



(11) **EP 3 667 059 B1**

(12) **EUROPEAN PATENT SPECIFICATION**

(45) Date of publication and mention of the grant of the patent:
28.07.2021 Bulletin 2021/30

(51) Int Cl.:
F02M 57/02^(2006.01) F02M 43/04^(2006.01)

(21) Application number: **19213763.6**

(22) Date of filing: **05.12.2019**

(54) **A LARGE TWO-STROKE COMPRESSION-IGNITED INTERNAL COMBUSTION ENGINE WITH FUEL INJECTION SYSTEM FOR A LOW FLASHPOINT FUEL AND A FUEL VALVE THEREFORE**

GROSSER ZWEITAKT-VERBRENNUNGSMOTOR MIT KOMPRESSIONSZÜNDUNG MIT KRAFTSTOFFEINSPRITZUNG FÜR KRAFTSTOFF MIT NIEDRIGEM FLAMMPUNKT UND KRAFTSTOFFVENTIL DAFÜR

GRAND MOTEUR À COMBUSTION INTERNE À ALLUMAGE PAR COMPRESSION ET À DEUX TEMPS COMPORTANT UN SYSTÈME D'INJECTION DE CARBURANT POUR CARBURANT À FAIBLE POINT D'INFLAMMABILITÉ ET SOUPEPE DE CARBURANT ASSOCIÉE

(84) Designated Contracting States:
AL AT BE BG CH CY CZ DE DK EE ES FI FR GB GR HR HU IE IS IT LI LT LU LV MC MK MT NL NO PL PT RO RS SE SI SK SM TR

(30) Priority: **11.12.2018 DK PA201870807**

(43) Date of publication of application:
17.06.2020 Bulletin 2020/25

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DescriptionTECHNICAL FIELD

[0001] The disclosure relates to large slow-running two-stroke compression-ignited internal combustion crosshead engines with a fuel injection system for injecting a low flashpoint fuel into the combustion chambers.

BACKGROUND

[0002] Large two-stroke uniflow turbocharged compression-ignited internal combustion crosshead engines are typically used in propulsion systems of large ships or as prime mover in power plants. The sheer size, weight and power output renders them completely different from common combustion engines and places large two-stroke turbocharged compression-ignited internal combustion engines in a class for themselves.

[0003] Large two-stroke compression-ignited internal combustion engines are conventionally operated with a liquid fuel such as e.g. fuel oil or heavy fuel oil but increased focus on environmental aspects has led to the development towards using alternative types of fuel such as gas, methanol, coal slurry, petroleum coke and the like. One group of fuels that is in increasing demand are low flashpoint fuels.

[0004] Many low flashpoint fuels, such as e.g. methanol, ethanol, LPG, DME or biofuel, naphtha, gasoline (petrol), crude gasoline and crude oil are relatively clean fuels that result in significantly lower levels of sulfurous components, NO_x and CO₂ in the exhaust gas when used as fuel for a large low-speed uniflow turbocharged two-stroke internal combustion engine when compared with e.g. using heavy fuel oil as fuel.

[0005] However, there are problems associated with using low flashpoint fuels in a large low-speed uniflow turbocharged two-stroke internal combustion engine. One of those problems is the low flashpoint, which causes significant problems if low flashpoint fuel leaks into one of the other systems of the engine and mixes with another fluid, such as e.g. the lubrication oil system. Low flashpoint fuel is inherently flammable, and vapors thereof can easily form explosive mixtures. Thus, should low flashpoint find its way into another system of the engine it is necessary to stop the engine operation for safety reasons and to clean or replace all of the liquid in such a system, a costly and cumbersome affair for the operator of the engine.

[0006] It is known in the art to provide large two-stroke compression ignited internal combustion engines with a common rail type system that stores and distributes the low flashpoint fuel at the required injection pressure of typically several hundred bar (depending on the type of low flashpoint fuel and the engine requirements), with accumulators close to the fuel valves. The common rail type system is connected to two or three fuel injection valves in the cylinder cover of each cylinder. The fuel

injection valves are electronically controlled and fuel injection is timed by electronically (the signal originates in an electronic control unit but the actual signal to the fuel valve is typically a hydraulic signal, i.e. electronic signal is converted to a hydraulic signal between the electronic control unit and the fuel valve) controlling the time (relative to the engine cycle) at which the fuel injection valve is opened.

[0007] The known common rail type gaseous fuel supply systems for large two-stroke compression-ignited internal combustion engines have disadvantages when operating on LPG or any other similar low flashpoint fuel with a relatively high compressibility. The injection pressure for LPG needs to be as high as 600 bar, which means that the common rail system including all valves, accumulators, pipes, etc., needs to be laid out for this high pressure. Furthermore, the safety concept with the window valves is not well suited for dense gas like LPG, since firstly the gas channels between window valve and fuel valve need to have a very small volume and secondly, monitoring of the gas channel pressure necessary to ensure detection of leakages is made very difficult due to high frequency oscillation excited from closing of the window valve.

[0008] It is also known in the art to use booster pumps and fuel pressure controlled fuel valves for injecting liquid gas, such as e.g. LPG. This concept has the problem associated therewith that the compressibility of LPG is rather large and dependent on pressure, temperature and gas composition. Hence, the delay between the actuation of the pressure booster and the actual gas injection is dependent on those parameters, which will make engine control, i.e. injection amount and particularly injection timing, very difficult. This is a significant problem, since injection timing is critical in compression-ignited engines.

[0009] EP3252291 discloses a fuel supply system that allows precise timing of the injection of the low flashpoint (and more compressible compared to fuel) fuel, using a pressure booster.

[0010] EP3343008 discloses an engine according to the preamble of claim 1, with fuel valves for injecting low flashpoint liquid fuel into the combustion chamber of the engine. Sealing oil is used in clearance around the valve needle and around the booster piston to avoid leakage of low flashpoint fuel.

[0011] However, many of the low flashpoint fuels do not have good ignition properties resulting in an unreliable ignition if no ignition liquid is injected into the combustion chamber just before or together with the low flashpoint fuel.

[0012] There is therefore a need to provide a fuel supply system for LPG and similar low flashpoint fuels that is safe, inexpensive, provides accurate control of the timing of the fuel admission into the cylinders and provides reliable ignition.

[0013] It is known to use separate valves for injecting the ignition liquid into the combustion chamber. However,

engines that operate on low flashpoint fuels are typically dual fuel engines that can also run on conventional fuel such as fuel oil or heavy fuel oil and the cylinder cover of the cylinders is provided with two or three fuel valves for the conventional fuel and two or three fuel valves for the low flashpoint fuel. Thus, with 4 to 6 fuel valves already present the cylinder cover is already crowded without additional valves for injecting the ignition liquid. Further, the amount of ignition liquid that is injected is typically very small compared to the amount of low flashpoint fuel. A typical range for the quantity of ignition liquid fuel is between 1.5 and 5% percent by weight of the low flashpoint fuel. Accordingly, these valves for injecting the ignition liquid are much smaller than the fuel valves for injecting the low flashpoint fuel and such small valves are typically too fragile to last in the harsh environment.

SUMMARY

[0014] It is an object of the invention to provide a large two-stroke turbocharged compression-ignited internal combustion crosshead engine that overcomes or at least reduces the problem indicated above.

[0015] The foregoing and other objects are achieved by the features of the independent claims. Further implementation forms are apparent from the dependent claims, the description and the figures.

[0016] According to a first aspect there is provided a large two-stroke turbocharged compression-ignited internal combustion crosshead engine comprising a plurality of cylinders, a source of pressurized ignition liquid, a source of pressurized sealing oil, and two or more fuel valves arranged in each cylinder, the fuel valve being configured for injecting low flashpoint liquid fuel into the combustion chamber of the engine, the fuel valve comprising:

an elongated valve housing with a rear end and a front end,

a nozzle with a plurality of nozzle holes, the nozzle being disposed at the front end of the elongated valve housing,

a fuel inlet port in the elongated valve housing for connection to a source of pressurized low flashpoint fuel, an actuation fluid port in the elongated valve housing for connection to a source of actuation fluid, an axially displaceable valve needle slidably received in a longitudinal bore in the fuel valve, the valve needle having a closed position and an open position and the valve needle being biased towards the closed position,

the valve needle allowing flow of fuel from the fuel cavity to the plurality of nozzle holes when the valve needle is in its open position and the valve needle preventing flow of fuel from the fuel cavity to the plurality of nozzle holes when the valve needle is in its closed position,

a pump piston received in a first bore in the valve

housing with a pump chamber in the first bore on one side of the pump piston,

the pump piston is slidably received in the first bore with a clearance between the pump piston and the first bore,

an actuation piston received in a second bore in the valve housing with an actuation chamber in the second bore on one side of the actuation piston, the pump piston being operably connected to the actuation piston,

the actuation chamber being fluidically connected to the actuation fluid port,

the pump chamber having an outlet fluidically connected to the fuel cavity and an inlet fluidically connected to the fuel inlet port,

a sealing liquid inlet port connected to the source of pressurized sealing liquid,

a sealing oil flow path connecting the sealing liquid inlet port to the longitudinal bore for sealing the valve

needle in the longitudinal bore, characterized by an ignition liquid inlet port connected to the source of pressurized ignition liquid,

an ignition liquid conduit extending from the ignition liquid inlet port to the clearance for sealing the pump piston in the first bore and for delivering ignition liquid to the pump chamber.

[0017] By providing an inlet port for ignition liquid connected to a source of pressurized ignition liquid and by providing a conduit that transports the ignition liquid to the first bore for sealing the pump piston in the first bore with an overpressure the ignition liquid will seal the pump piston and ignition liquid will reach the pump chamber since the clearance opens to the pump chamber. Thus, a small quantity of ignition liquid reaches the pump chamber and is mixed with the low flashpoint fuel. The small quantity of ignition liquid mixed with the flow flashpoint fuel is pumped to the nozzle and injected into the combustion chamber. The presence of the ignition liquid mixed with the low flashpoint fuel enhances reliable ignition.

[0018] According to a first possible implementation of the first aspect, the pump piston is sealed in the first bore by the ignition liquid delivered via the ignition liquid conduit to the clearance.

[0019] According to a second possible implementation of the first aspect, the clearance opens to the pump chamber and the ignition liquid is delivered to the pump chamber via the clearance.

[0020] According to a third possible implementation of the first aspect, the flow path comprises a conduit in the valve housing connected to the sealing liquid inlet port and/or a spring chamber of a spring that biases the valve needle to the valve seat and/or an axial bore in the valve needle and/or a transverse bore in the valve needle.

[0021] According to a fourth possible implementation of the first aspect, the sealing oil flow path connects the sealing liquid inlet port to the longitudinal bore at a first

position along the length of the longitudinal bore for sealing the valve needle in the longitudinal bore.

[0022] According to a fifth possible implementation of the first aspect the low-pressure conduit connects the low flashpoint fuel inlet port to the longitudinal bore at a second position along the length of the longitudinal bore, the second position being closer to the fuel cavity than the first position. The pressure in the low-pressure conduit is significantly lower than the injection pressure and thus, the connection to the low-pressure conduit "punctures" the high-pressure coming from the fuel cavity and thus, the sealing oil only needs to seal the clearance between the valve needle and the longitudinal bore against the lower fuel pressure in the low-pressure conduit.

[0023] According to a sixth possible implementation of the first aspect, the valve needle rests on a valve seat in the closed position and the valve needle has lift from the valve seat in the open position.

