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#### (54) **BIDIRECTIONAL WRENCH**

(57) The bidirectional wrench of the present invention achieves two working mode and can convert between them conveniently. During the use of the bidirectional wrench of the present invention, the input torque that the operator exerts is a clockwise torque or an anticlockwise torque, the output torque of the output end of the bidirectional wrench of the present invention is a clockwise torque or an anticlockwise torque alternatively.

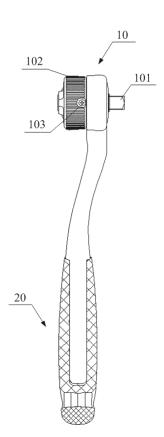


Fig. 1

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#### Field of the Invention

**[0001]** The present invention relates to a hand tool, particularly to a bidirectional wrench.

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#### **Background of the Invention**

[0002] During the using of common hand tools such as screwdrivers and torque wrenches, there is a movement limitation of the human hand in rotation direction, namely the inability of the human hand to turn continuously in one direction. The operation of such a tool which a rotation axis of the handle is coaxial with the tool's main shaft consists of a repetition of the following cycle: first, the hand rotates the handle in the desirable direction (e. g. tightening or loosening a screw); second, the hand rotates in the opposite direction to reposition the tool for next cycle. During the second portion of the above mentioned cycle, the hand's reversed rotation can be achieved by re-grasping the handle after releasing it, or by the tool which is equipped with an one-way means such as a ratchet surface to keep the main shaft stationary during the reversed rotation of the handle, or by reinserting the tool bit to the screw after extracting the bit from engagement with the screw. However, in any case, the hand's reversed rotation could not bring any effective advance of the fastener, and therefore it becomes a wasted movement.

[0003] U. S. Pat. No. 5,931,062 discloses a mechanical rectifier, which comprises a shaft; two driving element mounted on the shaft, each having a one-way clutch interposed between it and the shaft, with the clutches oriented in the same way on the shaft so that the shaft is always entrained in only one direction of rotation when either one of the two driving elements is rotated in that direction, and the shaft is overrun by a driving element that is rotated in the opposite direction; a rotation means positioned along the axis of the shaft and engaging a selected one of the driving elements; and a reversing mechanism coupling the two driving elements together and forcing them to always rotate in the opposite directions so that one driving element entrains the shaft and the other driving element overruns the shaft, thus causing the shaft to always turn in only one direction, regardless of the direction of rotation of the driving elements, so that transfers the bidirectional rotations of the rotation means (e.g. a handle) into the unidirectional rotation of the shaft. The mechanical rectifier can efficiently utilize the rotations of the rotation means in either way, i.e. no matter the handle rotates clockwise or anticlockwise, the shaft rotates in the same direction, therefore it can improve the efficiency of the hand motion, and save operation time.

**[0004]** However, the converting mechanism of the invention can only make the shaft rotate in on direction, which could not meet the requirement of rotating the shaft

in two directions, such as, tightening or loosening a fastener in the application of a torque wrench, the torque wrench equipped with the converting mechanism of the invention could only get the result of tightening a fastener (or loosening a fastener) no matter what operation it executes, either tightening or loosening a fastener, as the conventional wrenches do. If you expect the torque wrench equipped with the converting mechanism of the invention could executes the operation of tightening and loosening a fastener, the considerable solution can only be that the two ends of the shaft of the torque wrench are both mounted with output, and one end is to execute the operation of tightening a fastener, the other to execute the operation of tightening a fastener. But this design is cumbersome, it is inconvenient to choose appropriate output end when using the torque wrench.

**[0005]** Therefore, it is desired to develop a bidirectional wrench which is capable of switching the rotation direction of the shaft conveniently.

#### **Summary of the Invention**

**[0006]** In the view of the above, the technical object of the present invention is to provide a bidirectional wrench which can switch the rotation direction of the main shaft conveniently.

**[0007]** For the above purpose, the present invention provides a bidirectional wrench, and comprises: a working part and a handle, the working part consisting of a main shaft, the main shaft being used to output torque and its axis being perpendicular to the handle; a capstan gear, a follower gear, an transmission seat and an idle gear, the capstan gear, the follower gear and the transmission seat being all mounted on the main shaft, the axis of the transmission seat being perpendicular to the axis of the main shaft, the idle gear being mounted on the transmission seat and rotating between the capstan gear and the follower gear, wherein the handle entrains the capstan gear to rotate, and the transmission seat is equipped with a holding device, when holding the holding device and rotating the handle to entrain the capstan gear, the capstan gear entraining the follower gear to rotate reversely via the idle gear; a first ratchet surface rotating together with the capstan gear, and a second ratchet surface rotating together with the follower gear; a first pawl element and a second pawl element which are capable of driving the main shaft to rotate; wherein the first pawl element has a first pawl and a second pawl which are matched with the first ratchet selectively, wherein the first pawl skids on the first ratchet surface in the first direction but engages with the first ratchet surface in the second direction, and the second pawl engages with the first ratchet surface in the first direction but skids on the first ratchet surface in the second direction; the second pawl element has a third pawl and a forth pawl which is matched with the second ratchet selectively, wherein the third pawl skids on the first ratchet surface in the first direction but engages with the first ratchet sur-

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face in the second direction, and the forth pawl engages with the first ratchet surface in the first direction but skids on the first ratchet surface in the second direction; a reversing switch, which is capable of setting the first pawl element and the second pawl element on a first condition and a second condition, the first pawl and the third pawl are matched with the first ratchet surface and the second ratchet respectively under the first condition; the second pawl and the forth pawl are matched with the first ratchet surface and the second ratchet respectively under the second condition; the first direction is clockwise or anticlockwise, the second direction is opposite to the first direction.

