

## (11) **EP 2 620 723 A2**

(12)

## **EUROPEAN PATENT APPLICATION**

(43) Date of publication: 31.07.2013 Bulletin 2013/31

(51) Int Cl.: **F25B 1/02** (2006.01) **F25B 49/02** (2006.01)

**F25B 5/02 (2006.01)** F25B 1/10 (2006.01)

(21) Application number: 13153066.9

(22) Date of filing: 29.01.2013

(84) Designated Contracting States:

AL AT BE BG CH CY CZ DE DK EE ES FI FR GB
GR HR HU IE IS IT LI LT LU LV MC MK MT NL NO
PL PT RO RS SE SI SK SM TR
Designated Extension States:

Designated Extension States:

**BA ME** 

(30) Priority: 30.01.2012 KR 20120009083

(71) Applicant: LG Electronics, Inc. Seoul, 150-721 (KR)

(72) Inventors:

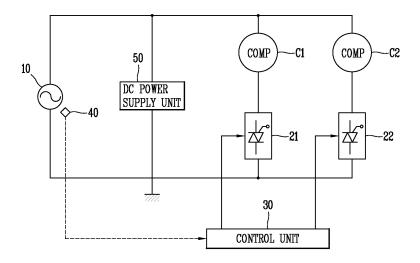
- Yoo, Jaeyoo Seoul (KR)
- Kim, Gyunam Seoul (KR)
- Lee, Boram Seoul (KR)
- (74) Representative: Ter Meer Steinmeister & Partner Patentanwälte
  Mauerkircherstrasse 45
  81679 München (DE)

## (54) Apparatus and method for controlling compressor, and refrigerator having the same

(57) There is disclosed herein a compressor control apparatus, and a refrigerator (700) including the same. According to the embodiments of the present disclosure, the operation of two compressors (C1, C2) may be controlled using an alternating current (AC) switch, thereby minimizing the use of elements as well as increasing the compressor capacity and enhancing the operation efficiency of a system. According to the embodiments of the

present disclosure, a plurality of operation modes may be used to correspond to a load or freezing capacity using two compressors (C1, C2). Furthermore, according to the present disclosure, two compressors (C1, C2) may be operated in a separate or simultaneous manner using a drive configured with two alternating current (AC) switches, thereby simplifying the configuration of a system to reduce the cost.

# FIG. 1



EP 2 620 723 A2

#### Description

15

20

30

35

40

45

50

55

#### BACKGROUND OF THE INVENTION

#### 5 1. Field of the invention

**[0001]** The present disclosure relates to a compressor control apparatus for controlling the operation of two compressors using one inverter and a refrigerator comprising the same.

#### 2. Description of the related art

**[0002]** In general, compressor is an apparatus for converting mechanical energy into compression energy of a compression fluid and used as part of a freezing device, such as a refrigerator, an air conditioner or the like.

[0003] Compressors may be divided into a reciprocating compressor, a rotary compressor, and a scroll compressor. The reciprocating compressor forms a compression space between a piston and a cylinder to inhale or discharge a working gas, thereby compressing refrigerant while moving in a linear reciprocating manner. The rotary compressor forms a compressor space between an eccentrically rotated roller and a cylinder to inhale or discharge a working gas, thereby compressing refrigerant while the roller is eccentrically rotated along an inner wall of the cylinder. The scroll compressor forms a compression space between an orbiting scroll and a fixed scroll to inhale or discharge a working gas, thereby compressing refrigerant while the orbiting scroll is rotated along the fixed scroll.

**[0004]** The reciprocating compressor allows an inner piston to move in a linear reciprocating manner within the cylinder, thereby inhaling, compressing and discharging a refrigerant gas. The reciprocating compressor is largely divided into a recipro type and a linear type depending on how the piston is driven.

**[0005]** The recipro type reciprocating compressor is a scheme in which a crank shaft is coupled to a rotating motor and a piston is coupled to the crank shaft, thereby converting a rotational movement into a linear reciprocating movement. On the contrary, the linear type reciprocating compressor is a scheme in which a piston is connected to a linearly moving mover of the motor, thereby converting a linear movement of the motor into a reciprocating movement of the piston.

**[0006]** The reciprocating compressor may include an electric power unit for generating a driving force, and a compression unit for receiving the driving force from the electric power unit to compress a fluid. A motor is typically used for the electric power unit, and a linear motor is used in case of the linear type.

