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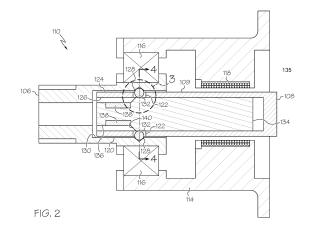
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(54) RESETTABLE ELECTRO-MECHANICALLY ACTUATED CONNECTION UNIT FOR GENERATORS

(57)An electrical generating system for an aircraft may include a first shaft connected with an engine gearbox, a second shaft connected with a generator of the aircraft, and a connection unit in which the first and second shafts are selectively engageable with one another. The connection unit may include a solenoid coil, an armature co-axial with the first and the second shafts, a plurality of balls positioned around the armature. The armature may include a first cylindrical segment and a second cylindrical segment. The armature may be selectively movable between a first axial position and a second axial position. When the armature is in a first axial position, the balls are positioned to transmit torque forces between the first shaft and the second shaft. When the armature is in a second axial position, the first shaft and the second shaft are free to rotate independently.



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Description

BACKGROUND OF THE INVENTION

[0001] The present invention relates to electrical generators driven by aircraft engines, and more particularly, to protection systems for preventing damage to aircraft engine gearboxes and generators.

[0002] A typical aircraft generates on-board electrical power by utilizing auxiliary power from its engines. The engines have a take-off shaft that goes through a gearbox and then to an electric generator to produce electricity for the aircraft.

[0003] If for some reason the temperature in an electric generator is suddenly higher or lower than normal or if any other abnormality occurs, like bearing failure or winding shorts, there is a need to discontinue driving the generator so that damage to the generator and/or the engine gearbox is precluded or at least minimized. Such disconnection may occur through operation of a shear section in generator driver shaft. The shear section is designed such that the increased torque will eventually shear the shaft resulting in an interruption of power being transmitted from the jet engine gearbox to the generator.

[0004] Many newer aircraft now employ variable speed, variable frequency or DC generators. A significant issue with variable speed generators is that the shaft shear section needs to be sized for minimum speed, maximum load operation. When variable speed generators are running at high speed, drive torque is much reduced for the same maximum load power. So at high speed the rotating energy to operate (i.e., break) the shear section can be very high, leading to high vibration and consequential damage.

[0005] As can be seen, there is a need for an externally operable disconnect. Additionally it is desirable that the disconnect be resettable so it can be periodically tested and then reset.

SUMMARY OF THE INVENTION

[0006] In one aspect of the present invention, an electrical generating system for an aircraft may comprise: a first shaft drivably connected with an engine gearbox of the aircraft; a second shaft co-axial with the first shaft and drivably connected with a generator of the aircraft; and a connection unit in which the first and second shafts are selectively engageable with one another. The connection unit may comprise, a solenoid coil, an armature co-axial with the first and the second shafts, a plurality of balls positioned circumferentially around the armature in contact with the armature, wherein the armature includes a first cylindrical segment and a second cylindrical segment, wherein the armature is selectively movable between a first axial position and a second axial position responsively to actuation of the solenoid coil, wherein the first axial position of the armature positions the armature so that the balls are in contact with the first cylindrical segment, wherein the second axial position of the armature positions the armature so that the balls are in contact with the second cylindrical segment, wherein the balls engage with both the first and second shaft when the armature is in the first axial position so that the balls are positioned to transmit torque forces between the first shaft and the second shaft, wherein the balls engage with only one of the first or second shaft when the armature is in the second axial position so that the first shaft and the second shaft are free to rotate independently of one another.

[0007] In another aspect of the present invention, a system for selectively connecting a generator to an aircraft engine gearbox may comprise: a stub shaft interconnected with the engine gearbox; a generator shaft interconnected with the generator, the generator shaft being co-axial with the stub shaft and having an overlapping segment positioned within a hollow overlapping segment of the stub shaft; an armature co-axial with and positioned within a hollow segment of the generator shaft, the armature having a first cylindrical segment with a first outside diameter and a second cylindrical segment with a second outside diameter smaller than the first outside diameter; a solenoid coil surrounding at least a portion of the armature, a plurality of balls circumferentially surrounding the armature; wherein the armature is selectively movable between a first axial position and a second axial position responsively to actuation of the solenoid coil, wherein the first axial position of the armature positions the armature so that the balls are in contact with the first cylindrical segment, wherein the second axial position of the armature positions the armature so that the balls are in contact with the second cylindrical seg-

wherein the balls engage with both the stub shaft and the generator shaft when the armature is in the first axial position so that the balls are positioned to transmit torque forces between the stub shaft and the generator shaft, wherein the balls engage with only the generator shaft when the armature is in the second axial position so that the stub shaft and the generator shaft are free to rotate independently of one another.

