



(12) **EUROPEAN PATENT APPLICATION**

(43) Date of publication:  
**10.03.2010 Bulletin 2010/10**

(51) Int Cl.:  
**B25C 1/04 (2006.01) B25F 5/00 (2006.01)**

(21) Application number: **08015757.1**

(22) Date of filing: **07.09.2008**

(84) Designated Contracting States:  
**AT BE BG CH CY CZ DE DK EE ES FI FR GB GR HR HU IE IS IT LI LT LU LV MC MT NL NO PL PT RO SE SI SK TR**  
Designated Extension States:  
**AL BA MK RS**

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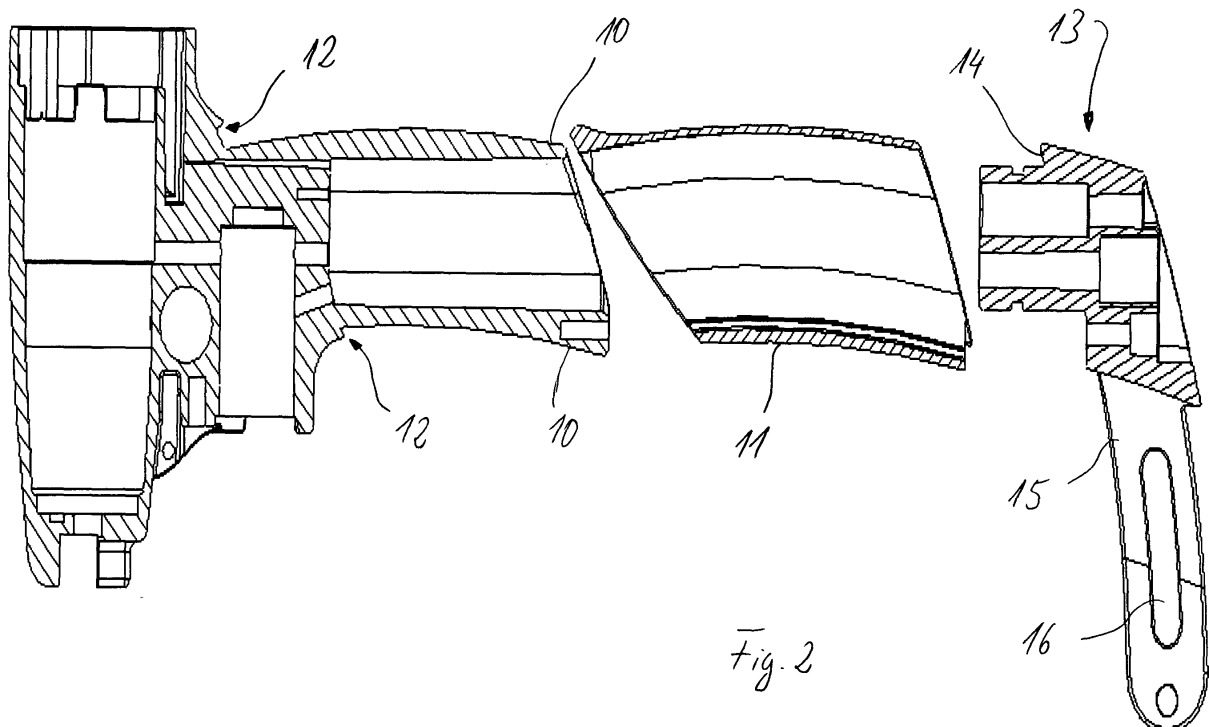
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(54) **Fastener driving tool with a handle portion**

(57) A pneumatic fastener driving tool comprising a housing assembly (2) including a main housing portion (3) and a handle portion (4) with a grip extending from the main housing portion (3), whereas a cylinder (62), a piston (63) which is movable arranged within the cylinder (62), and a fastener driving element (77) which is operatively connected to the piston are included in the main housing portion, said fastener driving element (77) is provided for driving fasteners which are successively feedable into a fastener driving track from a magazine assembly (5). In order to improve ergonomic properties of the handle portion and in order to allow a fast assembling and exchange of the grip the fastener driving tool is provided with a lattice work (10) of said handle portion (4) which is surrounded by a grip. The grip is designed as a flexural resistant circumferentially closed gap-free one piece plastic part.

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## Description

**[0001]** The invention concerns a pneumatic fastener driving tool comprising a housing assembly including a main housing portion and a handle portion with a grip, extending from the main housing portion, whereas a cylinder, a piston which is movable arranged within the cylinder, and a fastener driving element which is operatively connected to the piston are included in the main housing portion, said fastener driving element is provided for driving fasteners which are successively feedable into a fastener driving track from a magazine assembly.

**[0002]** Portable fastener driving tools which use a fluid, particularly compressed air, for accelerating fasteners like staples, headed and headless nails, clamp nails and the like and for driving them into items, are well known. Such fastener driving tools are used for example in the production of furniture for affixing overlays, for example leather covers, at furniture. Such fastener driving tools have a piston which is movable in a cylinder between a top and an upper dead center. A fastener driving element is arranged within the line of motion of the piston and is used for contacting and for ejection the fasteners which are fed into the fastener driving track from a magazine assembly.

**[0003]** Such fastener driving tools are known which are provided with a rubber tape around the metallic handle portion of the housing assembly. The rubber tape has the function to make the handle portion more slip-proof and is provided on one side with an adhesive in order to fix it at the handle portion. Such rubber tapes have to be wrapped in a helical manner around the handle portion. Because those rubber tapes are worn down after a short time of use it is necessary to replace the rubber tape by another rubber tape. Therefore the old rubber tape has to be removed together with its adhesive in a time consuming manner and the new rubber tape has to be wrapped round by round around the handle portion. Also if this is done very carefully the rubber tape often slips onto the handle portion during the use of the tool, which makes it necessary to replace it before it is worn down.

**[0004]** It is also prior known to use a cover-like flexible rubber tub which has no own flexible respective bending resistant. In order to assemble such a flexible rubber tub it is necessary to use water and soap which makes the rubber tub slippery and allows to put it over the handle portion. Since it is also necessary that the rubber tub is rolled before it is put over the handle the assembling of such a rubber tub is complicated. It has shown that those rubber tubs have only a low resistant against damages, which makes it necessary that the rubber tubs are often changed.

**[0005]** Another already known handle portion comprises a plastic part having a parting line which runs essentially parallel to the longitudinal direction of the handle portion and along the whole plastic part. The gap of the shell-like plastic part can be minimized by means of at least two screws. The plastic part should enclose the

metallic handle portion of the housing assembly and a turning of the screws should lead to a surface-to-surface contact of the plastic part with the handle portion. However, it has been found that such plastic parts are difficult to match with the housing assembly. The plastic part can therefore often be moved at least slightly with respect to the housing assembly which makes it difficult to orientate the tool correctly with respect to the item in which a fastener should be driven. It has also been found that the gap can't be closed completely so that the operator can easily scratch himself at sharp edges of plastic part.

**[0006]** It is therefore an object of the present invention to provide a fastener driving tool with a handle portion which has better ergonomic properties and which can be easily assembled to the tool and also easily exchanged in case of damages.

**[0007]** A fastener driving tool as initially mentioned and according to the invention comprises a lattice work of said handle portion which is surrounded by a grip which is designed as a flexural resistant circumferentially closed gap-free one piece plastic part. Such a grip according to the invention is slid on the lattice work of the handle portion and encloses said lattice work by contacting it. Since the plastic grip is provided on the one hand with an own flexural resistance it is inherently stable which allows a simple assembling of the grip to the lattice work. On the other hand the flexibility of a lattice work and the flexibility of the plastic material of the preferably hollow-cylindrical grip allows a form-fit arrangement of the grip onto the lattice work and which allows to avoid the danger of the possibility of a relative movement between the grip and the lattice work.

**[0008]** In connection with the present invention it is preferred that the grip is designed as a hollow-cylindrical one piece plastic part which can easily assembled to the driving tool. Such a design of the grip allows an easy form-fitting assembling by sliding it onto the lattice work from the free end of the handle portion. A preferred embodiment of the invention in which the grip is detachable fastened to the housing assembly allows a fast exchange of the grip in case that grip is damaged during the use of the tool.

**[0009]** In order to fix the grip in a predefined longitudinal position with respect to the handle portion, a locking piece can be arranged in the area of a free end of the handle portion, which secures the grip with respect to longitudinal position onto the lattice work by means of a form closure with the grip. In this connecting it can also be preferred that the grip is clamped on the handle portion between a shoulder of the housing assembly and the locking piece.

**[0010]** It is further preferred that the grip is provided with a hollow-cylindrical shape which is open at its both endings. One open end of the grip is used to push the grip onto the handle part of the housing assembly. The other open end allows to insert the locking piece into the grip and to rest against the grip in order to clamp the grip onto the handle part.

**[0011]** Further aspects and preferred embodiments are disclosed in the following detailed description, drawings and claims. A possible embodiment of the invention will be described, by way of example only, with reference to the accompanying drawings. These drawings show in a schematical illustration:

- Fig. 1 a preferred embodiment of a pneumatic fastener driving tool in a cross-section view;
- Fig. 2 housing parts of the fastener driving tool of fig. 1 in a cross-section view;
- Fig. 2a the housing parts of fig. 2 in a partly assembled situation;
- Fig. 2b a cross-sectional view according to the line II - II of fig. 2a;
- Fig. 3 said housing parts of fig. 2 in a perspective view;
- Fig. 4 a trigger assembly of the tool of fig. 1 in an enlarged illustration;
- Fig. 5 a housing part of the tool together with means of an single and automatic fire control in an exploded view;
- Fig. 6 a housing part of the tool together with means of a single and automatic fire control in a second arrangement;
- Fig. 7 the means of the single and automatic fire control in the single fire modus shown in a crossection view;
- Fig. 8 the means of the single and automatic fire control in the automatic fire modus shown in a crossection view;
- Fig. 9 a detail of fig. 1;
- Fig. 10-12 a power adjustment unit of the tool of fig. 1 in different modes;
- Fig. 13 a cross-section view through a cylinder and the single and automatic fire control;
- Fig. 14 a cross-section view through the trigger assembly and the single and automatic fire control;
- Fig. 15 a muffler means in a perspective view;
- Fig. 16 the muffler means in a cross-section view;
- Fig. 17 a detail of fig. 1 concerning the muffler

means.

