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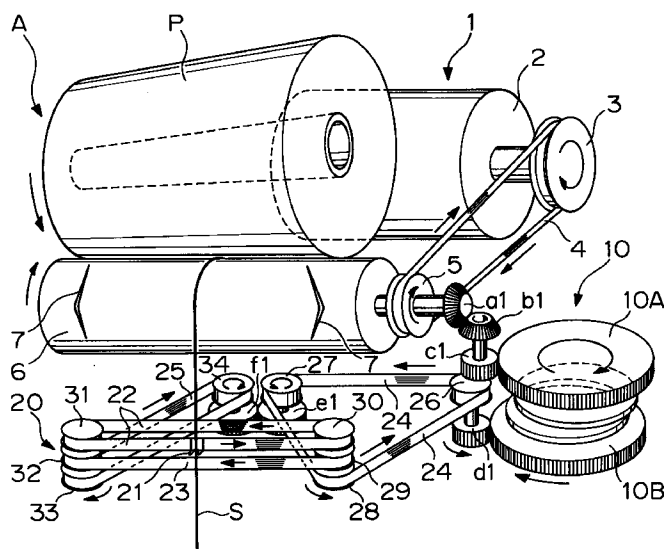
(54) **Winding device**

(57) A yarn winding device that is able to change the winding state applied to both the large diameter part and small diameter part of the cone package using a simple system.

A winding system connected to a motor and a traverse system driven by that motor are connected via

a speed change system arranged such that the traverse speed may be changed. The speed change system is able to change the speed change ratio non-gratuatedly by using non-aligned shafts which are able to control the amount of eccentricity.

FIG. 1



EP 0 834 462 A2

Description

Field of the Invention

The present invention relates to a winding device for forming a package by winding while traversing yarn and more specifically, relates to a winding device of the type that traverses yarn by a traverse system having a set of traverse belts that move in opposite directions to each other.

Background of the Invention

Winding devices for forming a package by winding yarn are generally provided with a winding system connected to a drive source such as a motor or the like and having a friction drum that rotates in contact with the package, and a traverse system for traversing the yarn to be wound on the package, and are arranged such that a package is formed by winding the yarn on a paper tube or the like by the winding system while traversing.

Conventional winding devices are arranged such that the friction drum that transmits rotational drive force by contacting the package performs traversing of the yarn. This device is arranged with a traverse groove in the friction drum and the yarn is traversed along the traverse groove simultaneous with winding of the yarn by rotating the drum.

When this type of friction drum with traverse groove is used, a problem whereby changes in the yarn twist appear due to the traversing. Thus in order to prevent this change in the twist, a winding device arranged with a separate winding system and traverse system has been used. An example of that is using a belt traverse having a set of traverse belts that move in opposite directions to each other and this type is particularly effective for the rewinding of silk yarn or synthetic yarns.

Types of package include cylindrical cheese packages, conical or truncated conical cone packages and the like. With the winding device of the type which uses a friction drum with a traverse groove, in short, performs simultaneous winding of the yarn and traversing by a single friction drum, when there are to be specification changes such as from a cheese package to a cone package or when the winding angle on the cone package is to be changed, it is necessary to exchange the friction drum corresponding to each type of specification. In short, when the type of package is to be changed, the friction drum must be exchanged, the operation is complicated and thus the operational efficiency is decreased.

Conversely, on a winding device arranged with a separate winding system and traverse system, when a cone package is to be formed, it is necessary to modify the winding speed at the small diameter part and large diameter part. Thus conventionally, a measure has been proposed that makes the winding speed when forming the large diameter part slower than when form-

ing the small diameter part by the intervention of a speed change system at some point along the path that transmits the drive force of the motor to the winding system having the friction drum.

However, by increasing and decreasing the rotational velocity (winding speed) of the friction drum on the winding system, not only is the mechanical load on the winding system increased but slippage is generated at the winding system between the package and the friction drum.

Summary of the Invention

It is an object of the present invention to propose a winding device that is able to form a cone package by performing precise changes in the traverse speed by a simple system.

In order to achieve the aforementioned object, the present invention is a winding device provided with a winding system connected to a drive source such as a motor or the like and having a friction drum that rotated in contact with the package and a traverse system for traversing the yarn to be wound on the package, wherein the traverse system is a belt traverse system having a set of traverse belts that move in opposite directions to each other and which is connected to the winding system via a speed change system arranged being an arrangement capable of adjusting the traverse speed.

On the aforementioned winding device, the speed change system may have non-aligned shafts which are able to adjust the amount of eccentricity.

Furthermore, the present invention is a winding device provided with a speed change system input with drive force from a motor or the like and a traverse system connected to the speed change system and having a set of traverse belts that move in opposite directions to each other, wherein the means for transmitting the drive force from the speed change system to the traverse system comprises the engagement of gears and the traverse system is arranged so as to be removable.

On this winding device, the part that connects the traverse system and speed change system may be positioned between the shaft of the input gear on the traverse system and the shaft of the output gear on the speed change system.

Furthermore, on this winding device, the input gear on the traverse system and the output gear on the speed change system are spur gears.

Yet further, the present invention is a winding device provided with a drive transmission part input with a drive force from a motor or the like and a traverse system having a set of traverse belts that move in opposite directions to each other by connection to the drive transmission part, wherein the means for transmitting the drive force from the drive transmission part to the traverse system comprises the engagement of gears and the traverse system is arranged so as to be remov-

able.

Brief Description of the Drawing

Figure 1 is a perspective view showing a schematic arrangement of one example of the winding device being a first embodiment of the present invention.

Figure 2A is a partially sectioned front view in one example of the speed change system used on the winding device being a first embodiment of the present invention with and Figure 2B is a side view of the main part.

Figure 3 is a perspective view showing a schematic arrangement of the winding device being a second embodiment of the present invention.

Figure 4 is a front view showing a schematic arrangement of the winding device being a second embodiment of the present invention.

Figure 5 is a top view showing a schematic arrangement of the winding device being a second embodiment of the present invention.

Figure 6A is a front sectional view of the main part in an example of the drum fixed structure on the winding device being a second embodiment of the present invention and Figure 6B is a side sectional view along the line VIB-VIB.

Figure 7 is a perspective view showing an auto-winder being one example of the winding device being a second embodiment of the present invention.

Figure 8 is a perspective view showing a removable belt traverse being the main part of Figure 7.

Detailed Description of the Preferred Embodiments

Firstly, a first embodiment of the present invention will be described using Figures 1 and 2. Figure 1 shows a schematic arrangement of one example of a winding device A being a first embodiment of the present invention.

A winding system 1 comprises a pulley 3 connected to the drive shaft of a motor 2 being the drive source, a pulley 5 to which the drive force is transmitted from the pulley 3 by a belt 4 and a friction drum 6 (hereafter, referred to simply as a drum) connected to the pulley 5 and that contacts the periphery of a package P. The drive force of the motor 2 is transmitted to the drum 6 via the pulley 3, belt 4 and pulley 5 and winding of a yarn S is performed by applying a rotational drive force to the package P by the contact rotation of this drum 6. A guide groove 7 for preventing yarn stitching is arranged in part of the circumferential direction close to the left and right ends of the surface of the drum 6.

