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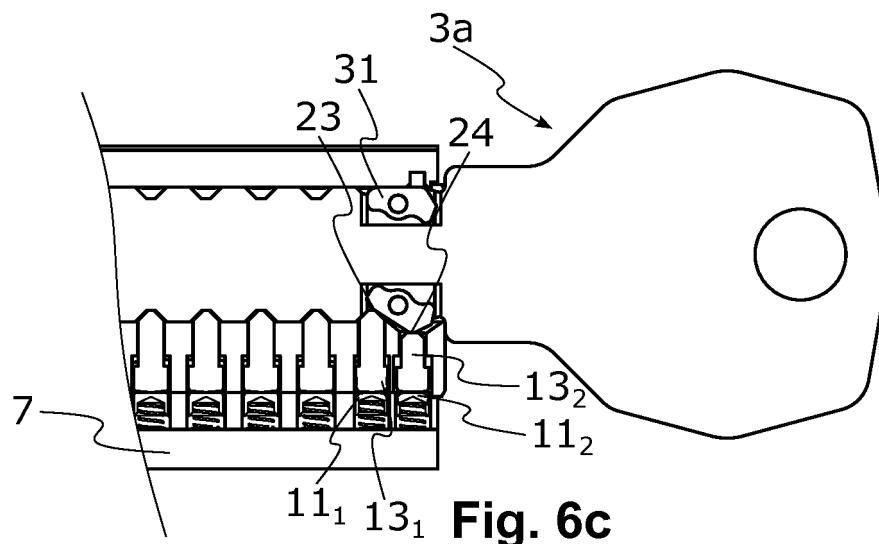
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(54) **KEY WITH A PIVOTING ELEMENT AND A LOCK**

(57) The present invention concerns a key for a cylinder lock. The key (3a) comprises a key shaft (22) for inserting into a key channel of the cylinder lock; and a force transferring means (31) disposed in the key shaft (22) and comprising a first pin engaging surface (39<sub>1</sub>) for engaging with a first rotor pin of the cylinder lock, and a second pin engaging surface (39<sub>2</sub>) for engaging with a second rotor pin of the cylinder lock. The force transfer-

ring means (31) is configured such that, in response to a first force exerted by the first pin on the first pin engaging surface (39<sub>1</sub>) in a first direction, the second pin engaging surface (39<sub>2</sub>) exerts a second force on to the second pin in a second direction. The second direction is substantially opposite to the first direction. The invention also relates to the corresponding cylinder lock.



**Fig. 6c**



## Description

### TECHNICAL FIELD

**[0001]** The present invention relates to a lock, a key comprising a pivoting element and to a method of coding the key.

### BACKGROUND OF THE INVENTION

**[0002]** In the last couple of decades many new optimisations and improvements have been introduced in the field of cylinder locks and keys with the main purpose of increasing the number of lock combinations and/or the complexity of mechanical key geometries. Many significant electronic features have also been introduced to mechanical cylinder locks in this way enhancing the security based on electronic devices, which relied on external electrical supply.

**[0003]** Some typical optimisations which involve increasing the number of lock combinations on mechanical keys and/or cylinder locks are:

- increasing the number of interacting components, such as pins and cavities and/or tumbler pins and cavities;
- varying and/or increasing the complexity of the shape of the pins, tumbler pins, keys and/or the corresponding channels in the cylinder lock;
- manufacturing keys with at least one static vacancy with a special form, which reacts with their counter-acting parts in the cylinder lock.

**[0004]** These types of mechanical improvements have made the reproduction of keys and cylinder locks by unauthorised third parties more difficult. However, these improvements do not comprise an active moving element which may make the cloning of these parts with conventional machines faster and simpler. The geometry of a key can as well be interpreted by an expert who is then able to clone a key with either rapid prototyping machines or conventional machines.

**[0005]** As commonly known, cylinder locks comprise at least one housing and/or a stator, which contains a rotor, sometimes also referred to as a plug, arranged to rotate within the cylinder. At least one row of cavities or one row of stator through holes is defined inside the stator, and at least one row of rotor through holes, rotor channels and a key channel are defined in the rotor. Rotor pins are accommodated in the rotor through holes, such that one end of each rotor pin is arranged to protrude into the key channel. Stator pins, supported by springs, are accommodated in the cavities, which may be blind holes. Tumbler pins are accommodated in the rotor channels. The ends of the tumbler pins are arranged to protrude into cavities in the stator. The corresponding key has a

row of teeth or dimples defined along the top and bottom edge and grooves on the lateral sides of the key shaft, which is inserted into the key channel of the lock. The row of rotor pins engages with the teeth or dimples of the key, and the row of tumbler pins engages with the grooves of the key. When the key is fully inserted in the key channel, causing the plane of contact between the rotor pins and the stator pins and between the tumbler pins and the key grooves to align with the plane of contact between the rotor and the stator. When this alignment occurs on all the rotor pins and tumbler pins, the rotor becomes free to rotate in the cylinder. The rotation of the cylinder is used to operate a further mechanism, such as a cam for moving a dead bolt element and thereby unlocking a door.

**[0006]** The disadvantage of this type of lock and key is that the key can be reproduced by unauthorised third parties using a series of relatively simple machining operation such as milling and drilling. Some solutions have been proposed to overcome this problem, as known for example from Spanish patent application ES2088345. However, the solution proposed in ES2088345 is not optimal, since the key is not reversible, and also in terms of coding permutations the key and/or lock design is not optimal, since the moving part on the lateral side of the key reduces the permutation ability, because the moving part uses a part of the key in which a key coding could have been stored. Another solution is proposed in EP0416500, however, this solution does not overcome the drawbacks of ES2088345. Furthermore, the solution disclosed in EP0416500 is complicated in that it requires auxiliary holes in the rotor and lock cylinder, perpendicular to and on opposite sides of a plane defined by the key channel. The auxiliary holes are arranged to accommodate auxiliary pins and springs.

**[0007]** It is an object of the present invention to overcome the problems identified above related to cylinder locks and associated keys.

### SUMMARY OF THE INVENTION

**[0008]** According to a first aspect of the invention, there is provided a cylinder lock as recited in claim 1.

**[0009]** According to a second aspect of the invention, there is provided a key for cylinder lock as recited in claim 3.

**[0010]** According to a third aspect of the invention, there is provided a method of coding a key as recited in claim 15.

**[0011]** With the proposed new solution more advantages are attained through further developments, in which enhancements are made to increase the safety of key blanks, keys and/or locks by using mechanical means.

**[0012]** The proposed new solution has therefore the advantage that key blanks, keys and/or the cylinder locks are more difficult to reproduce by unauthorised third parties, due to the inclusion of a force transferring means



such as a pivoting element. Furthermore, the inventive key and lock provide increased security, because the force transferring means can be used to code the key. Furthermore, fraudulent opening of the lock is made more difficult, because at least one of the rotor pins has to be in a recessed position in its through hole in order to rotate the rotor. Moreover, the proposed key may be reversible.

**[0013]** Other aspects of the invention are recited in the dependent claims attached hereto.

#### BRIEF DESCRIPTION OF THE DRAWINGS

**[0014]** Other features and advantages of the invention will become apparent from the following description of a non-limiting exemplary embodiment, with reference to the appended drawings, in which:

- Figure 1 is a simplified cross-sectional side view of a cylinder lock mechanism together with associated keys taken in a vertical plane along the longitudinal axis of the key channel;
- Figure 2 is a cross-sectional view of a cylinder lock mechanism together with an associated key taken in a plane orthogonal to the longitudinal axis of the key channel;
- Figure 3 is an isometric view of a key according to an embodiment of the present invention;
- Figure 4 is an isometric view of the key of Figure 3, showing the pivoting element in a different position;
- Figure 5 is a cross-sectional side view of the key of Figure 3 taken in a vertical plane along the longitudinal axis of the key;
- Figures 6a to 6c are cross-sectional side views of a part of the lock mechanism of Figure 1 together with a key, taken in a vertical plane along the longitudinal axis of the key, and showing the process of inserting the key into the key channel;
- Figure 7 is an enlarged cross-sectional side view of a part of the cylinder lock mechanism together with the associated key, taken in a vertical plane along the longitudinal axis of the key, according to a variant of the present invention;
- Figure 8 is an enlarged cross-sectional side view of a part of the cylinder lock mechanism together with the associated key, taken in a vertical plane along the longitudinal axis of the key, according to another variant of the present invention;
- Figure 9 is a cross-sectional side view of a key, taken in a vertical plane along the longitudinal axis of the key, according to another variant of the present in-

vention;

- Figure 10 is a cross-sectional side view of a key, taken in a vertical plane along the longitudinal axis of the key, according to another variant of the present invention;
- Figure 11 is an isometric view of a key according to another variant of the present invention;
- Figure 12 is an isometric view of a key according to another variant of the present invention;
- Figure 13 is an isometric view of a key according to another variant of the present invention;
- Figure 14 is an enlarged cross-sectional side view of a part of the cylinder lock mechanism together with the associated key, taken in a vertical plane along the longitudinal axis of the key, according to another variant of the present invention;
- Figure 15 is an enlarged cross-sectional side view of a part of the cylinder lock mechanism together with the associated key, taken in a vertical plane along the longitudinal axis of the key, according to another variant of the present invention;
- Figure 16 is a cross-sectional view of a cylinder lock mechanism together with an associated key, taken in a plane orthogonal to the longitudinal axis of the key channel, according to another variant of the present invention; and
- Figure 17 is a cross-sectional view of a cylinder lock mechanism together with an associated key, taken in a plane orthogonal to the longitudinal axis of the key channel, according to another variant of the present invention.

#### DETAILED DESCRIPTION OF AN EMBODIMENT OF THE INVENTION

**[0015]** An embodiment of the present invention will now be described in detail with reference to the attached figures. This embodiment is described in the context of mechanical keys and mechanical cylinder locks, but the teachings of the invention are also applicable to mechatronic keys and mechatronic cylinder locks. Identical or corresponding functional and structural elements which appear in the different drawings are assigned the same reference numerals.

**[0016]** Figure 1 shows a cross-sectional side view of a double cylinder lock 1 in which authorised keys 3a, 3b, also called safety keys, are inserted in the key channels 5 (see Figure 6a), also known as a key slot, key hole or key way. The double cylinder lock 1 comprises two cylinder locks 1 a, 1 b, which are connected to each other



by a crosspiece 2. Both locks 1 a, 1 b comprise a mechanical key coding system following e.g. the principles disclosed in CH4077989. The lock 1 a on the right further comprises an additional mechatronic system having a coding scheme e.g. according to WO98/28508. In the following description, for simplicity, only the lock 1 a and the mechanical key 3a on the right are described in more detail. However, the teachings below also apply to the lock 1 b and the mechatronic key 3b on the left.