[0008] In still another aspect of the present invention, a system for selectively connecting a generator to an aircraft engine gearbox may comprise: a stub shaft interconnected with the engine gearbox; a generator shaft interconnected with the generator, the generator shaft being co-axial with the stub shaft and having a hollow segment overlapping a portion of the stub shaft; a hollow armature co-axial with and surrounding the hollow segment of the generator shaft, the armature having a first cylindrical segment with a first inside diameter and a second cylindrical segment with a second inside diameter greater than the first inside diameter; a solenoid coil surrounding at least a portion of the armature, a plurality of balls circumferentially surrounding the stub shaft and contacting the armature; wherein the armature is selectively movable between a first axial position and a second

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axial position responsively to actuation of the solenoid coil, wherein the first axial position of the armature positions the armature so that the balls are in contact with the first cylindrical segment, wherein the second axial position of the armature positions the armature so that the balls are in contact with the second cylindrical segment.

wherein the balls engage with both the stub shaft and the generator shaft when the armature is in the first axial position so that the balls are positioned to transmit torque forces between the stub shaft and the generator shaft, wherein the balls engage with only the generator shaft when the armature is in the second axial position so that the stub shaft and the generator shaft are free to rotate independently of one another.

[0009] These and other features, aspects and advantages of the present invention will become better understood with reference to the following drawings, description and claims.

BRIEF DESCRIPTION OF THE DRAWINGS

[0010]

Figure 1 is a schematic diagram of an electrical generating system for an aircraft in accordance with an exemplary embodiment of the invention;

Figure 2 is a cross-sectional view of a connection system in accordance with an exemplary embodiment of the invention;

Figure 3 is a detailed view of a portion of the connection system of Figure 2 accordance with an exemplary embodiment of the invention;

Figure 4 is a cross-sectional view taken along the lines 4-4 of a portion of the connection system of Figure 2 in accordance with an exemplary embodiment of the invention;

Figure 5 is a detailed view of a radiused detent and ball connection in accordance with an exemplary embodiment of the invention;

Figure 6 is a cross-sectional view of the connection system of Figure 2 in a second operational mode in accordance with an exemplary embodiment of the invention:

Figure 7 is a cross-sectional view of a connection system in accordance with a second exemplary embodiment of the invention;

Figure 8 is a cross-sectional view of the connection system of Figure 7 in a second operational mode in accordance with the second exemplary embodiment of the invention;

Figure 9 is a detailed view of a portion of the connection system of Figure 7 accordance with the second exemplary embodiment of the invention; and Figure 10 is a cross-sectional view taken along the lines 10-10 of a portion of the connection system of Figure 7 in accordance with an exemplary embodiment of the invention;

DETAILED DESCRIPTION OF THE INVENTION

[0011] The following detailed description is of the best currently contemplated modes of carrying out the invention. The description is not to be taken in a limiting sense, but is made merely for the purpose of illustrating the general principles of the invention, since the scope of the invention is best defined by the appended claims.

[0012] Various inventive features are described below that can each be used independently of one another or in combination with other features.

[0013] The present invention generally provides an aircraft generator disconnect system that employs a solenoid actuated armature which may selectively release torque transmitting balls from their torque transmitting locations to allow a generator driving shaft to rotate independently from a generator input shaft, thereby achieving selective mechanical disconnection of the generator from its mechanical power source. More particularly, the present invention provides for a re-settable disconnect system. Moreover, the present invention accommodates torque transmission in both a clockwise and counterclockwise direction

[0014] Turning now to the description and with reference first to Figure 1, a schematic diagram may illustrate an electrical generating system 100 that may be employed in an aircraft (not shown). An engine 102 may be connected to gearbox 104. A stub shaft 106 may extend from the gear box 104 and connect with a generator shaft 108 within a connection unit 110. The stub shaft 106 may be considered to be drivably connected with the engine 102. Under normal operating conditions rotation of the stub shaft 106 may drive the generator shaft 108 to drive a generator 112. In the event of an operational anomaly of the generator 112, the connection unit 110 may be actuated, as described hereinbelow, to disengage the stub shaft 106 from the generator shaft 108. The stub shaft 106 and the generator shaft 108 may be considered to be selectively connected.