It should be noted that the structure (bent angles or the like) of these left and right guide grooves 7 differ between the large diameter side and small diameter side.

The drive force of the motor 2 is transmitted to the traverse system 20 via the speed change system 10 by

the system described hereafter. The rotational drive force of the motor 2 is transmitted to the input side gear 10A of the speed change system 10 via a gear a1 linked to the rotation shaft of the drum, a gear b1 that engages with the gear a1 and a gear c1 linked to the gear b1 and on the same shaft. Due to the rotational driving of the input side gear 10A, the output side gear 10b rotates at a under predetermined speed change conditions and this speed varying rotational drive force is transmitted to the traverse system 20.

The traverse system 20 is for moving a set (two) of synchronous toothed belts 22,23 (traverse belts) arranged horizontally above each other and each provided with traverse guides 21 that catch the yarn S, in opposite directions to each other and comprises synchronous toothed belts 22,23,24,25 pulleys 26,27,28,29,30,31,32,33,34 and gears d,e,f.

It should be noted that the pulleys 28 and 30 arranged on the same shaft at one end of one set of synchronous toothed belts 22,23 rotate in unison but pulley 29 rotates freely with respect to those. Similarly; the pulleys 32 and 33 arranged on the same shaft at the other end of one set of synchronous toothed belts 22,23 rotate in unison but pulley 31 rotates freely with respect to these. Due to this arrangement, one set of synchronous toothed belts 22,23 arranged on concentric pulleys arranged at both ends of the traverse width are able to move in opposite directions to each other.

The rotational drive force of the output side gear 10B of the speed change system 10 is output to the gear d1 making the pulley 26 connected to the same shaft rotate. Due to the rotation of the pulley 26, the synchronous toothed belt 24 arranged on the pulleys 27,28 is driven. The aforementioned pulley 28 rotates the pulley 30 connected to the same shaft and drives the synchronous toothed belt 22 arranged on the pulley 31 in the direction of the arrow.

Conversely, as the pulley 27 is rotated, the gear e connected to the same shaft as pulley 27 rotates and the gear f1 that engages with this gear e1 is rotated in the opposite direction. Thus the pulley 34 connected to the same shaft as gear f1 rotates, the synchronous toothed belt 25 is driven by the rotation of that pulley 34 and the pulley 33 is rotated. As a result, the synchronous toothed belt 23 arranged on the pulley 29 is driven in the opposite direction to synchronous toothed belt 22 by the pulley 32 rotating.

Due to this arrangement, the traverse system 20 is able to reciprocally move the yarn S within a predetermined range equivalent to the winding width of the package P due to the transfer of the yarn S from the traverse guide 21 of the upper synchronous toothed belt 22 to the traverse guide (not shown in the drawings) of the lower synchronous toothed belt 23.

Next, the structure and speed change functions of the speed change system 10 will be described.

As shown in Figure 2, the rotation shaft 11 of the input side gear 10A and the rotation shaft 12 of the out-

put side gear 10B on the speed change system 10 are arranged as non-aligned shafts where the center of rotation is eccentric. In short, as shown in Figure 2A, the rotation shaft 11 of the input side gear 10A and the rotation shaft 12 of the output side gear 10B are positioned such that the centers of rotation of each r_1, r_2 are eccentric by the distance X and furthermore, a first disk 13 is arranged on the tip of the rotation shaft 11 of the input side gear 10A and a second disk 14 is arranged on the tip of the rotation shaft 12 of the output side gear 10B. Also, a pin 15 fixed in a suitable place in the first disk 13 connects both disks 13, 14 by being movably insertable into a long hole 16 arranged in the second disk 14.

Due to the above arrangement, the rotational drive force of the input side gear 10A is transferred to the output side gear 10B by the pin 15 connecting the first disk 13 and the second disk 14. As shown in figure 2B, as the center of rotation r_1 of the first disk 13 and the center of rotation r_2 of the second disk 14 are eccentric by the distance X, the distance from the center of rotation r_2 of the second disk 14 to the pin 15 gradually changes while the first disk 13 is rotating. In short, this means that the pin 15 rotates at a uniform peripheral velocity but during this rotation, the angular speed of the second disk 13 changes over time as the rotational radius from the pin 15 to the center of rotation r_2 of the second disk 14 changes. As a result, while the first disk 13 makes one revolution at a uniform speed, the second disk 14 makes one revolution at a non-uniform speed. Accordingly, as the rotational velocity of the rotational drive force output to the traverse system 20 through the output side gear 10B connecting to the second disk 14 changes over a fixed cycle, the movement speed of the traverse belts 21, 22, in short, the traverse speed increases and decreases over a fixed cycle.

On the speed change system 10 using these non-aligned shafts, one revolution is one cycle of speed variation. Conversely, when a cone package is to be formed, it is necessary to increase the traverse speed at the small diameter part and decrease the traverse speed at the large diameter part. Accordingly, the traversing of the yarn S completes one reciprocal movement for each one revolution of the input/output side gears 10A, 10B of the speed change system 10 and the intended cone package may be obtained if the traverse speed is maximized at the smallest diameter part on the cone package and minimized at the largest diameter part of the package P.

It should be noted that if the dimension of from the center of rotation r_1 of the first disk 13 to the center of the pin 15 is given the symbol R and the amount of eccentricity between the first disk 13 and the second disk 14 is given the symbol X, the speed charge ratio is defined as $(R+X)/(R-X)$. For normal cone winding, the speed change ratio is set around 1.3 but by changing the value of the amount of eccentricity X, the formation of a cone package having the desired shape corresponding to each type of specifications is possible by

changing the speed change ratio non-graduatedly. Furthermore, if it is possible to set the speed change ratio to 1, the formation of a cheese package is also possible.

It should be noted that the changing of the speed change ratio is possible by fixing the position of the input side gear 10A and first disk 13 and increasing or reducing the amount of eccentricity X by moving the output side gear 10B and second disk 14 in a direction at right angles to the rotation shaft. However, in this case, as can be understood from Figure 1, by changing the position of the output side gear 10B, the position of the pulley 26 and the gear d1 that inputs rotational drive force into the traverse system 20 must also be changed. Thus, a tension pulley (not shown in the drawings) is arranged with respect to the synchronous toothed belt 24 and it is preferable to adjust the belt tension using this.

The first embodiment of the present invention is not limited to that stated here and does not disclose suitable modifications in response to various conditions. For example, although not shown in the drawings, a wing traverse system or equi-velocity cam traverse system may be utilized as a traverse device.

Next, using Figures 3 to 6, a second embodiment of the present invention will be described. Figure 3 shows a schematic arrangement of one example of the winding device A being a second embodiment of the present invention.

This winding device A is provided with a drive source such as a motor 102 or the like, a pulley 105 to which the drive force is transmitted via a belt 104 from a pulley 103 linked to a drive shaft 102a of the motor 102, and a friction drum 106 (hereafter, referred to simply as a drum 106) connected to a rotation shaft 106a of the pulley 105 and is in contact with the periphery of the package P. Guide grooves 107 for preventing yarn stitching are formed in part of the circumferential direction near both the left and right ends on the periphery of the drum 106.

