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(54) **Rotary valve in a sheet feeding unit of an offset printing press**

Drehschieber in einer Bogenzuführeinrichtung einer Offsetdruckmaschine

Vanne rotative dans une marge des feuilles d'une presse offset

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Description

Background of the Invention

[0001] The present invention relates to a rotary valve disposed between intake and exhaust air sources and suction and discharge units, to supply or stop supplying air.

[0002] In the sheet feeding unit of a sheet-fed offset printing press, a suction unit connected to an intake air source and a discharge unit connected to an exhaust air source are used to feed stacked sheets to the feeder board one by one. More specifically, in order to draw the highest sheet by suction with the suction unit, the air of the suction unit is taken by an intake pump serving as the intake air source. In order to blow air to the stacked sheets or to separate the highest sheet drawn by the suction unit and the second sheet from each other, or in order to discharge reverse air that facilitates separation of the sheet conveyed from the suction unit to the feeder board, exhaust air is supplied by an exhaust pump serving as the exhaust air source.

[0003] The ON/OFF timings of each of the intake air and exhaust air correspond to the rotation angle of the printing press main body. This series of timings is controlled by a rotary valve.

[0004] Fig. 9 shows a conventional rotary valve.

[0005] Referring to Fig. 9, in a rotary valve indicated by reference numeral 30, a main body 2 formed into a substantially rectangular parallelepiped shape is fixed to a frame 5 of a sheet feeding unit through a bracket 5a. The first and second suckers, a leveling foot, and an air blower (not shown) are provided to the sheet feeding unit. As shown in Figs. 10A and 10B, the lower portion of the main body 2 in one end side in the direction of an arrow Y swells in an arcuated manner to constitute a swelling portion 2a.

[0006] A through hole 3 extending in the direction of an arrow X is formed in the swelling portion 2a, and a cylindrical sleeve 4 is fixed to the inner circumferential surface of the through hole 3. A valve body 6 is engaged in the sleeve 4. End shafts 6a and 6b on the two ends of the valve body 6 are rotatably supported by the sleeve 4 through bearings 7a and 7b. The valve body 6 rotates in an interlocked manner with rotation of the printing press through one end shaft 6a.

[0007] Four air passages 9a, 9b, 9c, and 9d, each having an open upper end and a lower end communicating with the through hole 3, are formed in the upper portion side of the main body 2 corresponding to the swelling portion 2a, to extend in the vertical direction (the direction of an arrow Z). Of the air passages 9a to 9d, the air passages 9a and 9b, on their upper end side, are connected to an intake pump (to be described later) through hoses 17a and 17b. The air passages 9a and 9b constitute an intake air passage. The air passages 9c and 9d, on their upper end side, are connected to an exhaust pump (to be described later) through hoses 17c

and 17d. The air passages 9c and 9d constitute an exhaust air passage.

[0008] Air passages 10a, 10b, 10c, and 10d are formed in the main body 2 to extend in the direction of the arrow Y perpendicularly to the intake air passages 9a and 9b and the exhaust air passages 9c and 9d, respectively. One end of each of the air passages 10a to 10d opens to the outside of the main body 2 while the other end thereof communicates with the through hole 3.

[0009] Of the air passages 10a to 10d, the air passages 10a and 10b are connected to suction heads (to be described later), serving as the first and second suckers, through hoses 18a and 18b. The air passages 10a and 10b constitute a suction air passage. The air passages 10c and 10d are connected to nozzles (to be described later), respectively serving as a leveling foot and an air blower, through hoses 18c and 18d. The air passages 10c and 10d constitute a discharge air passage.

[0010] Reference numeral 31 denotes a reverse air passage for the suction heads. The reverse air passage 31 is formed between the suction air passages 10a and 10b to extend from the upper end of the main body 2 to the circumferential surface of the valve body 6 through the sleeve 4. The exhaust pump (described above) is connected to the upper opening end side of the reverse air passage 31 through a hose (not shown). At a certain machine angle of rotation of the valve body 6, the lower end of the reverse air passage 31 communicates with the suction air passages 10a and 10b through a notch (to be described later) formed in the valve body 6.

[0011] Vent holes 11a, 11b, 11c, and 11d are formed in the sleeve 4 to respectively correspond to the intake air passages 9a and 9b and the exhaust air passages 9c and 9d. Vent holes 12a, 12b, 12c, and 12d are also formed in the sleeve 4 to respectively correspond to the suction air passages 10a and 10b and the discharge air passages 10c and 10d.

[0012] As shown in Fig. 10A, a notch 13a through which the vent holes 11a and 12a communicate with each other is formed in the circumferential surface of the valve body 6 corresponding to the intake air passage 9a and the suction air passage 10a. Similarly, a notch 13b through which the vent holes 11b and 12b communicate with each other is formed in the circumferential surface of the valve body 6 corresponding to the intake air passage 9b and the suction air passage 10b, at a position displaced from the notch 13a in the axial direction and to be phase-shifted from the notch 13a in the rotating direction of the valve body 6.

[0013] As shown in Fig. 10B, a notch 13c through which the vent holes 11c and 12c communicate with each other is formed in the circumferential surface of the valve body 6 corresponding to the intake air passage 9c and the suction air passage 10c. Similarly, a notch 13d through which the vent holes 11d and 12d communicate with each other is formed in the circumferential surface of the valve body 6 corresponding to the intake air passage 9d and the suction air passage 10d, at a position

displaced from the notch 13c in the axial direction and to be phase-shifted from the notch 13c in the rotating direction of the valve body 6.

[0014] In this arrangement, when the valve body 6 is rotated in an interlocked manner with rotation of the printing press main body, the notch 13d of the valve body 6 opposes the vent holes 11d and 12d of the sleeve 4, and the exhaust air passage 9d and the discharge air passage 10d communicate with each other through the notch 13d. Thus, air exhausted from the exhaust pump flows through the air passages 9d and 10d that communicate with each other through the notch 13d, and is discharged from the nozzles to blow air to the sheets.

[0015] When the valve body 6 is continuously rotated, the notch 13a opposes the vent holes 11a and 12a, and the intake air passage 9a and the suction air passage 10a communicate with each other through the notch 13a. Thus, as shown in Fig. 10A, intake air A taken by the intake pump flows through the air passages 9a and 10a that communicate with each other through the notch 13a, to draw a sheet by suction with the first sucker.

[0016] When the valve body 6 is continuously rotated, the notch 13c opposes the vent holes 11c and 12c of the sleeve 4, and the exhaust air passage 9c and the discharge air passage 10c communicate with each other through the notch 13c. Thus, as shown in Fig. 10B, exhaust air B exhausted from the exhaust pump flows through the air passages 9c and 10c that communicate with each other through the notch 13c, and is discharged from the nozzle serving as the leveling foot. At this time, the discharged air is blown to a portion between the highest sheet drawn by the suction heads and the second sheet, to separate them from each other.

[0017] When the valve body 6 is further rotated, the notch 13b opposes the vent holes 11b and 12b, and the intake air passage 9b and the suction air passage 10b communicate with each other through the notch 13b. Thus, the intake air taken by the intake pump flows through the air passages 9b and 10b that communicate with each other through the notch 13b, to draw a sheet by suction with the second sucker.

[0018] Simultaneously, the notch 13a is displaced from the vent hole 12a, and the intake air passage 9a and the suction air passage 10a are disconnected from each other. Thus, suction air supply is stopped, and the sheet suction operation with the first sucker is stopped.

[0019] Since a notch 32a of the valve body 6 opposes the reverse air passage 31, the reverse air passage 31 and the suction air passage 10a communicate with each other through the notch 32a. The exhaust air exhausted from the exhaust pump flows through the air passages 31 and 10a and is blown out from the first sucker, and the sheet is quickly released from the first sucker that has ended the suction operation. As a result, a sheet which is conveyed to above the feeder board while being drawn by the second sucker will not be cut or bent.

[0020] When the second sucker during sheet conveyance is located above the feeder board, the notch 13b

is displaced from the vent hole 12b in accordance with rotation of the valve body 6, and the intake air passage 9b and the suction air passage 10b are disconnected from each other. Thus, intake air supply is stopped, and the sheet suction operation with the second sucker is stopped. At this time, a notch 32b opposes the reverse air passage 31, and the reverse air passage 31 and the suction air passage 10b communicate with each other through the notch 32b. The exhaust air exhausted from the exhaust pump flows through the air passages 31 and 10b and is blown out from the second sucker. The sheet is quickly released from the second sucker that has ended the suction operation, and is supplied to above the feeder board.

[0021] In the discharge operation of the nozzles, if the air discharge time of the nozzles is shorter than the suction time of the suction heads, the notch 13c constituting the air passage from the exhaust pump is formed smaller than the notch 13a constituting the air passage from the intake pump, as shown in Figs. 10A and 10B. Inversely, in the suction operation of the suction heads, if the suction time of the suction heads is shorter than the air discharge time of the nozzles, the notch 13a is formed smaller than the notch 13c, as shown in Figs. 12A and 12B.

[0022] As shown in Fig. 10B, if the notch 13c is made small to shorten the air discharge time from the nozzles, along with rotation of the valve body 6, as the opening of the vent hole 11c is enlarged, the opening of the vent hole 12c is narrowed. For this reason, a predetermined air pressure cannot be obtained on the nozzle side.

[0023] Accordingly, the amount of air from the nozzles becomes short and air blowing to the stacked sheets is not performed sufficiently, and two or more sheets are undesirably drawn by the suction heads. In this case, operation of the printing press must be stopped, or the printing press may cause a trouble to decrease the productivity. Since the supply amount of reverse air from the suction heads becomes short to delay sheet release from the suction heads, the sheet may be cut or bent to degrade the printing quality.

[0024] When the operation speed of the printing press increases, the time of forming the air passage in the rotary valve is shortened. Then, a predetermined discharge air pressure from the nozzles cannot be obtained, in the same manner as described above.

[0025] Meanwhile, as shown in Fig. 12A, if the notch 13a is made small to shorten the suction time of the suction heads, along with rotation of the valve body 6, as the opening of the vent hole 11a is enlarged, the opening of the vent hole 12a is narrowed. As a result, a predetermined air pressure cannot be obtained with the suction heads, and defective sheet supply may occur.

[0026] Fig. 11 shows the relationship in pressure of the input/output air of the rotary valve of Fig. 10B during the discharge operation. Referring to Fig. 11, reference numeral b1 denotes the pressure of the exhaust air input to the rotary valve 30. The pressure b1 is the pressure

of the exhaust air from the exhaust pump. Reference numeral b2 denotes the pressure of the discharge air output from the rotary valve. The pressure b2 is the pressure of the discharge air from the nozzles.