[0007] For the linear motor, the motor itself directly generates a linear driving force, and thus a mechanical conversion device is not required, and the structure thereof is not complicated. Furthermore, the linear motor may reduce loss due to energy conversion, and there is no connecting portion causing friction and abrasion, thereby greatly reducing noise. Furthermore, when a linear type reciprocating compressor (hereinafter, referred to as a "linear compressor") is used for a refrigerator or air conditioner, a stroke voltage applied to the linear compressor may be changed to change the compression ratio, thereby providing an advantage that the linear compressor can be used for variable freezing capacity control.

**[0008]** On the other hand, since a reciprocating compressor, particularly, linear compressor, performs a reciprocating movement in a state that the piston is not mechanically restricted in the cylinder, the piston may collide with the cylinder wall or the piston cannot move forward due to a large load when voltage is suddenly and excessively applied, thereby causing difficulty in properly performing the compression. Accordingly, a control apparatus for controlling the movement of the piston for the variation of a load or voltage is essentially required.

**[0009]** In general, a compressor control apparatus detects voltage and current applied to the compressor motor and estimates a stroke with a sensorless method to perform feedback control. At this time, the compressor control apparatus may include a triac or inverter as a means for controlling the compressor.

### SUMMARY OF THE INVENTION

**[0010]** According to the embodiments of the present disclosure, an object is to provide a compressor control apparatus and method capable of operating two compressors using an alternating current (AC) switch, and a refrigerator including the same

**[0011]** According to the embodiments of the present disclosure, another object is to provide a compressor control apparatus and method for detecting current and voltage applied to two compressor motors, respectively, and estimating a stroke of each compressor to control the stroke or frequency of the two compressors, in a separate or simultaneous manner, and a refrigerator including the same.

[0012] A compressor control apparatus according to an embodiment may include a first and a second alternating current (AC) switch switched based on a first and a second control signal to drive a first and a second compressor, and a control unit configured to generate the first and the second control signal based on a load of the first and the second

compressor to output them to the first and the second alternating current switch. Here, the first and the second compressor may be operated in a simultaneous manner, or the first compressor or the second compressor may be operated in a separate manner.

**[0013]** In the compressor control apparatus, the control unit may vary a firing angle of the first alternating current switch or second alternating current switch based on a freezing capacity of the first compressor or second compressor.

**[0014]** The compressor control apparatus may further include an input voltage detection unit configured to detect the power voltage of commercial alternating current (AC) power. Furthermore, compressor control apparatus may further include a direct current (DC) power supply unit configured to convert the commercial alternating current power into direct current power to apply it to the first and the second alternating current switch.

**[0015]** A compressor control method for controlling a first compressor and a second compressor using a first and a second alternating current (AC) switch, respectively, according to an embodiment may include receiving a compressor operation mode, and driving the first and the second alternating current switch in a simultaneous manner or driving the first alternating current switch or the second alternating current switch in a separate manner based on the compressor operation mode.

[0016] A refrigerator according to an embodiment may include a refrigerator body, a first and a second compressor provided in the refrigerator body to compress refrigerant, respectively, a first and a second alternating current switch switched based on a first and a second control signal to drive the first and the second compressor, and a control unit configured to generate the first and the second control signal based on a load of the first and the second compressor and output them to the first and the second alternating current switch, wherein the first and the second compressor are operated in a simultaneous manner or the first compressor or the second compressor is operated in a separate manner. [0017] According to the embodiments of the present disclosure, the operation of two compressors may be controlled using an alternating current (AC) switch, thereby minimizing the use of elements as well as increasing the compressor

**[0018]** According to the embodiments of the present disclosure, a plurality of operation modes may be used to correspond to a load or freezing capacity using two compressors. Furthermore, according to the present disclosure, two compressors may be operated in a separate or simultaneous manner using two alternating current (AC) switches, thereby simplifying the configuration of a system to reduce the cost.

#### BRIEF DESCRIPTION OF THE DRAWINGS

capacity and enhancing the operation efficiency of a system.

**[0019]** The accompanying drawings, which are included to provide a further understanding of the invention and are incorporated in and constitute a part of this specification, illustrate embodiments of the invention and together with the description serve to explain the principles of the invention.

