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Échangeur de chaleur

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**WO-A1-2011/034633**

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**US-A1- 2010 006 276**

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## Description

### FIELD OF THE INVENTION

**[0001]** The invention relates to a heat exchanger, notably a micro-channel heat exchanger.

**[0002]** The invention further relates to an air conditioning system.

### BACKGROUND OF THE INVENTION

**[0003]** A heat exchanger according to the preamble of claim 1 is known from document JPS58160797.

**[0004]** An embodiment of a heat exchanger is known from US 7, 281, 387. The known heat exchanger comprises a plurality of parallel fins arranged between the adjacent tubes. The known fins have a substantially uniform depth and may be arranged in a suitable profile, like a zigzag between the adjacent tubes. An inflowing air stream impinging on the fins of the known heat exchanger is intercepted by the whole cross-section of the fin structure arranged between the tubes.

**[0005]** It is a disadvantage of the known heat exchanger that the relatively fine structure of the fin sequence is relatively sensitive for particle contamination present in the inflowing air stream. When such particle contamination is deposited on the fins or between the fins local efficiency of the known heat exchanger may be substantially reduced.

**[0006]** It is a still further disadvantage of the known heat exchanger that air cooling applications may be impeded by water condensing on the surfaces of the individual fins. The known fins structure is not able to drain this water, so it accumulates in the fine fin structure, thereby blocking the stream of the inflowing air.

**[0007]** It is a still further disadvantage of the known heat exchanger that for air cooling applications aiming at the target evaporator temperatures below zero degrees centigrade undesirable frost may be accumulated on the surfaces of the individual fins substantially reducing the overall efficiency of the heat exchanger. The fact that the known fin structure is not able to drain water well, defrost cycles are not effective. The melt water during the defrost cycle accumulates in the fine fin structure. Subsequently this water is transformed into substantially solid ice formations and blocking the air flow.

### SUMMARY OF THE INVENTION

**[0008]** It is an object of the invention to provide a heat exchanger, notably a micro-channel heat exchanger with improved operational characteristics. In particular, it is an object of the invention to provide an improved heat exchanger, in particular it is an object of the invention to provide a heat exchanger having improved fin structure.

**[0009]** To this end the heat exchanger according to the invention is defined in claim 1.

**[0010]** An additional advantage is that the fin structure

has a reduced resistance for the inflowing stream of air. In particular, it is found advantageous that the substantially parallel surfaces of the meander structure are shaped as respective openings diverging towards the external face of the heat exchanger conceived to intercept the inflowing stream of air.

**[0011]** In a particular embodiment of the heat exchanger the depth profile provided in the substantially parallel surfaces of the fins is V-shaped, having the wider portion of the V-shape at an external face of the heat exchanger conceived to receive an inflowing stream of air.

**[0012]** It is found that in this particular embodiment the heat exchanger is less sensitive to particle contamination present in the inflowing stream of air, which contributed to the improved operational characteristics of the heat exchanger according to the invention.

**[0013]** It is further found that the heat exchanger having the V-shaped depth profiles in the substantially parallel surfaces of the fins has an increased cooling capacity in condensing or frost accumulating conditions.. This may be explained by the fact that in the heat exchanger according to the invention an improved water draining capacity is achieved.

**[0014]** In a still further embodiment of the heat exchanger according to the invention the depth profile is provided for a portion of the said meander of canted shape.

**[0015]** It is found that for some circumstances it may be sufficient to provide the depth profile in the substantially parallel surfaces of the fins only for a portion of the fins forming the meander structure.

**[0016]** The invention further relates to an air conditioning device comprising the heat exchanger as is discussed with reference to the foregoing.

**[0017]** These and other aspects of the invention will be discussed with reference to drawings wherein like reference signs correspond to like elements. It will be appreciated that the drawings are presented for illustrative purposes only and may not be used for limiting the scope of the appended claims.

### BRIEF DESCRIPTION OF THE DRAWINGS

#### **[0018]**

Figure 1 presents in a schematic way an embodiment of a known heat exchanger.

Figure 2 presents in a schematic way an embodiment of the heat exchanger according to an aspect of the invention.

Figure 3 presents in a schematic way an embodiment of the heat exchanger according to a further aspect of the invention.

Figure 4 presents in a schematic way a further embodiment of the heat exchanger according to a still further aspect of the invention.

## DETAILED DESCRIPTION OF THE DRAWINGS

**[0019]** Figure 1 presents in a schematic way an embodiment of a known heat exchanger. The known heat exchanger 10 comprises a set of tubes 2 enclosing a set of fins 4, which may propagate according to a meander shape in a meander propagation direction M. The fins 4 in the meander structure form mutually substantially parallel surface 3a, ..., 3n, which are transversely arranged to the tubes 2. Preferably, the surfaces 3a, ...3n are substantially perpendicular to the tubes 2. The known heat exchanger 10 further comprises a manifold 8 comprising a set of channels 6.