**[0017]** In Figure 1 there is also shown a housing 7, in which stators 8 are provided. The stator 8 is fixed in the housing, or may be of one piece with the housing, and is not arranged to rotate or move in another manner. A rotor 9 is located in the stator 8 such that the rotor 9 is axially fixed but rotatable within the stator 8. It is to be noted that in some configurations the stator and housing may be merged into one element. The stator has cavities or channels, in this case located radially with respect to the longitudinal axis of the stator. In this example, the cavities are aligned in one row, but other configurations of the cavities are also possible. Stator pins 11, also known as housed pins, are provided in the cavities. Each stator pin 11 is supported by a biasing means 12, in this example a compression spring, hereinafter simply referred to as a spring. The springs are arranged to urge the stator pins 11 towards the key channel 5. The stator pins 11 in turn cooperate with rotor pins 13 accommodated in rotor pin channels, referred to as through holes, or simply pin channels and located radially in the rotor 9. The distal ends of the rotor pins 13 are arranged to engage with the key 3a in the key channel 5, as explained later in more detail. When the proximal ends (the ends in contact with the stator pins) of the rotor pins 13 are substantially flush with the rotor outer surface, the lock 1 a can be unlocked by rotating the rotor within the stator 8. The turning of the rotor causes a cam 14 cooperating with a coupling unit 15 to rotate so that the cam can engage with a door latch, for example. In the configuration of Figure 1, the rotor is in its unlocking state. However, when there is no key 3a in the key channel 5, the rotor pins 13 are biased against a shoulder stop in the through holes, the rotor pins and the shoulder stops being such that the proximal ends of the rotor pins 13 are not flush with the rotor outer surface, and the rotor pins therefore block the rotation of the rotor, so that the lock 1 a cannot be unlocked.

**[0018]** In Figure 1 there are shown two types of keys, namely a mechanical key on the right 3a and a mechatronic key on the left 3b. The mechatronic key has an electronic structural element 16a and may as well have an electric circuit 16b. The electronic structural element 16a and/or the electric circuit 16b can communicate with an electronic control device (not shown), such as an RFID reader (not shown), which may either be comprised in the lock or be a part of the lock as an outstanding member. In this manner the electronic structural element 16a, the electric circuit 16b, and the electronic control device comprise the key coding. In order to be able to unlock the

cylinder lock 1 b, the mechatronic key 3b has to be fully inserted in the key channel 5 in order to first unlock the mechanical locking mechanism. The electronic structural element 16a and/or the electric circuit 16b communicate(s) with the electronic control device (not shown), which in turn will change the state of a locking unit device 17 to an unlocking state, or when the key coding is not recognised, the locking unit device 17 remains in a locking state. The data exchange between the key 3b and the cylinder lock can either be hard-wired or by radio transmission, which means that the mechatronic key 3b does not have to be inserted into the key channel 5 in order to change the state of the locking unit device 17. In Figure 1 the keys 3a and 3b are further shown to comprise force transferring means 31 of which the purpose will be explained later.

**[0019]** Figure 2 shows a cross-sectional view of the lock mechanism, taken through the first rotor pin 13 from the right in Figure 1 in a plane orthogonal to the plane defined by the longitudinal axis of the key channel 5. As shown, in this configuration the rotor pins 13 are aligned with the key 3a. In other words, the longitudinal axis of the blind holes is aligned with the key shaft 22 as viewed in the cross-section. Figure 2 also shows a safety ring 18, which fixes the rotor 9 to the stator 8. The safety ring 18 is fitted in place with two safety springs 19 and one safety screw 20. The safety springs 19 ensure that the safety ring is kept under pre-tension. The safety ring 18 ensures that the rotor 9 is kept in the stator 8 and cannot be fraudulently dismounted. Without the safety ring 18, the rotor 9 could theoretically fall out of the stator 8. The structure of the key 3a is explained next in more detail with reference to Figures 3 to 5.

**[0020]** Figure 3 is an isometric view of the key 3a according to an embodiment of the present invention. The key 3a comprises a grip head 21 connected to a key shaft 22, also known as a key blade, body or shank. In this example the key shaft 22 has a substantially rectangular cross-section, but other cross-sectional shapes, such as a substantially circular shape, are also possible. The key 3a further comprises pin-engaging means, referred to as dimples 25, on the top and/or bottom edge of the shaft 22 which are arranged to engage with the rotor pins 13 when the key shaft 22 is inserted into the key channel 5 of the lock. In this example, the lateral side(s) of the shaft 22 comprises two grooves 27 to engage with further validating means 30, known as tumblers (see Figure 6a) in the rotor 9. However, the teachings of the invention would also apply to keys with no grooves, or having a number of grooves other than two, for example one groove, possibly on either lateral side of the shaft. Instead of, or in addition to, the groove(s), the lateral side(s) could comprise further dimples, cavities and/or protrusions. In Figure 2, a longitudinal axis 29 of the key 3a is also illustrated. This axis 29 substantially coincides with the longitudinal axis of the key channel when the key is in the key channel 5. By moving the key along this axis 29, in this example to the left in Figure 3, the key can be inserted



into the key channel 5.

**[0021]** According to the present invention, the key 3a, and more specifically the key shaft 22, comprises force transferring means 31, in the example shown in Figure 3 it is a pivoting element. The force transferring means 31 may also be called an active moving element, since it is arranged to be actively urged by at least one rotor pin 13 as explained later more in detail. The pivoting element 31 may at least be partly made of metal, plastic or ceramic, or a combination thereof. In the examples discussed below, the key 3a comprises at least two substantially identical pivoting elements 31 to make the key reversible, but the teachings of the present invention are not limited to keys with two pivoting elements. For example, the number of the pivoting elements could be one, or more than two, as explained later in more detail. In the discussed examples, the pivoting element is provided in a cavity 33, better shown for example in Figure 5. According to this example, the pivoting element 31 is held in place by holding means, such as a rivet. In this example the pivoting element 31 is arranged to rotate or pivot about a rotation axis 37, which coincides with the length axis of the rivet 35. The holding means (rivet 35) may or may not pass through the whole lateral cross section of the key. In another variant, the rotation axis 37 is not fixed, but may be arranged to move so that the rotation axis 37 remains substantially orthogonal to the longitudinal axis 29 of the key. In the illustration of Figure 3 the pivoting element 31 is shown in a fully extended or pressed position as explained later in more detail, whereas in the illustration of Figure 4 the pivoting element 31 is shown in a position such that the upper surface of the pivoting element 31 is substantially flush with the top edge of the key shaft 22.

**[0022]** Figure 5 is a cross-sectional side view of the key 3a. The pivoting element 31 in this example comprises two pin engaging surfaces 39, namely a first pin engaging surface 39<sub>1</sub> and a second pin engaging surface 39<sub>2</sub>, in this example facing both substantially in the same direction. The first pin engaging surface 39<sub>1</sub> is arranged to engage with a first key engaging surface 23 (see Figure 6a) of the first rotor pin 13<sub>1</sub>, while the second pin engaging surface 39<sub>2</sub> is arranged to engage with a second key engaging surface 24 (see Figure 6a) of the second rotor pin 13<sub>2</sub>. In this example the first key engaging surface 23 and the second key engaging surface 24 face substantially in the same direction. The first rotor pin 13<sub>1</sub> located in the first through hole is arranged to be urged by a first stator pin 11<sub>1</sub> biased by a first spring 12<sub>1</sub> (see Figure 6a) towards the key channel 5, while the second rotor pin 13<sub>2</sub> located in the second through hole is arranged to be urged by a second stator pin 11<sub>2</sub> biased by a second spring 12<sub>2</sub> towards the key channel 5 (see Figure 6a).

**[0023]** Figures 6a to 6c are cross-sectional lateral views illustrating in enlarged views the positions of the stator pins 11 and rotor pins 13 when inserting the key shaft 22 into the key channel 5. In Figure 6a the key is

almost fully outside the key channel. In this case the springs 12 are in their extended positions and urge the stator pins 11, which further push the rotor pins 13 to be in their most distant positions from the springs 12 (the end positions being determined by the stop shoulders as discussed above). Since the stator pins 11 and therefore the rotor pins 13 are not flush with the rotor outer surface when they are in this no-key configuration, it is not possible to rotate the rotor 9 and the lock 1 a cannot be unlocked or locked. The key grooves 27 are based on a key coding, which in turn have their validating means 30 in the lock 1a. The validating means 30 are in this case referred to as tumblers, which are in this example positioned in the rotor 9 parallel to the key channel 5. The rotation of the rotor 9 is being blocked by the ends of the tumblers 30, which protrude into cavities within the stator 8. In Figure 6a the tumblers 30 are in a blocking state, thus blocking the lock 1 a from being unlocked. As the key 3a is being inserted into the key channel 5, the tumblers 30 are guided through the key grooves 27 which will force a vertical movement of the tumblers 30.

**[0024]** Figure 6b illustrates a situation where the key shaft 22 is partially inserted into the key channel 5. As can be seen, four of the central stator pins 11 and the rotor pins 13 have now moved to different vertical positions compared to the situation in Figure 6a, and thus some of the rotor pins 13 have engaged with dimples 25 in the lower edge of the key. In Figure 6b, as in Figure 6a, the rotor pins 13 have not yet reached their unlocking positions and the lock 1a cannot therefore be unlocked yet.

**[0025]** In Figure 6c the key shaft 22 is fully inserted in the key channel 5. In this situation, the pivoting element 31 is in its pivoted or fully extended position. This position is reached by the first rotor pin 13<sub>1</sub> urging the first pin engaging surface 39<sub>1</sub> of the pivoting element 31 upwards so that the pivoting element rotates in the clockwise direction in this example. At the same time, the second pin engaging surface 39<sub>2</sub> engages with the second rotor pin 13<sub>2</sub> and urges it downwards in this configuration. This kind of rotation of the pivoting element 31 is possible if more torque is generated by the action of the first rotor pin 13<sub>1</sub> at the left hand side than is generated by the second rotor pin 13<sub>2</sub> on the right hand side of the pivoting element 31. Torque can be increased by increasing the spring strength of the first spring 12<sub>1</sub> and/or by increasing its rotational moment, i.e. the distance between the point where the force is exerted on the pivoting element 31 and the rotation axis 37. Thus in one configuration, the strengths of the first and second springs, 12<sub>1</sub>, 12<sub>2</sub> are chosen such that the first spring 12<sub>1</sub> it exerts a stronger force on its stator pin 11<sub>1</sub> than the second spring 12<sub>2</sub> exerts on its stator pin 11<sub>2</sub>. However, wear on the key 3a and the components can be reduced by creating more torque on the pivoting element by increasing the distance instead of, or in addition to, increasing the force of the dominant spring. The force exerted by the first rotor pin 13<sub>1</sub> on the pivoting element 31 causes the pivoting ele-



ment to rotate against the biasing force of the second spring 12<sub>2</sub>. In this way, the rotatory movement of the pivoting element 31 controls the movement of the rotor pins 13 and more specifically in this example the movement of the second rotor pin 13<sub>2</sub>.