[0024] According to a seventh possible implementation of the first aspect, the nozzle has a nozzle body that is secured to the front of the elongated valve housing.

[0025] According to an eighth possible implementation of the first aspect, the valve seat is located in the tip of the nozzle.

[0026] According to a ninth possible implementation of the first aspect, the longitudinal bore is formed at least partially in the nozzle body.

[0027] According to a tenth possible implementation of the first aspect, the lift of the valve needle is controlled by the fuel pressure in the fuel cavity.

[0028] According to an eleventh possible implementation of the first aspect, the valve needle is operably connected to a needle actuation piston with a pressure surface of the needle actuation piston facing a needle actuation chamber in the fuel valve, the needle actuation chamber being fluidically connected to a control port in the fuel valve for connection to a controllable source of control fluid.

[0029] According to a twelfth possible implementation of the first aspect, the fuel valve is provided with a valve needle that controls the flow of fuel to nozzle holes of the nozzle of the fuel valve, the position of the valve needle preferably being controlled by a control signal and not by the fuel pressure.

[0030] According to a thirteenth possible implementation of the first aspect, the pump chamber has an inlet fluidically connected to the fuel inlet port via a first one-way valve.

[0031] According to a fourteenth possible implementation of the first aspect, the pump chamber having an outlet fluidically connected to the fuel cavity via a second one-way valve.

[0032] According to a fifteenth possible implementation of the first aspect, the effective pressure area of the pump piston is smaller than the effective pressure area of the actuation piston.

[0033] According to a sixteenth possible implementation of the first aspect, the nozzle is part of a nozzle body

that is secured to the front of the elongated valve housing.

[0034] According to a seventeenth possible implementation of the first aspect, the inlet is located in the pump piston.

5 **[0035]** According to an eighteenth possible implementation of the first aspect, a first one-way valve is provided in the inlet, the first one-way valve being configured to allow flow of low flashpoint fuel through the inlet into the pump chamber and to prevent flow from the pump chamber into the inlet.

10 **[0036]** According to a nineteenth possible implementation of the first aspect, the outlet of the pump chamber is connected to the fuel cavity by one or more fuel channels.

15 **[0037]** According to a twentieth possible implementation of the first aspect a second one-way valve is provided in the one or more fuel channels, the second one-way valve being configured to allow flow of low flashpoint fuel from the pump chamber to the fuel cavity and to prevent flow from the fuel cavity to the pump chamber.

20 **[0038]** According to twenty-first possible implementation of the first aspect, the pump piston is operably connected to the actuation piston to move in unison therewith.

25 **[0039]** According to a twenty-third possible implementation of the first aspect, the pressure of the source of pressurized ignition liquid is electronically controllable, and the engine comprises an electronic control unit configured to control the pressure of the source of pressurized ignition liquid.

30 **[0040]** According to a twenty-fourth possible implementation of the first aspect the engine is provided with a non-low flashpoint fuel supply system, and wherein the cylinders are provided with two or more fuel valves for injecting the non-low flashpoint fuel into the cylinders.

35 **[0041]** According to a twenty-fifth possible implementation of the first aspect the engine is provided with a low flashpoint fuel supply system.

40 **[0042]** These and other aspects of the invention will be apparent from the embodiments described below.

BRIEF DESCRIPTION OF THE DRAWINGS

45 **[0043]** In the following detailed portion of the present disclosure, the invention will be explained in more detail with reference to the example embodiments shown in the drawings, in which:

Fig. 1 is an elevated front view of a large two-stroke diesel engine according to an example embodiment.

Fig. 2 is an elevated side view of the large two-stroke engine of Fig. 1.

Fig. 3 is a diagrammatic representation of the large two-stroke engine according to Fig. 1.

55 Fig. 4a is a diagrammatic representation of the fuel injection system for injecting a low flashpoint fuel into the engine of Figs. 1 and 2.

Fig 4b is a diagrammatic representation of an exam-

ple of a fuel valve for use in the engine of Figs. 1 and 2, showing the sources of liquid connected to the fuel valve.

Fig. 5 is an elevated view of a fuel valve according to an embodiment.

Figs. 6 to 9 are different sectional views of the fuel valve of Fig. 5,

Fig. 10 illustrates an alternative nozzle for the fuel valve of Fig. 5.

DETAILED DESCRIPTION

[0044] In the following detailed description, an internal combustion engine will be described with reference to a large two-stroke low-speed uniflow turbocharged compression-ignited internal combustion engine with crossheads in the example embodiments, but it is understood that the internal combustion engine could be of another type, such as a two-stroke Otto, a four-stroke Otto or Diesel, with or without turbocharging, with or without exhaust gas recirculation.

[0045] Figs. 1, 2 and 3 show a large low-speed turbocharged two-stroke diesel engine with a crankshaft 8 and crossheads 9. Fig. 3 shows a diagrammatic representation of a large low-speed turbocharged two-stroke diesel engine with its intake and exhaust systems. In this example embodiment the engine has six cylinders in line. Large low-speed turbocharged two-stroke diesel engines have typically between four and fourteen cylinders in line, carried by a cylinder frame 23 that is carried by an engine frame 11. The engine may e.g. be used as the main engine in a marine vessel or as a stationary engine for operating a generator in a power station. The total output of the engine may, for example, range from 1,000 to 110,000 kW.

[0046] The engine is in this example embodiment a compression-ignited engine of the two-stroke uniflow type with scavenge ports 18 at the lower region of the cylinder liners 1 and a central exhaust valve 4 at the top of the cylinder liners 1. The scavenge air is passed from the scavenge air receiver 2 to the scavenge ports 18 of the individual cylinders 1. A piston 10 in the cylinder liner 1 compresses the scavenge air. Fuel is injected through fuel valves 50 in the cylinder cover 22. Combustion follows, and exhaust gas is generated. The fuel valves 50 are suitable for injecting a low flashpoint fuel into the combustion chamber. In an embodiment the engine is additionally provided with fuel valves 51 that are suitable for injecting a conventional (non-low flashpoint fuel, such as e.g. fuel oil or heavy fuel oil) fuel into the combustion chamber.

[0047] When an exhaust valve 4 is opened, the exhaust gas flows through an exhaust duct associated with the cylinders 1 into the exhaust gas receiver 3 and onwards through a first exhaust conduit 19 to a turbine 6 of the turbocharger 5, from which the exhaust gas flows away through a second exhaust conduit via an economizer 20 to an outlet 21 and into the atmosphere. Through

a shaft, the turbine 6 drives a compressor 7 supplied with fresh air via an air inlet 12. The compressor 7 delivers pressurized scavenge air to a scavenge air conduit 13 leading to the scavenge air receiver 2. The scavenge air in the scavenge air conduit 13 passes an intercooler 14 for cooling the scavenge air.

[0048] The cooled scavenge air passes via an auxiliary blower 16 driven by an electric motor 17 that pressurizes the scavenge air flow when the compressor 7 of the turbocharger 5 does not deliver sufficient pressure for the scavenge air receiver 2, i.e. in low or partial load conditions of the engine. At higher engine loads the turbocharger compressor 7 delivers sufficient compressed scavenge air and then the auxiliary blower 16 is bypassed via a non-return valve 15.

[0049] The engine is operated with a low flashpoint fuel, such as e.g. LPG, methanol or naphtha and supplied by a low flashpoint fuel supply system 30 in liquid or supercritical form at a substantially stable pressure and temperature. However, depending on the details of the low flashpoint fuel supply system and the type of gas supplied slight variations in temperature and pressure are unavoidable. Further, slight variations in the composition of the low flashpoint fuel can also occur. In an embodiment, the engine is a dual fuel engine and is also provided with a conventional fuel supply system (not shown) for supplying a non-low flashpoint fuel, such as e.g. fuel oil or heavy fuel oil.

[0050] The low flashpoint fuel supply system 30 supplies fuel injection valves 50 with low flashpoint fuel at relatively low supply pressure (e.g. 8 to 100 bar pressure) via a supply conduit 31.

[0051] Fig. 4a is a diagram showing the fuel injection system that receives the low flashpoint fuel via the supply conduit 31. The fuel injection system comprises a pressure booster 40 for pressurizing the fuel to the injection pressure. The pressure booster 40 is hydraulically actuated under control of a first control valve 41. The fuel valves 50 are hydraulically actuated under control of a second control valve 45.

[0052] The diagram in Fig. 4a shows the fuel injection system for a single cylinder 1, with one pressure booster 40 and three fuel injection valves 50. Instead of three, there could also be two fuel valves 50 for each cylinder 1. Each cylinder 1 will require a pressure booster 40 supplying two or three fuel valves 50.

[0053] The pressure booster 40 comprises a large diameter plunger connected to a smaller diameter plunger to move in unison therewith. The large diameter plunger and the small diameter plunger are received in respective matching bores in the housing of the pressure booster 40. The large diameter plunger faces an actuation chamber that is supplied with high pressure hydraulic fluid or with tank under control of the first control valve 41.

[0054] The small diameter plunger faces a pump chamber that is connected to the fuel supply conduit 31 via a one-way valve and to a high-pressure fuel supply line 35 for the delivery of high pressure fuel to the fuel

valves 50. A one-way valve prevents backflow of fuel from the high-pressure fuel supply line 35 to the pump chamber. The pressure of the fuel in the supply conduit 31 is sufficient to cause the pressure booster 40 to make a return stroke when the actuation chamber is connected to tank. A position sensor 34 senses the position of the large and small diameter plungers.

[0055] In the present embodiment the first control valve 41 comprises a preferably proportional first hydraulically controlled three-way valve 42. The first three-way valve 42 is connected to the actuation chamber via an actuation conduit 44, to a source of high-pressure hydraulic fluid and to tank. The first three-way valve 42 is configured to connect the actuation chamber selectively to tank or to the source of high-pressure hydraulic fluid. Since the first hydraulically controlled three-way valve 42 is in an embodiment a proportional valve capable of assuming any intermediate position between connection to the source of high-pressure hydraulic fluid and tank. The position of the first three-way valve 42 is controlled by a first smaller two-way valve 43 and the position of the first smaller two-way valve 43 is electronically controlled. The first smaller two-way valve 43 is connected to an electronic control unit 25 via a first signal cable 26. In an embodiment, the first control valve 41 is under command of a separate electronic control unit that is mainly configured for maintaining safety and will deactivate the pressure booster when a safety issue, such as e.g. a gas leak, has been detected. Alternatively, the first control valve is connected to a source of high pressure hydraulic fluid that is controlled by an engine safety system.

[0056] The high-pressure fuel supply line 35 is split up into three high-pressure fuel supply lines 35-1, 35-2, 35-3, i.e. one high-pressure fuel supply line for supplying each fuel valve 50 with high-pressure low flashpoint fuel. In an embodiment with two fuel valves 50 per cylinder the high-pressure fuel supply line 35 will be split in two lines.

[0057] As shown in Fig. 4b, each fuel valve 50 is connected to a supply of pressurized sealing oil Ps via a sealing oil supply line 36 and with a sealing oil return line. The flow sealing oil through the fuel valve 50 is in an embodiment relatively large so that the sealing oil also acts as a cooling medium for the fuel valve 50.