[0015] Referring now to Figures 2, 3, 4, 5 and 6, a cross-sectional view of an exemplary embodiment of the connection unit 110 illustrates operational features of the unit. The connection unit 110 may include a housing 114 which may be supported on the generator 112 (See Figure 1). The housing 114 may support a stub shaft bearing 116 and a solenoid coil 118. The stub shaft 106 and the generator shaft 108 may be positioned in an overlapping relationship along an interface 120. The generator shaft 108 may have a hollow segment 109 that is in the overlapped relationship with the stub shaft 106. A plurality of holes 122 may pass between an outer surface 124 of the generator shaft and in inner surface 126 of the generator shaft 108 (i.e., through a wall of the generator shaft 108). The stub shaft 106 may be provided with a plurality of detents 128 formed into its inner surface 130. A plurality of balls 132 may be positioned within the holes 122. As shown in Figures 2, 3 and 4, the balls 132 may rest against an armature 134 that may be positioned inside

the generator shaft 108. The armature 134 may comprise a first cylindrical segment 135 and a second reduced-diameter cylindrical segment 136. The balls 132 may be large enough and the wall thickness of the generator shaft 108 may be small enough so that a portion of each of the balls 132 may project into and beyond the interface 120. The detents 128 may be axially aligned with the holes 122. Each one of the balls 132 may be aligned with one of the detents 128. The holes 122 and the detents 128 may be considered to be ball-engagement openings. It may be seen that selective radial displacement of the balls 132 may result is selective interconnection between the stub shaft 106 and the generator shaft 108.

[0016] Rotation the stub shaft 108 may transmit torque forces through the balls 132 and into the generator shaft 108. Conversely, rotation of the generator shaft 108 may result in torque forces being transmitted to the stub shaft 106. In other words, the connection unit 110 may transmit torque forces during clockwise or counterclockwise rotation of the shafts 106 and 108. Consequently, the generator 112 may function as a starter-generator.

[0017] Referring now to Figure 6, the connection unit 110 is illustrated in a disconnected state. In that regard, it may be seen that the armature 134 is repositioned so that the reduced-diameter cylindrical segment 136 is positioned under the balls 132. The armature 134 may be moved between a first axial position and a second axial position by energizing the solenoid coil 118. The diameter of the segment 136 is small enough so that when the balls 132 rest against the segment 136, the balls 132 may be released from the detents 128 of the stub shaft 106. The diameter of the segment 136 is large enough so than when the balls 132 are in contact with the segment 136, the balls 132 remain engaged with the holes 122. In that configuration, the stub shaft 106 and the generator shaft 108 may be disconnected and thus may be free to rotate independently of one another.

[0018] One or more permanent magnets 138 may be positioned on the armature 134. When the connection unit 110 is in a disconnected state the magnets 138 may be aligned with the balls 132. The magnets 138 may magnetically hold the balls 132 in their disconnected state so that the balls 132 may not inadvertently re-engage with the detents 128.

[0019] The solenoid coil 118 may be connected so that it may selectively drive the armature 134 into a disconnection position as shown in Figure 6 or return the armature 134 into a connection position as shown in Figure 2. When the armature is driven into a connection position, a conical transition segment 140 of the armature 134 may drive the balls away from the magnets 138 and into reengagement with the detents 128. In other words, the connection unit 110 may be advantageously re-settable with a simple operation of the solenoid coil 118.

[0020] Referring now more particularly to Figures 3 and 4 it may be seen, that in an exemplary embodiment of the connection unit 110, the holes 122 may be formed as slots. The holes may have a width W that is only slightly

larger than a diameter of the balls 132. The holes 122 may have a length L is larger than the width W. In this regard, the balls 132 may be provided with axial clearance so that upon operation of the solenoid coil 118 (Figures 1 and 5), the balls 132 may roll within the slotted holes 122. Consequently, the armature 134 may be axially moved into its disconnection position while encountering only minimal rolling friction relative to the balls 132. Rolling action of the balls 132 during disconnection may be desirable because resultant disconnection force may be less than that which might result from sliding interaction between the balls 132 and the armature 134.