The drive force of the motor 102 is transmitted from the pulley 103 to the pulley 105 via the belt 104 and the rotational drive force is transmitted from that pulley 105 to the speed charge system 10 of the traverse device T and drum 106. The drum 106 is arranged such that the yarn S is wound with a tapered paper tube Q as the core by rotating the package P.

The traverse device T comprises speed change system 110 and traverse system 120 connected to this. The system that transmits the drive force of the motor 102 from the speed change system 110 of the traverse device T to the traverse system 120 is as described hereafter. The rotational drive force of the motor 102 is input into the speed change system 110 from the gear a2 linked to the rotation shaft 106a of the drum 106. In short, is transmitted to the primary side gear 110A via the gear b2 that engages with the gear a2 (in the present embodiment, the gears a2, b2 are bevel gears) and the gear c2 that is linked to the same shaft as gear

b2. Due to the driving of the primary side gear 110A, the secondary side gear 110B rotates under predetermined speed change conditions by the system described hereafter and that speed varied rotational drive force rotates the gear d2 that engages with the secondary side gear 110B and the gear e2 linked to the same shaft as gear d2. Also, the drive force is transmitted to the input gear g2 of the traverse system 120 through the output gear f2 that engages with the gear e2.

The traverse system 120 rotates the traverse belts 121,122 arranged parallel above each other and provided with traverse guides 140 on each that connect with the yarn S, in opposite directions to each other and as shown in Figure 5, is arranged as a unit removable by a bolt 119 from a frame 117,118 that supports the speed change system 110. The arrangement of the traverse system 120 comprises not only traverse belts 121,122 but also synchronous toothed belts 125,126, pulleys 123,124,128-134 and input gear g2, gears j2 and k2. It should be noted that gears j2 and k2 are mounted at a position from which a removing operation is possible from the outer side.

The upper traverse belt 121 is arranged between the pulleys 130,131 and the lower traverse belt 122 is arranged between pulleys 129,132. The pulley 123 is linked to the same shaft as the input gear g2 that engages with output gear f2 of the speed change system 110 and a synchronous toothed belt 125 is arranged between this pulley 123, the pulley 124 and pulley 128. Furthermore, the synchronous toothed belt 126 is arranged between the pulleys 133,134. The gear j2 and pulley 124 as well as gear k2 and pulley 134 of which both sets are arranged on the same shaft, rotate in unison and moreover, the gear j2 and gear k2 are arranged such that they move while engaging.

Of the pulleys 128,129,130 arranged on the same shaft, the pulleys 128 and 130 rotate in unison and the pulley 129 rotates freely with respect to the other two. Similarly, of the pulleys 131,132,133 arranged on the same shaft, the pulleys 132 and 133 rotate in unison and the pulley 131 rotates freely with respect to the other two. It should be noted that the aforementioned pulleys 128,129,130 arranged on the same shaft and the pulleys 131,132,133 are arranged so as to each be removable by a boss 141 and boss 142 respectively (refer to Figure 4).

It should be noted that the tension pulleys 135-138 omitted from Figures 3 and 4 are inserted in Figure 5 showing the schematic arrangement of the traverse system 120 when viewed from above. The tension pulley 135 is for adjusting the tension of the upper traverse belt 121, the tension pulley 136 is for adjusting the tension of the lower traverse belt 122, the tension pulley 137 is for adjusting the tension of the synchronous toothed belt 125 arranged between the pulleys 123,124,128 and the tension pulley 138 is for adjusting the tension of the synchronous toothed belt 126 arranged between the pulleys 133,134.

The rotational drive force input into the input gear g2 from the output gear f2 of the speed change system 10 is transmitted to the pulley 123. Due to this, the synchronous toothed belt 125 arranged between the pulleys 123,124,128 is driven and the traverse belt 121 arranged between the pulleys 130,131 moves in the direction of the arrow (refer to Figure 3) due to the rotation of the pulley 130 linked to the same shaft as pulley 128.

Due to the rotation of the aforementioned pulley 124, the gear j2 linked to the same shaft rotates and the gear k2 that engages with this is rotated in the opposite direction. Thus as the pulley 134 linked to the same shaft rotates, the pulley 133 rotates via the synchronous toothed belt 126. As a result, the pulley 132 linked to the same shaft rotates and the traverse belt 122 arranged between the pulleys 129,132 moves in the opposite direction to the aforementioned traverse belt 121.

Due to the aforementioned arrangement, the upper and lower traverse belts 121,122 move in opposite directions to each other. Then due to the transfer of the yarn S between the upper and lower traverse guides 140,140 arranged on each of the traverse belts 121,122 respectively, the yarn S is able to be reciprocally moved within a predetermined range equivalent to the winding width of the package P.

Similar to the first embodiment, the changing of the speed change ratio is possible by fixing the position of the primary side gear 110A and first disk 113 and increasing or reducing the amount of eccentricity by moving the secondary side gear 110B and second disk 114 in a direction at right angles to the rotation shaft. However, in the case of the second embodiment, as it becomes necessary to change the position of the gears d2 and e2 in association with the positional change of the secondary side gear 110B, the spacing of the gear e2 and the input gear g2 of the traverse system 120 changes. However, changing the position of the input gear g2 is difficult. Thus in this second embodiment, the output gear f2 is movable and moreover, its diameter is larger than the spacing between the gear e2 and gear g2 and by changing the gear position such that it suitably engages corresponding to the increase or decrease in the spacing between the gears e2,g2, this problem is solved.

When the traverse device T is to be used on a yarn winding device, the following adjustment operations ① through ③ are necessary.

① Adjustment of the yarn transfer position between the traverse guides arranged on the traverse belts.

Normally, the upper and lower traverse guides are set so that both the left and right sides of the traverse belts overlap.

② Adjustment of the transfer position between the traverse guides and the speed change cycle of the speed change system.

In short, this must be set such that the yarn

traverse speed is maximized at the smallest diameter part of the package and minimized at the largest diameter part.

③ Setting of the drum position such that the yarn contacts the guide groove in the drum periphery for preventing yarn stitching at the time of yarn transfer.

In order to easily carry out each of the aforementioned adjustment operations, the following arrangements a through d are utilized by the second embodiment.

(a) Forming the traverse system 120 into a unit and making it removable.

(b) An arrangement such that the drive force is transmitted by the engagement of the output gear f2 of the speed change system 110 and the input gear g2 of the traverse system 120, and a setting such that the output gear f2 and input gear g2 engage due to the mounting of the traverse system 120.

(c) An arrangement such that the gear k2 of the traverse system 120 may be removable and the removing operation of that gear k2 from the outer side may be performed.

(d) The utilization of a taperlock system such that the drum may be fixed at the desired rotation position.

The aforementioned transfer position adjustment ① is performed as described hereafter. The traverse system 120 is removed from the traverse device T and the gear k2 removed. The desired tension is then applied to each of the traverse belts 121,122 and synchronous toothed bolts 125,126 arranged between the pulleys by the tension pulleys 136-138 (refer to Figure 5) and the traverse guides 140,140 are adjusted such that one of the traverse belts 121,122 overlap at one end in the vertical direction. This state is then held by a suitable tool and if the gear k2 is attached and engages with the gear j2, the transfer position of the yarn S between the traverse guides 140,140 is fixed in the correct state. It should be noted that the aforementioned operation may be performed with the gear j2 being removable.

