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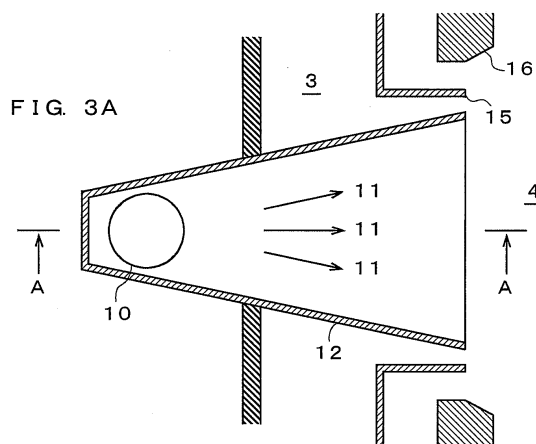
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(54) **BURNER, AND COMBUSTION EQUIPMENT AND BOILER COMPRISING BURNER**

(57) The present invention provides a solid fuel burner, which, while rendering the capacity larger than that in the conventional art, can suppress an increase in an unignited region and thus can realize the prevention of an increase in NO_x concentration in a combustion gas and the prevention of a lowering in combustion efficiency, and a combustion equipment and boiler including the burner. The burner includes a fuel-containing fluid supply nozzle (12) which supplies a fuel-containing fluid, from a connecting part in a fluid transfer flow passage (10) for transferring a fuel-containing fluid including a fuel and a medium for transfer of the fuel, toward an outlet part provided on the wall of a furnace (4). The fuel-containing fluid supply nozzle (12) in its cross section perpendicular to the direction of flow of the fluid is in a rectangular, elliptical, or substantially elliptical form having major and minor axis parts from a connecting part (10a) in the fluid transfer flow passage (10) toward the outlet part provided on the wall surface of the furnace (4). Further, the area of a cross section perpendicular to the direction of flow of the fluid is gradually increased from the connecting part in the fluid transfer flow passage (10) toward the

outlet part. One or more air supply nozzles (15) for supplying combustion air are provided on the outer peripheral part of the nozzle (12).



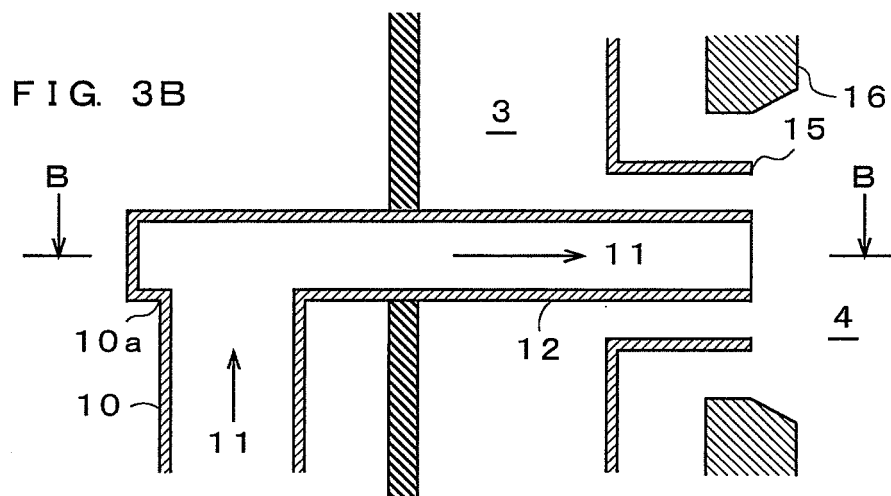
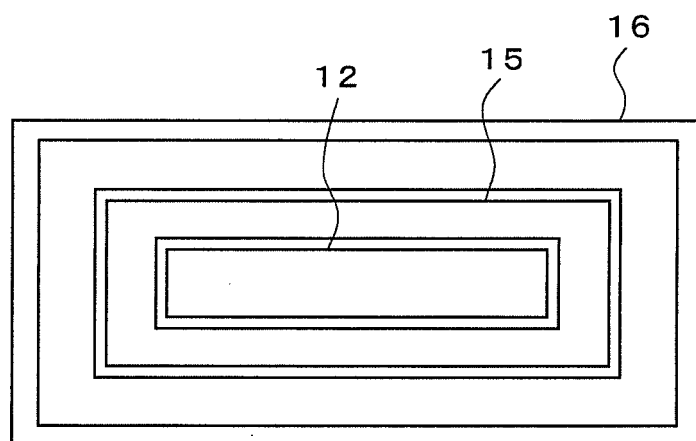


FIG. 3C



Description

FIELD OF THE ART

[0001] The present invention relates to a burner and a combustion equipment and a boiler including the burner, and particularly relates to a burner capable of performing low nitrogen oxide (NOx) combustion at high efficiency.

BACKGROUND ART

[0002] FIG. 28 shows an example of a solid fuel (pulverized coal, biomass fuel, etc.) burner according to a conventional art. FIG. 28A is a side sectional view of the burner, and FIG. 28B is a front view of the burner as viewed from a furnace (4) side. The solid fuel burner includes a fuel-containing fluid supply nozzle (12), defining a fuel-containing fluid flow passage through which a fuel-containing fluid (11), containing a solid fuel and a conveying primary air, flows toward the furnace (4), and a combustion air sleeve (15) disposed at an outer periphery of the fuel-containing fluid supply nozzle (12), and air inside a windbox (3) is supplied as a secondary air (13) and a tertiary air (14) through a combustion air flow passage defined by the sleeve (15). A flame stabilizer (17) is disposed at a front end of the fuel-containing fluid supply nozzle (12), and ignition of the fuel from a vicinity of the burner is enabled by an effect of a circular vortex formed at a wake of the flame stabilizer (17).

[0003] A front end of the combustion air sleeve (15) is disposed at a position facing a burner throat (16), a combustion air guide plate (15a), spreading outside the burner, is disposed at the front end of the sleeve (15), the tertiary air (14) is spread outward by the combustion air guide plate (15a) to delay mixing of air into a central part of a flame, and by promotion of combustion under a reducing atmosphere condition of insufficient air, generation of nitrogen oxides (NOx) in a combustion gas is suppressed.

[0004] FIG. 1C is a sectional view taken in a direction along an ejection flow of the fuel-containing fluid (11) at the fuel-containing fluid supply nozzle (12) in the burner of the conventional art, and as shown in FIG. 1D, which is a front view of an outlet part of the fuel-containing fluid supply nozzle (12) of the burner in FIG. 1C as viewed from the furnace (4) side, with the burner of the conventional art, a cross section of the outlet part of the fuel-containing fluid supply nozzle (12) has a shape close to a circular shape. When the fuel-containing fluid (11) is loaded into the furnace (4), the fuel is ignited near the outlet of the fuel-containing fluid supply nozzle (12) by heating due to radiation inside the furnace (4) and actions of the circular vortex at the wake of the flame stabilizer (17).

[0005] FIG. 29A is a sectional view taken in a direction along the ejection flow of the fuel-containing fluid (11) at the fuel-containing fluid supply nozzle (12) in the burner of the conventional art, and an ignition position (33) of

the fuel in the fuel-containing fluid ejected from the fuel-containing fluid supply nozzle (12) into the furnace (4) is formed as shown in FIG. 29B, which is a front view of the outlet part of the fuel-containing fluid supply nozzle (12) as viewed from the furnace (4) side. After the fuel is ignited at a surface of the ejection flow of the fuel containing fluid (11), a flame that is formed gradually propagates toward a central part of the ejection flow of the fuel-containing fluid (11). FIG. 30 schematically shows a propagation behavior of the flame inside the furnace (4) in a cross-sectional direction along the ejection flow of the fuel-containing fluid (11) at the fuel-containing fluid supply nozzle (12) in the burner of the conventional art. An ignited region (32) is formed around an unignited region (31) of conical shape.

[0006] The burner of circular cross section, shown in FIGS. 28 to 30, is frequently used in a so-called opposed firing configuration in which the burners is disposed at each of a pair of opposing furnace walls. Meanwhile, in so-called tangential firing in which the fuel is combusted while the fuel-containing fluid is ejected into the furnace (4) in directions of applying a rotation along a furnace wall surface from outlets of a plurality of fuel-containing fluid supply nozzles (12), an outlet shape of a transverse section (section orthogonal to the flow of the fuel-containing fluid) of each fuel-containing fluid supply nozzle (12) is made a square shape or a rectangular shape close to a square shape in many cases.

[0007] Burners, with which the outlet shape of the transverse section (section orthogonal to the flow of the fuel-containing fluid) of the fuel-containing fluid supply nozzle (12) is made a rectangular shape, an elliptical shape, or a substantially elliptical shape with major and minor axis parts, are disclosed in the following Patent Documents 1 to 3.

Patent Document 1: Japanese Translation of International Application (Kohyo) No. Sho 59-500981

Patent Document 2: Japanese Published Patent Application No. Hei 8-226615

Patent Document 3: Japanese Published Patent Application No. Hei 11-281009

DISCLOSURE OF THE INVENTION

OBJECT(S) OF THE INVENTION

[0008] In general, a cross section of an outlet part of a fuel-containing fluid supply nozzle (12) of a burner has a shape close to a circular shape or a square shape, and there are cases where, as shown in FIG. 30, a flame ignited at an outer side of a fuel-containing fluid ejection flow in a furnace (4) must propagate a considerable distance to reach a central part of the fuel-containing fluid ejection flow. A distance in a fuel-containing fluid (11) ejection flow direction from the fuel-containing fluid supply nozzle (12) that is required for the ignited flame to propagate to the central part of the fuel ejection flow, in

other words, an unignited distance L1' shown in FIG. 30 is longer and an unignited region (31) is more expanded the larger a diameter or a peripheral part of the fuel-containing fluid supply nozzle (12). Although promotion of combustion in a reducing region in a vicinity of the burner is important for suppressing NOx generation in a combustion gas, expansion of the unignited region (31) inhibits the NOx concentration suppression characteristic. Expansion of the unignited region (31) also means that a combustion time after ignition is short and causes lowering of combustion efficiency.

[0009] Although increasing a burner capacity (decreasing a number of burners) is an effective method for reducing cost and improving operability, with the conventional art, when the burner capacity is increased, a diameter or a length of an outer diameter part of the fuel-containing fluid supply nozzle (12) becomes long and the unignited region (31) expands, causing increase of NOx and lowering of the combustion efficiency. This problem was due to the distance from an ignited region (32) at a fuel-containing fluid ejection flow surface to the central part of the fuel-containing fluid ejection flow being large. Also, with the inventions described in Patent Documents 1 to 3 where the outlet shape of the transverse section (section orthogonal to the flow of the fuel-containing fluid) of the fuel-containing fluid supply nozzle (12) is made a rectangular shape, etc. that combines major and minor axis parts, nothing is mentioned in regard to a countermeasure for the expansion of the unignited region (31) due to increase of the burner capacity and the resulting increase of NOx and lowering of the combustion efficiency.

[0010] An object of the present invention is to provide a solid fuel burner that is increased in capacity over the conventional art and yet is suppressed in expansion of an unignited region to prevent increase of NOx concentration in a combustion gas and prevent lowering of combustion efficiency, a combustion equipment and a boiler including the burner.

SUMMARY OF THE INVENTION

[0011] The above object of the present invention is achieved by the following solutions.

[0012] A first aspect of the present invention provides a burner including: a fuel-containing fluid supply nozzle (12) supplying a fuel-containing fluid (11), containing a solid fuel and a medium for transfer of the solid fuel, to an outlet part disposed on a wall surface of a furnace (4) from a connecting part (10a) of a fuel-containing fluid transfer flow passage (10) that transfers the fluid (11); and one or more air supply nozzles (15) supplying combustion air and disposed at an outer peripheral part of the fuel-containing fluid supply nozzle (12); and where, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part disposed on the wall surface of the furnace (4), a cross section of the fuel-containing fluid supply nozzle (12) perpendicular to a flow

of the fluid (11) has a rectangular, elliptical, or substantially elliptical shape with major and minor axis parts and, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, a size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along a direction of the flow of the fluid (11).

[0013] A second aspect of the present invention provides the burner according to the first aspect where the fuel-containing fluid supply nozzle (12) has a configuration such that, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, the size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along the direction of the flow of the fluid (11) and a size of the minor axis part is unchanged.

[0014] A third aspect of the present invention provides the burner according to the first aspect where the fuel-containing fluid supply nozzle (12) has a configuration such that, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, the size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along the direction of the flow of the fluid (11) and a size of the minor axis part decreases gradually along the direction of the flow of the fluid (11).

[0015] A fourth aspect of the present invention provides the burner according to any of the first to third aspects where the fuel-containing fluid supply nozzle (12) has, in an interior thereof, fuel-containing fluid guide plates (19) plurally partitioning the flow of the fuel-containing fluid (11).

[0016] A fifth aspect of the present invention provides the burner according to the fourth aspect where the fuel-containing fluid guide plates (19) are disposed at a plurality of different inclination angles with respect to a plane passing along a line extending a central axis in the direction of the flow of the fluid (11) in the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a shortest axis of the minor axis part of the nozzle (12).

[0017] A sixth aspect of the present invention provides the burner according to any of the first to fifth aspects where the fuel-containing fluid supply nozzle (12) has, in an interior of the outlet thereof, fuel-containing fluid direction changing guide plates (21) forcibly changing a direction of ejection flow of the fuel-containing fluid (11).

[0018] A seventh aspect of the present invention provides the burner according to the sixth aspect where the fuel-containing fluid direction changing guide plates (21) are disposed in a plurality of mutually different directions with respect to a plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a longest axis of the major axis part of the nozzle (12).

[0019] An eighth aspect of the present invention provides the burner according to the sixth aspect where the fuel-containing fluid direction changing guide plates (21) for a portion of the fuel-containing fluid (11) are disposed

parallel to the plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to the longest axis of the major axis part of the nozzle (12), and the fuel-containing fluid direction changing guide plates (21) for another portion of the fuel containing fluid (11) are disposed at an inclination angle with respect to the plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to the longest axis of the major axis part of the nozzle (12).

[0020] A ninth aspect of the present invention provides the burner according to the fourth aspect where the fuel-containing fluid supply nozzle (12) is partitioned into a plurality of flow passages by the fuel-containing fluid guide plates (19), and central axes of the respective flow passages are disposed at the wall surface of the furnace (4) at a plurality of mutually different inclination angles with respect to the plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to the longest axis of the major axis part of the nozzle (12) outlet.

[0021] A tenth aspect of the present invention provides the burner according to any of the first to ninth aspects where fuel-containing fluid partitioning plates (22), capable of plurally partitioning the outlet part of the fuel-containing fluid supply nozzle (12), are disposed at the outlet part.

[0022] An eleventh aspect of the present invention provides the burner according to any of the first to tenth aspects where a flame stabilizer (17) with an L-shaped cross section is disposed at the outlet part of the fuel-containing fluid supply nozzle (12).

[0023] A twelfth aspect of the present invention provides the burner according to the eleventh aspect where a guide plate (17a) outwardly changing an ejection direction of the combustion air in a periphery of the flame stabilizer (17) is disposed at a front end of the L-shaped flame stabilizer (17).

[0024] A thirteenth aspect of the present invention provides the burner according to any of the first to twelfth aspects where a combustion air guide plate (15a), outwardly spreading the ejection direction of the combustion air at an outer side of the one or more combustion air supply nozzles (15) disposed at the outer peripheral part of the nozzle (12) with respect to a fuel ejection direction, is disposed at a front end of the fuel-containing fluid supply nozzle (12).

[0025] A fourteenth aspect of the present invention provides the burner according to any one of the first to thirteenth aspects where a condenser (23), narrowing the flow passage of the fuel-containing fluid (11) once and then expanding the flow passage again, is disposed in an interior of the fuel-containing fluid supply nozzle (12).

[0026] A fifteenth aspect of the present invention provides the burner according to any one of the first to fourteenth aspects where a fluid distribution plate (24) dis-

tributing the fuel uniformly inside the fuel-containing fluid supply nozzle (12) is disposed at an inlet part of the fuel-containing fluid supply nozzle (12).

[0027] A sixteenth aspect of the present invention provides the burner according to any of the first to fifteenth aspects where a nozzle (41, 44), ejecting a liquid fuel or a gas fuel that is an auxiliary fuel to a vicinity of the fluid (11) ejected from the fuel-containing fluid supply nozzle (12), is disposed at the vicinity of the fuel-containing fluid supply nozzle (12).

[0028] A seventeenth aspect of the present invention provides a combustion equipment where the burners according to any of the first to sixteenth aspects are disposed in a plurality of stages in an up/down direction at each of two opposing furnace walls, and a plurality of burners disposed at each stage are disposed respectively symmetrically in wall surface regions divided in two at a central part of width in a horizontal direction of the same furnace wall.

[0029] An eighteenth aspect of the present invention provides a combustion equipment where the burners according to any of the first to sixteenth aspects are disposed in the plurality of stages in the up/down direction at each of the two opposing furnace walls, and burners, which, among the plurality of burners disposed in each stage of the same furnace wall, are adjacent each other in the horizontal direction, are burners of the same structure.

[0030] A nineteenth aspect of the present invention provides a boiler including: a furnace wall formed by spirally winding a set of water wall tubes (25) inclined with respect to a horizontal direction; and where openings (26) of rectangular, elliptical, or substantially elliptical shape are disposed in the furnace wall along a longitudinal direction of the water wall tubes (25) and the burner according to any of the first to sixteenth aspects is mounted in each opening (26).

[0031] A twentieth aspect of the present invention provides a boiler including: a furnace wall formed by a set of water wall tubes (25) extending in a vertical direction; and where openings (26) of rectangular, elliptical, or substantially elliptical shape are disposed in the furnace wall along a longitudinal direction of the water wall tubes (25) and the burner according to any of the first to sixteenth aspects is mounted in each opening (26).

EFFECT(S) OF THE INVENTION

[0032] According to the first aspect of the present invention, expansion of an unignited region can be suppressed even if a burner capacity is increased, an unignited distance can be reduced effectively in comparison with the conventional art because the fuel-containing fluid (11) spreads in the width direction even after the fuel-containing fluid (11) is loaded into the furnace (4) so that a cross-sectional area of the ejection flow of the fuel-containing fluid (11) increases and a flow velocity decreases, and also because the fuel-containing fluid (11)

spreads inside the furnace (4), a combustion space can be utilized effectively and a practical furnace retention time is made long, thereby providing effects of reducing NOx concentration in a combustion gas and improving combustion efficiency.

[0033] According to the second aspect of the present invention, the size of the minor axis part is unchanged and this is effective for simplification of structure. Also, the flow velocity at an upstream side of the fuel-containing fluid supply nozzle (12) can be made high and this is effective for preventing backfiring in a case of a readily ignitable fuel, etc.

[0034] According to the third aspect of the present invention, the increase of the flow velocity of the fuel-containing fluid (11) from the fuel-containing fluid connecting part (10a) toward the outlet part of the fuel-containing fluid supply nozzle (12) can be suppressed to minimize pressure loss and suppress wear of component parts inside the fuel-containing fluid supply nozzle (12).

[0035] According to the fourth and fifth aspects of the present invention, because the flow of the fuel-containing fluid (11) is partitioned plurally by the fuel-containing fluid guide plates (19) in the interior of the fuel-containing fluid supply nozzle (12), the fuel-containing fluid (11) is supplied uniformly in the direction in which the fuel-containing fluid supply nozzle (12) expands from the fuel-containing fluid connecting part (10a) toward the outlet part of the nozzle (12), and the effects of NOx reduction, improvement in combustion efficiency, suppression of flow velocity increase, minimization of pressure loss, and suppression of wear of component parts are improved in comparison with other those of the third aspect of the present invention.

[0036] According to the sixth aspect of the present invention, effects of promoting dispersion of the fuel-containing fluid ejection flow (20) inside the furnace (4) and promoting combustion at a wake part of the furnace (4) are provided.

[0037] According to the seventh aspect of the present invention, because the fuel-containing fluid guide plates (19) are respectively disposed in mutually opposing directions with respect to the plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to the longest axis of the major axis part of the nozzle (12), the fuel-containing fluid (11) can be ejected into the furnace (4) in two or more groups and the fuel-containing fluid ejection flow (20) can thereby be divided into groups by a simple structure to provide the effects of promoting the dispersion of the fuel-containing fluid ejection flow (20) inside the furnace (4) and promoting the combustion at the wake part of the furnace (4).

[0038] According to the eighth aspect of the present invention, by dividing four fuel-containing fluid ejection flows (20), formed by the fuel-containing fluid supply nozzle (12) and the fuel-containing fluid guide plates (19) into two groups (20a, 20b) and thereby making, for example, fuel-containing fluid ejection flows (20a), adjacent

a furnace side wall, rectilinear flows and making fuel-containing fluid ejection flows (20b), not adjacent the furnace side wall, be ejected upon applying an inclination with respect to a horizontal direction, an effect of preventing ash deposition by suppressing flame inflow to a vicinity of the furnace side wall while maintaining promotion of combustion at the furnace wake part by dispersion of the fuel is provided.

