch the stinger and the stinger receptacle together.

Preferably, a releasable stop member is provided to stop the engaging member in the latched position, whereby when the stop member is released, the engaging member moves to a released position such that the stinger and stinger receptacle are separable. The releasable stop member may be adapted to be released by a clamp. The loaded engaging member may be mounted to the stinger receptacle. The releasable stop member may be mounted to the stinger receptacle.

The engaging member may be loaded by the stored potential energy of a spring retained in a compressed condition by the release member.

The connector may be provided with an internal explosive transfer system, whereby the tool connector has particular application for connecting perforating gun sections.

In an embodiment, the stinger comprises a stinger subassembly having:

- (i) a probe portion; and
- (ii) a stinger connector portion connected to the probe portion, the stinger connector portion being adapted to make up the stinger subassembly with a first tool or perforating gun section; and

wherein the stinger receptacle comprises a latch subassembly having:

- (i) a body portion, the body portion being adapted to receive the probe portion of the stinger subassembly;
- (ii) a latch connector portion connected to the body portion, the latch connector portion being adapted to make up the latch subassembly with a second tool or perforating gun section;
- (iii) a spring-loaded housing mounted to slide on the body portion, the spring-loaded housing adapted to be urged from a running position to a latched position, the spring-loaded housing having at least one

retaining pin to restrain the spring-loaded housing in the running position on the body portion until the retaining pin is sheared; and

(iv) collet fingers connected to the body portion substantially within the spring-loaded housing, the spring-loaded housing having a deflecting structure adapted to deflect the collet fingers to engage the probe portion of the stinger assembly,

whereby in the running position, the spring-loaded housing of the latch subassembly is retained in a running position for being moved onto the probe portion of the stinger subassembly; and whereby when a set force is applied to the latch subassembly against the stinger subassembly, the set force shears the retaining pin to release the spring-loaded housing, which is urged on the body portion toward the latched position such that the deflecting structure of the spring-loaded housing deflects the collet fingers to engage the probe portion of the stinger assembly and retains the collet fingers in latched engagement with the probe portion.

In an embodiment, the connector further comprises: spring-loaded stop/release pads mounted to the body portion of the latch subassembly so that the pads stop the spring-loaded housing in the latched position, the spring-loaded stop/release pads being adapted to be engaged and compressed by a clamp, whereby when the spring-loaded stop/release pads are clamped and compressed, the spring-loaded housing is urged on the body portion to a released position, which retains the spring-loaded stop/release pads in the compressed condition, and whereby the deflecting structure of the spring-loaded housing moves to free the collet fingers from engagement with the probe portion of the stinger subassembly.

In an embodiment, the stinger subassembly further comprises a stinger internal explosive transfer system, and the latch subassembly may further comprise a latch internal explosive transfer system, whereby the tool connector has particular application to perforating gun sections.

In an embodiment, the latch subassembly internal explosive transfer system comprises:

- (a) a latch internal chamber extending through the latch connector portion and through the body portion;
- (b) a latch receiving booster charge positioned in the latch internal chamber adjacent the latch connector portion;
- (c) a latch detonating cord positioned substantially throughout the length of the latch internal chamber;
- (d) a latch sending booster charge positioned in the latch internal chamber adjacent the body portion; and
- (e) a latch shaped charge positioned in the latch release chamber distal to the latch sending booster charge;

whereby the latch internal explosive transfer system is adapted to continue and transfer the detonation of perforating charges from the first tool or perforating gun section, through the latch subassembly, and to fire the latch shaped charge toward the probe portion of the stinger subassembly.

In an embodiment the stinger internal explosive transfer system comprises:

- (a) a stinger internal chamber extending from adjacent a tip end of the probe portion and to the stinger connector portion;
- (b) a stinger booster charge positioned in the stinger internal chamber adjacent the tip end of the probe portion;
- (c) a stinger detonating cord positioned substantially throughout the length of the stinger internal chamber;
- (d) a stinger firing pin positioned in the stinger internal chamber; and
- (e) a stinger initiator;

whereby the stinger internal explosive transfer system is adapted to be detonated by the latch internal explosive transfer system to continue and transfer the detonation of perforating charges from the latch subassembly, through the stinger subassembly, and to the second tool or perforating gun section made up with the stinger connector portion of the stinger subassembly.

The stinger connector portion of the stinger subassembly may be a pin connector portion. The latch connector portion of the latch subassembly may be a bell connector portion.

The stinger subassembly may further comprise: a slip landing portion connected to the probe portion, the slip landing portion being adapted to be engaged and held by a slip assembly.

In another embodiment, the stinger comprises a stinger subassembly having:

- (i) a probe portion; and
- (ii) means for connecting the probe portion to a first tool or perforating gun section; and wherein the stinger receptacle comprises: a latch subassembly having:
 - (i) a body portion, the body portion being adapted to receive the probe portion of the stinger subassembly;
 - (ii) means for connecting the body portion to a second tool perforating gun section;
 - (iii) spring-loaded housing mounted to slide on the body portion, the spring-loaded housing adapted to be urged from a running position to a latched position;
 - (iv) means for retaining the spring-loaded housing in the running position on the body portion; and
 - (v) means for latching the body portion to the probe portion of the stinger subassembly when the spring-

loaded housing is moved to the latched position;

whereby when a set force is applied to the latch subassembly against the stinger subassembly, the set force shears the means for retaining the spring-loaded housing such that the spring-loaded housing is urged on the body portion to the latched position.

In this embodiment, the tool may further comprise: spring-loaded stop/release means mounted to the body portion of the latch subassembly, the stop/release means for stopping the spring-loaded housing in the latched position after the set force shears the retaining means for the spring-loaded housing, and the spring-loaded stop/release means being adapted to be engaged and compressed by a clamp, whereby when the spring-loaded stop/release means is clamped and compressed, the spring-loaded housing is urged on the body portion from the latched position to a released position, which retains the spring-loaded stop release means in a compressed condition, and whereby the means for latching the body portion to the probe portion is released from engagement with the probe portion of the stinger subassembly.

According to another aspect of the invention there is provided, a method of connecting a first tool section to a second tool section, the method comprising the steps of:

- (a) connecting a stinger to the first tool section;
- (b) connecting a stinger receptacle to the second gun section;
- (c) stabbing the stinger to mate with the stinger receptacle;
- (d) applying a set force to the stinger and stinger receptacle to release a loaded engaging member from a running position to a latched position to latch the stinger and the stinger receptacle together.

According to another aspect of the invention there is provided a method of connecting a first perforating gun section to a second perforating gun section, the method comprising the steps of:

- (a) connecting a stinger to the first perforating gun section;
- (b) connecting a stinger receptacle to the second perforating gun section;
- (c) stabbing the stinger to mate with the stinger receptacle (100);
- (d) applying a set force to the stinger and stinger receptacle to release a loaded engaging member from a running position to a latched position to latch the stinger and the stinger receptacle together.

A releasable stop member may be clamped to disengage the loaded engaging member from the latched position such that the stinger and stinger receptacle are separable.

In one embodiment, the step of clamping the releasable stop member to disengage the loaded engaging member may be accomplished by use of a hand clamp.

In another embodiment, the step of clamping the releasable stop member to disengage the loaded engaging member may be accomplished by use of operating rams of a blowout preventer.

Reference is now made to the accompany drawings, in which:

FIG. 1 is an axial cross-section view of a stinger subassembly for an embodiment of a latch and release tool connector according to the invention;

FIG. 2 is a detail cross-section view of part of the internal explosive transfer system of the stinger subassembly shown in FIG. 1;

FIG. 3 is a detail cross-section view of an alternative embodiment of the probe portion of the stinger subassembly shown in FIG. 1, wherein the tip is disposable;

FIG. 4 is an axial cross-section view of a latch and release subassembly for an embodiment of a latch and release tool connector according to the invention;

FIG. 5 is a horizontal cross-section view through the line 5-5 of FIG. 4 showing the spring-loaded stop/release pads in more detail;

FIG. 6 is a horizontal cross-section view through the line 6-6 of FIG. 4 showing the collet fingers in more detail;

FIG. 7 is an axial cross-section view of latch and release subassembly shown in FIG. 4 in a running position for engaging the stinger subassembly according to FIG. 1;

FIG. 8 is an axial cross-section view of the latch and release subassembly shown in FIG. 4 in a latched position on the stinger subassembly according to FIG. 1; and

FIG. 9 is an axial cross-section view of the latch and release subassembly shown in FIG. 4 in a released position on the stinger subassembly according to FIG. 1.

The present invention will be described by referring to drawings of examples of how the invention can be made and used. Like reference characters are used throughout the several figures of the drawings to indicate like or corresponding parts.

The presently most preferred embodiment of the invention is a latch and release connector for use with perforating gun sections, which is one of the most challenging applications for the invention. It is to be understood, however, that the present invention can be employed to connect other types of downhole tools and simple tubing sections.

The structures of the stinger subassembly 10 shown in FIG. 1 will first be described in detail, and then the structures of the latch subassembly 100 shown in

FIG. 2. Thereafter, how the structures cooperate and are used to latch perforating gun sections with an ordinary slip assembly and a clamp or through a blowout preventer stack will be described in detail. Regarding the use with a blowout preventer stack, the stack is assumed to have lower seal/slip rams and upper operating rams.

STINGER SUBASSEMBLY

Referring now to FIG. 1, a stinger subassembly 10 according to the presently most preferred embodiment of the invention is shown in an axial cross-section view. In general, the stinger subassembly 10 has a probe portion 12, a slip landing portion 14, a bell connector portion 16, and a stinger internal explosive transfer system 18. According to the presently most preferred embodiment of the invention, the stinger subassembly is generally symmetrical about a stinger central axis A_1 .

In FIG. 1, the stinger subassembly 10 is shown with its central axis A_1 in a vertical orientation and such that the probe portion 12 is oriented upward. This illustrated orientation is how the stinger subassembly 10 would normally be oriented for use at the well head of a well. References to "upward," "downward," "above," "below," and other relative terms are understood to be with reference to the orientation of the stinger subassembly 10 shown in FIG. 1 of the drawing.