[0021] In the exemplary embodiment of the connection unit 110 of Figures 3 and 4, the detents 128 may be formed as conical pockets in the inner surface 130 of the stub shaft 106. The conical configuration of the detents 128 may reduce the risk of a failure of the balls 132 to release from the detents 128 when there is a high torque loading between the shafts 106 and 108. High torque loading may actually result in the balls 132 being forced out of the detents 128 when the armature 134 is moved to its disconnection position. In other words, conical configuration of the detents 128 may contribute to assuring positive disconnection by the connection unit 110 under a wide range of torque loading conditions.

[0022] In still another embodiment shown in Figure 5, detents 129 may be formed as radiused slots or pockets. A radius (Rd) of the detent 129 may be larger than a radius (Rb) of the ball 132. The radiused detents 129 may provide particularly high load carrying geometry for hertzian contact stress. Outer edges 131 of the detents 129 may radiused to facilitate positive disconnection by the connection unit 110 under a wide range of torque loading conditions.

[0023] Referring now to Figures 7, 9 and 10, a crosssectional view of an exemplary embodiment of a connection unit 210 illustrates operational features of the unit. The connection unit 210 may include a housing 214 which may be supported on the generator 112 (See Figure 1). The housing 214 may support a stub shaft bearing 216 and a solenoid coil 218. A stub shaft 206 and a generator shaft 208 may be positioned in an overlapping relationship along an interface 220. The generator shaft 208 may be hollow along that portion of its length that is in the overlapped relationship with the stub shaft 206. A plurality of holes 222 may pass between an outer surface 224 of the generator shaft 208 and an inner surface 226 of the generator shaft 208. The stub shaft 206 may be provided with a plurality of detents 228 formed into its outer surface 230.

[0024] A plurality of balls 232 may be positioned within the holes 222. As shown in Figures 6 and 8, the balls 232 may be held in engagement with the detents 228 by an armature 234 that may be positioned around the generator shaft 208. The armature 234 may comprise a first cylindrical segment 235 and a second cylindrical segment 236 with an enlarged internal diameter. The balls 232 may be large enough and a wall thickness of the

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generator shaft 208 may be small enough so that a portion of each of the balls 232 may project into and beyond the interface 220. The detents 228 may be aligned with the holes 222. Each one of the balls 232 may be aligned with one of the detents 228.

[0025] Rotation the stub shaft 206 may transmit torque forces through the balls 232 and into the generator shaft 208. Conversely, rotation of the generator shaft 208 may result in torque forces being transmitted to the stub shaft 206. In other words, the connection unit 210 may transmit torque forces during clockwise or counterclockwise rotation of the shafts 206 and 208. Consequently, the generator 112 may function as a starter-generator.

[0026] Referring now to Figure 8, the connection unit 210 is illustrated in a disconnected state. In that regard, it may be seen that the armature 234 is repositioned so that the enlarged-diameter cylindrical segment 236 is positioned over the balls 232. The armature 234 may be moved in an axial direction by energizing the solenoid coil 218. The internal diameter of the segment 236 is great enough so that when the balls 232 rest against the segment 236, the balls 232 may be released from the detents 228 of the stub shaft 206. The diameter of the segment 236 is small enough so than when the balls 232 are in contact with the segment 236, the balls 232 remain engaged with the holes 222. In that configuration, the stub shaft 206 and the generator shaft 208 may be disconnected and thus may be free to rotate independently of one another.

[0027] One or more permanent magnets 238 may be positioned on the armature 234. When the connection unit 210 is in a disconnected state the magnets 238 may be aligned with the balls 232. The magnets 238 may hold the balls 232 in their disconnected state so that the balls 232 may not inadvertently re-engage with the detents 228.