**[0008]** In a further embodiment, the handle has a ring-shaped head, and the first ratchet surface is disposed on the inner circumference of the ring-shaped head.

**[0009]** In a further embodiment, the first ratchet is disposed on the inner circumference of the capstan gear.

[0010] In a further embodiment, the second ratchet is disposed on the inner circumference of the follower gear.
[0011] In a further embodiment, the holding device is a holding ring.

[0012] In the further embodiment, the first pawl element and/or the second pawl element are/is fan-shaped.
[0013] In a further embodiment, the first pawl element and the second pawl element are mounted on the countershaft, the axis of the countershaft is parallel to but not overlap in the main shaft, and the countershaft engages with the main shaft and is capable of entraining the main shaft to rotate.

**[0014]** In a further embodiment, the countershaft drills through the main shaft.

**[0015]** In a further embodiment, the reversing switch comprises a newel, a first spring-loaded plunger and a second spring-loaded plunger, the newel is disposed inside the main shaft, the first spring-loaded plunger and the second spring-loaded plunger is fix on the newel in turn, and the first spring-loaded plunger and the second spring-loaded plunger matches with the respectively.

**[0016]** In a further embodiment, springs are disposed inside the first spring-loaded plunger and the second spring-loaded plunger.

[0017] The bidirectional wrench of the present invention achieves two working modes between which the wrench can convert conveniently. During the use of the bidirectional wrench of the present invention, the input torque that the operator exerts is a clockwise torque or an anticlockwise torque, and the output torque of the output end of the bidirectional wrench of the present invention is a clockwise torque or an anticlockwise torque alternatively.

**[0018]** The present invention would be described in detail hereinafter in combination with the attached drawings for better understanding the purpose, features and effects of the present invention.

#### **Brief Description of the Drawings**

#### [0019]

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Figure 1 is a front view of the bidirectional wrench of the present invention;

Figure 2 is a sectional view of the bidirectional wrench in Figure 1;

Figure 3 shows the driving mechanism in the bidirectional wrench of the present invention;

Figure 4 shows the reversing mechanism in the bidirectional wrench of the present invention;

Figure 5 shows the main shaft mounted first pawl element and second pawl element;

Figure 6 is a front view of the first pawl element in Figure 5;

Figure 7 shows cooperation between the first pawl element and the first ratchet surface when the bidirectional wrench of the present invention is on the first working mode;

Figure 8 shows the knob for changing working mode of the bidirectional wrench of the present invention;

Figure 9 shows the blocking device in the bidirectional wrench of the present invention;

Figure 10 is a side view of the blocking device in Figure 5.

#### <u>Detailed Description of the Preferred Embodiments</u>

**[0020]** As shown in Figure 1 and 2, the bidirectional wrench, which refers to one detailed embodiment of the present invention, comprises a handle 20 and a working part 10, the handle 20 is socketed to the working part 10 across the ring-shaped head 21 (see Figure 3) through longitudinal extension. Inside the working part 10 is main shaft 100, outside is a holding ring. One end of the main shaft is an output end 101 which extends beyond the working part 10 and the head 21 of the handle 20. The output end 101 can be a component, which is suitable to operate various fasteners like quoining screw, by mounting different sleeves.

**[0021]** The bidirectional wrench of the present invention consists of a driving mechanism coupling with a reversing mechanism, the input torque from the handle 20 is transmitted to the main shaft 100 of the working part 10 through the driving mechanism, and make the direction of the output torque from the output end 101 be the first direction or the second direction, wherein the first direction and the second direction are opposite. Such as,

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when the input torque of the working part 10 is a clockwise torque or an anticlockwise torque, the output torque of the output end 101 is a clockwise torque; or, when the input torque of the working part 10 is a clockwise torque or an anticlockwise torque, the output torque of the output end 101 is an anticlockwise torque.

[0022] The structure of the driving mechanism of the bidirectional wrench of the present invention is shown in Figure 3, consists of a first ratchet surface 311, a capstan gear 312, a second ratchet surface 321, a follower gear 322, a transmission seat 330, an idle gear 331 and 332. Wherein, the first ratchet surface 311 and the capstan gear 312 connect and be coaxial with each other, in this embodiment, the first ratchet surface 311 is disposed in the inner circumference of the ring-shaped head 21 of the handle 20, the driving engages with the head 21 of the handle 20, thus the head 21 will drive driving gear to rotate when the handle 20 rotates, in other embodiment, the first ratchet surface 311 can be disposed in the inner circumference of capstan gear 312; the second ratchet surface 321 can be disposed in the inner circumference of follower gear 322. The faces of the first ratchet surface 311 and the second ratchet surface 321 connect with the outer face of the main shaft 100; the capstan gear 312 and the follower gear 322 are face-gears, faces of the capstan gear 312 and the follower gear 322 are face to face. The first ratchet 311, the second ratchet 321, the capstan gear 312 and the follower gear 322 are coaxial and the center axes thereof overlap in that of the main shaft 100.