[0039] According to the ninth aspect of the present invention, because the fuel-containing fluid (11) can be ejected into the furnace (4) at mutually different angles with respect to the horizontal direction or the vertical direction from the fuel-containing fluid supply nozzle (12) and the fuel-containing fluid ejection flow (20) can be varied in direction without using parts inside the fuel-containing fluid supply nozzle (12) with which pulverized coal or other solid fuel collides directly, wear of parts can be suppressed effectively.

[0040] According to the tenth aspect of the present invention, the fuel-containing fluid ejection flow (20) is partitioned by the fuel-containing fluid partitioning plates (22) to be increased in surface area, and radiant heat inside the furnace (4) is thereby increased and a negative pressure region is formed at the wake side of the fluid partitioning plates (22), thereby making a high temperature gas in a periphery flow into the negative pressure region to contribute to early ignition of the fuel, promote combustion at a reducing region in the vicinity of the burner, and effectively contribute to the reduction of the NOx concentration in the combustion gas and the improvement in the combustion efficiency.

[0041] According to the eleventh aspect of the present invention, by disposing the flame stabilizer (17) with the L-shaped cross section at the outlet part of the fuel-containing fluid supply nozzle (12), a circular vortex is formed at the wake of the flame stabilizer (17) and draws back the high-temperature combustion gas to a vicinity of the flame stabilizer (17) to contribute to early ignition of the fuel, promote combustion at the reducing region in the vicinity of the burner, and effectively contribute to the reduction of the NOx concentration in the combustion gas and the improvement in the combustion efficiency in comparison with the eleventh aspect of the invention.

[0042] According to the twelfth aspect of the present invention, by the secondary air guide plate (17a) at the front end of the flame stabilizer (17) of L-shaped cross section, the secondary air is spread outward and the circular vortex at the wake of the flame stabilizer (17) is enlarged, thereby increasing a recirculation amount of the high-temperature combustion gas to further quicken ignition of the fuel, promote combustion at the reducing region near the burner, and effectively contribute to the reduction of the NOx concentration in the combustion gas and the improvement in the combustion efficiency in comparison with the eleventh aspect of the invention.

[0043] According to the thirteenth aspect of the present invention, by provision of the combustion air guide plate (15a) that spreads the combustion air ejection direction

at the outer side of the combustion air supply nozzles (15) outward with respect to the fuel-containing fluid ejection direction, the combustion air is spread outward, thereby enlarging the reducing region at a central part of the flame and effectively contributing to the reduction of the NO_x concentration in the combustion gas and the improvement in the combustion efficiency.

[0044] According to the fourteenth aspect of the present invention, the fuel in the vicinity of the flame stabilizer (17) is condensed by the condenser (23) thereby contributing to early ignition of the fuel to promote combustion at a reducing region in the vicinity of the burner and effectively contributing to the reduction of the NO_x concentration in the combustion gas and the improvement in the combustion efficiency.

[0045] According to the fifteenth aspect of the present invention, the fuel concentration at the inlet part of the fuel-containing fluid supply nozzle (12) is made uniform by the fluid distribution plate (24) to suppress imbalance of concentration of the fuel flowing into the respective flow passages partitioned by the fuel-containing fluid guide plates (19), and this is effective for NO_x reduction and improvement in the combustion efficiency.

[0046] According to the sixteenth aspect of the present invention, because the liquid fuel or the gas fuel is ejected to the burner outlet, the fuel-containing fluid (11) that contains the solid fuel can be ignited reliably.

[0047] According to the seventeenth aspect of the present invention, by the burners according to any of the first to sixteenth aspects being disposed in the plurality of stages in the up/down direction at each of the opposing furnace walls of the opposed firing type furnace (4) and by disposing the plurality of burners of each stage respectively symmetrically at the wall surface regions divided in two at the central part of width in the horizontal direction of the same furnace wall, the directions of the fluid ejection flows (20a, 20b) can be made left/right symmetrical at a single furnace wall surface and good left/right balance of flow and combustion states can be maintained in the furnace (4).

[0048] According to the eighteenth aspect of the present invention, by the burners according to any of the first to sixteenth aspects being disposed in the plurality of stages in the up/down direction at each of the two opposing furnace walls of the opposed firing type furnace (4) and by making the burners, which, among the plurality of burners disposed in each stage of the same furnace wall, are adjacent each other in the horizontal direction, burners of the same structure, collision of the fuel-containing fluid ejection flows (20a, 20b) can be avoided, especially in a furnace (4) of small capacity, to suppress localized concentration of fuel and provide the effects of reducing the NO_x concentration in the combustion gas and improving the combustion efficiency.

[0049] According to the nineteenth aspect of the present invention, by aligning a longitudinal direction of the water wall tubes (25) with a longitudinal direction of the major axis parts of the openings (26), a number of

the spiral water wall tubes (25) necessary for forming the openings (26) can be made small and an economical boiler can be constructed with few processed and bent parts in the water wall tubes (25). The number of the spiral water wall tubes (25) necessary for forming the openings (26) can be minimized to improve economy. Also, because the fuel-containing fluid (11) spreads in the horizontal (width) direction of the furnace (4), distribution of the fuel-containing fluid (11) in the horizontal (width) direction of the furnace (4) is made uniform, the practical furnace retention time is made longer, and the effects of reducing the NO_x concentration in the combustion gas and improving the combustion efficiency are provided.

[0050] According to the twentieth aspect of the present invention, because the rectangular openings (26) are installed on the furnace wall along the arrangement of the water wall tubes (25) in the vertical direction, by aligning the longitudinal direction of the water wall tubes (25) with the longitudinal direction of the major axis parts of the openings (26), an economical boiler can be constructed with few processed and bent parts in the water wall tubes (25).

BRIEF DESCRIPTION OF THE DRAWINGS

[0051]

FIG. 1 shows explanatory diagrams of ignited regions of burner outlets according to the present invention and a conventional art.

FIG. 2 shows explanatory diagrams of unignited regions of the burner outlets according to the present invention and the conventional art.

FIG. 3 shows a structural example of a burner according to an embodiment of the present invention (FIG. 3A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 3B is a sectional view taken on line A-A of FIG. 3A, and FIG. 3C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 4 shows a structural example of a burner according to an embodiment of the present invention (FIG. 4A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 4B is a sectional view taken on line A-A of FIG. 4A, and FIG. 4C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 5 shows a structural example of a burner according to an embodiment of the present invention (FIG. 5A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 5B is a sectional view taken on line A-A of FIG. 5A, and FIG. 5C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 6 shows a structural example of a burner according to an embodiment of the present invention (FIG. 6A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 6B is a sectional view taken on line A-A of FIG. 6A, and FIG. 6C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 7 shows a structural example of a burner according to an embodiment of the present invention (FIG. 7A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 7B is a sectional view taken on line B-B of FIG. 7A, and FIG. 7C is a sectional view taken on line A-A of FIG. 7A).

FIG. 8 shows a structural example of a burner according to an embodiment of the present invention (FIG. 8A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 8B is a sectional view taken on line B-B of FIG. 8A, and FIG. 8C is a sectional view taken on line A-A of FIG. 8A).

FIG. 9 shows a structural example of a burner according to an embodiment of the present invention (FIG. 9A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 9B is perspective view of the burner, FIG. 9C is a sectional view taken on line B-B of FIG. 9A, and FIG. 9D is a sectional view taken on line A-A of FIG. 9A).

FIG. 10 shows a structural example of a burner according to an embodiment of the present invention (FIG. 10A is a sectional view taken along a line, passing through a central axis of the burner and extending toward the furnace, in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 10B is a sectional view taken on line A-A of FIG. 10A, and FIG. 10C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 11 is an explanatory diagram of effects of the invention shown in FIG. 10.

FIG. 12 shows a structural example of a burner according to an embodiment of the present invention shown in FIG. 10 (FIG. 12A is a sectional view taken along a line, passing through a central axis of the burner and extending toward the furnace, in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 12B is a sectional view taken on line A-A of FIG. 12A, and FIG. 12C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 13 shows a structural example of a burner according to an embodiment of the present invention (FIG. 13A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 13B

is a sectional view taken on line A-A of FIG. 13A, and FIG. 13C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 14 shows a structural example of a burner according to an embodiment of the present invention (FIG. 14A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 14B is a sectional view taken on line A-A of FIG. 14A, and FIG. 14C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 15 shows a structural example of a burner according to an embodiment of the present invention (FIG. 15A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 15B is a sectional view taken on line A-A of FIG. 15A, and FIG. 15C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 16 shows a structural example of a burner according to an embodiment of the present invention (FIG. 16A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 16B is a sectional view taken on line A-A of FIG. 16A, and FIG. 16C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 17 shows a structural example of a burner according to an embodiment of the present invention (FIG. 17A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 17B is a sectional view taken on line A-A of FIG. 17A, and FIG. 17C is a front view of the outlet part of the burner as viewed from the furnace (4) side).

FIG. 18 shows a structural example of a burner according to an embodiment of the present invention (FIG. 18A is a sectional view parallel to a fuel-containing fluid supply nozzle (12) surface formed through a long edge of an outlet part of the nozzle (12) (sectional view taken on line B-B of FIG. 18B), FIG. 18B is a sectional view taken on line A-A of FIG. 18A, and FIG. 18C is a front view of the outlet part of the burner as viewed from the furnace side).

FIG. 19 shows an example where an oil supply nozzle is installed at a central part of a fuel-containing fluid supply nozzle.

FIG. 20 shows an example where gas supply nozzles are installed in a periphery of a flame stabilizer.

FIG. 21 shows a front view (FIG. 21A) and a plan view (FIG. 21B) of a fuel-containing fluid supply nozzle.

FIG. 22 shows a front view (FIG. 22A) and a plan view (FIG. 22B) of a fuel-containing fluid supply nozzle of another configuration.

FIG. 23 shows an example where a plurality of the fuel-containing fluid supply nozzles, shown in FIG. 21, are positioned in three stages in an up/down di-

rection and four columns in a horizontal direction on a single furnace wall surface.

FIG. 24 shows an example where a plurality of the fuel-containing fluid supply nozzles, shown in FIGS. 21 and 22, are positioned in three stages in an up/down direction and four columns in a horizontal direction on a single furnace wall surface.

FIG. 25 shows another embodiment where a plurality of the fuel-containing fluid supply nozzles, shown in FIG. 21, are positioned in three stages in an up/down direction and four columns in a horizontal direction on a single furnace wall surface.

FIG. 26 is a plan view of a furnace wall of a boiler in which burners according to an embodiment of the present invention are disposed.

FIG. 27 is a plan view of a furnace wall of a boiler in which burners according to an embodiment of the present invention are disposed.

FIG. 28 shows an example of a solid fuel burner according to a conventional art (FIG. 28A is a side sectional view of the burner, and FIG. 28B is a front view of the burner as viewed from a furnace side).

FIG. 29A is a sectional view taken in a direction along an ejection flow of a fuel-containing fluid at a fuel-containing fluid supply nozzle in the burner of the conventional art, and FIG. 29B is a front view of an outlet part of a fuel-containing fluid supply nozzle as viewed from the furnace side.

FIG. 30 is schematic diagram of a flame propagation behavior inside the furnace in a cross-sectional direction along the ejection flow of the fuel-containing fluid at the fuel-containing fluid supply nozzle in the burner of the conventional art.

BEST MODE(S) FOR CARRYING OUT THE INVENTION

[0052] Embodiments of the present invention shall now be described along with the drawings.

[0053] Basic concepts of the present invention shall now be described using FIGS. 1 and 2. In FIG. 1, an ignition position (33) in a furnace (4) using a fuel-containing fluid supply nozzle (12) of a burner according to an embodiment shown in FIGS. 1A and 1B is compared with that in a conventional art shown in FIGS. 1C and 1D. FIG. 1A is a sectional view of the fuel-containing fluid supply nozzle (12) of the burner according to the present embodiment taken in an ejection flow direction of a fuel-containing fluid (11), FIG. 1B is a front view of an outlet part of the fuel-containing fluid supply nozzle (12) of FIG. 1A as viewed from the furnace (4) side, FIG. 1C is a sectional view of the fuel-containing fluid supply nozzle (12) of the burner according to the conventional art taken in the ejection flow direction of the fuel-containing fluid, and FIG. 1D is a front view of the outlet part of the fuel-containing fluid supply nozzle (12) as viewed from the furnace (4) side.

[0054] As shown in FIG. 1D, in the conventional art,

the ring-shaped ignition position (33) is present in a periphery of the ejection flow of circular cross section of the fuel-containing fluid (11) that is ejected from the fuel-containing fluid supply nozzle (12) of the burner. Meanwhile, in the present embodiment, the ignition position (33) is present in a periphery of the ejection flow of rectangular cross section of the fuel-containing fluid (11) that is ejected from the fuel-containing fluid supply nozzle (12) of the burner as shown in FIG. 1B.

[0055] In the present embodiment, by making the shape of the outlet part of the fuel-containing fluid supply nozzle (12) of the burner a rectangular shape and reducing a length of a short edge, a length L2 (FIG. 1B) from the ignition position (33) to a central part of the ejection flow of the fuel-containing fluid (11) inside the furnace (4) in a direction perpendicular to the ejection flow of rectangular cross section of the fuel-containing fluid (11) is reduced significantly in comparison with the length L2' (FIG. 1D) from the ignition position (33) to the central part of the ejection flow of the fuel-containing fluid (11) in the direction perpendicular to the ejection flow of circular cross section.

[0056] FIG. 2 shows sectional views of the fuel-containing fluid supply nozzle (12) of the burner showing that a distance (unignited distance) L1 (FIG. 2A) that a flame propagates from the ignition position (33) to the central part of the fuel ejection flow in an ejection flow direction of the fuel-containing fluid (11) from the fuel-containing fluid supply nozzle (12) in the embodiment is reduced in comparison with the unignited distance L1' (FIG. 2B) in the conventional art. In the present embodiment, in accordance with the reduction of the distance L2 from the ignition position (33) to the central part of the ejection flow of the fuel-containing fluid (11) in the direction perpendicular to the ejection flow of rectangular cross section in comparison with the distance L2' of the conventional art, the unignited distance L1 is reduced significantly in comparison with the unignited distance L1' of the conventional art.

[0057] FIG. 3 shows a structural example of a burner according to an embodiment of the present invention. FIG. 3A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 3B), FIG. 3B is a sectional view taken on line A-A of FIG. 3A, and FIG. 3C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0058] A cylindrical fuel-containing fluid flow passage (10) is connected via a connecting part (10a) of circular cross section to the fuel-containing fluid supply nozzle (12) having a rectangular cross section and has an adequate configuration for forming the ejection flow of rectangular cross section from the fuel-containing fluid supply nozzle (12) into the furnace (4). Even after the fuel-containing fluid (11) is loaded into the furnace (4), the fuel-containing fluid (11) spreads along the ejection flow directions and a cross-sectional area of the ejection flow

of the fuel-containing fluid (11) expands while a flow velocity decreases, thereby effectively reducing the unignited distance L1 shown in FIG. 2 further. Also, because the fuel-containing fluid (11) spreads inside the furnace (4), a combustion space can be utilized effectively and a practical furnace retention time is made long, thereby contributing effectively to reduction of NOx concentration in a combustion gas and improvement in combustion efficiency. A combustion air sleeve (15) of rectangular cross section and a burner throat (16) of rectangular cross section are disposed in a periphery of the fuel-containing fluid supply nozzle (12).

[0059] FIG. 4 shows a structural example of a burner according to an embodiment of the present invention. FIG. 4A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 4B), FIG. 4B is a sectional view taken on line A-A of FIG. 4A, and FIG. 4C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0060] In the burner shown in FIG. 4, the cross section of the burner in the direction perpendicular to the ejection flow of the fuel-containing fluid (11) has an elliptical shape, and configurations of other parts are the same as those of the burner shown in FIG. 3.

[0061] Although as cross-sectional shapes in a vertical direction of the burner (direction perpendicular to the ejection flow of the fuel-containing fluid (11)), the representative shapes of rectangular and elliptical were shown in FIGS. 3 and 4, respectively, the same effects as those mentioned above can be obtained even when a similar shape, such as a shape with which short sides of a rectangle have an arcuate shape, an expanded rhombus, etc., is employed.

[0062] FIG. 5 shows a structural example of a burner according to an embodiment of the present invention. FIG. 5A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 5B), FIG. 5B is a sectional view taken on line A-A of FIG. 5A, and FIG. 5C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0063] In the burner shown in FIG. 5, whereas a size of the major axis part gradually increases along the flow direction of the fuel-containing fluid (11) from the fuel-containing fluid connecting part (10a) of the fuel-containing fluid supply nozzle (12) toward the outlet part, a size of a minor axis part gradually decreases along the direction of flow of the fuel-containing fluid (11), and configurations of other parts are the same as those of the burner shown in FIG. 3.

[0064] A characteristic of the burner structure shown in FIG. 5 is that increase of the flow velocity of the fuel-containing fluid (11) from the fuel-containing fluid connecting part (10a) toward the outlet part of the fuel-containing fluid supply nozzle (12) can be suppressed to min-

imize pressure loss and suppress wear of component parts inside the fuel-containing fluid supply nozzle (12).

[0065] FIG. 6 shows a structural example of a burner according to an embodiment of the present invention. FIG. 6A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 6B), FIG. 6B is a sectional view taken on line A-A of FIG. 6A, and FIG. 6C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0066] In the burner shown in FIG. 6, fuel-containing fluid guide plates (19) are disposed so that the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) is uniformly supplied in directions in which the fuel-containing fluid supply nozzle (12) expands along the ejection flow direction, and configurations of other parts are the same as those of the burner shown in FIG. 5. In the present example, three fuel-containing fluid guide plates (19) are installed, and to make the fuel-containing fluid (11) spread uniformly in accordance with the spreading of the fuel-containing fluid supply nozzle (12), a central guide plate (19) is disposed along the central axis and guide plates (19) at both sides that sandwich the central guide plate (19) are disposed at angles α and β with respect to a vertical section passing through the central axis.

[0067] Because the flow of the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) is partitioned plurally by the fuel-containing fluid guide plates (19), the fuel-containing fluid (11) is spread uniformly according to the spreading of the fuel-containing fluid supply nozzle (12) from the fuel-containing fluid connecting part (10a) toward the outlet part of the fuel-containing fluid supply nozzle (12) and can be combusted without imbalance. Also, by the fuel-containing fluid (11) being spread uniformly, the effects of suppression of localized increase of flow velocity, minimization of pressure loss, and suppression of wear of component parts are improved over those of the configuration shown in FIG. 5.

[0068] FIG. 7 shows a structural example of a burner according to an embodiment of the present invention. FIG. 7A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 7B is a sectional view taken on line B-B of FIG. 7A, and FIG. 7C is a sectional view taken on line A-A of FIG. 7A.

[0069] In the burner shown in FIG. 7, the fuel-containing fluid guide plates (19) are disposed in the same manner as in the burner shown in FIG. 6 so that the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) is uniformly supplied in directions in which the fuel-containing fluid supply nozzle (12) expands along the flow direction, and in the section taken on line A-A of FIG. 7A, fuel-containing fluid direction changing guide plates (21a) that change the flow of the fluid (11) downward with respect to a plane along a line, passing through the central axis of the fuel-containing

fluid supply nozzle (12) and extending toward the furnace (4), and parallel to the longest axis of the major axis part of the outlet part are installed at the burner outlet part, and in the section taken on line B-B of FIG. 7A, fuel-containing fluid direction changing guide plates (21b) that change the flow of the fluid (11) upward with respect to the abovementioned plane are installed at the burner outlet part. Four fuel-containing fluid ejection flows 20 (20a, 20b) formed by the fuel-containing fluid supply nozzle (12) and the fuel-containing fluid guide plates (19) are formed to downwardly inclining fuel-containing fluid ejection flows (20a) and upwardly inclining fuel-containing fluid ejection flows (20b) by the above-mentioned fuel-containing fluid direction changing guide plates (21a, 21b). By the burner configuration shown in FIG. 7, dispersion of the fuel-containing fluid ejection flow (20) inside the furnace (4) is promoted to provide an effect of promoting combustion at a wake part of the furnace (4).

[0070] FIG. 8 shows a structural example of a burner according to an embodiment of the present invention. FIG. 8A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 8B is a sectional view taken on line B-B of FIG. 8A, and FIG. 8C is a sectional view taken on line A-A of FIG. 8A.