**[0012]** Fig. 1 shows a portable pneumatically actuated fastener driving tool 1 having a housing assembly 2. The housing assembly 2 has a main housing portion 3 and a handle part 4, to which a hollow magazine assembly 5 for receiving a supply of fastener is connected. The handle part 4 and the magazine 5 assembly are aligned essentially parallel to each other, whereby the main housing portion 3 is orientated essentially perpendicular with respect to the handle part 4. The handle part 4 contains a connection 6 for an air supply.

**[0013]** The handle part 4 is provided with a kind of lattice work 10 which is build by the metallic housing in order to provide the handle part 4 with a good torsional and bending strength (Fig. 2). As shown in Fig. 2, 2a, 2b and 3 a hollow one piece plastic part 11 which is build us an adapting sleeve and is slid onto an over the lattice work 10 of the handle part 4. As can be seen in Fig. 2b the lattice work 10 has an outer shape particularly with an extension 10a which is provided with edges, which allows with respect to the radial position of the hollow plastic part 11 a form-locking arrangement onto the the lattice work 10. The tub-shaped plastic part 11 can be manufactured as an injection molding part out of an appropriate plastic material, for example TPE (thermoplastic elastomer), PUR (polyurethane) or EPDM (ethylene propylene diene M-class rubber). Such materials are particularly in the form of a hollow-cylindrical gap-free part with respect to its mechanical stability strength as plastic although it is soft as rubber with respect to its ability to adapt its shape in a certain amount to the shape of handle part of the metallic housing. The plastic part 11 should fit on the handle part 4 of the metallic housing preferably without clearance and can be fastened to the handle part with only one screw. As shown in Fig. 1, 2 and 2a in the transition area between the handle part 4 and the main housing portion 3 the metallic housing can be provided with a shoulder 12 resp. border, which in circumferential direction of the handle part at least partly surrounds the handle part 4. The plastic part 11 can rest with its front side against said shoulder 12 in order to allow a predefined position of the plastic part 11 onto the handle part. At the other end of the plastic part a locking piece 13 is partly inserted into the free end of the handle part 4 and of the plastic part 11. The locking piece 13 is also provided with a shoulder 14 and rests with said shoulder 14 against the handle part 4 and plastic part 11. The plastic part 11 is therefore clamped between these two shoulders 12, 14. As can be seen in Fig. 1 - 3 the locking piece 13 is provided with a lug 15 which is used to fasten the magazine assembly 5 at the housing.

**[0014]** A manually actuatable trigger assembly 20 is arranged in an intersection part between the handle part 4 and the main housing portion 3. The trigger assembly 20 can be actuated by a user of the tool by means of a trigger 21. A trigger valve 22 acts as a barrier between a pressure reservoir 25 for compressed air (Fig. 1), and

a main valve 26 of the tool. The pressure reservoir 25 can be connected by means of a hose (not shown in the drawings) to an external supply of compressed air. A trigger assembly 20 connects the pressure reservoir 25 with a supply channel 28 which ends in a lower area of the main valve 26. The pressure reservoir is also connected via a connecting channel 27 with the main valve 26. The connecting channel leads to an upper area of the main valve which is sealed against the lower area. By actuating the trigger assembly 20 the supply channel 28 is disconnected from the pressure reservoir 25. As long as the trigger is not actuated the trigger assembly connects the pressure reservoir 25 with a supply channel 28.

**[0015]** An assembly group of a means for single fire control and of a means for automatic frequency fire control 40 (hereinafter called "SFC AFC") is arranged in flow direction of the compressed air behind the trigger assembly 20. The assembly group is arranged in the handle part 4 of the housing assembly in a through hole 41 and is provided with a single fire control means 42, a frequency control means 43 and a valve 44, which are inserted at the same side of the housing (Fig. 5). At the opposite side of the housing a throttle valve 45 of the SFC AFC is inserted in the housing. The actuatable single fire control means 42 has the function to allow a limitation of the amounts of shots of fasteners by actuating the trigger to only one shot. Contrary to that, the frequency control means 43 has the function to allow an adjustment of the frequency in a certain range of frequencies with which a plurality of fasteners are successively shoot out of the tool in case that the single fire control is not actuated. A typical adjustable range of frequencies are from 0 to 1800 fasteners per minute. In order to adjust the fastener driving tool to a single shot it is necessary to push the single fire control means 42 inside the frequency control means 43. Simultaneously the function of the frequency control means 43 is switched off. In case that the single fire control is not actuated the frequency of the fasteners which are shoot out of the tool can be adjusted by revolving the frequency control means 43 at the knurl 46, which is a one-piece part of the frequency control means.

**[0016]** In order to allow right-handed persons as well as left-handed persons an ergonomic use of the fastener driving tool it is possible to interchange the sides on which the throttle valve 45 on one hand and the single fire control means 42, the frequency control means 43 and the valve 44 on the other hand are arranged in the fastener driving tool. The possible two different configurations of the SFC AFC are shown in the exploded views of Fig. 5 and 6 as well as in the intersection views of the Fig. 7 and 8. For this adaptation according to the preferences of left- or right-handed persons it is necessary to remove the throttle valve 45 and also the single fire control means 42, the frequency control means 43 and the valve 44 from the housing. These parts can be removed from the handle part 4 as assembled. Then the throttle valve 45 is inserted at the other side of the housing where the single fire control 42 means, the frequency control means 43 and the

valve 44 have been arranged before. Furthermore the single fire control means 42, the frequency control means 43 and the valve 44 are inserted at the opposite side of the housing where the throttle valve 45 has been arranged before. As soon as the parts of the SFC AFC are interchanged and connected to each other in the subsequent described manner the SFC AFC is ready for use.

**[0017]** The throttle valve 45 is provided with a conical plug 50 which interacts with an also conical valve seat 51 of the frequency control means 43. By means of varying the penetration depth of the plug 50 into the opening of the valve seat 51 the restriction between the valve seat 51 and the plug 50 can be varied. As can be seen in Fig. 7 and 8 the frequency control means 43 is provided at one of its ends with an outer thread which can be screwed into an inner thread of the valve 44, whereby the valve 44 can be screwed with its outer thread into an inner thread of the throttle valve 45. The inner thread of the throttle valve 45 is situated at the inner surface of a hollow cylindrical part of the throttle valve 45 which surrounds the plug 50 with distance. The wall of the hollow cylindrical part of the throttle valve is provided with openings 52 through which air can flow in direction to the supply channel 28 via the trigger assembly 20.

**[0018]** At the other end of the frequency control means 43 the pin-shaped single fire control means 42 is coaxially inserted into the hollow cylindrical frequency control means 43. The single fire control means 42 can be displaced in the frequency control means in longitudinal direction between two end positions. In one of these two end positions the front end of the single fire control closes openings 53 in the wall of the frequency control means 43 so that the passageway of the frequency control means is closed for air coming through openings 52 of the throttle valve and through the supply channel 28. This position of the single fire control means is shown in Fig. 7, whereby the flow direction of the air is illustrated by dotted lines 57. This is the position in which the single fire control is switched on.

**[0019]** The other possible end-position of the single fire control means 42 is shown in Fig. 8. towards the throttle valve 45 and its restriction. In its other position the single fire control means is partly retreated in the frequency control means so that the passageway of the frequency control means is free for air which flows from the openings 54, 53 through the wall of the frequency control means 43 towards the throttle valve 45 and its restriction and openings 52.

**[0020]** A cage 60, which is part of the main valve 26 can be moved between an upper and a lower position. In Fig. 9 the cage 60 is shown in its lower position. The cage 60 is arranged in flow-direction behind the throttle valve 45 of the SFC AFC and at the end of the supply channel 28. The cage 60 is also arranged at the end of the connecting channel 27. In its upper position the cage closes openings 65 which acts as passageways and connects the connecting channel 27 with the power adjustment unit 61, whereby in its lower position the cage opens

the openings 65 for a flow-through. The cage 60 surrounds the upper part of a cylinder 62 which contains a piston 63 and surrounds also the lower part of a valve seating 64 of the main valve.

**[0021]** If the trigger 21 is not activated the cage 60 is loaded with the hose pressure at its lower front side, the cage is held in its upper position (Fig. 1 and 10). In said upper position the cage seals openings 65 which lead to the power adjustment unit 61. By actuating the trigger 21 the air pressure under the cage is taken away and the cage 60 is moving down in direction towards its lower position, which is shown in Fig. 9 and 11.

**[0022]** Now the cage releases the openings 65 and air can flow now to a throttle valve 70 of the power adjusting assembly.

**[0023]** The throttle valve 70, which is part of the power adjustment unit 61 is shown in Fig. 10 - 12. The valve 70 includes a screw 72 which is screwed in a hollow cylindrical part 76 which is provided with a inner thread and which is fixed at main housing portion 3 of the housing assembly. The screw 72 can be moved up or down by revolving an actuating part 71 in one of the two rotating directions. The screw 72 acts as a stop for the valve body 74. A helical spring 73 is arranged between the actuating part 71 and the valve body 74 of the throttle valve 70. The force of the spring 73 presses the valve body 74 with its cone-end onto the valve seating 64. As deeper the screw 72 is screwed into the main housing as shorter is the length of the maximum lifting movement of the valve body. In order to get a high resolution of the adjustable position of the the screw 72, the screw 72 is provided with a relatively low pitch and it can be adjusted over several revolutions.

**[0024]** The valve body 74 is provided with an outer inclined peripherally surface 75. A normal of said surface has a component parallel to the direction of the spring force. The air pressure acting against said surface 75 leads therefore to a lifting of the valve body 74 from the valve seating 64 in case that the air pressure is high enough. This lifting movement proceeds until the valve body 74 reaches the screw and gets into contact with the screw 72 which stops the movement of the valve body 74. By means of the possibility to adjust via the position of the screw 72 it is possible to vary for each cycle the maximum restriction of the throttle valve and therefore the possible discharge flow-through which can reach in flow-direction the piston and affects pressure onto a surface of the piston. The pressure in the piston chamber accelerates than the piston 63 downwards in its cylinder 62 from its top dead center (Fig. 10 - 12) to the direction of its bottom dead center (Fig. 9). Near at the end of this movement a fastener driving element 77 (Fig. 1) which is operatively connected the piston 63 acts with its contact surface onto a fastener which will therefore be accelerated and shot out of the tool 1 in order to be driven into a subject. Than the next fastener is fed from the magazine assembly 5 for fasteners into the fastener driving track, which is the line of motion of the fastener driving element

77.