**[0026]** In the present description, in order to maintain consistent terminology, it will be assumed that the first rotor pin 13<sub>1</sub> is the dominant rotor pin (i.e. the one which exerts the greater rotational moment on the pivoting element 31. However, depending on the configuration of the pivoting element 31, the first rotor pin 13<sub>1</sub> could for example be the right-hand most of the rotor pins 13. In this case a greater moment would be applied on the right hand side of the pivoting element 31 than on the left hand side of the pivoting element 31, and the rotation would be in the anti-clockwise direction. As can be seen in Figure 6c, when the key shaft 22 is inserted in the key channel 5, the proximal ends (i.e. the radially outer ends) of the rotor pins 13 and the ends of the tumblers (not shown) are now flush with the rotor outer surface, thereby enabling the rotor to be rotated and thus allowing the lock 1 a to be locked or unlocked. This is referred to as the unlocking position of the lock. In this position, the first rotor pin 13<sub>1</sub> is arranged to protrude into the key channel 5, while the second rotor pin 13<sub>2</sub> is held in a recessed or depressed position in the second through hole by the second pin-facing surface 39<sub>1</sub> of the pivoting element. According to the present invention the first and second through holes emerge into the key channel 5 at substantially the same circumferential position, with respect to the longitudinal axis, in the key channel 5.

**[0027]** In the example described above, the pivoting element 31 is designed such that the leading end, i.e. the end first entering the key channel, as opposed to the trailing end, is prevented or deterred from projecting out of the cavity 33 or is designed such that leading end can project only very slightly out of the cavity 33 so that this does not prevent the key 3a from being inserted into the key channel 5. This can be achieved by having for example a protrusion 41 at the trailing end of the pivoting element protruding from the cavity bottom facing side of the pivoting element 31. In this way, the pivoting element 31 cannot rotate beyond its flush orientation in the rotational direction opposite to the pin-actuating rotational direction or can rotate only very slightly beyond this orientation. In the case shown in Figure 6a, the protrusion 41 of the upper pivoting element 31 prevents or substantially prevents the upper pivoting element from rotating beyond its flush orientation (i.e. flush or parallel with the edge of the key shaft 22) in the clockwise direction. Similarly, the protrusion 41 of the lower pivoting element 31 prevents or substantially prevents the lower pivoting element 31 from rotating beyond its flush orientation in the anti-clockwise direction. This avoids a situation in which the leading edge of the pivoting element could protrude outwards from its key edge and thereby interfere with the insertion of the key shaft 22 into the key channel 5.

**[0028]** Other configurations of the pivoting element 31

whose leading end is not able to protrude from its cavity 33 or is able to protrude only very slightly from the cavity 33 so that this does not prevent the key 3a from being inserted into the key channel 5 are illustrated in Figures 7 and 8. In the configuration of Figure 7, the cross-section of the cavity 33 has a substantially semi-circular shape, which is simple to manufacture. In this case, the pivoting element 31 may have a simple form. The pivoting element 31 in the configuration of Figure 8 has an opening 42 arranged to receive a rotation limiting pin 43 arranged substantially parallel to the rotation or pivot axis 37 of the pivoting element 31. Thus, in this configuration, the rotation limiting pin 43 and opening 42 limit the movement of the pivoting element 31. The shape and length of the opening 42 defines the possible range of rotational movement for the pivoting element 31. Figures 7 and 8 also identify a shoulder 47 on the rotor 9, at the opening of the key channel 5. It is also a security feature. It prevents a lock-picker from actuating the nearest rotor pin 13<sub>2</sub> (and therefore the angular position of the pivoting element 31) by hand. The shoulder 47 also contributes to the functioning of the pivoting element. It ensures that only keys in which the pivoting element 31 is flush with or below the edge surface of the key can be inserted into the key channel 5.

**[0029]** The pivoting element 31 can also be designed such that the leading end could, unless precautionary measures are taken, protrude from its cavity 33. This could impede the insertion of the key 3a into the key channel 5, as mentioned above. To overcome this issue, a biasing element 44, in this example a cavity spring, is provided in the cavity 33 to exert a rotational torque on the pivoting element 31. The cavity spring 44 acts on the pivoting element 31 such that the leading end of the pivoting element is biased in a rotational direction which results in the leading edge remaining in the cavity 33. In practical terms, the cavity spring 44 exerts a force on the trailing end of the pivoting element 31. However, as the first rotor pin 13<sub>1</sub> would also urge the trailing end in this scenario, the moment exerted by the rotor pin 13<sub>1</sub> is arranged to be stronger than the moment exerted by the cavity spring 44 to allow the pivoting member 31 to make the desired rotational movement when the authorised key is in the unlocking position. In the example configuration of Figure 9, the cavity spring 44 is a leaf spring, whereas in the configuration of Figure 10 it is a helical spring, also known as a coil spring.

**[0030]** Figure 11 shows another variant of the present invention. In the configurations explained above, the pivoting elements 31 are located at the trailing end of the shaft 22 or in the trailing end region, namely one pivoting element at the top edge of the key shaft 22, and the other pivoting element 31 at the bottom edge of the key shaft 22. In the configuration of Figure 11, by contrast, the pivoting elements 31 are located at the leading end of the shaft 22, or in the leading end region, namely one pivoting element at the top edge of the key shaft 22, and the other pivoting element 31 at the bottom edge of the key shaft



22. By having the pivoting element 31 at the leading end of the shaft, more space is available at the trailing end of the shaft. On the other hand, arranging the pivoting element 31 at the leading end of the shaft may require a slimmer design of the pivoting element 31, especially if there is a groove or dimples on the lateral side of the key shaft 22.

**[0031]** Figure 12 shows a further variant of the present invention. In this variant there is a series of pivoting elements 31 disposed on the top and bottom side of the shaft 22. Having a series of pivoting elements arranged in the shaft makes it even more difficult to fraudulently reproduce the key 3a. Because each pivoting element 31 is associated with one or two dimples 25, of which the dimple 25<sub>1</sub> receiving the dominant rotor pin 13<sub>1</sub> can be used to code the key, the number of coding permutations is increased when compared to the number of coding permutation of a key 3a which only has known dimples and grooves. The position, size and seating depth of the pivoting element 31 in the key shaft 22 can be chosen in various ways. Thus the number of key coding permutations can be increased and various subsystems or main systems can be created that differ from each other. The geometry of the pivoting element 31 may be constant or variable in order to be able to increase the key coding combinations. The position, size and depth of the dimples may be constant or variable in order to be able to control the protrusion distance of the pivoting element 31 in the rotor 9. The interaction between the dimples 25, the rotor pins 13<sub>1</sub>, 13<sub>2</sub>, the pivoting element 31 (including the protrusion distance in the rotor 9) increase the number of coding permutations, since more variables are involved, which a coding system can be based upon.

**[0032]** Figure 13 shows a further variant of the present invention. In this variant there are three pivoting elements 31 disposed on the top and bottom side of the shaft 22. As can be seen, each of the pivoting elements, and more specifically the first pin engaging surface 39<sub>1</sub>, is associated with a dimple 25<sub>1</sub>, which can seat the first rotor pin 13<sub>1</sub>. These dimples 25<sub>1</sub>, which can seat the first rotor pin may be used for coding the key 3a. Of course the dimples 25 which are not associated with any pivoting element may also be used to code the key. The position, for example along the longitudinal axis 29 of the key 3a, size and depth of the dimples 25, 25<sub>1</sub> on the key or a protrusion distance of the second or third pin engaging surface 39<sub>2</sub>, 39<sub>3</sub> when the first rotor pin 13<sub>1</sub> is seated in the dimple 25<sub>1</sub> associated with the first engaging surface 39<sub>1</sub> may be varied in order to be able to create keys which are different from each other. It is to be noted that the cross-section of the dimples 25, 25<sub>1</sub> may or may not be circular. Thus, the position, size and/or depth of the dimples 25, 25<sub>1</sub> and/or the protrusion distance may be used as coding parameters. Also the position, for example along the longitudinal axis 29 of the key, size and/or depth of the cavity 33 in which the pivoting element 31 is fitted may also vary in order to be able to create various keys, which are different from each other. It is to be noted that the dimple

25<sub>1</sub> associated with the first pin engaging surface 39<sub>1</sub> may or may not be located at the end of the cavity 33 accommodating the pivoting element 31. For example, in the configuration of Figure 13, the dimple 25<sub>1</sub> of the middle pivoting element 31 is not located at the end of the cavity 33, but close to the centre point of the cavity along the longitudinal axis 29 of the key shaft 22.

**[0033]** Figures 14 and 15 illustrate further variants of the present invention. In these variants there is provided a third rotor pin 13<sub>3</sub> arranged to engage with the pivoting element 31. The third rotor pin 13<sub>3</sub> cooperates with a third stator pin 11<sub>3</sub> biased by a third biasing means 12<sub>3</sub>. In the variant of Figure 14 the pivoting element 31 has the same shape as in the configuration of Figure 5 for example. When the authorised key is in the unlocking position, the key facing end of the third rotor pin is substantially flush with the inner surface of the key channel. The variant of Figure 15 differs from the variant of Figure 14 in that the pivoting element 31 has a recess at the third rotor pin engaging surface 39<sub>3</sub>. This means that in the unlocking position of the lock, the third rotor pin protrudes slightly into the key channel 5. The dimple 25 associated with the third pin engaging surface 39<sub>3</sub> may be used to code the key. In further variants it would be possible to have more than three rotor pins engaging at the same time with the pivoting element 31.