[0023] The transmission seat 330 and the holding ring 102 are fixed together. The idle gear 331, 332 are mounted on the transmission seat 330, which is perpendicular to the main shaft 100. The idle gear 3331, 333 are matched between the capstan gear 312 and the follower gear 322, their tooth engage with the tooth of the capstan gear 312 and the follower gear 322 to drive respectively, when the holding ring 102 is fixed, or the transmission seat 330 is fixed, the capstan gear 312 will drive the follower gear 322 to rotate via the idle gear 331, 331. In this embodiment, the idle gear 331, 332 are angle gears.

[0024] The structure of the reversing mechanism of the bidirectional wrench of the present invention is shown in Figure 4, comprises a newel, a reversing switch consisting of a first spring-loaded plunger 221 and a second spring-loaded plunger 222, a first pawl element 211 and a second pawl element 212. Wherein, the newel 220 is fixed in the main shaft 100, the first spring-loaded plunger 221 and the second spring-loaded plunger 222 are fixed on the newel 220 in turn, the first spring-loaded plunger 221 and the second spring-loaded plunger 222 are perpendicular to the main shaft 110 along the active direction, preferably, the first spring-loaded plunger 221 and the second spring-loaded plunger 222 have elastic elements such as a spring. The first pawl element 221 and second pawl element 222 are fixed on the main shaft 100 across the countershaft 210, as shown in Figure 5, the countershaft 210 is parallel to the center axis of the main

shaft 100 but it does not overlap in it, the pawl element 221 and the second pawl element 222 can rotate round the countershaft 210.

[0025] The first pawl element 211 and the first pawl element 212 have a similar structure, namely a first fanshaped pawl, a second fan-shaped pawl and a fanshaped space between them. Take the first pawl element 211 for example, Figure 6 shows the top view of the first pawl element 211 (direction towards the output end 101 along the main shaft 100), as can be seen from Figure 6, the first pawl element 211 consists of the first fanshaped pawl 2111, the second fan-shaped pawl 2112 and the fan-shaped space between them. The fanshaped face of the first fan-shaped pawl 2111, the fanshaped space center section and the fan-shaped face of the second fan-shaped pawl 2112 constitutes the first surface of the first pawl element 211. The first pawl element 211 also has a second surface which is a special shaped surface and contains a concave section 2113, which has a first side wall 2114 and a second side wall 2115, in this embodiment. The first side wall 2114 and the second side wall 2115 extend along the main shaft 100. The first pawl element 211 has a hole 2101 which is matched with the countershaft 210, the countershaft 210 fixes the first pawl element 211 on the main shaft across the hole 2101 (see Figure 5). In this embodiment, the hole 2101 is arranged on the fan-shaped center section of the first pawl element 211, preferably, on the center of gravity of the first pawl element 211. The structure of the second pawl element 212 is similar to the first pawl element 211, no more description, its thickness is less than that of the first pawl element 211 in this embodiment, but in other embodiments, its thickness can be equal to or more than that of the first pawl element 211.

[0026] The first surface of the first pawl element 211 and the second pawl element 212 face to the first ratchet surface 311 and the second ratchet surface 321 respectively, specifically, the tooth of the fan-shaped pawl (which contains the first fan-shaped pawl 2111 and the second fan-shaped pawl 2112) of the first pawl element 211 face to the tooth of the first ratchet 311, the tooth of the fan-shaped pawl (contains the first fan-shaped pawl and the second fan-shaped pawl) of the second pawl element 212 face to the tooth of the second ratchet surface 321. The second surface of the first pawl element 211 and the second pawl element 212 face to the surface of the newel 220, specifically, the second surface of the first pawl element 211 faces to the ball-head section of the first spring-loaded plunger 221, the second surface of the second pawl element 212 faces to the ball-head section of the second spring-loaded plunger 222. Across the newel 220, the ball-head section of the first springloaded plunger 221 connects with the first side wall 2114 of the concave section 2113 of the first pawl element 211, the ball-head section of the second spring-loaded plunger 222 connects with the first side wall of the concave section of the second pawl element 212, when the bidirectional wrench of the present invention is on the first

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working mode; or, the ball-head section of the first spring-loaded plunger 221 connects with the second side wall 2115 of the concave section 2113 of the first pawl element 211, the ball-head section of the second spring-loaded plunger 222 connects with the second side wall of the concave section of the second pawl element 212, when the bidirectional wrench of the present invention is on the second working mode.

[0027] When the bidirectional wrench of the present invention is on the first working mode, see Figure 7, the tooth of the first fan-shaped pawl 2111 of the first pawl element 211 connects with the tooth of the first ratchet surface 311, similarly, the tooth of the first fan-shaped pawl of the second pawl element 212 connects with the tooth of the second ratchet surface 321. When the head 21 of the handle 20 drives the first ratchet surface 311 to rotate, when the moving direction of the first ratchet surface 311 beside the first fan-shaped pawl 2111 is from the first fan-shaped section 2111 to the second fanshaped section 2112, namely the first ratchet surface 311 rotates clockwise seen in the Figure 7, since the ballhead section of the first spring-loaded plunger 211 connects with the first side wall 2114 of the concave section 2113 of the first pawl element 211, the first ratchet surface 311 cannot drive the first pawl element 211 to rotate, namely the tooth of the first fan-shaped pawl 2111 does not engage with the tooth of the first ratchet surface 311; but when the moving direction of the first ratchet surface 311 beside the first fan-shaped pawl 2111 is from the second fan-shaped section 2112 to the first fan-shaped section 2111, namely the first ratchet surface 311 rotates anticlockwise seen in the Figure 7, since the ball-head section of the first spring-loaded plunger 211 connects with the first side wall 2114 of the concave section 2113 of the first pawl element 211, the first ratchet surface 311 can drive the first pawl element 211 to rotate, namely the tooth of the first fan-shaped pawl 2111 engages with the tooth of the first ratchet surface 311. And the rotation of the first pawl element 211 is transferred to the countershaft 210 through the main shaft 100, thus driving the main shaft 100 to rotate.