Next, the aforementioned timing adjustment operation ② will be described. Firstly, the phase of the speed change system 110 is adjusted such that it is the same as the traverse system 120. In short, the positional relationship between the primary side gear 110A and secondary side gear 110B is adjusted such that the speed variation conditions are in alignment with the traverse guide position after adjustment ① (the rotational velocity has been either maximized or minimized). If a marking that shows the position where the rotational velocity

of the secondary side gear 110B is at a maximum (or minimum), is performed on the frame or the like that supports the speed change system 110, the aforementioned adjustment operation is extremely simple. In continuance, based on the aforementioned procedure, after the yarn transfer position has been correctly adjusted, the traverse system 120 that has been held in this state by a suitable tool is fixed to the predetermined place on the frame 118 (refer to Figure 5) by 3 or 4 bolts 119 and the input gear g2 is engaged with the output gear f2. Due to this, a state where the yarn transfer timing and speed variation of the traverse speed have been correctly adjusted may be obtained.

Concerning the aforementioned ③, after performing the adjustments of the yarn transfer position, transfer timing and speed variation conditions, the traverse guide is held in the yarn transfer position and the position of the drum 106 is adjusted and fixed so that it is in alignment with this held state. The drum 106 of the present embodiment is arranged such that it may be fixed at a desired angle (phase) with respect to the rotation shaft 106a by a tapered fixing member 108 positioned on the end as shown in Figures 4 and 6.

As shown in Figure 6, the fixing member 108 comprises first taper members 108a,108a that contact the outer periphery of the rotating shaft 106a, second taper members 108b,108b that contact the inner periphery of the drum 106, and bolts 108c,108c. Each member is positioned in two part sets axially symmetric about the rotation shaft 106a. Furthermore, a hole 108d into which may be freely inserted the bolt 108c is formed in the first taper member 108a and a screw hole 108e into which the bolt 108c may be screwed is formed in the second taper member 108b. The bolt 108c is inserted into the hole 108d of the first taper member 108a and by tightening by screwing into the screw hole 108e of the second taper member 108b, the second taper member 108b is pulled by the first taper member 108a. Due to this, both taper members 108a,108b slide in contact with each other along a slope, the first taper member 108a presses the outer periphery of the rotation shaft 106a, the second taper member 108b presses the inner periphery of the drum 106 and as a result, the drum 106 and rotation shaft 106a are fixed so as to rotate in unison.

In the present embodiment, it is possible to fix the drum 106 at a desired angle (phase) with respect to the rotation shaft 106a due to the fixing member 108. Accordingly, if a marking that shows the correct drum 106 position during the yarn S transfer, is performed on the frame or the like that supports the drum 106, the position adjustment operations of the drum 106 may be performed extremely easily and as a result, the contact state of the yarn S with the guide groove 107 at the time of transfer is correct and yarn stitching may be reliably prevented.

Figure 7 shows the entire arrangement of an auto-winder 150 being one example of the winding device

being a second embodiment of the present invention and Figure 8 shows those main parts. The auto-winder 150 shows forms a cone package by winding the yarn S on a tapered paper tube Q held so as to be removable by the upper most cradle 151 while unwinding the yarn S from an supply yarn package 160. It should be noted that the auto-winder of figures 8 and 9 may be used as winding devices being the first embodiment of the present invention.

In the second embodiment of the present invention, the part (refer to Figure 8), of the belt traverse device T, comprising the drum 106 that applies a rotational drive force to the paper tube Q and the traverse system 120 that traverses the yarn S to the left and right is an arrangement removable as a single unit by a bolt from the frame 159 of the winder 150 and the speed change system 110 and a mechanical element for transmitting the rotational drive force of the motor 102 to that speed change system 110 (in short, belt 104, pulley 105 and gears a2,b2,c2) are stored inside the frame 159.

Furthermore, as the marking for aligning the phase of the speed change system 110 to the traverse system 120 is easily visible from the outer side, it is preferable to arrange a winder or the like (not shown in the drawings) in a location on the frame 159.

Hereafter, the actions of the aforementioned auto-winder 150 will be described. When normal winding is to be performed, the yarn S unwound from the supply yarn package 160 is rewound onto the tapered paper tube Q rotated by the drum 106 after being made to follow the guide 157. The yarn S at this time is traversed to the left and right while the speed is increased and decreased in the uniform cycle ideal for forming a cone package. During this winding operation, a yarn defect detector 153 inspects whether there are any yarn defects in the yarn Y and furthermore, a tensor 156 controls the tension of the yarn S.

When the yarn defect detector 153 detects some kind of defect or abnormality in the yarn S, the driving of the drum 106 by the motor 102 is stopped and the yarn S is cut by a cutter provided on the yarn defect detector 153. The supply side part of the cut yarn S (lower yarn) is sucked by the continuously sucking lower yarn sucking mouth 155. Conversely, the winding side part of the cut yarn S (upper yarn) is wound on to the paper tube Q by the drum 106 rotating by inertial rotation. In continuance, the yarn S is slightly payed out by reverse rotating the drum 106 and after grabbing the upper yarn be rotating the upper yarn suction member 158 upwards, the grasped upper yarn is guided to the yarn piecing device 152 by rotating the upper yarn suction member 158 downwards. Conversely, after grasping the lower yarn sucked into the lower yarn suction mouth 155 by rotating the lower yarn suction member 158 downwards, it is once again rotated upwards and the grasped lower yarn is guided to the yarn piecing device 152. In this way, the upper and lower yarns guided to the yarn piecing device 152 are joined by a rotating air current at the

yarn piecing device 152. After the yarn piecing processing is complete, the rotation of the motor 102 in the forward direction restarts and the winding of the yarn S resumes.

The aforementioned second embodiment is not limited to that disclosed here. For example, the winding device being a second embodiment of the present invention may be provided with a belt traverse device not possessing a speed change system specifically for a cheese winding. In this case, it is necessary to align the drum position with the traverse system phase but after individually adjusting each, a brief adjustment operation may be completed by simply engaging the output gear with the input gear by connecting the traverse system to the drive transmission part and thus is extremely simple.

The present invention does not preclude suitable modifications corresponding to other conditions.

As shown by the above description, the winding device of the present invention is able to reliably form a cone package by a simple arrangement. Furthermore, by changing the speed of the traverse system, a winding state suitable for each of the small diameter part and large diameter part when forming a package is realised. There is no increase in the mechanical load on the winding system and there is no danger of slippage being generated between the package and friction drum. In particular, by arranging the traverse system as a belt traverse system that may be arranged as a low inertia rotating member, the problems such as mechanical load and noise may be reduced and speed variation may be more quickly and more reliably performed by the speed change system.

Furthermore, as the present invention is a simple arrangement achieved by utilizing a system whereby the speed change ratio is changed by increasing or decreasing the amount of eccentricity, it is possible to non-graduatedly change the speed change ratio on an identical device.

Yet further, as the present invention has a removable traverse system and as the means for transmission of the drive force from the speed change system to the traverse system is comprised of engaged gears, after independently performing the phase adjustments of each of the traverse system and speed change system, both the phases of both may be aligned by simply mounting the traverse system and engaging the gears. Accordingly, the number of necessary operations is decreased and results in operation only at the front of the device thus making the yarn transfer adjustment operations extremely simple. Furthermore, as a result, the time when the device must be stopped for maintenance is reduced.

