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## Description

**[0001]** The disclosure relates to a hinge, and more particularly to an adjustable hinge.

**[0002]** A conventional hinge disclosed in Taiwanese Patent No. 1580856 includes a leaf unit that has first and second leaves that are rotatable relative to each other, and two action units that are mounted in the leaf unit.

**[0003]** Each of the action units includes a casing that is co-rotatable with the first leaf, and an operating shaft that is co-rotatable with the second leaf. The casing and the operating shaft of each of the action units are rotated relative to each other upon the relative rotation between the first and second leaves, so as to generate an actuating force that acts between the first and second leaves.

**[0004]** EP3246501A1 and DE102009035682A1 both disclose a hinge with action units.

**[0005]** However, to co-rotatably mount the operating shaft of each of the action units to the second leaf, an inner surrounding surface of the second leaf need to be formed with mounting structures that correspond to the operating shafts of the action units.

**[0006]** Such mounting structures may not be machined easily.

**[0007]** Therefore, an object of the disclosure is to provide a hinge that can alleviate the drawback of the prior art.

**[0008]** The invention proposes a hinge
according to claim 1. Embodiments of the hinge are defined in the dependent claims.

**[0009]** In the whole application, the terms action unit and force-providing unit are equivalent.

**[0010]** Other features and advantages of the disclosure will become apparent in the following detailed description of the embodiments with reference to the accompanying drawings, of which:

Figure 1 is a top view illustrating a first embodiment of the hinge which does not form part of the claimed invention;
Figure 2 is a partly exploded perspective view illustrating the first embodiment;
Figure 3 is an exploded perspective view illustrating a first action unit of the first embodiment;
Figure 4 is a sectional view illustrating the first action unit;
Figure 5 is an exploded perspective view illustrating a second action unit of the first embodiment;
Figure 6 is a sectional view illustrating the second action unit;
Figure 7 is an assembled perspective view illustrating the first embodiment;
Figure 8 is a sectional view illustrating the first embodiment;
Figure 9 is a partly exploded perspective view illustrating a second embodiment of the hinge which does not form part of the claimed invention;
Figure 10 is an exploded perspective view illustrating one of two torsional action units of the second embodiment;
Figure 11 is a sectional view illustrating the one of the torsional action units;
Figure 12 is an exploded perspective view illustrating the other one of the torsional action units of the second embodiment;
Figure 13 is a sectional view illustrating the other one of the torsional action units;
Figure 14 is an assembled perspective view illustrating the second embodiment;
Figure 15 is a sectional view illustrating the second embodiment;
Figure 16 is a sectional view illustrating a third embodiment of the hinge which does not form part of the claimed invention;
Figure 17 is a sectional view illustrating a fourth embodiment of the hinge which does not form part of the claimed invention;
Figure 18 is a sectional view illustrating a modification of the first action unit;
Figure 19 is a partly exploded perspective view illustrating a fifth embodiment of the hinge which does not form part of the claimed invention;
Figure 20 is a sectional view illustrating the fifth embodiment;
Figure 21 is a partly exploded perspective view illustrating a sixth embodiment of the hinge according to the disclosure;
Figure 22 is a sectional view illustrating the sixth embodiment;
Figure 23 is a partly exploded perspective view illustrating a modification of a ring unit of the hinge according to the disclosure;
Figure 24 is a sectional view illustrating the modification of the ring unit;
Figure 25 is a sectional view illustrating a modification of the torsional action unit of the hinge according to an embodiment of the invention; and
Figures 26 and 27 are sectional views illustrating operation of the modification of the torsional action unit.

**[0011]** Before the disclosure is described in greater detail, it should be noted that where considered appropriate, reference numerals or terminal portions of reference numerals have been repeated among the figures to indicate corresponding or analogous elements, which may optionally have similar characteristics.

**[0012]** Referring to Figures 1 and 2, the first embodiment of the hinge according to the disclosure and not forming part of the claimed invention, is for interconnecting first and second objects 11, 12 (e.g., a door frame and a door leaf) , and includes a leaf unit 2, a first action unit 6, a second action unit 7, an axle unit 4 and a ring unit 5.

**[0013]** The leaf unit 2 includes first and second leaves 21, 22 that are rotatable relative to each other. Each of the first leaf 21 and the second leaf 22 is made of metal.

**[0014]** The first leaf 21 has two first barrels 211 that are spaced apart from each other along an axis (X), a first clinging surface 212 that clings to the first object 11, and a first positioning surface 213 that is parallel to the axis (X), that is connected to the first clinging surface 212 and that is not coplanar with the first clinging surface 212. The first positioning surface 213 permits an edge 111 of the first object 11 to abut thereagainst. Each of the first barrels 211 has two inner limiting planes 2111 that are formed on an inner surrounding surface thereof.