[0027] When the machine angle of rotation of the printing press becomes β_1 , the notch 13c of the valve body 6 opposes the vent holes 11c and 12c of the sleeve 4, and discharge air is supplied from an exhaust pump 36 to the nozzles through the exhaust air passage 9c, the notch 13c, and the discharge air passage 10c. The pressure of air supplied to the nozzles at this time is expressed as a pressure P. Subsequently, when the machine angle of rotation becomes β_2 , the notch 13c is displaced from the vent hole 12c, the vent hole 12c is closed with the circumferential surface of the valve body 6, and supply of the discharge air to the discharge air passage 10c is stopped.

[0028] At this time, in the conventional rotary valve 30, a pressure b3 of the discharge air between the machine angles β_1 and β_2 of rotation becomes lower than a necessary pressure b4 by a pressure difference ΔP . This is due to the following reason. Since the notch 13c is small, along with rotation of the valve body 6, as the opening of the vent hole 11c is enlarged, the opening of the vent hole 12c is narrowed, so the air exhausted from the exhaust pump 36 is not sufficiently supplied to the nozzles.

[0029] Fig. 13 shows the relationship of the input/output air of the rotary valve shown in Fig. 12A during the suction operation. As shown in Fig. 13, even during the suction operation, a pressure loss in air of the rotary valve occurs. Reference numeral a1 denotes the pressure of the intake air input to the rotary valve 30. The pressure a1 is the pressure of intake air from the intake pump. Reference numeral a2 denotes the pressure of the suction air output from the rotary valve 30. The pressure a2 is the pressure of the suction air of the suction heads. As shown in Fig. 13, a pressure a3 of the suction air between the machine angles β_1 and β_2 of rotation becomes higher than a necessary pressure a4 by a pressure difference ΔP .

[0030] Document DE-A-4 215 226 discloses a rotary valve in which the compressed air source and the hollow portion are connected through a hole extending towards a radial direction of valve body and whole of a main body. Therefore, the compressed air sources in the hollow portion are either disconnected or connected from each other by a rotation of the valve body. The opening in the rotary valve according to the mentioned document is in the circumferential surface of the valve body. This means that the opening and air source on the circumferential surface of the valve are intermittently connected to each other, where the intermittence is subjected to the rotation of the valve body. In a structure where the intermittence is subject to the rotation of the valve body, the air flow is interrupted.

[0031] Document US-A-3 884 460 shows a spool valve member in which a chamber is formed along the central axis. Pressurized air supplied into the chamber

flows into the outlet channel through the radial bores, so as to provide a blast to the suction cup. Pressurized air is supplied into the chamber in a radial direction, i.e. through the supply channel, the inlet opening and the distribution channel.

[0032] Document DE-195 15 301 relates to an arrangement for controlling air with an air diffuser comprising a controller cylinder which is pivotably driveable during a cycle, which, with its cylindrical shell surface, is pivotably run on bearings in the bore hole of a control bearing bush which closely surrounds the controller cylinder, whereby the controller cylinder discloses an air admittable inner chamber from which radial control recesses lead to the cylindrical shell surface of the controller cylinder, which, with radial control connections formed in the control socket/liner, can be overlapped.

Summary of the Invention

[0033] It is an object of the present invention to provide a rotary valve which can obtain a predetermined air pressure even when the actuation time by the air is short.

[0034] It is another object of the present invention to provide a rotary valve in which a pressure loss in air in its interior is decreased.

[0035] In order to achieve the above objects, according to the present invention, there is provided a rotary valve, which is interposed between a first and a second air source for supplying air, and at least one air unit that performs a predetermined operation upon reception of the air supplied from the first and second air sources, to supply/stop supplying air, comprising a valve body rotatably driven in a cylinder, a main body for rotatably supporting the valve body, notches formed in a circumferential surface of the valve body, to be connected to the first air source, and further comprising a first hollow portion formed in said valve body, and wherein said rotary valve comprises a first air passage formed in said main body and having one end side connected to said first air source and the other end side that opens in an inner circumferential surface of said cylinder, a second air passage formed in said main body and having one end side connected to the first air unit and the other end side that opens in said inner circumferential surface of said cylinder, said first air passage being connected to said second air passage through said notches and a first communication hole formed in said valve body to connect said first hollow portion and said second air passage to each other, characterized in that the first hollow portion is formed in said valve body in an axial direction and having, in one end face of said valve body, an opening through which said first hollow portion is connected to the second air source, and in that the rotary valve further comprises a third air passage formed in said main body and having one end side connected to the second air unit and the other end side that opens in an inner circumferential surface of said cylinder, a second

hollow portion formed in said valve body in an axial direction and connected to said first air source, and a second communication hole formed in said valve body to connect said second hollow portion and said third air passage to each other.

Brief Description of the Drawings

[0036]

Fig. 1 is a plan view of a rotary valve;
 Fig. 2A is a sectional view taken along the line I - I of Fig. 1, and Fig. 2B is a sectional view taken along the line II - II of Fig. 1;
 Fig. 3 is a diagram showing the schematic arrangement of the sheet feeding unit of a printing press to which the embodiment according to Fig. 1 is applied;
 Fig. 4 is a plan view of a rotary valve according to the present invention;
 Fig. 5A is a sectional view taken along the line III - III of Fig. 4, and Fig. 5B is a sectional view taken along the line IV - IV of Fig. 4;
 Fig. 6 is a diagram showing the schematic arrangement of a sheet feeding unit to which the embodiment according to the invention is applied;
 Fig. 7 is a plan view of another rotary valve; Fig. 8 is a sectional view taken along the line V - V of Fig. 7;
 Fig. 9 is a plan view of a conventional rotary valve; Fig. 10A is a sectional view taken along the line VI - VI of Fig. 9, and Fig. 10B is a sectional view taken along the line VII - VII of Fig. 9;
 Fig. 11 is a graph showing the relationship in pressure of the input/output air of the rotary valve of Fig. 10B during discharge operation;
 Fig. 12A is a sectional view showing the main part of another example of the rotary valve that corresponds to Fig. 10A, and Fig. 12B is a sectional view showing the main part of still another example of the rotary valve that corresponds to Fig. 10B; and
 Fig. 13 is a graph showing the relationship in pressure of the input/output air of the rotary valve of Fig. 12A during suction operation.

Description of the Preferred Embodiments

[0037] The present invention will be described in detail with reference to the accompanying drawings.

[0038] Fig. 3 shows the schematic arrangement of a sheet feeding unit for a printing press. Referring to Fig. 3, the air source side of a rotary valve 101 is connected to an intake pump 134 serving as an intake air source, and an exhaust pump 136 serving as an exhaust air source, through hoses 117a and 117b, and a hose 119, respectively. The suction/discharging side of the rotary valve 101 is connected to suction heads 135a and 135b serving as the suction units, and nozzles 137a and 137b serving as the discharge unit, through hoses 118a and

118b, and hoses 118c and 118d, respectively. The suction heads 135a and 135b serve as the first and second suckers of the sheet feeding unit, and the nozzles 137a and 137b serve as a leveling food and an air blower.

[0039] Fig. 1 shows the rotary valve shown in Fig. 3. Referring to Fig. 1, a main body 102 formed into a substantially rectangular parallelepiped shape is fixed to a frame 105 of the sheet feeding unit through a bracket 105a. As shown in Figs. 2A and 2B, the lower portion of the main body 102 in one end side in the direction of an arrow Y swells in an arcuated manner to constitute a swelling portion 102a.

[0040] A through hole 103 extending in the direction of an arrow X is formed in the swelling portion 102a, and a cylindrical sleeve 104 is fixed to the inner surface of the through hole 103. A valve body 106 is engaged in the sleeve 104. End shafts 106a and 106b on the two ends of the valve body 106 are rotatably supported by the sleeve 104 through bearings 107a and 107b. The valve body 106 rotates in an interlocked manner with rotation of the printing press through one end shaft 106a.

[0041] Two air passages 109a and 109b, each having an open upper end and a lower end communicating with the through hole 103, are formed in the upper portion side of the main body 102 corresponding to the swelling portion 102a, to extend in the vertical direction (the direction of an arrow Z). As described above, the air passages 109a and 109b, on their upper end side, are connected to the intake pump 134 through the hoses 117a and 117b, and the air passages 109a and 109b constitute an intake air passage (air passage for the intake air source).

[0042] Air passages 110a and 110b are formed in the main body 102 to extend in the direction of the arrow Y perpendicularly to the intake air passages 109a and 109b. Also, air passages 110c and 110d are formed in parallel to the air passages 110a and 110b. One end of each of the air passages 110a to 110d opens to the outside of the main body 102 while the other end thereof is connected to the through hole 103.

[0043] Of the air passages 110a to 110d, the air passages 110a and 110b are connected to the suction heads 135a and 135b through the hoses 118a and 118b, as described above. The air passages 110a and 110b constitute a suction air passage (air passage for the suction unit). As described above, the air passages 110c and 110d are connected to the nozzles 137a and 137b through the hoses 118c and 118d, and the air passages 110c and 110d constitute a discharge air passage (air passage for the discharge unit).

[0044] Vent holes 111a and 111b are formed in the sleeve 104 to respectively correspond to the intake air passages 109a and 109b. Vent holes 112a, 112b, 112c, and 112d are also formed in the sleeve 104 to respectively correspond to the suction air passages 110a and 110b and the discharge air passages 110c and 110d.

[0045] As shown in Figs. 2A and 2B, a blind hole-like

hollow portion 114 is formed in the valve body 106 to have an opening in the end face of the valve body 106 on one end shaft 106b side and to extend in the direction of an arrow X to a position corresponding to the intake air passage 109a. As described above, the exhaust pump 136 is connected to the opening side of the hollow portion 114 through the hose 119. Accordingly, the hollow portion 114 constitutes an exhaust air passage (air passage for the exhaust air source).

[0046] A communication hole 115a (Fig. 2A) is formed in the valve body 106. The communication hole 115a allows the hollow portion 114 to communicate with the suction air passage 110a through the vent hole 112a. Similarly, a communication hole 115b is formed in the valve body 106 at a position displaced from the communication hole 115a in the axial direction and to be phase-shifted from the communication hole 115a in the rotating direction. The communication hole 115b allows the hollow portion 114 to communicate with the suction air passage 110b through the vent hole 112b.