As the contact location of the speed change system and traverse system is positioned between the output gear and input gear, the contact operation for both is simplified and moreover, the engagement of the output gear and input gear is reliable.

Due to the arrangement of both the output gear of the speed change system and the input gear of the traverse system as flat gears, the engagement of both is simple and thus operability is improved.

Furthermore, the present invention is also effective with winding devices provided with a belt traverse device not possessing a speed change system.

Claims

1. A winding device provided with a winding system having a friction drum connected to a drive source such as a motor or the like and that rotates in contact with a package, and a traverse system for traversing a yarn to be wound on the package, where the traverse system is a belt traverse system having a set of traverse belts that move in opposite directions to each other and which is connected to the winding system via a speed change system which enables the traverse speed to be adjusted.
2. A winding device as in claim 1, wherein the speed change system has non-aligned axes for which the amount of eccentricity may be adjusted.
3. A winding device provided with a speed change system that is input with drive force from a drive source such as a motor or the like and a traverse system connected to the speed change system and having a set of traverse belts that move in opposite directions to each other, wherein the means for transmitting drive force from the speed change system to the traverse system comprises the engagement of gears and the traverse system is removable.
4. A winding device as in claim 3, wherein the connecting part of the traverse system and speed change system is positioned between the shaft of the input gear on the traverse system and the shaft of the output gear on the speed change system.
5. A winding device as in either of claims 3 or 4, wherein the input gear on the traverse system and the output gear on the speed change system are flat gears.
6. A winding device provided with a drive transmission part that is input with drive force from a drive source such as a motor or the like and a traverse system connected to the drive transmission part and having a set of traverse belts that move in opposite directions to each other, wherein the means for transmitting drive force from the drive transmission part to the traverse system comprises the engagement of gears and the traverse system is removable.