**[0025]** As soon as the piston reaches the area of its bottom dead center (Fig. 9) the air can leave the cylinder 62 through an orifice 78 in the wall of the cylinder 62 near the position of the piston bottom dead center, as indicated by a dotted arrow 79. The air flows from there into a return chamber 80, which is arranged as a hollow cylinder around the cylinder 62 of the piston 63. This enables to vent the piston chamber of the cylinder 62. The lower pressure in the piston chamber supports to fastening the movement of the piston back at its return stroke to its top dead center. Additionally the return chamber acts as a pressure chamber for the return stroke.

**[0026]** As can be seen in Fig. 13 and is indicated by dotted line 81 the high pressure air flows from the return chamber 80 with direction to the frequency control means 43. Before the air reaches the frequency control means 43 it has to flow through the restriction of the throttle valve 45 of SCAFC. The time it takes for the air to flow through the restriction and to fill the chamber of the frequency control means 43 depends from the size of the restriction. The size of the restriction is adjustable by revolving the throttle valve 45, which therefore influences the frequency with which the tool 1 ejects fasteners. Behind the frequency control means the air flows into the trigger valve 22. When the pressure is high enough under the automatic cage, the automatic cage moves to its upper position. Because in Fig. 8 the SCAFC is assembled for right handed persons and in Fig. 13 the SCAFC is assembled for left handed persons in these two illustrations are shown two different flow directions of the air as a result of the two different assembly possibilities.

**[0027]** Then the air flows through the trigger valve under a cage 85 of the trigger valve 22 and moves it to its upper position (Fig. 14). Also the piston 63 returns to its upper position. When the piston is at its upper position the pressure in the return chamber 80 flows out. When the piston 63 leaves the bottom the air can flow out through a guide of the fastener driving element 77. The pressure can therefore be lowered in two ways, namely leaving through the guide of the fastener driving element 77 and by expanding when it pushes the piston upwards. When the return pressure is gone, the (automatic) cage 60 moves to its lowest position, because at the same time the trigger valve 20 closes supply channel 28 which takes the pressure from the lower side of the cage 60. This activates the cage 60 of the main valve and it moves to its lowest position. Air flows now into the cylinder and starts a new stroke. The piston strokes down and stays at its lowest position. Now, the higher pressure in the return chamber supports to bring the piston back to its upper position. This will go on as long as the trigger is activated. If the trigger is not activated the air stops inside the trigger valve, which results in an automatic fire and tool stop. Without de-activating the trigger there will be no automatic fire and tool stop.

**[0028]** For the modus of "single fire" the initial situation is the same as shown in Fig. 9. In order to start a single

fire shot the single fire control means 42 has to be activated which means that the single fire control means 42 has to be pushed inside the bushing of frequency control means 43 - as shown in Fig. 7 - which closes the passage way from the return chamber 80 to the cage 85. Therefore the air which comes from the return chamber 80 remains in the frequency control means 43. No air can lift the automatic cage 85. In order to cause a new shot, first the trigger 21 has to be deactivated and then the trigger 21 has to be activated again.

**[0029]** As shown in Fig. 1 a tubular muffler means 90 is arranged in the handle part 4 of the tool 1. By means of the muffler means 90, which shown in more detail in Figs. 15 - 17, the noise of the compressed air can be reduced which exhaust the fastener driving tool 1 after its flow through the tool. The muffler means 90 has two parts, a tubular passage way pipe 91 and a tubular muffler pipe 92. The passage way pipe 91 and the muffler pipe 92 are preferably connected to each other as one piece part. The muffler pipe 92 is provided in its ring-shaped wall 93 with a plurality of geometrically predefined recesses 94. With respect to a certain length of the muffler pipe 92 and also with respect to its circumferential direction the recesses 94 are equally allocated at the muffler pipe. In the preferred embodiment the recesses 94 have a circular cross-sectional form, whereby the diameter of the recesses should be as big as possible. The size of the recesses is limited by the circumstance that too large recesses do not reduce the noise anymore. In a preferred embodiment the diameters of the recesses can have a value out of a range from 1 mm to 4 mm. In the embodiment shown in Fig. 17 the recesses have a diameter of 1,8mm. In case that non-circular recesses are used the largest aperture of those non-circular openings can also be out of this range of values. The front wall of the tubular muffler pipe is closed.

**[0030]** As can be seen in Fig. 1 and 17 the muffler pipe 92 is arranged in a tubular hole 95 of the locking piece 13. The muffler means 90 is provided at its outer peripheral surface with a plurality of ring-shaped extensions 96, with which the muffler means 90 rests again the inner surface of the tubular hole 95 of the locking piece. In the space between two extensions a sealing can be arranged, for example an o-ring. The outer peripheral surface of the muffler pipe 92 is arranged with clearance to the inner peripheral surface of the hole 95. The compressed air, which enter with high energy, particularly in form of a high velocity, the muffler means 90 flows through the passage way pipe 91 and will be therefore guided to the free end of the handle part 4. In the muffler pipe 92 the compressed air strikes against the closed front wall 97 of the muffler pipe 92. The compressed air has therefore the tendency to be retained within the muffler pipe 92, which means the air loses velocity. In the following the air can exit the muffler pipe through the plurality of recesses 94 in the wall of the muffler pipe as a result of a deflection of its flow direction. The air, which is now located in the hollow cylindrical area between the

outer surface of the muffler pipe 92 and the surface of the hole 95 can leave the handle part through the open end of the hole 95.

## 5 Reference numbers

### [0031]

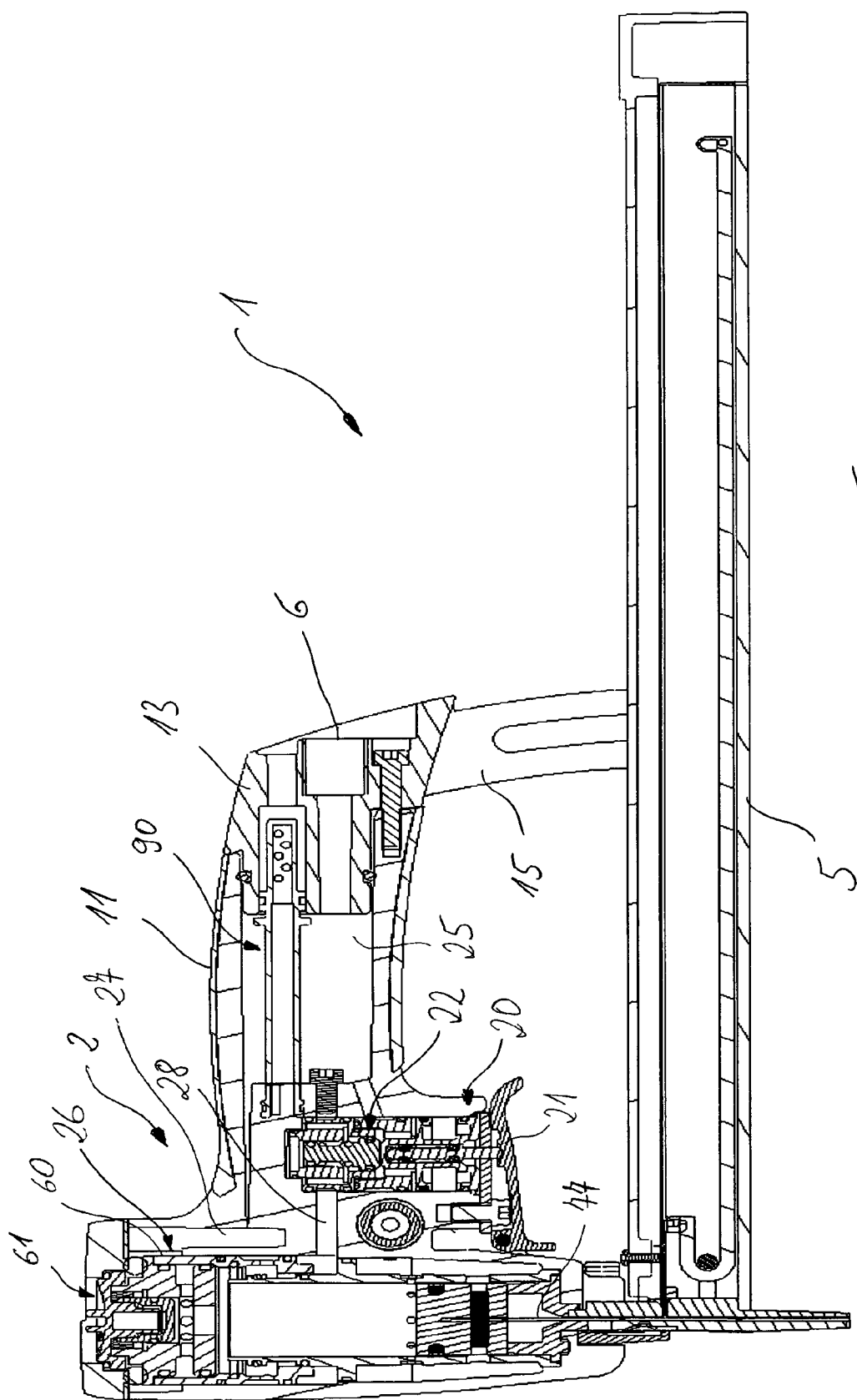
1	Fastener driving tool
2	Housing assembly
3	Main housing portion
4	Handle part
5	Magazine assembly
6	Connection
10	Lattice work
11	Plastic part
12	Shoulder
13	Locking piece
14	Shoulder
15	Lug
16	
20	Trigger assembly
21	Trigger
22	Trigger valve
25	Pressure reservoir
26	Main valve
27	Connecting channel
28	Supply channel
40	Single fire and automatic fire control
30	Through hole
42	Single fire control means
43	Frequency control means
44	Valve
45	Throttle valve
35	Knurl
50	Conical plug
51	Conical valve seat
52	Opening
53	Opening
40	Opening
54	Dotted line
57	Cage
60	Lower front side
60a	Power adjustment unit
45	Cylinder
61	Piston
62	Valve seating
63	opening
64	Throttle valve
65	Actuating part
50	Screw
70	Spring
71	Valve body
72	Surface
73	Hollow cylindrical part
74	Fastener driving element
75	Orifice
76	
77	
78	