**[0015]** The second leaf 22 has a second barrel 221 that is disposed between the first barrels 211 and that is spaced apart from the first barrels 211 along the axis (X), a second clinging surface 222 that clings to the second object 12, and a second positioning surface 223 that is parallel to the axis (X), that is connected to the second clinging surface 222 and that is not coplanar with the second clinging surface 222. The second positioning surface 223 permits an edge 121 of the second object 12 to abut thereagainst.

**[0016]** Referring further to Figures 3 and 4, the first action unit 6 includes a first tubular member 61 that is inserted into the first and second barrels 211, 221 and that is co-rotatable with the first leaf 21, a hydraulic module 62 that is disposed in the first tubular member 61, a distal acting member 63 that is co-rotatably mounted in the first tubular member 61, a proximal acting member 64 that is co-rotatably mounted in the first tubular member 61, and a cap member 65 that is mounted to an end of the first tubular member 61.

**[0017]** The first tubular member 61 has a first tube section 611, and a second tube section 612 that abuts against the first tube section 611. The first tube section 611 has two outer limiting planes 6111 that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211, and two mounting grooves 6112 each of which extends from an end of the first tube section 611 in the direction of the axis (X) . The second tube section 612 has two outer limiting planes 6121 that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211, two spaced-apart positioning recesses 6122 that are formed in an inner surrounding surface thereof, and two mounting grooves 6123 each of which extends from an end of the second tube section 612 in the direction of the axis (X). Each of the mounting grooves 6112 of the first tube section 611 cooperates with a respective one of the mounting grooves 6123 of the second tube section 612 to form an mounting space 610 (see Figure 4) . By such, the first tubular member 61 is co-rotatable with the first leaf 21 by the cooperation among the outer limiting planes 6111, 6121 and the inner limiting planes 2111. It should be noted that the two-piece first tubular member 61 is easy to be assembled with other components, and the first and second tube sections 611, 612 can be made of different materials. A junction between the first and second tube sections 611, 612 of the first tubular member 61 should be located within one of the first barrels 211.

**[0018]** The hydraulic module 62 includes a hydraulic cylinder 621, an abutment pin 622 that abuts against the hydraulic cylinder 621, and a resilient member 623 that abuts against the hydraulic cylinder 621. The hydraulic cylinder 621 threadably engages the first tube section 611 of the first tubular member 61, and has a hexagonal setting hole 6211 that extends along the axis (X) and that is accessible through the cap member 65, a hexagonal throttle hole 6212, and a telescopic protrusion 6213 that is opposite to the setting hole 6211 and that abuts against the abutment pin 622.

**[0019]** The distal acting member 63 is mounted to the first and second tube sections 611, 612 of the first tubular member 61, and has a distal inclined surface 631, and two mounting blocks 632 each of which engages a respective one of the mounting grooves 6112 of the first tube section 611 and a corresponding one of the mounting grooves 6123 of the second tube section 612 (i.e., resides within a respective one of the mounting spaces 610), so that the distal acting member 63 is co-rotatable with the first tubular member 61.

**[0020]** The proximal acting member 64 has a proximal inclined surface 641, a through hole 642, and two spaced-apart positioning protrusions 643 that are formed on an outer surrounding surface thereof. The positioning protrusions 643 of the proximal acting member 64 respectively engage the positioning recesses 6122 of the second tube section 612, so that the proximal acting member 64 is co-rotatable with the first tubular member 61.

**[0021]** Referring further to Figures 5 and 6, the second action unit 7 includes a second tubular member 71 that is inserted into the first and second barrels 211, 221 and that is co-rotatable with the first leaf 21, a disc spring assembly 72 that is disposed in the second tubular member 71, a friction member 73 that abuts against the disc spring assembly 72 and that is co-rotatable with the second tubular member 71, an adjusting member 74 that engages threadably the second tubular member 71 and that pushes the disc spring assembly 72, and a plurality of washers 75 disposed in the second tubular member 71. In this example, the second action unit 7 includes two tab washers 75.

**[0022]** The second tubular member 71 has a first tube section 711, and a second tube section 712 that abuts against the first tube section 711. The first tube section 711 has two outer limiting planes 7111 (only one is visible in Figure 5) that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211, and two mounting blocks 7112 each of which extends from an end of the first tube section 711 in the direction of the axis (X). The second tube section 712 has two outer limiting planes 7121 that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211, two spaced-apart positioning recesses 7122 (only one is

shown in Figure 5) that are formed at an end of the second tube section 712, and two spaced-apart mounting grooves 7123 each of which extends from an opposite end of the second tube section 712 in the direction of the axis (X) and is engaged with a respective one of the mounting blocks 7112 of the first tube section 711.

**[0023]** The disc spring assembly 72 includes a plurality of disc springs 721 that are disposed between the friction member 73 and one of the washers 75, and a padding member 722 that is disposed between the washers 75. The friction member 73 has two spaced-apart positioning protrusions 731 that are formed on an outer surrounding surface thereof and that respectively engage the positioning recesses 7122 of the second tube section 712 so that the friction member 73 is co-rotatable with the second tubular member 71. The friction member 73 further has a friction surface 732 that is formed at an end thereof distal from the disc spring assembly 72.