[0047] A communication hole 115c (Fig. 2B) is formed in the valve body 106. The communication hole 115c allows the hollow portion 114 to communicate with the discharge air passage 110c through the vent hole 112c. Similarly, a communication hole 115d is formed in the valve body 106 at a position displaced from the communication hole 115c in the axial direction and to be phase-shifted from the communication hole 115c in the rotating direction. The communication hole 115d allows the hollow portion 114 to communicate with the discharge air passage 110d through the vent hole 112d.

[0048] As shown in Fig. 2A, a notch 113a through which the vent holes 111a and 112a communicate with each other is formed in the circumferential surface of the valve body 106 corresponding to the intake air passage 109a and the suction air passage 110a. Similarly, a notch 113b through which the vent holes 111b and 112b communicate with each other is formed in the circumferential surface of the valve body 106 corresponding to the intake air passage 109b and the suction air passage 110b, at a position displaced from the notch 113a in the axial direction and to be phase-shifted from the notch 113a in the rotating direction of the valve body 106.

[0049] Therefore, the characteristic feature of this embodiment resides in that the blind hole-like hollow portion 114 having a circular section is formed in the valve body 106 to extend in the axial direction and that this hollow portion 114 serves as the exhaust air passage. The conventional exhaust air passages 9c and 9d and reverse air passage 31 shown in Fig. 9 and Figs. 10A and 10B are replaced with the hollow portion 114, so that the reverse air passage 31 and the notches 32a and 32b formed in the valve body 6 become unnecessary.

[0050] The switching operation between intake and exhaustion done by the rotary valve having this arrangement will be described with reference to Fig. 3.

[0051] When the valve body 106 is rotated in an interlocked manner with rotation of the printing press, the

communication hole 115d of the valve body 106 opposes the vent hole 112d of the sleeve 104, and the hollow portion 114 serving as the exhaust air passage and the discharge air passage 110d communicate with each other through the communication hole 115d and the vent hole 112d. Therefore, exhaust air from the exhaust pump 136 flows through the hollow portion 114 and the air passage 110d, and is discharged from the nozzle 137b to blow the air to the sheets (step S11).

[0052] When the valve body 106 is continuously rotated, the notch 113a opposes the vent holes 112a and 111a, and the intake air passage 109a and the suction air passage 110a communicate with each other through the notch 113a. Thus, as shown in Fig. 2A, intake air A from the intake pump 134 flows through the air passages 109a and 110a that communicate with each other, and is supplied to the suction head 135a serving as the first sucker, to draw a sheet by suction (step S12).

[0053] When the valve body 106 is continuously rotated, the communication hole 115c opposes the vent hole 112c of the sleeve 104, and the hollow portion 114 serving as the exhaust air passage and the discharge air passage 110c communicate with each other through the communication hole 115c and a vent hole 111c, as shown in Fig. 2B. Exhaust air B from the exhaust pump 134 is supplied to the nozzle 137a, serving as the leveling foot, through the hollow portion 114 and the air passage 110c. The air is blown to a portion between the highest sheet and the second sheet, to separate them from each other (step S13).

[0054] When the valve body 106 is continuously rotated, the notch 113b opposes the vent holes 112b and 111b, and the intake air passage 109b and the suction air passage 110b communicate with each other through the notch 113b. Therefore, intake air from the intake pump 134 is supplied to the suction head 135b, serving as the second sucker, through the air passages 109b and 110b, to draw a sheet by suction (step S14).

[0055] Simultaneously, the communication hole 115a of the valve body 106 opposes the vent hole 112a, and the hollow portion 114 and the suction air passage 110a communicate with each other through the communication hole 115a and the vent hole 112a. The exhaust air from the exhaust pump 136 is supplied to the suction head 135a, serving as the first sucker, through the hollow portion 114 and the air passage 110a. The sheet is quickly released from the suction head 135a that has ended the suction operation (step S15). As a result, a sheet which is conveyed to above the feeder board while being drawn by the suction head 135b, serving as the second sucker will not be cut or bent.

[0056] When the suction head 135b during sheet conveyance is located above the feeder board, as the valve body 106 is rotated, the communication hole 115b opposes the vent hole 112b, and the hollow portion 114 and the suction air passage 110b communicate with each other through the communication hole 115b and vent hole 112b. The exhaust air from the exhaust pump

136 is supplied to the suction head 135b through the hollow portion 114 and the air passage 110b. The sheet is quickly released from the suction head 135b, and is supplied onto the feeder board (step S16).

[0057] According to this embodiment, during the air discharge operation, the air is supplied from the exhaust pump 136 to the nozzles 137a and 137b through the hollow portion 114 and the communication holes 115c and 115d. Therefore, the air blowing and leveling foot operation can be reliably performed within a short period of time with sufficiently high-pressurized air.

[0058] During the air discharge operation, the hollow portion 114, the communication holes 115c and 115d, the vent holes 111c and 111d, and the discharge air passages 110c and 110d are located in a substantially straight line, so that the air loss during the leveling foot operation and air blowing is decreased. Therefore, an exhaust pump having a comparatively small power can be used.

[0059] Since the exhaust air is supplied through the hollow portion 114 of the valve body 106 which has a large sectional area, the pressure loss of the air at the valve portion is decreased. Also, the exhaust air passages 9c and 9d conventionally formed in the main body 2 or the vent holes 11c and 11d conventionally formed in the sleeve 4 become unnecessary, and only the hollow portion 114 and the communication holes 115a to 115d need be formed in the valve body 106. Therefore, the structure is simplified and machining becomes easy.

[0060] Since the notches 32a and 32b of the valve body 6 and the reverse air passage 31, which are conventionally necessary, becomes unnecessary, the structure is simplified. Since the hollow portion 114, the communication holes 115a and 115b, the vent holes 111a and 111b, and the suction air passages 110a and 110b are located in a substantially straight line, the air pressure loss during reverse blowing is decreased.

[0061] Fig. 4 shows a rotary valve according to the first embodiment of the present invention. Fig. 5A shows a section taken along the line III-III of Fig. 4, and Fig. 5B shows section taken along the line IV-IV of Fig. 4. In Fig. 4 and Figs. 5A and 5B, portions that are identical to those of Fig. 1 and Figs. 2A and 2B are denoted by the same reference numerals as in Fig. 1 and Figs. 2A and 2B, and a detailed description thereof will be omitted.

[0062] The first embodiment is different from the embodiment shown in Fig 1 in that, as shown in Fig. 4, a partition wall 114c is formed at the center of a hollow portion 114 to divide the hollow portion 114 into a first hollow portion 114a and a second hollow portion 114b.

[0063] As shown in Fig. 6, the first hollow portion 114a is connected to an intake pump 134 through a hose 119a to constitute an intake air passage. The second hollow portion 114b is connected to an exhaust pump 136 through a hose 119b, in the same manner as in the first embodiment. Air passages 116a and 116b formed in the upper end side of a main body 102 are connected to the

exhaust pump 136 through hoses 117a and 117b to constitute an exhaust air passage.

[0064] A communication hole 215a (Fig. 5A), where the hollow portion 114 extends to communicate with a suction air passage 110a through a vent hole 112a, is formed in the valve body 106. Similarly, a communication hole 215b, where the hollow portion 114 extends to communicate with the suction air passage 110b through a vent hole 112b, is formed in the valve body 106, at a position displaced from the communication hole 215a in the axial direction and to be phase-shifted from the communication hole 215a in the rotating direction. A communication hole 215c (Fig. 5B), where the hollow portion 114 extends to communicate with a discharge air passage 110c through a vent hole 112c, is also formed in the valve body 106. Similarly, a communication hole 215d, where the hollow portion 114 extends to communicate with a suction air passage 110d through a vent hole 112d, is formed in the valve body 106, at a position displaced from the communication hole 215c in the axial direction and to be phase-shifted from the communication hole 215c in the rotating direction.

[0065] The switching operation between intake and exhaustion done by the rotary valve having this arrangement will be described with reference to Fig. 6.

[0066] When the valve body 106 is rotated in an interlocked manner with rotation of the printing press, the communication hole 215d of the valve body 106 opposes the vent hole 112d of the sleeve 104, and the second hollow portion 114b serving as the exhaust air passage and the discharge air passage 110d communicate with each other through a communication hole 115d and the vent hole 112d. Exhaust air from the exhaust pump 136 is supplied to a nozzle 137b through the second hollow portion 114b and the air passage 110d, to blow air to the sheets (step S21).

[0067] When the valve body 106 is continuously rotated, its communication hole 215a opposes the vent hole 112a, and the first hollow portion 114a serving as the intake air passage and the suction air passage 110a communicate with each other through the communication hole 215a and the vent hole 112a. Intake air from the intake pump 134 is supplied to a suction head 135a, serving as the first sucker, through the suction air passage 110a and the first hollow portion 114a, to draw a sheet by suction (step S22).

[0068] When the valve body 106 is continuously rotated, the communication hole 215c opposes the vent hole 112c of a sleeve 104, and the second hollow portion 114b serving as the exhaust air passage and the discharge air passage 110c communicate with each other through the communication hole 215c and the vent hole 112c, as shown in Fig. 5B. Exhaust air B from the exhaust pump 136 is supplied to a nozzle 137a, serving as the leveling foot, through the second hollow portion 114b and the suction air passage 110c. The air is blown to a portion between the highest sheet and the second sheet, to separate them from each other (step S23).

[0069] When the valve body 106 is continuously rotated, the communication hole 215b of the valve body 106 opposes the vent hole 112b, and the first hollow portion 114a and the suction air passage 110a communicate with each other through the communication hole 215b and the vent hole 112b. Intake air from the intake pump 134 is supplied to a suction head 135b, serving as the second sucker, through the first hollow portion 114a and the suction air passage 110a, to draw a sheet by suction (step S24).

[0070] Simultaneously, a notch 113a opposes the vent hole 112a and a vent hole 111a, and an exhaust air passage 116a and the discharge air passage 110a communicate with each other through the notch 113a, as shown in Fig. 5A. Thus, the exhaust air B from the exhaust pump 136 is supplied to the suction head 135a, serving as the first sucker, through the air passages 116a and 110a. The sheet is quickly released from the suction head 135a that has ended the suction operation (step S25). As a result, a sheet which is conveyed to above the feeder board while being drawn by the suction head 135b, serving as the second sucker, will not be cut or bent.