The stinger subassembly 10 is adapted to mate with the latch subassembly 100 shown in FIG. 2 of the drawing and as hereinafter described in detail.

Probe Portion of Stinger Subassembly

Referring to FIG. 1, the probe portion 12 of the stinger subassembly 10 preferably has tip 20, a probe first ramp surface 22, a shank surface 24, a probe second ramp surface 26, a probe recess 28, a probe first shoulder surface 30, a probe landing surface 32, a probe second shoulder surface 34, and a centralizer surface 36. Of the stinger overall axial length L_1 of the stinger subassembly 10, the probe portion 12 has an axial probe length L_2 .

According to the presently most preferred embodiment of the invention, the tip 20 presents a flat, circular surface that has a tip diameter D_1 . From the tip 20, the probe first ramped surface 22 is frusto-conical and expands in diameter downward along the axis A_1 from the tip 20 to the shank surface 24. This probe first ramp surface 22 faces upward and helps deflect and guide the probe portion 12 of the stinger subassembly 10 into the latch subassembly 100 as hereinafter described in detail. The shank surface 24 provides a structure for mating with the latch subassembly 100 and has a shank diameter D_2 .

Below the shank surface 24 is the probe second ramp surface 26, the probe recess 28, and probe first shoulder surface 30. According to the presently most preferred embodiment of the stinger subassembly 10 il-

illustrated in FIG. 1, the probe second ramp surface 26 is preferably frusto-conical and reduces in diameter downward along the axis A_1 from the shank surface 24. Thus, this probe second ramp surface 26 faces downward and helps deflect collet fingers of the latch sub-assembly 100 out of the recess 28 when the collet fingers are moved upward relative to the stinger sub-assembly 10 as will hereinafter be described in detail. According to the presently most preferred embodiment of the invention, the probe recess 28 is preferably a circumferential recess. Thus, the collet fingers can engage the probe recess 28 regardless of the relative rotational positions of the stinger subassembly 10 and the latch subassembly 100 as hereinafter described in detail. The circumferential probe recess 28 has a recess diameter D_3 . The probe first shoulder surface 30 faces upwards and defines the lower end of the recess 28.

Below the probe first shoulder surface 30 is the probe landing surface 32 and the probe second shoulder surface 34. According to the presently most preferred embodiment of the stinger subassembly 10 illustrated in FIG. 1, the probe landing surface 32 is cylindrical and adapted to fit within the lower portion of the housing of the latch subassembly 100 as hereinafter described in detail. The cylindrical probe landing surface 32 has a landing diameter D_4 . The probe second shoulder surface 34 faces upward and serves as a mechanical stop to the further insertion of the probe portion 12 of the stinger subassembly 10 into the housing of the latch subassembly 100 as hereinafter described in detail.

Below the probe second shoulder surface 34 is the centralizer surface 36. According to the presently most preferred embodiment of the stinger subassembly 10 illustrated in FIG. 1, the centralizer surface 36 is cylindrical having a centralizer diameter D_5 and is adapted to help centralize the stinger subassembly 10 within the tubulars of a well bore.

Slip Landing Portion of Stinger Subassembly

Continuing to refer to FIG. 1 of the drawing, the slip landing portion 14 of the stinger subassembly 10 is below the centralizer surface 36 of the probe portion 12. The slip landing portion 14 has a slip landing first shoulder surface 38, a slip landing surface 40, and a slip landing second shoulder surface 42. The slip landing portion 14 is preferably integrally formed with the probe portion 12 of the stinger subassembly. Of the overall length L_1 of the stinger subassembly, the slip landing portion 14 of the stinger subassembly 10 has an axial landing length L_3 .

The slip landing first shoulder surface 38 faces downwards and defines the upper end of the slip landing surface 40. The slip landing surface 40 is cylindrical having a slip landing diameter D_6 and is structurally adapted to be engaged and held by a slip assembly at the rig floor or the seal/slip rams of a blowout preventer as

hereinafter described in detail. The slip second shoulder surface 42 faces upwards and defines the lower end of the slip landing surface 40. The recessed slip landing surface 40 helps indicate a positive engagement of the seal/slip rams of a blowout preventer. However, it is to be understood that the slip landing surface 40 need not be recessed compared with the largest overall diameter of the stinger subassembly 10.

10 Bell Connector Portion of Stinger Subassembly

Continuing to refer to FIG. 1, the bell connector portion 16 of the stinger subassembly 10 is below the slip second shoulder surface 42 defining the lower end of the slip landing portion 14. The structure of the bell connector portion 16 can be of a standard form to adapt with correspondingly standard pin connectors on perforating gun sections. The bell connector portion 16 is preferably integrally formed with the slip landing portion 14 of the stinger subassembly. Of the overall length L_1 of the stinger subassembly, the bell connector portion 16 of the stinger subassembly 10 has an axial bell length L_4 .

According to the presently most preferred embodiment of the invention, the bell connector portion 16 is a generally tubular body symmetrical about stinger central axis A_1 and defining a cylindrical connector surface 44 having a bell diameter D_7 . The interior of the bell connector portion 16 has a bell sealing area 46, a female threaded bore section 48, and an end seat section 50 formed therein. The interior of the bell connector portion 16 is adapted for receiving and engaging a correspondingly threaded and structured male pin connector. For example, the bell sealing area 46 is adapted to provide a surface for compressing one or more O-ring seals on a correspondingly structured pin connector. The cooperation of the bell sealing area 46 with the corresponding structure and O-ring seals of a corresponding pin connector forms a pressure-tight seal. Thus, the bell connector portion 16 is structurally adapted to be made-up with the correspondingly structured and threaded male pin connector of a perforating gun connector (not shown). The bell diameter D_7 is normally also adapted to help centralize the stinger subassembly 10 within the tubulars of a well bore.

45 Stinger Internal Explosive Transfer System of Stinger Subassembly

Continuing to refer to FIG. 1 of the drawing, the stinger internal explosive transfer system 18 is preferably located centrally within the stinger subassembly 10. According to the presently most preferred embodiment of the invention, the stinger internal explosive transfer system 18 includes a stinger internal chamber 52 that extends from a first end 54 adjacent the tip 20 of the probe portion 12 through the probe portion, through the slip/seal ram landing portion 14, and into the bell connector portion 16 to a second end 56 adjacent the end

seat section 50 of the bell connector portion. The first end 54 of the stinger internal chamber 52 is sealed by the web material 58 defining the tip 20 of the probe portion 12. Positioned within the stinger internal tubular chamber 52 adjacent the first end 54 is a stinger booster charge 60. The booster charge is adapted to ignite a stinger detonating cord 62 positioned throughout substantially the entire length of the chamber 52. A stinger initiator section 64 is located at the second end 56 of the stinger internal chamber 52.

Referring now to FIG. 2 of the drawing, the stinger initiator section 64 of the stinger internal explosive transfer system 18 is shown in more detail. The section 64 is shown adjacent the threads 48 of the bell connector portion 16 of the stinger subassembly. According to the presently most preferred embodiment of the invention, the stinger initiator section 64 includes a firing pin housing 66 with initiator retainer 68 that are threaded into the second end 56 of the stinger internal chamber 52 and sealed with initiator O-ring seals 70 and 72. The end of the detonating cord 62 is provided with an end seal 74 adjacent the firing pin housing 66. A firing pin 76 is mounted within the firing pin housing 66 with shear pins 78. The firing pin 76 is adapted to be fired by the detonating cord 62 toward the stinger initiator 80. According to the invention, the initiator 80 is deformed, but not breached by the firing pin 76, thus, a seal between the interior of the bell connector portion 16 is maintained.

As will hereinafter be described in detail, the stinger internal explosive transfer system 18 is adapted to continue and transfer the detonation of the perforating charges from one perforating gun section, through the stinger subassembly 10, and to the next perforating gun section made-up with the bell connector portion 16 of the stinger subassembly 10. To help with the transfer of the detonation from the stinger subassembly 10 through the bell connector portion 16 to the next perforating gun section made up with the bell connector portion, the interior of the bell connector portion 16 is sealed against well fluids as previously described.

Alternative End Portion and Disposable End Cap for Stinger Subassembly

Referring to FIG. 3 of the drawing, according to an alternative embodiment of the present invention, an alternative structure is provided for a probe portion 12a of a stinger subassembly. The probe portion 12a includes an upper end portion 82, which is adapted to receive a disposable end cap 84.

The upper end portion 82 of the probe portion 12 of the stinger subassembly 10 has the first end 54 of the stinger internal chamber 52 formed therein. The stinger receiving initiator charge 60 is positioned within the first end 54 of the stinger internal chamber 52. The upper end portion 82 has male threads 86 formed thereon. Beneath the male threads 86 is formed an O-ring groove 88 adapted to receive and trap a sealing O-ring 90.

The disposable end cap 82 has outer surfaces 20a, 22a, and 24a that substantially conform to the surfaces 20, 22, and 24 previously described for the probe portion 12. The disposable end cap 82 also has an end web portion 58a that corresponds to the web portion 58 previously described for the probe portion 12. The body of the end cap 82 has a generally bell-shaped interior with a female threaded portion 92. The female threaded portion 92 of the end cap 82 is adapted to be threaded onto correspondingly male threaded portion 86 formed on the body of the probe portion 12a. Below the female threaded portion 92 is an end cap sealing surface 94, which is adapted to seal against the O-ring 90 positioned in the O-ring groove 88 when the end cap is threaded onto the probe portion 12a. Thus, the stinger subassembly 10 can be provided with a disposable end cap 82, thereby making the stinger subassembly reusable.

LATCH SUBASSEMBLY

Referring now to FIG. 4 of the drawing, a latch subassembly 100 according to the presently most preferred embodiment of the invention is shown in an axial cross-section view. In general, the latch subassembly 100 has a pin connector portion 102, a body portion 104, spring-loaded stop/release pads 106, a spring-loaded housing 108, collet fingers 110, and a latch internal explosive transfer system 112. According to the presently most preferred embodiment of the invention, the latch subassembly 100 is generally symmetrical about its central axis A_2 except as otherwise noted.