[0020] In the drawings:

10

20

30

35

40

45

50

55

FIGS. 1 and 2 are views schematically illustrating a compressor control apparatus according to the embodiments of the present disclosure;

FIGS. 3A and 3B are graphs for explaining the operation of controlling the working of two compressors;

FIG. 4 is a flow chart schematically illustrating a compressor control method according to an embodiment;

FIG. 5 is a perspective view illustrating a refrigerator to which two compressors are applied; and

FIG. 6 is a cross-sectional view illustrating a reciprocating compressor having a compressor control apparatus according to the embodiments of the present disclosure.

## DETAILED DESCRIPTION OF THE INVENTION

**[0021]** Referring to FIG. 1, a compressor control apparatus according to an embodiment may include a first and a second alternating current (AC) switch 21, 22 switched based on a first and a second control signal to drive a first and a second compressor (C1, C2), and a control unit 30 configured to generate a first and a second control signal based on a load of the first and the second compressor (C1, C2) to output them to the first and the second alternating current switch 21, 22. Here, the compressor control apparatus may operate the first and the second compressor in a simultaneous manner or operate the first compressor or the second compressor in a separate manner.

**[0022]** The first and the second alternating current (AC) switch are opened or closed to apply the motor drive voltage and motor drive current to compressor motors provided in the first and the second compressor. The alternating current (AC) switch may include a thyrister, a triac, or the like, but the triac is mostly used for the alternating current (AC) switch. The control unit 30 varies a firing angle of the first alternating current switch 21 or second alternating current switch 22 based on the freezing capacity of the first compressor (C1) and second compressor (C2).

[0023] FIGS. 3A and 3B illustrate a method of varying a firing angle of the triac to control a size of freezing capacity of the compressor when the triac is used for the alternating current switch. Referring to FIG. 3A, when a gate voltage is

received at the first alternating current switch 21 based on a first control signal, the first alternating current switch 21 supplies a first motor drive current to the first compressor (C1). Similarly, referring to FIG. 3B, when a gate voltage is received at the second alternating current switch 22 based on a second control signal, the second alternating current switch 22 supplies a second motor drive current to the second compressor (C2). At this time, the control unit 30 may generate a first and a second signal for varying a firing angle of the first and the second alternating current switch to perform a load corresponding operation.

**[0024]** Referring to FIG. 1 again, the compressor control apparatus may further include a power voltage detection unit 40 configured to detect the power voltage of the commercial alternating current (AC) power 10. Furthermore, the compressor control apparatus may further include a zero voltage detection unit (not shown) configured to detect the zero voltage of the power voltage, and a power frequency detection unit (not shown) configured to detect the power frequency of the commercial alternating current (AC) power.

[0025] Furthermore, the compressor control apparatus may further include a direct current (DC) power supply unit 50 configured to convert the commercial alternating current power 10 into direct current power to apply it to the first and the second alternating current switch. The direct current (DC) power supply unit 50 is a switching mode power supply (SMPS) for receiving commercial alternating current power and performing AC-DC conversion, and supplies drive voltages (for example, 5, 15 V) for the elements of the control unit 30, the first and the second alternating current switch, and the like. In other words, direct current voltages required for the two alternating current switches and two compressors are supplied using one SMPS.

**[0026]** Here, at least one of the first and the second compressor (C1, C2) may be a reciprocating compressor, particularly, a linear compressor. Furthermore, the two compressors may be configured to have different capacities. The first and the second compressor may be operated in a simultaneous manner or operated in a separate manner, respectively, by a compressor control apparatus using two alternating current switches. It may be merely defined as a compressor operation mode. The compressor operation mode is an operation mode determined by a load or required freezing capacity of the first and the second compressor. The compressor operation mode may be an operation mode for controlling the stroke, frequency or the like of each compressor by dividing it into a predetermined value. Here, the compressor operation mode may be merely divided into a separate operation mode of the first compressor, a separate operation mode of the second compressor, and a simultaneous operation mode of the first and the second compressor.

**[0027]** Referring to FIG. 6, the first and the second compressor may include a casing 100 communicated with a gas suction pipe (SP) and a gas discharge pipe (DP), a frame unit 200 elastically supported by an inner portion of the casing 100, a motor 300 supported by the frame unit 200 to allow a mover 330 to perform a linear reciprocating movement, a compression unit 400 in which a piston 420 is coupled to the mover 330 of the motor 300 and supported by the frame unit 200, a plurality of resonant units 500 for elastically supporting the mover 330 of the motor 300 and the piston 420 of the compression unit 400 in the movement direction to induce a resonant movement.