[0058] Each fuel valve 50 is connected to a fuel valve actuation signal conduit 48. The pressure in the fuel valve actuation signal conduit 48 is controlled by a second control valve 45 that comprises in an embodiment a second hydraulically controlled proportional three-way valve 46 and a second smaller electronically controlled two-way valve 47. The second hydraulically controlled three-way valve 46 is preferably a proportional valve and is configured to connect the actuation signal conduit 48 to a source of high-pressure hydraulic fluid or to tank. Since the second hydraulically controlled three-way valve 46 is in an embodiment a proportional valve it is capable of assuming any intermediate position between connection to the source of high-pressure hydraulic fluid and tank.

The position of the second three-way valve 46 is controlled by a second smaller two-way valve 47 and the position of the second smaller two-way valve 47 is electronically controlled. The second smaller two-way valve 47 is connected to the electronic control unit 25 via a third signal cable 28. The electronic control unit 25 is informed of the position of the second three-way valve 46 via a second signal cable 27.

[0059] The electronic control unit 25 is in receipt of signals from various sensors via signal cables that are illustrated in Fig. 4a as interrupted lines. The signals from the various sensors include e.g. scavenging pressure, temperature, exhaust pressure, temperature, crank angle and speed, although it is noted that this list is not exhaustive and will depend on the construction of the engine, for example whether it includes exhaust gas recirculation or not, whether it includes a turbocharger or not, etc. The electronic control unit 25 controls the fuel injection valves 50 i.e. the electronic control unit determines when the fuel valve 50 is open and determines the duration of the opening time. The electronic control unit 25 also controls the operation of the pressure booster 40.

[0060] The timing of the fuel injection highly affects the combustion pressure in the large two-stroke turbocharged diesel engine (compression-ignited engine). The timing of the opening of the fuel valves 50 relative to the crankshaft angle or relative to the engine cycle largely determines the combustion pressure. The duration of the opening of the fuel valves 50 determines the amount of fuel admitted to the cylinders 1, with increasing duration leading to increasing amount of fuel being admitted to the cylinders 1.

[0061] The electronic control unit 25 is configured to control the timing of the opening of the fuel valves by an electronic signal to the second control valve 45 via the third signal cable 28. Upon receipt of the signal the electronic control valve switches position and connects the actuation signal conduit 48 to the source of high-pressure hydraulic fluid. The high pressure in the actuation signal conduit 48 opens the fuel valve 50.

[0062] In the embodiment above the pressure booster 40 and the fuel valve 50 are separate physical units. In an embodiment the pressure booster 40 is an integral part of the fuel valve 50.

[0063] An embodiment of a fuel valve 50 with an integrated pressure booster is shown in Figs. 5 to 9.

[0064] Fig. 5 is a perspective view of the fuel valve 50 with its elongated valve housing 52, a nozzle 54 secured to the front end of the elongated valve housing 52. The nozzle 54 is provided with a plurality of nozzle holes 56 for creating jets of fuel into the combustion chamber. The nozzle 54 is removably secured to the elongated valve housing 52, so that it can be easily replaced if the nozzle 54 should fail or be worn out.

[0065] Figs. 6,7,8 and 9 show different sectional views of the fuel valve 50. The fuel valve 50 has an elongated valve housing 52 with a rearmost end and a nozzle 54 at its front end. The nozzle 54 is a separate body that is

attached to the front end of the valve housing 52. The rearmost end of the valve housing 52 is provided with a plurality of ports, including a control port 86, an actuation fluid port 78 and a gas leak detection port (not shown). The rearmost end is enlarged to form a head that protrudes from the cylinder cover when the fuel valve 50 is mounted in the cylinder cover. In the present embodiment, the fuel valves 50 are placed around the central exhaust valve 4, i.e. relatively close to the wall of the cylinder liner. The elongated valve housing 52 and the other components of the fuel injection valve 50, as well as the nozzle, are in an embodiment made of steel, such as e.g. tool steel or stainless steel.

[0066] The nozzle 54 is provided with nozzle holes that are connected to the interior of the nozzle 54 and the nozzle holes are arranged in different directions in order to distribute the fuel in the combustion chamber. The nozzle holes are directed away from the cylinder liner which is relatively near due to the location of the fuel valve 50 in the cylinder head. The nozzle holes are arranged in the tip 56 of the nozzle 54. Further, the nozzle holes are directed such that they are roughly in the same direction as the direction of the swirl of the scavenge air in the combustion chamber caused by the configuration of the scavenge ports (this swirl is a well-known feature of large two-stroke turbocharged internal combustion engines of the uniflow type).

[0067] The nozzle 54 is connected to the front end of the valve housing 52 with a union nut 57 securing and surrounding a portion of the nozzle body 55, surrounding an intermediate section 53 and surrounding a distal portion of the elongated valve housing 52. The nozzle body 55 is provided with a longitudinal bore in which the valve needle 61 is received. The longitudinal bore has a diameter that is larger than the diameter of the valve needle 61 in the portion of the longitudinal bore closest to the tip 56. The space between the longitudinal bore and the valve needle 61 forms a fuel cavity 58. An intermediate section of the longitudinal bore has a small clearance with the valve needle 61. The portion of the longitudinal bore in the nozzle body 55 most distant from the tip 56 of the nozzle 54 has an enlarged diameter matching and an enlarged diameter portion of the valve needle 61. The enlarged diameter portion of the valve needle 61 forms a needle actuation piston 62 with a pressure surface of the needle actuation piston 62 facing a needle actuation chamber 88 in the nozzle 54. The needle actuation chamber 88 is fluidically connected to the control port 86 via a control conduit 87. The control port 86 is connected to a source of control oil P_c .

[0068] The enlarged diameter section of the longitudinal bore is aligned with a spring chamber 96 in the intermediate section 53. The spring chamber 96 is aligned with a longitudinal bore in the elongated valve housing 52. The distal section of the longitudinal bore in the elongated valve housing 52 closest to the distal end of the elongated valve housing 52 has a diameter that corresponds to the diameter and the spring chamber 96. A

helical wire spring 68 extends between the distal section of the longitudinal bore in the elongated valve housing 52 and the enlarged diameter section 62 of the valve needle 61. The valve needle 61 is resiliently biased towards its closed position by the pre-tensioned helical wire spring 68. The helical wire spring 68 is a helical wire spring that is received in a spring chamber 96 in the elongated fuel valve housing 52. The helical wire spring 68 biases the valve needle 61 towards the tip 56 of the nozzle 54, i.e. to its closed position. In the closed position of the valve needle 61 the, preferably conical, tip of the valve needle 61 abuts with a preferably conical valve seat 63 in the tip 56 of the interior of the nozzle 54 and closes the fluidic connection between the fuel cavity 58 and the nozzle holes. The fluidic connection between the fuel cavity 58 and the nozzle holes is established when the valve needle 61 is lifted, i.e. when the valve needle 61 is forced towards the proximate end of the fuel valve 50 against the bias of the helical wire spring 68. The valve needle 61 is lifted when the needle actuation chamber 88 is pressurized.

[0069] A spring guide 69 extends concentrically in the spring chamber 96 for guiding the helical wire spring 68. The proximate end of the spring guide 69 is sealingly received in the longitudinal bore in the elongated valve housing 52.

[0070] The axially displaceable valve needle 61 is slidably received with a small clearance in a longitudinal bore in the nozzle body 55, and lubrication between the axially displaceable valve needle 61 and the longitudinal bore is critical. Hereto, pressurized sealing liquid is delivered to the small clearance between the longitudinal bore in the valve needle via a conduit (channel) 93. The channel 93 connects the small clearance between the valve needle 61 and the longitudinal bore to a sealing liquid inlet port 70, which in turn can be connected to the source of pressurized sealing liquid. The connection between the small clearance and the channel 93 includes a transverse bore 99 (Fig. 8) in the valve needle 61 that connects to an axial bore 97 (Fig. 8) in the valve needle 61 that extends all the way through the enlarged diameter section that forms the needle actuation piston 62 to the spring chamber 96. The channel 93 connects to the spring chamber 96 and supplies the spring chamber 96 with pressurized sealing liquid. In order to allow a substantial flow of sealing liquid through the spring chamber 96 so that the sealing liquid may act as a cooling medium the spring chamber 96 is connected via a bore to a sealing liquid outlet port 95. The sealing liquid prevents leakage of low flashpoint fuel through the small clearance between the valve needle 61 and the axial bore and provides cooling to the fuel valve 50. Further, the sealing liquid, which is preferably an oil, provides for lubrication between the valve needle 61 and the longitudinal bore.

[0071] The elongated valve housing 52 is provided with a fuel inlet port 76 for connection to a source of pressurized low flashpoint liquid fuel, for example via the low flashpoint liquid fuel supply conduit 31. The fuel inlet port

76 connects to a pump chamber 82 in the valve housing 52 via a conduit 73 in a pump piston 80 and a one-way valve 89, preferably a spring loaded poppet valve. The one-way valve 89 (suction valve) is provided in the pump piston 80 at an inlet 71 of the conduits 73. The one-way valve 89 is a spring loaded poppet valve that ensures that liquid low flashpoint fuel can flow from the fuel inlet port 76 via conduit 73 to the pump chamber 82, but not in the opposite direction. The fluidic connection between the conduit 73 in the pump piston 80 and the fuel inlet port 76 in the elongated valve housing 52 is established by a receded area 74 in the pump piston 80 that in axial direction overlaps with the bore in the elongated valve housing 52 that forms the fuel inlet port 76.

[0072] The pump piston 80 is slidably and sealingly received in a first bore 81 in the elongated fuel valve housing 52 with a pump chamber 82 in the first bore 81 on one side of the pump piston 80. An actuation piston 83 is slidably and sealingly received in a second bore 84 in the valve housing 52 with an actuation chamber 85 in the second bore 84 on one side of the actuation piston 83. The pump piston 80 is connected to the actuation piston 83 to move in unison therewith, i.e. the pump piston 80 and the actuation piston 83 can slide in unison with their respective bores 81,84. In the present embodiment the pump piston 80 and the actuation piston 83 are formed as a single body. However, it is noted that the pump piston 80 and the actuation piston 83 can be separate interconnected bodies.

[0073] The actuation chamber 85 is fluidically connected to an actuation fluid port 78. The first control valve 41 controls the flow pressurized actuation liquid to and from the actuation fluid port 78 and thereby to and from to the actuation chamber 85.

[0074] A lead time before the start of injection event, the electronic control unit 25 commands the first control valve 41 to allow high pressure actuation liquid into the actuation chamber 85. At this moment the actuation piston 83 and pump piston 80 combination is in the position shown in Fig. 6. The pressurized actuation liquid in the actuation chamber 85 acts on the actuation piston 83, thereby creating a force that urges the pump piston 80 into the pump chamber 82. Thereby, the pressure of the low flashpoint liquid fuel in the pump chamber 82 increases. In an embodiment the diameter of the actuation piston 83 is larger than the diameter of the pump piston 80 and thus the pressure in the pump chamber 82 will be correspondingly higher than the pressure in the actuation chamber 85, and the combination of the actuation piston 83 and pump piston 80 acts as a pressure booster.

[0075] One or more fuel channels 79 fluidically connect the pump chamber 82 to the fuel cavity 58 and thereby to the valve seat that is located at the bottom of the fuel cavity 58. A one-way valve 90 is placed between the fuel channels 79 and the pump chamber 82. The outlet 66 of the pump chamber 82 is connected to the inlet of the one-way valve 90. The one-way valve 90 comprises a valve member slidably received in an axial bore in the elongat-

ed valve housing 52 and the valve member is resiliently biased towards its seat, i.e. towards its closed position and prevents backflow of fuel from the fuel channels 79 into the pump chamber 82.