**[0034]** Figures 16 and 17 illustrate further variants of the present invention. In these variants, the longitudinal axis of one or more of the blind holes is oriented at an angle to the key shaft 22 as viewed in the cross-section. The cross section of Figure 16 is taken at the position of the second rotor pin 13<sub>2</sub> (referring for example to Figure 7) and in a plane orthogonal to the longitudinal axis of the key channel 5. The orientation of the first rotor pin 13<sub>1</sub> is also indicated with dashed lines. As can be seen, the first and second rotor pins 13<sub>1</sub>, 13<sub>2</sub> are angled with respect to the key channel 5, such that the first and second rotor pins are not parallel to each other. The angle between the longitudinal axis of the rotor pin and the longitudinal axis of the cross section of the key channel taken in a plane orthogonal to the longitudinal axis of the key channel can for example be in the range of 10 to 60 degrees, or more specifically between 20 and 50 degrees. The variant of Figure 15 is advantageous, if the design constraints of the lock mean that the region directly under the key channel cannot be occupied by the rotor pins 13. In the variant of Figure 17, a single pin engaging surface of the pivoting element 31 is arranged to engage with two angled rotor pins 13. Instead of only having one rotor pin 13 per pin engaging surface, having two rotor pins increases the security of the lock. Using the right tools, a lock could in theory be manipulated by trying to guess or find the unlocking position of the rotor pins 13. However, having two pins arranged as shown in Figure 17 increases the resistance against lock picking.

**[0035]** In the above examples, the keys are reversible. In other words the key shaft 22 can be inserted into the key channel 5 either way up. To achieve this, the same



coding is applied to the top side and bottom side of the key and on both lateral sides, if they happen to have a coding pattern. This means that the top side of the key 3a has the same pivoting element configuration as the bottom side. However, the teachings of the present invention may also be applied to non-reversible keys. In this case, it is possible that the pivoting element(s) 31 is/are only provided on any one side of the key shaft 22. Alternatively, the pivoting elements may be provided on at least two sides of the key shaft 22, but using a different coding. This would further increase the number of coding permutations.

**[0036]** In the above examples, the function of the force transferring means 31 was illustrated using the example of a pivoting element 31. However, according to the present invention, the force transferring means can be any element able to apply a depressing force on the second rotor pin 13<sub>2</sub> as a result of the biasing force exerted by the first rotor pin 13<sub>1</sub>. In this manner the movement of the first rotor pin 13<sub>1</sub> in a first direction causes the second rotor pin 13<sub>2</sub> to move in a second direction, substantially opposite to the first direction, thereby controlling the movement of the second rotor pin 39<sub>2</sub>. Thus, the force transferring means 31 could be for instance be a hydraulic circuit. Or it could simply be a rubber element held in the cavity 33 by friction. Or it could be an elastic element, which is compressed slightly when inserting the element into the cavity, but could then easily move in the cavity 33 once inside it. Or as mentioned above, it could be the pivoting element 31 of which pivot axis is movable. Or it could be a loose mechanical element similar to the pivoting element, constrained in the cavity 33 but without a fixed pivot bearing. In this case, the rotational motion could be guided by the shape of the mechanical element's outer edge and the interaction between this outer edge and the outline shape of the cavity 33. Of course it could be a combination of any of the above examples.

**[0037]** To summarise, the lock 1a according to the present invention comprises: a stator 8 comprising a first stator pin 11<sub>1</sub> and a second stator pin 11<sub>2</sub>; a rotor 9 comprising a key-channel 5, a first rotor pin 13<sub>1</sub> and a second rotor pin 13<sub>2</sub>, the rotor being rotatable in the stator 8 about a rotation axis by rotating an authorised key 3a inserted into the key channel 5, and the first and second rotor pins 13<sub>1</sub>, 13<sub>2</sub> being displaceable along first and second pin channels of the rotor 9 respectively, whereby the first and second pin-channels each open into the key channel 5 at first and second pin-channel openings respectively; and biasing means 12<sub>1</sub>, 12<sub>2</sub> for biasing the first and second stator pins 11<sub>1</sub>, 11<sub>2</sub> towards the key channel 5 along first and second pin-channel axes respectively. The stator pins 11<sub>1</sub>, 11<sub>2</sub> are arranged to align axially with the rotor pins 13<sub>1</sub>, 13<sub>2</sub> when the lock is in a key-insertion state such that the first and second stator pins 11<sub>1</sub>, 11<sub>2</sub> bias the first and second rotor pins 13<sub>1</sub>, 13<sub>2</sub> towards the key channel 5. The first and second rotor pins 13<sub>1</sub>, 13<sub>2</sub> have first and second pin lengths respectively, wherein the axial lengths are such that, when the authorised key

3a is inserted, the stator and rotor pins are displaceable by pin engaging surfaces 39<sub>1</sub>, 39<sub>2</sub> of the key 3a to an unlocking position which permits the rotation of the rotor 9. The first and second pin lengths are such that, when the stator and rotor pins are in the unlocking position, the first rotor pin 13<sub>1</sub> protrudes from the first pin channel opening into the key channel 5, and the second rotor pin 13<sub>2</sub> stops short of the second pin channel opening. The first and second pin channel openings are arranged at substantially the same circumferential position in the key channel 5 relative to the rotation axis.

**[0038]** Furthermore, to summarise, the key 3a, 3b according to the present invention is for a cylinder lock 1 a, 1 b as explained above. The key 3a, 3b comprises:

- the key shaft 22 for inserting into the key channel 5 of the cylinder lock 1 a, 1 b; and
- the force transferring means 31 disposed in the key shaft 22 and comprising the first pin engaging surface 39<sub>1</sub> for engaging with the first rotor pin 13<sub>1</sub> of the cylinder lock 1a, 1b, and the second pin engaging surface 39<sub>2</sub> for engaging with the second rotor pin 13<sub>2</sub> of the cylinder lock 1 a, 1 b, the force transferring means 31 being configured such that, in response to a first force exerted by the first pin 13<sub>1</sub> on the first pin engaging surface 39<sub>1</sub> in a first direction, the second pin engaging surface 39<sub>2</sub> exerts a second force on to the second pin 13<sub>2</sub> in a second direction.

**[0039]** The second direction is substantially opposite to the first direction.

**[0040]** While the invention has been illustrated and described in detail in the drawings and foregoing description, such illustration and description are to be considered illustrative or exemplary and not restrictive, the invention being not limited to the disclosed embodiment. Other embodiments and variants are understood, and can be achieved by those skilled in the art when carrying out the claimed invention, based on a study of the drawings, the disclosure and the appended claims. For example, it is possible to combine teachings from at least two variants explained above to obtain further variants.

**[0041]** In the claims, the word "comprising" does not exclude other elements or steps, and the indefinite article "a" or "an" does not exclude a plurality. The mere fact that different features are recited in mutually different dependent claims does not indicate that a combination of these features cannot be advantageously used. Any reference signs in the claims should not be construed as limiting the scope of the invention.

## Claims

1. A cylinder lock (1 a, 1 b) comprising:
  - a stator (8) comprising a first a stator pin (11<sub>1</sub>),



and a second stator pin (11<sub>2</sub>);

- a rotor (9) comprising a key channel (5) for receiving a key (3a, 3b), the rotor (9) being arranged to be rotated in the stator (8) about a longitudinal axis of the key channel (5) by turning an authorised key (3a, 3b) in the key channel (5);
- biasing means for biasing the first stator pin (11<sub>1</sub>) and a second stator pin (11<sub>2</sub>) towards the key channel (5);
- a first pin channel in the rotor (9) for accommodating a first rotor pin (13<sub>1</sub>) arranged to cooperate with the first stator pin (11<sub>1</sub>), and a second pin channel in the rotor (9) for accommodating a second rotor pin (13<sub>2</sub>) arranged to cooperate with the second stator pin (11<sub>2</sub>), the first and second rotor pins (13<sub>1</sub>, 13<sub>2</sub>) being arranged to engage with the key (3a, 3b), wherein

in an unlocking position of the lock (1a, 1b), the first rotor pin (13<sub>1</sub>) is arranged to protrude into the key channel (5), while a key facing end of the second rotor pin (13<sub>2</sub>) is located recessed in the second pin channel, **characterised in that** the first and second pin channels emerge into the key channel (5) at substantially the same circumferential position, with respect to the longitudinal axis, in the key channel (5).

2. A cylinder lock (1a, 1b) according to claim 2, wherein the first rotor pin (13<sub>1</sub>) and the second rotor pin (13<sub>2</sub>) are angled with respect to each other, and the angle between the first rotor pin (13<sub>1</sub>) and the second rotor pin (13<sub>2</sub>) is between 10 and 60 degrees and more preferably between 20 and 50 degrees.
3. A key (3a, 3b) for a cylinder lock (1 a, 1 b), the key (3a, 3b) comprising:
  - a key shaft (22) for inserting into a key channel (5) of the cylinder lock (1); and
  - a force transferring means (31) disposed in the key shaft (22) and comprising a first pin engaging surface (39<sub>1</sub>) for engaging with a first rotor pin (13<sub>1</sub>) of the cylinder lock (1a, 1b), and a second pin engaging surface (39<sub>2</sub>) for engaging with a second rotor pin (13<sub>2</sub>) of the cylinder lock (1 a, 1 b), the force transferring means (31) being configured such that, in response to a first force exerted by the first pin (13<sub>1</sub>) on the first pin engaging surface (39<sub>1</sub>) in a first direction, the second pin engaging surface (39<sub>2</sub>) exerts a second force on to the second pin (13<sub>2</sub>) in a second direction,

**characterised in that**

the second direction is substantially opposite to the first direction.

4. A key (3a, 3b) according to claim 1, wherein the force transferring means (31) comprises a pivoting element (31) arranged to pivot about a pivot axis (37) perpendicular to a longitudinal axis (29) of the key (3a, 3b).
5. A key (3a, 3b) according to claim 4, wherein the pivot axis (37) is substantially tangential relative to the longitudinal axis (29) of the key (3a, 3b).
6. A key (3a, 3b) according to claim 4 or 5, wherein the pivot axis (37) is movable relative to key (3a, 3b).
7. A key (3a, 3b) according to any one of claims 3 to 6, wherein the force transferring means (31) comprises a third pin engaging surface (39<sub>3</sub>) for engaging with a third rotor pin (13<sub>3</sub>) of the cylinder lock (1).
8. A key (3a, 3b) according to any one of claims 3 to 7, wherein the key (3) comprises at least two force transferring means (31).
9. A key (3a, 3b) according claim 8, wherein the at least two force transferring means (31) are arranged to engage simultaneously with the rotor pins (13) of the cylinder lock (1a, 1b).
10. A key (3a, 3b) according to claim 8, wherein the at least two force transferring means are substantially identical and located on opposite sides of the key shaft (22) substantially in an identical location relative to the longitudinal axis (29) of the key (3a, 3b).
11. A key (3a, 3b) according to any one of claims 3 to 10, wherein the force transferring means (31) is located in a cavity in the key shaft (22), which is a blind hole.
12. A key (3a, 3b) according to any one of claims 3 to 11, wherein the force transferring means (31) is located in a cavity (33) in the key shaft (22), and force transferring means biasing means (44) is provided in the cavity (33) for urging the force transferring means (31) for preventing a leading end of the force transferring means (31) from protruding out of the cavity (33) when the force transferring means (31) is not engaged with rotor pins (13).
13. A key (3a, 3b) according to any one of claims 3 to 12, wherein the force transferring means (31) is located in a cavity (33) in the key shaft (22), and a rotation limiting means (41, 42, 43) is provided in the cavity (33) for limiting the range of rotation of the force transferring means (31).
14. A key (3a, 3b) according to any one of claims 3 to 13, wherein the key (3a, 3b) comprises a first dimple (25<sub>1</sub>) for receiving an end of the first rotor pin (13<sub>1</sub>).