**[0020]** It is found that the known heat exchanger has a limitation that the spaces between the substantially parallel surfaces 3a, ..., 3n may be polluted with particles present in the inflowing air thereby reducing cooling capacity of the known heat exchanger.

**[0021]** Figure 2 presents in a schematic way an embodiment, based on standard heat exchanger 10, of the heat exchanger 20 according to an aspect of the invention having a cross-sectional dimension X, Y and a depth dimension Z. The heat exchanger 20 comprises a set of tubes 23 enclosing the set of fins having the substantially parallel surface 22a, ..., 22n. The fins and the tubes are connected with the manifold 21 provided with channels 21a, ...21n.

**[0022]** In accordance with the invention the substantially parallel surfaces 22a, ..., 22n are provided with a depth profile 25, which has a wider portion at a face surface 24 of the fins conceived to intercept the stream of the inflowing air 26. It will be appreciated that the depth profile may have different shapes.

**[0023]** In accordance with the invention, the depth profile 25 provided in the parallel fins 22a, ..., 22n decrease the resistance for the inflowing air 26. In addition, the depth profile 25 ensures that the particles possible present in the inflowing air have a lower probability to deposit on the external face portion 24a and thereby to block the pathway of the inflowing air 26 towards the inner volume of the heat exchanger.

**[0024]** In accordance with the invention, the depth and slant profile 25 provided in the parallel fins 22a, ..., 22n increase the water drain capacity. In addition, with horizontal application of the heat exchanger, with the profiled air intake surface facing the ground, the gravity helps to drain the water so it drips off the point of the canted profile.

**[0025]** Accordingly, the heat exchanger of the invention has in practice an increased cooling capacity and an increased robustness with respect to operational malfunction.

**[0026]** Figure 3 presents in a schematic way an embodiment in a still different shape of the heat exchanger 30 according to an aspect of the invention. In the cross-sectional view shows the staggered tubes 37 with the parallelogram shaped fins having surfaces 32. Air can enter the heat exchanger with reduced flow resistance

due to the V-shaped profile 36 and water drains itself easy from the projecting edge of the tube 37.

**[0027]** Figure 4 presents in a schematic way a further embodiment of the heat exchanger 40 according to a still further aspect of the invention. In this embodiment the tubes 42 and the fins 41 are affixed to the manifold 43. The fins 41 are provided with the depth profile (i.e. groove) as is discussed with the reference to the foregoing along the complete length of the meander structure having a suitable plurality of substantially parallel surfaces (not shown for clarity).

**[0028]** While specific embodiments have been described above, it will be appreciated that the invention may be practiced otherwise than as described. The descriptions above are intended to be illustrative, not limiting. Thus, it will be apparent to one skilled in the art that modifications may be made to the invention as described in the foregoing without departing from the scope of the claims set out below.

## Claims

1. A heat exchanger comprising a first set of tubes (23) enclosing a second set of fins such that an external plane is formed by fins between said tubes that intercepts a stream of inflowing air, the first set of tubes (23) cooperating with a manifold, wherein the fins of the second set are substantially meander or canted-shaped having a number of substantially mutually parallel surfaces (22a, ..., 22n) in a direction transverse to the meander propagation direction, wherein a depth profile (25) is provided in the said mutually parallel surfaces (22a, ..., 22n) **characterized in that** the depth profile forms a groove along a length of the meander or canted shape of said fins at the external plane of the heat exchanger, the groove (25) having a wider portion formed at the external plane (24) conceived to intercept the stream of inflowing air (26) at a reduced flow resistance.
2. The heat exchanger according to claim 1, wherein the depth profile is V-shaped.
3. An air conditioning system comprising a heat exchanger according to any one of the preceding claims.
4. Use of a heat exchanger according to any of claims 1-3, with a horizontal application of the heat exchanger, with the profiled air intake plane facing the ground.

## Patentansprüche

1. Wärmetauscher mit einem ersten Satz von Rohren (23), die einen zweiten Satz von Rippen derart um-

schließen, dass eine äußere Ebene durch Rippen zwischen den Rohren gebildet wird, die einen Strom einströmender Luft abfängt, wobei der erste Satz von Rohren (23) mit einem Verteiler zusammenwirkt, wobei die Rippen des zweiten Satzes im Wesentlichen mäander- oder schrägförmig sind und eine Anzahl von im Wesentlichen parallel zueinander verlaufenden Flächen (22a, ..., 22n) in einer Richtung quer zur Mäanderausbreitungsrichtung aufweisen, wobei ein Tiefenprofil (25) in den parallel zueinander verlaufenden Flächen (22a, ..., 22n) vorgesehen ist, **dadurch gekennzeichnet, dass** das Tiefenprofil eine Nut entlang einer Länge der Mäander- oder Schrägform der Rippen in der äußeren Ebene des Wärmetauschers bildet, die Nut (25) einen breiteren Abschnitt aufweist, der an der äußeren Ebene (24) ausgebildet ist und so konzipiert ist, dass er den Strom einströmender Luft (26) mit einem verringerten Strömungswiderstand abfängt.