[0071] In the burner shown in FIG. 8, the fuel-containing fluid guide plates (19) are disposed in the same manner as in the burner shown in FIG. 7 so that the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) is uniformly supplied in directions in which the fuel-containing fluid supply nozzle (12) expands along the flow direction, and in the section taken on line A-A of FIG. 8A, fuel-containing fluid direction changing guide plates (21a) that rectify and make the flow of the fluid (11) rectilinear with respect to a plane along a line, passing through the central axis of the fuel-containing fluid supply nozzle (12) and extending toward the furnace (4), and parallel to the longest axis of the major axis part of the outlet part are installed at the burner outlet part, and in the section taken on line B-B of FIG. 8A, fuel-containing fluid direction changing guide plates (21b) that change the flow of the fluid (11) upward with respect to the abovementioned plane are installed at the burner outlet part. Four fuel-containing fluid ejection flows 20 (20a, 20b) formed by the fuel-containing fluid supply nozzle (12) and the fuel-containing fluid guide plates (19), are formed to fuel-containing fluid ejection flows (20a) in a rectilinear direction and fuel-containing fluid ejection flows (20b) that are upwardly directed ejection flows by installation of the above-mentioned fuel-containing fluid direction changing guide plates (21a, 21b).

[0072] In a case where the fuel-containing fluid direction changing guide plates (21a) are not installed and only the guide plates (21b) that change the direction are installed, the same fuel-containing fluid ejection flows (20a, 20b) are formed.

[0073] For example, by making fuel-containing fluid ejection flows, close to a water wall side at a side wall

side of the furnace (4), rectilinear flows and making fuel-containing fluid that are not close to the water wall side at the side wall side of the furnace (4) flows that are oblique with respect to a central side of the furnace by the burner configuration shown in FIG. 8, the dispersion of the fuel-containing fluid ejection flow (20) inside the furnace (4) is promoted to maintain the effect of promoting the combustion at the wake part of the furnace (4) and provide an effect of suppressing inflow of a flame to a vicinity of the side wall of the furnace (4) to prevent ash deposition.

[0074] FIG. 9 shows a structural example of a burner according to an embodiment of the present invention. FIG. 9A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part, FIG. 9B is perspective view of the burner, FIG. 9C is a sectional view taken on line B-B of FIG. 9A, and FIG. 9D is a sectional view taken on line A-A of FIG. 9A.

[0075] In the burner shown in FIG. 9, the fuel-containing fluid guide plates (19) are disposed in the same manner as in the burner shown in FIG. 7 so that the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) is uniformly supplied in directions in which the fuel-containing fluid supply nozzle (12) expands along the flow direction. A front side of an inlet part of the fuel-containing fluid supply nozzle (12) has a parallelepiped shape, one side surface (12a) of the fuel-containing fluid supply nozzle (12) leads to the outlet of the fuel-containing fluid supply nozzle (12) while being disposed obliquely downward along the flow direction and the other side surface (12b) leads to the outlet while being disposed obliquely upward along the flow direction.

[0076] By this configuration, an obliquely downwardly directed fuel-containing fluid ejection flow (20a) is formed at a portion close to the side surface (12a) of the fuel-containing fluid supply nozzle (12) as shown in FIG. 9D, and an obliquely upwardly directed fuel-containing fluid ejection flow (20d) is formed at a portion close to the side surface (12b) of the fuel-containing fluid supply nozzle (12) as shown in FIG. 9C. From two central flow passages into which the fuel-containing fluid supply nozzle (12) is partitioned by the fuel-containing fluid guide plates (19), a fuel-containing fluid ejection flow (20b) having an ejection flow direction intermediate the fuel-containing fluid ejection flow (20a) and the central line and a fuel-containing fluid ejection flow (20c) having an ejection flow direction intermediate the fuel-containing fluid ejection flow (20d) and the central line are formed.

[0077] Although the effects of the burner configuration of FIG. 9 are equivalent to those of the burner shown in FIG. 7, because the fuel-containing fluid direction changing guide plates (21) are not used to change the ejection direction of the fuel-containing fluid (11), a problem of wear that is of concern with the guide plates (21) does not occur.

[0078] FIG. 10 shows a structural example of a burner according to an embodiment of the present invention.

FIG. 10A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 10B), FIG. 10B is a sectional view taken on line A-A of FIG. 10A, and FIG. 10C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0079] Fuel-containing fluid partitioning plates 22, perpendicular to the flow of the fuel-containing fluid (11) and partially blocking the flow, are disposed at the outlet part of the fuel-containing fluid supply nozzle (12). The fuel-containing fluid ejection flow (20) is partitioned into four by the fuel-containing fluid partitioning plates (22) as shown in FIG. 11. By the partitioning, the fuel-containing fluid ejection flow (20) increases in surface area, the radiant heating inside the furnace (4) increases, negative pressure regions (22a) are formed at the wake side of the fuel-containing fluid partitioning plates (22), and high-temperature gas in the periphery flows into the negative pressure regions (22a) as indicated by arrows in the figure. Increase of the radiant heating and inflow of the high-temperature gas into the negative pressure regions (22a) both contribute to early ignition of the fuel, and combustion in a reducing region in a vicinity of the burner is promoted to contribute effectively to reduction of the NOx concentration of the combustion gas and improvement in the combustion efficiency.

[0080] FIG. 12 shows a structural example of a burner according to an embodiment of the present invention. FIG. 12A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 12B), FIG. 12B is a sectional view taken on line A-A of FIG. 12A, and FIG. 12C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0081] The fuel-containing fluid partitioning plates (22), perpendicular to the flow of the fuel-containing fluid (11) and partially blocking the flow, are disposed at the outlet parts of the fuel-containing fluid guide plates (19) at the outlet part of the fuel-containing fluid supply nozzle (12). Because the fuel-containing fluid (11) is supplied uniformly inside the fuel-containing fluid supply nozzle (12) by the fuel-containing fluid guide plates (19), the reduction of NOx and improvement in the combustion efficiency are realized more effectively.

[0082] FIG. 13 shows a structural example of a burner according to an embodiment of the present invention. FIG. 13A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 13B), FIG. 13B is a sectional view taken on line A-A of FIG. 13A, and FIG. 13C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0083] A flame stabilizer (17) with an L-shaped cross section is installed at the outlet part of the fuel-containing fluid supply nozzle (12). Because a circular vortex (not

shown) is formed at a wake of the flame stabilizer (17) and draws back the high-temperature combustion gas to the vicinity of the flame stabilizer (17), the configuration contributes to early ignition of the fuel and promotes combustion in the reducing region in the vicinity of the burner to effectively contribute to reduction of the NOx concentration in the combustion gas and improvement in the combustion efficiency.

[0084] FIG. 14 shows a structural example of a burner according to an embodiment of the present invention. FIG. 14A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 14B), FIG. 14B is a sectional view taken on line A-A of FIG. 14A, and FIG. 14C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0085] A secondary air guide plate (17a) that outwardly spreads ejection directions of a secondary air is installed at a front end of the flame stabilizer (17) of L-shaped cross section shown in FIG. 14. By the secondary air being spread outward by the guide plate (17a), the vortex flow (not shown) at the wake of the flame stabilizer (17) is enlarged, a recirculation amount of the high-temperature gas is increased, ignition of the fuel is quickened further, and combustion in the reducing region in the vicinity of the burner is promoted to effectively contribute to reduction of the NOx concentration in the combustion gas and improvement in the combustion efficiency.

[0086] FIG. 15 shows a structural example of a burner according to an embodiment of the present invention. FIG. 15A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 15B), FIG. 15B is a sectional view taken on line A-A of FIG. 15A, and FIG. 15C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0087] In the burner shown in FIG. 15, a tertiary air guide plate (15a) that outwardly spreads ejection directions of a tertiary air is installed at a front end of a secondary air sleeve (15). By the tertiary air being spread outward, a reducing region at a center part of the flame is enlarged to effectively contribute to reduction of the NOx concentration in the combustion gas and improvement in the combustion efficiency.

[0088] FIG. 16 shows a structural example of a burner according to an embodiment of the present invention. FIG. 16A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 16B), FIG. 16B is a sectional view taken on line A-A of FIG. 16A, and FIG. 16C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0089] In the burner shown in FIG. 16, a fuel-containing fluid condenser (23), combining a triangular prism gradually increasing in cross-sectional area from an upstream

side and an oppositely directed triangular prism gradually decreasing in cross-sectional area at a downstream side, is installed inside the fuel-containing fluid supply nozzle (12). The fuel in the vicinity of the flame stabilizer (17) is condensed by the fuel-containing fluid condenser (23) and this contributes to early ignition of the fuel, promotes combustion in the reducing region near the burner, and thereby effectively contributes to reduction of the NOx concentration in the combustion gas and improvement in the combustion efficiency.

[0090] FIG. 17 shows a structural example of a burner according to an embodiment of the present invention. FIG. 17A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 17B), FIG. 17B is a sectional view taken on line A-A of FIG. 17A, and FIG. 17C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0091] A fuel-containing fluid condenser (23') combining a triangular prism gradually increasing in cross-sectional area at an upstream side, a quadrangular prism at an intermediate part, and an oppositely directed triangular prism gradually decreasing in cross-sectional area at a downstream side, is installed inside the fuel-containing fluid supply nozzle (12). In the present configuration, delamination is suppressed by making an angular variation around the condenser (23') small and a fuel condensing effect is thereby promoted to heighten the NOx reducing effect and improve the combustion efficiency.

[0092] Although effective configuration examples of the condenser (23, 23') are shown in FIGS. 16 and 17, the same effects are obtained even when a condenser with a similar structure, such as a triangular prism, etc., is employed.

[0093] FIG. 18 shows a structural example of a burner according to an embodiment of the present invention. FIG. 18A is a sectional view parallel to a fuel-containing fluid supply nozzle (12) surface formed through a long edge of an outlet part of the nozzle (12) (sectional view taken on line B-B of FIG. 18B), FIG. 18B is a sectional view taken on line A-A of FIG. 18A, and FIG. 18C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0094] In FIG. 18, a dam-like fluid distribution plate (24) is disposed at an inlet part of the fuel-containing fluid supply nozzle (12). The fuel-containing fluid (11) collides once with an upstream side of the dam-like fluid distribution plate (24) and, after being dispersed uniformly in the direction of a long edge of the fuel-containing fluid supply nozzle (12), is guided uniformly into the four flow passages partitioned by the fuel-containing fluid guide plates (19) inside the fuel-containing fluid supply nozzle (12) and supplied into the furnace (4) while being maintained in a uniform state.

[0095] FIG. 19 shows an example where an oil supply nozzle (41) is installed at a central part of the fuel-containing fluid supply nozzle (12). FIG. 19A is a sectional

view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 19B), FIG. 19B is a sectional view taken on line A-A of FIG. 19A, and FIG. 19C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0096] FIG. 20 shows an example where a gas ejection part, connected from a gas introduction tube (42) to gas supply nozzles (44) via a horizontal tube (43), is installed in a periphery of the flame stabilizer (17).

[0097] FIG. 20A is a sectional view taken through a central axis of the burner in a direction parallel to a longest axis of a major axis part of an outlet part (sectional view taken on line B-B of FIG. 20B), FIG. 20B is a sectional view taken on line A-A of FIG. 20A, and FIG. 20C is a front view of the outlet part of the burner as viewed from the furnace (4) side.

[0098] FIG. 21 shows, for a burner structure, a front view as viewed from the furnace (4) side (FIG. 21A) and a plan view (FIG. 21B) of the fuel-containing fluid supply nozzle (12), with which the cross-sectional shape perpendicular to the flow of the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) of the burner is rectangular, in a case where the fuel-containing fluid supply nozzle (12) is positioned with its long edge side directed in an up/down direction.

[0099] As viewed from the furnace (4) front side, the fuel-containing fluid ejection flows (20a, 20b) from the fuel-containing fluid supply nozzle (12) shown in FIG. 21 are formed obliquely toward mutually opposite sides in the horizontal direction to the left and right with respect to a plane perpendicular to the furnace wall surface in upper and lower directions of the long edge side of the nozzle (12). The forming of the present fuel-containing fluid ejection flows is achieved by applying the burner structure of FIG. 7 or FIG. 9.

[0100] FIG. 22 shows, for a burner structure, a front view as viewed from the furnace (4) side (FIG. 22A) and a plan view (FIG. 22B) of the fuel-containing fluid supply nozzle (12), with which the cross-sectional shape perpendicular to the flow of the fuel-containing fluid (11) flowing inside the fuel-containing fluid supply nozzle (12) of the burner is rectangular, in a case where the fuel-containing fluid supply nozzle (12) is positioned with its long edge side directed in an up/down direction.

[0101] As viewed from the furnace (4) front side, one fuel-containing fluid ejection flow (20b) from the fuel-containing fluid supply nozzle (12) shown in FIG. 22 is formed obliquely to the horizontal direction with respect to a plane perpendicular to the furnace wall surface in upper and lower directions of the long edge side of the nozzle (12) and the other fuel-containing fluid ejection flow (20c) is formed perpendicular to the furnace wall surface. The forming of the present fuel-containing fluid ejection flows is achieved by applying the burner structure of FIG. 8.

[0102] FIG. 23 shows an example where a plurality of the fuel-containing fluid supply nozzles (12), shown in FIG. 21, are positioned in three stages in the up/down

direction and four columns in the horizontal direction on a single furnace wall surface. Fuel-containing fluid supply nozzles (12) forming ejection flows (20a, 20b) in the same directions as the nozzles (12) shown in FIG. 21 are disposed at a right half of the single furnace wall, and fuel-containing fluid supply nozzles (12) forming ejection flows (20a, 20b) at mirror symmetric positions with respect to the fuel-containing fluid supply nozzles (12) shown in FIG. 21 are disposed at a left half of the furnace wall. By configuring the directions of the fuel-containing fluid ejection flows (20a, 20b) to be left/right symmetrical on the single furnace wall surface, a good left/right balance of flow and combustion states can be maintained in the furnace (4).

[0103] As long as the ejection flows (20a, 20b) from the fuel-containing fluid supply nozzles (12), disposed distributedly at left and right halves of a single furnace wall, are formed at mirror symmetrical positions, the directions of the fuel-containing fluid ejection flows (20a, 20b) from the nozzles (12) do not necessary have to be as illustrated.

[0104] FIG. 24 shows an example where a plurality of the fuel-containing fluid supply nozzles (12) are positioned in three stages in the up/down direction and four columns in the horizontal direction on a single furnace wall surface, with the fuel-containing fluid supply nozzles (12) shown in FIG. 22 being disposed mirror symmetrically at respective end columns at the left and right sides and the fuel-containing fluid supply nozzles (12) shown in FIG. 21 being disposed mirror symmetrically at two central columns. By aligning the burners with the rectilinear fuel-containing fluid ejection flows (20c) and the oblique fuel-containing fluid ejection flows (20b) along and adjacent the water walls of the side wall of the furnace (4), and aligning the fuel-containing fluid ejection flows (20a, 20b), which are inclined obliquely to the respective sides, along and adjacent the center, the dispersion within the furnace (4) is promoted to maintain the effect of promoting the combustion at the wake part of the furnace (4) while providing the effect of suppressing inflow of the flame to the vicinity of the side wall of the furnace (4) to prevent ash deposition.

[0105] FIG. 25 shows an example where the fuel-containing fluid supply nozzles (12) that are all the same and form the fuel-containing fluid ejection flows (20a, 20b) shown in FIG. 21 are disposed as the fuel-containing fluid supply nozzles (12) of all of the burners on the single furnace wall. The present embodiment provides a configuration with which collision of the fuel-containing fluid ejection flows (20a, 20b) can be avoided, especially in a furnace (4) of small capacity, and localized concentration of fuel is suppressed to effectively reduce the NOx concentration in the combustion gas and improve the combustion efficiency.

[0106] By selecting appropriately from the burner structures of FIGS. 21 to 25 in accordance with furnace dimensions, burner configurations, and other conditions, optimal combustion characteristics can be realized.

[0107] FIG. 26 is a plan view of a furnace wall of a boiler in which burners according to an embodiment of the present invention are disposed. In FIG. 26, in a boiler having spiral water wall tubes (25) on the furnace wall, rectangular openings (26) are installed and the various burners described as the embodiments of the present invention are mounted along an arrangement of the water wall tubes (25) that is oblique with respect to the horizontal direction. By disposing the openings (26) along the spiral water wall tubes (25), a number of the water wall tubes (25) necessary for forming the openings (26) can be minimized to improve economy.

[0108] As described above, a combustion equipment configured from the respective embodiments of the present invention has a characteristic of enabling the combustion space to be utilized effectively because the fuel-containing fluid ejection flows (20) spread inside the furnace (4), and with the configuration shown in FIG. 26, because the fuel-containing fluid ejection flows (20a, 20b) spread in the horizontal (width) direction of the furnace (4), a distribution of the fuel-containing fluid (11) in the horizontal (width) direction of the furnace (4) is made uniform and the practical furnace retention time is made even longer, thereby effectively contributing to the reduction of NOx concentration in the combustion gas and improvement in combustion efficiency.

[0109] FIG. 27 is a plan view of a furnace wall of a boiler in which burners according to an embodiment of the present invention are disposed. In FIG. 27, in a boiler having water wall tubes (25) extending in the vertical direction on the furnace wall, rectangular openings (26) are installed and the various burners described as the embodiments of the present invention are mounted along the arrangement of the water wall tubes (25). By disposing the openings (26) along the water wall tubes (25), the number of the water wall tubes (25) necessary for forming the openings (26) can be minimized to improve economy.

[0110] In the present configuration, by using the burners shown in FIGS. 7 to 9 so that the directions of the fuel-containing fluid ejection flows (20a, 20b) are directed in mutually different directions to promote the dispersion of the fuel-containing fluid (11) even in the horizontal (width) direction of the furnace (4), the dispersion of the fuel-containing fluid (11) in the entirety of the furnace is promoted, thereby effectively contributing to NOx reduction and improvement in combustion efficiency.

[0111] An oil or a gas is generally used as an auxiliary fuel in a burner, and even when supply nozzles for such fuels are installed at a part of the burners according to the embodiments of the present invention, the characteristics and effects of the burners according to the embodiments of the present invention are maintained.

INDUSTRIAL APPLICABILITY

[0112] As a burner structure capable of following a trend toward burners of large capacity while reducing cost without lowering combustion performance, the

present invention is high in future industrial applicability.