**[0024]** By such, the second tubular member 71 is co-rotatable with the first leaf 21 by the cooperation among the outer limiting planes 7111, 7121 and the inner limiting planes 2111. It should be noted that the two-piece second tubular member 71 is easy to be assembled with other components, and the first and second tube sections 711, 712 can be made of different materials. A junction between the first and second tube sections 711, 712 of the second tubular member 71 should be located within one of the first barrels 211.

**[0025]** Referring back to Figures 2 and 3, the axle unit 4 includes a fixing member 41 that is removably mounted in the second barrel 221 of the second leaf 22 by a fastener 23 and that is co-rotatable with the second leaf 22, a first axle 43 (see Figure 3) that is mounted to the first action unit 6 and that is co-rotatably connected to the fixing member 41, and a second axle 44 (see Figure 2) that is mounted to the second action unit 7 and that is co-rotatably connected to the fixing member 41. In one example, the first axle 43 is located between the fixing member 41 and the hydraulic cylinder 621.

**[0026]** The fixing member 41 has a rectangular fixing hole 411 that is formed in one of two opposite end surfaces of the fixing member 41 along the axis (X) and that extends along the axis (X), a fixing recess 412 (see Figure 8) that is formed in the other one of the opposite end surfaces of the fixing member 41, and two fixing grooves 413 (only one is visible in Figure 2) that are respectively formed in the opposite end surfaces of the fixing member 41. In one example, the fixing recess 412 is configured as a circular recess. In one example, the fixing hole 411 is formed through the opposite end surfaces of the fixing member 41. In one example, the fixing member 41 has a circular outer surrounding surface that abuts against an inner surrounding surface of the second barrel 221 of the second leaf 22.

**[0027]** Referring back to Figure 3, the first axle 43 has a follower portion 430 that is disposed between the distal acting member 63 and the proximal acting member 64, and an axle portion 431 that extends through the through hole 642 of the proximal acting member 64 and that co-rotatably engages the fixing hole 411 of the fixing member 41. The follower portion 430 has an abutment surface 432 (see Figure 8) that is opposite to the axle portion 431 and that abuts against the abutment pin 622 and the resilient member 623, a surrounding wall 433 that cooperates with the abutment surface 432 to define a recess, and a proximal follower surface 434 that is opposite to the abutment surface 432 and that faces toward the proximal inclined surface 641 of the proximal acting member 64. The surrounding wall 433 has a distal follower surface 4331 that is opposite to the proximal follower surface 434 and that faces toward the distal inclined surface 631 of the distal acting member 63. In this example, the axle portion 431 has configured as a rectangular cross-section.

**[0028]** Referring to Figure 2, the second axle 44 has a fixing hole 440 that is co-rotatably engaged with the axle portion 431 of the first axle 43, a post 441 that co-rotatably engages a corresponding one of the fixing grooves 413 of the fixing member 41, and two protrusions 442 (only one is visible in Figure 2) that protrude toward the friction surface 732 of the friction member 73 of the second action unit 7. In this example, the fixing hole 440 is configured as a rectangular hole. The post 441 may co-rotatably engages the fixing recess 412 of the fixing member 41 by modifying the shape of the fixing recess 412. The protrusions 442 of the second axle 44 are in frictional contact with the friction surface 732 of the friction member 73 of the second action unit 7, so that the second action unit 7 may generate an actuating force that acts between the first and second leaves 21, 22 when the second axle 44 and the second action unit 7 are rotated relative to each other. The profile of the friction surface 732 of the friction member 73 may be configured such that the first and second leaves 21, 22 are held relative to each other when an angle formed between the first and second leaves 21, 22 reaches a predetermined value or range, or may be configured such that the second action unit 7 retards the relative rotation between the first and second leaves 21, 22 when the angle formed between the first and second leaves 21, 22 reaches a predetermined value or range, and is not limited to such.

**[0029]** Referring to Figures 2 and 8, the ring unit 5 includes two ring members 51 and two spacer assemblies 52. The ring members 51 are respectively disposed between the first tubular member 61 and the second barrel 221 and between the second tubular member 71 and the second barrel 221. Each of the spacer assemblies 52 includes a spacer 521. Each of the spacers 521 of the spacer assemblies 52 has an surrounding wall 522 that is disposed between the second barrel 221 and a respective one of the first tubular member 61 and the second tubular member 71, and a flange wall 523 that is disposed between the second barrel 221 and a respective one of the first barrels 211. Each of the ring members 51 and the spacer assemblies 52 may be made of Polyoxymethylene (POM) or Polytetrafluoroethylene (PTFE), and

serves as a bushing for facilitating relative rotation between the corresponding components.