FIG. 1

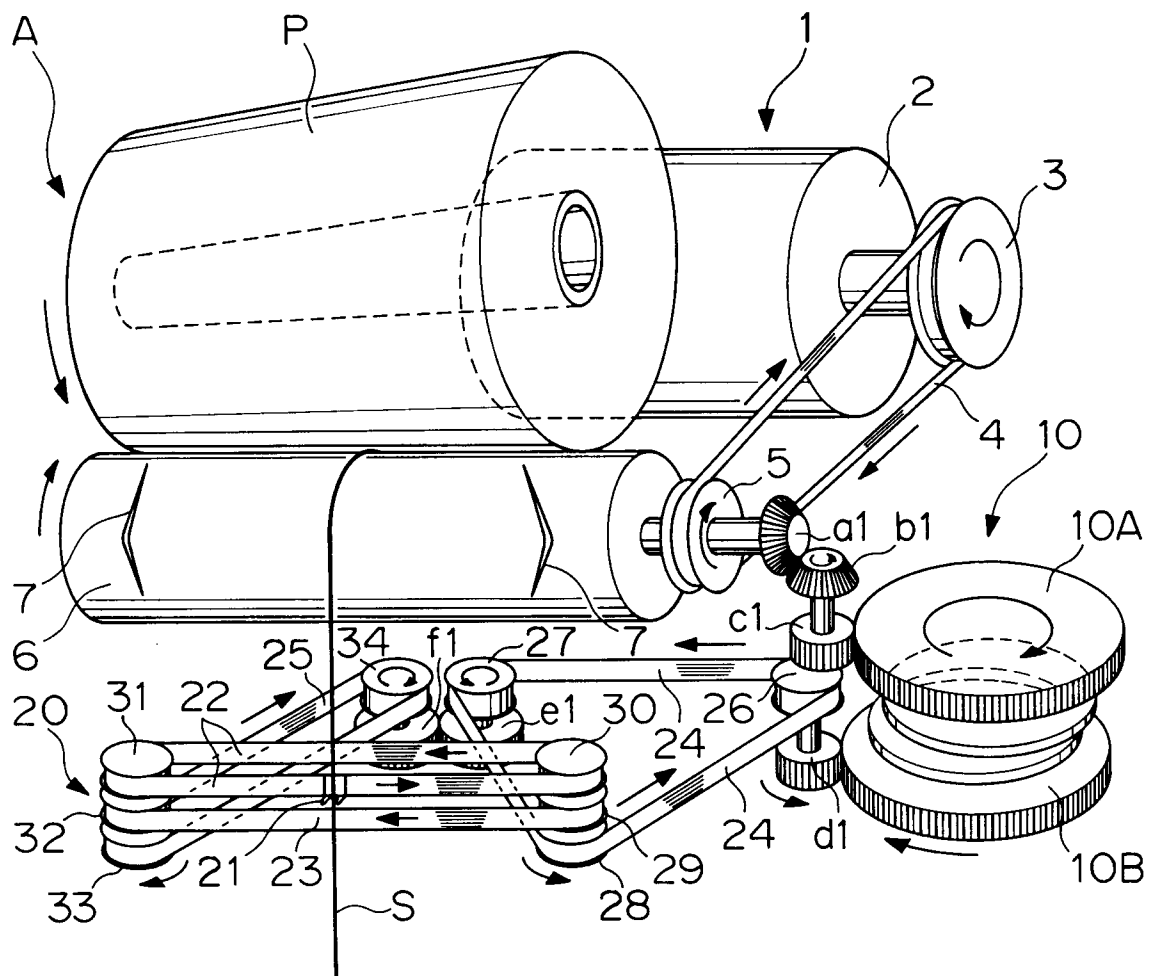


FIG. 2A

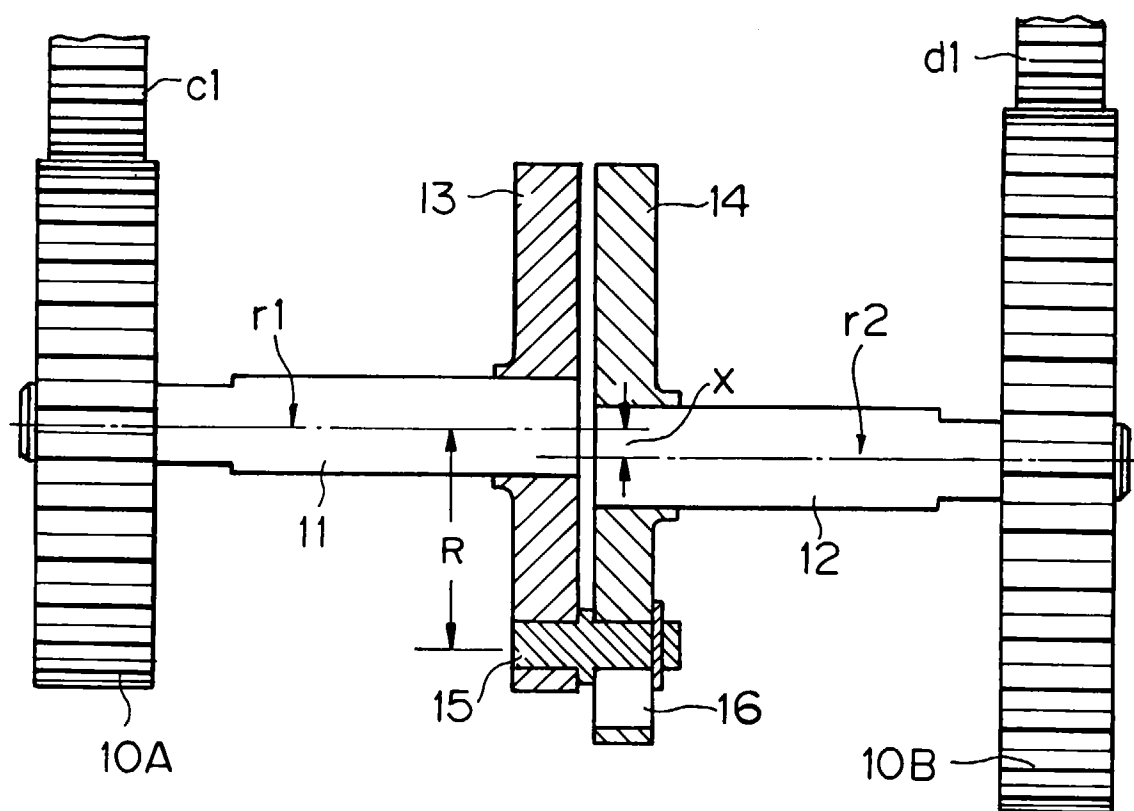


FIG. 2B

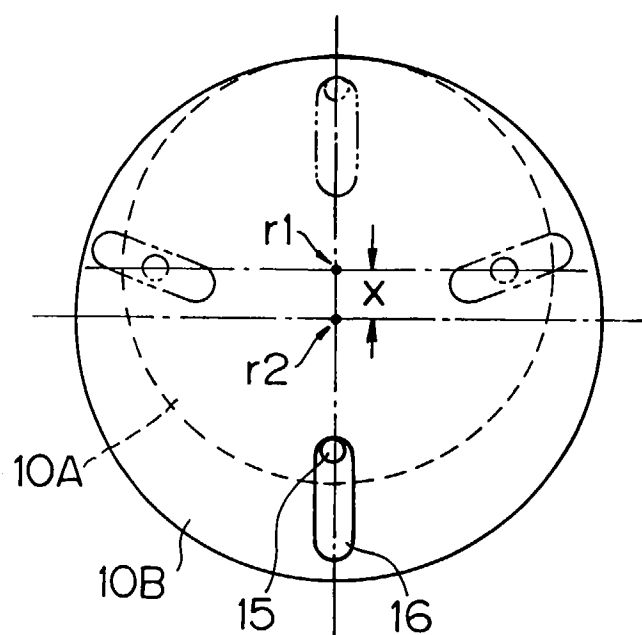