[0076] The pressurized fluid in the actuation chamber 85 will cause the actuation piston 83 and the pump piston 80 to move downwards (downwards as in Figs. 6 to 9) as shown in Fig. 7. The pressure in the pump chamber 82 will after a short compression phase be the product of the ratio between the effective pressure area of the pump piston 80 and the effective pressure area of the actuation piston 83 and the pressure in the actuation chamber 85. At this moment the pressure in the actuation chamber 85 will be substantially equal to the pressure of the source of high pressure fluid. With an effective pressure surface ratio of e.g. 2,5:1 and a supply pressure of the hydraulic system of e.g. 225 to 300 bar, the pressure in the fuel in pump chamber will be approximately 500 bar at the end of the compression phase. Thus, the combination of the actuation piston 83 and pump piston 80 acts as a pressure booster.

[0077] The electronic control unit 25 pressurizes the actuation chamber 85 before the start of the fuel injection by a lead time sufficient to ensure that the pressure in the pump chamber 82 has reached the required injection pressure of e.g. 500 bar. The electronic control unit 25 determines when the valve needle 61 needs to lift and thus when the fuel injection commences. The valve needle 61 is configured to move in the direction away from the nozzle 54 to obtain lift, and towards the nozzle 54 to reduce lift. The valve needle 61 gets lift when the needle actuation chamber 88 is pressurized. The electronic control unit 25 instructs the second control valve 45 to connect the fuel valve actuation signal conduit 48 to the source of high pressure hydraulic fluid at the moment in the engine cycle when the fuel injection has to commence. The fuel valve actuation signal conduit 48 is connected to the control port 86 and the high pressure fluid reaches the needle actuation chamber 88 via control conduit 87. When the valve needle 61 has lift from its seat it allows flow of low flashpoint liquid fuel from the fuel cavity 58 through the nozzle holes into the combustion chamber.

[0078] The electronic control unit 25 ends an injection event by instructing the second control valve 45 to connect the needle actuation chamber 88 to tank and thereupon the valve needle 61 returns to its seat and prevents further injection of fuel. Simultaneously or shortly thereafter, the electronic control unit 25 instructs the first control valve 41 to connect the actuation chamber 85 to tank. The pump chamber 82 is connected to the pressurized source of low flashpoint fuel 30 and the supply pressure of the low flashpoint liquid fuel that flows in via the one-way valve 89 urges the actuation piston 83 into the actuation chamber 85 until it has reached the position that is shown in Fig. 6 with the pump chamber 82 completely filled with low flashpoint liquid fuel so that the fuel valve 50 is ready for the next injection event. Fig. 8 shows the

position of the pump piston 80 and the actuation position 83 near the end of an injection event with a major part of the pump chamber 80 depleted from low flashpoint fuel.

[0079] An injection event of the low flashpoint fuel is controlled by the electronic control unit 25 by the timing and the duration of lift of the valve needle 61. The electronic control unit can also control the injection event by regulating the pressure supplied to the actuation chamber 85 in order to perform rate shaping.

[0080] The fuel valve 50 is provided with a sealing liquid inlet port 70 for connection to a source of pressurized sealing liquid Ps. In an embodiment, the pressure of the source of sealing liquid is at least as high as the maximum pressure in the pump chamber 82 during an injection event. In another embodiment the pressure of the source of sealing liquid is at least as high as the supply pressure of the low flashpoint fuel.

[0081] The sealing liquid is provided to the small clearance between the longitudinal bore and the valve needle 61 via transverse bore 99 and needs only to seal against the supply pressure of the fuel since the fuel inlet port 76 is connected to a low-pressure fuel conduit 98 that extends from the fuel inlet port 76 through the elongated valve housing 52, through the intermediate section 53 into the nozzle body 55 and to the longitudinal bore in which the valve needle 61 is slidably received. The position at which the low-pressure fuel conduit 98 connects to the longitudinal bore is axially between the position where the fuel channels 79 connect to the longitudinal bore and the position where the transverse bore 99 connects to the longitudinal bore. Thus, any low flashpoint fuel leaking upwards (upward as in Figs. 6 to 9)) through the small clearance between the longitudinal bore and the valve needle 61 from the high pressure fuel cavity 58 during an injection event will have its pressure reduced to the much lower fuel feed pressure when it reaches the position where the low-pressure fuel conduit 98 connects to the longitudinal bore. Thus, the low-pressure in the low-pressure fuel conduit 98 "punctures" the pressure of the fuel in the small clearance between the valve needle 61 and the longitudinal bore and thus, the sealing liquid from the transverse bore 99 needs only to seal against the feed pressure of the low flashpoint fuel and not against the injection pressure. Thus, the pressure of the sealing oil needs only to be marginally higher than the feed pressure of the low flashpoint fuel and does not need to be as high or higher than the injection pressure of the fuel.

[0082] Each of the fuel valves 50 is connected to a source of pressured ignition liquid Pi (also referred to as pilot oil). The ignition liquid is a liquid with suitable properties for initiating the ignition of the main fuel. Fuel oil, such as e.g. marine diesel, is an example of a suitable ignition liquid. However other liquids with good ignition qualities, such as bio diesel, lubricating oil, heavy fuel oil or dimethyl ether (DME) can also be used.

[0083] The valve housing 52 is provided with an ignition liquid inlet port 92 for connection to the source of pres-

surized ignition liquid Pi. The valve housing 52 is provided with an ignition liquid bore 94 extending from an ignition liquid inlet port 92 to the clearance 91 between the pump piston 80 and the first bore 81 for sealing the pump piston 80 in the first bore 81 in order to prevent the high-pressure fuel in the pump chamber 82 from leaking into the space under the actuation piston 83. Since the pump piston 80 is slidably received in the first bore 81 with a preferably calibrated clearance 91 between the pump piston 80 and the first bore 81 and since the clearance 91 opens to the pump chamber 82, a small quantity of ignition liquid will reach the pump chamber 82 for each injection event via the clearance 91 between the pump piston 80 and the first bore 81. The amount of ignition liquid that will reach the pump chamber for each injection event will depend on the pressure of the source of ignition liquid Pi, and on the dimensions of the clearance 91. The ignition liquid that reaches the pump chamber will mix with the fuel and will be delivered to the nozzle 54 and injected with the fuel into the combustion chamber. The small quantity of ignition liquid that is mixed with the main fuel assists in reliably igniting the main fuel in the combustion chamber.

[0084] The quantity (for each injection event) of ignition liquid that is required for reliably igniting the main fuel in the combustion chamber will depend on the number of circumstances, such as e.g. the type of main fuel, the type of ignition liquid, the constructional details of the combustion chamber, the constructional details of the nozzle 54, the timing of the fuel injection, the compression pressure, the temperature of the scavenging air and the ratio of recirculated exhaust gas. However, this is not an exhaustive list of aspects that influence the ignition of the main fuel. Through simple trial and error, the skilled person is able to determine the required amount.

[0085] In an embodiment, the pressure of the source of ignition liquid is adjusted in order to adjust the quantity of ignition liquid that is delivered to the pump chamber 82, i.e. the pressure of the source of ignition liquid Pi is increased when the quantity of ignition liquid needs to be increased and the pressure of the source of the ignition liquid is reduced when the quantity of ignition liquid needs to be reduced.

[0086] The pressure of the source of the ignition liquid Pi is always higher than the pressure at which the main fuel is delivered to the fuel inlet port 76. The pressure at which the ignition liquid Pi is delivered can be higher than the maximum pressure in the pump chamber 82, but this is generally not necessary since it is acceptable that some main fuel enters the clearance 91 between the pump piston 80 and the first bore 81 during the pump stroke.

[0087] Fig. 10 shows an alternative type of nozzle 54 for use with the fuel valve 50 described above with reference to Figs. 5 to 9. In this embodiment the nozzle 54 is of a so-called slider type in which the valve seat 63 is arranged at a distance from the tip 56 of the nozzle 54 and the tip 56 of the nozzle is in this embodiment closed, i.e. the tip 56 does not have any nozzle holes. Instead,

the nozzle holes are arranged along the length of the nozzle starting at a position close to the tip 56 going upwards (upward as in the orientation of Fig. 10). The valve needle 61 is provided with a conical section that cooperates with the valve seat 63 and a slider section that extends from the conical section of the valve needle 61 towards the tip of the valve needle 61. This type of nozzle with a closed tip and a slider of the valve needle 61 extending inside the nozzle 54 towards the tip 56 is well known in the art and will therefore not be described in any greater detail.

[0088] In an embodiment (not shown) the pressure of the source of pressurized ignition liquid P_i is electronically controllable, and the electronic control unit 25 is configured to control the pressure of said source of pressurized ignition liquid P_i . Thus, the amount of ignition liquid delivered through the clearance 91 to the pump chamber 82 can be controlled by the electronic control unit by adjusting the pressure of the source of pressurized ignition liquid P_i .

[0089] In an embodiment the engine is provided with a non-low flashpoint fuel supply system (not shown), and the cylinders are provided with two or more fuel valves 51 for injecting said non-low flashpoint fuel into said cylinders.

[0090] The concept of the invention is not restricted to high compressibility fuels and neither to low flash point fuels.

[0091] The invention has been described in conjunction with various embodiments herein. However, other variations to the disclosed embodiments can be understood and effected by those skilled in the art in practicing the claimed invention, from a study of the drawings, the disclosure, and the appended claims. In the claims, the word "comprising" does not exclude other elements or steps, and the indefinite article "a" or "an" does not exclude a plurality. The mere fact that certain measures are recited in mutually different dependent claims does not indicate that a combination of these measures cannot be used to advantage. The reference signs used in the claims shall not be construed as limiting the scope.

Claims

1. A large two-stroke turbocharged compression-ignited internal combustion crosshead engine comprising:

a plurality of cylinders (1),
 a source of pressurized ignition liquid (P_i),
 a source of pressurized sealing oil (P_s), and
 two or more fuel valves (50) arranged in each cylinder (1), said fuel valve (50) being configured for injecting low flashpoint liquid fuel into the combustion chamber of said engine, said fuel valve (50) comprising:

an elongated valve housing (52) with a rear end and a front end,
 a nozzle (54) with a plurality of nozzle holes, said nozzle (54) being disposed at the front end of said elongated valve housing (52),
 a fuel inlet port (76) in said elongated valve housing (52) for connection to a source of pressurized low flashpoint fuel,
 an actuation fluid port (78) in said elongated valve housing (52) for connection to a source of actuation fluid,
 an axially displaceable valve needle (61) slidably received in a longitudinal bore in said fuel valve (50), said valve needle (61) having a closed position and an open position and said valve needle (61) being biased towards said closed position,

said valve needle (61) allowing flow of fuel from said fuel cavity (58) to said plurality of nozzle holes when the valve needle (61) is in its open position and said valve needle preventing flow of fuel from said fuel cavity (58) to said plurality of nozzle holes when the valve needle (61) is in its closed position,

a pump piston (80) received in a first bore (81) in said valve housing (52) with a pump chamber (82) in said first bore (81) on one side of said pump piston (80),

said pump piston (80) is slidably received in said first bore (81) with a clearance (91) between said pump piston (80) and said first bore (81),

an actuation piston (83) received in a second bore (84) in said valve housing (52) with an actuation chamber (85) in said second bore (84) on one side of said actuation piston (83), said pump piston (80) being operably connected to said actuation piston (83),

said actuation chamber (85) being fluidically connected to said actuation fluid port (78),
 said pump chamber (82) having an outlet (66) fluidically connected to said fuel cavity (58) and an inlet (71) fluidically connected to said fuel inlet port (76),

a sealing liquid inlet port (70) connected to said source of pressurized sealing liquid (P_s),

a sealing oil flow path (93,96,97,99) connecting said sealing liquid inlet port (70) to said longitudinal bore for sealing said valve needle (61) in said longitudinal bore,

characterized by

an ignition liquid inlet port (92) connected to said source of pressurized ignition liquid (P_i),

an ignition liquid conduit (94) extending from said ignition liquid inlet port (92) to said clearance (91) for sealing said pump piston (80) in said first bore (81) and for delivering ignition liquid to said pump chamber (82).