**[0030]** During installation of the hinge onto the first and second objects 11, 12, the first leaf 21 can be quickly and accurately positioned relative to the first object 11 by moving the first positioning surface 213 to abut against the edge 111 of the first object 11, and the second leaf 22 can be quickly and accurately positioned relative to the second object 12 by moving the second positioning surface 223 to abut against the edge 121 of the second object 12. As such, the first and second objects 11, 12 are accurately positioned relative to each other, and can be smoothly rotated relative to each other.

**[0031]** Referring to Figures 1 to 8, when the first and second leaves 21, 22 are rotated relative to each other in the direction of the arrow shown in Figure 1 by an external force, the first axle 43 is rotated relative to the distal and proximal acting members 63, 64, and the proximal inclined surface 641 of the proximal acting member 64 pushes the proximal follower surface 434 of the first axle 43 to move the first axle 43 toward the hydraulic cylinder 621 and to push the distal follower surface 4331 against the distal inclined surface 631 of the distal acting member 63. As such, the first axle 43 pushes the abutment pin 622 to press the telescopic protrusion 6213 of the hydraulic cylinder 621 for controlling the relative rotational speed between the first and second leaves 21, 22, and the abutment surface 432 of the first axle 43 pushes and compresses the resilient member 63 to generate a restoring force (i.e., an actuating force).

**[0032]** At the same time, the second axle 44 is rotated relative to the friction member 73, and pushes the friction member 73 to compress the disc spring assembly 72 to generate the actuating force.

**[0033]** When the external force is removed, the resilient member 623 pushes the first axle 43 to move away from the hydraulic cylinder 621, and therefore the proximal follower surface 434 of the first axle 43 pushes the proximal inclined surface 641 of the proximal acting member 64 to rotate the first axle 43 and the proximal acting member 64 relative to each other, so as to rotate the first and second leaves 21, 22 relative to each other in a direction opposite to the arrow shown in Figure 1.

**[0034]** It should be noted that, the distal follower surface 4331 of the first axle 43 is in contact with the distal inclined surface 631 of the distal acting member 63 when the first and second leaves 21, 22 are rotated relative to each other in the direction opposite to the arrow shown in Figure 1.

**[0035]** It should also be noted that, the first leaf 21 can be connected to any one of a door leaf and a door frame while the second leaf 22 is connected to the other one of the door leaf and the door frame.

**[0036]** The hexagonal setting hole 6211 of the hydraulic cylinder 621 permits a hand tool to engage therewith. By rotating the hand tool, the hydraulic cylinder 621 is moved relative to the first tubular member 61 along the axis (X), and the relative position between the hydraulic cylinder 621 and the first axle 43 is adjusted, so that the range of the angle formed between the first and second leaves 21, 22 within which the hydraulic cylinder 621 works can be adjusted. The hexagonal throttle hole 6212 of the hydraulic cylinder 621 permits another hand tool to engage therewith. By rotating the hand tool, the damping coefficient of the hydraulic cylinder 621 can be adjusted.

**[0037]** In addition, by moving the adjusting member 74 along the axis (X), the actuating force generated by the disc spring assembly 72 can be adjusted. By substituting the friction member 73 with another friction member 73 that has a friction surface 732 with different profile, the disc spring assembly 72 is able to generate the actuating force when the angle formed between the first and second leaves 21, 22 reaches a predetermined value or range.

**[0038]** Referring to Figures 9 and 10, a second embodiment of the hinge which does not form part of the claimed invention, is similar to the first embodiment, and includes the leaf unit 2, the axle unit 4, the ring unit 5, and two torsional action units 3.

**[0039]** In this embodiment, the first leaf 21 is U-shaped and defines a receiving space, and the second leaf 22 is disposed in the receiving space of the first leaf 21.

**[0040]** Referring to Figures 10 to 13, the torsional action units 3 are inserted into the first barrels 211 and the second barrel 221 respectively in two opposite directions along the axis (X). Each of the torsional action units 3 includes a torsional tubular member 31 that is inserted into the first and second barrels 211, 221 and that is co-rotatable with the first leaf 21, a torsion spring 32 that is disposed in the torsional tubular member 31 for generating a restoring force, an adjusting member 33 that is rotatably disposed in the torsional tubular member 31 and that can be positioned relative to the torsional tubular member 31, two limiting rings 34 (see Figures 11 and 12), and a set screw 35.

**[0041]** The torsional tubular member 31 has a first tube section 311, and a second tube section 312 that abuts against the first tube section 311. The first tube section 311 has a toothed portion 3111 formed at an inner surrounding surface thereof, two mounting blocks 3112 each of which extends from an end of the first tube section 311 in the direction of the axis (X), and two outer limiting planes 3113 (only one is visible in Figure 10) that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211. The second tube section 312 has two spaced-apart mounting grooves 3121 each of which extends from an end of the second tube section 312 in the direction of the axis (X) and is engaged with a respective one of the mounting blocks 3112 of the first tube section 311, and two outer limiting planes 3122 that are formed at an outer surrounding surface thereof and that respectively abut against the inner limiting planes 2111 of one of the first barrels 211.