30

35

40

45

50

55

**[0028]** The frame unit 200 may include a first frame 210 by which the compression unit 400 is supported and which supports a front side of the motor 300, a second frame 220 coupled to the first frame 210 to support a rear side of the motor 300, and a third frame 230 coupled to the second frame 220 to support a plurality of resonant springs 530. The first frame 210, second frame 220, and third frame 230 may be all formed of a non-magnetic body, such as aluminium, to reduce core loss.

**[0029]** Furthermore, the first frame 210 is formed with a frame portion 211 having a ring plate shape, a cylinder portion 212 having a cylindrical shape into which a cylinder 410 is inserted is formed on a rear surface, namely, lengthwise as an integral body in the motor direction, at the center of the frame portion 211. The frame portion 211 is preferably formed such that the outer diameter of the frame portion 211 is at least not less than the inner diameter of the outer stator 310 of the motor 300 to support both an outer stator 310 and an inner stator 320.

**[0030]** Furthermore, the first frame 210 is fixed such that the inner stator 320 is inserted into an outer circumferential surface of the cylinder portion 212. In this case, the first frame 210 is preferably formed of a non-magnetic body, such as aluminium, to reduce magnetic loss. Furthermore, the cylinder portion 212 may be formed on the cylinder 410 as an integral body using an insert die casting method. However, the cylinder portion 212 may be screw-assembled such that the cylinder 410 is pressurized or a screw thread is formed at an inner circumferential surface thereof. Furthermore, it may be preferable in the stability aspect of the cylinder 410 that a step surface or inclined surface is formed between a front side inner circumferential surface and a rear side inner circumferential surface of the cylinder portion 212, thereby allowing the cylinder 410 coupled to an inner circumferential surface of the cylinder portion 212 to be supported in the piston direction.

[0031] The motor 300 may include an outer stator 310 supported between the first frame 210 and second frame 220 and around which a coil 311 is wound, an inner stator 320 coupled to an inner side of the outer stator 310 with a predetermined interval and inserted into the cylinder portion 212, and a mover 330 in which a magnet 331 is provided to correspond to the coil 311 of the outer stator 310 to perform a linear reciprocating movement along the magnetic flux direction between the outer stator 310 and inner stator 320. The outer stator 310 and inner stator 320 are formed by laminating a plurality of thin stator core sheets in a cylindrical shape for each sheet or laminating a plurality of thin stator

core sheets in a block shape and laminating the stator block in a radial shape.

20

30

35

40

45

50

55

[0032] The compression unit 400 may include a cylinder 410 formed on the first frame 210 as an integral body, a piston 420 coupled to the mover 330 of the motor 300 to perform a reciprocating movement in the compression space (P) of the cylinder 410, a suction valve 430 mounted at a front end of the piston 420 to control the suction of the refrigerant gas while opening or closing the suction passage 421 of the piston 420, a discharge valve 440 mounted at a discharge side of the cylinder 410 to control the suction of the compression gas while opening or closing the compression space (P) of the cylinder 410, a valve spring 450 elastically supporting the discharge valve 440, and a discharge cover 460 fixed to the first frame 210 at a discharge side of the cylinder 410 to accommodate the discharge valve 440 and valve spring 450.

[0033] The cylinder 410 is formed in a cylindrical shape and inserted and coupled to the cylinder portion 212 of the first frame 210.

**[0034]** The cylinder 410 may be formed of a material having a hardness higher than that of cast iron or at least that of the first frame 210, more accurately, that of the cylinder portion 212 by considering abrasion due to the piston 420 as forming a bearing surface with the piston 420 an inner circumferential surface of which is made of cast iron.

**[0035]** The piston 420 may be preferably formed of the same material as the cylinder 410 or formed of a material having a hardness similar to that of the cylinder 410 to reduce abrasion with the cylinder 410. Furthermore, the suction passage 421 is formed in a penetrated manner within the piston 420 such that refrigerant is inhaled into the compression chamber (P) of the cylinder 410.

**[0036]** The resonant unit 500 may include a spring supporter 510 coupled to a connecting portion between the mover 330 and the piston 420, first resonant springs 520 supported at a front side of the spring supporter 510, and second resonant springs 530 supported at a rear side of the spring supporter 510.

**[0037]** In the drawing, non-described reference numeral 422 denotes a piston connecting portion and non-described reference numeral 600 denotes an oil feeder.