79 Arrow  
 80 Return chamber  
 81 Dotted line  
 85 Cage  
 90 Muffler Means  
 91 Passage way pipe  
 92 Muffler pipe  
 93 Wall  
 94 Recess  
 95 hole  
 96 Extension  
 97 Front wall

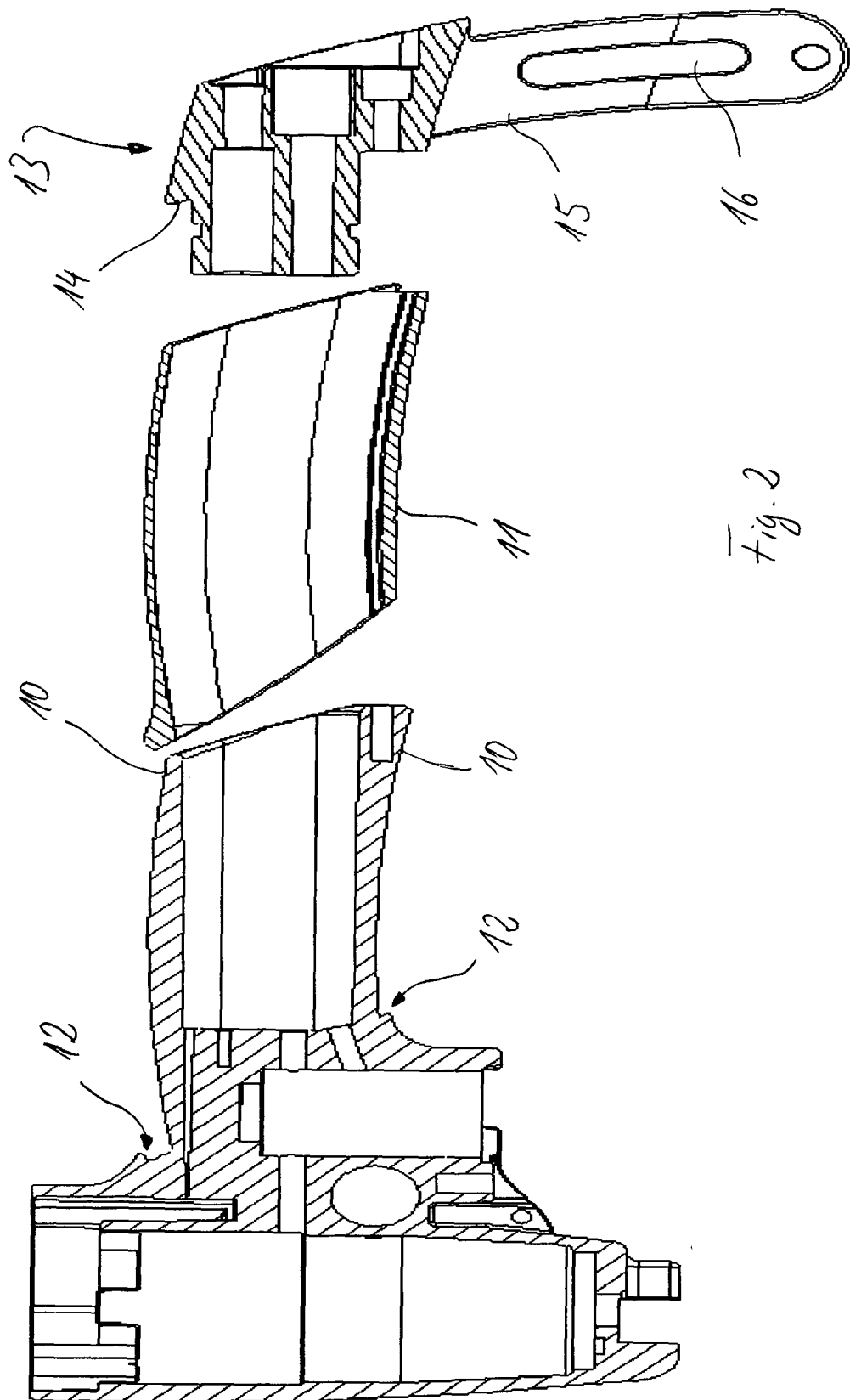
## Claims

1. A pneumatic fastener driving tool comprising a housing assembly (2) including a main housing portion (3) and a handle portion (4) with a grip extending from the main housing portion (3), whereas a cylinder (62), a piston (63) which is movable arranged within the cylinder (62), and a fastener driving element (77) which is operatively connected to the piston are included in the main housing portion, said fastener driving element (77) is provided for driving fasteners which are successively feedable into a fastener driving track from a magazine assembly (5), **characterized in that** a lattice work (10) of said handle portion (4) is surrounded by a grip which is designed as a circumferentially closed gap-free one piece plastic part. 20
2. A pneumatic fastener driving tool according to claim 1, **characterized in that** the grip is designed as hollow-cylindrical one piece plastic part. 35
3. A pneumatic fastener driving tool according to claim 1 or 2, **characterized in that** the grip is detachable fastened to the housing assembly. 40
4. A pneumatic fastener driving tool according to one or more to the preceding claims, **characterized in that** the one piece plastic part is at its both front sides open. 45
5. A pneumatic fastener driving tool according to one or more to the preceding claims, **characterized in that** in the area of a free end of the handle portion (4) a locking piece (13) is arranged, which secures the grip onto the lattice work (10) by means of a form closure with the grip. 50
6. A pneumatic fastener driving tool according to one or more to the preceding claims, **characterized in that** the grip is clamped on the handle portion between a shoulder of the housing assembly and between the locking piece. 55

7. A pneumatic fastener driving tool according to claim 5 **characterized** that the locking piece (13) is inserted into the grip and that the locking piece is provided with a shoulder which is in contact with a front face of the grip. 5
8. A pneumatic fastener driving toll according to one or more of the preceding claims 4 to 6, **characterized by** a muffler, which is arranged in the locking piece (13). 10
9. A pneumatic fastener driving toll according to one or more of the preceding claims, **characterized by** a single fire control means for switching the fastener driving tool into a single fire mode in which the fastener driving tool performs for each actuation of the tool, particularly the trigger (21), only one stroke of the piston in order to eject only one fastener and an automatic fire frequency control means for adjusting in another mode than the single fire mode a frequency of the fastener driving tool with which the fastener driving tool performs for each actuation of the tool, particularly of the trigger (21), a plurality of strokes in order to eject successively a plurality of strokes, whereas the grip is provided with at least two holes in which the single fire control means and the automatic fire frequency control means are exchangeable arranged. 15