15. A method of coding a key according to claims 14, wherein at least one of the following parameters is used to code the key (3a, 3b): a diameter of the cross-section of the first dimple (25<sub>1</sub>), a depth of the first dimple (25<sub>1</sub>), a position of the first dimple (25<sub>1</sub>) along a longitudinal axis (29) of the key (3a, 3b), a protrusion distance of the second pin engaging surface (39<sub>2</sub>) when the first rotor pin (13<sub>1</sub>) is seated in the first dimple (25<sub>1</sub>), and a location along a longitudinal axis (29) of the key (3a, 3b) of a cavity (33) accommodating the force transferring means (31).

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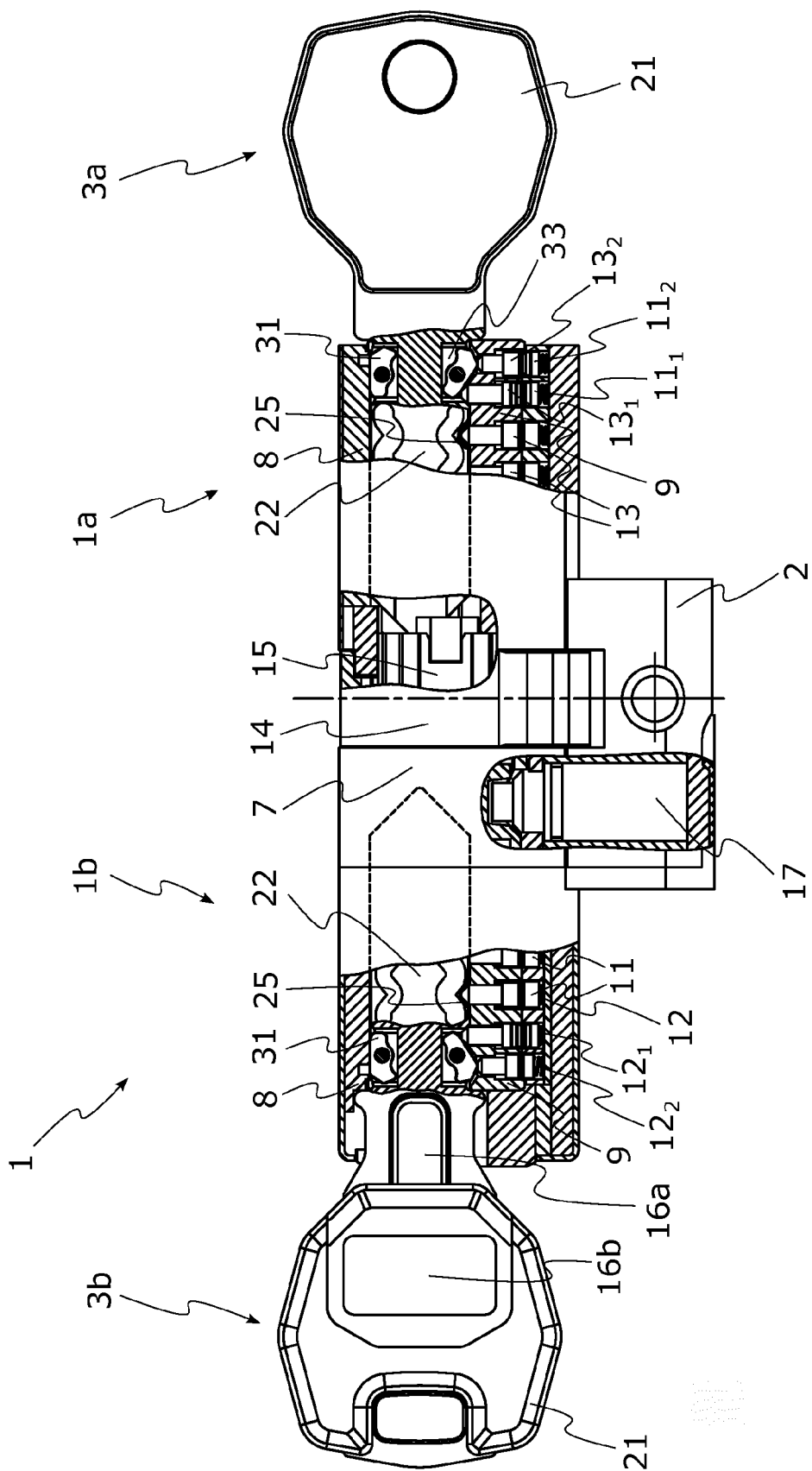
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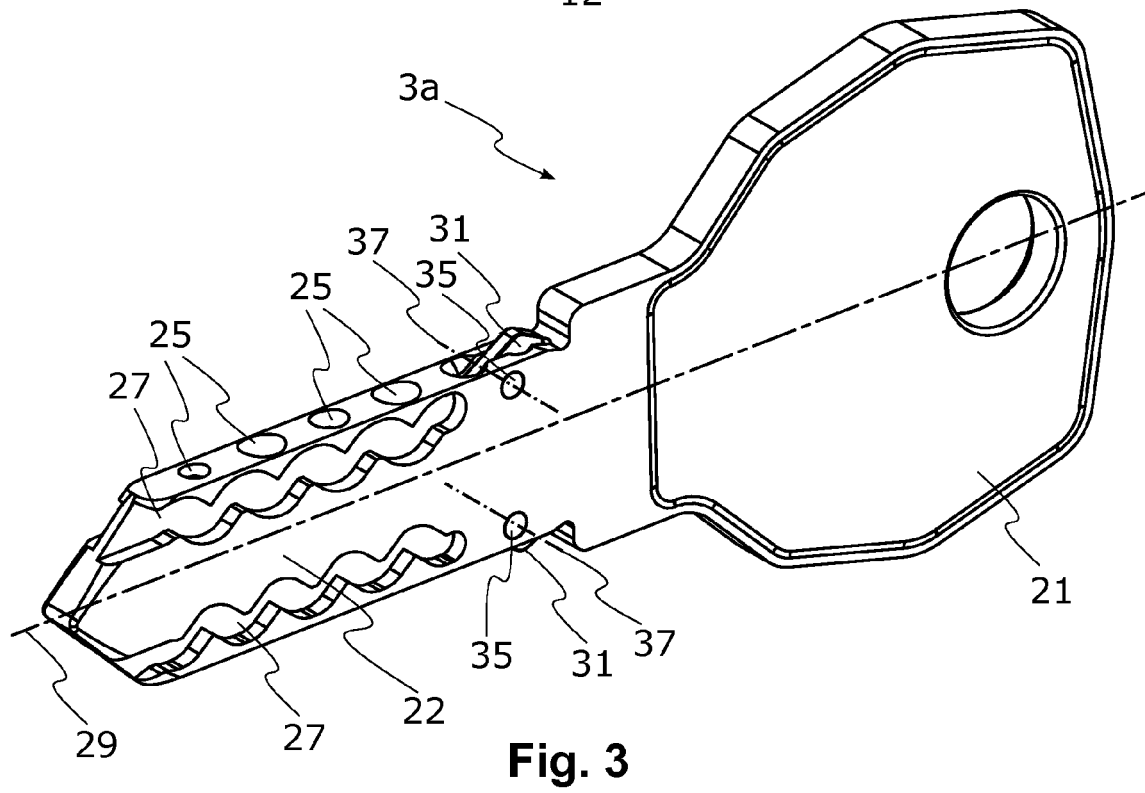
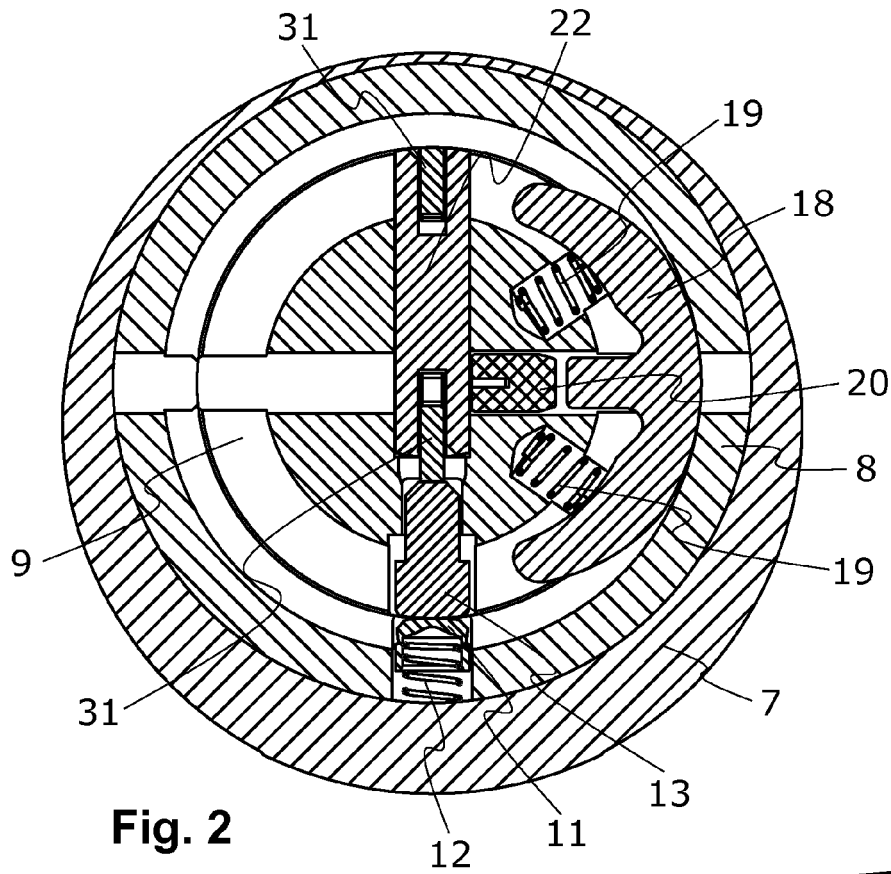
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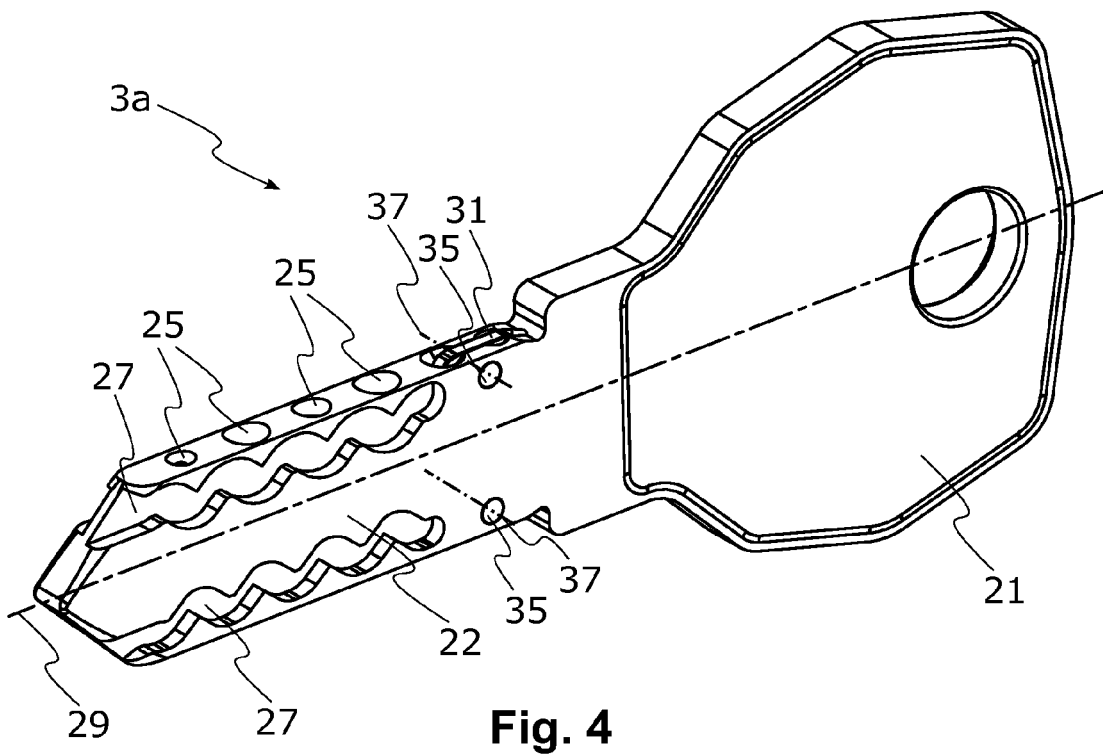