FIG. 3

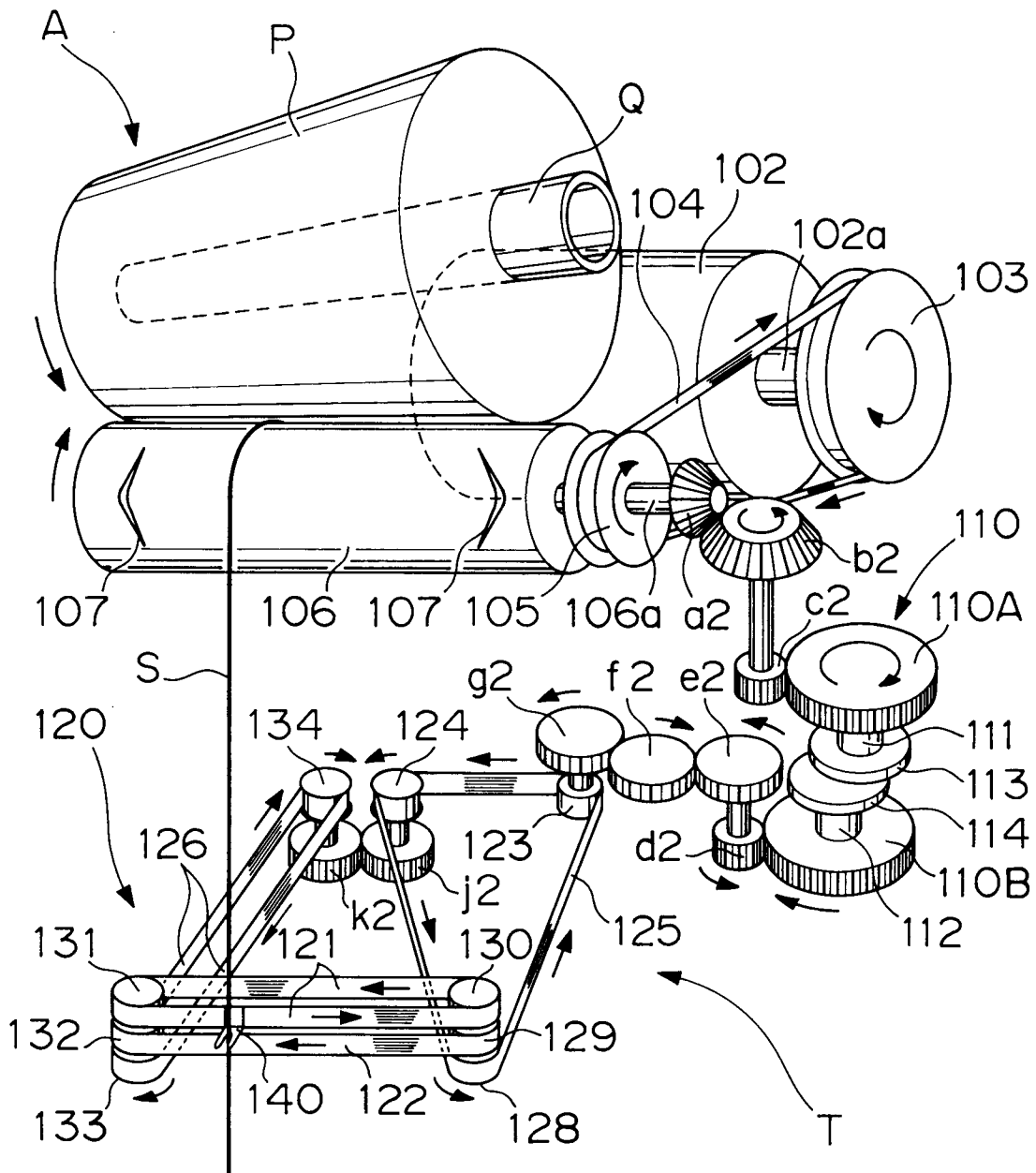


FIG. 4

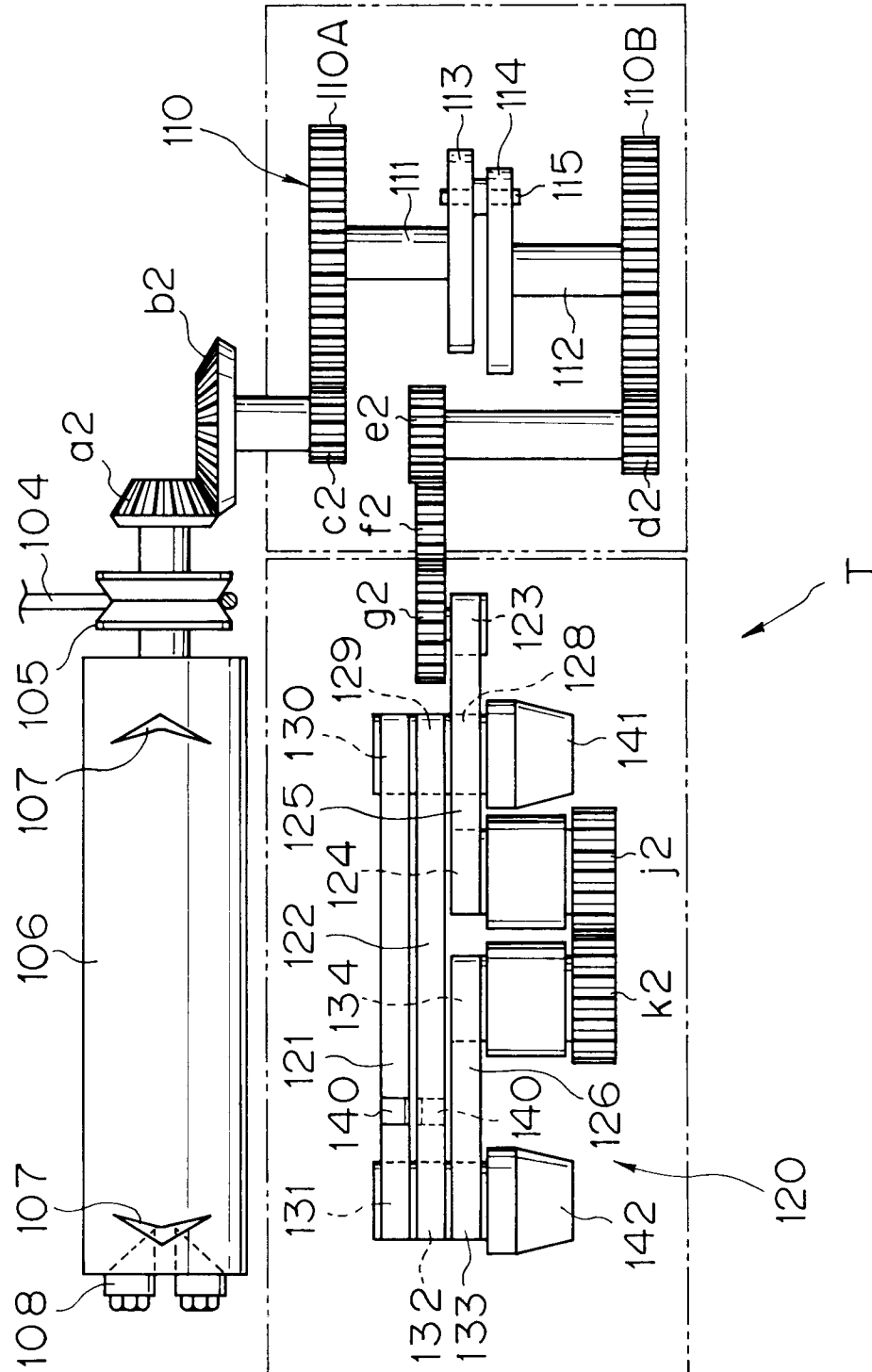


FIG. 5

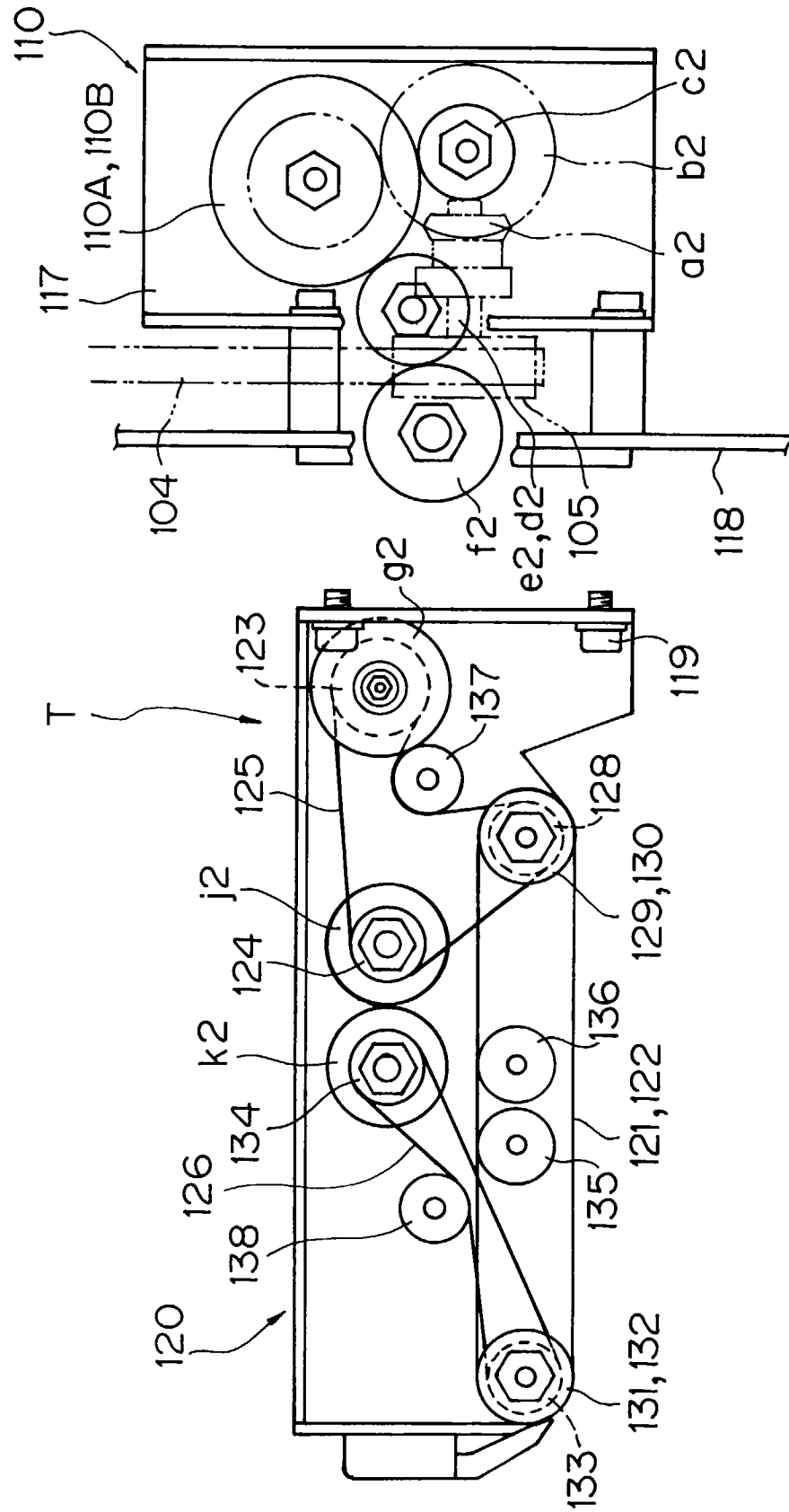


FIG. 6A