2. The engine according to claim 1, wherein said pump piston (80) is sealed in said first bore by said ignition liquid delivered via said ignition liquid conduit (94) to said clearance (91). 5
3. The engine according to claim 2, wherein said clearance opens to said pump chamber (82) and wherein said ignition liquid is delivered to said pump chamber (82) via said clearance (91). 10
4. The engine according to any one of claims 1 to 3, wherein said flow path comprises a conduit in said valve housing (52) connected to said sealing liquid inlet port (70) and/or a spring chamber of a spring (96) that biases said valve needle (61) to said valve seat (63) and/or an axial bore (97) in said valve needle (61) and/or a transverse bore (99) in said valve needle (61). 15
5. The engine according to any one of claims 1 to 4, wherein said sealing oil flow path (93,96,97,99) connects said sealing liquid inlet port (70) to said longitudinal bore at a first position along the length of said longitudinal bore for sealing said valve needle (61) in said longitudinal bore. 20
6. The engine according to claim 5, wherein a conduit (98) connects the low flashpoint fuel inlet port (76) to said longitudinal bore at a second position along the length of said longitudinal bore, said second position being closer to said fuel cavity (58) than said first position. 25
7. The engine according to any one of claims 1 to 6, wherein said valve needle (61) rests on a valve seat (63) in said closed position and said valve needle (61) has lift from said valve seat (63) in said open position. 30
8. The engine according to any one of claims 1 to 7, wherein said nozzle (54) has a nozzle body (55) that is secured to the front of said elongated valve housing (52). 35
9. The engine according to any one of claims 1 to 8, wherein said valve seat is located in the tip (56) of said nozzle (54). 40
10. The engine (50) according to any one of claims 1 to 9, wherein said longitudinal bore is formed at least partially in said nozzle body (55). 45
11. The engine (50) according to any one of claims 1 to 10, wherein lift of said valve needle (61) is controlled by the fuel pressure in said fuel cavity (58). 50
12. The engine according to any one of claims 1 to 10, wherein said valve needle (61) is operably connect-

ed to a needle actuation piston (62) with a pressure surface of said needle actuation piston (62) facing a needle actuation chamber (88) in said fuel valve (50), said needle actuation chamber (88) being fluidically connected to a control port (86) in said fuel valve (50) for connection to a controllable source (Pc) of control fluid.

13. The engine according to any one of claims 1 to 12 comprising a low flashpoint fuel supply system. 10
14. The engine according to claim 13, wherein the pressure of said source of pressurized ignition liquid (Pi) is electronically controllable, and wherein said engine comprises an electronic control unit configured to control the pressure of said source of pressurized ignition liquid (Pi). 15
15. The engine according to claim 13 or 14, wherein said engine is provided with a non-low flashpoint fuel supply system, and wherein said cylinders (1) are provided with two or more fuel valves (51) for injecting said non-low flashpoint fuel into said cylinders (1). 20

Patentansprüche

1. Großer turboaufgeladener kompressionsgezündeter Zweitakt-Kreuzkopf-Verbrennungsmotor, umfassend: 30
- mehrere Zylinder (1),
eine Quelle einer mit Druck beaufschlagten Zündungsflüssigkeit (Pi),
eine Quelle eines mit Druck beaufschlagten Dichtungsöls (Ps) und
zwei oder mehr Kraftstoffventile (50), die in jedem Zylinder (1) angeordnet sind,
wobei das Kraftstoffventil (50) zum Einspritzen eines flüssigen Kraftstoffs mit niedrigem Flammpunkt in die Brennkammer des Motors eingerichtet ist, wobei das Kraftstoffventil (50) umfasst: 35
- ein längliches Ventilgehäuse (52) mit einem hinteren Ende und einem vorderen Ende,
eine Düse (54) mit mehreren Düsenlöchern, wobei die Düse (54) an dem vorderen Ende des länglichen Ventilgehäuses (52) angeordnet ist,
eine Kraftstoff-Einlassöffnung (76) in dem länglichen Ventilgehäuse (52) zur Verbindung mit einer Quelle eines mit Druck beaufschlagten Kraftstoffs mit niedrigem Flammpunkt,
eine Betätigungsfluidöffnung (78) in dem länglichen Ventilgehäuse (52) zur Verbindung mit einer Quelle eines Betätigungsflu-

ids,
 eine axial verschiebbare Ventilmadel (61),
 die in einer Längsbohrung in dem Kraftstoff-
 ventil (50) gleitend aufgenommen ist, wobei
 die Ventilmadel (61) eine geschlossene Po-
 sition und eine offene Position aufweist und
 die Ventilmadel (61) in Richtung der ge-
 schlossenen Position vorgespannt ist,

wobei die Ventilmadel (61) einen Kraftstofffluss
 von dem Kraftstoffhohlraum (58) zu den mehre-
 ren Düsenlöchern ermöglicht, wenn sich die
 Ventilmadel (61) in ihrer offenen Position befin-
 det, und die Ventilmadel einen Kraftstofffluss von
 dem Kraftstoffhohlraum (58) zu den mehreren
 Düsenlöchern verhindert, wenn sich die Ventil-
 madel (61) in ihrer geschlossenen Position be-
 findet,

einen Pumpenkolben (80), der in einer ersten
 Bohrung (81) in dem Ventilgehäuse (52) mit ei-
 ner Pumpenkammer (82) in der ersten Bohrung
 (81) auf einer Seite des Pumpenkolbens (80)
 aufgenommen ist,

wobei der Pumpenkolben (80) in der ersten Boh-
 rung (81) mit einem Zwischenraum (91) zwi-
 schen dem Pumpenkolben (80) und der ersten
 Bohrung (81) gleitend aufgenommen ist,

einen Betätigungskolben (83), der in einer zwei-
 ten Bohrung (84) in dem Ventilgehäuse (52) mit
 einer Betätigungskammer (85) in der zweiten
 Bohrung (84) auf einer Seite des Betätigungs-
 kolbens (83) aufgenommen ist, wobei der Pum-
 penkolben (80) mit dem Betätigungskolben (83)
 wirkverbunden ist,

wobei die Betätigungskammer (85) mit der Be-
 tätigungsfluidöffnung (78) fluidverbunden ist,
 wobei die Pumpenkammer (82) einen Auslass
 (66), der mit dem Kraftstoffhohlraum (58) fluid-
 verbunden ist, und einen Einlass (71), der mit
 der Kraftstoff-Einlassöffnung (76) fluidverbun-
 den ist, aufweist,

eine Dichtungsflüssigkeit-Einlassöffnung (70),
 die mit der Quelle der mit Druck beaufschlagten
 Dichtungsflüssigkeit (Ps) verbunden ist,
 einen Dichtungsöl-Strömungsweg (93, 96, 97,
 99), der die Dichtungsflüssigkeit-Einlassöffnung
 (70) mit der Längsbohrung verbindet, um die
 Ventilmadel (61) in der Längsbohrung abzudich-
 ten,

gekennzeichnet durch

eine Zündungsflüssigkeit-Einlassöffnung (92),
 die mit der Quelle der mit Druck beaufschlagten
 Zündungsflüssigkeit (Pi) verbunden ist,
 eine Zündflüssigkeitsleitung (94), die sich von
 der Zündflüssigkeit-Einlassöffnung (92) zu dem
 Zwischenraum (91) erstreckt, zum Abdichten
 des Pumpenkolbens (80) in der ersten Bohrung
 (81) und zum Leiten von Zündungsflüssigkeit in

die Pumpenkammer (82).

2. Motor nach Anspruch 1, wobei der Pumpenkolben (80) in der ersten Bohrung durch die Zündungsflüssigkeit abgedichtet ist, die über die Zündungsflüssigkeitsleitung (94) in den Zwischenraum (91) geleitet wird.
3. Motor nach Anspruch 2, wobei der Zwischenraum zu der Pumpenkammer (82) öffnet und wobei die Zündungsflüssigkeit über den Zwischenraum (91) in die Pumpenkammer (82) geleitet wird.
4. Motor nach einem der Ansprüche 1 bis 3, wobei der Strömungsweg eine Leitung in dem Ventilgehäuse (52), welche mit der Dichtungsflüssigkeit-Einlassöffnung (70) verbunden ist, und/oder eine Federkammer einer Feder (96), welche die Ventilmadel (61) zu dem Ventilsitz (63) vorspannt, und/oder eine axiale Bohrung (97) in der Ventilmadel (61) und/oder eine Querbohrung (99) in der Ventilmadel (61) umfasst.
5. Motor nach einem der Ansprüche 1 bis 4, wobei der Dichtungsöl-Strömungsweg (93, 96, 97, 99) die Dichtungsflüssigkeit-Einlassöffnung (70) mit der Längsbohrung an einer ersten Position entlang der Länge der Längsbohrung verbindet, um die Ventilmadel (61) in der Längsbohrung abzudichten.
6. Motor nach Anspruch 5, wobei eine Leitung (98) die Einlassöffnung (76) des Kraftstoffs mit niedrigem Flammpunkt mit der Längsbohrung an einer zweiten Position entlang der Länge der Längsbohrung verbindet, wobei die zweite Position näher zu dem Kraftstoffhohlraum (58) liegt als die erste Position.
7. Motor nach einem der Ansprüche 1 bis 6, wobei die Ventilmadel (61) in der geschlossenen Position auf einem Ventilsitz (63) ruht und die Ventilmadel (61) in der offenen Position von dem Ventilsitz (63) angehoben ist.
8. Motor nach einem der Ansprüche 1 bis 7, wobei die Düse (54) einen Düsenkörper (55) aufweist, der an dem Vorderteil des länglichen Ventilgehäuses (52) befestigt ist.
9. Motor nach einem der Ansprüche 1 bis 8, wobei sich der Ventilsitz in der Spitze (56) der Düse (54) befindet.
10. Motor (50) nach einem der Ansprüche 1 bis 9, wobei die Längsbohrung zumindest teilweise in dem Düsenkörper (55) gebildet ist.
11. Motor (50) nach einem der Ansprüche 1 bis 10, wobei ein Anheben der Ventilmadel (61) durch den Kraftstoffdruck in dem Kraftstoffhohlraum (58) gesteuert

wird.