[0028] The solenoid coil 218 may be connected so that it may selectively drive the armature 234 into a disconnection position as shown in Figure 8 or return the armature 234 into a connection position as shown in Figure 7. When the armature 234 is driven into a connection position, a conical transition segment 240 of the armature 234 may drive the balls 232 away from the magnets 238 and into re-engagement with the detents 228. In other words, the connection unit 210 may be advantageously re-settable with a simple operation of the solenoid coil 218. A locking latch 242 may be provided to lock the armature in a disconnected position so that inadvertent re-engagement of the connection unit may be precluded without positive release of the locking latch 242.

[0029] As described above with respect to the connection unit 110 and for the same advantageous reasons, the holes 222 may be slotted holes and the detents 228 may have a conical configuration.

[0030] It should be understood, of course, that the foregoing relates to exemplary embodiments of the invention and that modifications may be made without departing from the spirit and scope of the invention as set forth in

the following claims.

Claims

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 An electrical generating system (100) for an aircraft comprising:

a first shaft (106, 206) drivably connected with an engine gearbox (104) of the aircraft;

a second shaft (108, 208) co-axial with the first shaft (106, 206) and drivably connected with a generator (112) of the aircraft; and

a connection unit (110, 210) in which the first and second shafts are selectively engageable with one another, the connection unit comprising,

a solenoid coil (118, 218),

an armature (134, 234) co-axial with the first (106, 206) and the second (108, 208) shafts, a plurality of balls (132, 232) positioned circumferentially around the armature (134, 234) in contact with the armature,

wherein the armature includes a first cylindrical segment (135, 235) and a second cylindrical (136, 236) segment,

wherein the armature is selectively movable between a first axial position and a second axial position responsively to actuation of the solenoid coil,

wherein the first axial position of the armature positions the armature so that the balls (132, 232) are in contact with the first cylindrical segment (135, 235),

wherein the second axial position of the armature positions the armature so that the balls (132, 232) are in contact with the second cylindrical segment (136, 236),

wherein the balls engage with both the first and second shaft when the armature is in the first axial position so that the balls are positioned to transmit torque loading between the first shaft and the second shaft,

wherein the balls engage with only one of the first or second shaft when the armature is in the second axial position so that the first shaft and the second shaft are free to rotate independently of one another.

- 50 2. The system of claim 1 wherein the first cylindrical segment (135) of the armature (134) has an outside diameter greater than an outside diameter of the second cylindrical segment (136) of the armature (134).
- 55 **3.** The system of claim 1 wherein the first cylindrical segment (235) of the armature (234) has an inside diameter smaller than an inside diameter of the second cylindrical segment (236) of the armature (234).

4. The system of claim 1 further comprising one or more permanent magnets (138, 238) positioned in the second cylindrical segment (136, 236) to magnetically hold the balls (132, 232) in contact with the second cylindrical segment when the first and the second shafts are free to rotate independently of one another.

5. The system of claim 1:

wherein the first shaft (106, 206) and the second shaft (108, 208) are positioned in a co-axial overlapping configuration within the connection unit (110, 210);

wherein the first shaft is provided with a plurality of first ball-engagement openings (122, 228); wherein the second shaft is provided with a plurality of second ball-engagement openings (128, 222);

wherein the first ball-engagement openings (122, 228) are axially aligned with the second ball-engagement openings (128,222) so that selective radial displacement of the balls produces selective connection or disconnection of the first and the second shafts.

6. The system of claim 5:

wherein the first shaft (206) has an overlapping segment that has an inside diameter greater than an outside diameter of an overlapping segment of the second shaft (208);

wherein the second ball-engagement openings (222) are slotted holes which pass through a wall of the overlapping segment of the second shaft.

7. The system of claim 1 wherein the armature (134, 234) includes a conical transition segment (140, 240) interposed between the first cylindrical segment (135, 235) and the second cylindrical segment (136, 236).