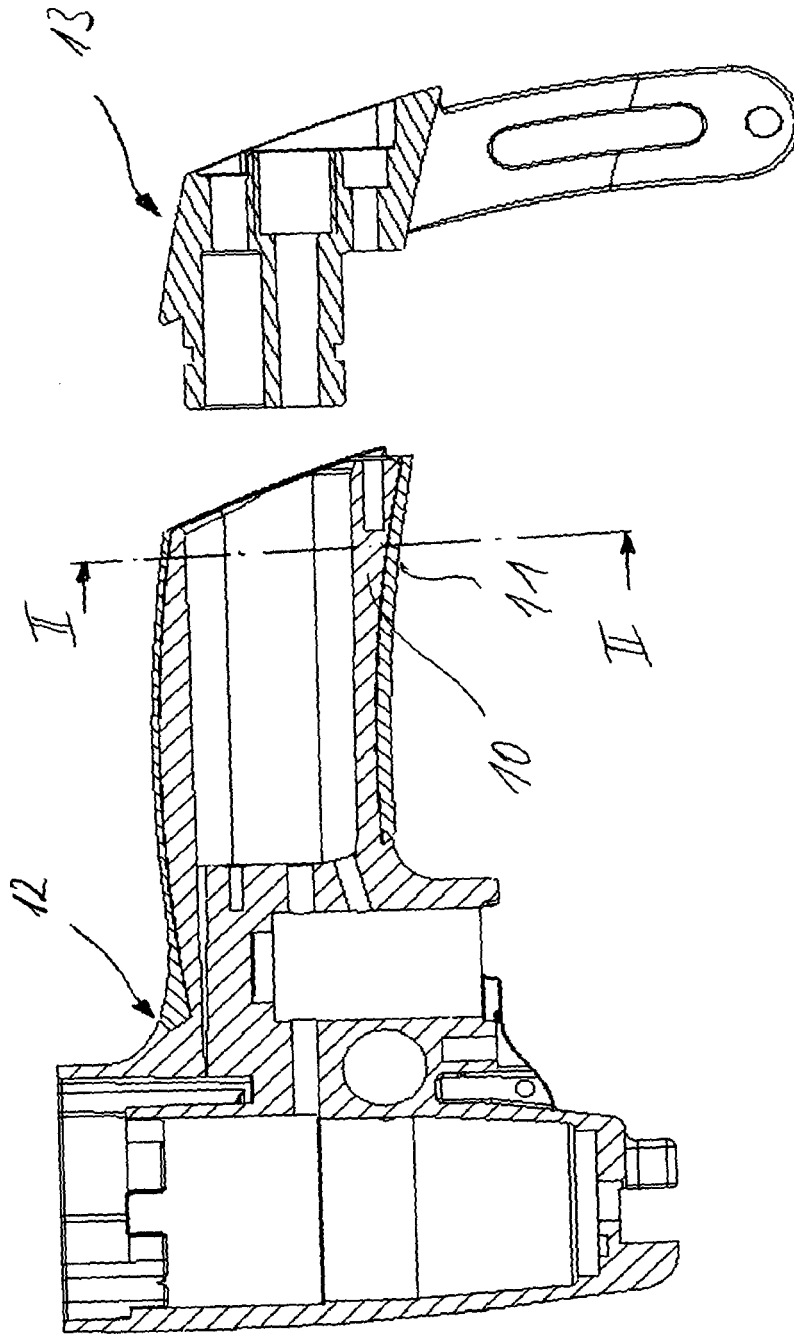


Fig. 2a

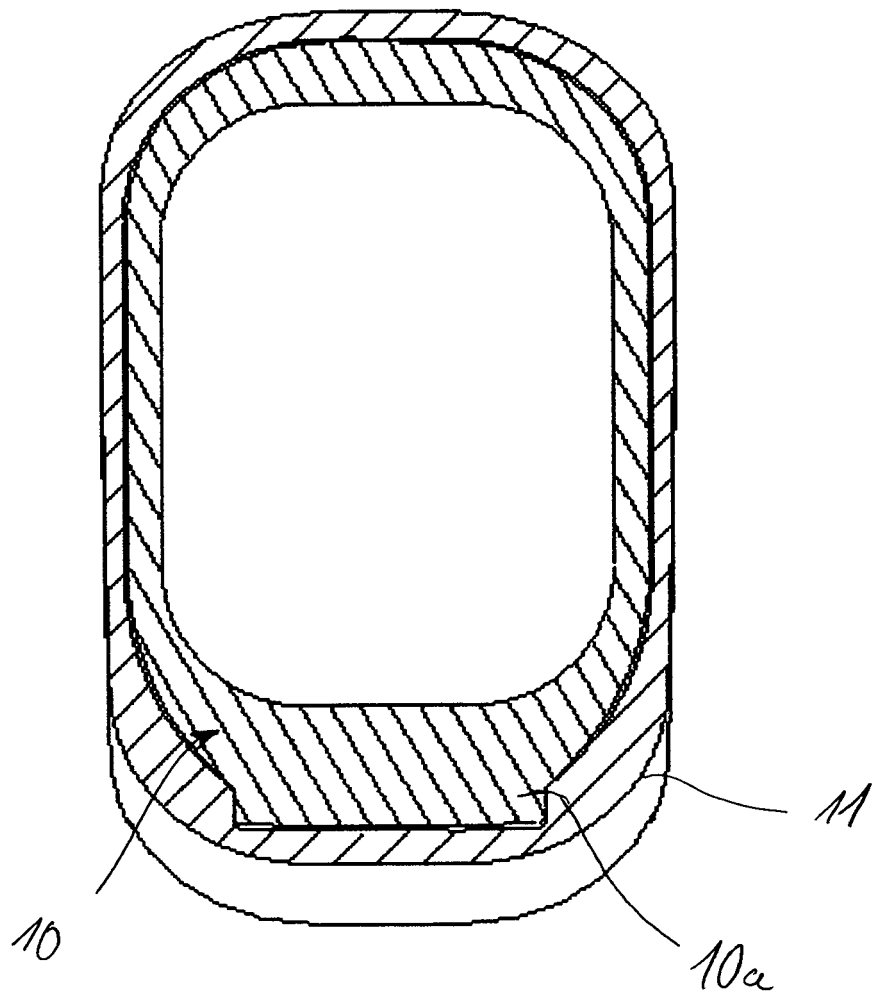
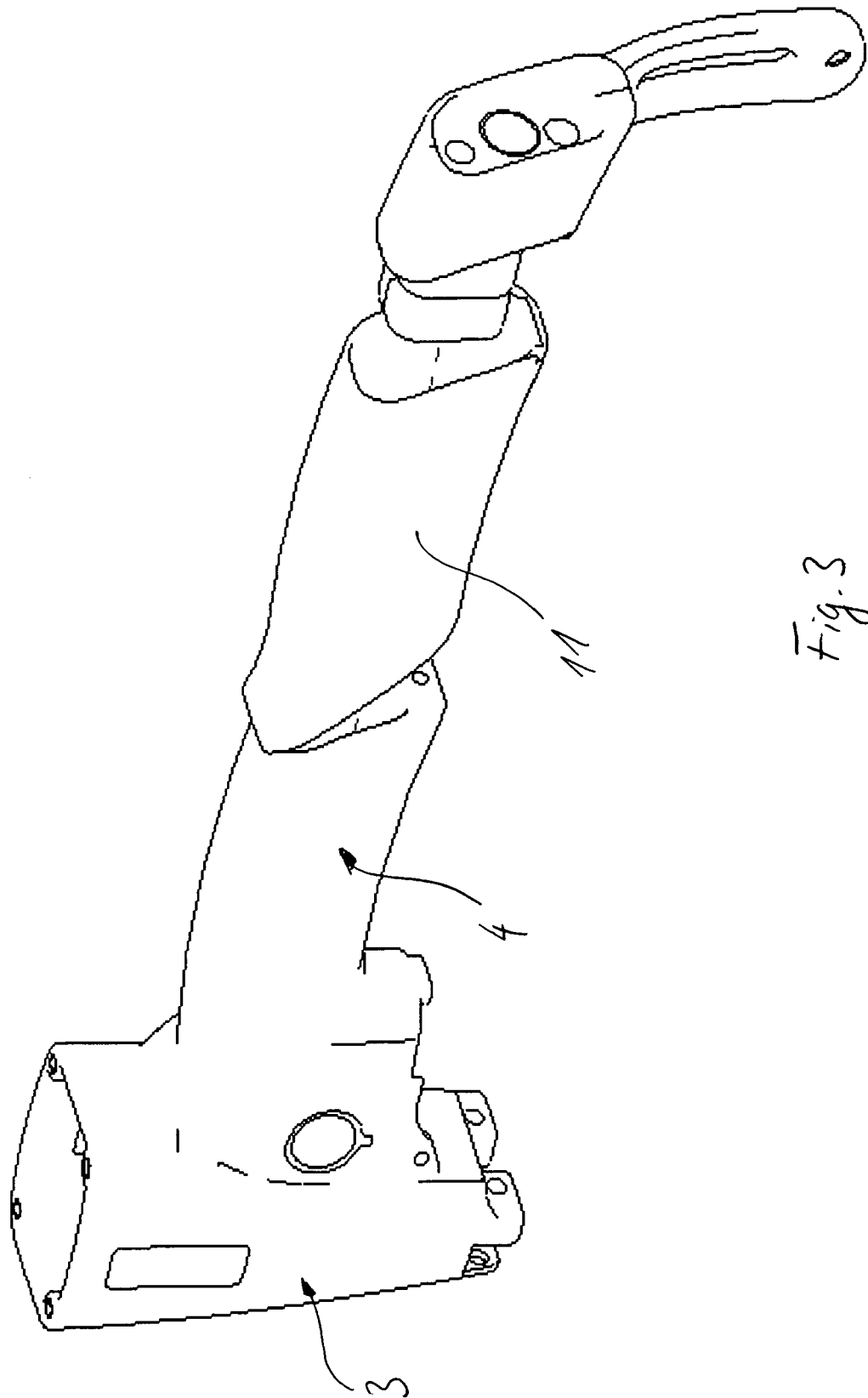