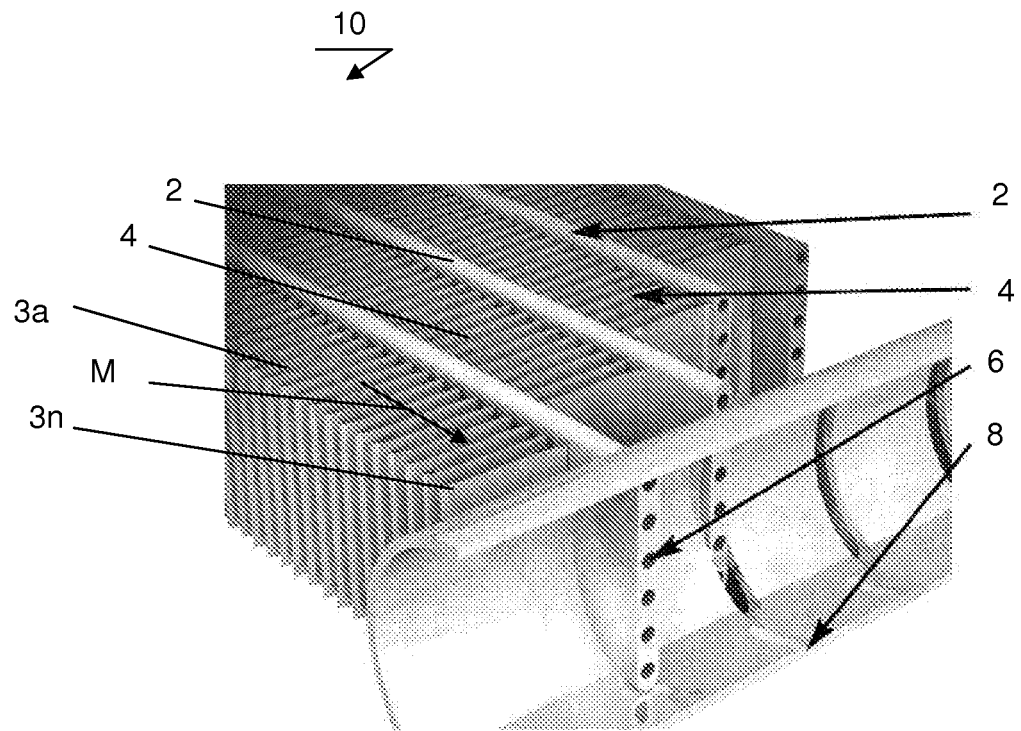
2. Wärmetauscher nach Anspruch 1, wobei das Tiefenprofil V-förmig ist.
3. Klimatisierungssystem mit einem Wärmetauscher nach einem der vorhergehenden Ansprüche.
4. Verwendung eines Wärmetauschers nach einem der Ansprüche 1 bis 3 bei horizontaler Anwendung des Wärmetauschers, wobei die profilierte Lufteinlassenebene dem Boden zugewandt ist.

3. Système de climatisation comprenant un échangeur de chaleur selon l'une quelconque des revendications précédentes.

- 5 4. Utilisation d'un échangeur de chaleur selon l'une quelconque des revendications 1 à 3, avec une application horizontale de l'échangeur de chaleur, le plan d'admission d'air profilé faisant face au sol.

## Revendications

1. Echangeur de chaleur comprenant un premier ensemble de tubes (23) renfermant un second ensemble d'ailettes de sorte qu'un plan externe est formé par des ailettes entre lesdits tubes qui intercepte un flux d'air entrant, le premier ensemble de tubes (23) coopérant avec un collecteur, dans lequel les ailettes du second ensemble sont sensiblement en forme de méandre ou de forme inclinée comportant un certain nombre de surfaces sensiblement mutuellement parallèles (22a, ..., 22n) dans une direction transversale à la direction de propagation de méandre, dans lequel un profilé de profondeur (25) est prévu dans lesdites surfaces mutuellement parallèles (22a, ..., 22n), **caractérisé en ce que** le profil de profondeur forme une rainure sur une longueur de la forme de méandre ou inclinée desdites ailettes au niveau du plan extérieur de l'échangeur de chaleur, la rainure (25) ayant une partie plus large formée au niveau du plan extérieur (24) conçue pour intercepter le flux d'air entrant (26) à une résistance à l'écoulement réduite.
2. Echangeur de chaleur selon la revendication 1, dans lequel le profil de profondeur est en forme de V.