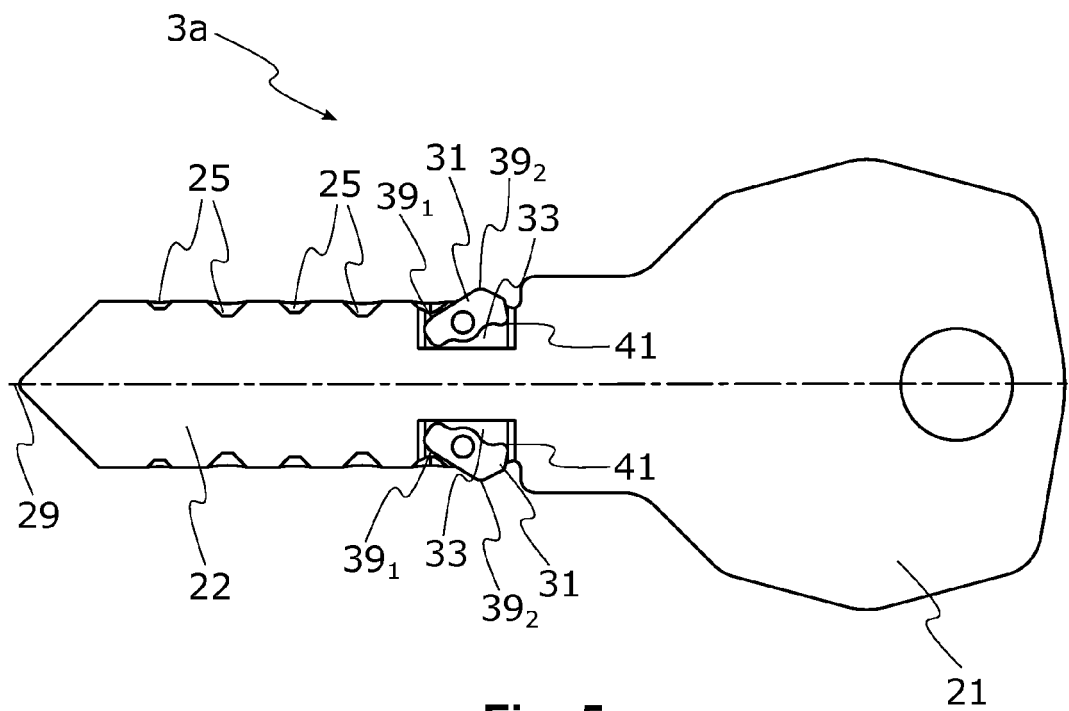






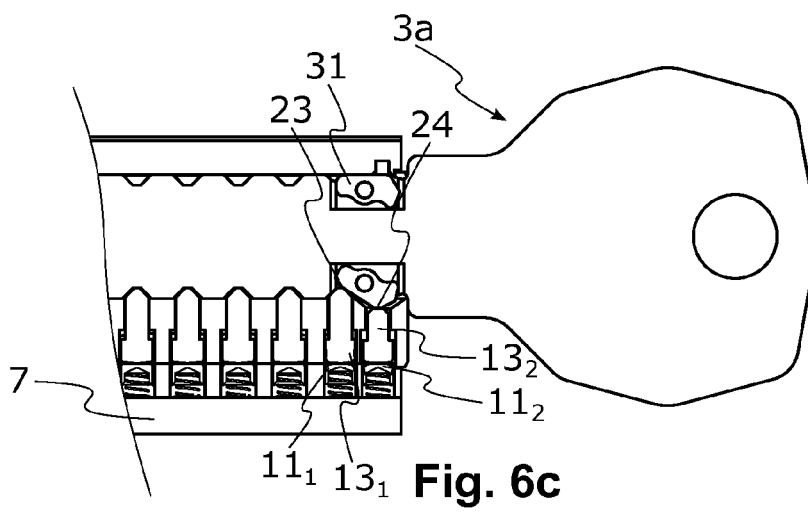
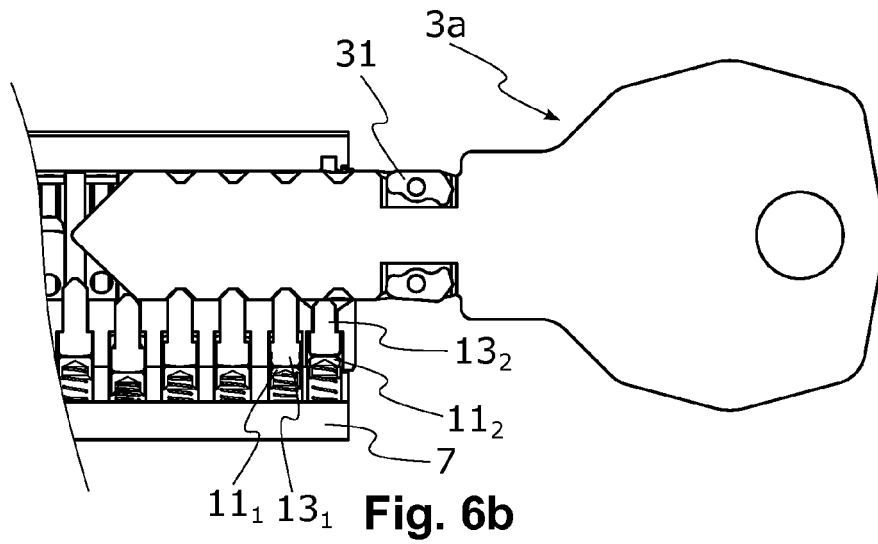
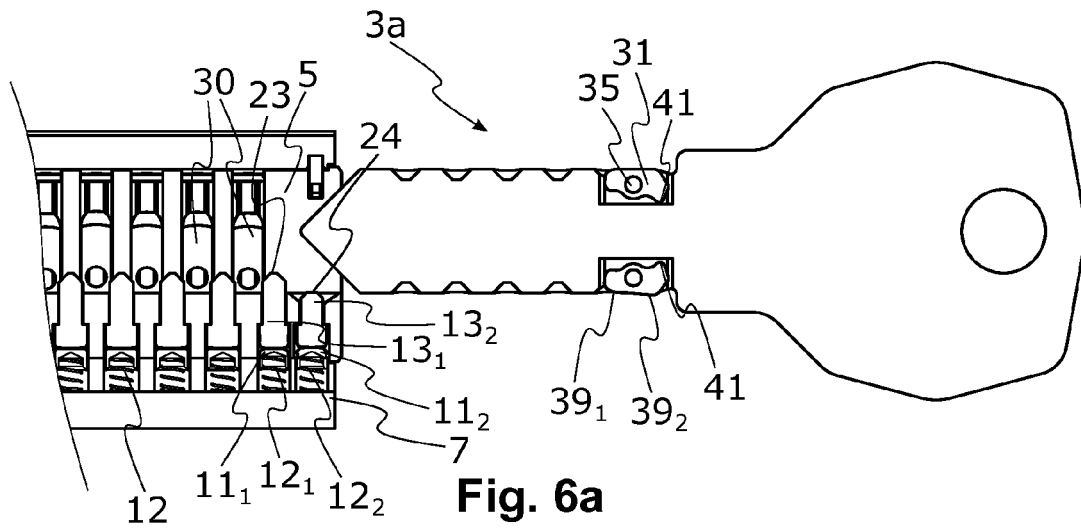