Fig. 26



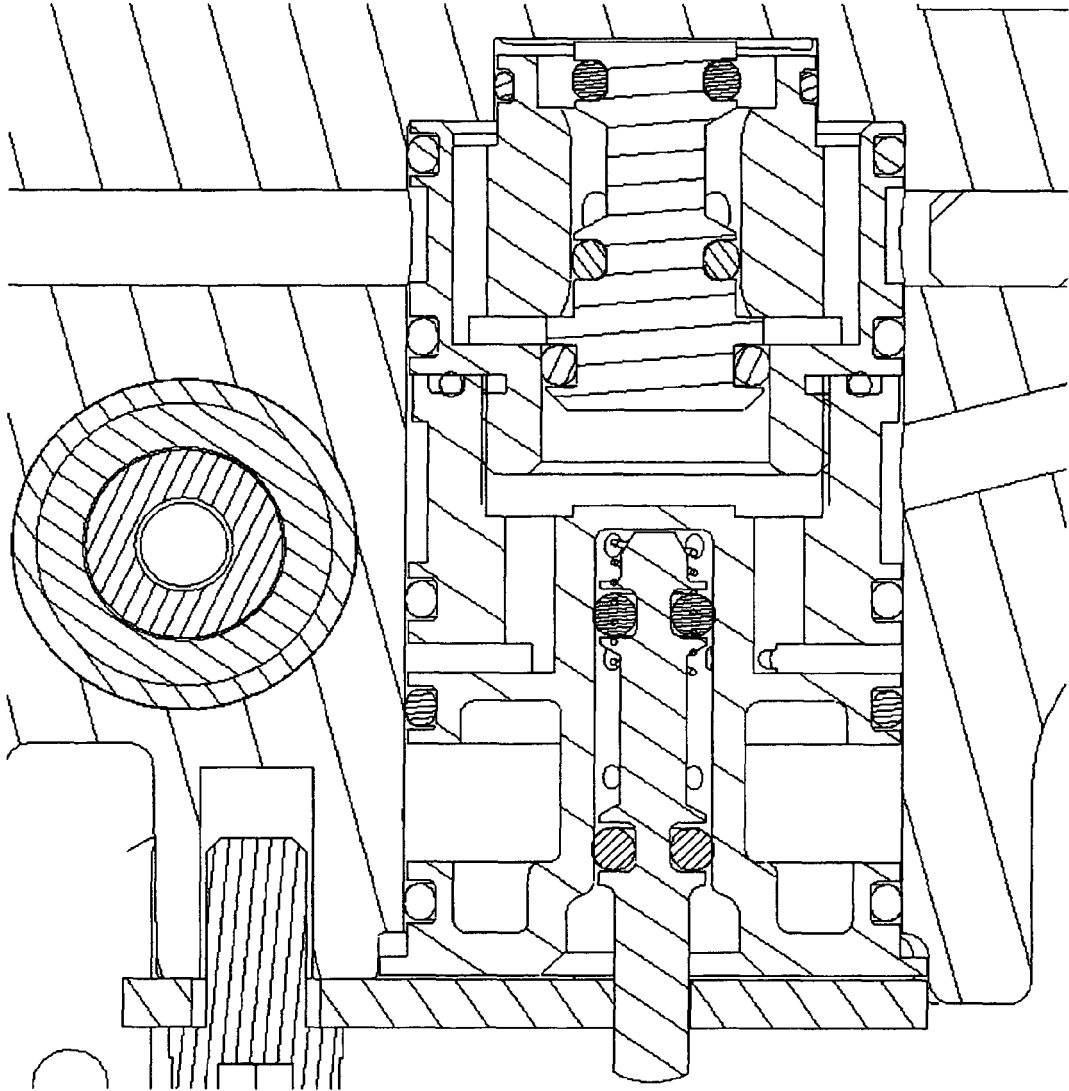
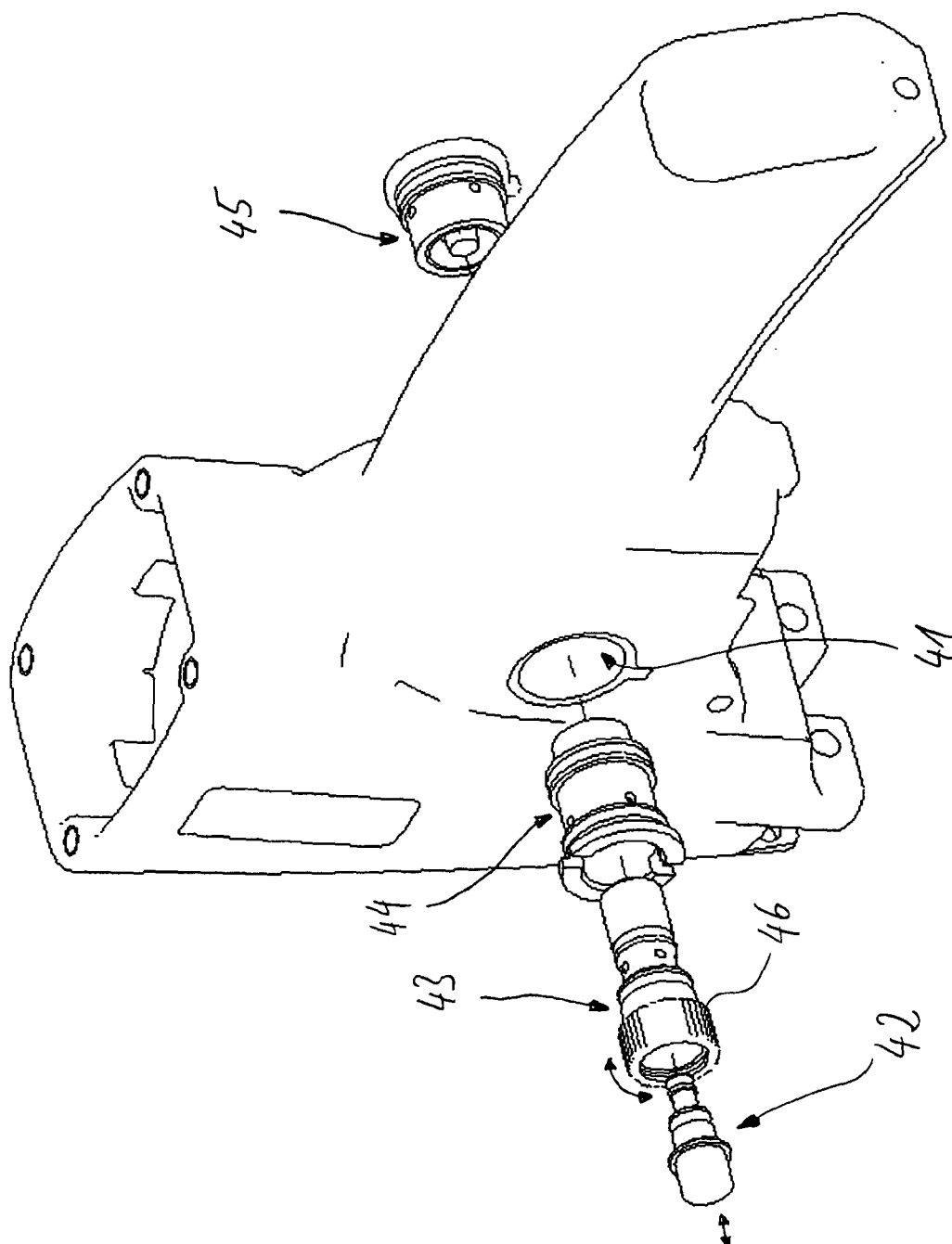
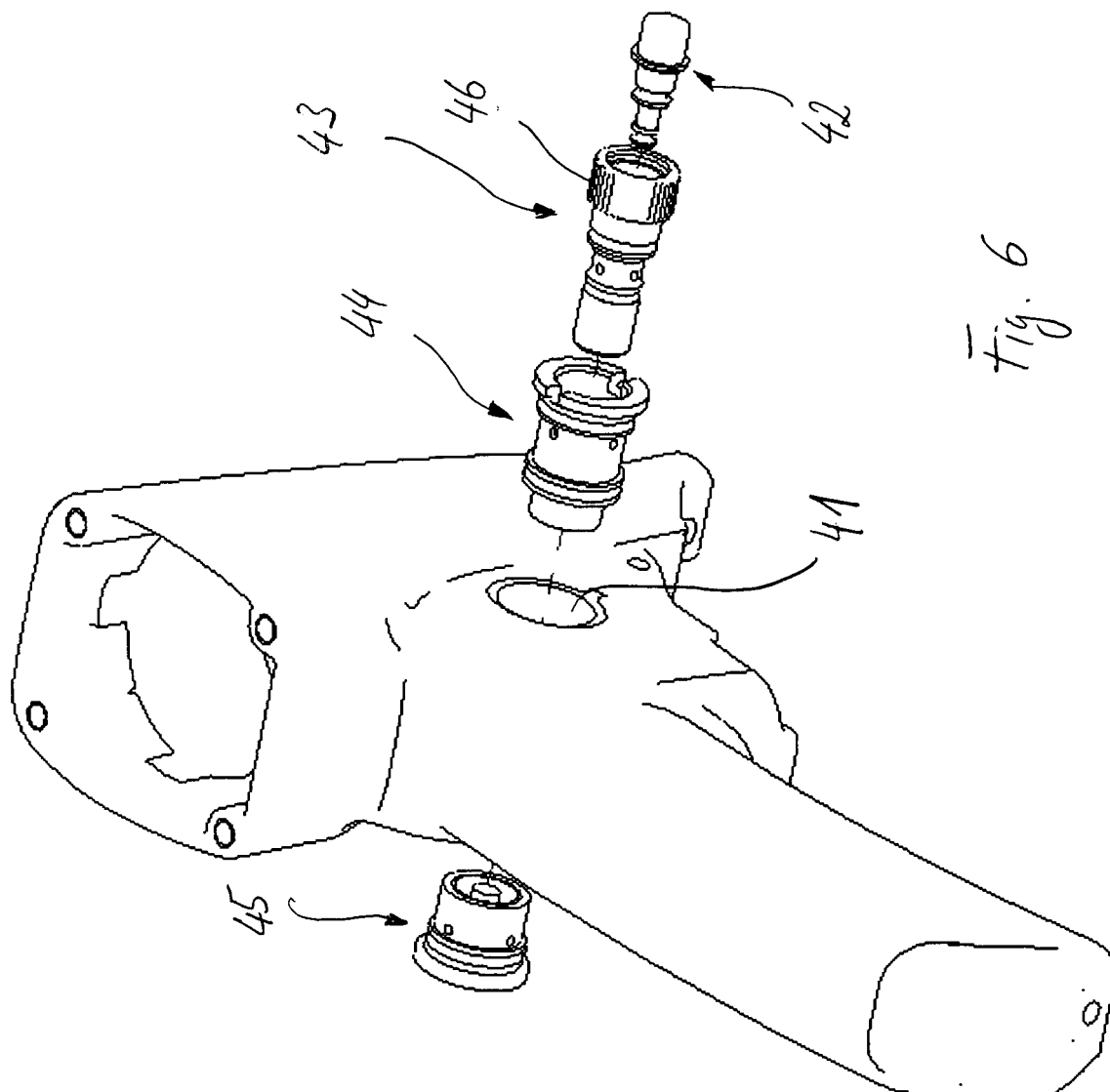
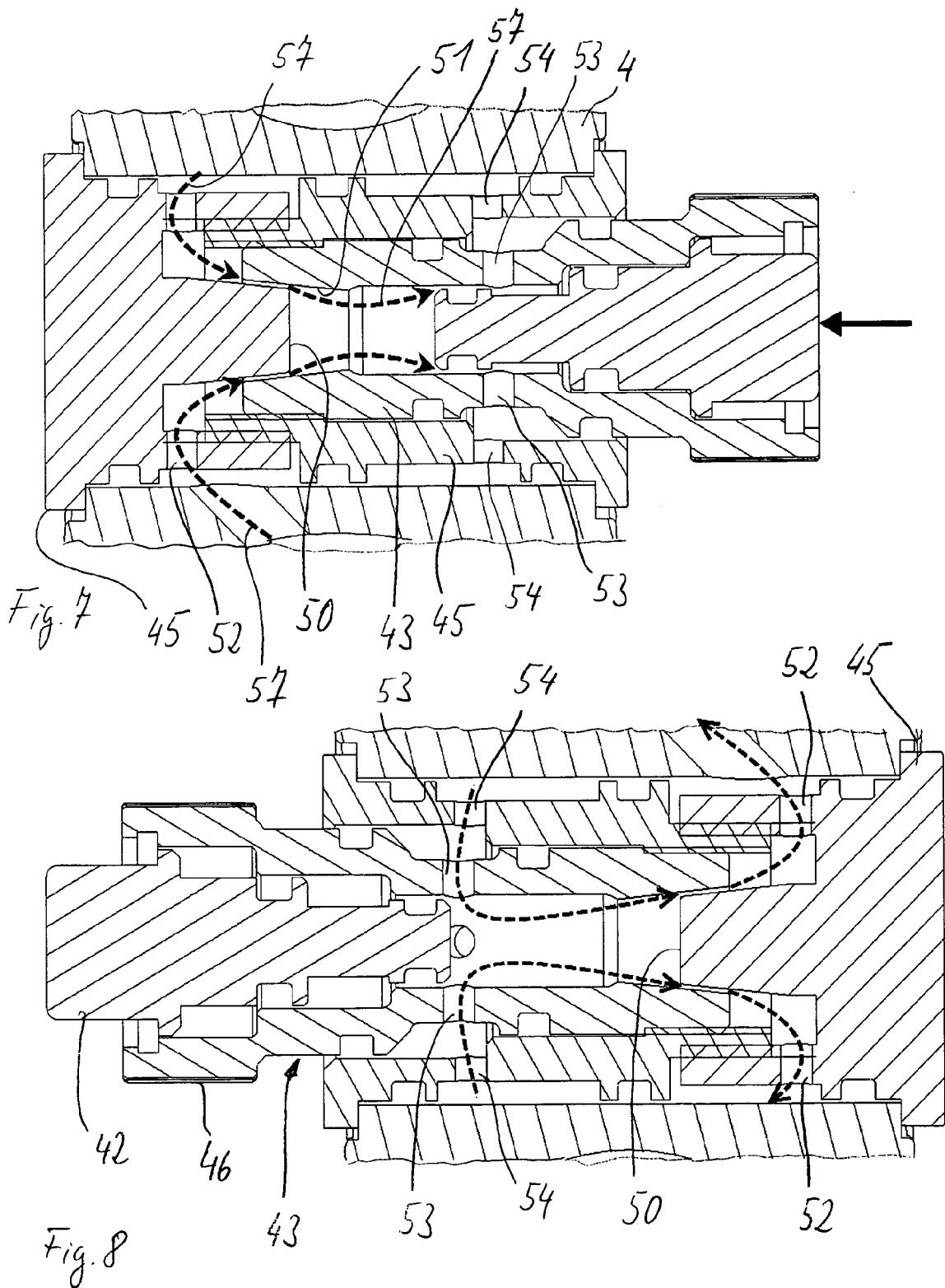