[0071] When the second suction head 135b during sheet conveyance is located above the feeder board, as the valve body 106 is rotated, a notch 113b opposes the vent hole 112b and a vent hole 111b, and an exhaust air passage 116b and the discharge air passage 110b communicate with each other through the notch 113b. The exhaust air from the exhaust pump 136 is supplied to the suction head 135b through the air passages 116b and 110b. The sheet is quickly released from the suction head 135b, and is supplied to above the feeder board (step S26).

[0072] According to this embodiment, during the suction operation, the air is supplied from the intake pump 134 to the suction heads 135a and 135b through the first hollow portion 114a and the communication holes 215a and 215b. Therefore, the suction operation can be reliably performed within a short period of time with sufficiently high-pressurized air.

[0073] During the suction operation, the first hollow portion 114a, the communication holes 215a and 215b, and the vent holes 112a and 112b are located in a substantially straight line, so that the pressure loss of the suction air is decreased. Since the first hollow portion 114a of the valve body 106, which has a large sectional area, serves as the intake air passage, the pressure loss of the air at the valve portion is small.

[0074] Fig. 7 shows another rotary valve, and Fig. 8 shows a section taken along the line V - V of Fig. 7.

[0075] A rotary valve 120 of the embodiment according to Fig. 7 controls to supply/stop supplying of the air of the first sucker of the sheet feeding unit. More specifically, a through hole 122 is formed in a main body 121 of the rotary valve 120. A valve body 123 which rotates in an interlocked manner with the printing press is rotatably engaged with the inner surface of the through

hole 122. An intake air passage 124 and a suction air passage 125 are formed in the upper portion of the main body 121. The air passage 124 has an open upper end and a lower end that communicates with the through hole 122. The suction air passage 125 has an L-shaped section.

[0076] The upper end open side of the intake air passage 124 is connected to an intake pump through a hose 127. The upper end open side of the suction air passage 125 is connected to a suction head, serving as the first sucker, through a hose 128. A notch 126a is formed in part of the circumferential surface of the valve body 123. A blind hole-like hollow portion 126b is formed in the valve body 123 to extend in the axial direction. A communication hole 126c, through which the hollow portion 126b and the suction air passage 125 communicate with each other, is also formed in the valve body 123. The open end side of the hollow portion 126b is connected to an exhaust pump through a hose 129. The hollow portion 126b constitutes an exhaust air passage.

[0077] In this arrangement, when the valve body 123 is rotated in an interlocked manner with the printing press, the notch 126a opposes the intake air passage 124 and the suction air passage 125, and the two air passages 124 and 125 communicate with each other through the notch 126a. Intake air A is supplied to a suction head, serving as the first sucker, through the air passages 124 and 125, to draw a sheet by suction.

[0078] When the valve body 123 is continuously rotated, air intake is controlled by another rotary valve (not shown), and a sheet is drawn by suction with a suction head serving as the second sucker.

[0079] Simultaneously, the communication hole 126c of the valve body 123 opposes the suction air passage 125, and the hollow portion 126b and the suction air passage 125 communicate with each other. Exhaust air B from the exhaust pump is supplied to the suction head, serving as the first sucker, through the suction air passage 125, and a sheet is quickly released from the suction head that has ended the suction operation. As a result, a sheet which is drawn by the suction head, serving as the second sucker, and is conveyed to above the feeder board will not be cut or bent.

[0080] According to this embodiment, during the reverse operation, a sufficient amount of air can be supplied to the suction heads through the hollow portion 126b and the communication hole 126c within a short period of time. Since the hollow portion 126b, the communication hole 126c, and the suction air passage 125 are located in a substantially straight line and the distance between the hollow portion 126b and the suction air passage 125 reaches a minimum, the air pressure loss in the air passage is decreased.

[0081] In this embodiment, when the intake air operation time is short, an exhaust pump 137 may be connected through the hose 127 so that the air passage 124 serves as the exhaust air passage, and an intake pump 134 may be connected to the open end side of the hollow

portion 126b through the hose 129 so that the hollow portion 126b serves as the intake air passage.

[0082] In the embodiments according to Figs. 1 and 4, the sleeve 104 formed with the vent holes 111a and 111b is provided. However, the valve body 123 may be directly engaged in the through hole 122 of the main body 121, as in the third embodiment. With this structure, the vent holes 111a to 111d become unnecessary. Specifically, it suffices if the valve body is rotatably held in a cylinder including a sleeve formed in the main body.

[0083] In the above embodiments, the rotary valve is applied to the sheet feeding unit. However, the present invention is not limited to this, but can similarly be applied to, e.g., the convertible cylinder unit or a sheet discharge unit of a printing press main body. Although the hollow portion is formed to have a circular section, it can have various other sections, e.g., an elliptical section, a square section, or a triangular section. Although only one rotary valve is used in the above embodiments, a series of a plurality of rotary valves may be used.

[0084] As has been described above, according to the present invention, the valve body of the rotary valve has a hollow portion and a notched portion respectively connected to different air sources, and the hollow portions and notched portions are combined with air passages that require predetermined characteristics. If the hollow portion is used as an air passage having a short air operation time and the notched portion is used as an air passage having a long air operation time, predetermined air pressures required for the respective air passages can be obtained.

[0085] Since the hollow portion and the notched portion are formed in the valve body, the entire rotary valve can be made compact. The reverse air passage from the suction unit can also be constituted by a hollow portion or a notch formed in the valve body, thus decreasing the manufacturing cost.

Claims

1. A rotary valve, which is interposed between a first air source (136) and a second air source (134) for supplying air, and at least one air unit (135a, 135b, 137a, 137b) that performs a predetermined operation upon reception of the air supplied from said first and second air sources (136, 134), to supply/stop supplying air, comprising:

a valve body (106) rotatably driven in a cylinder;

a main body (102) for rotatably supporting said valve body (106); and

notches (113a, 113b) formed in a circumferential surface of said valve body (106) to be connected to the first air source (136), and further comprising a first hollow portion (114a) formed

in said valve body (106), and wherein said rotary valve comprises

a first air passage (116a, 116b) formed in said main body (102) and having one end side connected to said first air source (136) and the other end side that opens in an inner circumferential surface of said cylinder,

a second air passage (110a, 110b) formed in said main body (102) and having one end side connected to the first air unit (135a, 135b) and the other end side that opens in said inner circumferential surface of said cylinder, said first air passage (116a, 116b) being connected to said second air passage (110a, 110b) through said notches (113a, 113b), and

a first communication hole (215a, 215b) formed in said valve body (106) to connect said first hollow portion (114a) and said second air passage (110a, 110b) to each other, **characterized in that** the first hollow portion (114a) is formed in said valve body (106) in an axial direction and having, in one end face of said valve body (106), an opening through which said first hollow portion (114a) is connected to the second air source (134), and **in that** the rotary valve further comprises

a third air passage (110c, 110d) formed in said main body (102) and having one end side connected to the second air unit (137a, 137b) and the other end side that opens in an inner circumferential surface of said cylinder,

a second hollow portion (114b) formed in said valve body (106) in an axial direction and connected to said first air source (136), and

a second communication hole (215c, 215d) formed in said valve body (106) to connect said second hollow portion (114b) and said third air passage (110c, 110d) to each other.

2. A rotary valve according to claim 1, wherein one of said first and second air sources comprises an intake pump for supplying intake air, and the other one of said first and second air sources comprises an exhaust pump for supplying exhaust air.

3. A rotary valve according to claim 2, wherein said one end side of said first air passage is connected to said exhaust pump, said one end side of said second air passage is connected to said suction unit, said first hollow portion is connected to said intake pump, and said suction unit is connected to said exhaust pump through said second air passage, said

notch, and said first air passage, and to said intake pump through said second air passage, said first communication hole, and said first hollow portion.

4. A rotary valve according to claim 1, wherein one of said first and second air sources comprises an intake pump for supplying intake air, the other one of said first and second air sources comprises an exhaust pump for supplying exhaust air, one of said first and second air units comprises a suction unit which performs suction operation upon receiving the intake air from said intake pump, and the other one of said first and second air units comprises a discharge unit which performs discharge operation upon receiving the exhaust air from said exhaust pump.
5. A rotary valve according to claim 4, wherein said first and second communication holes are arranged to be displaced from each other in an axial direction of said cylinder, said first communication hole is arranged at the same position as an opening of said second air passage in the axial direction of said cylinder, and said second communication hole is arranged at the same position as an opening of said third air passage in the axial direction of said cylinder.
6. A rotary valve according to claim 1, wherein openings of said first and third air passages are arranged to be displaced from each other in an axial direction of said cylinder.
7. A rotary valve according to claim 1, wherein openings of said first and second air passages are arranged at the same position in an axial direction of said cylinder and to be phase-shifted from each other in a rotational direction of said valve body.

Patentansprüche

1. Ein Drehschieber, der sich zwischen einer ersten Luftquelle (136) und einer zweiten Luftquelle (134) zum Zuführen von Luft befindet, und mindestens eine Lufteinheit (135a, 135b, 137a, 137b), die auf Empfang der von der ersten und zweiten Luftquelle zugeführten Luft hin einen vorbestimmten Vorgang ausführt, um Luft zuzuführen bzw. die Luftzufuhr zu stoppen, umfassend:
 - einen Schieberkörper (106), der in einem Zylinder rotierbar angetrieben wird;
 - einen Hauptkörper (102), um den Schieberkörper (106) rotierbar zu tragen/stützen; und
 - Ausnehmungen (113a, 113b), die in einer Um-

fangsfläche des Schieberkörpers (106) gebildet sind, um mit der ersten Luftquelle (136) verbunden zu werden, und weiterhin umfassend einen ersten hohlen Abschnitt (114a), der im Schieberkörper (106) gebildet ist, und wobei der Drehschieber umfaßt

einen ersten Luftdurchtritt (116a, 116b), der im Hauptkörper (102) gebildet ist und dessen eine Endseite mit der ersten Luftquelle (136) verbunden ist und dessen andere Endseite sich in einer inneren Umfangsfläche des Zylinders öffnet,

einen zweiten Luftdurchtritt (110a, 110b), der im Hauptkörper (102) gebildet ist und dessen eine Endseite mit der ersten Lufteinheit (135a, 135b) verbunden ist und dessen andere Endseite sich in der inneren Umfangsfläche des Zylinders öffnet, wobei der erste Luftdurchtritt (116a, 116b) durch die Ausnehmungen (113a, 113b) mit dem zweiten Luftdurchtritt (110a, 110b) verbunden ist, und

ein erstes Kommunikationsloch (215a, 215b), das im Schieberkörper (106) gebildet ist, um den ersten hohlen Abschnitt (114a) und den zweiten Luftdurchtritt (110a, 110b) miteinander zu verbinden, **dadurch gekennzeichnet, daß** der erste hohle Abschnitt (114a) im Schieberkörper (106) in einer axialen Richtung gebildet ist und in einer Endfläche des Schieberkörpers (106) eine Öffnung aufweist, durch die der erste hohle Abschnitt (114a) mit der zweiten Luftquelle (134) verbunden ist, und daß der Drehschieber weiterhin umfaßt

einen dritten Luftdurchtritt (110c, 110d), der im Hauptkörper (102) gebildet ist und dessen eine Endseite mit der zweiten Lufteinheit (137a, 137b) verbunden ist und dessen andere Endseite sich in einer inneren Umfangsfläche des Zylinder öffnet,

einen zweiten hohlen Abschnitt (114b), der im Schieberkörper (106) in einer axialen Richtung gebildet ist und mit der ersten Luftquelle (136) verbunden ist, und

ein zweites Kommunikationsloch (215c, 215d), das im Schieberkörper (106) gebildet ist, um den zweiten hohlen Abschnitt (114b) und den dritten Luftdurchtritt (110c, 110d) miteinander zu verbinden.