[0038] When power is applied to the motor 300 and a magnetic flux is formed between the outer stator 310 and inner stator 320, the mover 330 placed at a gap between the outer stator 310 and inner stator 320 continuously performs a reciprocating movement by the resonant unit 500 while moving along the direction of the magnetic flux. Furthermore, when the piston 420 performs a backward movement within the cylinder 410, refrigerant filled in an inner space of the casing 100 passes through the suction passage 421 of the piston 420 and the suction valve 430 and inhaled into the compression space (P) of the cylinder 410. When the piston 420 performs a forward movement within the cylinder 410, refrigerant gas inhaled into the compression space (P) is compressed to repeat a series of processes of discharging while opening the discharge valve 440.

**[0039]** The reciprocating compressor may be widely used for a freezing device such as a refrigerator or air conditioner. When the first and the second compressor are applied to a refrigerator as illustrated in FIG. 5, it may be designed such that each compressor takes charge of the refrigerating chamber and freezing chamber, respectively.

**[0040]** Referring to FIGS. 1 and 5, a refrigerator according to an embodiment may include a refrigerator body, a first and a second compressor provided in the refrigerator body to compress refrigerant, respectively, a first and a second alternating current switch switched based on a first and a second control signal to drive the first and the second compressor, and a control unit configured to generate the first and the second control signal based on a load of the first and the second compressor and output them to the first and the second alternating current switch, wherein the first and the second compressor are operated in a simultaneous manner or the first compressor or the second compressor is operated in a separate manner.

**[0041]** Referring to FIG. 5, the refrigerator 700 is provided with a main board 710 therein for controlling the entire operation of the refrigerator, and connected to the first and the second compressor (C1, C2). The compressor control apparatus may be provided in the main board 710. The refrigerator 700 is operated by driving the first and the second compressor. Cool air supplied to an inner portion of the refrigerator is generated by the operation of heat exchange with refrigerant, and continuously supplied to an inner portion of the refrigerator while repeatedly performing compression-condensation-expansion-evaporation cycles. The supplied refrigerant is uniformly transferred to an inner portion of the refrigerator by convection, thereby allowing foods within the refrigerator to be stored at a desired temperature.

**[0042]** The compressor control apparatus may further include a first and a second load detection unit configured to detect a load of the first and the second compressor. Here, the control unit 30 generates a first and a second control signal based on the load of the first and the second compressor (C1, C2) to operate the first and the second compressor in a separate or simultaneous manner. The control unit 30 generates the first and the second control signal using a first and a second stroke of the first and the second compressor (C1, C2), and stroke instruction values for the first and the second compressor. Here, the load of the compressor may include a motor current, a motor voltage, a stroke, their phase difference, a frequency, and the like. For example, when a compressor is provided in a refrigerator, the load of the compressor may be detected using a load of the refrigerator.

[0043] Referring to FIG. 2, the compressor control apparatus may further include a first current detection unit 611 configured to detect a first motor drive current applied to a first motor provided in the first compressor (C1) and a first

voltage detection unit 612 configured to detect a first motor drive voltage applied to the first motor. Furthermore, the compressor control apparatus may further include a second current detection unit 621 configured to detect a second motor drive current applied to a second motor provided in the second compressor (C2) and a second voltage detection unit 622 configured to detect a second motor drive voltage applied to the second motor.

**[0044]** The first and the second current detection unit 611, 621 detect a drive current applied to the compressor based on a load of the compressor or a load of the freezer. The current detection units detect a motor current applied to the compressor motor. The first and the second voltage detection unit 612, 622 detect a motor voltage applied to the compressor. The voltage detection units detect a motor voltage applied between both ends of the compressor motor based on a load of the compressor.

**[0045]** The compressor control apparatus according to the embodiments may further include a first and a second stroke calculation unit 613, 623 configured to calculate a first and second stroke of the first and the second compressor, respectively, using the motor drive current and the motor drive voltage. The relationship among the motor voltage, motor current, and stroke is as follows. The first and the second stroke calculation unit 613, 623 may calculate a stroke using the following equation based on a motor voltage detected through the first and the second voltage detection unit 612, 622, and a motor current detected through the first and the second current detection unit 611, 621.

# [Equation 1]

15

20

25

30

35

40

50

55

 $x = \frac{1}{\alpha} \int (Vm - Ri - L\frac{di}{dt})dt$ 

**[0046]** Here, x is a stroke,  $\alpha$  is a motor constant, Vm is a motor voltage, R is a resistance, L is an inductance, and i is a motor current.