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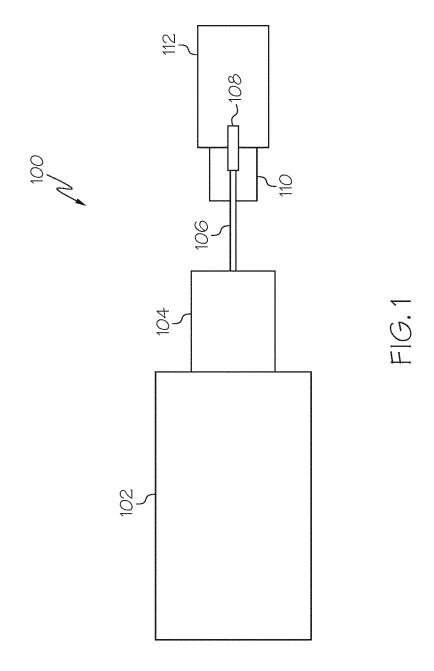
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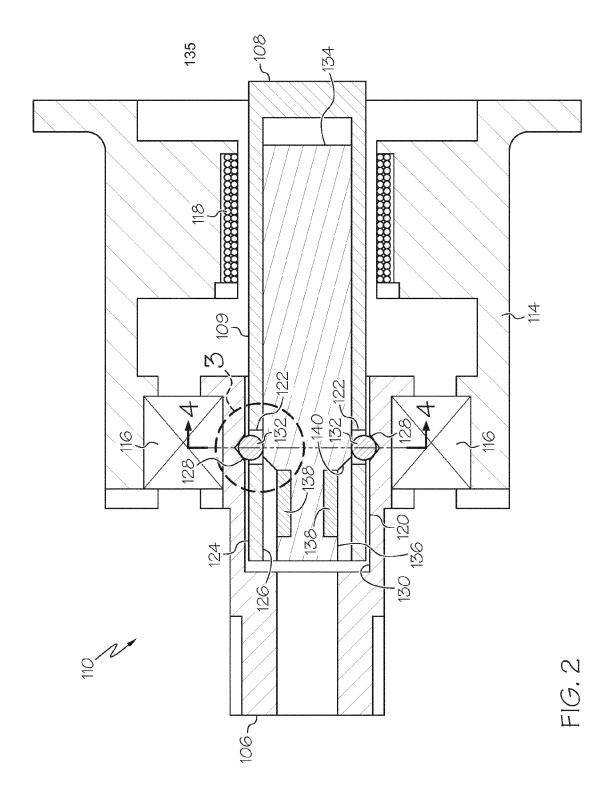
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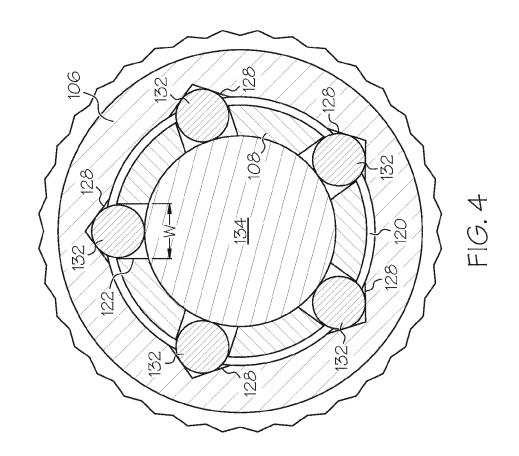