2. Ein Drehschieber nach Anspruch 1, wobei eine der ersten und zweiten Luftquelle eine Einlaßpumpe zum Zuführen von Einlaßluft umfaßt und die andere

der ersten und zweiten Luftquelle eine Abluftpumpe zum Zuführen von Abluft umfaßt.

3. Ein Drehschieber nach Anspruch 2, wobei die eine Endseite des ersten Luftdurchtritts mit der Abluftpumpe verbunden ist, die eine Endseite des zweiten Luftdurchtritts mit der Saugereinheit verbunden ist, der erste hohle Abschnitt mit der Einlaßpumpe verbunden ist und die Saugereinheit durch den zweiten Luftdurchtritt, die Ausnehmung und den ersten Luftdurchtritt mit der Abluftpumpe und durch den zweiten Luftdurchtritt, das erste Kommunikationsloch und den ersten hohlen Abschnitt mit der Einlaßpumpe verbunden ist. 5
4. Ein Drehschieber nach Anspruch 1, wobei eine der ersten und zweiten Luftquelle eine Einlaßpumpe zum Zuführen von Einlaßluft umfaßt, die andere der ersten und zweiten Luftquelle eine Abluftpumpe zum Zuführen von Abluft umfaßt, eine der ersten und zweiten Lufteinheit eine Saugereinheit umfaßt, die auf den Empfang der Einlaßluft von der Einlaßpumpe hin einen Saugvorgang durchführt, und die andere der ersten und zweiten Lufteinheit eine Ausstoßeinheit umfaßt, die auf den Empfang der Abluft von der Abluftpumpe hin einen Ausstoßvorgang durchführt. 10
5. Ein Drehschieber nach Anspruch 4, wobei die ersten und zweiten Kommunikationslöcher angeordnet sind, um voneinander in einer axialen Richtung des Zylinders verschoben zu sein, wobei das erste Kommunikationsloch an der gleichen Position wie eine Öffnung des zweiten Luftdurchtritts in der axialen Richtung des Zylinders angeordnet ist, und das zweite Kommunikationsloch an der gleichen Position wie eine Öffnung des dritten Luftdurchtritts in der axialen Richtung des Zylinders angeordnet ist. 15
6. Ein Drehschieber nach Anspruch 1, wobei Öffnungen der ersten und dritten Luftdurchtritte angeordnet sind, um voneinander in einer axialen Richtung des Zylinders verschoben zu sein. 20
7. Ein Drehschieber nach Anspruch 1, wobei Öffnungen der ersten und zweiten Luftdurchtritte an der gleichen Position in einer axialen Richtung des Zylinders angeordnet sind und voneinander in einer Rotationsrichtung des Schieberkörpers phasenverschoben werden sollen. 25

Revendications

1. Distributeur rotatif disposé entre une première source d'air (136) et une seconde source d'air (134) destinées à transmettre de l'air, et au moins une unité pneumatique (135a, 135b, 137a, 137b) qui exécute 55

une opération prédéterminée lors de la réception de l'air transmis par les première et seconde sources d'air (136, 134), afin que la transmission d'air soit assurée ou arrêtée, comprenant :

un corps (106) de distributeur entraîné en rotation dans un cylindre,
un corps principal (102) destiné à supporter en rotation le corps de distributeur (106), et des encoches (113a, 113b) formées à une surface circonférentielle du corps de distributeur (106) afin qu'elles soient reliées à la première source d'air (136), et comportant en outre une première partie creuse (114a) formée dans le corps de distributeur (106), et dans lequel le distributeur rotatif comprend :

un premier passage d'air (116a, 116b) formé dans le corps principal (102) et ayant un premier côté d'extrémité relié à la première source d'air (136) et l'autre côté d'extrémité qui débouche à une surface circonférentielle interne du cylindre,
un second passage d'air (110a, 110b) formé dans le corps principal (102) et ayant un premier côté d'extrémité relié à la première unité pneumatique (135a, 135b) et l'autre côté d'extrémité qui débouche à la surface circonférentielle interne du cylindre, le premier passage d'air (116a, 116b) étant relié au second passage d'air (110a, 110b) par les encoches (113a, 113b), et un premier trou de communication (215a, 215b) formé dans le corps de distributeur (106) afin qu'il relie la première partie creuse (114a) et le second passage d'air (110a, 110b) l'un à l'autre, **caractérisé en ce que** la première partie creuse (114a) est formée dans le corps de distributeur (106) dans une direction axiale et elle comporte, à une première face d'extrémité du corps de distributeur (106), une ouverture par laquelle la première partie creuse (114a) est reliée à la seconde source d'air (134), et **en ce que** le distributeur rotatif comporte en outre :

un troisième passage d'air (110c, 110d) formé dans le corps principal (102) et ayant un premier côté d'extrémité relié à la seconde unité pneumatique (137a, 137b) et l'autre côté d'extrémité qui débouche à une surface circonférentielle interne du cylindre,
une seconde partie creuse (114b) formée dans le corps de distributeur (106) en direction axiale et reliée à la première source d'air (136), et
un second trou de communication (215c,

- 215d) formé dans le corps de distributeur (106) pour relier la seconde partie creuse (114b) et le troisième passage d'air (110c, 110d) l'un à l'autre.
2. Distributeur rotatif selon la revendication 1, dans lequel l'une des première et seconde sources d'air comporte une pompe d'admission destinée à transmettre de l'air d'admission, et l'autre des première et seconde sources d'air comporte une pompe d'évacuation destinée à transmettre de l'air d'évacuation. 5 10
 3. Distributeur rotatif selon la revendication 2, dans lequel le premier côté d'extrémité du premier passage d'air est relié à la pompe d'évacuation, le premier côté d'extrémité du second passage d'air est relié à l'unité d'aspiration, la première partie creuse est reliée à la pompe d'admission, et l'unité d'aspiration est reliée à la pompe d'évacuation par l'intermédiaire du second passage d'air, de l'encoche et du premier passage d'air, et à la pompe d'admission par l'intermédiaire du second passage d'air, du premier trou de communication et de la première partie creuse. 15 20 25
 4. Distributeur rotatif selon la revendication 1, dans lequel l'une des première et seconde sources d'air comprend une pompe d'admission destinée à transmettre de l'air d'admission, l'autre des première et seconde sources d'air comprend une pompe d'évacuation destinée à transmettre de l'air d'évacuation, l'une des première et seconde unités pneumatiques comprend une unité d'aspiration qui exécute une opération d'aspiration lors de la réception d'air d'admission de la pompe d'admission, et l'autre des première et seconde unités pneumatiques comporte une unité de refoulement qui effectue une opération de refoulement lors de la réception de l'air d'évacuation de la pompe d'évacuation. 30 35 40
 5. Distributeur rotatif selon la revendication 4, dans lequel les premier et second trous de communication sont disposés afin qu'ils soient décalés l'un par rapport à l'autre dans la direction axiale du cylindre, le premier trou de communication est placé à la même position qu'une ouverture du second passage d'air dans la direction axiale du cylindre, et le second trou de communication a la même position qu'une ouverture du troisième passage d'air dans la direction axiale du cylindre. 45 50
 6. Distributeur rotatif selon la revendication 1, dans lequel des ouvertures des premier et troisième passages d'air sont disposées afin qu'elles soient décalées les unes par rapport aux autres dans la direction axiale du cylindre. 55
 7. Distributeur rotatif selon la revendication 1, dans lequel des ouvertures des premier et second passages d'air ont la même position dans la direction axiale du cylindre et sont déphasées l'une par rapport à l'autre dans la direction de rotation du corps de distributeur.