**Fig. 4**

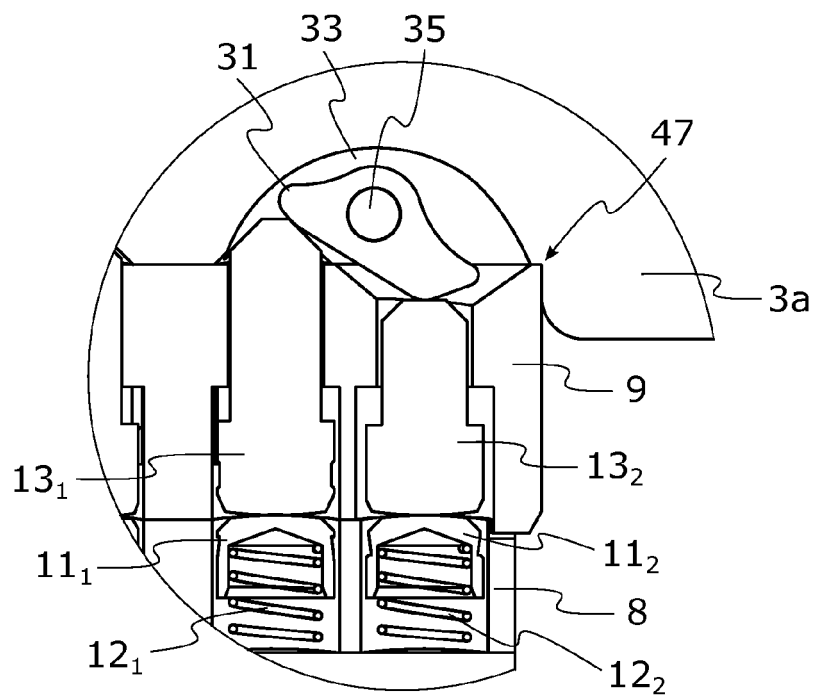


**Fig. 5**

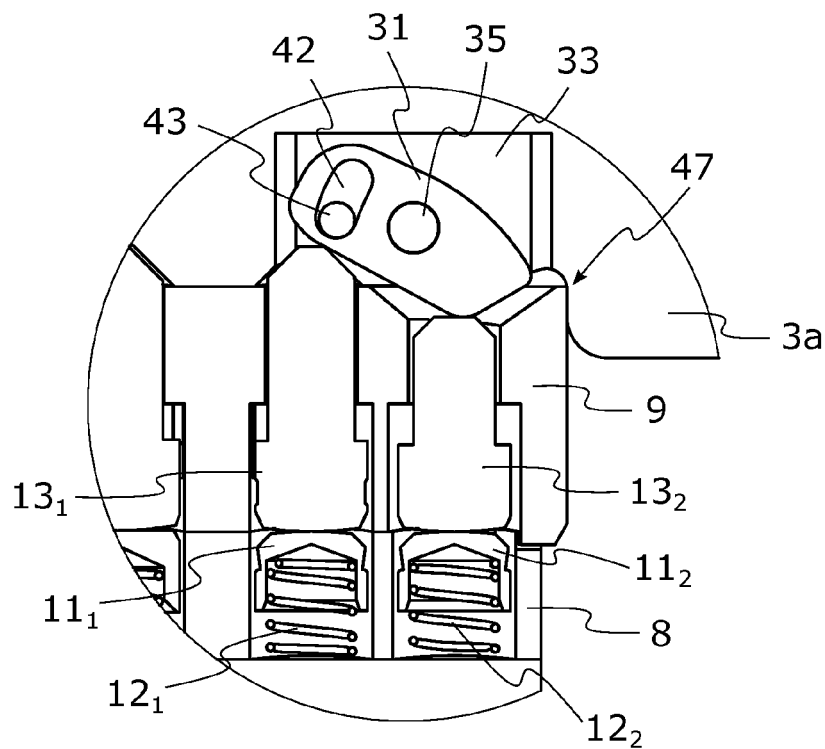






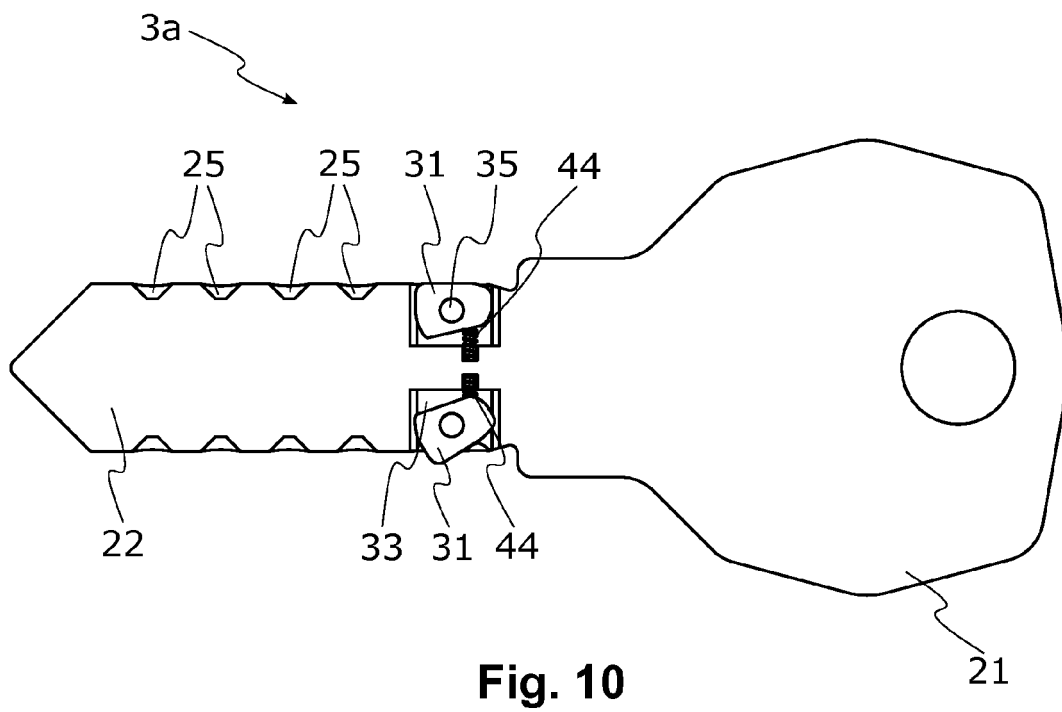
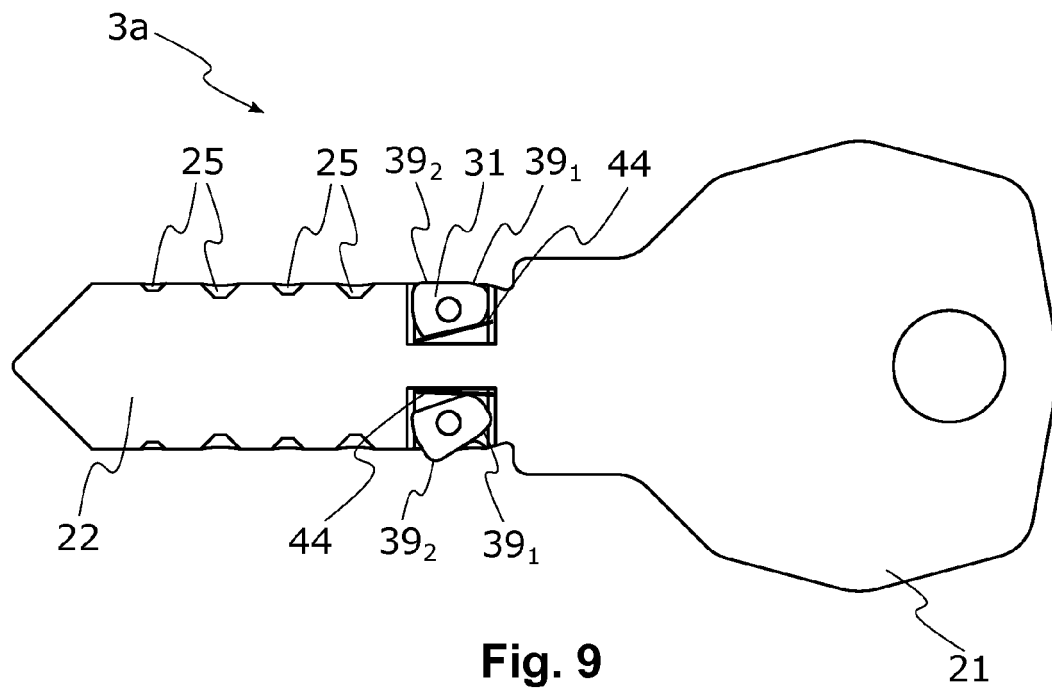