**[0042]** The torsion spring 32 has a middle coil 324, two

end coils 323 that are respectively connected to two opposite ends of the middle coil 324, and two end portions 321, 322 each of which is connected to a distal end of a respective one of the end coils 323. Each of the end coils 323 has at least two spirals that are spaced apart from each other by a first distance (D1). The middle coil 324 has a plurality of spirals. Two adjacent ones of the spirals of the middle coil 324 are spaced apart from each other by a second distance (D2) . The first distance (D1) is smaller than the second distance (D2).

**[0043]** The adjusting member 33 has a hexagonal adjusting hole 331 (see Figure 11) that is formed in an end surface thereof and that is exposed from the torsional tubular member 31, a limiting groove 332 that is formed in an outer surrounding surface thereof, a toothed portion 333 that separably engages the toothed portion 3111 of the first tube section 311, and a spring groove 334 that is formed in an end surface thereof and that is co-rotatably engaged with the end portion 321 of the torsion spring 32. The hexagonal adjusting hole 331 of the adjusting member 33 permits a hand tool (not shown) to engage therewith. By rotating the hand tool in a direction, the engagement between the toothed portion 333 of the adjusting member 33 and the toothed portion 3111 of the first tube section 311 can be adjusted so as to adjust the restoring force (i.e., an actuating force) generated by the torsion spring 32.

**[0044]** The limiting rings 34 are respectively disposed between the first tube section 311 and the second tube section 312 and at an end of the second tube section distal from the first tube section 311, and respectively surrounds the end portions 321, 322 of the torsion spring 32 to prevent the end portions 321, 322 of the torsion spring 32 from being separated from the spring groove 334 of the adjusting member 33. The set screw 33 engages threadedly the first tube section 311 of the torsional tubular member 31, and extends into the limiting groove 332 of the adjusting member 33 to limiting movement of the adjusting member 33 along the axis (X).

**[0045]** Referring to Figures 9, 10 and 12, the axle unit 4 includes a fixing member 41 that is removably mounted in the second barrel 221 of the second leaf 22 by a fastener 23 (see Figure 15) and that is co-rotatable with the second leaf 22, two torsional axles 42 (see Figures 10 and 12) each of which is mounted to a respective one of the torsional action units 3 and is co-rotatably connected to the fixing member 41.

**[0046]** The fixing member 41 has a rectangular fixing hole 411 that is formed in one of two opposite end surfaces of the fixing member 41 along the axis (X) and that extends along the axis (X), a fixing recess 412 (see Figure 9) that is formed in the other one of the opposite end surfaces of the fixing member 41, and two fixing grooves 413 (only one is visible in Figure 9) that are respectively formed in the opposite end surfaces of the fixing member 41. In one example, the fixing recess 412 is configured as a rectangular recess. In one example, the fixing hole 411 is formed through the opposite end surfaces of the fixing member 41.

**[0047]** Each of the torsional axles 42 extends along the axis (X) through the end coils 323 and the middle coil 324 of the torsion spring 32 of the corresponding torsional action unit 3, and has an axle portion 421, and a flange portion 423 that is formed with a breach 422. The axle portions 421 of the torsional axles 42 respectively and co-rotatably engage the fixing hole 411 and the fixing recess 412 of the fixing member 41 (see Figure 15) . The breaches 422 of the torsional axles 42 are respectively aligned with the fixing grooves 413 of the fixing member 41, so that the end portions 322 of the torsion springs 32 of the torsional action units 3 respectively extend through the breaches 422 of the torsional axles 42 to respectively engage the fixing grooves 413 of the fixing member 41.

**[0048]** Referring to Figure 14, since the second leaf 22 is disposed in the receiving space defined by the U-shaped first leaf 21, the second embodiment is suitable for use on the occasion that a gap between the first and second objects 11, 12 (with reference to Figure 1) is equal to or slightly greater than the thickness of the first leaf 21. During installation of the hinge onto the first and second objects 11, 12, the first leaf 21 can be quickly and accurately positioned relative to the first object 11 by moving the first positioning surface 213 to abut against the edge 111 of the first object 11, and the second leaf 22 can be quickly and accurately positioned relative to the second object 12 by moving the second positioning surface 223 to abut against the edge 121 of the second object 12. As such, the first and second objects 11, 12 are accurately positioned relative to each other, and can be smoothly rotated relative to each other.

**[0049]** Referring to Figures 9, 14 and 15, when the first and second leaves 21, 22 are rotated relative to each other by an external force, each of the torsional axles 42 is rotated relative to the torsional tubular member 31 of the corresponding torsional action unit 3 to twist the torsion spring 32 of the corresponding torsional action unit 3 in a direction such that the diameter of the torsion spring 32 decreases and that each of the first and distances (D1, D2) decreases so as to generate a restoring force (i.e., an actuating force) . By such, when the external force is removed, the torsion spring 32 of each of the torsional action units 3 restores to rotate the first and second leaves 21, 22 relative to each other.