In FIG. 4, the latch subassembly 100 is shown with its central axis A_2 in a vertical orientation and such that the housing portion 106 is downward. This orientation is how the latch subassembly 100 would normally be oriented for use at the well head of a well. Again, references to "upward," "downward," "above," "below," and other relative terms are understood to be with reference to the orientation of the latch subassembly 100 shown in FIG. 4 of the drawing.

Pin Connector Portion of Latch Subassembly

Referring now to FIG. 4 of the drawing, the latch subassembly 100 is described and shown in detail. In particular, the pin connector portion 102 is at the upper end of the latch subassembly 100. The structure of the pin connector portion 102 can be of a standard form to adapt with correspondingly standard bell connectors on perforating gun sections. Of the overall length L_5 of the latch subassembly 100, the pin connector portion 102 of the latch subassembly has an axial pin length L_6 .

For the purposes of this description, it will be assumed that a corresponding bell connector portion of a perforating gun assembly (not shown) to be made up with the latch subassembly will have the same structure as the bell connector portion 16 previously described for the stinger subassembly 10. Thus, the pin connector

portion 102 is a generally tubular body symmetrical about latch axis A_2 and defining an end surface 114, a male threaded pin section 116, a pin ramped surface 118, pin sealing surfaces 120, pin O-ring grooves 122, a pin shoulder surface 124, and a connector centralizer surface 126. The pin connector portion 102 is adapted to be made up with a correspondingly structured and threaded bell connector portion of a perforating gun section. When the pin connector portion 102 and a corresponding bell connector portion of a perforating gun section are moved toward each other, the pin connector portion 102 is guided into the open end section of the bell connector portion. The male threaded pin section 116 is made up with the female threaded section of the corresponding bell connector portion. The pin ramped surface 118 helps guide the pin connector portion 102 into the open end section of the corresponding bell connector portion. The pin O-ring grooves 122 formed in the pin sealing surface 120 are adapted to receive O-rings for helping to seal the pin sealing surface 120 with the bell sealing area of a corresponding bell connector portion of a perforating gun section. The pin sealing surface 120 also helps in aligning the latch central axis A_2 of the latch subassembly and its pin connector portion 102 with the corresponding bell connector portion of a perforating gun section. The pin end surface 114 and pin shoulder surface 124 provide mechanical stops against over-tightening the threaded connection between the pin connector portion 102 and a corresponding bell connector portion of a perforating gun section. The connector centralizer surface 126 having a pin diameter D_8 is adapted to help centralize the latch subassembly 100 within the tubulars of a well bore.

According to the presently most preferred embodiment of the invention, the lower end of the bell connector portion 102 further has an inwardly facing shelf 128. As will hereinafter be described in detail, this shelf 128 helps in retaining the spring-loaded stop/release pads on the body portion 104.

Body Portion of Latch Subassembly

Continuing to refer to FIG. 4 of the drawing, the body portion 104 of the latch subassembly 100 is a structural member attached to the pin connector portion 102. The body portion 104 has an upper body portion 130 extending into the pin connector portion 102, a central body portion 132, and a lower body portion 134. The upper body portion 130 is for securely mounting the body portion 104 to the pin connector portion 102. As will hereinafter be described in detail, the spring-loaded stop/release pads 106 are connected to the central body portion 132, and the spring-loaded housing 108 and the collet fingers 110 are mounted to the lower body portion 134.

According to the presently most preferred embodiment of the invention, the upper body portion 130 is a structural member in the general form of a cylindrical

mandrel or other solid structural member adapted for connecting to the pin connector portion 102 of the latch subassembly 100. The upper body portion has a male threaded section adapted to be threaded into corresponding female threads formed in the pin connector portion 102.

According to the presently most preferred embodiment of the invention, the central body portion 132 is a structural member having a generally cylindrical structure with an overall central body diameter D_9 . The central body portion 132 is preferably integrally formed with the upper body portion 130. The overall central body diameter D_9 is less than the connector centralizer diameter D_8 of the pin connector portion 102 to allow the spring-loaded stop/release pads 106 to be mounted to the outside of the central body portion 132. Nevertheless, the spring-loaded stop/release pads 106 still present an overall profile for the latch subassembly 100 that is not greater than the connector centralizer diameter D_8 . Thus, the latch subassembly 100 can pass through downhole tubing of a desired size.

A plurality of alignment bores are formed in the central body portion 132, such as the illustrated two alignment bores 136a and 136b. Each of the alignment bores is preferably a cylindrical bore formed in the central body portion 132 and oriented radially about the latch central axis A_2 . As will hereinafter be described in detail, the alignment bores 136a-b are adapted to help maintain the stop/release pads 106 on the central body portion 132. Two additional alignment bores (not shown) are preferably radially oriented 180 degrees from each other and 90 degrees from the alignment bores 136a and 136b, respectively. Thus, a total of four alignment bores are radially spaced apart 90 degrees about the latch central axis A_2 . A plurality of spring bores are formed in the central body portion 132, such as the illustrated two upper spring bores 138a-b and the two lower spring bores 140a-b illustrated in FIG. 4. Each of the spring bores 138a-b and 140a-b is preferably a cylindrical bore formed in the central body portion 132 and oriented radially about the latch central axis A_2 . The upper spring bores 138a-b are each adapted to receive an upper spiral spring 142 therein, and the lower spring bores 140a-b are similarly each adapted to receive a similar spiral spring 144 therein.

The two upper spring bores 138a and 138b are preferably radially opposed 180 degrees about the latch central axis A_2 as shown in FIG. 4. Thus, the upper spiral springs 142 positioned in these two upper spring bores can be loaded to exert opposed radial forces. Two additional upper spring bores (not shown) are preferably radially oriented 180 degrees from each other and 90 degrees from the upper spring bores 138a and 138b, respectively. Thus, a total of four upper spring bores are radially spaced apart 90 degrees about the latch central axis A_2 . As will hereinafter be described in detail, each of the four upper spiral springs 142 (only two shown in FIG. 4) mounted in the upper spring bores can be loaded

to exert a force opposed to another upper spiral spring 142 mounted in a radially opposed upper spring bore.

Similarly, the two lower spring bores 140a and 140b are preferably radially opposed 180 degrees about the latch central axis A_2 as shown in FIG. 4. Two additional lower spring bores (not shown) are preferably radially oriented 180 degrees from each other and 90 degrees from the lower spring bores 140a and 140b, respectively. Thus, a total of four lower spring bores are radially spaced apart 90 degrees about the latch central axis A_2 . As will hereinafter be described in detail, each of the four lower spiral springs 144 (only two shown in the FIG. 4) mounted in the lower spring bores are loaded to exert a force opposed to another lower spiral spring 144 mounted in a radially opposed lower spring bore.

According to the presently most preferred embodiment of the invention, the lower body portion 134 is a structural member having a generally cylindrical structure with a lower body diameter D_{10} . The lower body portion 134 is secured to the central body portion 132.

The lower body portion 134 has a collar portion 146, which is preferably integrally formed thereon. The collar portion 146 defines an upwardly facing collar shoulder surface 148. As will hereinafter be described in detail, the collar shoulder surface 148 helps in mounting the spring-loaded housing 108 to the lower body portion 134. Furthermore, the collar portion 146 provides added structural material for helping in connecting the spring-loaded housing 108 thereto.

The bottom end of the lower body portion 134 defines a generally bell-shaped opening 150. As will hereinafter be described in detail, the bell-shaped opening 150 is adapted to receive the probe tip 20 and the probe first ramped surface 22 of the probe portion 12 of the stinger subassembly 10.

Further according to the presently most preferred embodiment of the invention, the bottom end of the lower body portion 134 adjacent the bell-shaped opening 150 has the collet fingers 110 connected thereto.

The lower body diameter D_{10} is preferably substantially the same as the overall central body diameter D_9 for central body portion 132. The lower body diameter D_{10} of the lower body portion 134 is less than the connector centralizer diameter D_8 of the pin connector portion 102 to allow the spring-loaded housing 108 to be mounted to the outside of the lower body portion 134. Nevertheless, the spring-loaded housing still presents an overall profile for the latch subassembly 100 that is not greater than the connector centralizer diameter D_8 . Thus, the latch subassembly 100 can pass through downhole tubing of a desired size. Similarly, the diameter of the collar portion 146, although greater than the lower body diameter D_{10} , is still less than the connector centralizer diameter D_8 of the pin connector portion 102. This smaller diameter allows the spring-loaded housing 108 to be mounted to the outside of the lower body portion 134 yet still present an overall profile for the latch subassembly 100 that is not greater than the connector

centralizer D_8 . Thus, the latch subassembly 100 can pass through downhole tubing of a desired size.

Spring-Loaded Stop/Release Pads of Latch Subassembly

Referring now to FIGS. 4 and 5 of the drawing, the spring-loaded stop/release pads 106 are mounted to the central body portion 132. Of the overall length L_5 of the latch subassembly 100, the spring-loaded stop/release pads 106 have an axial pads length L_7 .

According to the presently most preferred embodiment of the invention, the structure of the spring-loaded stop/release pads 106 is based on a tubular structure divided into four identical portions, as represented in the drawing by the two pads 152a and 152b shown in FIG. 4. All four of the pads 152a-d are shown in FIG. 5. Together, the four pads of the spring-loaded stop/release pads 106 present an overall pads diameter D_{11} . The overall pads diameter D_{11} of the spring-loaded stop/release pads 106 is not greater than the connector centralizer diameter D_8 of the pin connector portion 102. Thus, the latch subassembly 100 can pass through downhole tubing of a desired size. As best shown in FIG. 5, the four pads 152a-d are positioned on the central body portion 132 over the radially oriented springs, such as upper springs 142. Thus, the springs 142 exert radially outward forces on the pads 152a-d.

The upper end of each of the pads, as shown in FIG. 4 for the two pads 152a and 152b, also includes a peg 154a and 154b, respectively, adapted to fit within any of the four alignment bores, such as illustrated in FIG. 4 for the alignment bores 136a and 136b. Thus, the pegs help in retaining the vertical position of the pads on the central body portion 132.