[0047] The control unit 30 receives a first stroke instruction value (xref1) and compares a first stroke estimate value (x1) calculated by the first stroke calculation unit 613 with the first stroke instruction value. The control unit compares the first stroke estimate value with the first stroke instruction value, and generates a first control signal for switching the first alternating current switch based on the comparison result. In addition, the control unit receives a second stroke instruction value (xref2) and compares a second stroke estimate value (x2) calculated by the second stroke calculation unit 623 with the second stroke instruction value. The control unit compares the second stroke estimate value with the second stroke instruction value, and generates a second control signal for controlling the second alternating current switch based on the comparison result. The compressor control apparatus typically performs sensorless control, and the detailed description thereof will be omitted.

**[0048]** The first and the second load detection unit 61, 62 may detect a load on the first compressor (C1) and the second compressor (C2), respectively, using the motor drive current, the motor drive voltage, or the first and the second stroke. The control unit 50 independently operates the first compressor and the second compressor based on the load on the first compressor and the second compressor detected through the first load detection unit and the second load detection unit.

**[0049]** The size of the compressor load may be detected using a phase difference between the motor current and stroke estimate value, and a phase difference between the motor voltage and stroke estimate value. Furthermore, the size of the compressor load may be detected using a gas spring constant (Kg). In addition, the size of the compressor load may be detected using a gas damping constant (Cg).

**[0050]** Referring to FIG. 4, a compressor control method according to an embodiment controls the first compressor and the second compressor using a first and a second alternating current (AC) switch, respectively. The compressor control method may include receiving a compressor operation mode (S10), and driving the first and the second alternating current switch in a simultaneous manner or driving the first alternating current switch or the second alternating current switch in a separate manner based on the compressor operation mode (S21 and below). Here, the compressor operation mode is a mode determined by a load or required freezing capacity of the first and the second compressor. The compressor operation mode may control a compression amount or the like of each compressor, but merely divided into a mode of operating only the first compressor, a mode of operating only the second compressor, and a mode of operating the first and the second compressor in a simultaneous manner. Hereinafter, the configuration of the apparatus refers to FIGS.

**[0051]** The driving step operates the first and the second compressor connected to the driven alternating current switch in a simultaneous manner or operates the first compressor or the second compressor in a separate manner (S33, S43, S53). Here, the operating step varies a firing angle of the first alternating current switch or second alternating current

## 6

switch based on a freezing capacity of the first compressor or the second compressor.

**[0052]** The first and the second alternating current switch are connected to the first and the second compressor, and then the compressor control apparatus receives a compressor operation mode (S10), and determines whether to operate only the first compressor, operate only the second compressor, or operate the first and the second compressor in a simultaneous manner (S21, S22, S23).

[0053] Referring to FIG. 2, when operating the first compressor, the compressor control apparatus compares a first stroke instruction value (xref1) with a first stroke estimate value (x1), and generates a first control signal for switching the first alternating current switch based on the comparison result (S31). In addition, when operating the second compressor, the compressor control apparatus compares a second stroke instruction value (xref2) with a second stroke estimate value, and generates a second control signal for switching the second alternating current switch based on the comparison result (S41). When the first and the second compressor are operated in a simultaneous manner, the compressor control apparatus generates the first and the second control signal to the first and the second alternating current switch, respectively (S51).

[0054] The compressor control apparatus varies a firing angle of the first alternating current switch or second alternating current switch based on a freezing capacity of the first and the second compressor, and performs voltage control for the compressor based on this. Referring to FIG. 3A, when a gate voltage is received at the first alternating current switch based on a first control signal, the first alternating current switch supplies a first motor drive current to the first compressor). Similarly, referring to FIG. 3B, when a gate voltage is received at the second alternating current switch based on a second control signal, the second alternating current switch supplies a second motor drive current to the second compressor (C2).

[0055] As described above, in a compressor control apparatus according to the embodiments of the present disclosure and a refrigerator including the same, the operation of two compressors may be controlled using an alternating current (AC) switch, thereby minimizing the use of elements as well as increasing the compressor capacity and enhancing the operation efficiency of a system. According to the embodiments of the present disclosure, a plurality of operation modes may be used to correspond to a load or freezing capacity using two compressors. Furthermore, according to the present disclosure, two compressors may be operated in a separate or simultaneous manner using two alternating current (AC) switches, thereby simplifying the configuration of a system to reduce the cost.