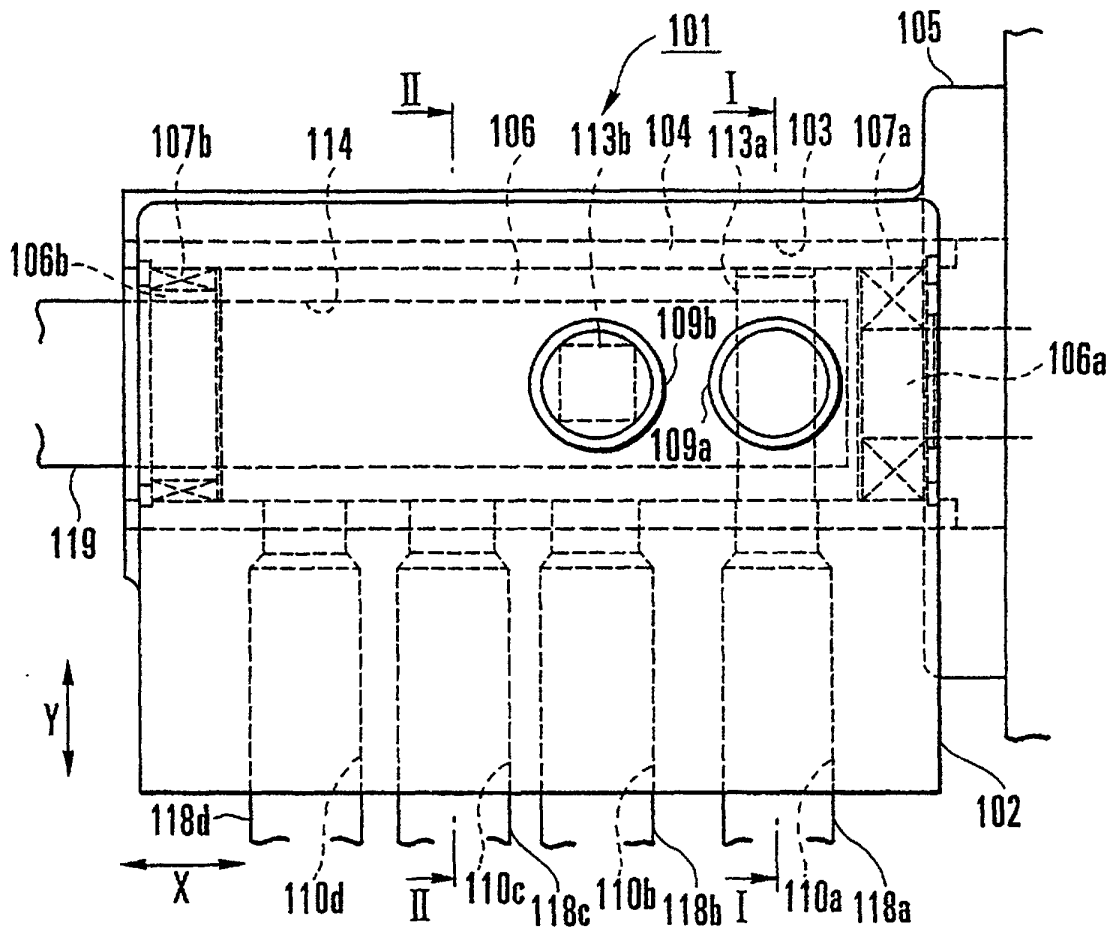


FIG. 1

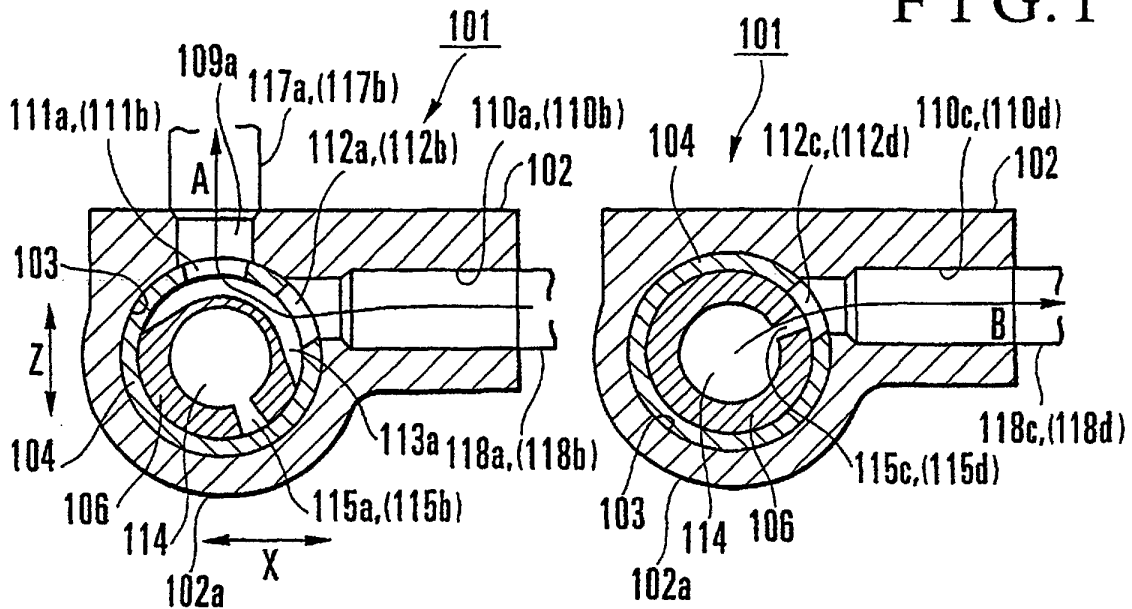


FIG. 2A

FIG. 2B

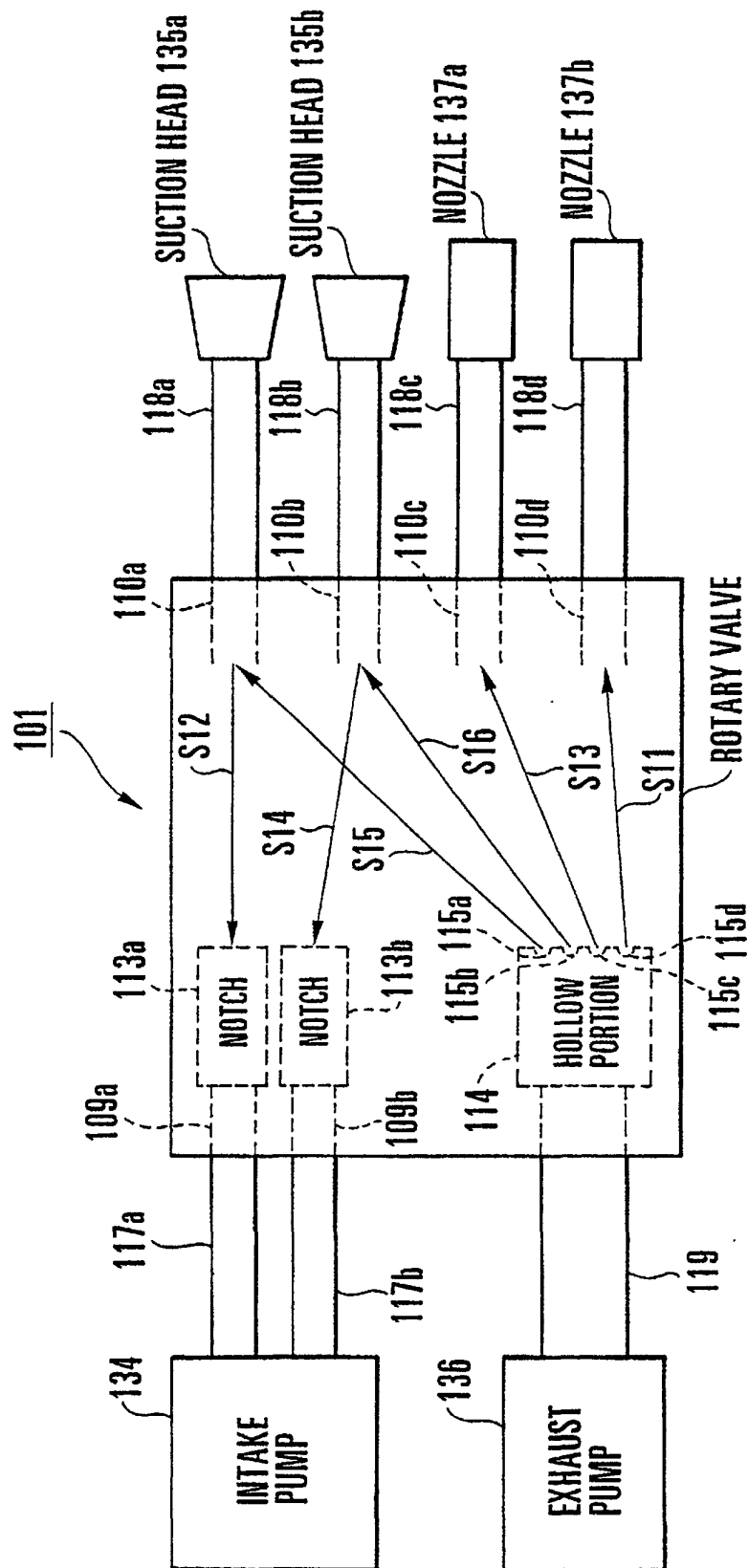


FIG. 3

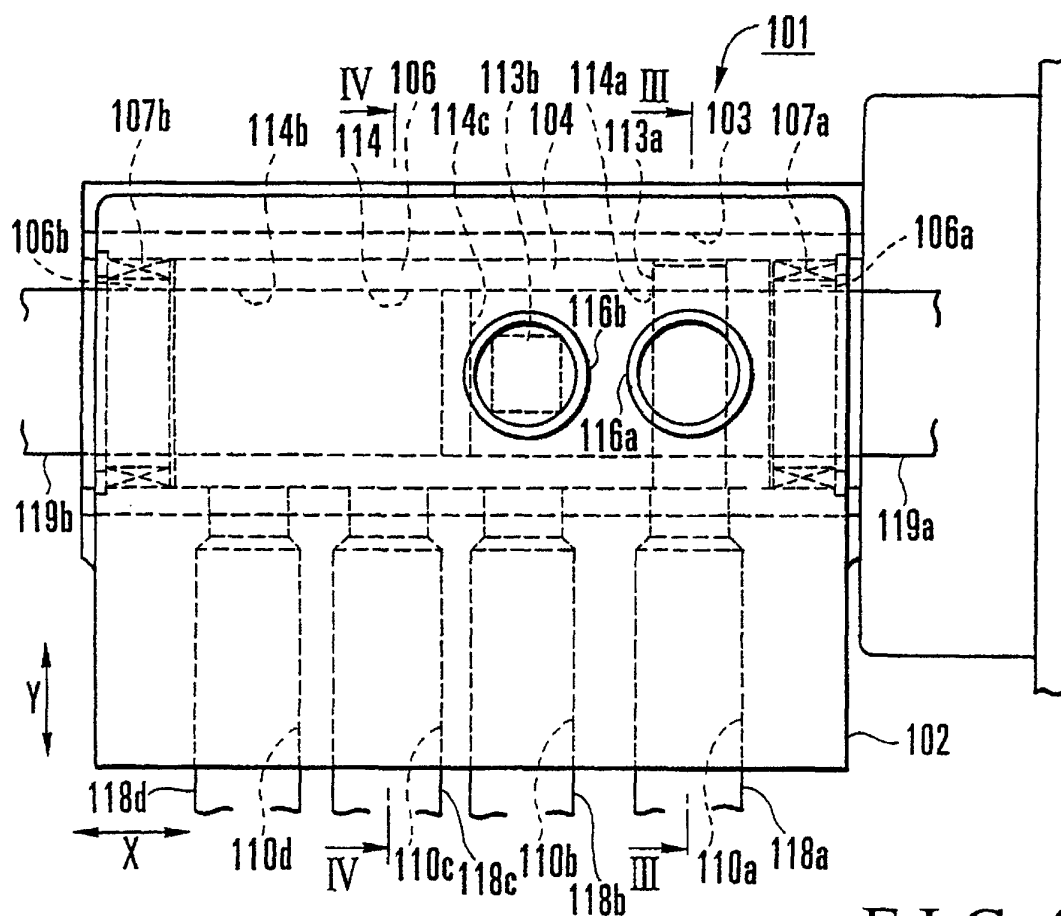


FIG. 4

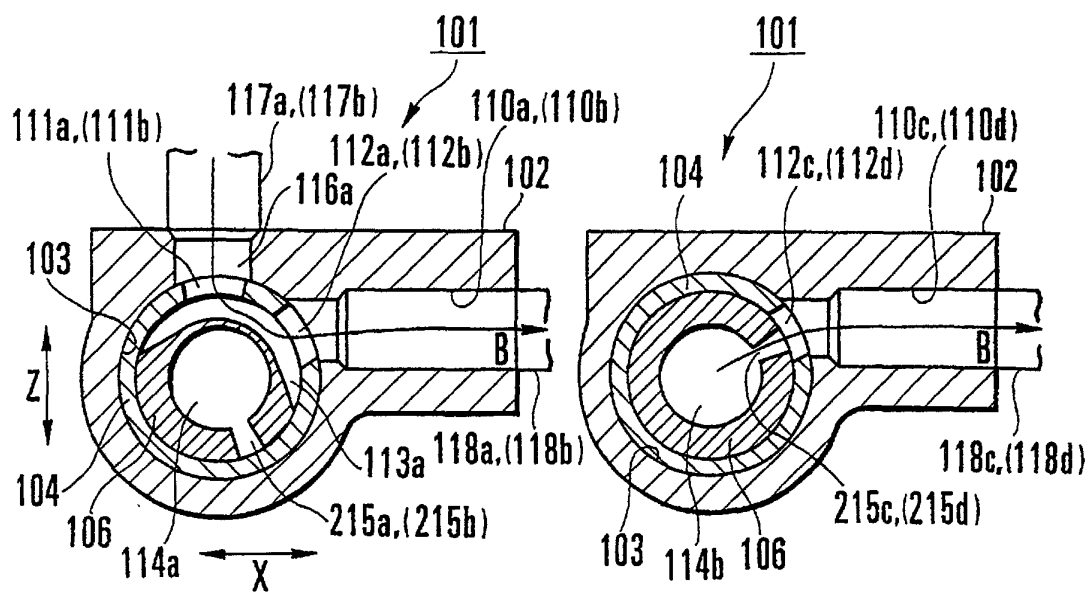