**Fig. 7**

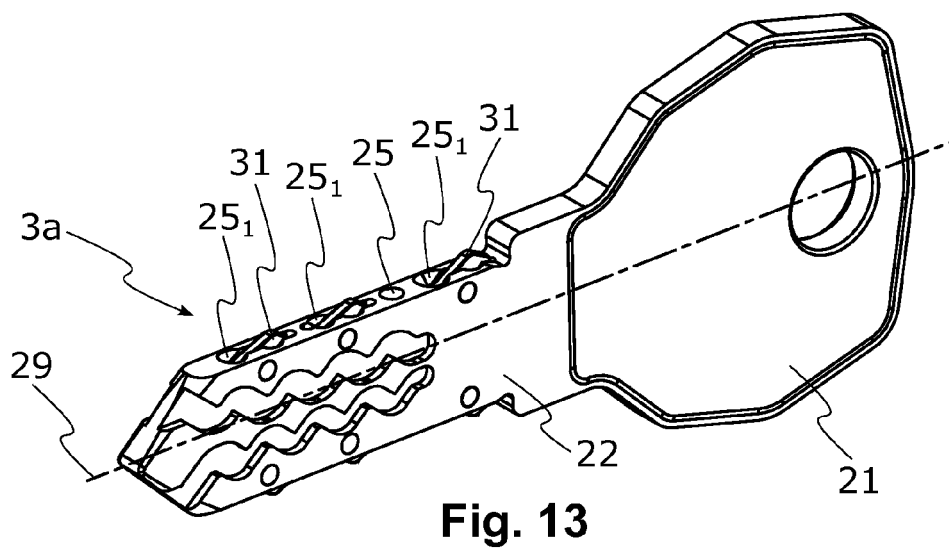
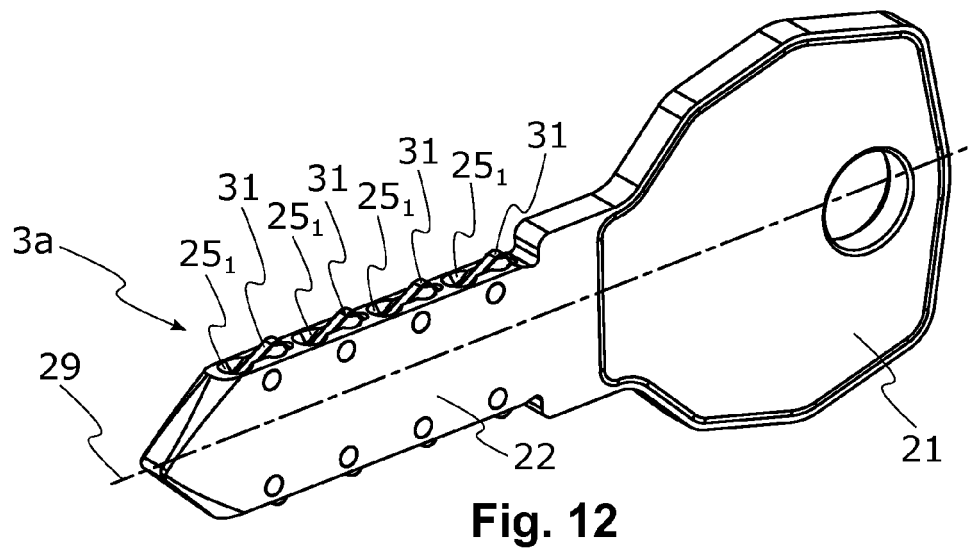
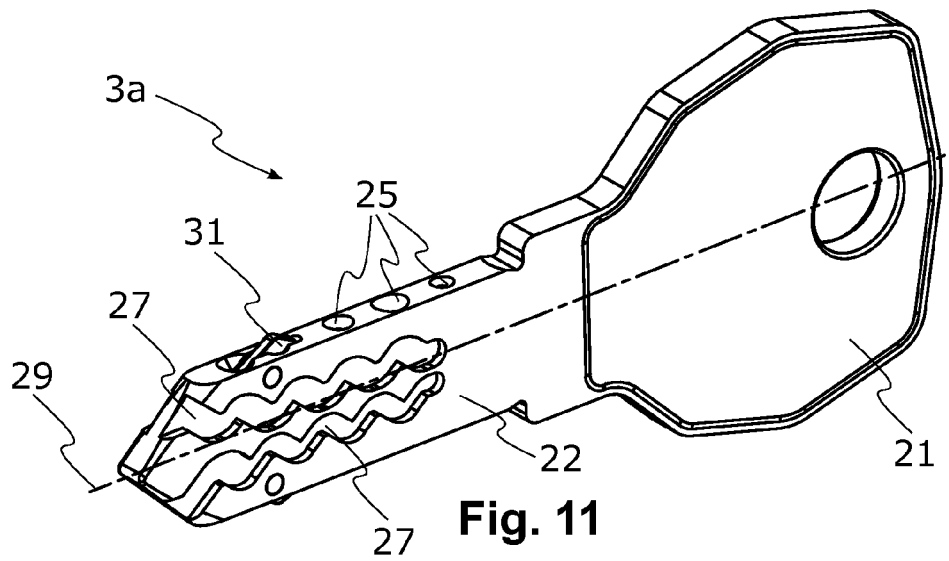


**Fig. 8**

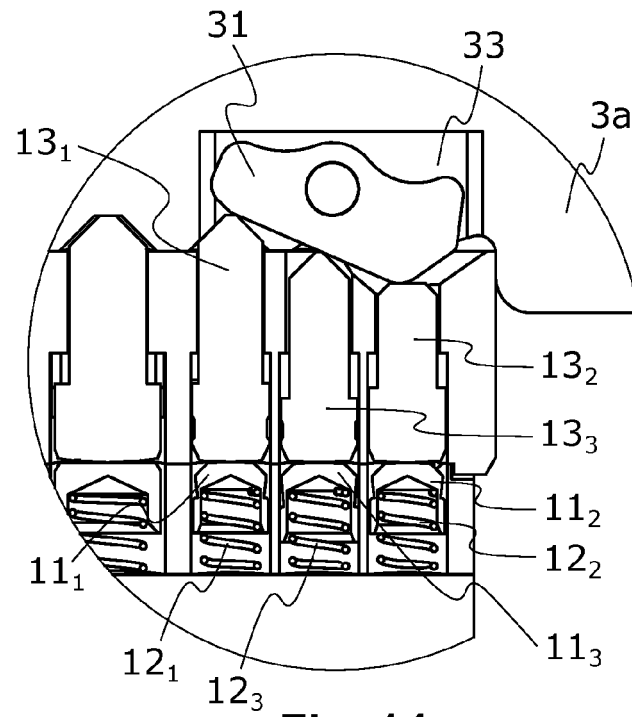




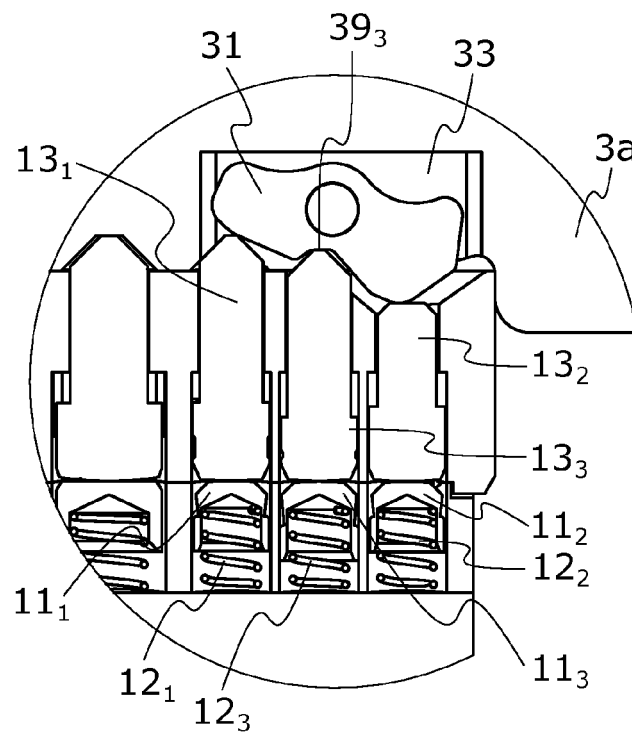






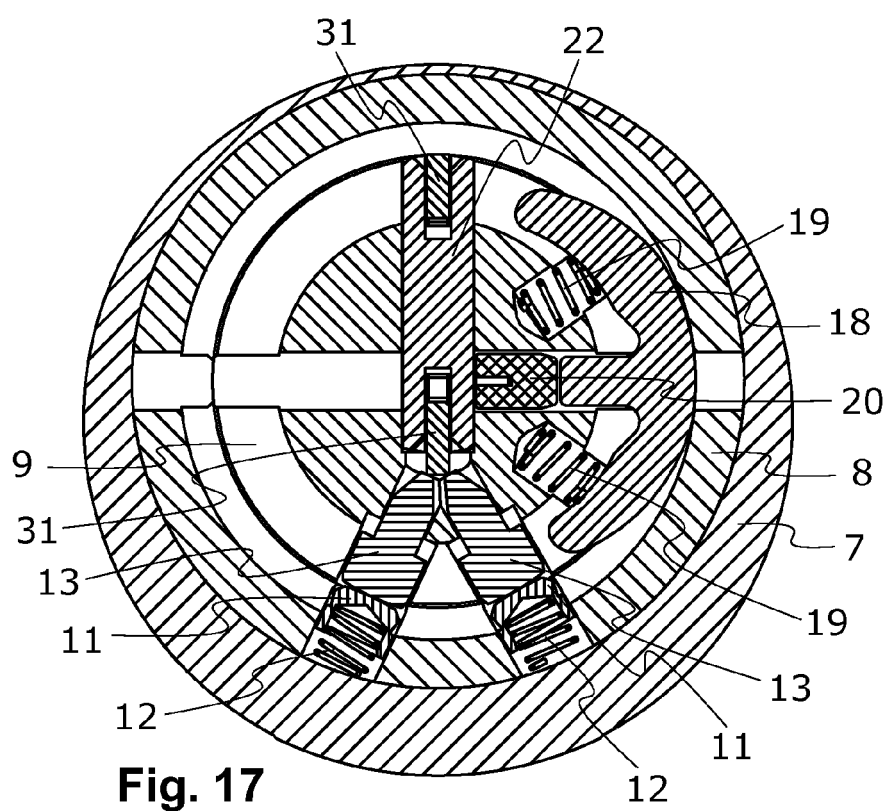
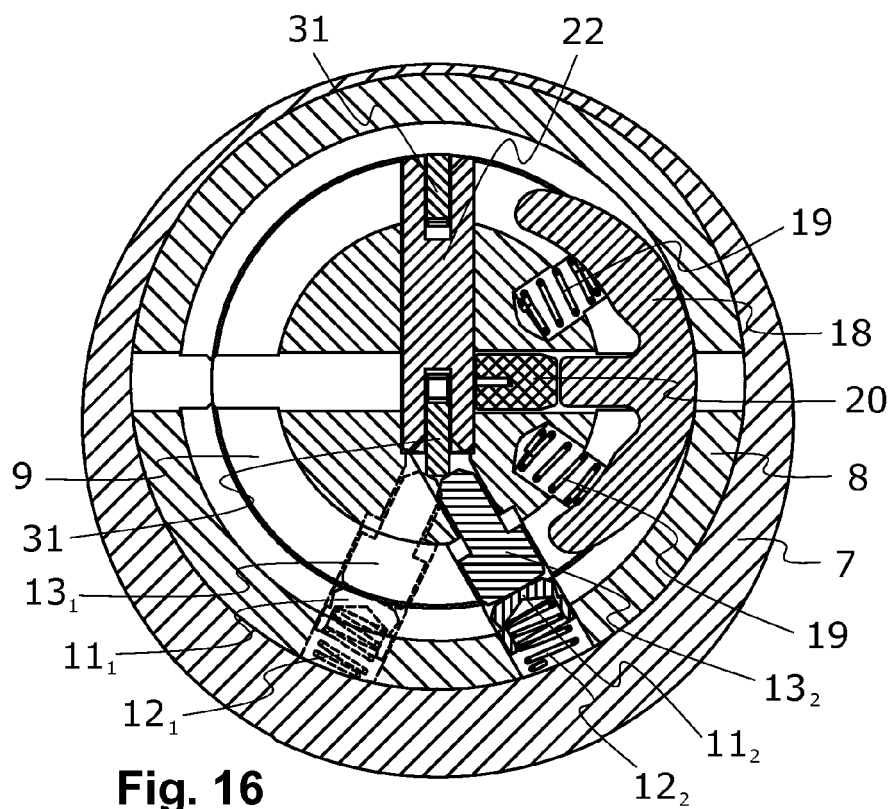


**Fig. 14**



**Fig. 15**









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