#### 30 Claims

10

20

35

40

1. A compressor control apparatus, comprising:

a first and a second alternating current (AC) switch (21,22) switched based on a first and a second control signal to drive a first and a second compressor (C1, C2); and a control unit (30) configured to generate the first and the second control signal based on a load of the first and the second compressor (C1, C2) to output them to the first and the second alternating current switch (21, 22), wherein the first and the second compressor (C1, C2) are operated in a simultaneous manner or the first compressor (C1) or the second compressor (C2) is operated in a separate manner.

- 2. The compressor control apparatus of claim 1, wherein the control unit (30) is adapted to vary a firing angle of the first alternating current switch (21) and/or second alternating current switch (22) based on a freezing capacity of the first compressor or second compressor (C1, C2).
- **3.** The compressor control apparatus of claim 1 or 2, further comprising:

an input voltage detection unit (40) configured to detect the power voltage of commercial alternating current (AC) power.

- 50 **4.** The compressor control apparatus as claimed in any one of the preceding claims, further comprising:
  - a direct current (DC) power supply unit (50) configured to convert the commercial alternating current power into direct current power to apply it to the first and/or the second alternating current switch (21, 22).
- 55 The compressor control apparatus as claimed in any one of the preceding claims, further comprising at least one of:

a first and a second current detection unit (611, 621) configured to detect a first and a second motor drive current applied to a first and a second motor provided in the first compressor and the second compressor (C1, C2),

respectively;

a first and a second voltage detection unit (612, 622) configured to detect a first and a second motor drive voltage applied to the first and the second motor.

- 5 **6.** The compressor control apparatus as claimed in any one of the preceding claims, further comprising:
  - a first and a second stroke calculation unit (613, 623) configured to calculate a first and second stroke of the first compressor and the second compressor, respectively.
- 7. The compressor control apparatus as claimed in any one of the preceding claims, wherein the first and a second stroke calculation unit (613, 623) is adapted to use the first and the second motor drive current and the first and the second motor drive voltage for calculating a first and second stroke of the first compressor and the second compressor, respectively.
- **8.** The compressor control apparatus as claimed in any one of the preceding claims, wherein the first compressor or the second compressor (C1, C2) is a reciprocating compressor.
  - **9.** A compressor control method for controlling a first compressor and a second compressor (C1, C2) using a first and a second alternating current (AC) switch (21,22), respectively, the method comprising:

receiving (S10) a compressor operation mode; and

driving (S32,S42, S52) the first and the second alternating current switch (21,22) in a simultaneous manner or driving the first alternating current switch or the second alternating current switch (21,22) in a separate manner based on the compressor operation mode.

25

35

20

- **10.** The method of claim 9, wherein said driving step operates the first and the second compressor (C1, C2) connected to the driven alternating current switch (21,22) in a simultaneous manner or operates the first compressor or the second compressor (C1, C2) in a separate manner.
- 30 11. The method of claim 10, wherein said operating step varies a firing angle of the first alternating current switch or second alternating current switch (21,22) based on a freezing capacity of the first compressor or the second compressor (C1, C2).
  - 12. The method as claimed in any one of the preceding claims 9-11,wherein the compressor operation mode is an operation mode determined by a load or required freezing capacity of the first and the second compressor (C1, C2), and comprises at least one of a mode of operating only the first compressor (C1), a mode of operating only the second compressor (C2), and a mode of operating the first and the second compressor (C1, C2) in a simultaneous manner.
- 40 **13.** A refrigerator (700), comprising:

a refrigerator body;

a first and a second compressor (C1, C2) provided in the refrigerator body to compress refrigerant, respectively; a first and a second alternating current switch (21, 22) switched based on a first and a second control signal to drive the first and the second compressor (C1, C2); and

a control unit (30) configured to generate the first and the second control signal based on a load of the first and the second compressor (C1, C2) and output them to the first and the second alternating current switch, wherein the first and the second compressor (C1, C2) are operated in a simultaneous manner or the first compressor (C1) or the second compressor (C2) is operated in a separate manner.

50

45

- **14.** The refrigerator of claim 13, wherein the control unit (30) is adapted to varys a firing angle of the first alternating current switch (21) or second alternating current switch (22) based on a freezing capacity of the first compressor or the second compressor (C1, C2).
- 15. The refrigerator of claim 13 or 14, further comprising:

an input voltage detection unit (40) configured to detect the power voltage of commercial alternating current (AC) power.