Further according to the presently most preferred embodiment of the invention, the upper end of each of the pads, as shown in FIG. 4 for the two pads 152a and 152b, extend into the shelf 128 of the pin connector portion 102. This helps in retaining the pads against the springs 142 and 144. As shown in FIG. 4, in the lower end of each of the pads, as shown for the pads 152a and 152b, is formed a shallow recess 156a and 156b, respectively. The shallow recesses are identically positioned on each of the pads such that when the four pads are positioned about the central body portion 132, the recesses define an at least partially circumferential recess. Thus, the recesses are adapted to position a tubular collar 158 over the lower end of the pads 152a-d. The cooperation of the shallow recesses with the tubular collar 158 retains the four pads, represented by pads 152a and 152b, against the upper springs 142 and lower springs 144. Thereby, the four pads are spring-loaded to the central body portion 132.

To assemble the spring-loaded stop/release pads onto the central body portion 132, the body portion 104 is separated from the bell connector portion 102. The plurality of upper springs 142 are positioned in the upper

spring bores 138a-d of the central body portion 132 as shown in FIGS. 4 and 5, and the plurality of lower springs 144 are positioned in the lower spring bores of central body portion, as shown in FIG. 4 for lower spring bores 140a-b. The pads 152a-d are then positioned over the central body portion 132, such that the peg 154 of each pad is positioned in one of the alignment bores, as shown in FIG. 4 for alignment bores 136a-b. The tubular collar 158 is positioned over the pads as shown in FIG. 4 to restrain them against the upper springs 142 and lower springs 144. The upper body portion 130 of the body portion 104 is then secured to the bell connector portion 102 such that the upper ends of the pads are restrained against the upper springs 142 and lower springs 144 as shown in FIG.4.

Spring-Loaded Housing of Latch Subassembly

Continuing to refer to FIG. 4 of the drawing, the spring-loaded housing 108 is mounted on the lower body portion 134. The overall housing diameter D_{12} of the spring-loaded housing 108 is not greater than the pin centralizer diameter D_8 , whereby the latch subassembly 100 can pass through downhole tubing of a desired size. When the spring-loaded housing 108 is set and ready for use as illustrated in FIG. 4 of the drawing, the housing 108 is spaced apart from the lower end of the spring-loaded stop/release pads 106 by an axial spacing length L_8 . As will hereinafter be described in detail, however, the spring-loaded housing 108 is adapted to be axially moved upward on the lower body portion 134, first to close the axial spacing length L_8 , and then to overlap with the lower end of the spring-loaded stop/release pads 106. Of the overall length L_5 of the latch subassembly 100 when it is in the set position shown of FIG.4, the spring-loaded housing 108 has an axial length L_9 .

According to the presently most preferred embodiment of the invention, the spring-loaded housing 108 includes a substantially tubular housing member 160 adapted to slide over the lower body portion 134. As will hereinafter be described in more detail, the tubular housing member 160 is preferably formed in two sections, an upper housing portion 160a and a lower housing portion 160b. The tubular housing member 160 has an inner diameter that is larger than the lower body diameter D_{10} of the lower body portion 134, but adapted to slide over the collar portion 146 of the lower body portion 134. Thus, there is a first annular space 162 defined between the lower body diameter D_{10} of the lower body portion 134 and the inner diameter of the tubular housing member 160 of the spring-loaded housing 108. The upper end of the first annular space 162 is open. The tubular member 160 has an inwardly facing flange 164 that can slide with the tubular member 160 along the lower body portion 134 and defines the lower end of the first annular space 162. As will hereinafter be described in detail, the first annular space 162 is adapted to move

over the lower ends of the four pads 152a-d when the pads are radially compressed against the springs 142 and 144 such that the pads 152a-d present a smaller diameter profile.

The flange 164 defines the upper end of a second annular space 166. The lower end of the second annular space 166 is defined by the upwardly facing collar shoulder surface 148 on the collar portion 146 of the lower body portion 134. The housing spring 168, which is trapped at its lower end by the upwardly facing collar shoulder surface 148 of the collar portion 146, exerts an upward force against the flange 164 of the tubular housing member 160. This upward force exerted by the spring 168 is parallel to the latch central axis A_2 .

One or more retaining pins, such as screws 170 are tapped or threaded through the tubular housing member 160 and into the collar portion 146 of the lower body portion 134. Thus, the retaining screws 170 retain the tubular housing member over the lower body portion 134 against the force of the housing spring 168 positioned within the second annular space 166.

The lower end of the tubular housing member 160 has an inwardly facing deflecting structure 172, which is for engaging the collet fingers 110 with the stinger subassembly 10 as will hereinafter be described in detail. According to the presently most preferred embodiment of the invention, the deflecting structure 172 has a deflecting first ramped surface 174, an engaging surface 176, and a deflecting second ramped surface 178. The deflecting first ramped surface 174 is frusto-conical and reduces in diameter downward along the axis A_2 of the latch subassembly 100. The engaging surface 176 defines an inner cylindrical wall below the deflecting first ramped surface 174. The deflecting second ramped surface 178 is frusto-conical and expands in diameter downward along the axis A_2 of the latch subassembly 100.

As previously mentioned, according to the presently most preferred embodiment of the invention, the tubular housing member 160 is preferably formed into two portions, upper housing portion 160a and lower housing portion 160b. The upper housing portion 160a and the lower housing portion 160b are threaded together and retained with one or more set screws 180. This separable housing structure permits the latch assembly 100 to be more easily assembled. For example, the lower body portion 134 is removed from the central body portion 132, so that the upper housing portion 160a can be placed over the lower body portion 134 from its upper end. Otherwise, if the lower housing portion 160b were integrally formed with the upper housing portion 160a, the deflecting structure 172 would not slide over the diameter of the collar portion 146 on the lower body portion 134.

Finally, according to the presently most preferred embodiment of the invention, a housing snap-ring seal 181 is provided between the lower body portion 134 and the tubular housing member 160 to prevent the housing

from moving downward and accidentally releasing while running into and out of the well. The snap-ring 181 expands beyond the inside diameter of the pin threads on housing 160a.

To assemble the spring-loaded housing 108 onto the lower body portion 134, the lower body portion 134 is separated from the central body portion 132. The housing spring 168 is positioned over the lower body portion 132 and slid downward until it is stopped by the upwardly facing collar shoulder surface 148 on the collar portion 146 of the lower body portion 134. The upper housing portion 160a is then positioned over the lower body portion 132 and slid downward such that the inwardly facing flange 164 compresses the spring 168 as shown in FIG. 4. The one or more retaining screws 170 are tapped or threaded through the tubular housing member 160 and into the collar portion 146 of the lower body portion 134. Thus, the retaining screws 170 retain the tubular housing member over the lower body portion 134 against the force of the housing spring 168 positioned within the second annular space 166. The lower housing portion 160b is slid upward from the lowermost end of the lower body portion 134. Then the lower housing portion 160b is threaded to the upper housing portion 160a and retained with one or more set screws 180.

Collet Fingers of Latch Subassembly

Continuing to refer to FIG. 4 of the drawing, the collet fingers 110 of the latch subassembly 100 are attached to the lower body portion 134. At least two collet fingers 110, such as the first and second collet fingers 182a and 182b are employed. However, it is to be understood that additional collet fingers can be used, which may be particularly desirable for a larger latch subassembly for use in larger downhole tubing applications. The arcuate extension of each of the collet fingers 182a and 182b is a matter of design choice, and is expected to range up to nearly 90 degrees of radial arc about the latch axis A_2 . Thus, if desired, four or more collet fingers 110 can be employed in the latch subassembly 100. According to the presently most preferred embodiment, as shown in FIG. 6 of the drawing of the invention, six collet fingers 182a-f are employed. Referring back to FIG. 4 of the drawing, each of the individual collet fingers, as represented by collet fingers 182a and 182b, has a dog portion 184 and a finger tip portion 186.

The upper end of the dog portion 184 of each collet finger 182a-b is an extension of the lower body portion 134. The dog portion 184 is adapted to be sufficiently deformable to be deflected inward or outward relative to the relaxed position shown in FIG. 4 of the drawing. Alternatively, the dog portion 184 of each collet finger 182a-b can be pivotally mounted to the lower body portion 134 adjacent the bottom of the bell-shaped opening 150.

According to the presently most preferred embodi-

ment of the invention, the finger tip portion 186 of each of the collet fingers 182a-b has a plurality of surfaces adapted to be deflected by and engage with other surfaces of the stinger subassembly 10 and the latch subassembly 100. In particular, the finger tip portion of each of the collet fingers 182a-b has a first outwardly facing ramped surface 188, an outwardly facing vertical surface 190, a second outwardly facing ramped surface 192, a first inwardly facing ramped surface 194, an inwardly facing vertical surface 196, and a second inwardly facing ramped surface 198. The cooperation of these surfaces 188-198 with other surfaces and structures will hereinafter be described in more detail.

Latch Internal Explosive Transfer System

Continuing to refer to FIG. 4 of the drawing, the latch internal explosive transfer system 112 is preferably located centrally within the latch subassembly 100. According to the presently most preferred embodiment of the invention, the latch internal explosive transfer system 112 includes a latch internal chamber 200. The chamber 200 extends from a first end 202 adjacent the end surface 114 of the pin connector portion 102 and through the entire body portion 104 to a second end 204 adjacent the bell-shaped opening 150 of the lower body portion 134. Positioned within the latch internal chamber 200 adjacent the first end 202 is a latch receiving booster charge 206. A latch detonating cord 208 is positioned through substantially the entire length of the chamber 200. A latch booster charge 210 and a downward focused shaped charge 212 are positioned in the chamber 200 adjacent the second end 204 of the chamber 200. As will hereinafter be described in detail, the latch internal explosive transfer system 112 is adapted to continue and transfer the detonation of the perforating charges from one perforating gun section made-up with the pin connector portion 102 of the latch subassembly 100, through the latch subassembly 100, and to a stinger subassembly 10 latched to the latch subassembly 100. As previously mentioned, the stinger subassembly 10 in turn continues and transfers the detonation to the next perforating gun section made-up with the bell connector portion 16 of the stinger subassembly 10.