**[0050]** The second embodiment employs two torsion springs 32 to generate the restoring force, and is therefore suitable for a heavy door leaf. It should be noted that after the torsion spring 32 is twisted by an external force such that any two adjacent ones of the spirals of each of the end coils 323 abut against each other (i.e., D1=0, $D2 \neq 0$), further relative rotation between the corresponding adjusting member 33 and the corresponding torsional axle 42 caused by the external force would only deform the middle coil 324 (because the end coils 323 cannot be further deformed). Accordingly, in the case that each of the middle coil 324 and the end coils 323 has the same number of spirals, upon each relative rotation between

the corresponding adjusting member 33 and the corresponding torsional axle 42 by a predetermined angle caused by the external force, the increment of the restoring force generated by the torsion spring 32 at the time that any two adjacent ones of the spirals of each of the end coils 323 abut against each other is three times the increment of the restoring force generated by the torsion spring 32 at the time that the spirals of each of the end coils 323 are spaced apart from each other. As such, the second embodiment is suitable for a heavy door leaf.

**[0051]** It should be noted that the first leaf 21 can be connected to any one of a door leaf and a door frame while the second leaf 22 is connected to the other one of the door leaf and the door frame.

**[0052]** Referring to Figure 16, a third embodiment of the hinge which does not form part of the claimed invention, is similar to the second embodiment, and includes the leaf unit 2, the axle unit 4, the ring unit 5, the torsional action unit 3 and the second action unit 7. The axle unit 4 of the third embodiment includes the fixing member 41 that is removably mounted in the second barrel 221 of the second leaf 22 by the fastener 23 and that is co-rotatable with the second leaf 22, the torsional axle 42 that is mounted to the torsional action unit 3 and that is co-rotatably connected to the fixing member 41, and the second axle 44 that is mounted to the second action unit 7 and that is co-rotatably connected to the fixing member 41.

**[0053]** The cooperation of the components of the third embodiment can be comprehended by one of ordinary skill in the art with reference to the preceding paragraphs, and would not be further described.

**[0054]** Referring to Figure 17, a fourth embodiment of the hinge which does not form part of the claimed invention, is similar to the second embodiment, and includes the leaf unit 2, the axle unit 4, the ring unit 5, the torsional action unit 3 and the first action unit 6. The axle unit 4 of the fourth embodiment includes the fixing member 41 that is removably mounted in the second barrel 221 of the second leaf 22 by the fastener 23 and that is co-rotatable with the second leaf 22, the torsional axle 42 that is mounted to the torsional action unit 3 and that is co-rotatably connected to the fixing member 41, and the first axle 43 that is mounted to the first action unit 6 and that is co-rotatably connected to the fixing member 41.

**[0055]** The cooperation of the components of the fourth embodiment can be comprehended by one of ordinary skill in the art with reference to the preceding paragraphs, and would not be further described.

**[0056]** Referring to Figure 18, a modification of the first action unit 6 includes the first tubular member 61 that is inserted into the first and second barrels 211, 221 (see Figure 2) and that is co-rotatable with the first barrels 211, the hydraulic module 62 that is disposed in the first tubular member 61, the proximal acting member 64 that is co-rotatably mounted in the first tubular member 61, and the cap member 65 that is mounted to an end of the first tubular member 61. It should be noted that the distal acting member 63 (see Figure 3) is omitted. The operation of the modification is similar to that of the first action unit 6 shown in Figure 4, and would not be further described.

**[0057]** Referring to Figures 19 and 20, a fifth embodiment of the hinge which does not form part of the claimed invention, is similar to the first embodiment, and includes the leaf unit 2, the axle unit 4, the ring unit 5, the first action unit 6 and the second action unit 7.

**[0058]** The fixing member 41 has a different configuration such that the fixing member 41 and the second axle 44 are moved into the second barrel 221 of the second leaf 22 via the lower opening of the second barrel 221. The axle portion 431 of the first axle 43 engages the fixing hole 411 of the fixing member 41 and the fixing hole 440 of the second axle 44, so the fixing member 41, the first axle 43 and the second axle 44 are co-rotatable. The protrusions 442 (only one is visible in Figure 19) of the second axle 44 are in frictional contact with the friction surface 732 of the friction member 73 of the second action unit 7.

**[0059]** Referring to Figures 21 and 22, a sixth embodiment of the hinge which does not form part of the claimed invention, is similar to the second embodiment, and includes the leaf unit 2, the axle unit 4, the ring unit 5, and the torsional action units 3.

**[0060]** The fixing member 41 has a different configuration, and is moved into the second barrel 221 of the second leaf 22 via the lower opening of the second barrel 221. The axle portions 421 of the torsional axles 42 respectively and co-rotatably engage the fixing hole 411 and the fixing recess 412 of the fixing member 41 (see Figure 22) . The breaches 422 of the torsional axles 42 are respectively aligned with the fixing grooves 413 of the fixing member 41, so that the end portions 322 of the torsion springs 32 of the torsional action units 3 respectively extend through the breaches 422 of the torsional axles 42 to respectively engage the fixing grooves 413 of the fixing member 41.