[0028] Meanwhile, when the moving direction of the second ratchet surface 321 beside the first fan-shaped pawl of the second pawl element 212 is from the first fanshaped section to the second fan-shaped section in the second pawl element 212, namely the second ratchet surface 321 rotates clockwise, since the ball-head section of the second spring-loaded plunger 222 connects with the first side wall of the concave section of the second pawl element 212, the second ratchet surface 321 cannot drive the second pawl element 212 to rotate, namely the tooth of the first fan-shaped pawl of the second pawl element 212 does not engage with the tooth of the second ratchet surface 312; but when the moving direction of the second ratchet surface 321 beside the first fan-shaped pawl of the second pawl element 212 is from the second fan-shaped section to the first fan-shaped section in the second pawl element 212, namely the second ratchet

surface 321 rotates anticlockwise, since the ball-head section of the second spring-loaded plunger 222 connects with the first side wall of the concave section of the second pawl element 212, the second ratchet surface 321 can drive the second pawl element 212 to rotate, namely the tooth of the first fan-shaped pawl of the second ratchet surface 321 engages with the tooth of the second ratchet surface 321. And the rotation of the second pawl element 212 is transferred to the countershaft 210 through the main shaft 100, thus driving the main shaft 100 to rotate.

[0029] Since the drive among the idle gear 331, 332, the capstan gear 312 and the follower gear 322, when the holding ring is fixed, the rotation direction of the second ratchet surface 321 is opposite to the first ratchet surface 311. It can be seen from this, when the bidirectional wrench of the present invention is on the first working mode, when the input torque from the working part 10 is a clockwise torque, it makes the first ratchet surface 311 rotate clockwise and the second ratchet surface 321 rotate anticlockwise, the first pawl element 211 does not engage with the first ratchet surface 311 and the second pawl element 212 engages with the second ratchet surface 321 at the moment, thus the second pawl element 212 drives the main shaft 100 to rotate anticlockwise and the output torque is an anticlockwise torque; when the input torque from the working part 10 is an anticlockwise torque, it makes the first ratchet 311 rotate anticlockwise and the second ratchet surface 321 rotate clockwise, the first pawl element 211 engages with the first ratchet surface 311 and the second pawl element 212 does not engage with the second ratchet surface 321 at the moment, thus the first pawl element 211 drives the main shaft 100 to rotate anticlockwise and the output torque is an anticlockwise torque.

[0030] When the bidirectional wrench of the present invention is on the second working mode, the tooth of the second fan-shaped pawl 2112 of the first pawl element 211 connects with the tooth of the first ratchet surface 311, similarly, the tooth of the second fan-shaped pawl of the second pawl element 212 connects with the tooth of the second ratchet surface 321. When the head 21 of the handle 20 drives the first ratchet surface 311 to rotate, when the moving direction of the first ratchet surface 311 beside the second fan-shaped pawl 2112 is from the first fan-shaped section 2111 to the second fanshaped section 2112, namely the first ratchet surface 311 rotates clockwise, since the ball-head section of the first spring-loaded plunger 211 connects with the second side wall 2115 of the concave section 2113 of the first pawl element 211, the first ratchet surface 311 can drive the first pawl element 211 to rotate, namely the tooth of the second fan-shaped pawl 2112 engages with the tooth of the first ratchet surface 311; and the rotation of the first pawl element 211 is transferred to the main shaft 100 through the countershaft 210, thus driving the main shaft 100 to rotate; but when the moving direction of the first ratchet surface 311 beside the second fan-shaped pawl

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2112 is from the second fan-shaped section 2112 to the first fan-shaped section 2111, namely the first ratchet surface 311 rotates anticlockwise, since the ball-head section of the first spring-loaded plunger 211 connects with the second side wall 2115 of the concave section 2113 of the first pawl element 211, the first ratchet surface 311 cannot drive the first pawl element 211 to rotate, namely the tooth of the second fan-shaped pawl 2112 does not engage with the tooth of the first ratchet surface 311.

[0031] Meanwhile, when the moving direction of the second ratchet surface 321 beside the second fanshaped pawl of the second pawl element 212 is from the first fan-shaped section to the second fan-shaped section in the second pawl element 212, namely the second ratchet surface 321 rotates clockwise, since the ball-head section of the second spring-loaded plunger 222 connects with the second side wall of the concave section of the second pawl element 212, the second ratchet surface 321 can drive the second pawl element 212 to rotate, namely the tooth of the second fan-shaped pawl of the second pawl element 212 engages with the tooth of the second ratchet surface 312; and the rotation of the second pawl element 212 is transferred to the main shaft 100 through the countershaft 210, thus driving the main shaft 100 to rotate. But when the moving direction of the second ratchet surface 321 beside the second fanshaped pawl of the second pawl element 212 is from the second fan-shaped section to the first fan-shaped section in the second pawl element 212, namely the second ratchet surface 321 rotates anticlockwise, since the ballhead section of the second spring-loaded plunger 222 connects with the second side wall of the concave section of the second pawl element 212, the second ratchet surface 321 cannot drive the second pawl element 212 to rotate, namely the tooth of the second fan-shaped pawl of the second ratchet surface 321 does not engage with the tooth of the f second ratchet surface 321.