DESCRIPTION OF THE SYMBOLS

[0113] 3 windbox, 4 furnace, 10 fuel-containing fluid flowpassage, 10a fuel-containing fluid connecting part, 11 fuel-containing fluid, 12 fuel-containing fluid supply nozzle 13 secondary air, 14 tertiary air, 15 combustion air sleeve, 15a tertiary air guide plate, 16 burner throat, 17 flame stabilizer, 17a secondary air guide plate, 19 fuel-containing fluid guide plate, 20, 20a, 20b, 20c, 20d fuel-containing fluid ejection flow, 21a, 21b fuel-containing fluid direction changing guide plate, 22 fuel-containing fluid partitioning plate, 22a negative pressure region, 23, 23' condenser, 24 fluid distribution plate, 25 water wall tube, 26 opening, 31 unignited region, 32 ignited region, 33 ignition position, 41 oil supply nozzle, 42 gas introduction tube, 43 horizontal tube, 44 gas supply nozzle, L1 unignited distance, L2 distance from ignition position to central part of fuel-containing fluid ejection flow

Claims

1. A burner including: a fuel-containing fluid supply nozzle (12) supplying a fuel-containing fluid (11), containing a solid fuel and a medium for transfer of the solid fuel, to an outlet part disposed on a wall surface of a furnace (4) from a connecting part (10a) of a fuel-containing fluid transfer flow passage (10) that transfers the fluid (11); and one or more air supply nozzles (15) supplying combustion air and disposed at an outer peripheral part of the fuel-containing fluid supply nozzle (12); wherein, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part disposed on the wall surface of the furnace (4), a cross section of the fuel-containing fluid supply nozzle (12) perpendicular to a flow of the fluid (11) has a rectangular, elliptical, or substantially elliptical shape with major and minor axis parts and, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, a size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along a direction of the flow of the fluid (11).
2. The burner according to Claim 1, wherein the fuel-containing fluid supply nozzle (12) has a configuration such that, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, the size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along the direction of the flow of the fluid (11) and a size of the minor axis part is unchanged.
3. The burner according to Claim 1, wherein the fuel-containing fluid supply nozzle (12) has a configuration such that, from the connecting part (10a) of the fluid transfer flow passage (10) toward the outlet part, the size of the major axis part of the cross section perpendicular to the flow of the fluid (11) increases gradually along the direction of the flow of the fluid (11) and a size of the minor axis part decreases gradually along the direction of the flow of the fluid (11).
4. The burner according to any of Claims 1 to 3, wherein the fuel-containing fluid supply nozzle (12) has, in an interior thereof, fuel-containing fluid guide plates (19) plurally partitioning the flow of the fuel-containing fluid (11).
5. The burner according to Claim 4, wherein the fuel-containing fluid guide plates (19) are disposed at a plurality of different inclination angles with respect to a plane passing along a line extending a central axis in the direction of the flow of the fluid (11) in the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a shortest axis of the minor axis part of the nozzle (12).
6. The burner according to any of Claims 1 to 5, wherein the fuel-containing fluid supply nozzle (12) has, in an interior of the outlet thereof, fuel-containing fluid direction changing guide plates (21) forcibly changing a direction of ejection flow of the fuel-containing fluid (11).
7. The burner according to Claim 6, wherein the fuel-containing fluid direction changing guide plates (21) are disposed in a plurality of mutually different directions with respect to a plane passing along a line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a longest axis of the major axis part of the nozzle (12).
8. The burner according to Claim 6, wherein the fuel-containing fluid direction changing guide plates (21) for a portion of the fuel-containing fluid (11) are disposed parallel to a plane passing along a line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a longest axis of the major axis part of the nozzle (12), and the fuel-containing fluid direction changing guide plates (21) for another portion of the fuel containing fluid (11) are disposed at an inclination angle with respect to the plane passing along the line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to the longest axis of the major axis part of the nozzle (12).
9. The burner according to Claim 4, wherein

- the fuel-containing fluid supply nozzle (12) is partitioned into a plurality of flow passages by the fuel-containing fluid guide plates (19), and central axes of the respective flow passages are disposed at the wall surface of the furnace (4) at a plurality of mutually different inclination angles with respect to a plane passing along a line extending the central axis of the fuel-containing fluid supply nozzle (12) toward the furnace (4) and parallel to a longest axis of the major axis part of the nozzle (12) outlet.
10. The burner according to any of Claims 1 to 9, wherein fuel-containing fluid partitioning plates (22), capable of plurally partitioning the outlet part of the fuel-containing fluid supply nozzle (12), are disposed at the outlet part.
 11. The burner according to any of Claims 1 to 10, wherein a flame stabilizer (17) with an L-shaped cross section is disposed at the outlet part of the fuel-containing fluid supply nozzle (12).
 12. The burner according to Claim 11, wherein a guide plate (17a) outwardly changing an ejection direction of the combustion air in a periphery of the flame stabilizer (17) is disposed at a front end of the L-shaped flame stabilizer (17).
 13. The burner according to any of Claims 1 to 12, wherein a combustion air guide plate (15a), outwardly spreading an ejection direction of the combustion air at an outer side of the one or more combustion air supply nozzles (15) disposed at the outer peripheral part of the nozzle (12) with respect to a fuel ejection direction, is disposed at a front end of the fuel-containing fluid supply nozzle (12).
 14. The burner according to any of Claims 1 to 13, wherein a condenser (23), narrowing the flow passage of the fuel-containing fluid (11) once and then expanding the flow passage again, is disposed in an interior of the fuel-containing fluid supply nozzle (12).
 15. The burner according to any of Claims 1 to 14, wherein a fluid distribution plate (24), distributing the fuel uniformly inside the fuel-containing fluid supply nozzle (12) is disposed at an inlet part of the fuel-containing fluid supply nozzle (12).
 16. The burner according to any of Claims 1 to 15, wherein a nozzle (41, 44), ejecting a liquid fuel or a gas fuel that is an auxiliary fuel to a vicinity of the fluid (11) ejected from the fuel-containing fluid supply nozzle (12), is disposed at a vicinity of the fuel-containing fluid supply nozzle (12).
 17. A combustion equipment wherein the burners according to any of Claims 1 to 16 are disposed in a plurality of stages in an up/down direction at each of two opposing furnace walls, and a plurality of burners disposed at each stage are disposed respectively symmetrically in wall surface regions divided in two at a central part of width in a horizontal direction of the same furnace wall.
 18. A combustion equipment wherein the burners according to any of Claims 1 to 16 are disposed in a plurality of stages in an up/down direction at each of two opposing furnace walls, and burners, which, among the plurality of burners disposed in each stage of the same furnace wall, are adjacent each other in a horizontal direction, are burners of the same structure.
 19. A boiler including: a furnace wall formed by spirally winding a set of water wall tubes (25) inclined with respect to a horizontal direction; wherein openings (26) of rectangular, elliptical, or substantially elliptical shape are disposed in the furnace wall along a longitudinal direction of the water wall tubes (25) and the burner according to any of Claims 1 to 16 is mounted in each opening (26).
 20. A boiler including: a furnace wall formed by a set of water wall tubes (25) extending in a vertical direction; wherein openings (26) of rectangular, elliptical, or substantially elliptical shape are disposed in the furnace wall along a longitudinal direction of the water wall tubes (25) and the burner according to any of Claims 1 to 16 is mounted in each opening (26).

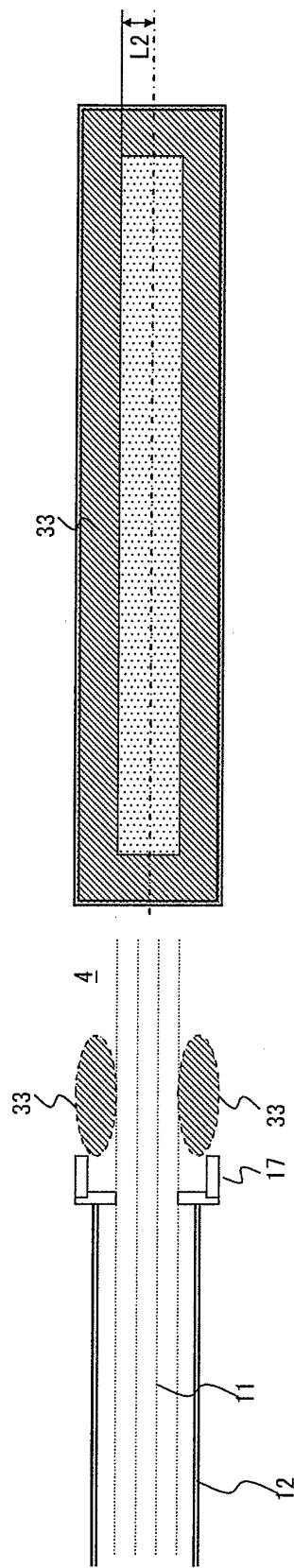


FIG. 1B

FIG. 1A

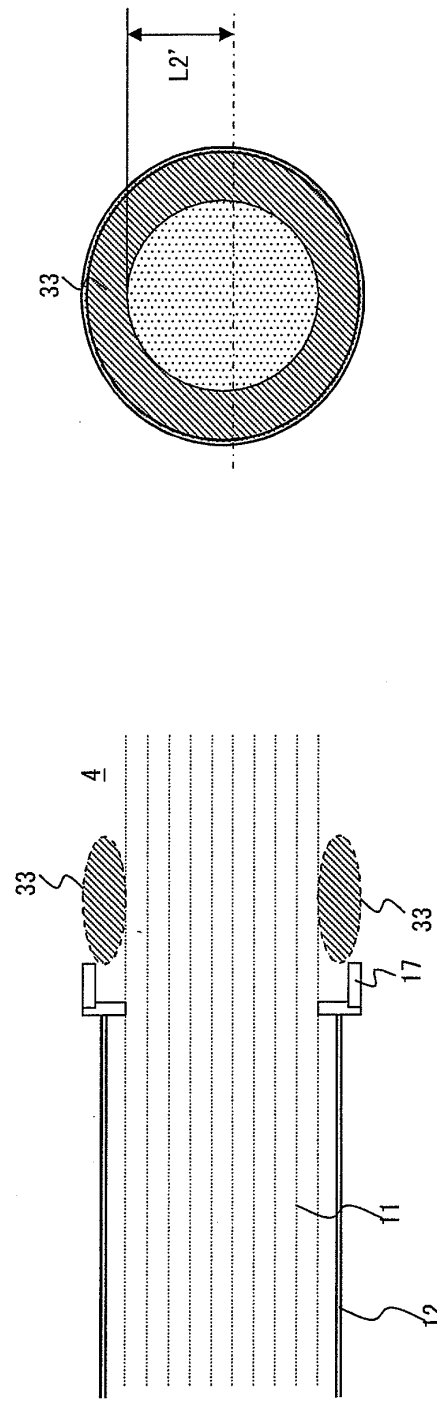


FIG. 1D
PRIOR ART

FIG. 1C
PRIOR ART

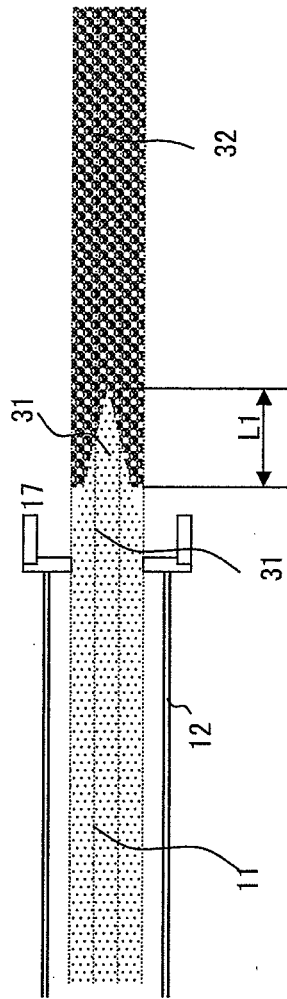


FIG. 2 A

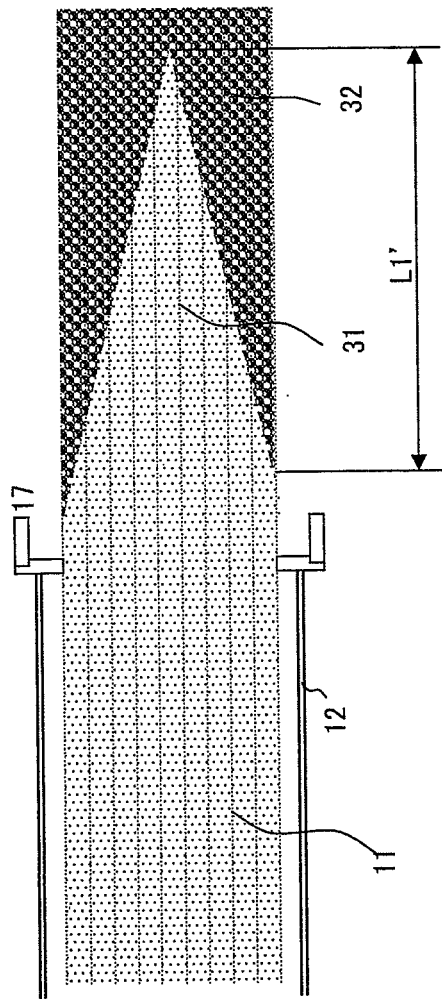


FIG. 2 B
PRIOR ART

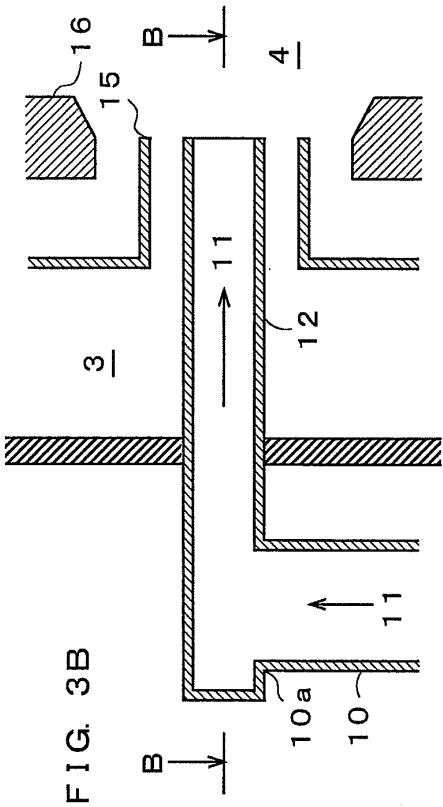
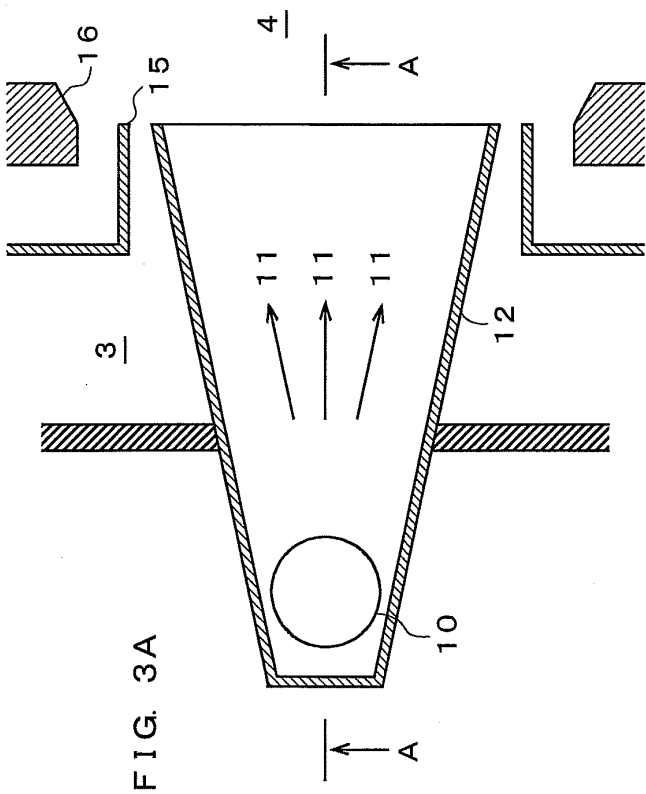
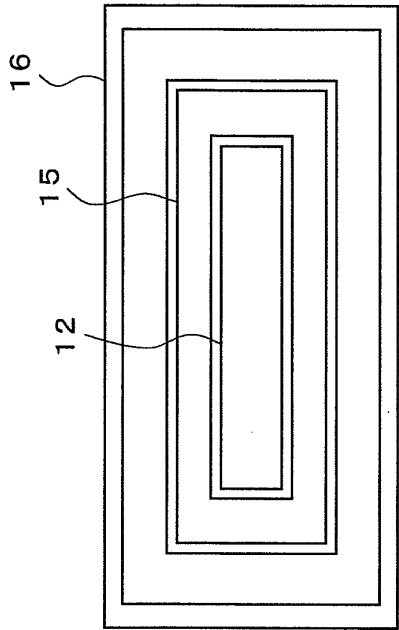


FIG. 3C



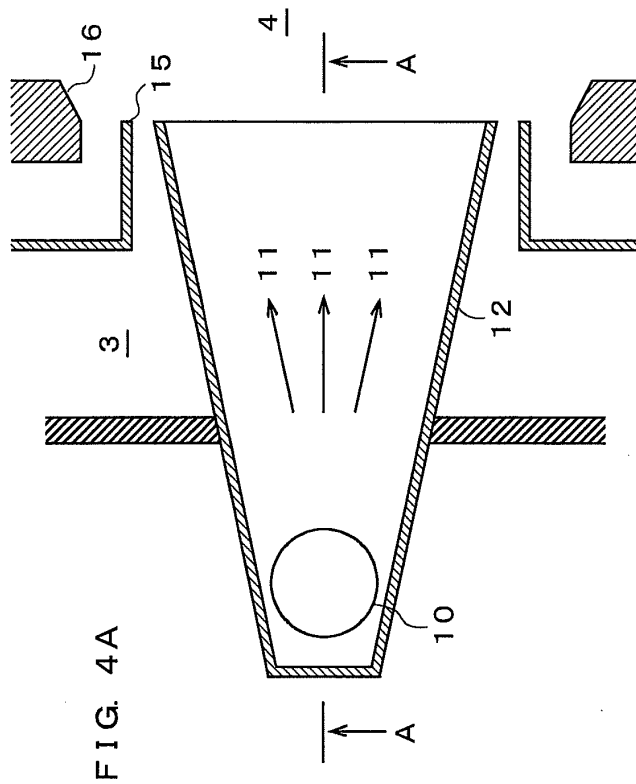
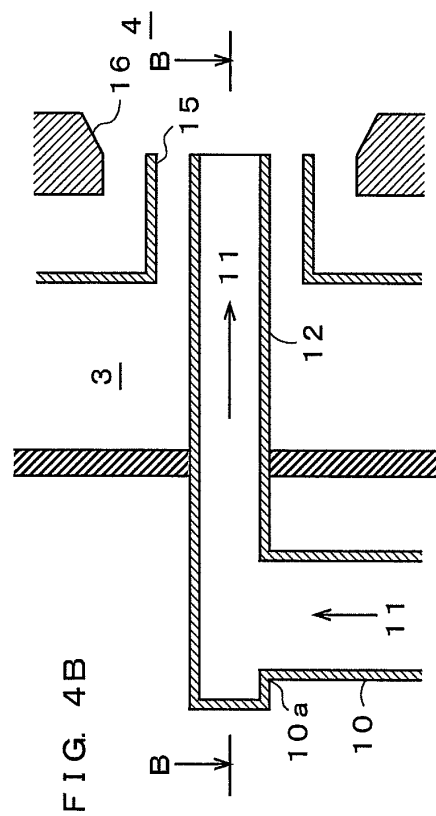
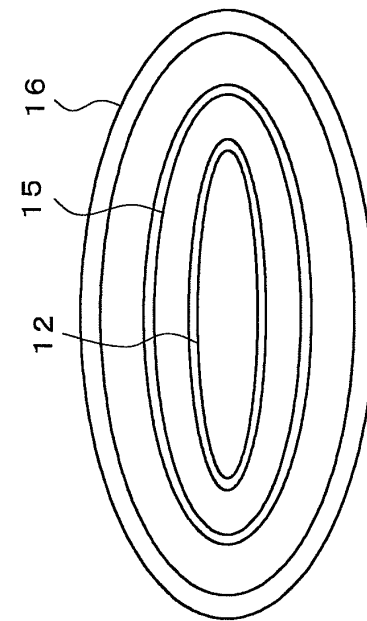
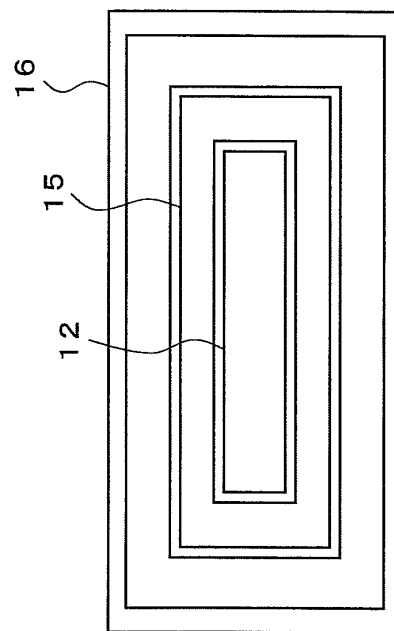
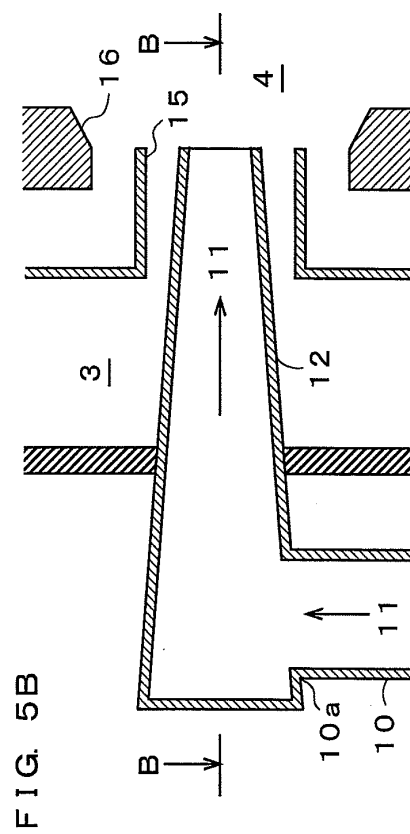
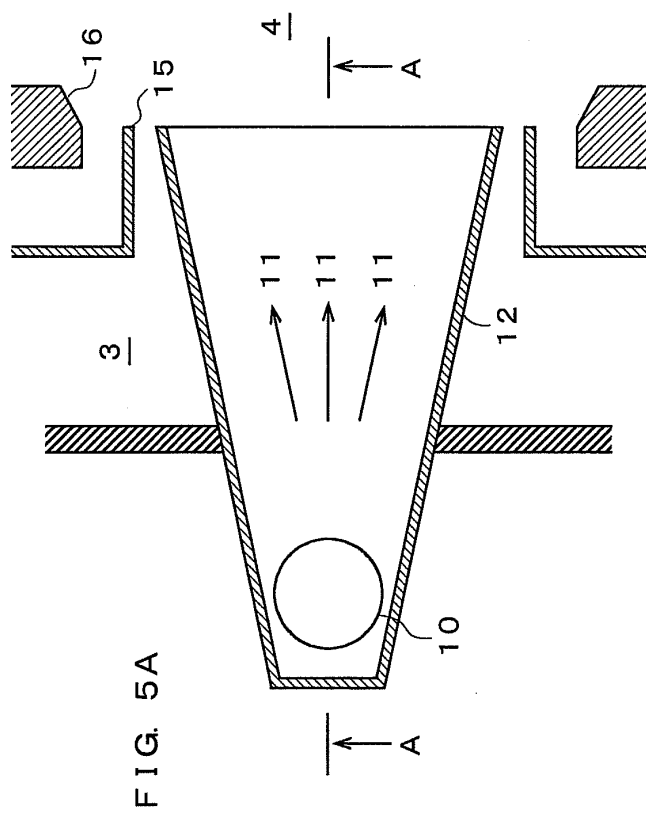