PRIOR ART

Fig. 1

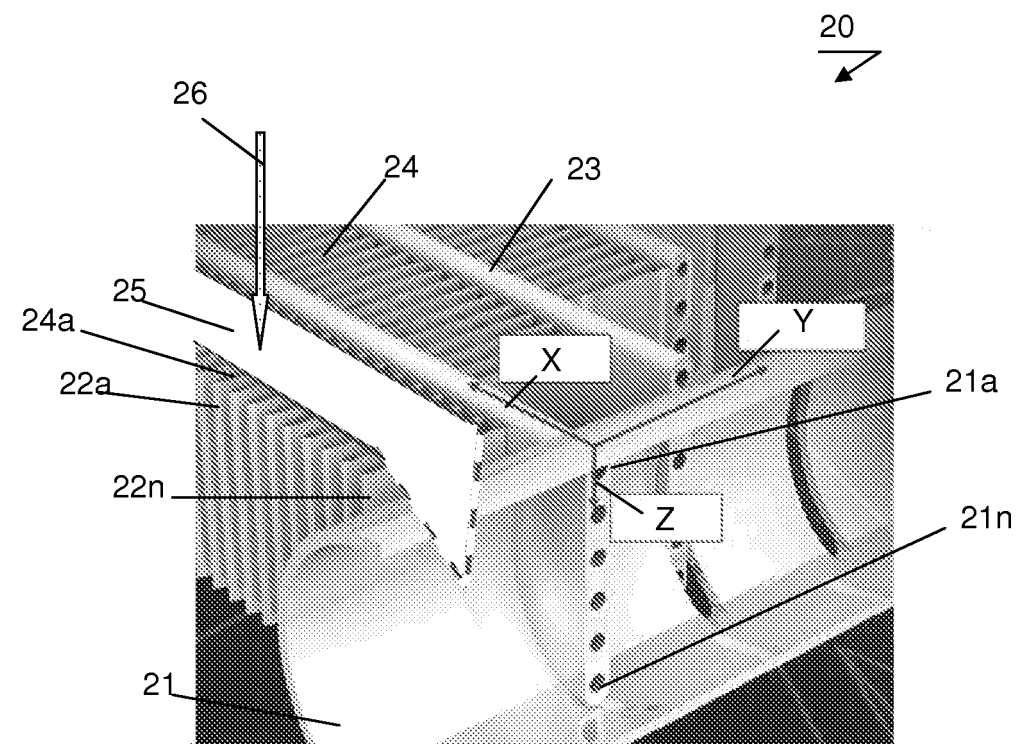


Fig. 2

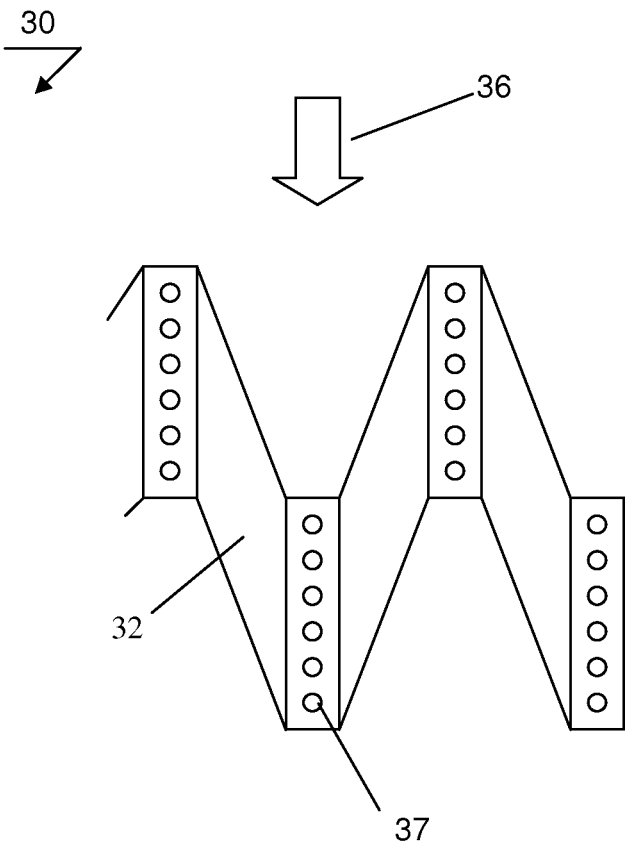


Fig. 3



Fig. 4



**REFERENCES CITED IN THE DESCRIPTION**

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**Patent documents cited in the description**

- JP S58160797 B [0003]
- US 7281387 B [0004]