[0032] Since the drive among the idle gear 331, 332, the driving gear 312 and the driven gear 322, when the holding ring is fixed, the rotation direction of the second ratchet surface 321 is opposite to the first ratchet surface 311. It can be seen from this, when the bidirectional wrench of the present invention is on the second working mode, when the input torque from the working part 10 is a clockwise torque, it makes the first ratchet surface 311 rotate clockwise and the second ratchet surface 321 rotate anticlockwise, the first pawl element 211 engages with the first ratchet surface 311 and the second pawl element 212 does not engage with the second ratchet surface 321 at the moment, thus the first pawl element 211 drives the main shaft 100 to rotate clockwise and the output torque is a clockwise torque; when the input torque from the working part 10 is an anticlockwise torque, it makes the first ratchet 311 rotate anticlockwise and the second ratchet surface 321 rotate clockwise, the first pawl element 211 does not engage with the first ratchet surface 311 and the second pawl element 212 engages

with the second ratchet surface 321 at the moment, thus the second pawl element 212 drives the main shaft 100 to rotate clockwise and the output torque is a clockwise torque.

[0033] As previously mentioned, the first working mode and the second working of the bidirectional wrench of the present invention can be switched and selected via the newel 220. To be convenient, in this embodiment, as shown in Figure 8, the first end of the newel 220 has a knob 223, which would be matched with the newel 220 by embedding the two ears (ear 2201 in Figure 8) of the newel 220 into the knob 223. In this way, the newel 220 will rotate when turning the knob 223. In this embodiment, two spines protrude out of the surface of the knob 223, such as spine 2231, turning of the knob 223 can be achieved by putting rotating torque on the two spines including the spine 2231.

**[0034]** The bidirectional wrench of the present invention also contains a blocking device, which keeps the bidirectional wrench of the present invention on the working mode selected until the operator switch it to the other mode artificially. In Figure 9 and 10, the blocking device in this embodiment consists of a ball 400 disposed between the output end 101 and the second end of the newel 220, a groove matched with the ball 400 on the second end of the newel 220, specifically, the first groove 410 and the second groove 421. The first groove 410 and the second groove 421 are parallel with each other and are separated by the smooth spine.

[0035] When the ball 400 is in the first groove 410, the head working part of the first spring-loaded plunger 221 and the second spring-loaded plunger 222 keeps connecting with the first side wall of the concave section of the first pawl element 211 and the second pawl element 222 respectively, namely the first working mode of the bidirectional wrench of the present invention; when the ball 400 is in the second groove 420, the head working part of the first spring-loaded plunger 221 and the second spring-loaded plunger 222 keeps connecting with the second side wall of the concave section of the first pawl element 211 and the second pawl element 222 respectively, namely the second working mode of the bidirectional wrench of the present invention. When turning the knob 223 to rotate the newel 220 and let the ball 400 move from the first groove 410 to the second groove 420, the bidirectional wrench of the present invention turns from the first working mode to the second working mode; when turning the knob 223 to rotate the newel 220 and let the ball 400 move from the second groove 420 to the first groove 410, the bidirectional wrench of the present invention turns from the second working mode to the first working mode. In this embodiment, the transmission seat 330 engages with the holding ring 102 constantly, the transmission seat 330 is fixed with respect to the holding ring 102, thus when the working part 20 rotates with respect to the holding ring 102, the idle gear 331, 332 makes the follower gear 322 and the capstan 312 rotate in the opposite direction. In use, to keep the idle gear

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331, 332 working and make sure that the second ratchet surface 321 and the first ratchet surface 311 rotate in the opposite direction, operator can orientate the transmission seat 330 by holding the holding ring 102, thus the capstan gear 312 drives the idle gear 331, 331 to rotate, and then drives the follower gear 322 to rotate, thereby making the second ratchet surface 321 and first ratchet surface 311 rotate in the opposite direction. It should be noted that in other embodiments of the present invention, other methods can also be taken to position the transmission seat 330 and thus drive the idle gear 331, 332 to work.

[0036] In addition, as described previously, the output end 101 of the bidirectional wrench of the present invention can be a component, which is suitable to operate various fasteners such as quoining screw, by mounting various sleeves, and the ball 400 in the blocking device can also be used to block the various sleeves which are mounted on the output end 101 at this moment.

[0037] What stated above described the preferred embodiment in detail. It should be understood that the ordinary skills in the art can make many modifications and variations according to the present invention without any creative work. Therefore, any modification, equivalent replacement and improvement made to the present invention without going beyond the spirit and principle of the present invention shall be within the scope of the appended claims.