Method of Using Latch and Release Perforating Gun Connector

Referring now to FIG. 7 of the drawing, the stinger subassembly 10 is shown as it is positioned when the slip landing surface 40 of the slip landing portion 14 are held by the seal/slip rams of a blowout preventer (not shown). For the purposes of this description, the stinger subassembly 10 has already been made up with a lower perforating gun section (not shown), which has been inserted through the blowout preventer seal/slip rams. The latch subassembly 100 has been made up with an upper perforating gun section (not shown), which has

been moved into a lubricator above the blowout preventer. The upper perforating gun section with the latch subassembly 100 at the lower end thereof is then lowered through the blowout preventer onto the probe portion 12 of the stinger subassembly 10. The latch subassembly 100 is lowered until the deflecting structure 172 of the spring-loaded housing 108 is stopped by the second shoulder surface 34 above the centralizer surface 36 of the stinger subassembly 10, as shown in FIG. 7.

In this running position illustrated in FIG. 7, the tip 20 of the probe portion 12 of the stinger subassembly 10 is slightly spaced apart from the upper end of the bell-shaped opening 150 formed in the lower body portion 134. In this running position, the finger tip portion 186 of each of the individual collet fingers 182a and 182b can at least partially begin to be deflected into the recess 28 of the probe portion 12 on the stinger subassembly 10. As can be seen in FIG. 7, the housing spring 168 is trapped in the second annular space 166 defined by the lower body portion 134, the tubular housing member 160, and the flange 164. As previously described, the potential energy of the housing spring 168 is retained by the retaining screws 170 threaded through the tubular housing portion 160 into the collar portion 146 of the lower body portion 134.

At this point, a downward force is applied to the latch subassembly 100. This force is transmitted axially through the latch subassembly 100 to the lower body portion, through the retaining screws 170, through the spring-loaded housing 108 at the deflecting structure 172 to the second shoulder surface 34 above the centralizer surface 36 of the stinger subassembly 10. A sufficiently strong downward force is applied to the latch subassembly that the retaining screws 170 are sheared between tubular housing member 160 and the lower body portion 134. Once the retaining screws 170 have been sheared, the tubular housing member 160 is released from the lower body portion 134. Thus, the housing spring 168, which is trapped between the surface 148 of the collar portion 146 of the lower body portion 134 and the flange 164 of the tubular housing member 160, is now free to drive the slidably mounted tubular housing body 160 upward on the lower body portion 134.

Referring now to FIG. 8 of the drawing, the latch subassembly 100 is shown in a latched position on the stinger subassembly 10. Each of the retaining screws 170 are shown as having been sheared into two portions. An outer portion 170a of the sheared retaining screw travels with the upwardly moving tubular housing member 160. An inner portion 170b of the sheared retaining screw remains with the collar portion 146 of the lower body portion 134. The upward movement of the tubular housing member 160 on the lower body portion 134 permits the latch subassembly 100 to settle onto the tip 20 of the probe portion 12 of the stinger subassembly 10. Driven by the released housing spring 168, the tubular housing member 160 moves upward on

the lower body portion 134 until it is stopped by the pads, such as pads 152a-b, of the spring-loaded stop/release pads 106. At this point, the potential energy of the housing spring 168 is only partially released in driving the tubular housing member 160 upward. The upward movement of the tubular housing member 160 also causes the deflecting structure 172 to force and deflect the collet fingers inward. More particularly, the deflecting first ramped surface 174 of the deflecting structure 172 engages the second outwardly facing ramped surface 192 of the finger tip portion 186 inward. Thus, the finger tip portion 186 of each of the collet fingers 182a and 182b are deflected into the probe recess 28 of the probe portion 12 of the stinger subassembly 10. The various surfaces on the probe portion 12 of the stinger subassembly and the deflecting structure 172 of the tubular housing member cooperate to trap the finger tip portions 186 of the collet fingers 182a-b in the probe recess 28. Thus, the latch subassembly 100 is securely latched onto the probe portion 12 of the stinger subassembly. This process of latching the latch subassembly 100 to the stinger subassembly 10 can be accomplished in a matter of seconds.

The stinger subassembly 10 and the latch subassembly 100 form a completed connection between the lower and upper perforating gun sections (not shown). The perforating gun sections can then be lowered downhole to perforate the well.

It is to be understood, of course, that additional perforating gun sections can be successively added to the string using successive additional pairs of stinger subassemblies 10 and latch subassemblies 100.

Furthermore, according to the presently most preferred embodiment of the invention, a detonating signal can be transmitted from the latch subassembly 100 to the stinger subassembly 10. Referring back to FIG. 4 of the drawing, a detonating signal is transmitted from an upper perforating gun to the latch internal explosive transfer system 112 of the latch subassembly 100. The detonating signal from the upper perforating gun detonates the latch receiving booster charge 206. The booster charge 206 in turn ignites the latch detonating cord 208 positioned within the latch internal chamber 200. The latch detonating cord 208 transfers the detonating signal to the latch booster charge 210, which detonates the latch downward focused shaped charge 212. The shaped charge 212 pierces the web material of the lower body portion 134 below the second end 204 of the chamber 200 and fires through the stinger tip web 58 of the stinger subassembly 10 that is latched to the latch subassembly 100.

Referring again to FIG. 8 of the drawing, which shows the latch subassembly 100 in a latched position on the stinger subassembly 10, the tip 20 of the probe 12 of the stinger subassembly 10 is preferably flush with the inner surface of the bell-shaped opening 150 of the lower body portion 134 of the latch subassembly 100. The latch shaped charge 212 pierces through the thick-

ness of the web material 58 defining the tip 20 of the probe portion 12. The latch downward focused shaped charge 212 is adapted to pierce the tip 20 of the sub-assembly 10. According to the previously described alternative embodiment of the stinger subassembly with respect to FIG. 3 of the drawing, the latch downward focused shape charge 212 pierces the disposable end cap 84.

Referring back to FIG. 1 of the drawing, which shows the stinger subassembly 10 in detail, piercing the web material 58 defining the tip 20 of the probe portion 12 initiates the stinger internal explosive transfer system 18. More particularly, the latch shaped charge 212 pierces the material to initiate the stinger booster charge 60. The stinger booster charge 60 in turn ignites the stinger detonating cord 62 within the stinger internal chamber 52. The stinger detonating cord 62 transfers the detonating signal to the stinger initiator section 64, best shown in FIG. 2. The firing pin 76 mounted within the firing pin housing 66 is fired by the detonating cord 62 toward the stinger initiator 80. According to the invention, the initiator 80 is deformed, but not breached by the firing pin 76, thus, a seal between the interior of the stinger internal chamber 52 and the bell connector portion 16 is maintained. The deforming material of the initiator drives downward to detonate the initiator. This detonation of the initiator initiates a booster charge in a perforating gun section connected to the bell connector portion 16 of stinger subassembly 10. Thus, the detonating signal is transferred from the stinger subassembly 10 to a booster charge and detonating cord in the lower perforating gun section (not shown). The detonating cord in the lower perforating gun section serially detonates the perforating charges in that perforating gun section.

If a plurality of perforating gun sections are connected using the stinger subassembly 10 and latch subassembly 100, the detonating signal is carried through the successive connections as described herein.

After the perforating gun sections have been detonated downhole to perforate the well, they are raised back toward the well head. The second (upper) perforating gun section is raised through the blowout preventer stack until the slip landing portion 14 of the stinger subassembly 10 aligns with the seal/slip rams of the blowout preventer stack. The seal/slip rams of the blowout preventer stack are engaged to seal and hold the perforating gun section string at the stinger subassembly 10. Since the integrity of the stinger subassembly 10 has been maintained, the latch subassembly 100 can be removed from the stinger subassembly 10 without allowing any fluid to escape through the seal/slip rams of the blowout preventer stack.

According to the presently most preferred embodiment of the invention, a clamp or the operating rams of another blowout preventer above the seal/slip rams in the blowout preventer stack are employed to release the latch subassembly 100 from the stinger subassembly

10. As used herein, the term "operating" rams refers to any of a number of different types of rams that are usually employed above the seal/slip rams, except shearing or other type rams that would undesirably damage the latch subassembly. Referring to FIG. 8, the operating rams engage the spring-loaded stop/release pads 106 and radially compress the pads 152a-b toward the latch central axis A_2 . This compressing force opposes the radially outward force of springs 142 and 144 and deflects the pads 152a-d inward toward the central body portion 132. Thus, the effective diameter of the spring-loaded stop release pads 106 is reduced. Meanwhile, the tubular housing member 160 is still being acted upon by the housing spring 168 trapped within the second annular space 166. Thus, once the spring-loaded stop release pads 106 are sufficiently compressed, the open end of the tubular housing member 160 can slide upward over the pads 152a-d.

Referring now to FIG. 9 of the drawing, the latch subassembly is shown in a released position. The housing spring 168 maintains the tubular housing member 160 over the pads 152a-d, which retains them in the reduced diameter form against the opposing forces of the springs 142 and 144 of the spring-loaded latch pads 106. The further upward movement of the tubular housing member 160 also causes the deflecting structure 172 to move upward. This releases the finger pads 186 of the collet fingers 182a-b, such that the latch subassembly 100 can be lifted off the probe portion 12 of the stinger subassembly 10. More particularly, as the latch subassembly 100 is lifted upward, the probe second ramp surface 26 deflects the second inwardly facing ramped surface 188 of the finger tip portion 186 of each of the collet fingers 182a-b. Thus, the finger tip portion 186 of each of the collet fingers 182a-b is deflected out of the probe recess 28 of the probe portion 12 of the stinger subassembly 10. This process of releasing the latch subassembly 100 from the stinger subassembly 10 can be accomplished within a few seconds. Throughout the process, the integrity of the blowout preventer stack pressure seal at the well head can be maintained.

An Example of Latch and Release Gun Connector for Use Through 5-Inch Tubing

Of course, the particular dimensions of the stinger subassembly 10 and latch subassembly 100 according to this invention are a matter of engineering design choice depending on many parameters. Such parameters, include, for example, the particular size of the well tubing and casing in which the stinger subassembly is to be used. The stinger subassembly 10 and latch subassembly 100 can be designed, for example, for use in 5-inch tubing. However, this illustrative example is for the purposes of more fully describing the presently most preferred embodiment of the invention, but not to limit the invention to the particular dimensions of such a disclosed preferred embodiment.