Fig. 4

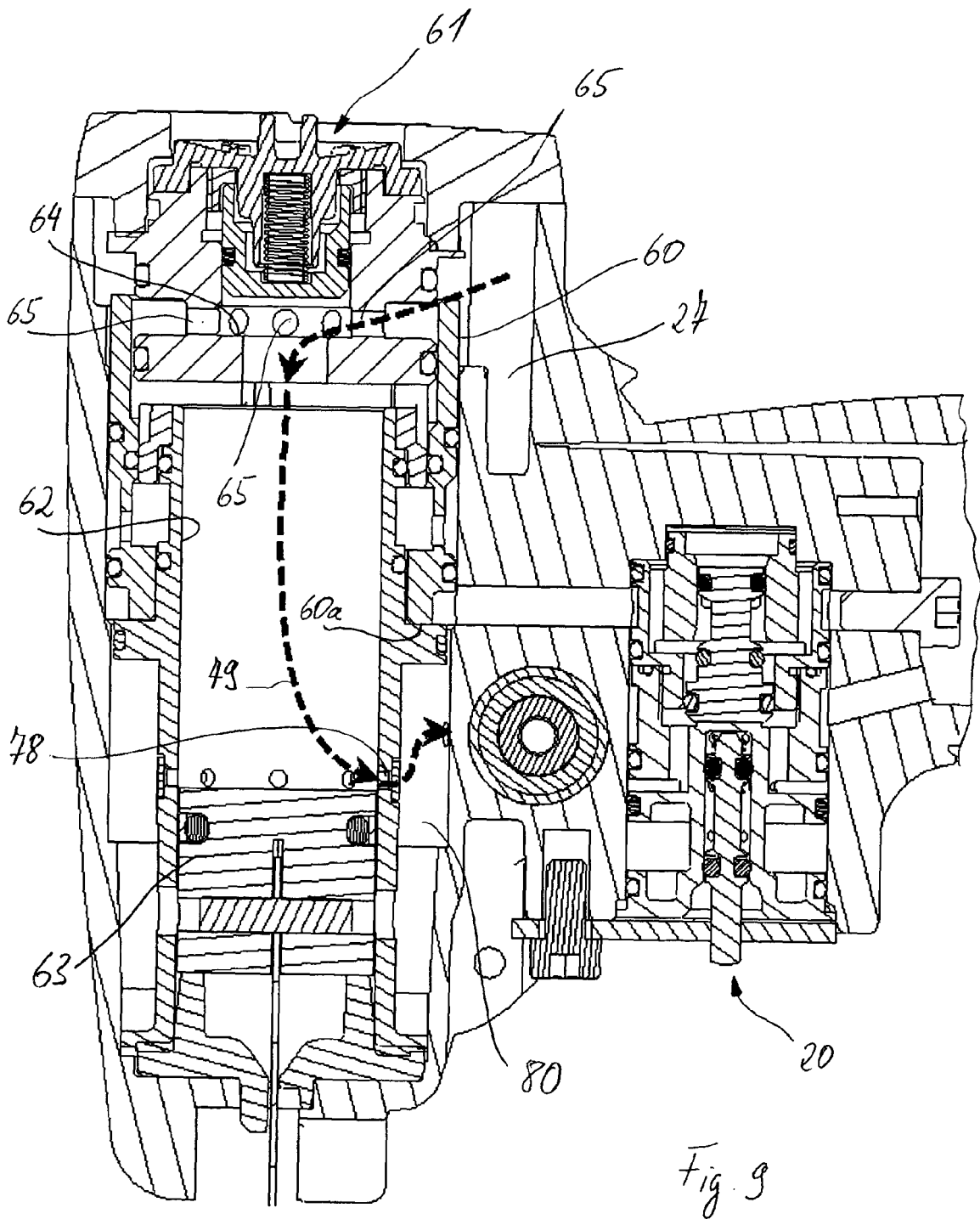
Fig. 5











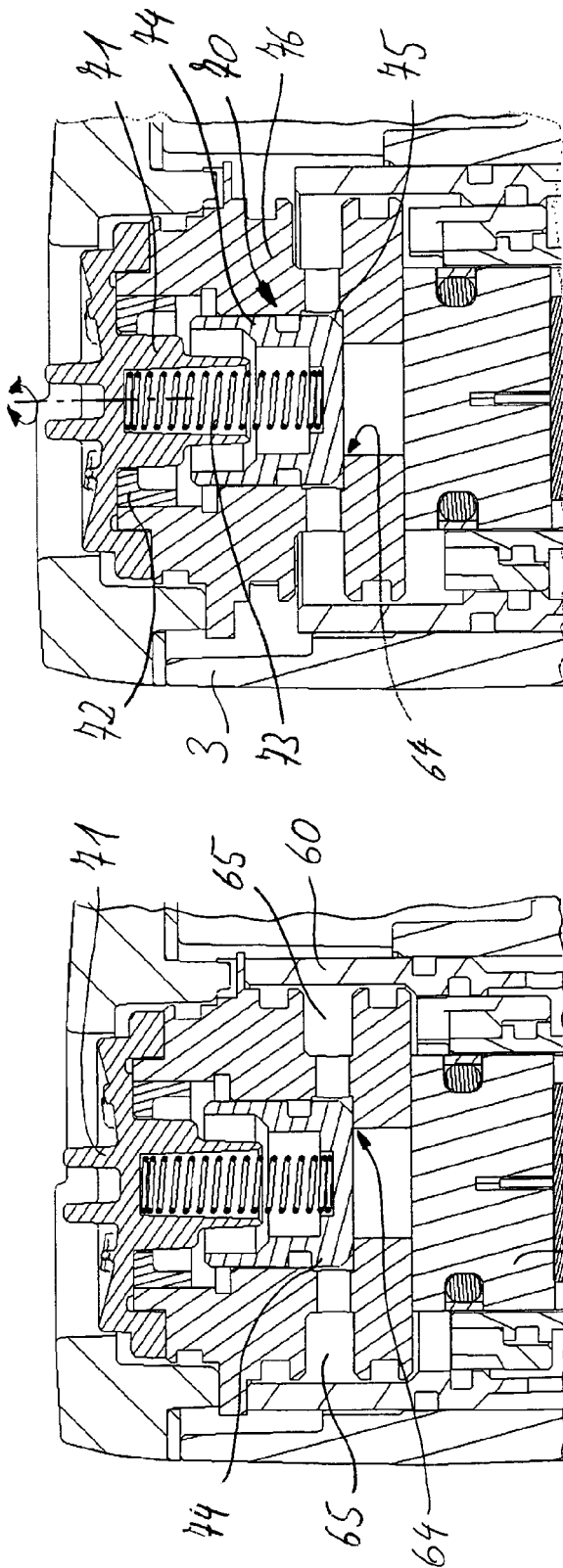


Fig. 11

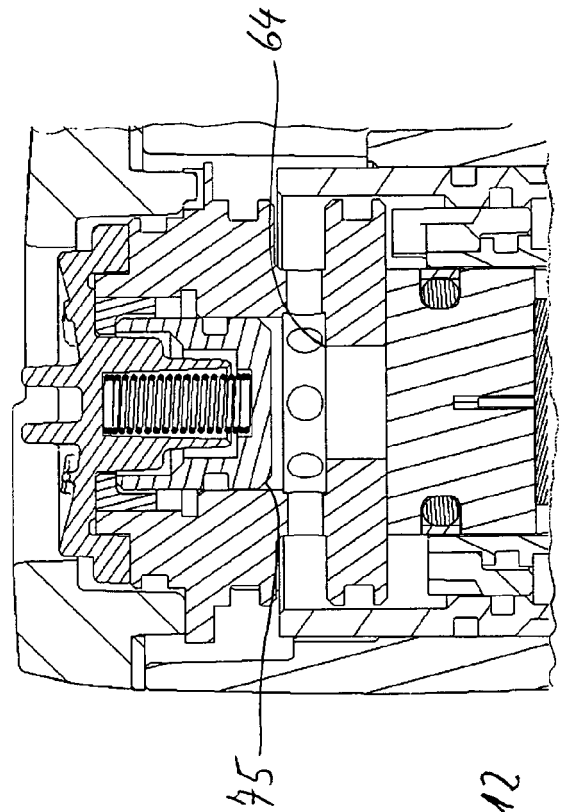


Fig. 12

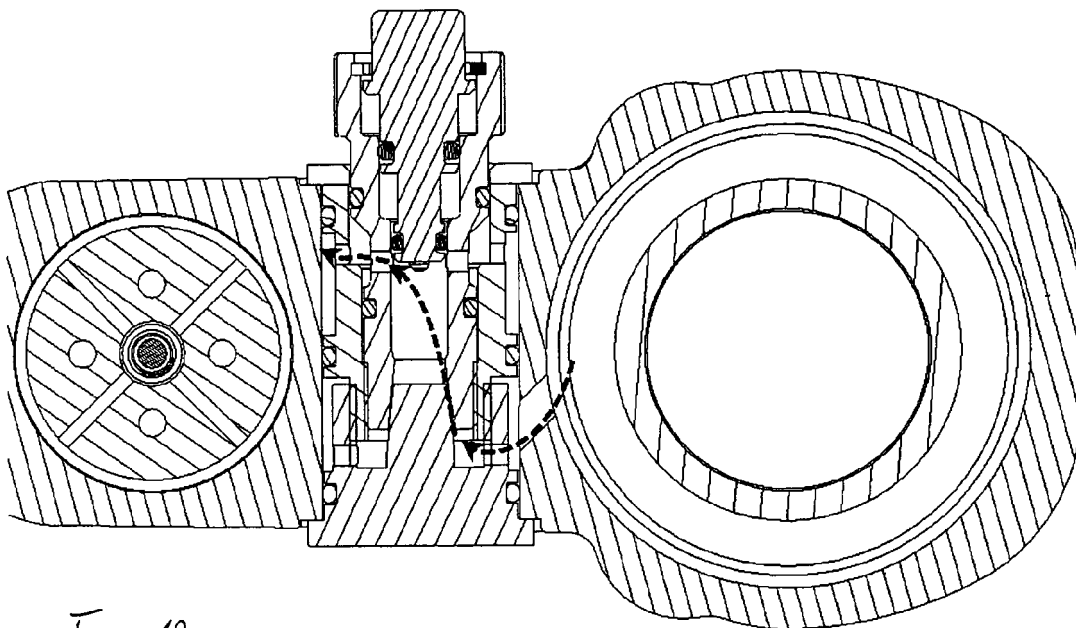