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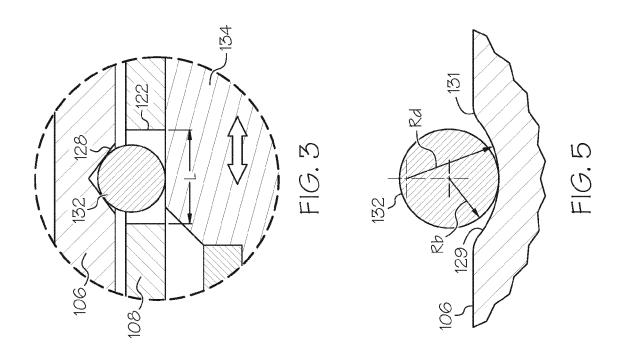
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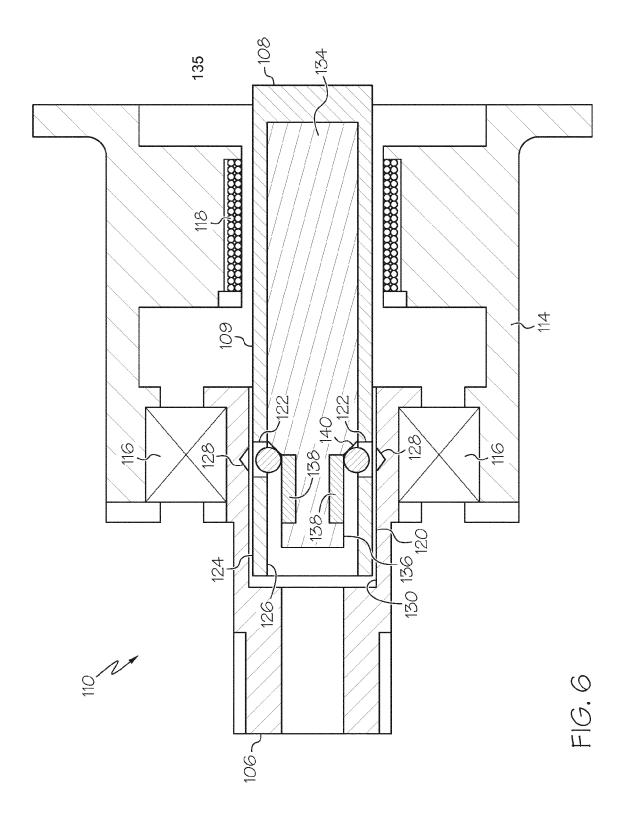
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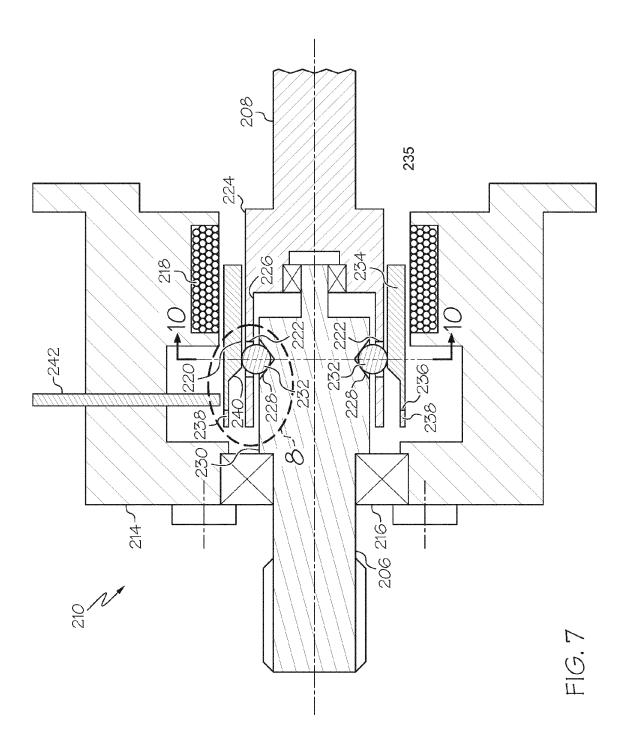