Accordingly, referring back to FIG. 1 of the drawing, the stinger subassembly 10 can have, for example, the following basic dimensions: an overall axial stinger length L_1 of about 24 inches (61 cm), an axial probe length L_2 of about 10 inches (26 cm); an axial landing length L_3 of about 10 inches (26 cm); an axial bell length L_4 of about 5 inches (13 cm); a tip diameter D_1 of about 1 inches (2.5 cm); a shank diameter D_2 of about 2 inches (5 cm); a recess diameter D_3 of about 1.5 inches (4 cm); a probe landing diameter D_4 of about 2.5 inches (6.5 cm); a centralizer diameter D_5 of about 3.5 inches (9 cm); a slip landing diameter D_6 of about 3 inches (8 cm); and a bell diameter D_7 of about 3.5 inches (9 cm).

Referring again to FIG. 4 of the drawing, the latch subassembly 100 can have, for example, the following basic dimensions: an overall axial latch length L_5 of about 30 inches (76 cm); an axial pin length L_6 of about 8 inches (20 cm); an axial pads length L_7 of about 9 inches (22 cm); an axial spacing length L_8 of about 1.2 inches (3 cm); an axial housing length L_9 of about 12 inches (30 cm); a pin diameter D_8 of about 3.5 inches (9 cm); an overall central body diameter D_9 of about 3.2 inches (8 cm); a lower body diameter D_{10} of about 2.2 inches (5.5 cm); an overall pads diameter D_{11} of about 3.2 inches (8 cm); and an overall housing diameter D_{12} of about 3.5 inches (9 cm).

The embodiments shown and described above are only exemplary. For example, the preferred embodiment for the spring-loaded housing is representative of a structure for storing potential energy for moving the housing. It will be appreciated that modifications may be made within the scope of the dependent claims.

Claims

1. A tool connector comprising: a stinger (10); a stinger receptacle (100), the stinger (10) being adapted to be stabbed into the stinger receptacle (100); a loaded engaging member movable between a running position before the stinger (10) is stabbed into the stinger receptacle (100) and a latched position when the stinger (10) is stabbed into the stinger receptacle (100) to latch the stinger (10) and the stinger receptacle (100) together; and a release member retaining the loaded engaging member in the running position, whereby when the stinger (10) is stabbed into the stinger receptacle (100) and a set force is applied to the stinger (10) and stinger receptacle (100), the release member releases the loaded engaging member to move to the latched position and latch the stinger (10) and the stinger receptacle (100) together.
2. A perforating gun connector comprising: a stinger (10); a stinger receptacle (100), the stinger (10) being adapted to be stabbed into the stinger receptacle (100); a loaded engaging member movable be-

tween a running position before the stinger (10) is stabbed into the stinger receptacle (100) and a latched position when the stinger (10) is stabbed into the stinger receptacle (100) to latch the stinger (10) and the stinger receptacle (100) together; and a release member retaining the loaded engaging member in the running position, whereby when the stinger (10) is stabbed into the stinger receptacle (100) and a set force is applied to the stinger (10) and stinger receptacle (100), the release member releases the loaded engaging member to move to the latched position and latch the stinger (10) and the stinger receptacle (100) together.

3. A connector according to Claim 1 or 2, further comprising: a releasable stop member to stop the engaging member in the latched position, whereby when the stop member is released, the engaging member moves to a released position such that the stinger (10) and stinger receptacle (100) are separable.
4. A connector according to Claim 2, wherein the releasable stop member is adapted to be released by a clamp.
5. A connector according to Claim 1, 2, 3 or 4, wherein the loaded engaging member is mounted to the stinger receptacle.
6. A connector according to any preceding claim, wherein the stinger (10) comprises a stinger subassembly (10) having:

- (i) a probe portion (12); and
- (ii) a stinger connector portion (16) connected to the probe portion (12), the stinger connector portion (16) being adapted to make up the stinger subassembly (10) with a first tool or perforating gun section; and

wherein the stinger receptacle (100) comprises a latch subassembly (100) having:

- (i) a body portion (104), the body portion (104) being adapted to receive the probe portion (12) of the stinger subassembly (10);
- (ii) a latch connector portion (102) connected to the body portion (104), the latch connector portion (102) being adapted to make up the latch subassembly (100) with a second tool or perforating gun section;
- (iii) a spring-loaded housing (108) mounted to slide on the body portion (104), the spring-loaded housing (108) adapted to be urged from a running position to a latched position, the spring-loaded housing (108) having at least one retaining pin (170) to restrain the spring-

loaded housing (108) in the running position on the body portion (104) until the retaining pin (170) is sheared; and

(iv) collet fingers (110) connected to the body portion (104) substantially within the spring-loaded housing (108), the spring-loaded housing (108) having a deflecting structure (172) adapted to deflect the collet fingers (110) to engage the probe portion (12) of the stinger sub-assembly (10),

whereby in the running position, the spring-loaded housing (108) of the latch subassembly (100) is retained in a running position for being moved onto the probe portion (12) of the stinger subassembly (10); and whereby when a set force is applied to the latch subassembly (100) against the stinger sub-assembly (10), the set force shears the retaining pin (170) to release the spring-loaded housing (108), which is urged on the body portion (104) toward the latched position such that the deflecting structure (172) of the spring-loaded housing (108) deflects the collet fingers (110) to engage the probe portion (12) of the stinger subassembly (10) and retains the collet fingers (110) in latched engagement with the probe portion (12).

7. A connector according to Claim 6, wherein the stinger subassembly (10) further comprises a stinger internal explosive transfer system (18).

8. A connector according to Claim 6, wherein the latch subassembly (100) further comprises an internal explosive transfer system (112) which comprises:

(a) a latch internal chamber (200) extending through the latch connector portion (102) and through the body portion (104);

(b) a latch receiving booster charge (206) positioned in the latch internal chamber (200) adjacent the latch connector portion (102);

(c) a latch detonating cord (208) positioned substantially throughout the length of the latch internal chamber (200);

(d) a latch sending booster charge (210) positioned in the latch internal chamber (200) adjacent the body portion (104); and

(e) a latch shaped charge (212) positioned in the latch internal chamber (200) distal to the latch sending booster charge (210);

whereby the latch internal explosive transfer system (112) is adapted to continue and transfer the detonation of perforating charges from the first tool or perforating gun section, through the latch sub-assembly (100), and to fire the latch shaped charge (212) toward the probe portion (12) of the stinger subassembly (10).

9. A connector according to Claim 8, wherein the stinger subassembly (10) further comprises a stinger internal explosive transfer system (18) which comprises:

(a) a stinger internal chamber (52) extending from adjacent a tip end (54) of the probe portion (12) and to the stinger connector portion (16);

(b) a stinger booster charge (60) positioned in the stinger internal chamber (52) adjacent the tip end (54) of the probe portion (12);

(c) a stinger detonating cord (62) positioned substantially throughout the length of the stinger internal chamber (52);

(d) a stinger firing pin (76) positioned in the stinger internal chamber (52); and

(e) a stinger initiator (80);

whereby the stinger internal explosive transfer system (18) is adapted to be detonated by the latch internal explosive transfer system (112) to continue and transfer the detonation of perforating charges from the latch subassembly (100), through the stinger subassembly (10), and to the second tool section or perforating gun section made up with the stinger connector portion (16) of the stinger sub-assembly (10).

10. A connector according to any one of Claims 1 to 5, wherein the stinger (10) comprises a stinger sub-assembly (10) having:

(i) a probe portion (12); and

(ii) means for connecting the probe portion (12) to a first tool or perforating gun section; and

wherein the stinger receptacle (100) comprises a latch subassembly (100) having:

(i) a body portion (104), the body portion (104) being adapted to receive the probe portion (12) of the stinger subassembly (10);

(ii) means for connecting the body portion (104) to a second tool or perforating gun section;

(iii) spring-loaded housing (108) mounted to slide on the body portion (104), the spring-loaded housing (108) adapted to be urged from a running position to a latched position;

(iv) means for retaining the spring-loaded housing (108) in the running position on the body portion (104); and

(v) means for latching the body portion (104) to the probe portion (12) of the stinger subassembly (10) when the spring-loaded housing (108) is moved to the latched position;

whereby when a set force is applied to the latch subassembly (106) against the stinger subassembly

(10), the set force shears the means for retaining the spring-loaded housing (108) such that the spring-loaded housing (108) is urged on the body portion (104) to the latched position.

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11. A method of connecting a first tool section to a second tool section, the method comprising the steps of:

- (a) connecting a stinger (10) to the first tool section; 10
- (b) connecting a stinger receptacle (100) to the second tool section;
- (c) stabbing the stinger (10) to mate with the stinger receptacle (100); 15
- (d) applying a set force to the stinger (10) and stinger receptacle (100) to release a loaded engaging member from a running position to a latched position to latch the stinger (10) and the stinger receptacle (100) together. 20

12. A method of connecting a first perforating gun section to a second perforating gun section, the method comprising the steps of:

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- (a) connecting a stinger (10) to the first perforating gun section;
- (b) connecting a stinger receptacle (100) to the second perforating gun section;
- (c) stabbing the stinger (10) to mate with the stinger receptacle (100); 30
- (d) applying a set force to the stinger (10) and stinger receptacle (100) to release a loaded engaging member from a running position to a latched position to latch the stinger (10) and the stinger receptacle (100) together. 35

13. A method according to Claim 11, further comprising the step of: clamping a releasable stop member to disengage the loaded engaging member from the latched position such that the stinger (10) and stinger receptacle (100) are separable. 40

14. A method according to Claim 13, wherein the step of clamping the releasable stop member to disengage the loaded engaging member is accomplished by use of a hand clamp, or by use of operating rams of a blowout preventer. 45