FIG. 5A

FIG. 5B

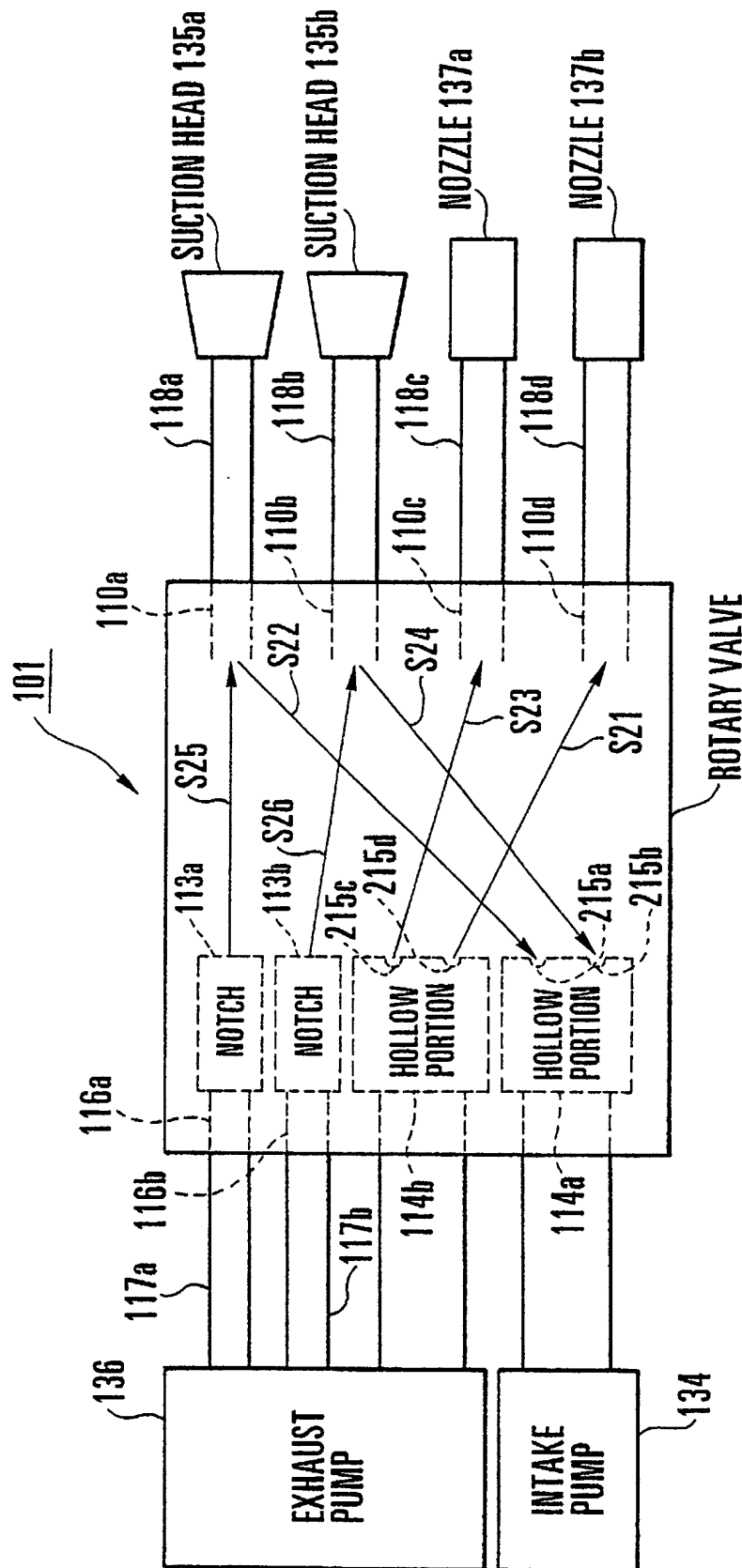


FIG. 6

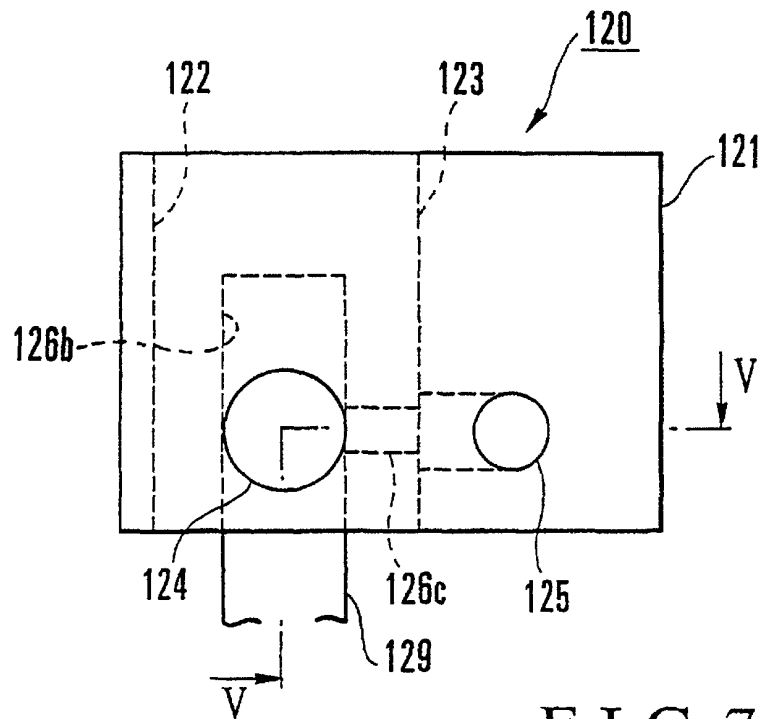


FIG. 7

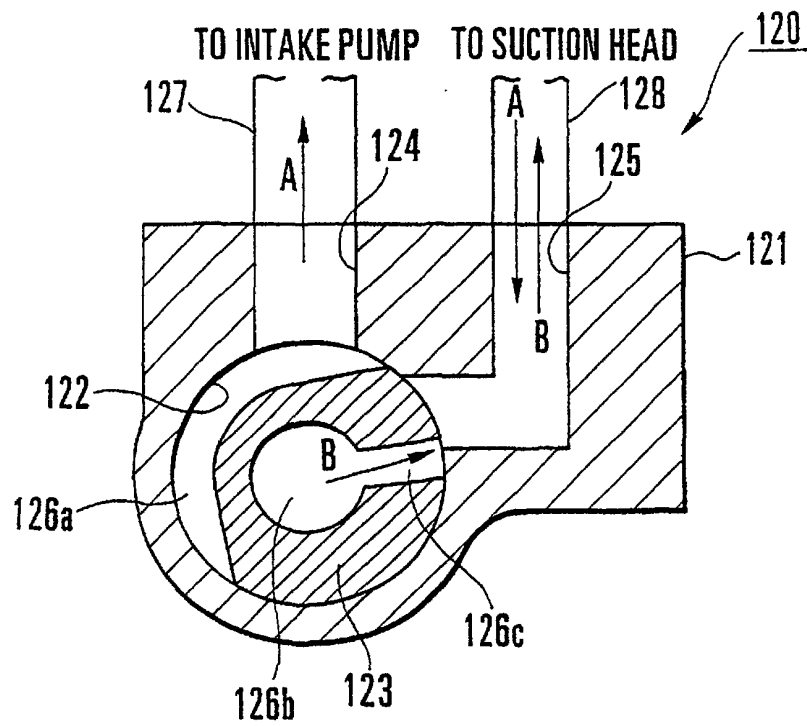


FIG. 8

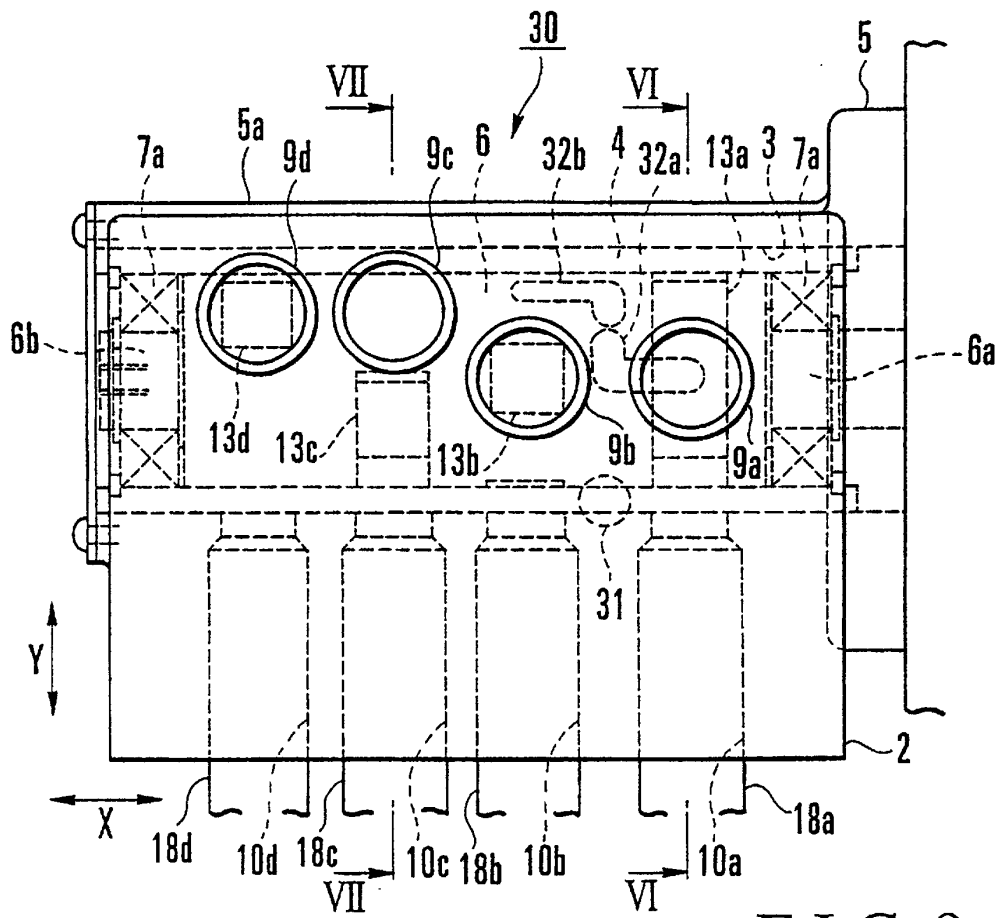


FIG. 9
PRIOR ART