12. Motor nach einem der Ansprüche 1 bis 10, wobei die Ventilmadel (61) mit einem Nadelbetätigungskolben (62) wirkverbunden ist, wobei eine Druckfläche des Nadelbetätigungskolbens (62) einer Nadelbetätigungskammer (88) in dem Kraftstoffventil (50) zugewandt ist, wobei die Nadelbetätigungskammer (88) mit einer Steueröffnung (86) in dem Kraftstoffventil (50) zur Verbindung mit einer steuerbaren Quelle (Pc) eines Steuerfluids fluidverbunden ist. 5 10
13. Motor nach einem der Ansprüche 1 bis 12, umfassend ein System zur Zufuhr von Kraftstoff mit niedrigem Flammpunkt. 15
14. Motor nach Anspruch 13, wobei der Druck der Quelle der mit Druck beaufschlagten Zündungsflüssigkeit (Pi) elektronisch steuerbar ist, und wobei der Motor eine elektronische Steuereinheit aufweist, die eingerichtet ist, um den Druck der Quelle der mit Druck beaufschlagten Zündungsflüssigkeit (Pi) zu steuern. 20
15. Motor nach Anspruch 13 oder 14, wobei der Motor mit einem System zur Zufuhr von Kraftstoff mit nicht niedrigem Flammpunkt vorgesehen ist, und wobei die Zylinder (1) mit zwei oder mehr Kraftstoffventilen (51) zum Einspritzen des Kraftstoffs mit nicht niedrigem Flammpunkt in die Zylinder (1) vorgesehen ist. 25 30

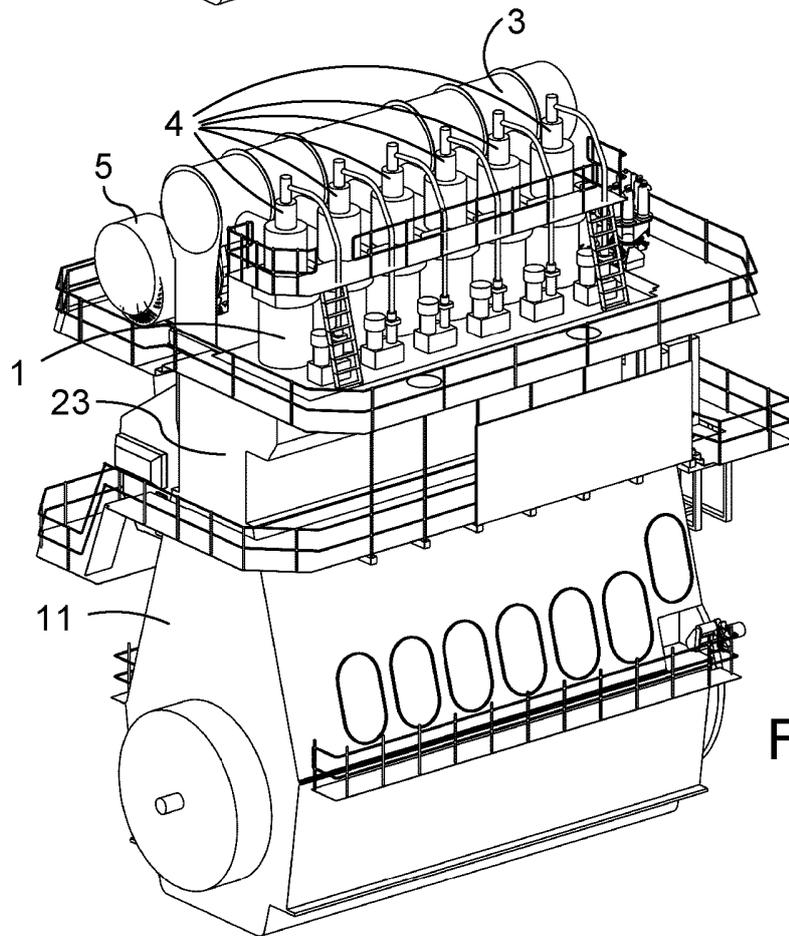
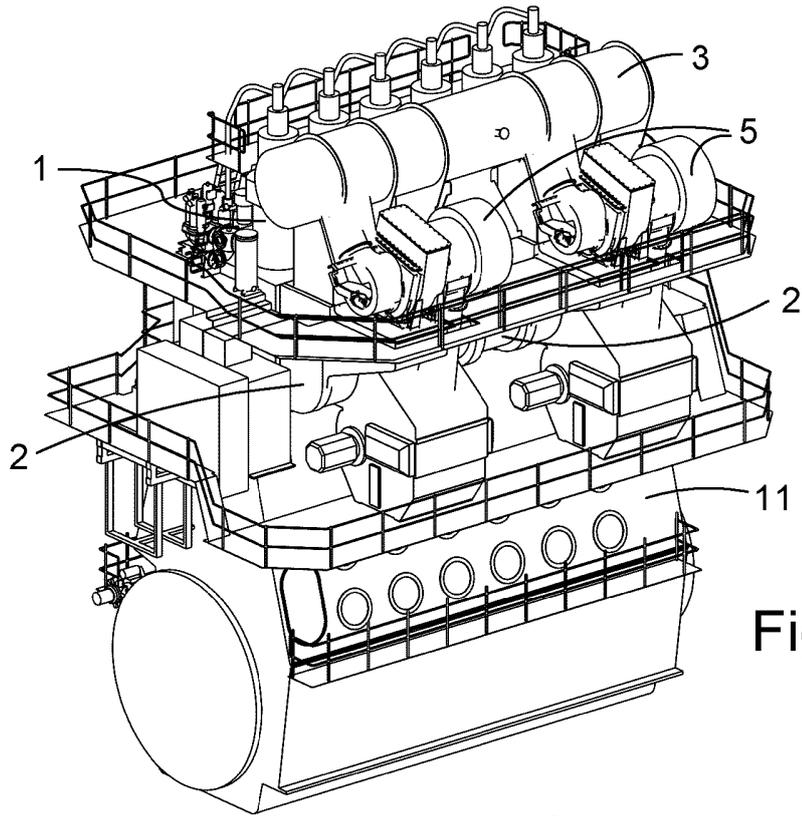
Revendications

1. Gros moteur à crosse à combustion interne à allumage par compression turbocompressé à deux temps comprenant : 35
- une pluralité de cylindres (1),
 une source de liquide d'allumage sous pression (Pi), 40
 une source d'huile d'étanchéité sous pression (Ps), et
 deux ou plusieurs vannes de carburant (50) disposées dans chaque cylindre (1),
 ladite vanne de carburant (50) étant configurée 45
 pour injecter du carburant liquide à faible point d'éclair dans la chambre de combustion dudit moteur, ladite vanne de carburant (50) comprenant : 50
- un boîtier de soupape allongé (52) avec une extrémité arrière et une extrémité avant,
 une buse (54) avec une pluralité de trous de buse, ladite buse (54) étant disposée à 55
 l'extrémité avant dudit boîtier de soupape allongé (52),
 un orifice d'admission de carburant (76) dans ledit boîtier de soupape allongé (52)

pour le raccordement à une source de carburant sous pression à faible point d'éclair, un orifice de fluide d'actionnement (78) dans ledit boîtier de soupape allongé (52) pour le raccordement à une source de fluide d'actionnement,
 un pointeau de soupape déplaçable axialement (61) reçu de manière coulissante dans un alésage longitudinal dudit robinet de carburant (50), ledit pointeau de soupape (61) ayant une position fermée et une position ouverte et ledit pointeau de soupape (61) étant rappelé vers ladite position fermée,

ledit pointeau de soupape (61) permettant l'écoulement de carburant depuis ladite cavité de carburant (58) vers ladite pluralité de trous de buse lorsque le pointeau de soupape (61) est dans sa position ouverte et ledit pointeau de soupape empêchant l'écoulement de carburant de ladite cavité de carburant (58) vers ladite pluralité de trous de buse lorsque le pointeau de soupape (61) est dans sa position fermée,
 un piston de pompe (80) reçu dans un premier alésage (81) dans ledit boîtier de soupape (52) avec une chambre de pompe (82) dans ledit premier alésage (81) sur un côté dudit piston de pompe (80),
 ledit piston de pompe (80) est reçu de manière coulissante dans ledit premier alésage (81) avec un jeu (91) entre ledit piston de pompe (80) et ledit premier alésage (81),
 un piston d'actionnement (83) reçu dans un deuxième alésage (84) dans ledit boîtier de soupape (52) avec une chambre d'actionnement (85) dans ledit deuxième alésage (84) sur un côté dudit piston d'actionnement (83), ledit piston de pompe (80) étant raccordé fonctionnellement audit piston d'actionnement (83),
 ladite chambre d'actionnement (85) étant raccordée fluidiquement audit orifice de fluide d'actionnement (78),
 ladite chambre de pompe (82) ayant une sortie (66) raccordée fluidiquement à ladite cavité de carburant (58) et une entrée (71) raccordée fluidiquement audit orifice d'admission de combustible (76),
 un orifice d'entrée de liquide d'étanchéité (70) raccordé à ladite source de liquide d'étanchéité sous pression (Ps),
 un trajet d'écoulement d'huile d'étanchéité (93, 96, 97, 99) raccordant ledit orifice d'entrée de liquide d'étanchéité (70) audit alésage longitudinal pour étanchéfier ledit pointeau de soupape (61) dans ledit alésage longitudinal,
caractérisé par
 un orifice d'entrée de liquide d'allumage (92) raccordé à ladite source de liquide d'allumage

- sous pression (Pi),
un conduit de liquide d'allumage (94) s'étendant dudit orifice d'admission de liquide d'allumage (92) audit espace (91) pour sceller ledit piston de pompe (80) dans ledit premier alésage (81) et pour fournir du liquide d'allumage à ladite chambre de pompe (82) .
2. Moteur selon la revendication 1, dans lequel ledit piston de pompe (80) est scellé dans ledit premier alésage par ledit liquide d'allumage délivré via ledit conduit de liquide d'allumage (94) audit espace (91).
 3. Moteur selon la revendication 2, dans lequel ledit jeu s'ouvre sur ladite chambre de pompe (82) et dans lequel ledit liquide d'allumage est délivré à ladite chambre de pompe (82) via ledit jeu (91).
 4. Moteur selon une quelconque des revendications 1 à 3, dans lequel ledit chemin d'écoulement comprend un conduit dans ledit boîtier de soupape (52) raccordé audit orifice d'admission de liquide d'étanchéité (70) et/ou une chambre de ressort d'un ressort (96) qui sollicite ledit pointeau de soupape (61) vers ledit siège de soupape (63) et/ou et un alésage axial (97) dans ledit pointeau de soupape (61) et/ou un alésage transversal (99) dans ledit pointeau de soupape (61).
 5. Moteur selon une quelconque des revendications 1 à 4, dans lequel ledit chemin d'écoulement d'huile d'étanchéité (93, 96, 97, 99) raccorde ledit orifice d'admission de liquide d'étanchéité (70) audit alésage longitudinal à une première position le long de la longueur de ledit alésage longitudinal pour sceller ledit pointeau de soupape (61) dans ledit alésage longitudinal.
 6. Moteur selon la revendication 5, dans lequel un conduit (98) relie l'orifice d'admission de carburant à faible point d'éclair (76) audit alésage longitudinal en une seconde position sur la longueur dudit alésage longitudinal, ladite seconde position étant plus proche de ladite cavité de carburant (58) que ladite première position.
 7. Moteur selon une quelconque des revendications 1 à 6, dans lequel ledit pointeau de soupape (61) repose sur un siège de soupape (63) dans ladite position fermée et ledit pointeau de soupape (61) se soulève dudit siège de soupape (63) dans ladite position ouverte.
 8. Moteur selon une quelconque des revendications 1 à 7, dans lequel ladite buse (54) a un corps de buse (55) qui est fixé à l'avant dudit boîtier de soupape allongé (52).
 9. Moteur selon une quelconque des revendications 1 à 8, dans lequel ledit siège de soupape est situé dans l'embout (56) de ladite buse (54).
 10. Moteur (50) selon une quelconque des revendications 1 à 9, dans lequel ledit alésage longitudinal est formé au moins partiellement dans ledit corps de buse (55).
 11. Moteur (50) selon une quelconque des revendications 1 à 10, dans lequel la levée dudit pointeau de soupape (61) est commandée par la pression de carburant dans ladite cavité de carburant (58).
 12. Moteur selon une quelconque des revendications 1 à 10, dans lequel ledit pointeau de soupape (61) est fonctionnellement raccordé à un piston d'actionnement de pointeau (62) avec une surface de pression dudit piston d'actionnement de pointeau (62) faisant face à une chambre d'actionnement de pointeau (88) dans ladite vanne de carburant (50), ladite chambre d'actionnement de pointeau (88) étant raccordée fluidiquement à un orifice de commande (86) dans ladite vanne de carburant (50) en vue d'un raccordement à une source contrôlable (Pc) de fluide de commande.
 13. Moteur selon une quelconque des revendications 1 à 12 comprenant un système d'alimentation en carburant à bas point d'éclair.
 14. Moteur selon la revendication 13, dans lequel la pression de ladite source de liquide d'allumage sous pression (Pi) est contrôlable électroniquement, et dans lequel ledit moteur comprend une unité de contrôle électronique configurée pour contrôler la pression de ladite source de liquide d'allumage sous pression (Pi).
 15. Moteur selon la revendication 13 ou 14, dans lequel ledit moteur est pourvu d'un système d'alimentation en carburant à point d'éclair non bas, et dans lequel lesdits cylindres (1) sont pourvus de deux ou plusieurs soupapes de carburant (51) pour injecter ledit point d'éclair de carburant non bas dans lesdits cylindres (1).