#### Claims

- A bidirectional wrench, comprising a working part and a handle, characterized in that the working part comprises
  - a main shaft, which is used to output torque and whose axis is perpendicular to the handle;
  - a capstan gear, a follower gear, an transmission seat and an idle gear, the capstan gear, the follower gear and the transmission seat being all mounted on the main shaft, the axis of the transmission seat being perpendicular to the axis of the main shaft, and the idle gear being mounted on the transmission seat and rotating between the capstan gear and the follower gear, wherein

the handle entrains the capstan gear to rotate, and the transmission seat is equipped with a holding device, when holding the holding device and rotating the handle to entrain the capstan gear, the capstan gear entraining the follower gear to rotate reversely via the idle gear;

- a first ratchet surface rotating together with the capstan gear, and a second ratchet surface rotating together with the follower gear;
- a first pawl element and a second pawl element which are capable of driving the main shaft to rotate; wherein

the first pawl element has a first pawl and a second

pawl which are matched with the first ratchet selectively, wherein the first pawl skids on the first ratchet surface in the first direction but engages with the first ratchet surface in the second direction, and the second pawl engages with the first ratchet surface in the first direction but skids on the first ratchet surface in the second direction;

the second pawl element has a third pawl and a forth pawl which are matched with the second ratchet selectively, wherein the third pawl skids on the first ratchet surface in the first direction but engages with the first ratchet surface in the second direction, and the forth pawl engages with the first ratchet surface in the first direction but skids on the first ratchet surface in the second direction:

a reversing switch, which is capable of setting the first pawl element and the second pawl element on a first condition and a second condition, the first pawl and the third pawl being matched with the first ratchet surface and the second ratchet respectively under the first condition; the second pawl and the forth pawl being matched with the first ratchet surface and the second ratchet respectively under the second condition;

the first direction being clockwise or anticlockwise, and the second direction being counter to the first direction.

- The bidirectional wrench as claimed in claim 1, wherein, the handle has a ring-shaped head, and the first ratchet surface is disposed on the inner circumference of the ring-shaped head.
- **3.** The bidirectional wrench as claimed in claim 1, wherein, the first ratchet is disposed on the inner circumference of the capstan gear.
- **4.** The bidirectional wrench as claimed in claim 1, wherein, the second ratchet is disposed on the inner circumference of the follower gear.
- **5.** The bidirectional wrench as claimed in claim 1, wherein, the holding device is a holding ring.
- 45 6. The bidirectional wrench as claimed in claim 1, wherein, the first pawl element and/or the second pawl element are/is fan-shaped.
  - 7. The bidirectional wrench as claimed in claim 1, wherein, the first pawl element and the second pawl element are mounted on the countershaft, the axis of the countershaft is parallel to but not overlap in the main shaft, and the countershaft engages with the main shaft and is capable of entraining the main shaft to rotate.
  - 8. The bidirectional wrench as claimed in claim 7, wherein, the countershaft drills through the main

shaft.

9. The bidirectional wrench as claimed in claim 1, wherein, the reversing switch comprises a newel, a first spring-loaded plunger and a second spring-loaded plunger, the newel being disposed inside the main shaft, the first spring-loaded plunger and the second spring-loaded plunger being fix on the newel in turn, and the first spring-loaded plunger and the second spring-loaded plunger being matched with the respectively.

**10.** The bidirectional wrench as claimed in claim 9, wherein, springs are disposed inside the first springloaded plunger and the second spring-loaded plunger.