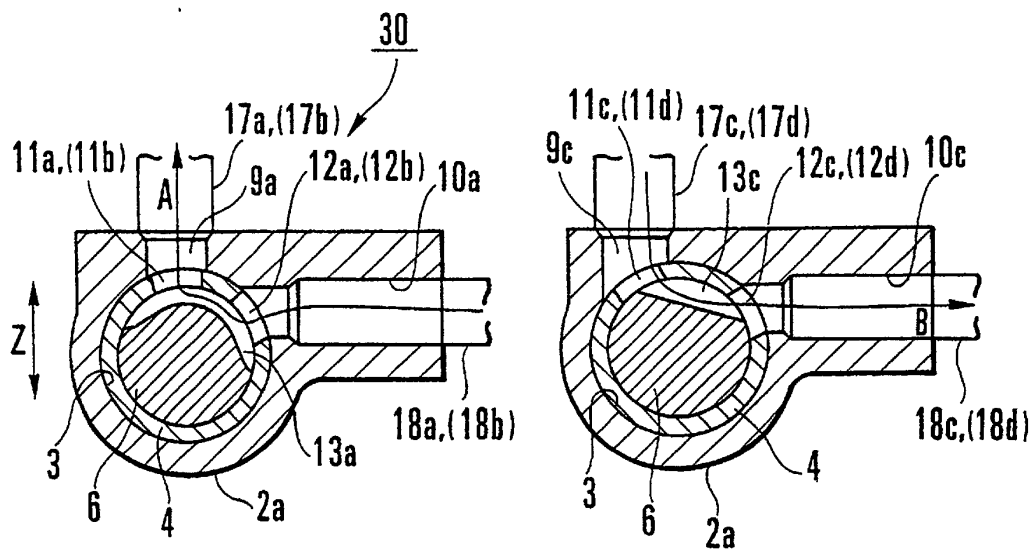


FIG. 10A
PRIOR ART

FIG. 10B
PRIOR ART

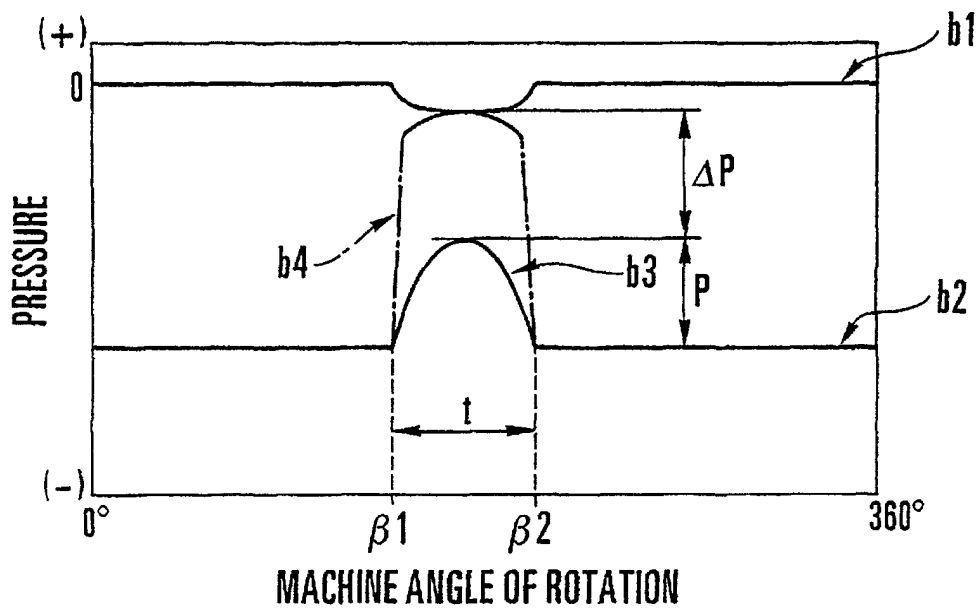


FIG. 11

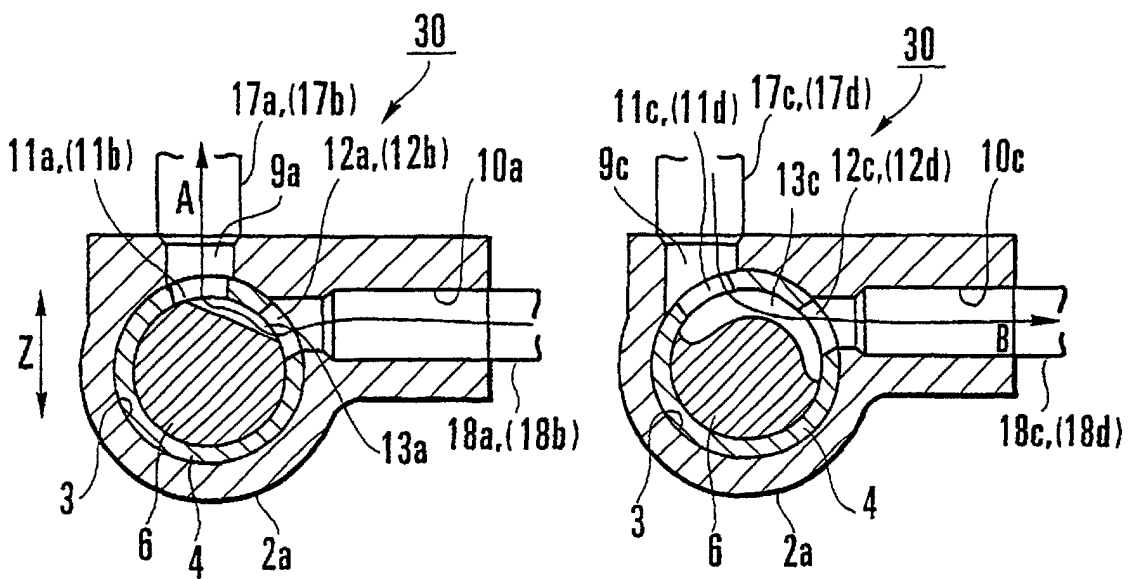


FIG. 12A
PRIOR ART

FIG. 12B
PRIOR ART

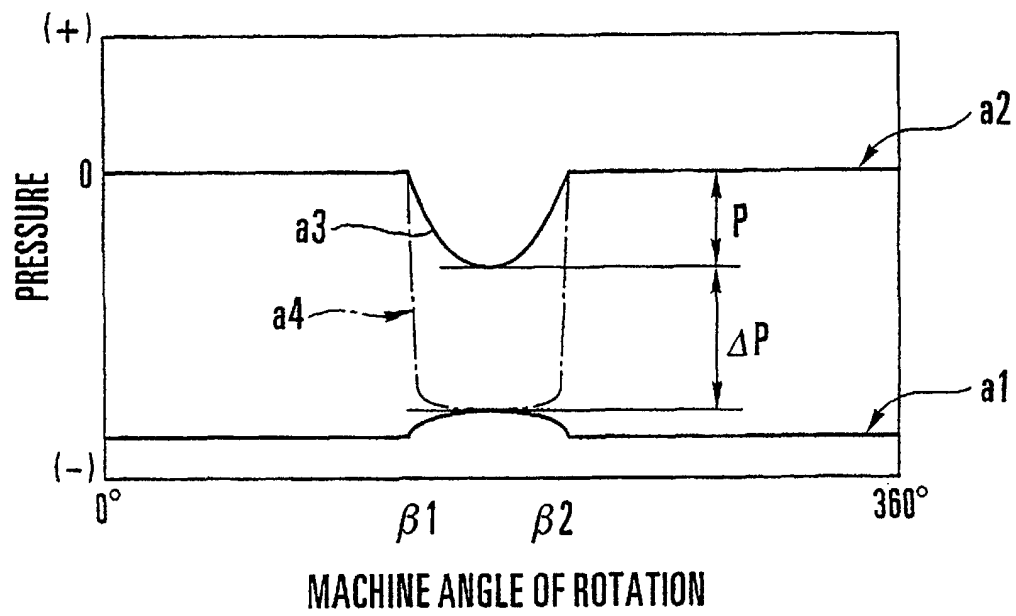


FIG. 13