Fig. 13

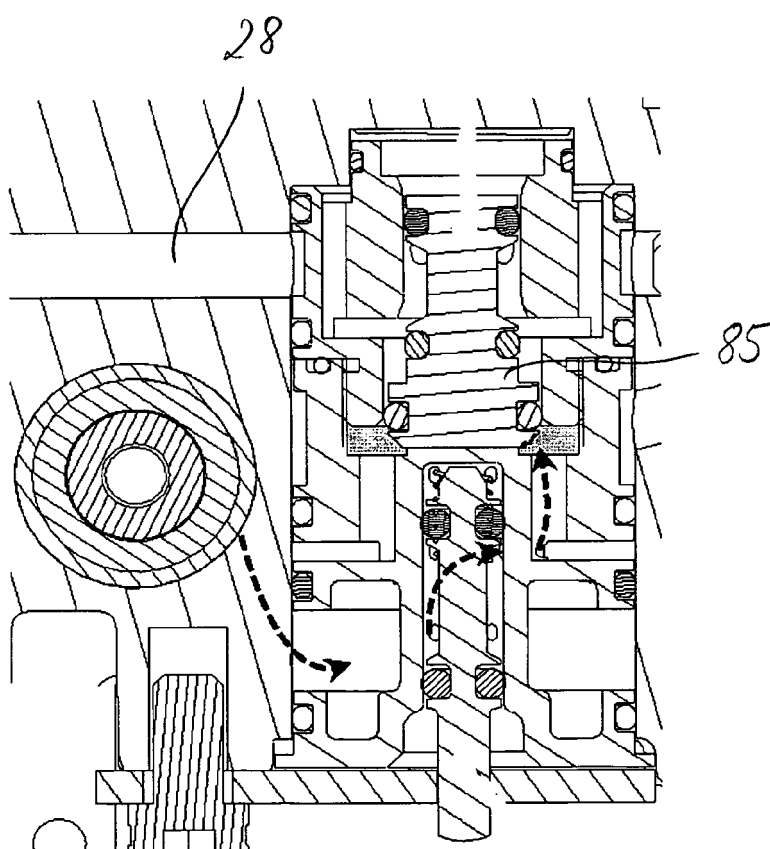
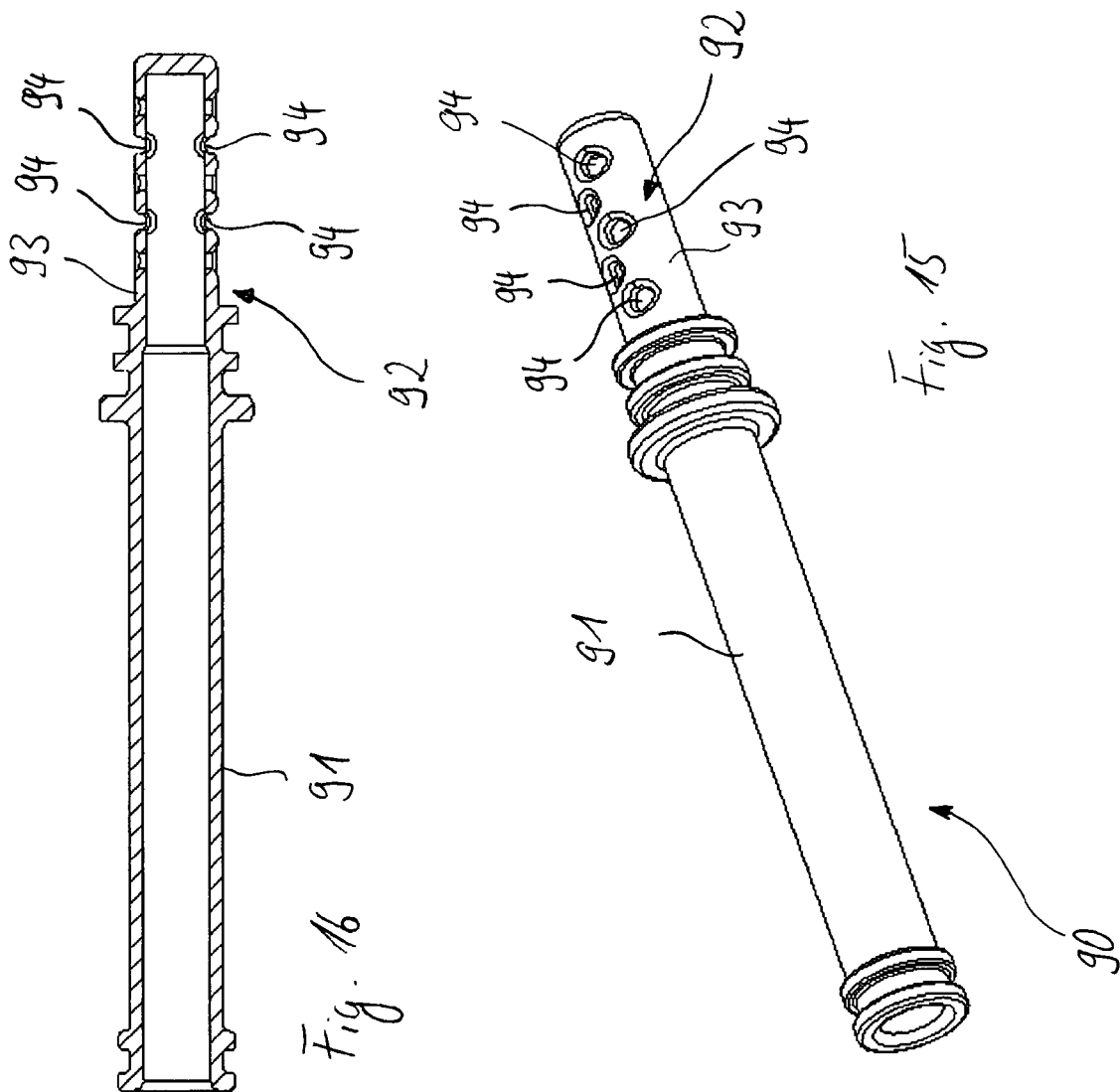


Fig. 14



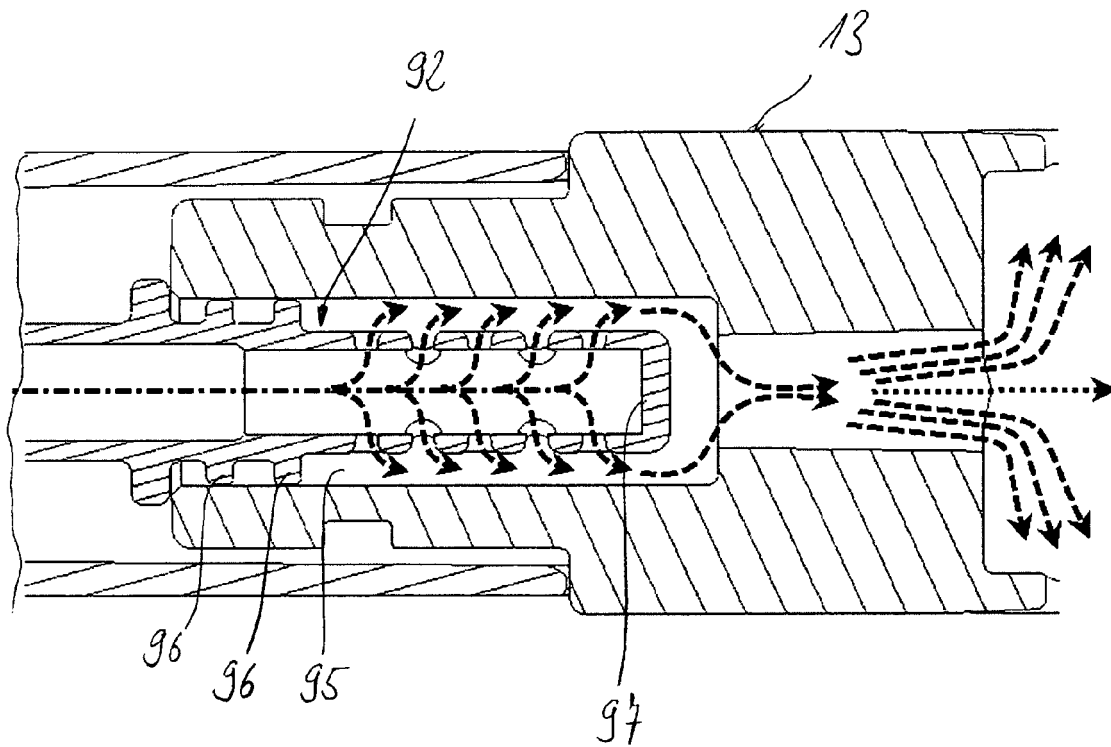


Fig. 17



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