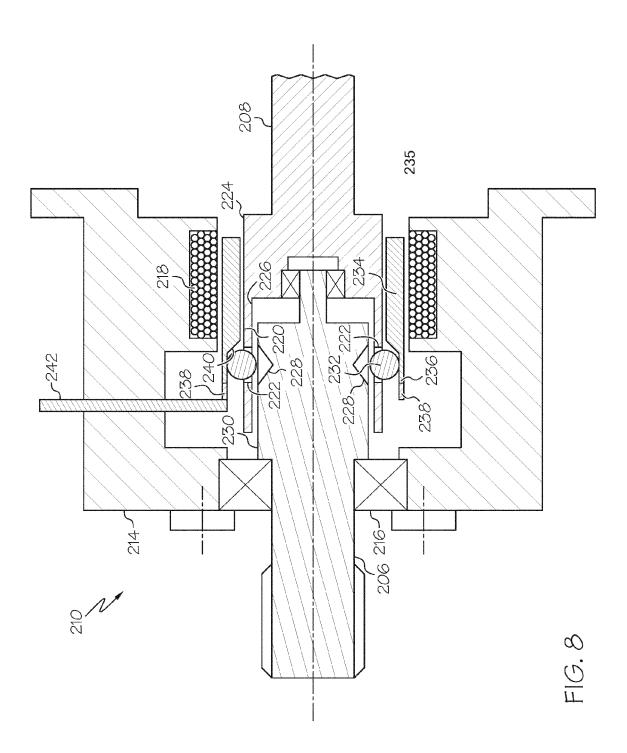


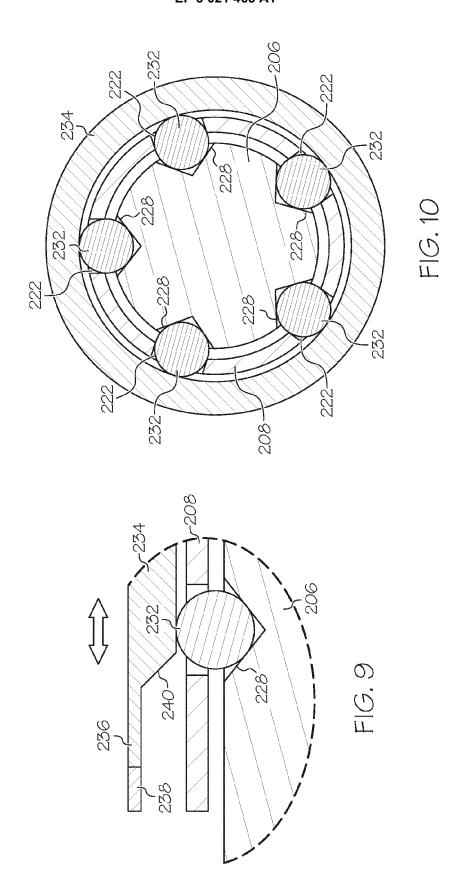














EUROPEAN SEARCH REPORT

Application Number EP 15 19 3314

		DOCUMENTS CONSID	ERED TO BE RELEVANT		1	
	Category	Citation of decument with in	ndication, where appropriate,	Relevant to claim	CLASSIFICATION OF THE APPLICATION (IPC)	
10	Х	US 4 685 550 A (MET 11 August 1987 (198 * column 1, line 7	CALF JEFFREY D [US])	1-3,5-7	INV. H02K7/00 F16D27/00	
15	A	EP 2 098 740 A2 (G0 [GB]) 9 September 2 * abstract *		1	H02K35/00	
20	A	EP 1 220 164 A1 (MA 3 July 2002 (2002-0 * paragraph [0027] * figures 7(A)-7(D)	7-03) - paragraph [0028] *	4		
25	A	EP 2 098 741 A2 (H0 9 September 2009 (2 * abstract *	NEYWELL INT INC [US])	1		
30					TECHNICAL FIELDS SEARCHED (IPC) H02K F16D	
35						
40						
45						
1	The present search report has been drawn up for all claims Place of search Date of completion of the search				Examiner	
50 (10070)		The Hague	13 April 2016		aci, Mutlu	
50 ((x0070d) 288 83 83 83 83 84 84 85 85 85 85 85 85 85 85 85 85 85 85 85	CATEGORY OF CITED DOCUMENTS X: particularly relevant if taken alone Y: particularly relevant if combined with anoth document of the same category A: technological background O: non-written disclosure P: intermediate document		after the filing dat ner D : document cited ir L : document cited fo	eument, but publise e n the application or other reasons	ıt, but published on, or application er reasons	
EPO FG			& : member of the sa document	& : member of the same patent family, corresponding document		

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ANNEX TO THE EUROPEAN SEARCH REPORT ON EUROPEAN PATENT APPLICATION NO.

EP 15 19 3314

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This annex lists the patent family members relating to the patent documents cited in the above-mentioned European search report. The members are as contained in the European Patent Office EDP file on The European Patent Office is in no way liable for these particulars which are merely given for the purpose of information.

13-04-2016

10	Patent document cited in search report	Publication date	Patent family member(s)	Publication date
15	US 4685550 /	11-08-1987	EP 0238659 A1 JP S63501588 A US 4685550 A WO 8702106 A1	30-09-1987 16-06-1988 11-08-1987 09-04-1987
	EP 2098740	A2 09-09-2009	EP 2098740 A2 US 2009223774 A1	09-09-2009 10-09-2009
20	EP 1220164	A1 03-07-2002	CN 1362697 A EP 1220164 A1 JP 4578746 B2 JP 2002230611 A US 2002117798 A1	07-08-2002 03-07-2002 10-11-2010 16-08-2002 29-08-2002
25	EP 2098741	A2 09-09-2009	EP 2098741 A2 US 2009224727 A1	09-09-2009 10-09-2009
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55	FORM P0459			

For more details about this annex : see Official Journal of the European Patent Office, No. 12/82