FIG. 4C





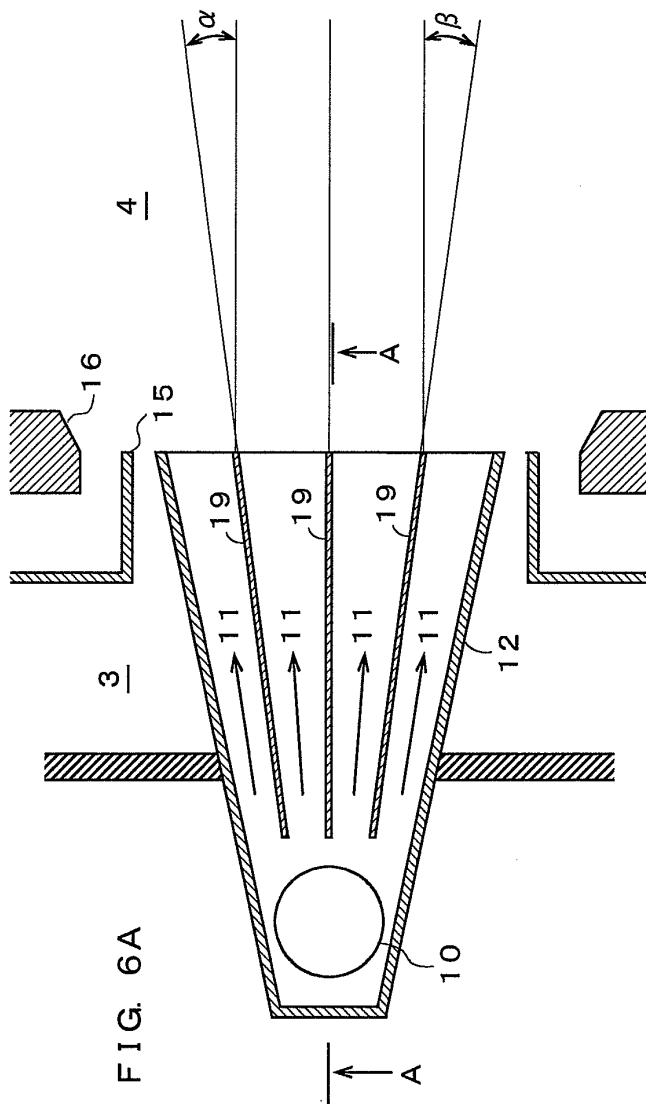
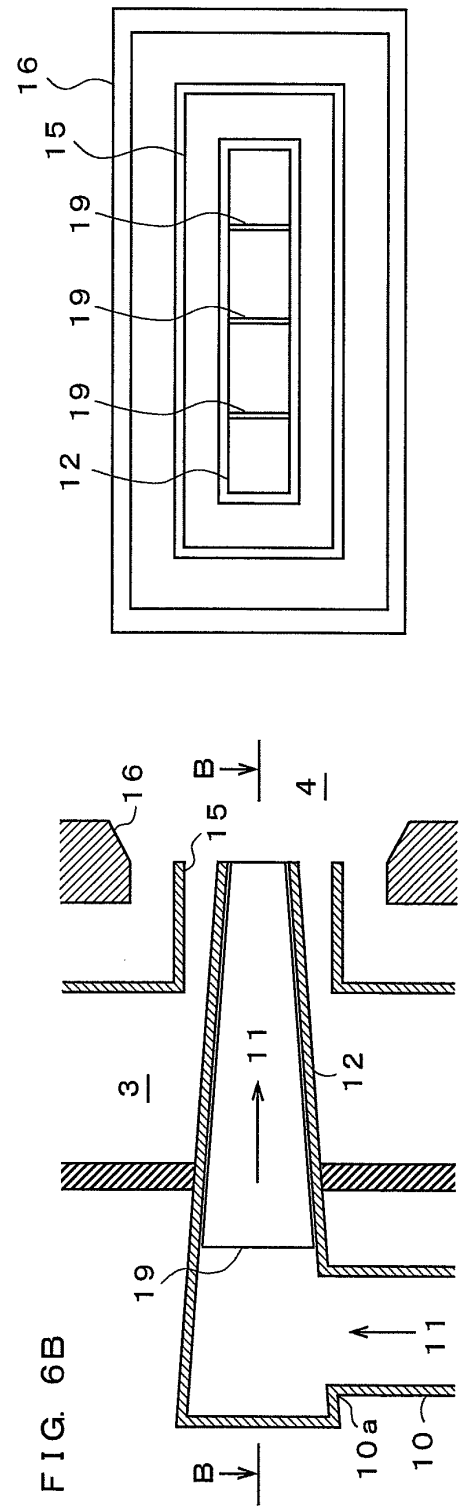


FIG. 6C



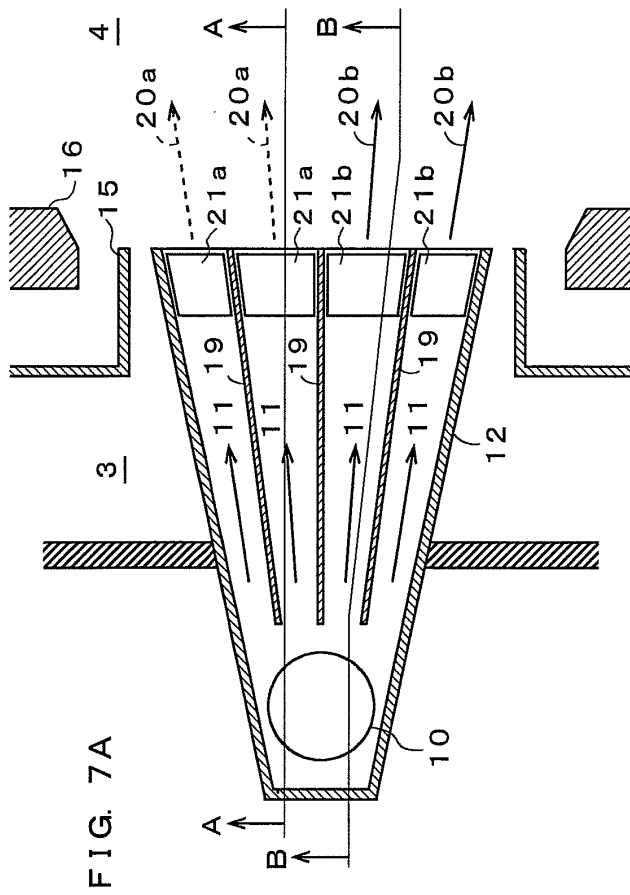


FIG. 7A

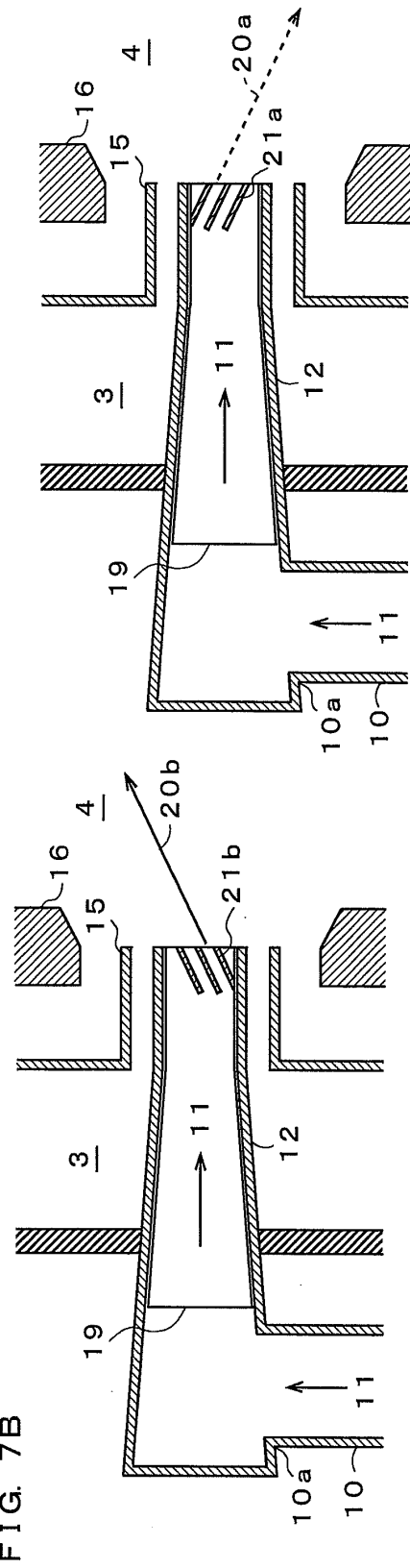
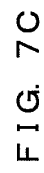


FIG. 7B

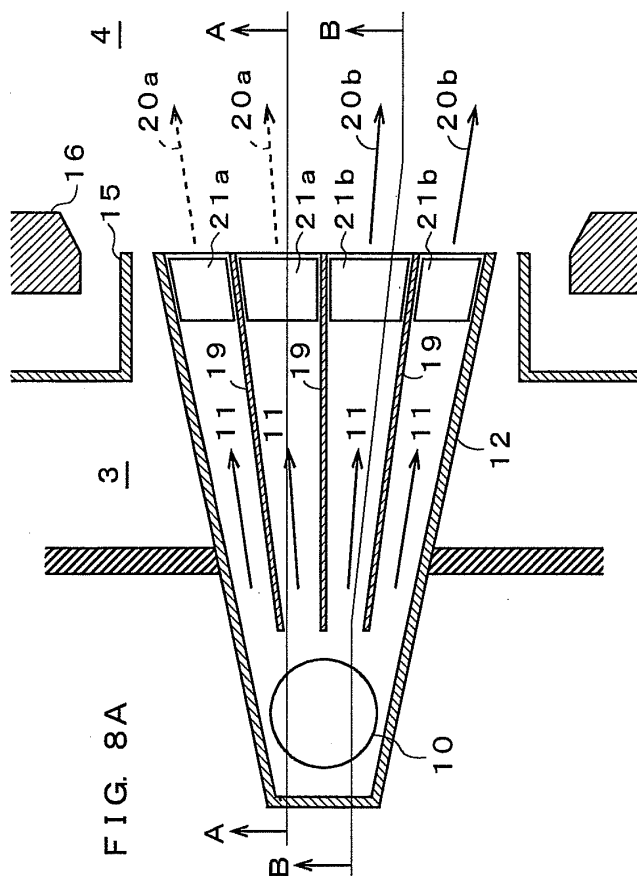
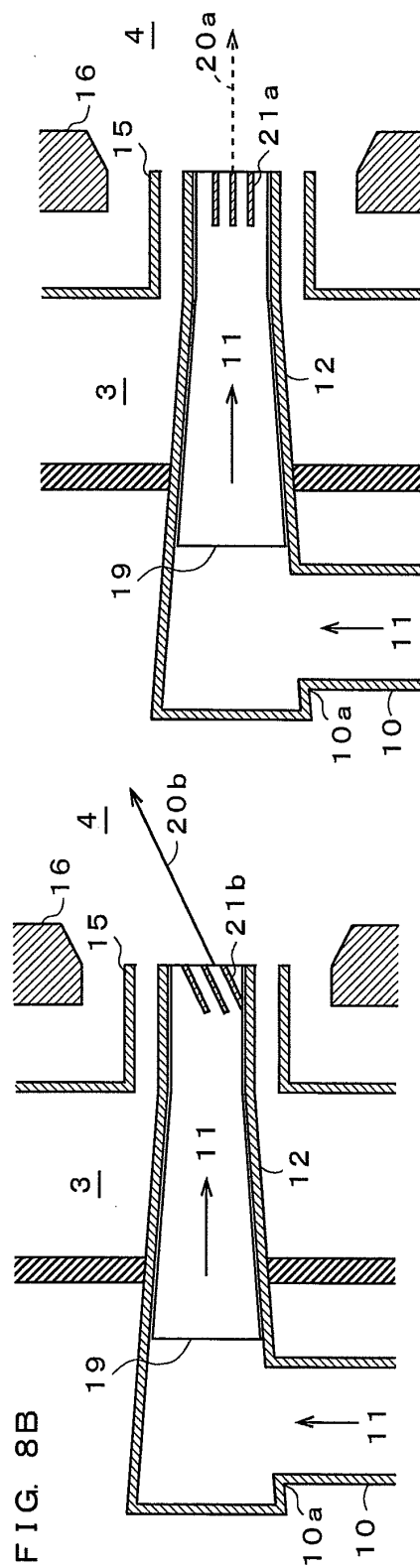
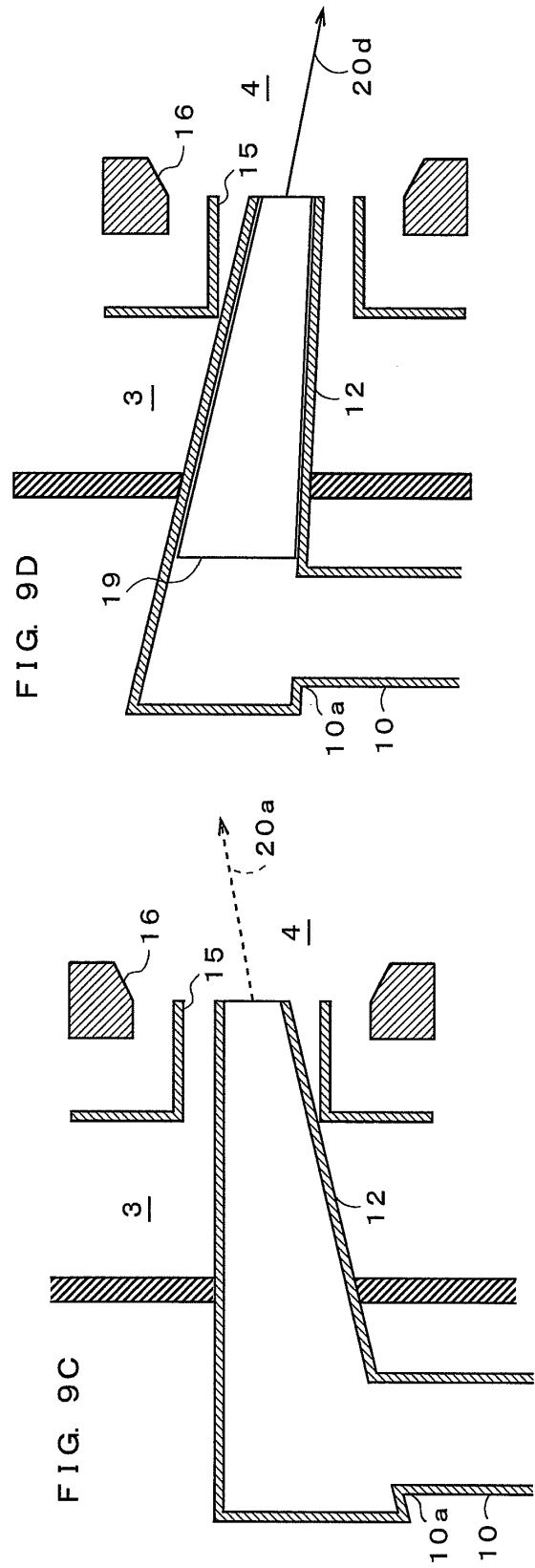
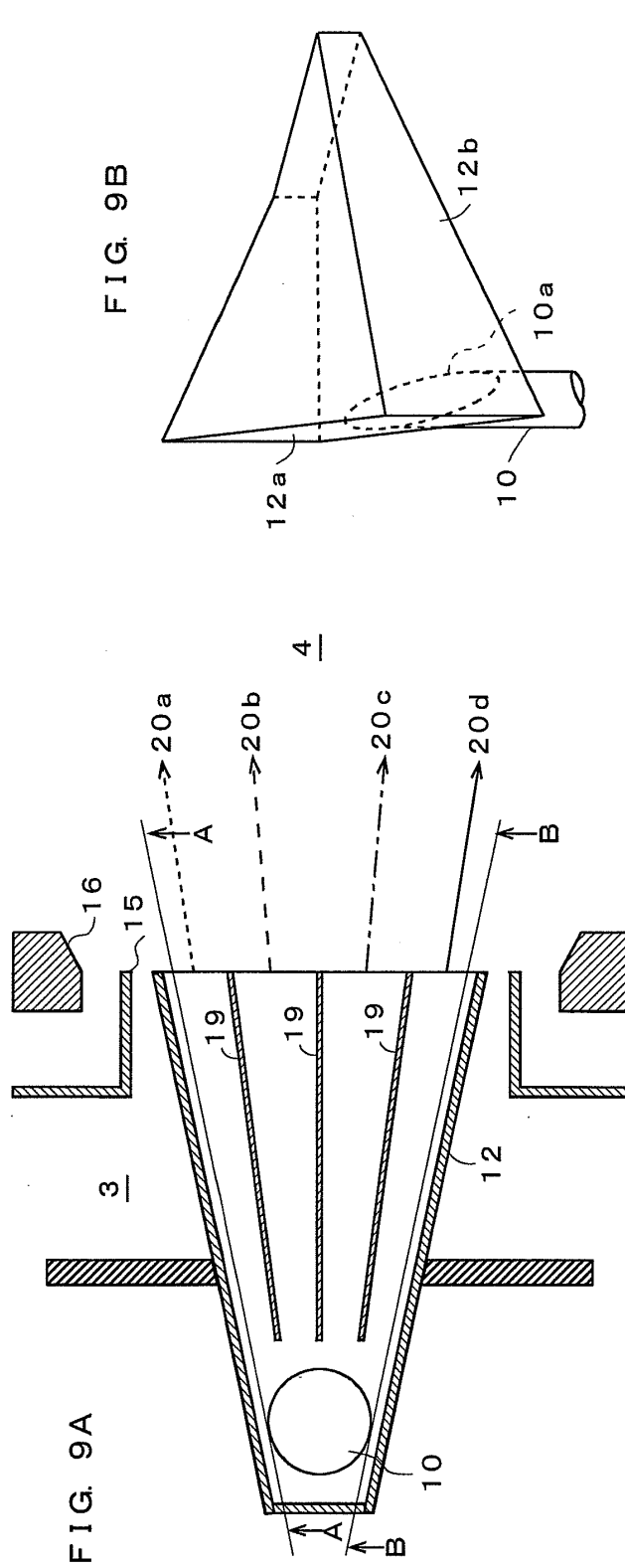