**[0061]** Referring to Figures 23 and 24, each of the spacer assemblies 52 may includes two spacers 521. The surrounding walls 522 of the spacers 521 of each of the spacer assemblies 52 respectively extend into the second barrel 221 and one of the first barrel 211, and the flange wall 523 of the spacers 521 of each of the spacer assemblies 52 abut against each other and are disposed between the second barrel 221 and the one of the first barrels 211.

**[0062]** In some examples, each of the ring members 51 may be made of Polyoxymethylene (POM) or Polytetrafluoroethylene (PTFE), and serves as a bushing for facilitating relative rotation between the corresponding components. Each of the spacer assemblies 52 may be made of metal, such as aluminum, so as to be wear-resistant. Moreover, the material of the first barrels 211, the second barrel 221 and the exposed flange wall 523 of the spacers 521 of each of the spacer assemblies 52 may be similar to each other, so the hinge may be visually

aesthetic.

**[0063]** Referring to Figure 25, a modification of the torsional action unit 3, according to an embodiment of the invention, further includes an auxiliary spring 36 and a slide block 37.

**[0064]** The torsional axle 42 further has a rectangular auxiliary axle portion 424 that is opposite to the axle portion 421.

**[0065]** The adjusting member 33 further has an inclined surface 335 that is opposite to the hexagonal adjusting hole 331.

**[0066]** The auxiliary spring 36 is sleeved on the torsional axle 42, and is surrounded by the torsion spring 32.

**[0067]** The slide block 37 abuts against an end of the auxiliary spring 36, and has a rectangular hole 371 that is engaged with the auxiliary axle portion 424 of the torsional axle 42, and an inclined surface 372 that is opposite to the auxiliary spring 36 and that is in slidable contact with the inclined surface 335 of the adjusting member 33. The slide block 37 is co-rotatable with the torsional axle 42, and is movable along the auxiliary axle portion 424 of the torsional axle 42 along the axis (X).

**[0068]** Referring to Figures 26 and 27, when the first and second leaves 21, 22 are rotated relative to each other by an external force, the torsional axle 42 and the slide block 37 are rotated relative to each other, so that the inclined surface 335 of the adjusting member 33 pushes the inclined surface 372 of the slide block 37 to move the slide block 37 away from the adjusting member 33 along the axis (X) to compress the auxiliary spring 36 so as to generate a restoring force. When the external force is removed, the torsion spring 32 and the auxiliary spring 36 restore to rotate the first and second leaves 21, 22 relative to each other, and to move the slide block 37 toward the adjusting member 33 along the axis (X).

**[0069]** In summary, the advantages of the disclosure are as follows:

1. The torsional axle 42, the first axle 43 or the second axle 44 can be easily and co-rotatably mounted to the second barrel 221 of the second leaf 22 by virtue of the fixing member 41 that is removably mounted in the second barrel 221 without forming mounting structures on the inner surrounding surface of the second barrel 221. Moreover, a worn fixing member 41 can be easily substituted with a new fixing member 41.
2. Each of the the ring members 51 and the spacer assemblies 52 serves as a bushing for facilitating relative rotation between the corresponding components.
3. The configuration of the torsion spring 32 enables the torsion spring 32 to generate a greater restoring force.
4. Each of the second and the subsequent embodiments is suitable for use on the occasion that a gap between the first and second objects 11, 12 (with reference to Figure 1) is equal to or slightly greater than the thickness of the first leaf 21 since the second leaf 22 is disposed in the receiving space defined by the U-shaped first leaf 21.
5. During installation of the hinge onto the first and second objects 11, 12, the first leaf 21 can be quickly and accurately positioned relative to the first object 11 by moving the first positioning surface 213 to abut against the edge 111 of the first object 11, and the second leaf 22 can be quickly and accurately positioned relative to the second object 12 by moving the second positioning surface 223 to abut against the edge 121 of the second object 12. Therefore, the first and second objects 11, 12 are accurately positioned relative to each other, and can be smoothly rotated relative to each other.

**[0070]** In the description above, for the purposes of explanation, numerous specific details have been set forth in order to provide a thorough understanding of the embodiments. It will be apparent, however, to one skilled in the art, that one or more other embodiments may be practiced without some of these specific details.

**[0071]** It should be further appreciated that in the description, various features are sometimes grouped together in a single embodiment, figure, or description thereof for the purpose of streamlining the disclosure and aiding in the understanding of various inventive aspects, and that one or more features or specific details from one embodiment may be practiced together with one or more features or specific details from another embodiment, where appropriate, without departing from the scope of the invention as defined by the claims.