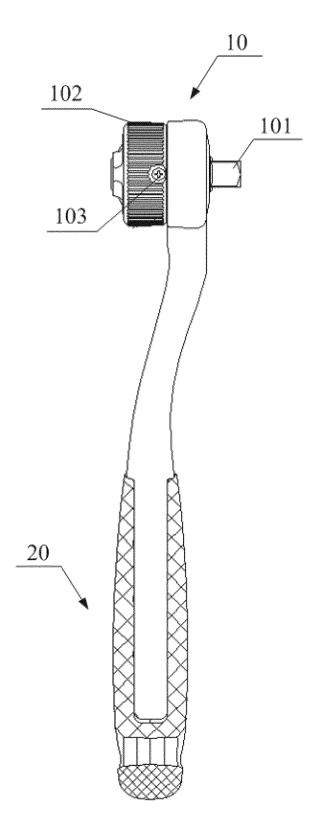


Fig. 1

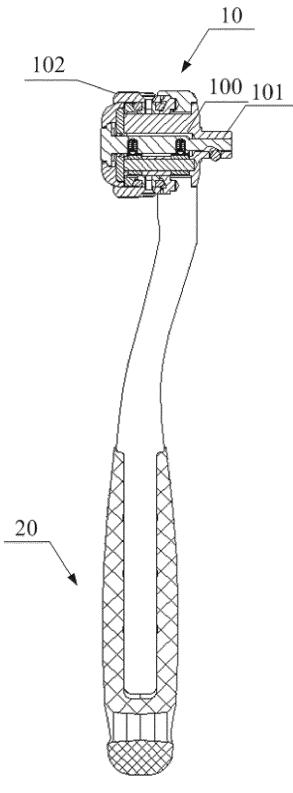
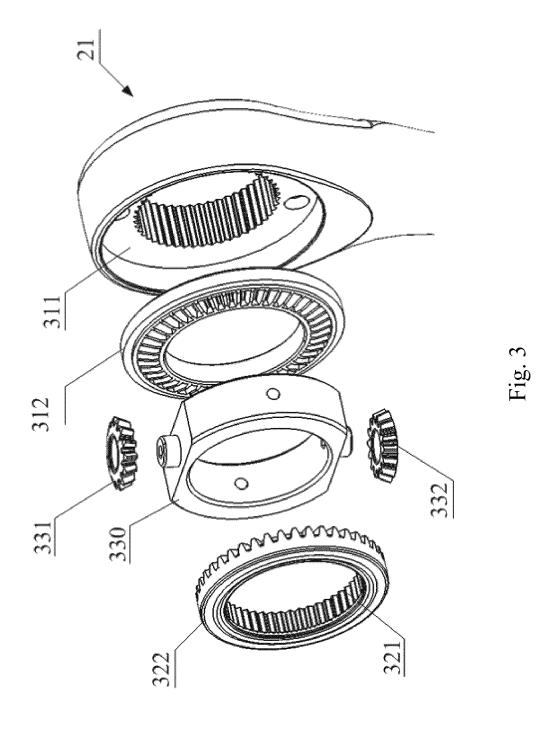
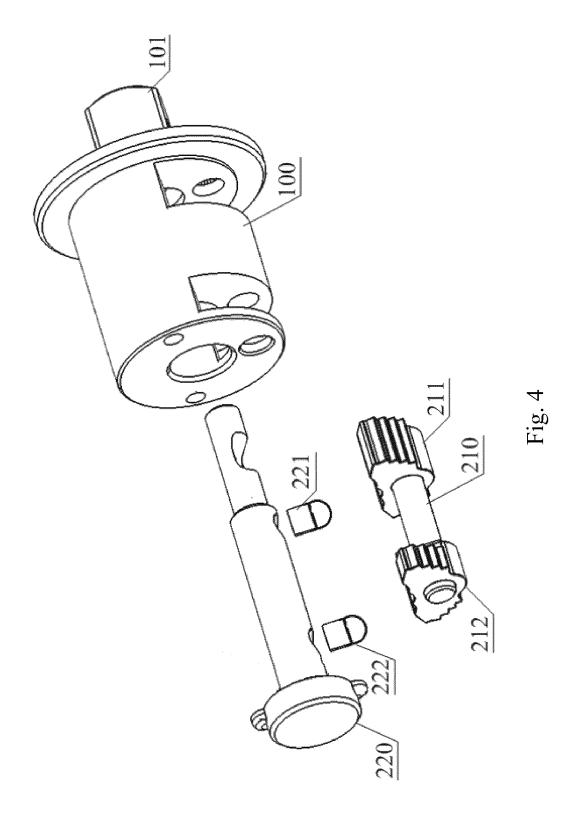