FIG. 8C





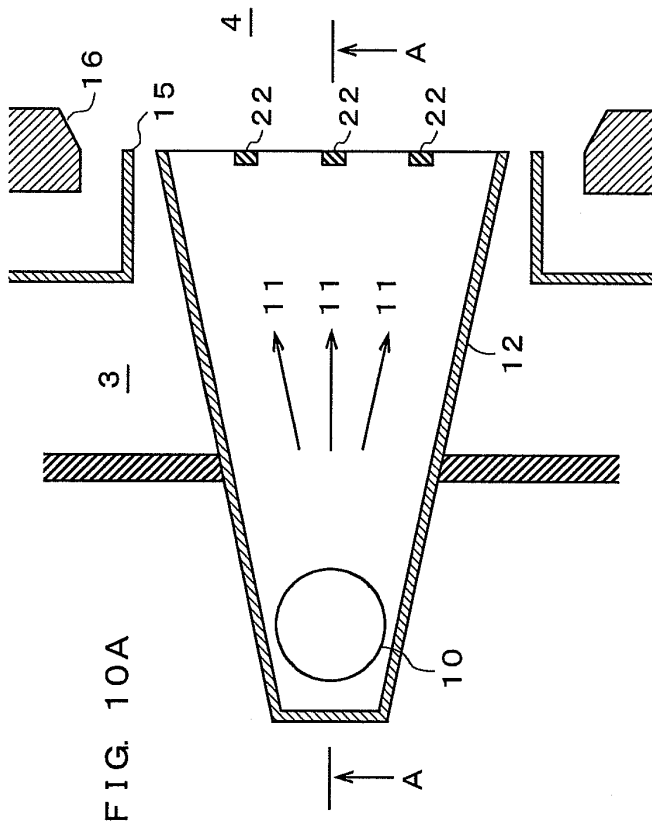


FIG. 10A

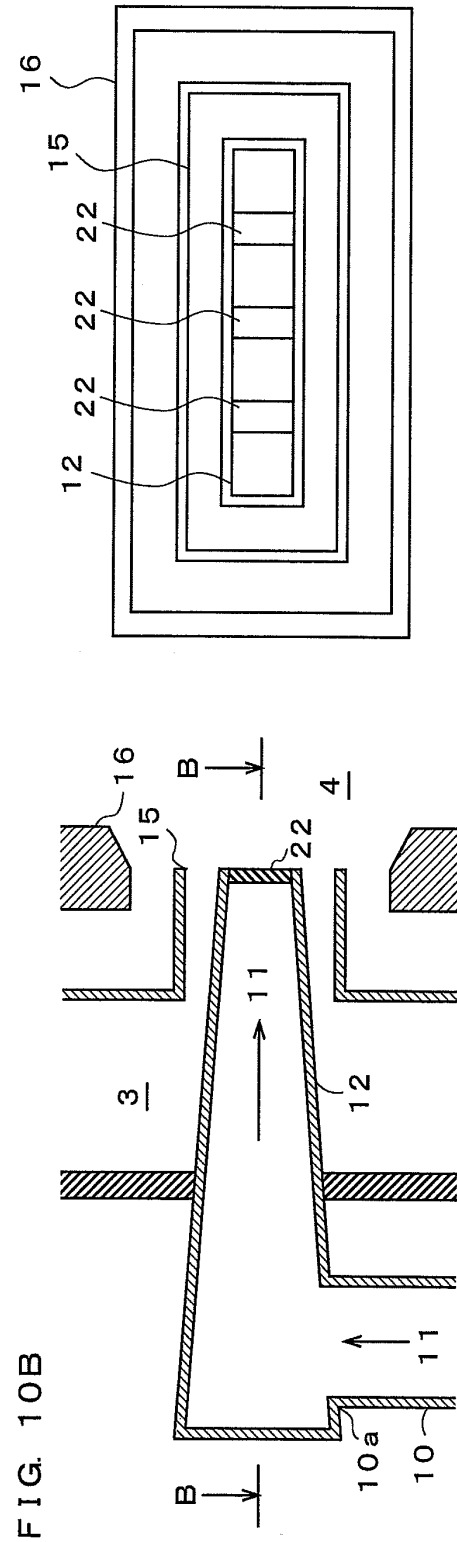
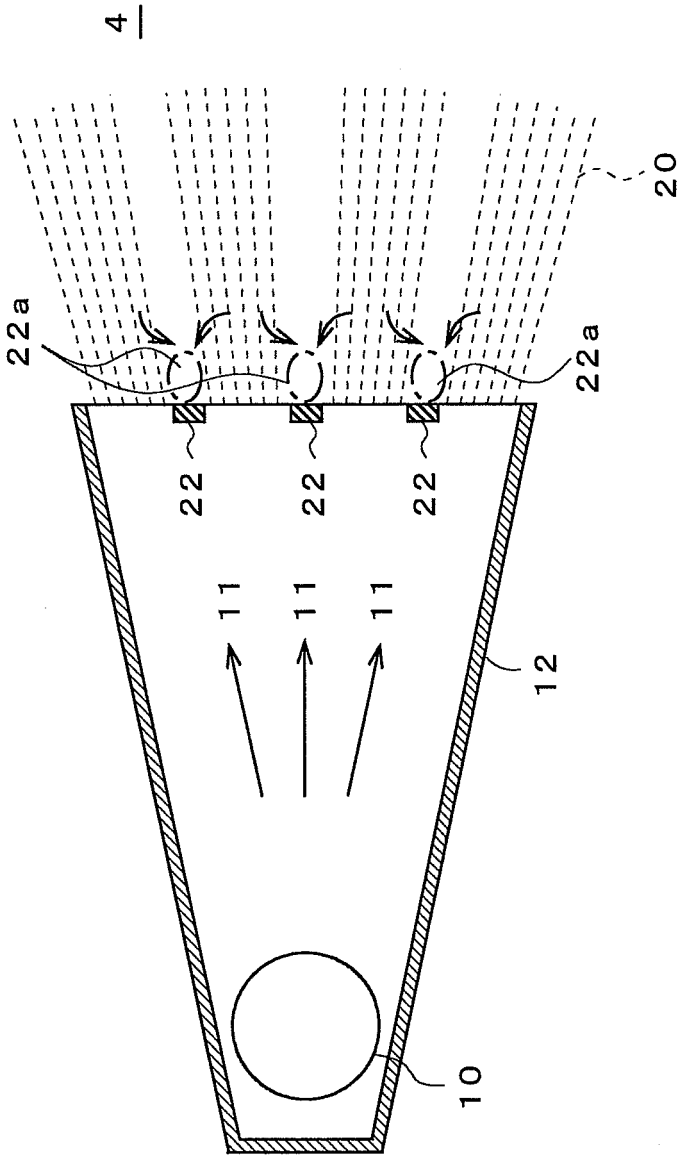


FIG. 10C

FIG. 11



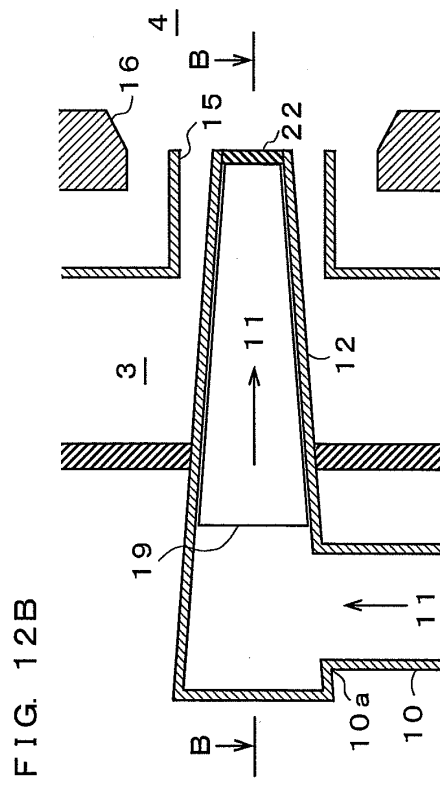
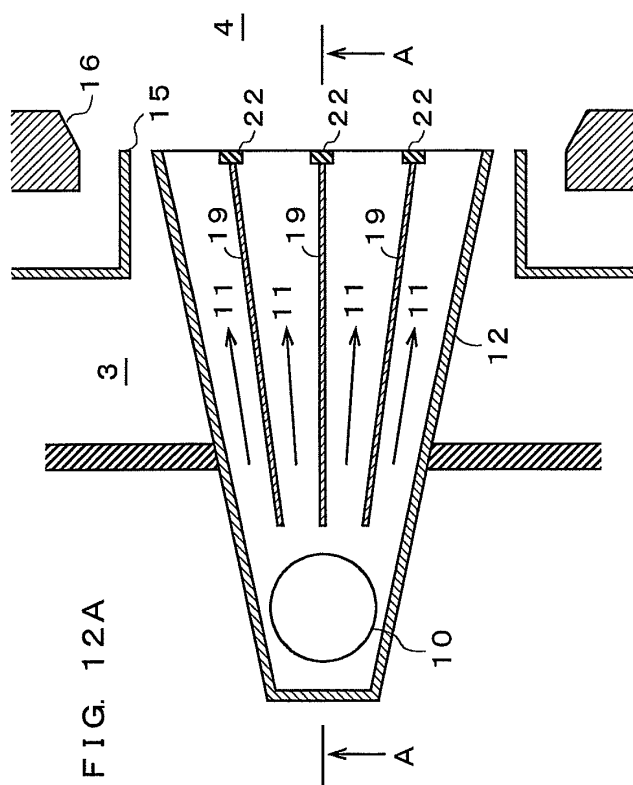
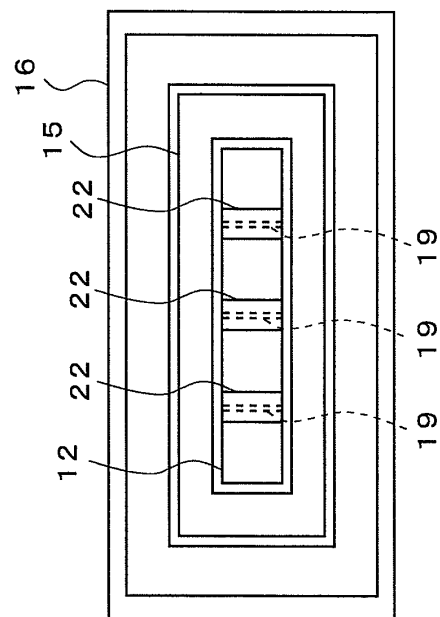


FIG. 12C



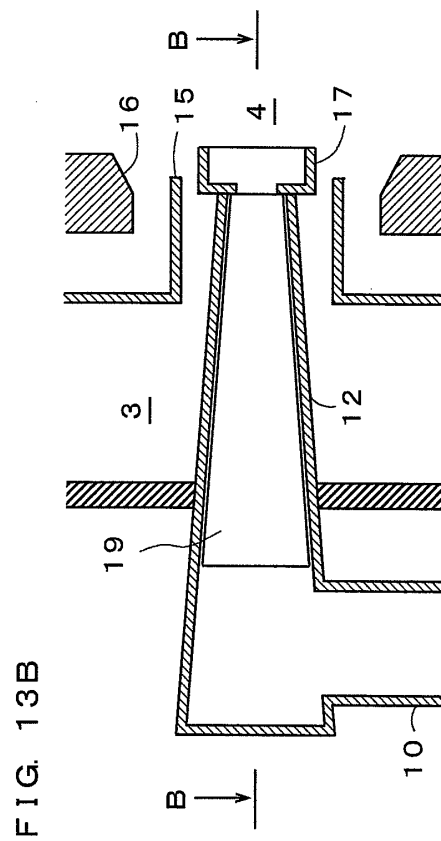
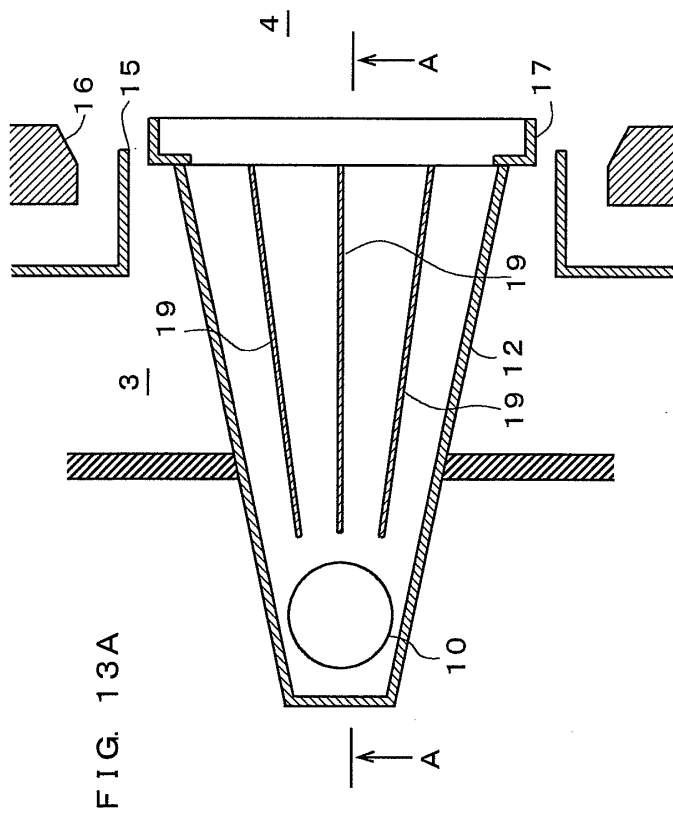
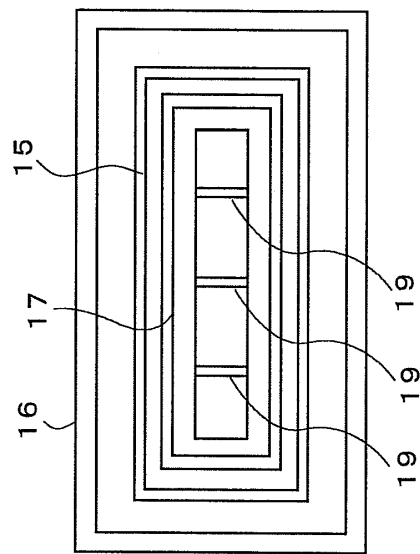


FIG. 13C



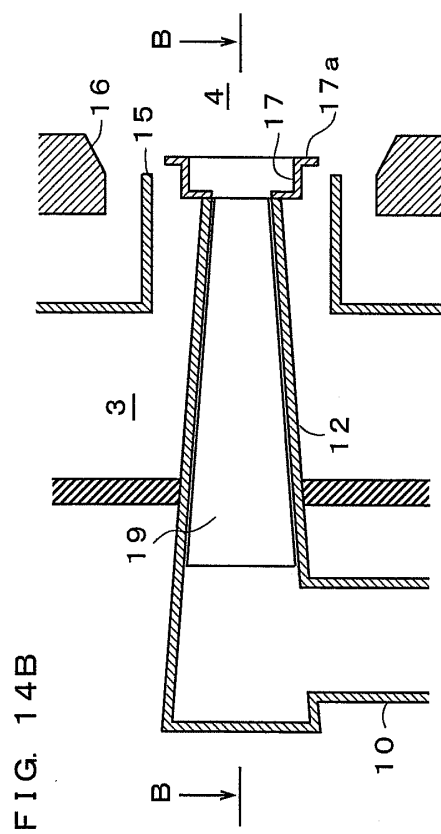
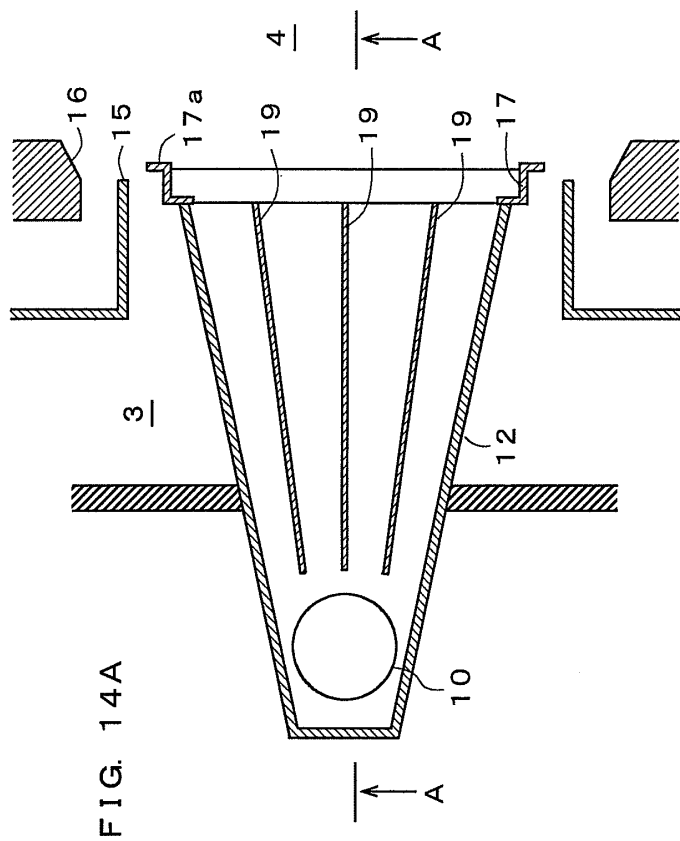
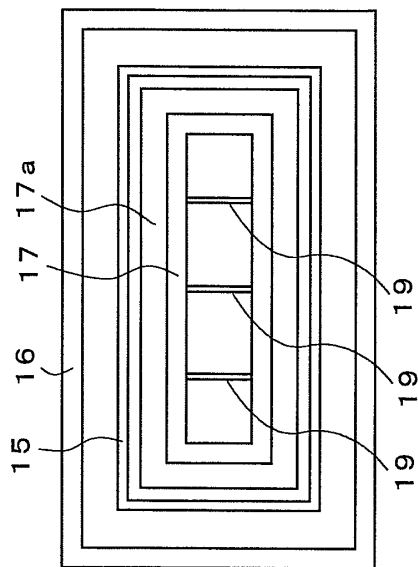


FIG. 14C



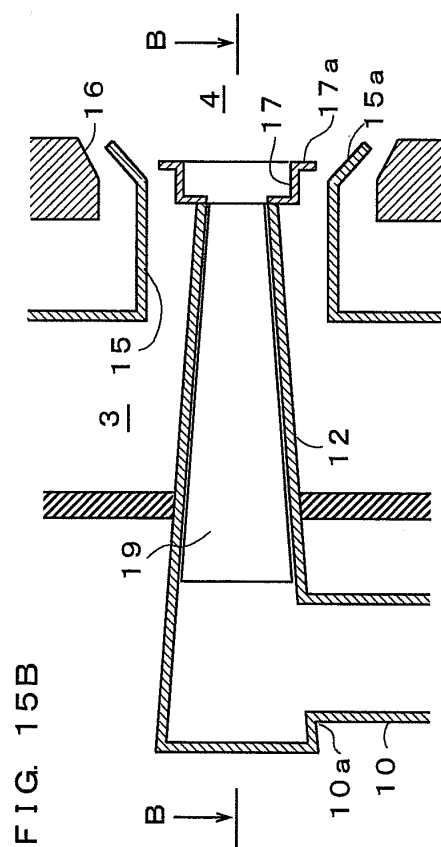
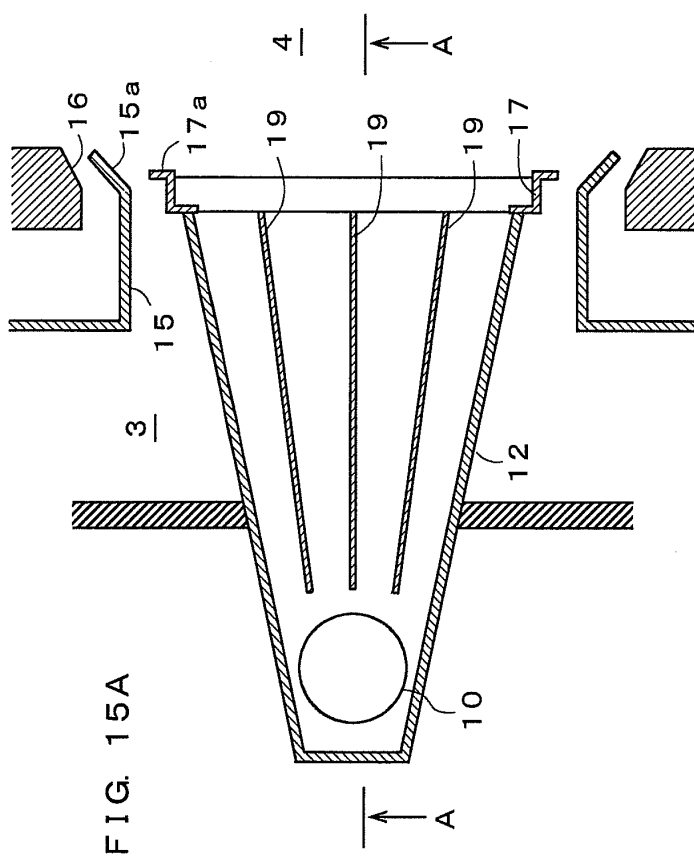
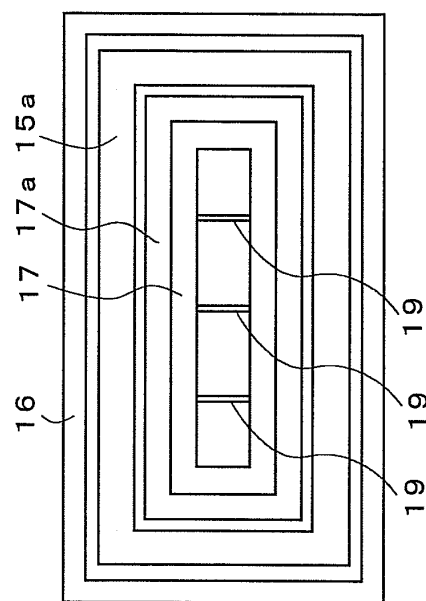