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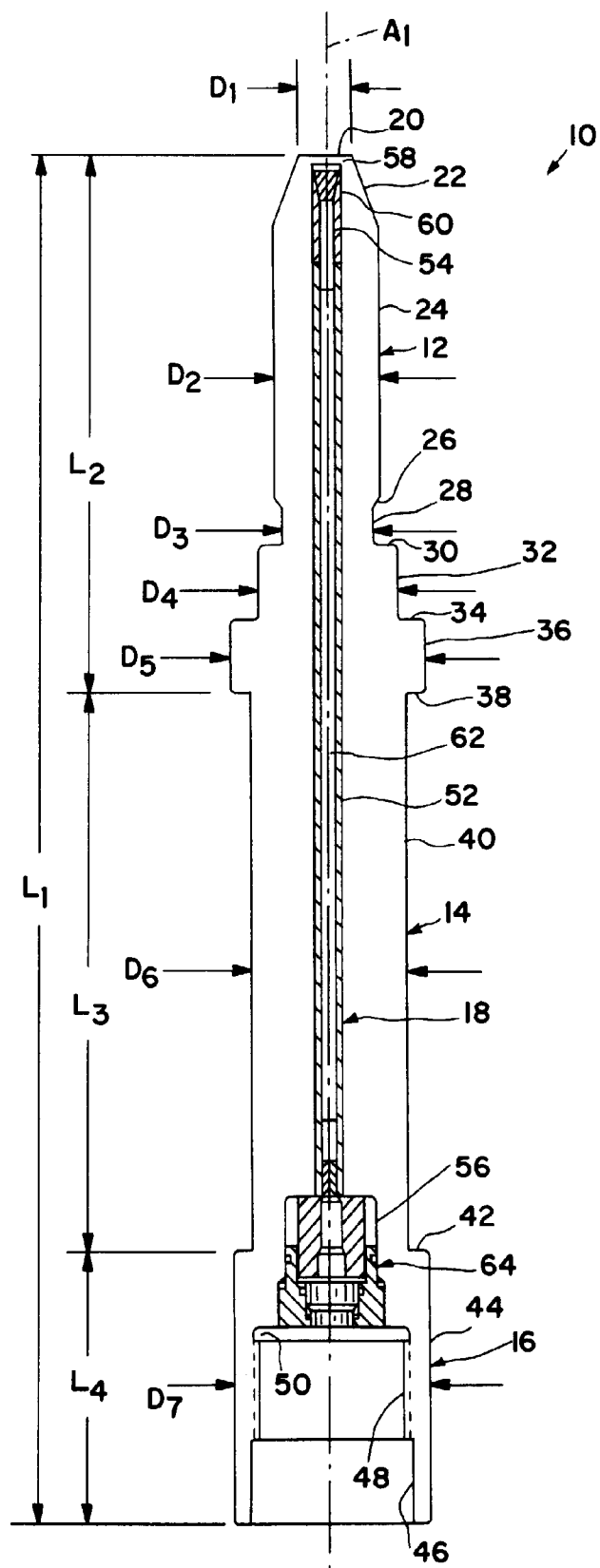