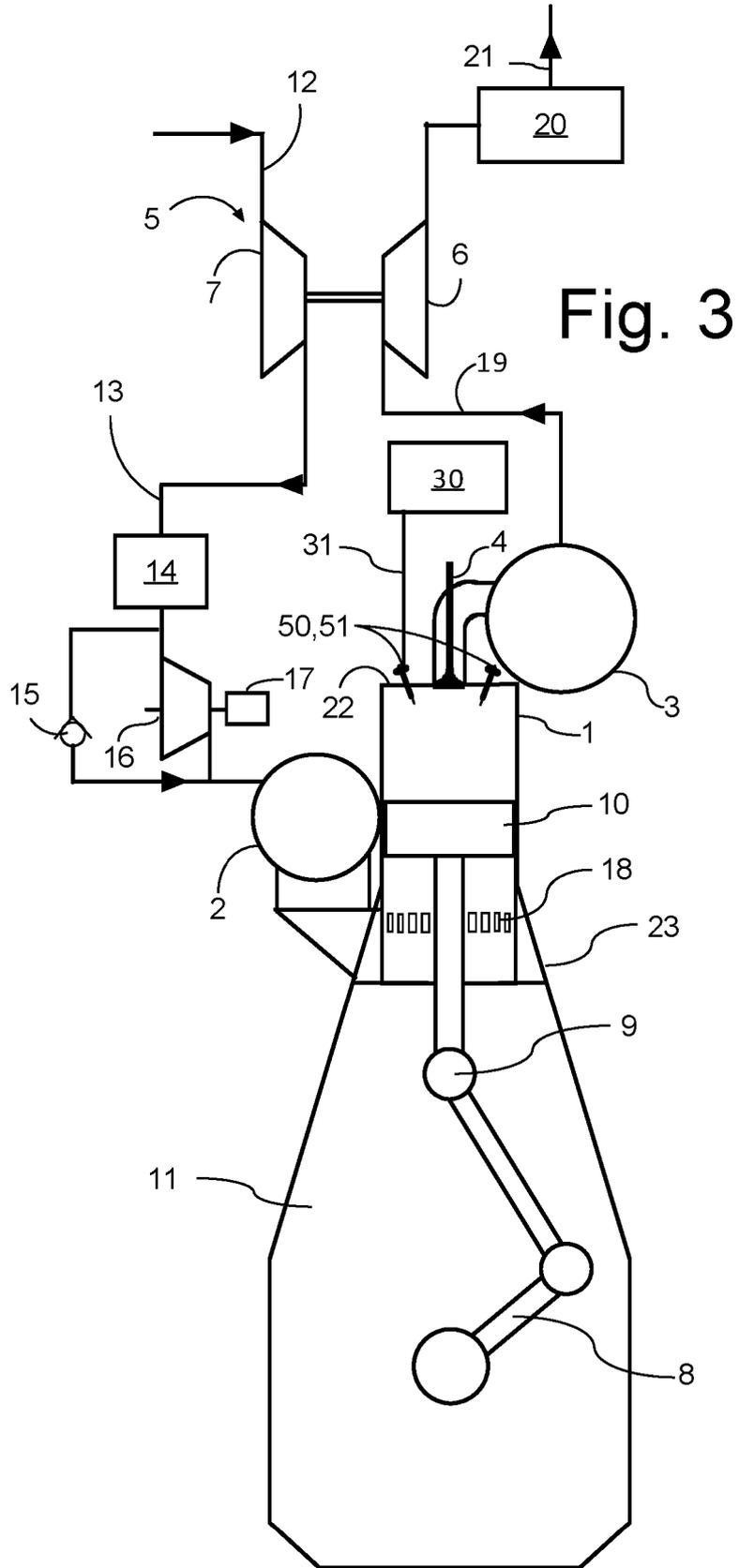
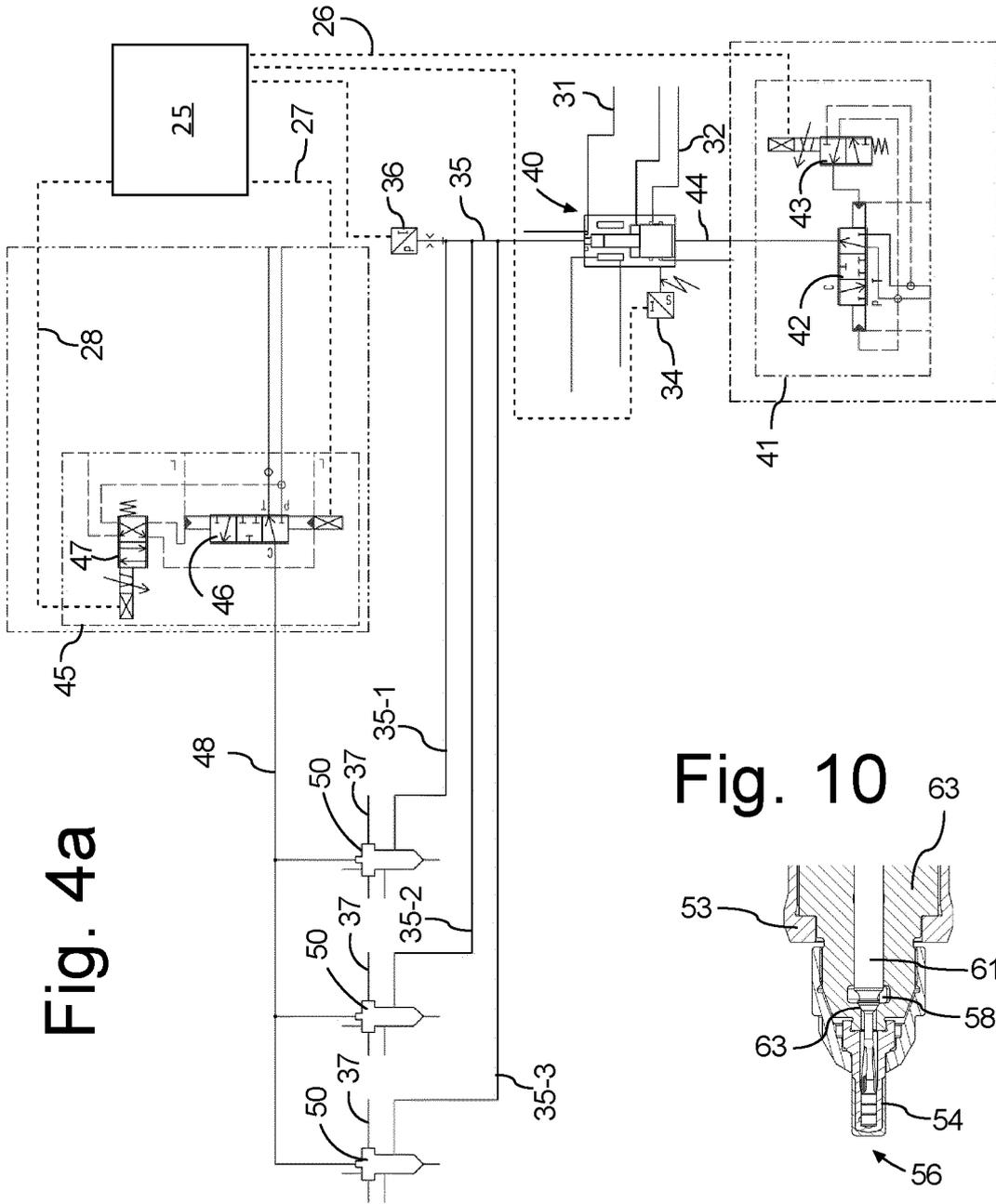