FIG. 15C



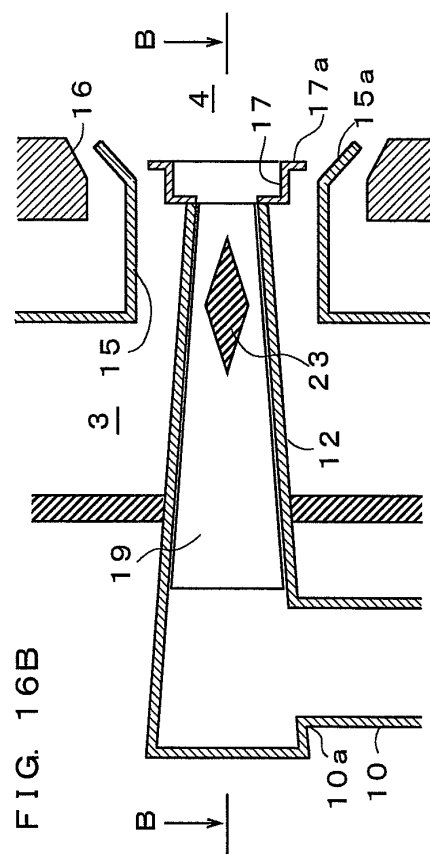
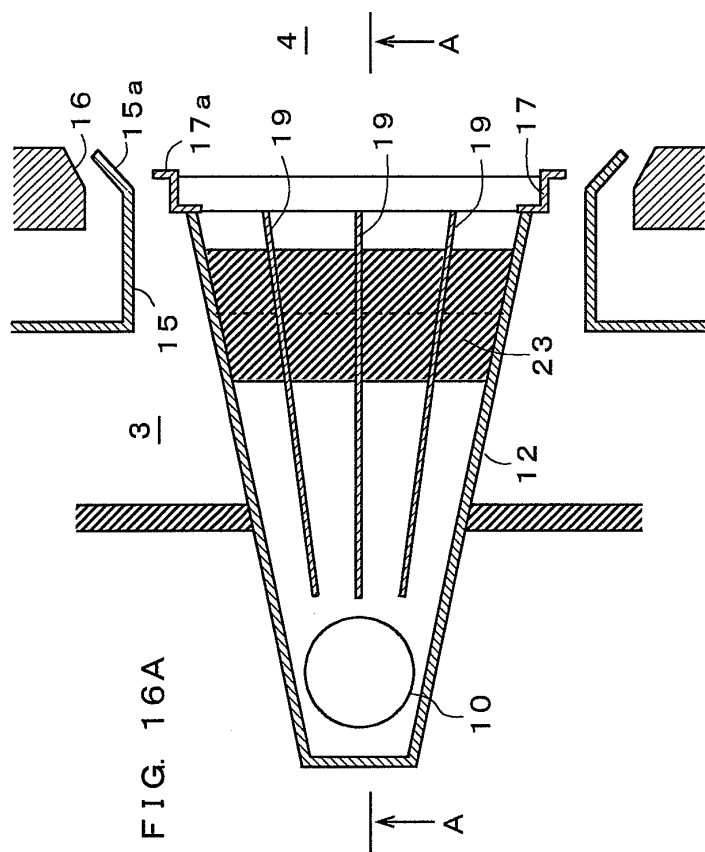
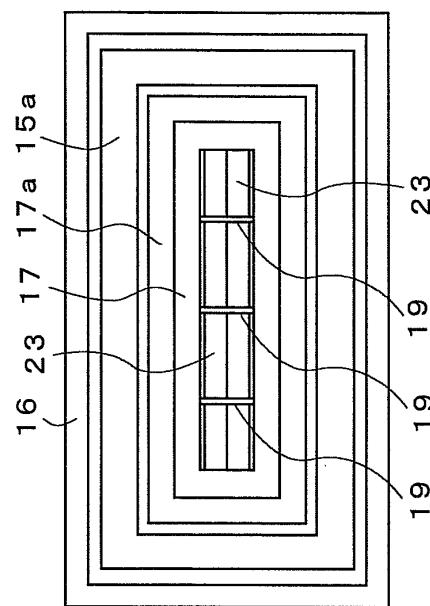


FIG. 16C



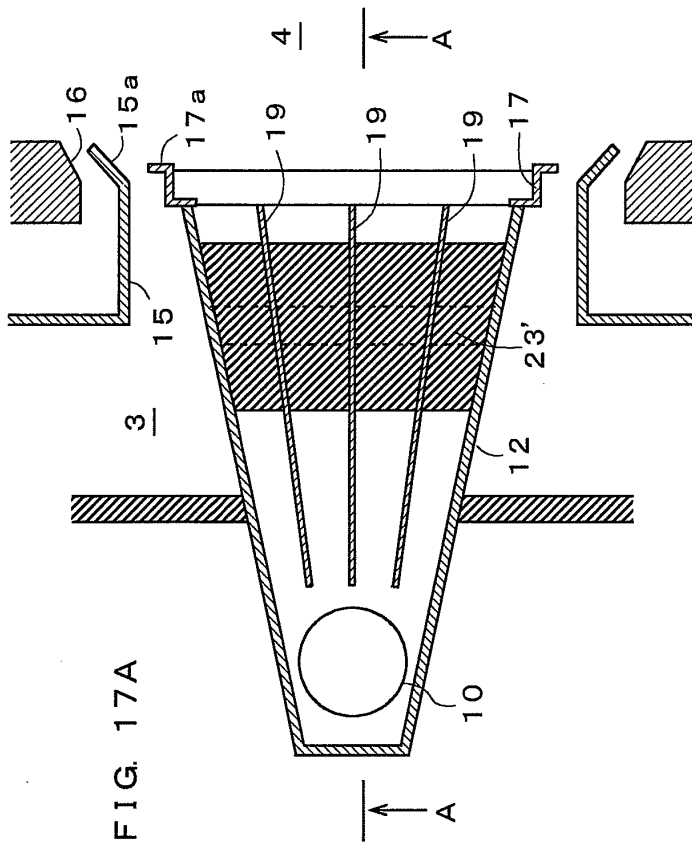
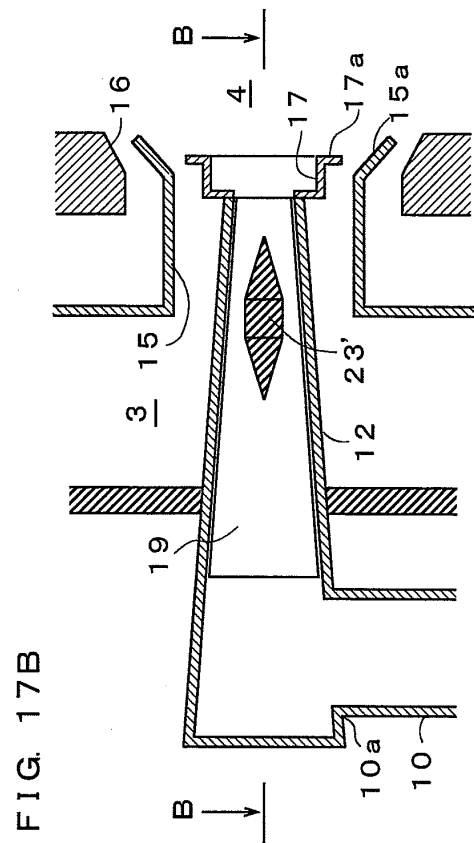
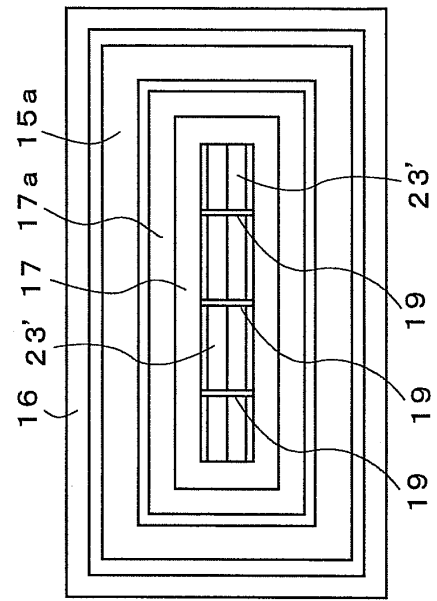


FIG. 17C



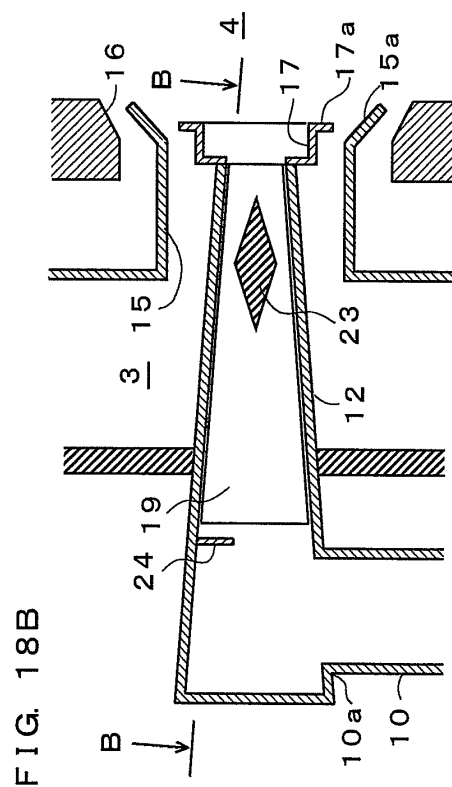
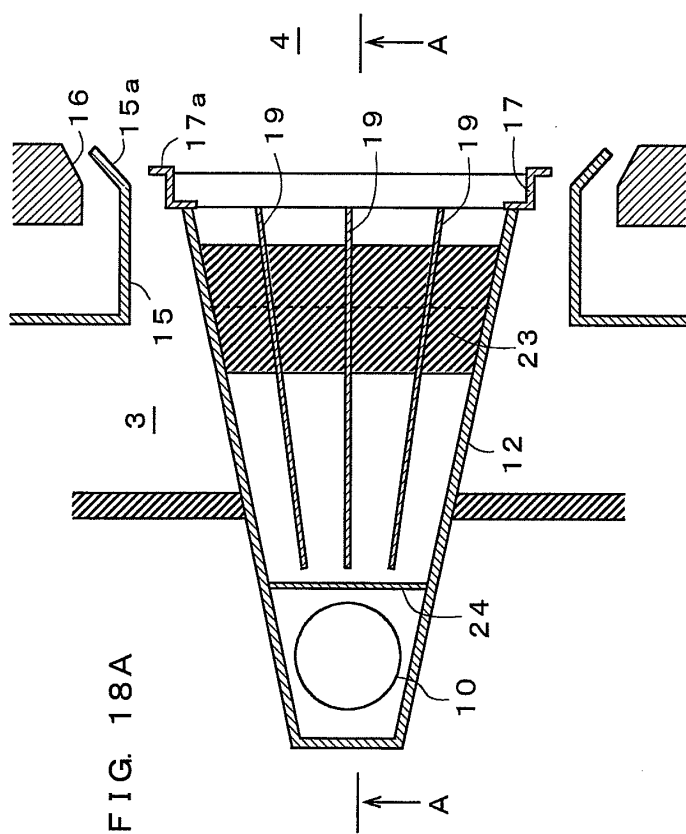
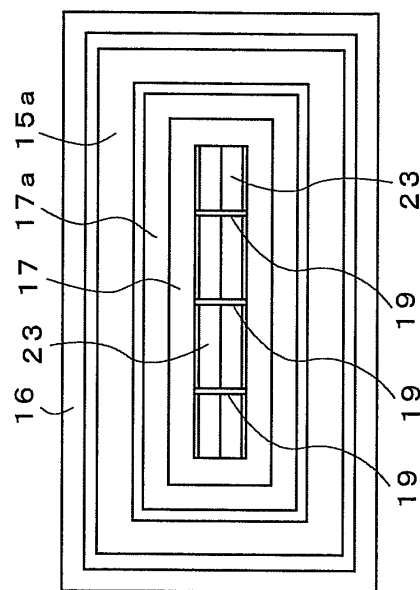


FIG. 18C



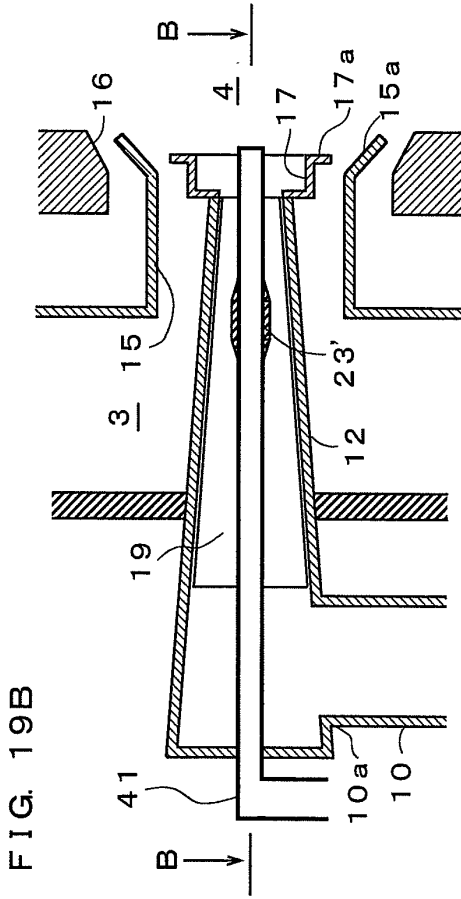
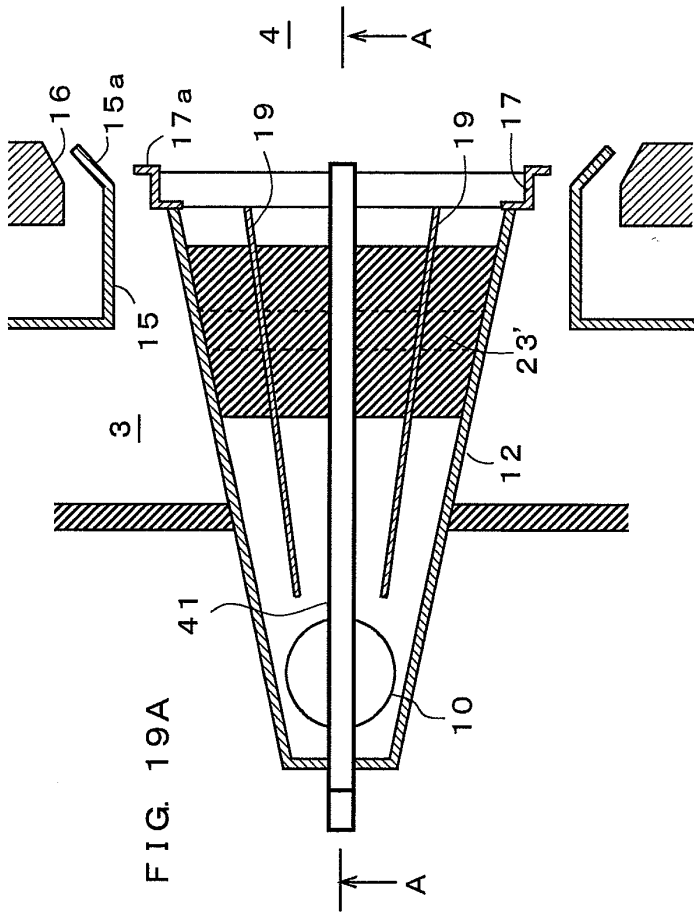
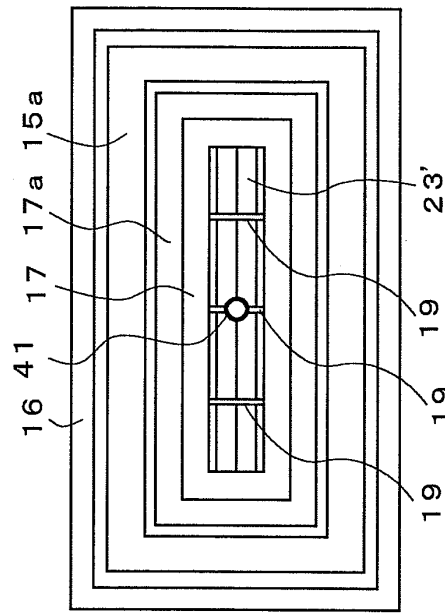


FIG. 19C



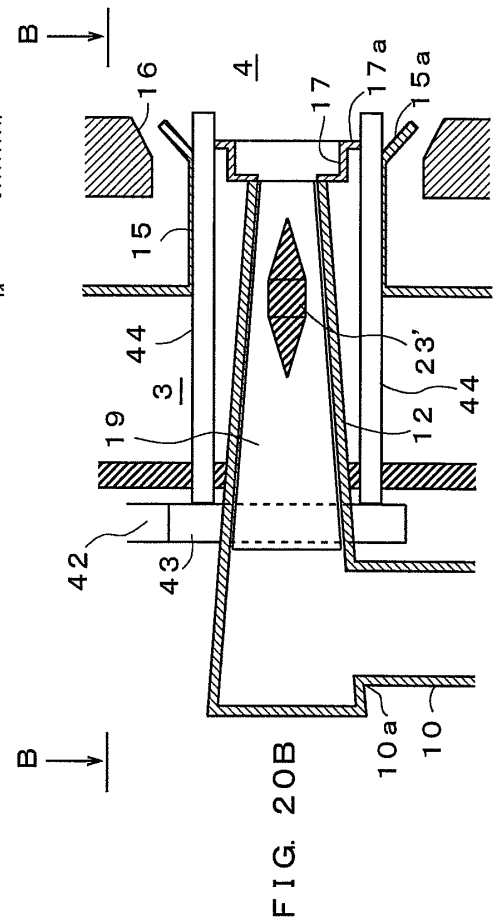
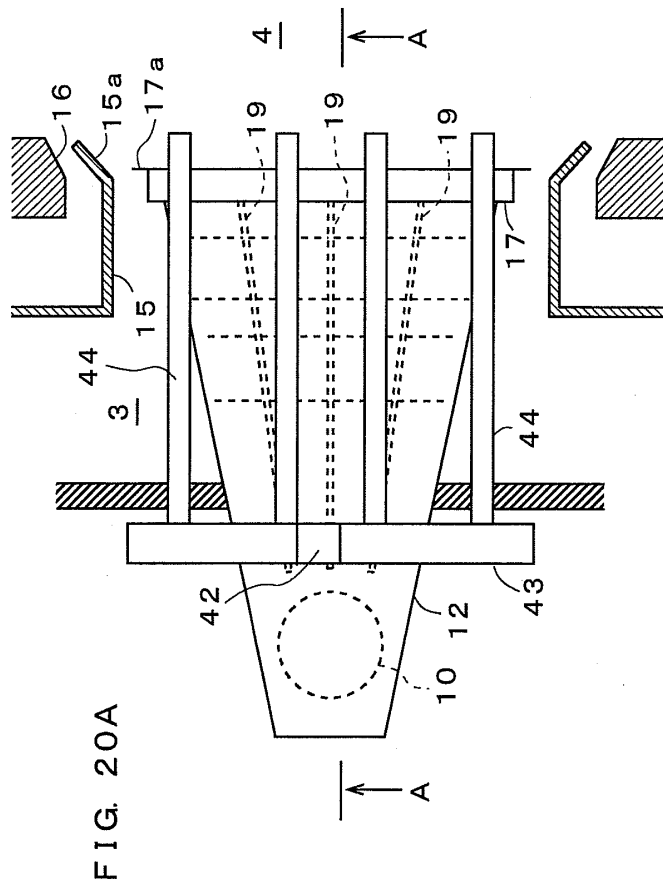
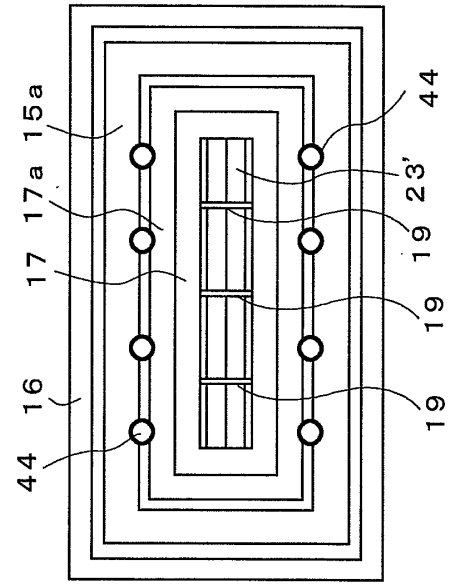