FIG. 1

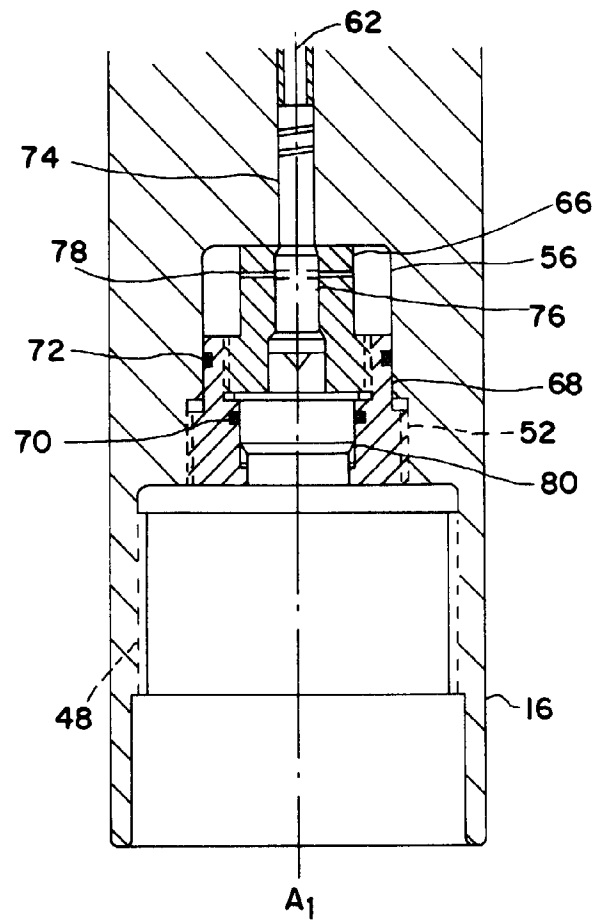


FIG. 2

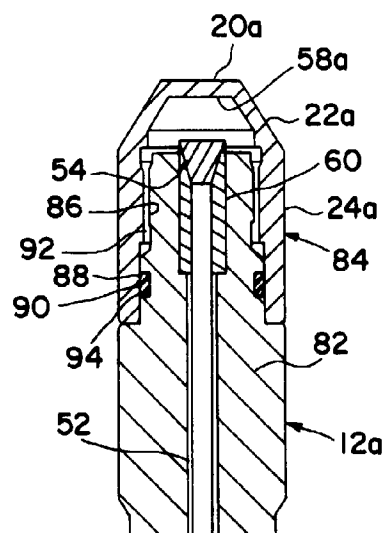


FIG. 3

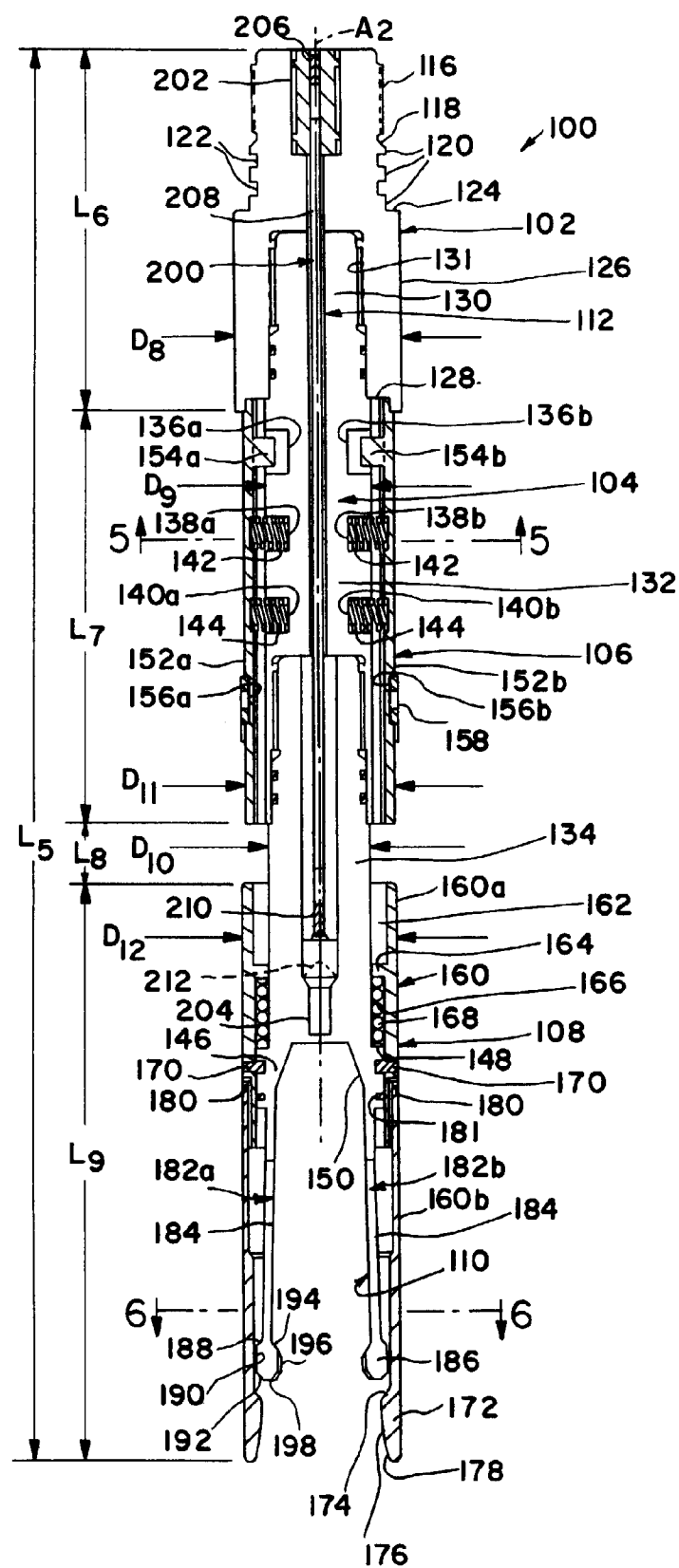


FIG. 4

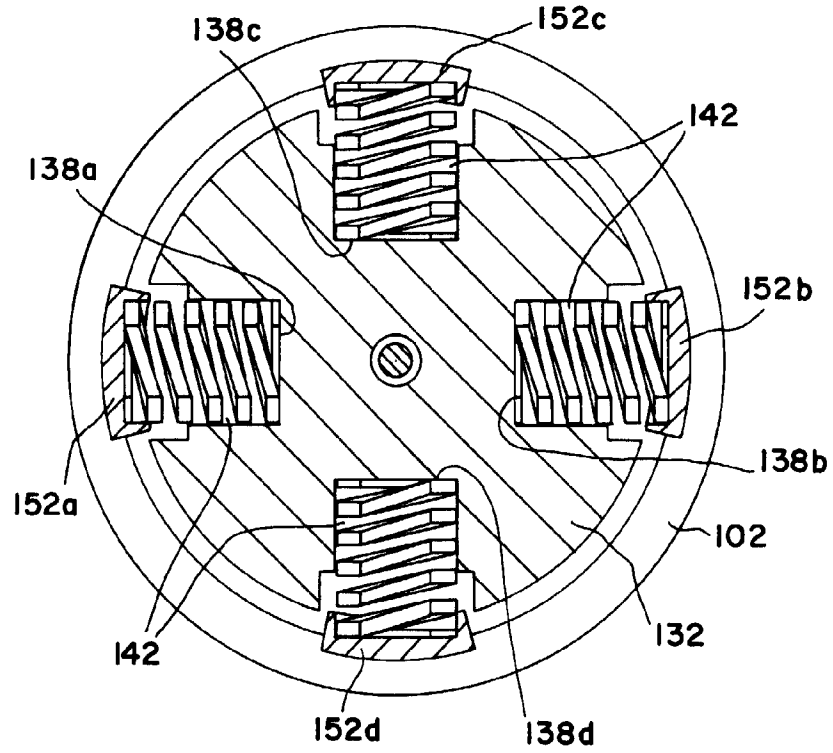


FIG. 5

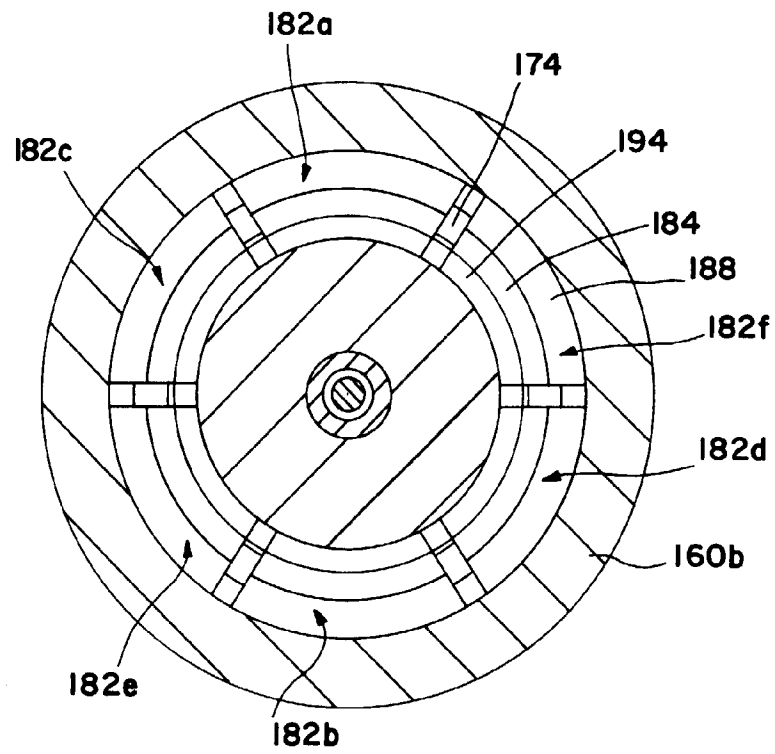


FIG. 6

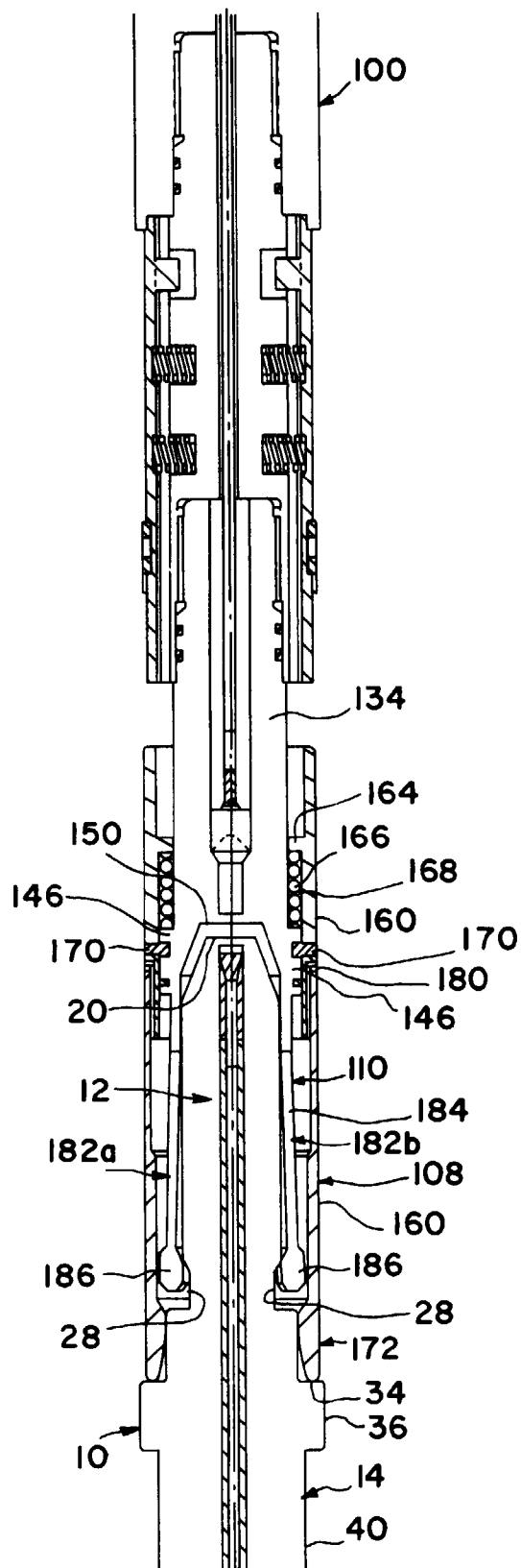


FIG. 7

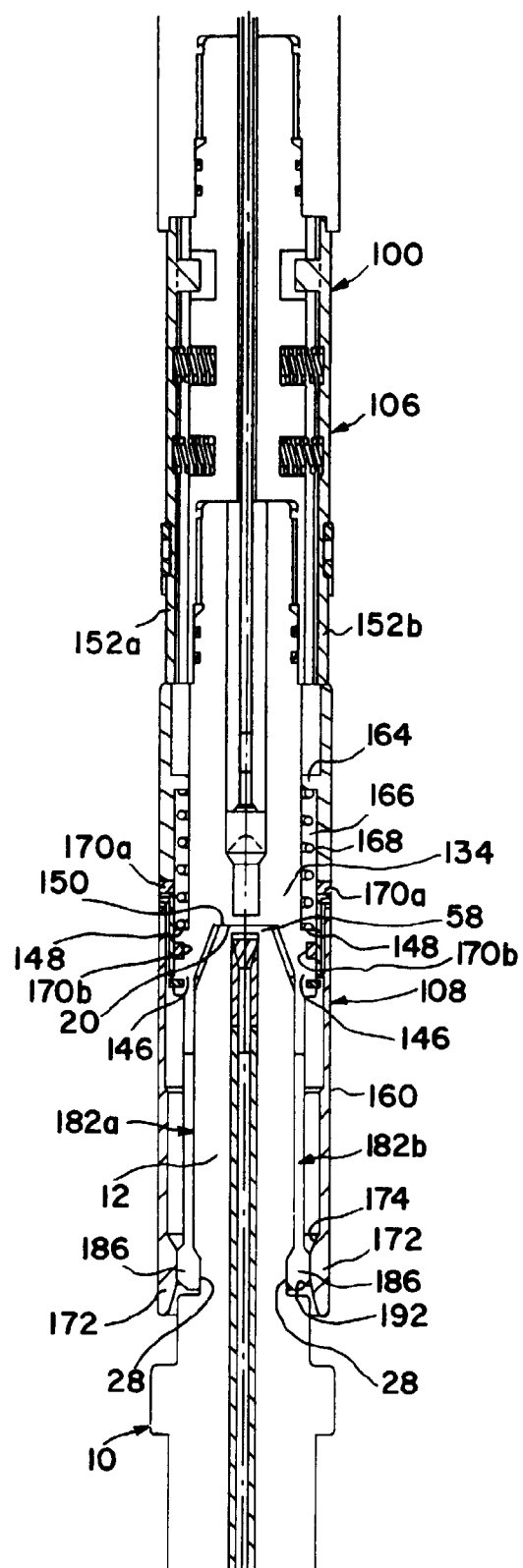


FIG. 8

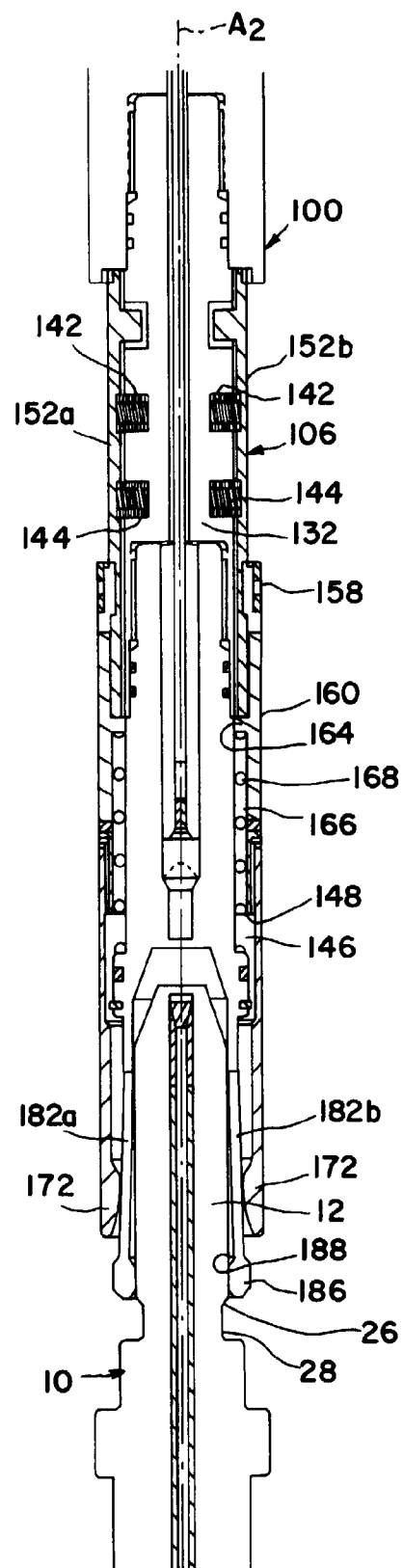


FIG. 9