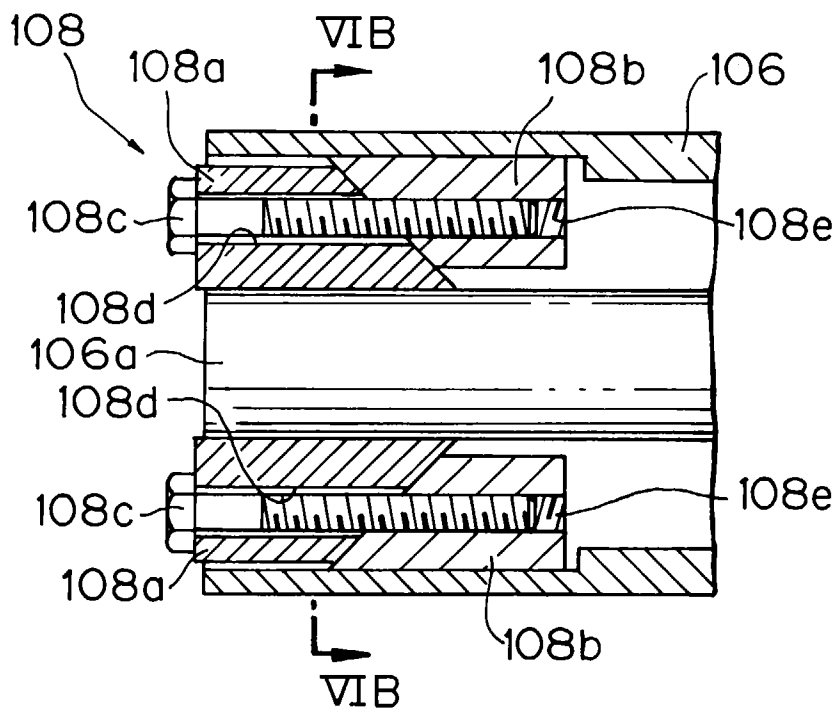


FIG. 6B

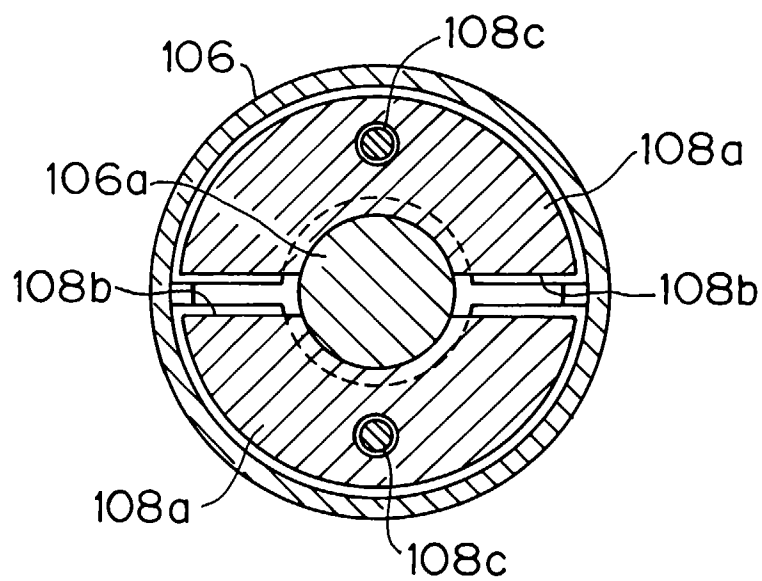


FIG. 7

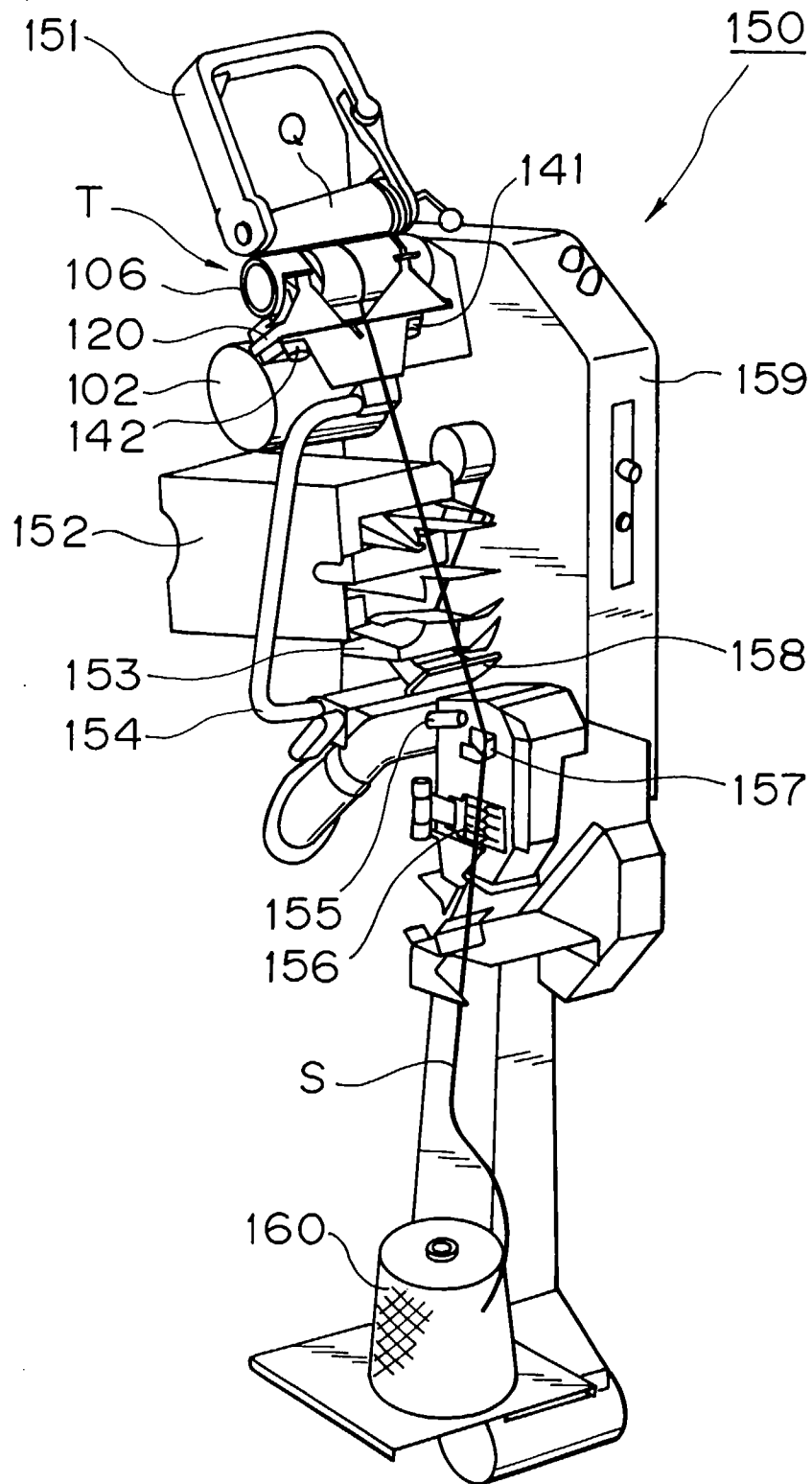


FIG. 8