Fig. 2





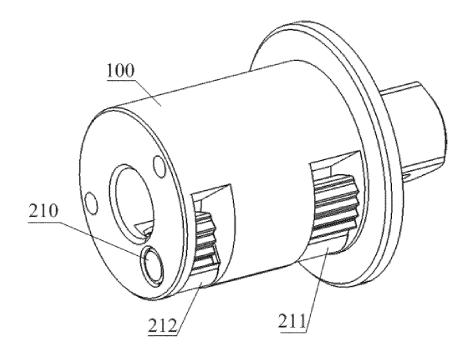


Fig. 5

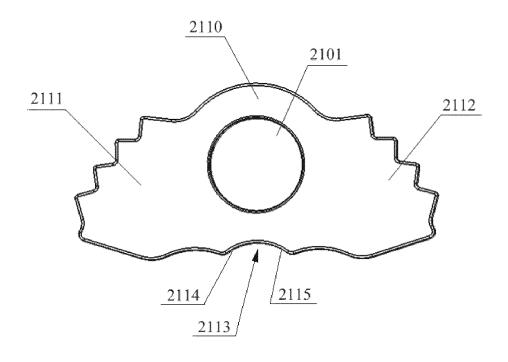


Fig. 6

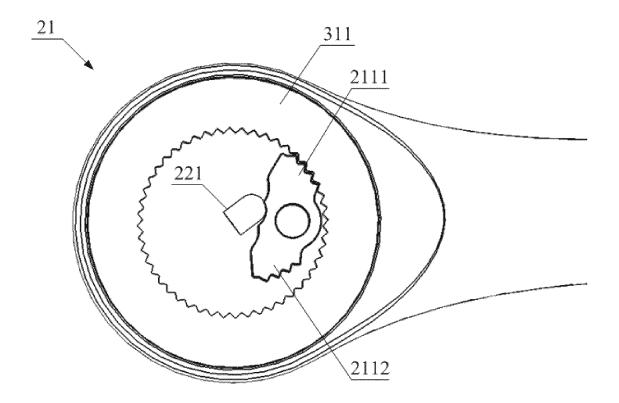


Fig. 7

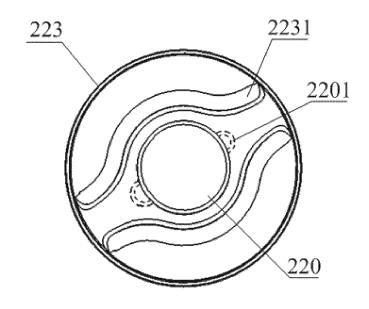


Fig. 8

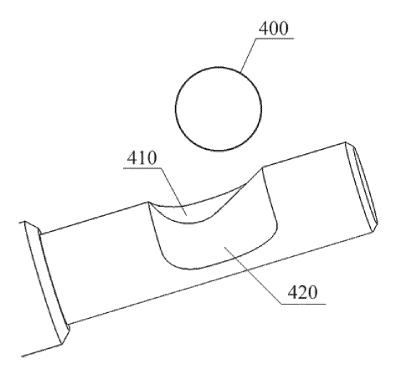
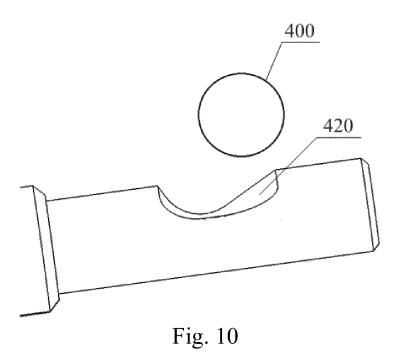


Fig. 9



## INTERNATIONAL SEARCH REPORT

International application No.

PCT/CN2013/070641

5	A. CLASS	CLASSIFICATION OF SUBJECT MATTER								
		B25B 13/4	6 (200	6.01) i						
	According to	International Patent Classification (IPC) or to both na	ational	classification and IPC						
10	B. FIELDS	S SEARCHED								
	Minimum documentation searched (classification system followed by classification symbols)									
		IPC: B2	5B, B6	2M						
15	Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched									
	Electronic da	ata base consulted during the international search (nam	ne of da	ata base and, where practicable, sear	ch terms used)					
	WPI EPODOC CNKI CNPAT: pipe vice, claw, reversing, Spanner, wrench, screwdriver, ratchet, gear, pawl, reversible, bidirection,									
20	exchange, tra	ansform, transition								
	C. DOCUMENTS CONSIDERED TO BE RELEVANT									
	Category*	Citation of document, with indication, where ap	propri	iate, of the relevant passages	Relevant to claim No.					
25	A	GB 2041272 A (PENNER J.), 10 September 1980 (1 60 to page 2, line 32, and figure 5	0.09.1	980), see description, page 1, line	1-10					
	A	US 6101901 A (STANLEY WORKS), 15 August 20	00 (15	6.08.2000), the whole document	1-10					
	A	US 7587961 B1 (WILLIAM TOOLS CO., LTD.), 1: whole document	5 Septe	ember 2009 (15.09.2009), the	1-10					
30	A	US 6789447 B1 (ZINCK, F.L.), 14 September 2004 (14.09.2004), the whole document			1-10					
	A	CN 2248119 Y (TANG, Zhenghan), 26 February 199	97 (26.	02.1997), the whole document	1-10					
	A	CN 1202438 A (JIA, Junying), 23 December 1998 (	23.12.1	1998), the whole document	1-10					
	A	CN 2485095 A (CHEN, Mulin), 10 April 2002 (10.0	4.2002	2), the whole document	1-10					
35	☐ Furthe	☐ Further documents are listed in the continuation of Box C. ☐ See patent family annex.								
	"A" docum	ial categories of cited documents: nent defining the general state of the art which is not ered to be of particular relevance	"T"	later document published after the or priority date and not in conflict cited to understand the principle of invention	with the application but					
40		application or patent but published on or after the ational filing date	"X"	document of particular relevance; cannot be considered novel or cannot an inventive step when the docume	be considered to involve					
	which	ent which may throw doubts on priority claim(s) or is cited to establish the publication date of another n or other special reason (as specified)	"Y"	•	the claimed invention inventive step when the					
45	"O" docum	nent referring to an oral disclosure, use, exhibition or means		documents, such combination bein skilled in the art	g obvious to a person					
		nent published prior to the international filing date er than the priority date claimed	"&"	document member of the same par	tent family					
	Date of the a	ctual completion of the international search	Date	of mailing of the international search	ch report					
50		17 September 2013 (17.09.2013)		10 October 2013 (10.1	10.2013)					
	State Intelle No. 6, Xitud Haidian Dis	ailing address of the ISA/CN: ectual Property Office of the P. R. China cheng Road, Jimenqiao strict, Beijing 100088, China o.: (86-10) 62019451		orized officer  ZHU, Zhenyo phone No.: (86-10) 62085468	u					
55	Form PCT/ISA	√210 (second sheet) (July 2009)								

## INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No.

PCT	CN2	013/	070	641

			PCT/CN2013/070641
Patent Documents referred in the Report	Publication Date	Patent Family	Publication Date
GB 2041272 A	10.09.1980	DE 2951823 A	14.08.1980
		CA 1112913 A	24.11.1981
		GB 2041272 B	15.09.1982
US 6101901 A	15.08.2000	WO 0110601 A1	15.02.2001
		AU 6533500 A	05.03.2001
US 7587961 B1	15.09.2009	None	
US 6789447 B1	14.09.2004	WO 0220220 A	14.03.2002
		EP 1341646 A1	10.09.2003
		JP 2004508211 A	18.03.2004
CN 2248119 Y	26.02.1997	None	
CN 1202438 A	23.12.1998	None	
CN 2485095 A	10.04.2002	None	

Form PCT/ISA/210 (patent family annex) (July 2009)

#### EP 2 946 884 A1

#### REFERENCES CITED IN THE DESCRIPTION

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#### Patent documents cited in the description

• US 5931062 A [0003]