FIG. 1

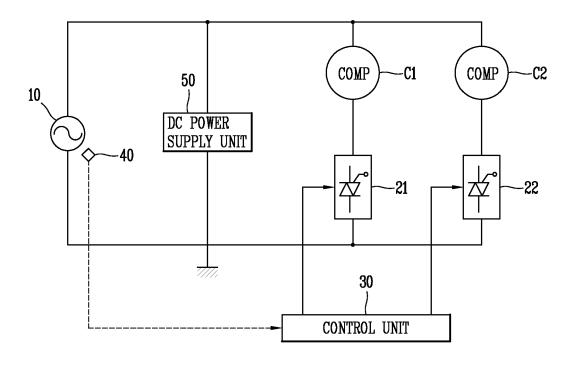


FIG. 2

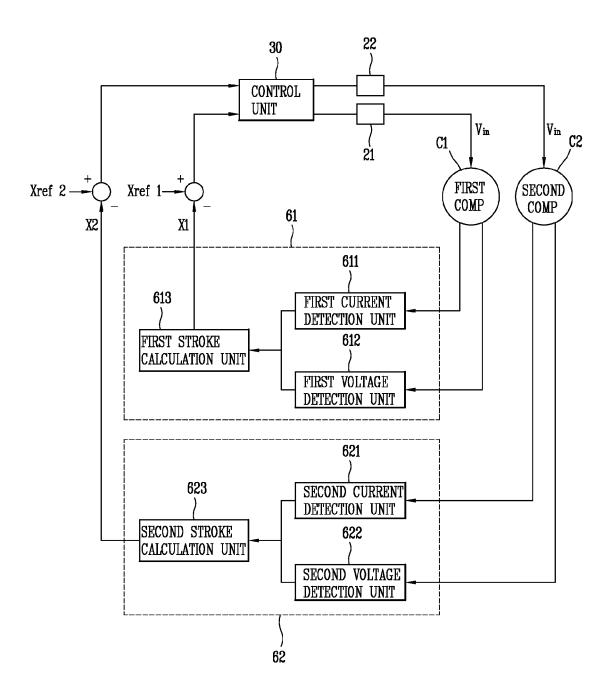


FIG. 3A

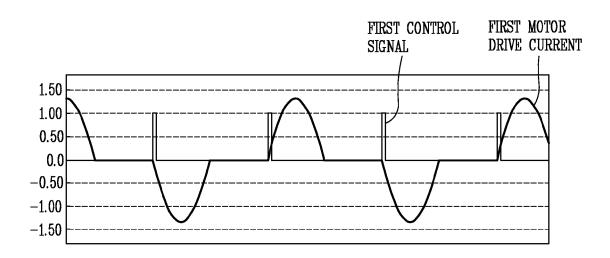


FIG. 3B

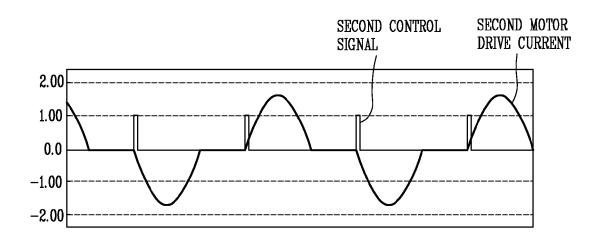


FIG. 4

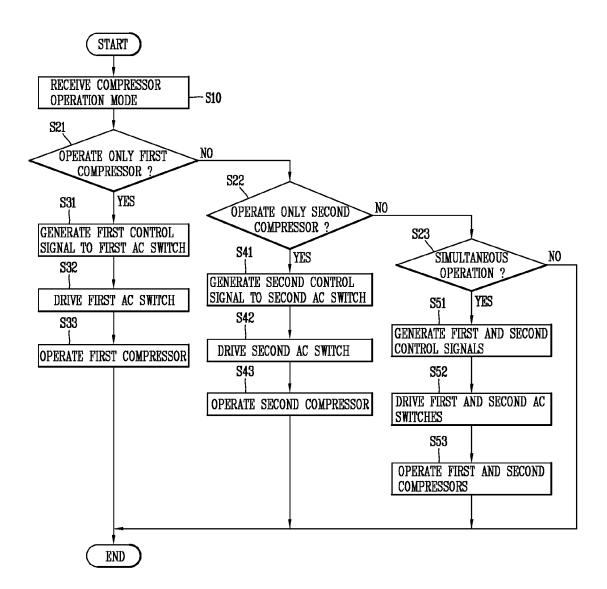


FIG. 5