Fig. 3



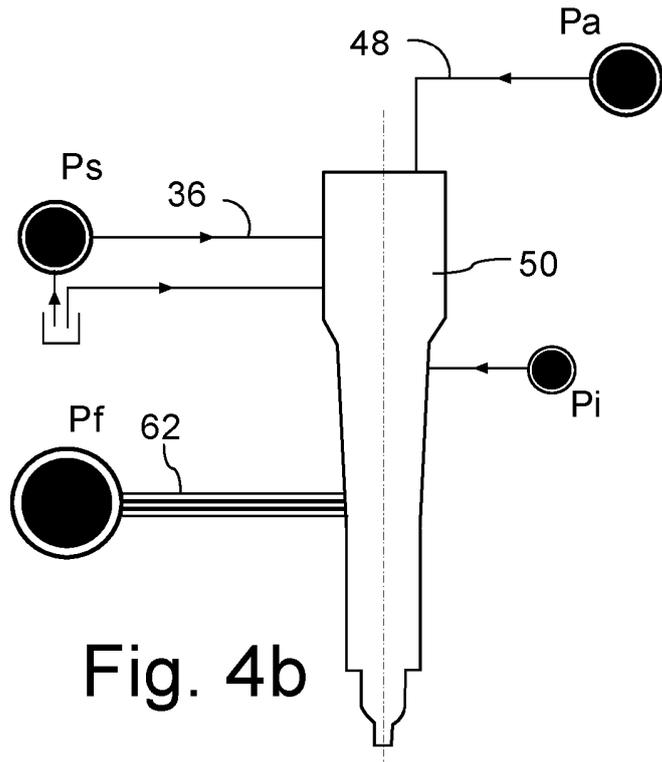


Fig. 4b

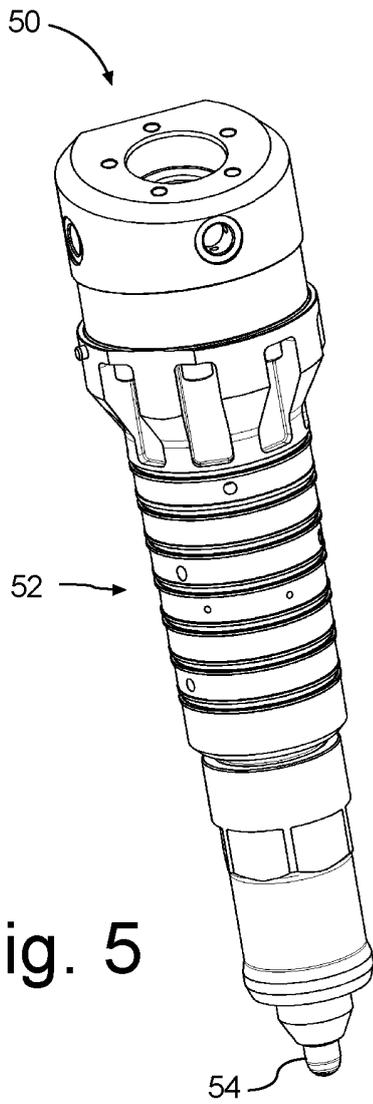


Fig. 5

Fig. 6

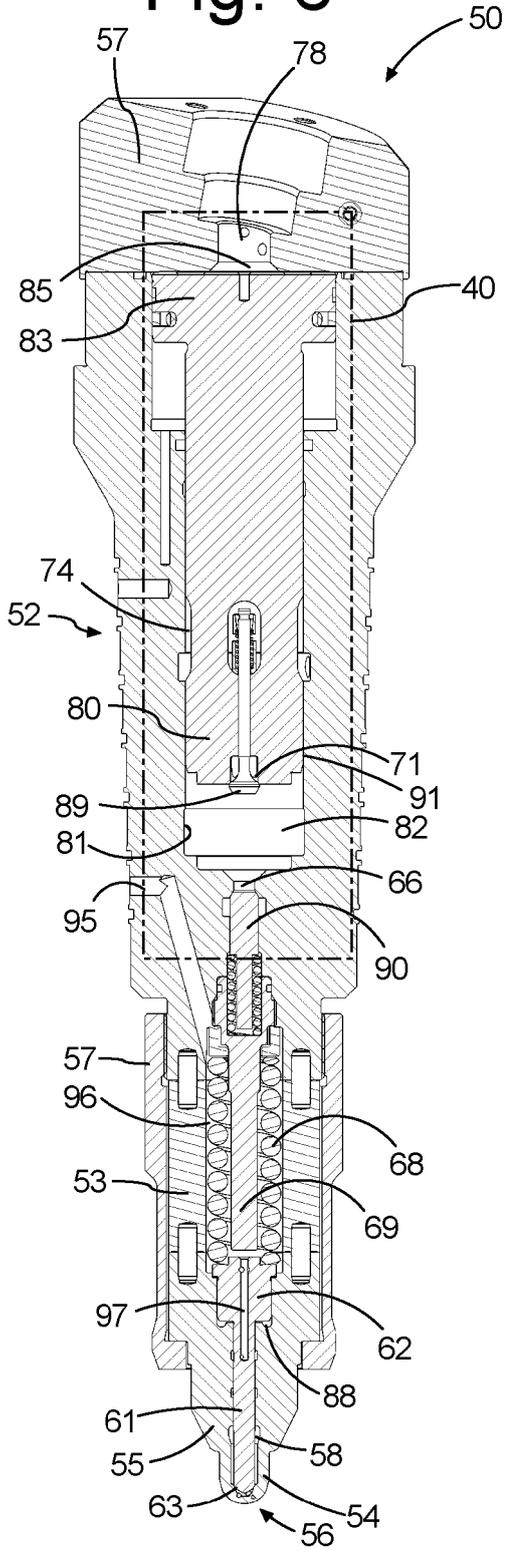


Fig. 7

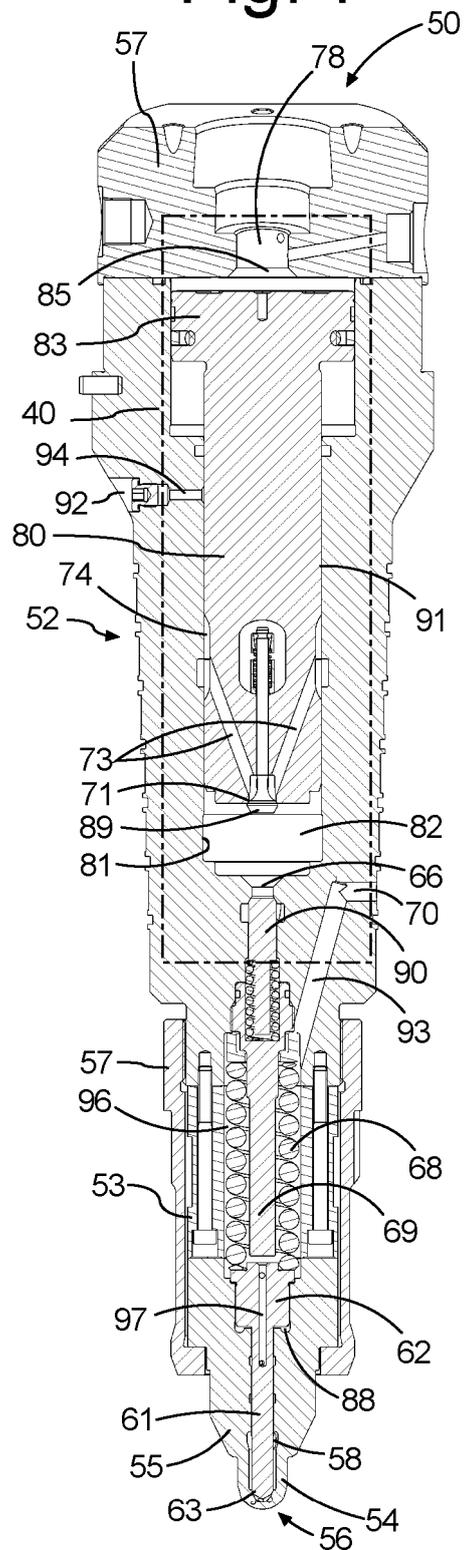


Fig. 8

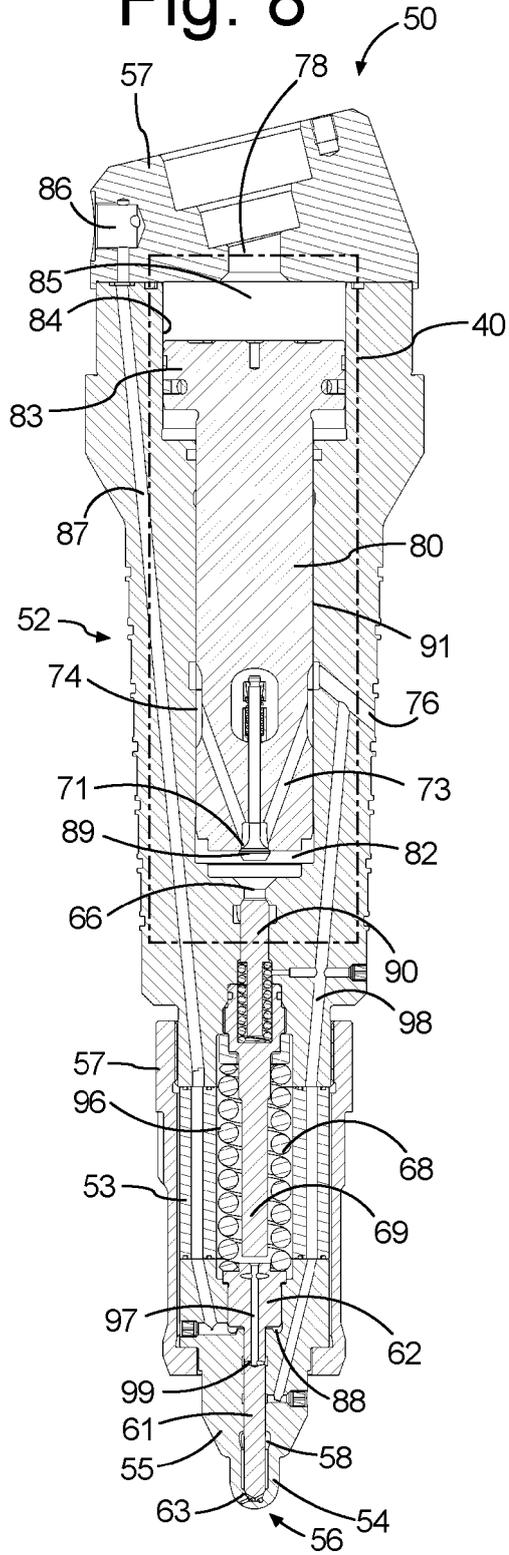
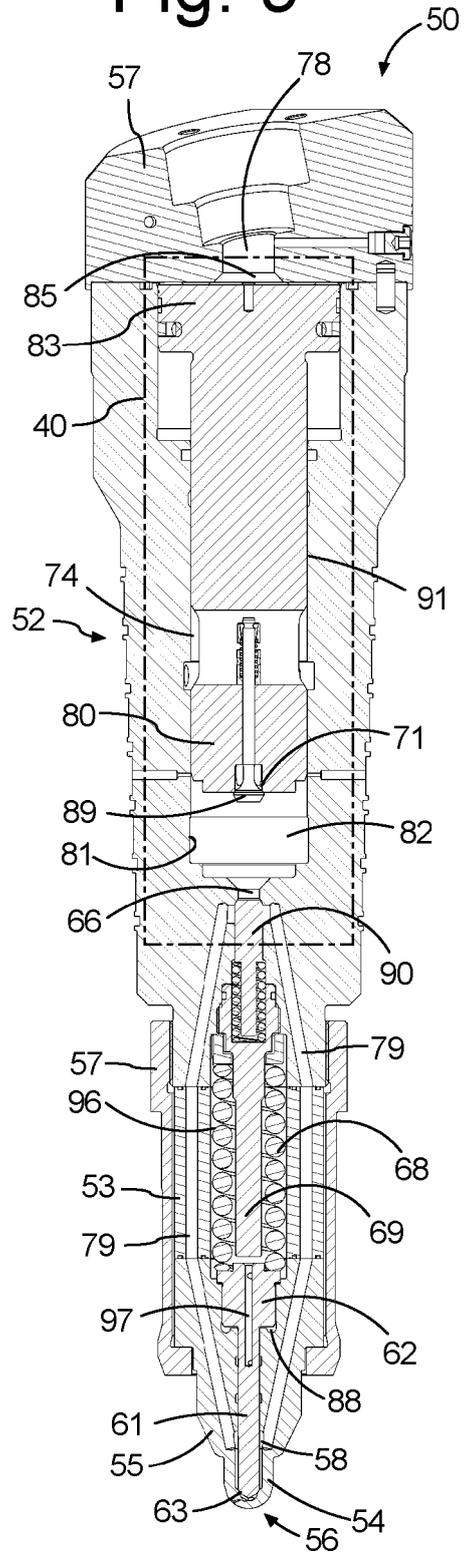


Fig. 9



REFERENCES CITED IN THE DESCRIPTION

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