FIG. 20C



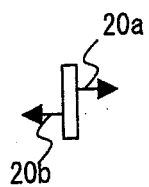


FIG. 21A

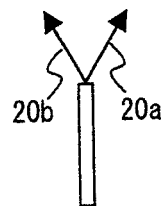


FIG. 21B

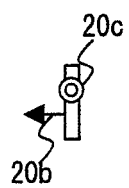


FIG. 22A

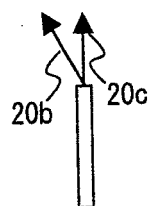


FIG. 22B

FIG. 23

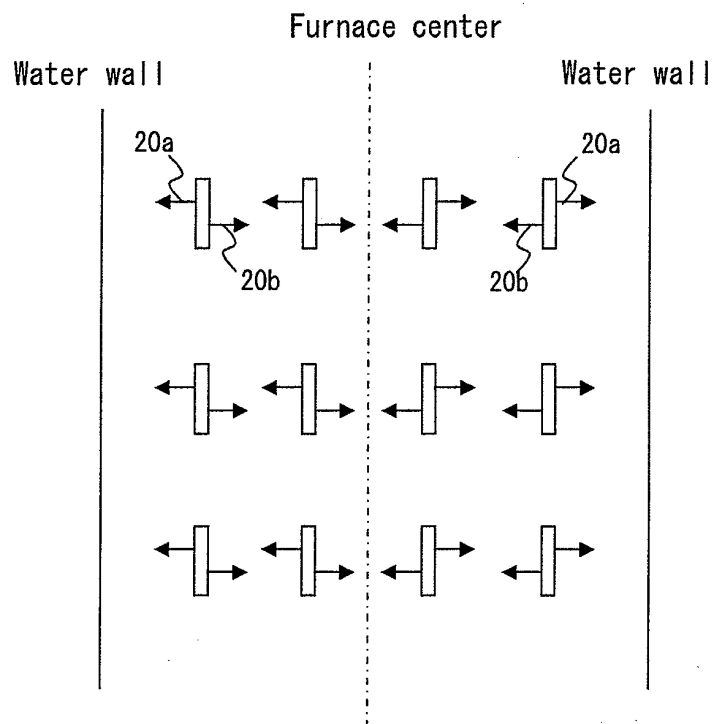


FIG. 24

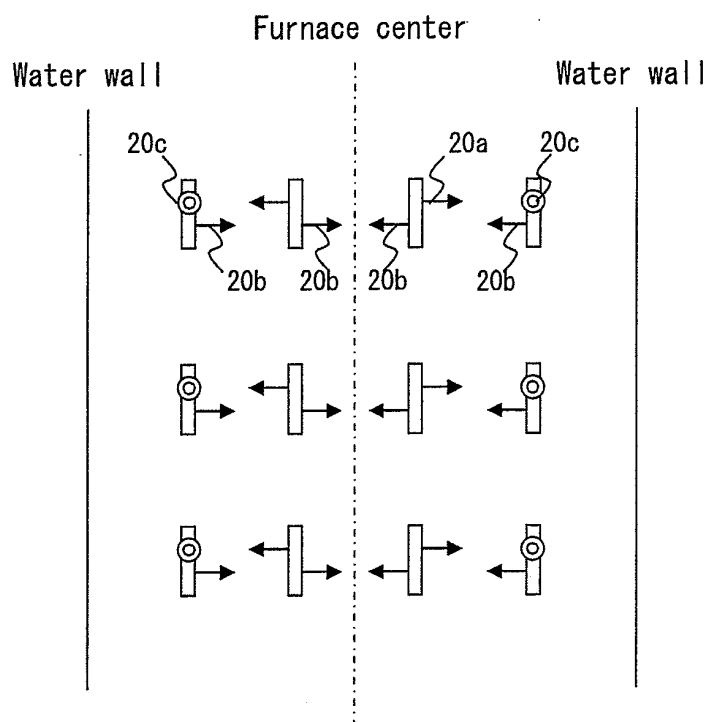


FIG. 25

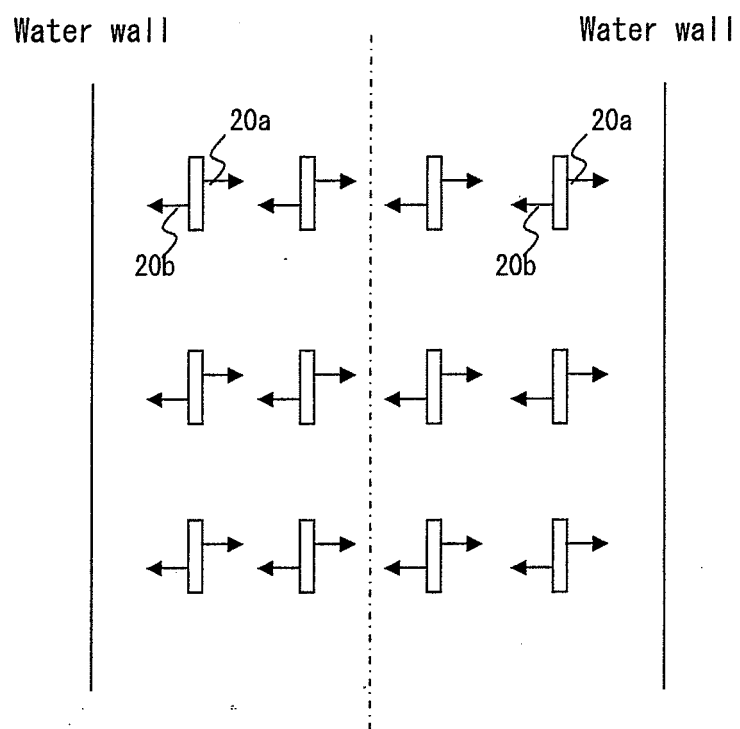
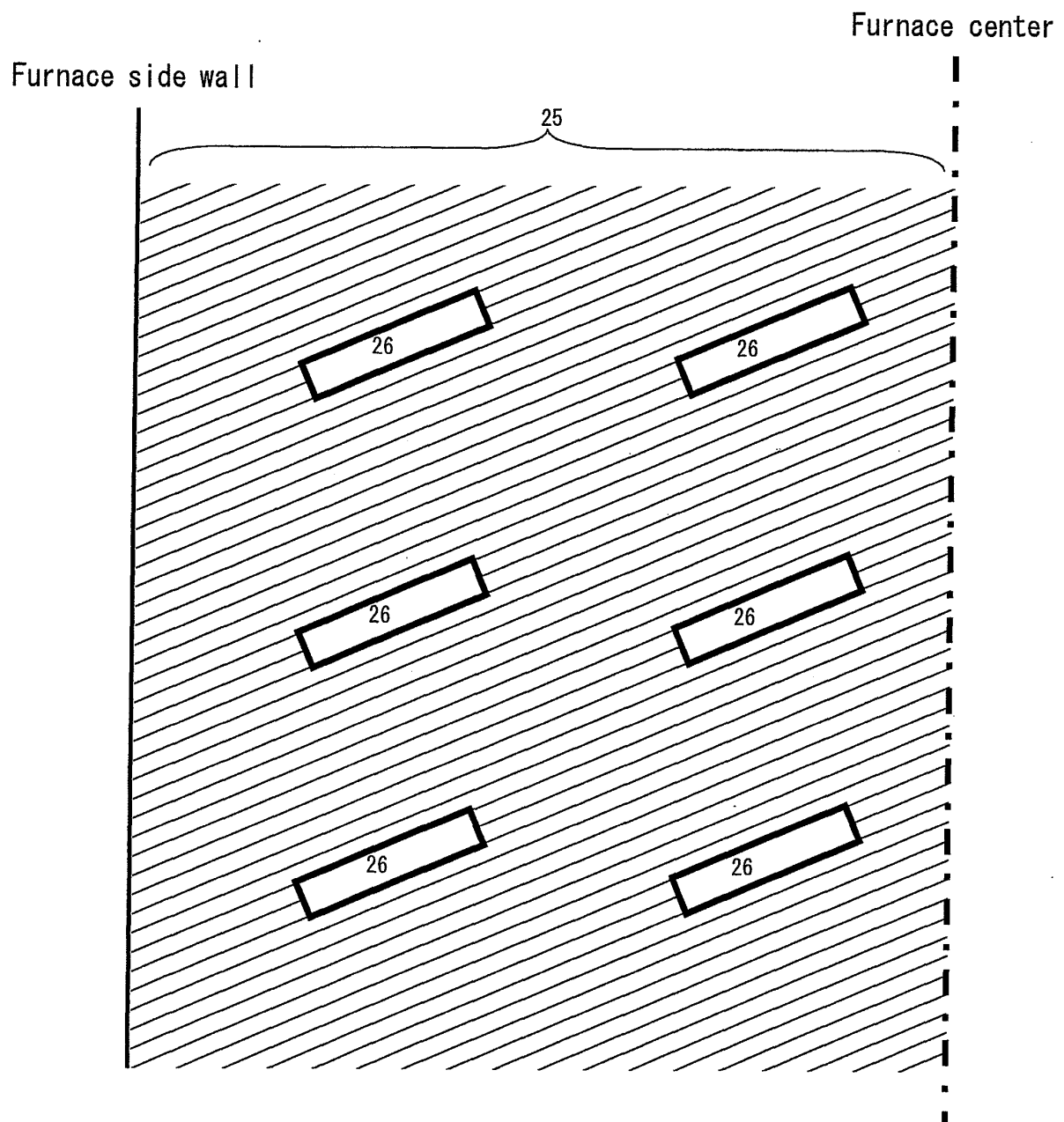
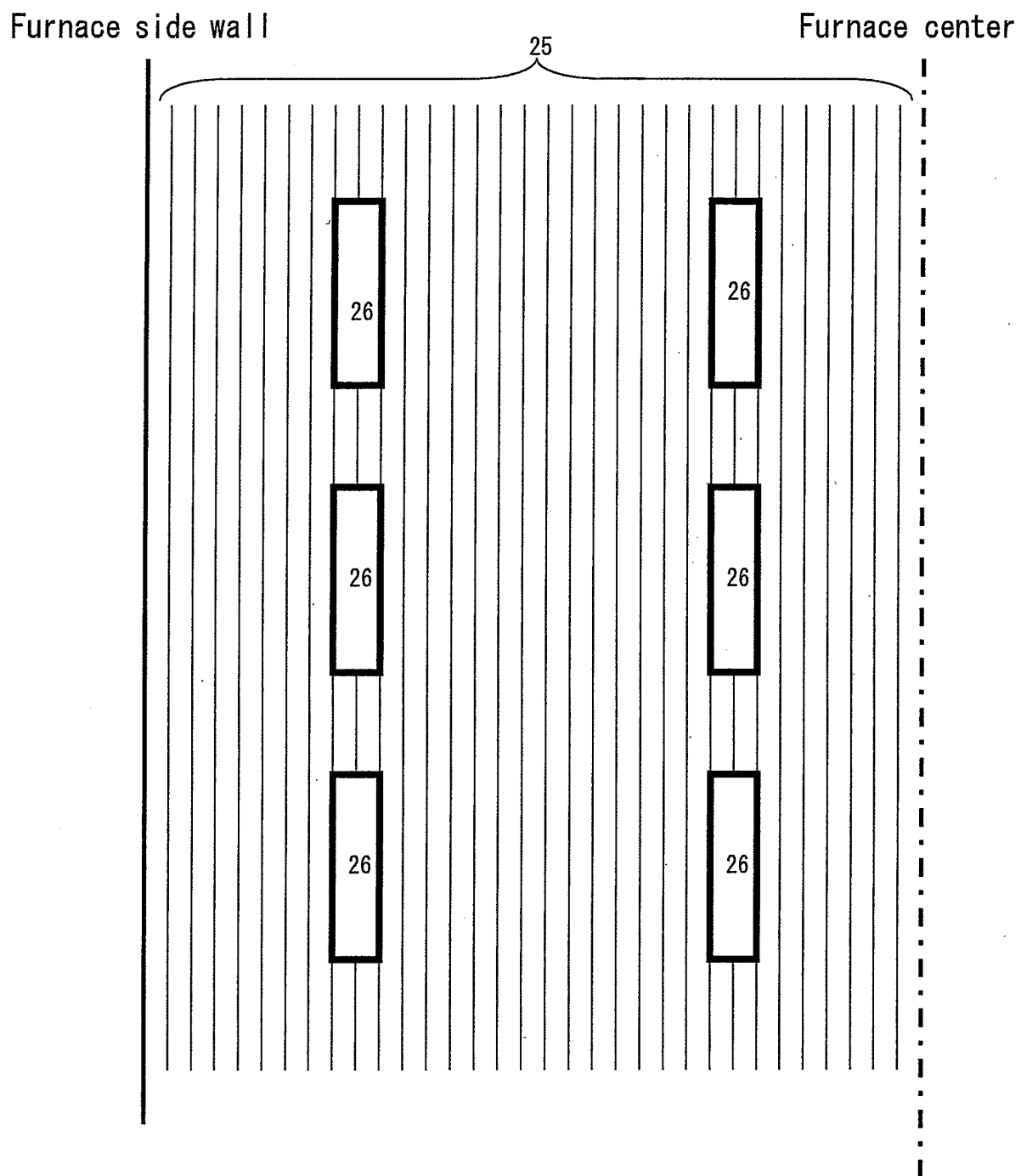


FIG. 26



F I G. 2 7



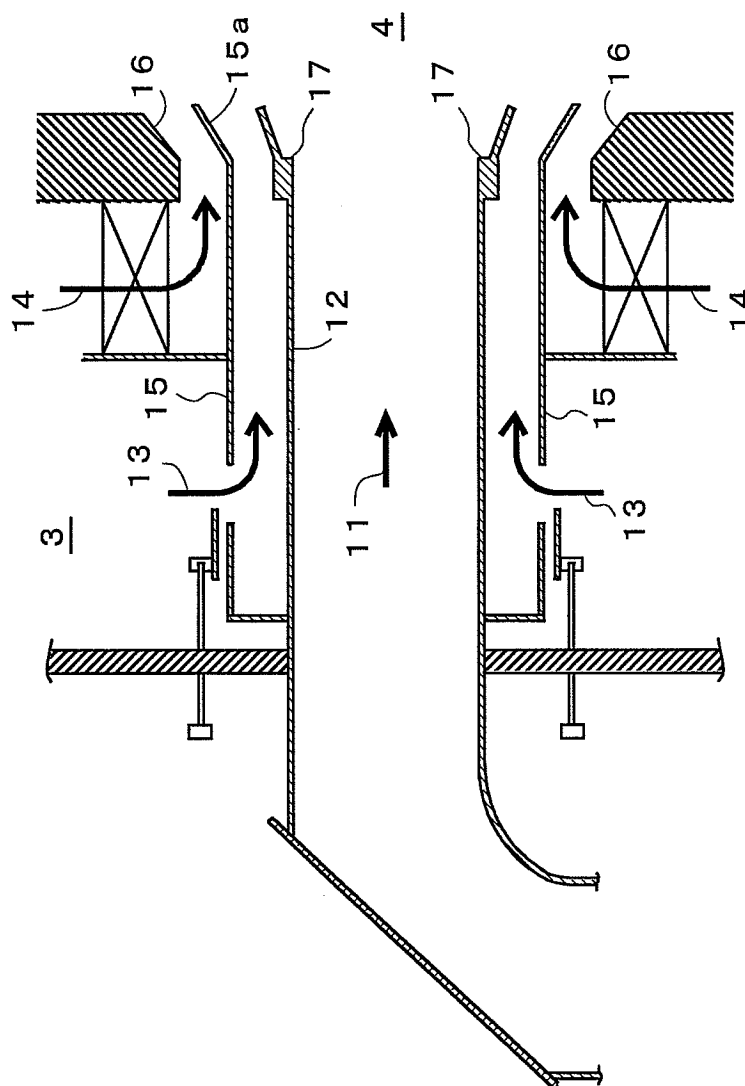


FIG. 28A
PRIOR ART

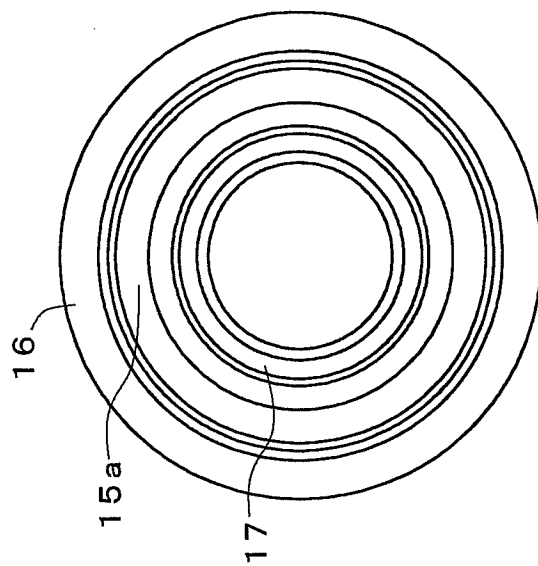


FIG. 28B
PRIOR ART

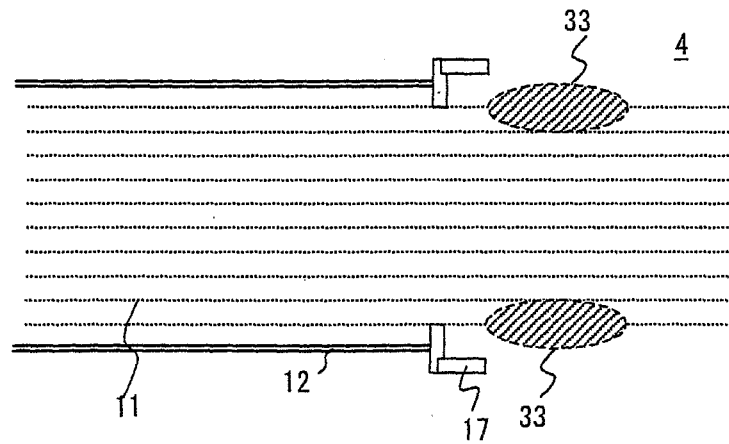


FIG. 29A

PRIOR ART

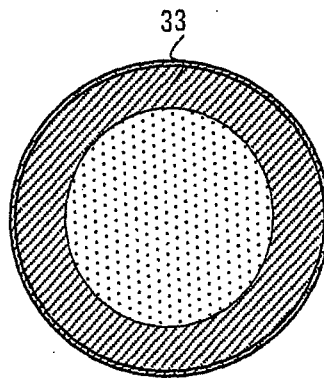
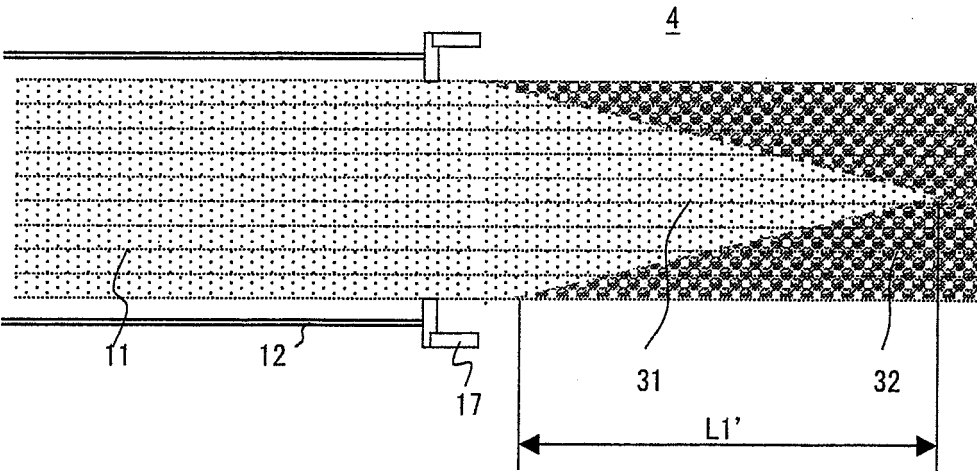


FIG. 29B

PRIOR ART

FIG. 30
PRIOR ART



INTERNATIONAL SEARCH REPORT

International application No.

PCT/JP2007/056311

A. CLASSIFICATION OF SUBJECT MATTER

F23D1/00(2006.01) i, F23C99/00(2006.01) i

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

F23D1/00, F23C99/00

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Jitsuyo Shinan Koho	1922-1996	Jitsuyo Shinan Toroku Koho	1996-2007
Kokai Jitsuyo Shinan Koho	1971-2007	Toroku Jitsuyo Shinan Koho	1994-2007

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X Y	JP 8-200616 A (Mitsubishi Heavy Industries, Ltd.), 06 August, 1996 (06.08.96), Column 2, lines 2 to 3; column 4, lines 1 to 45; Figs. 1 to 4 & US 5829367 A & EP 0687857 A3	1-3, 15 4, 6, 10-14, 16-18, 20
Y	JP 11-287433 A (Mitsubishi Heavy Industries, Ltd.), 19 October, 1999 (19.10.99), Column 1, line 38 to column 2, line 36; Figs. 1 to 11 (Family: none)	4, 6

☒ Further documents are listed in the continuation of Box C.☐ See patent family annex.

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Date of the actual completion of the international search
22 June, 2007 (22.06.07)Date of mailing of the international search report
03 July, 2007 (03.07.07)Name and mailing address of the ISA/
Japanese Patent Office

Authorized officer

Facsimile No.

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INTERNATIONAL SEARCH REPORT

International application No.

PCT/JP2007/056311

C (Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	JP 2000-314508 A (Hitachi, Ltd.), 14 November, 2000 (14.11.00), Column 11, line 31 to column 13, line 18; Figs. 9 to 10 & CN 1271826 A	10
Y	JP 2000-130710 A (Hitachi, Ltd.), 12 May, 2000 (12.05.00), Column 6, line 37 to column 7, line 31; Fig. 1 (Family: none)	11-14, 16
Y	JP 2000-234704 A (Babcock-Hitachi Kabushiki Kaisha), 29 August, 2000 (29.08.00), Column 2, lines 39 to 46; column 5, line 38 to column 6, line 26; Figs. 1 to 7 (Family: none)	17-18
Y	JP 39-11303 B1 (Combustion Engineering, Inc.), 22 June, 1964 (22.06.64), Figs. 1 to 3 (Family: none)	20

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REFERENCES CITED IN THE DESCRIPTION

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Patent documents cited in the description

- JP SHO59500981 B [0007]
- JP HEI8226615 B [0007]
- JP HEI11281009 B [0007]