## Claims

**1.** A hinge adapted to interconnect first and second objects (11, 12) including:

a leaf unit (2) including first and second leaves (21, 22) that are rotatable relative to each other, said first leaf (21) having at least two first barrels (211), said second leaf (22) having at least one second barrel (221) that is spaced apart from said first barrels (211) along an axis (X);
two force-providing units (3) being inserted into said first barrels (211) and said second barrel (221) respectively in two opposite directions along the axis (X), and being co-rotatable with said first leaf (21); and
an axle unit (4) including a fixing member (41) that is removably mounted in said second barrel (221) of said second leaf (22) and that is co-rotatable with said second leaf (22), and two axles (42) that are respectively associated with said force-providing units (3) and that are co-rotatably mounted to said fixing member (41), said fixing member (41) and said axles (42) be-

ing configured as independent components, each of said axles (42) and said corresponding force-providing unit (3) being rotated relative to each other upon relative rotation between said first and second leaves (21, 22) so that said corresponding force-providing unit (3) generates an actuating force that acts between said first and second leaves (21, 22);

said hinge being **characterized in that**

each of said force-providing units (3) includes a tubular member (31) that is inserted into said first and second barrels (211, 221) and that is co-rotatable with said first leaf (21), a torsion spring (32) that is disposed in said tubular member (31) and that surrounds said corresponding axle (42), and an adjusting member (33) that is rotatably disposed in said tubular member (31) and that is able to be positioned relative to said tubular member (31), said torsion spring (32) of each of said force-providing units (3) having two end portions (321, 322) that are respectively connected to said adjusting member (33) of said force-providing unit (3) and said corresponding axle (42); and

at least one of said force-providing unit (3) further includes an auxiliary spring (36) and a slide block (37), said axle (42) that corresponds to the one of said force-providing unit (3) having a rectangular auxiliary axle portion (424), said adjusting member (33) of the one of said force-providing unit (3) further having an inclined surface (335), said auxiliary spring (36) being sleeved on said corresponding axle (42) and surrounded by said torsion spring (32) of the one of said force-providing unit (3), said slide block (37) abutting against an end of said auxiliary spring (36), and having a rectangular hole (371) that is engaged with said auxiliary axle portion (424) of said corresponding axle (42), and an inclined surface (372) that faces said inclined surface (335) of said adjusting member (33), said slide block (37) being co-rotatable with said corresponding axle (42), and being movable along said auxiliary axle portion (424) of said corresponding axle (42) along the axis (X).

**2.** The hinge as claimed in claim 1, **characterized in that** said first leaf (21) further has a first clinging surface (212) that clings to the first object (11), and a first positioning surface (213) that is parallel to the axis (X), that is connected to said first clinging surface (212) and that is not coplanar with said first clinging surface (212), said first positioning surface (213) permitting an edge (111) of the first object (11) to abut thereagainst.

**3.** The hinge as claimed in any one of claims 1 and 2, **characterized in that** said second leaf (22) further has a second clinging surface (222) that clings to the second object (12), and a second positioning surface (223) that is parallel to the axis (X), that is connected to said second clinging surface (222) and that is not coplanar with said second clinging surface (222), said second positioning surface (223) permitting an edge (121) of the second object (12) to abut thereagainst.

**4.** The hinge as claimed in any one of claims 1 to 3, **characterized in that** said first leaf (21) is U-shaped and defines a receiving space, said second leaf (22) being disposed in said receiving space of said first leaf (21).

**5.** The hinge as claimed in any one of claims 1 to 4, further **characterized by** a ring unit (5), said ring unit (5) including a plurality of ring members (51) and a spacer assembly (52), said ring member (51) being respectively disposed between one of said force-providing units (3) and said second barrel (221) and between the other one of said force-providing units (3) and said second barrel (221), said spacer assembly (52) including at least one spacer (521), said spacers (521) having an surrounding wall (522) that is disposed between said second barrel (221) and one of said force-providing units (3), and a flange wall (523) that is disposed between one of said first barrels (211) and said second barrel (221) .

**6.** The hinge as claimed in claim 5, **characterized in that** each of said first leaf (21), said second leaf (22) and said spacer (521) is made of metal, each of said ring members (51) being made of Polyoxymethylene (POM) or Polytetrafluoroethylene (PTFE).

**7.** The hinge as claimed in any one of claim 1, further **characterized in that** said torsion spring (32) of at least one said force-providing units (3) has a middle coil (324), two end coils (323) that are respectively connected to two opposite ends of said middle coil (324), and two end portions (321, 322) that are respectively connected to said end coils (323)and that are respectively connected to said adjusting member (33) and said corresponding axle (42), each of said end coils (323) having at least two spirals that are spaced apart from each other by a first distance (D1), said middle coil (324) having a plurality of spirals, two adjacent ones of said spirals of said middle coil (324) being spaced apart from each other by a second distance (D2), the first distance (D1) being smaller than the second distance (D2) .

**8.** The hinge as claimed in claim 1, further **characterized in that** said fixing member (41) has a rectangular fixing hole (411) and a fixing recess (412) that

are respectively formed in two opposite end surfaces of said fixing member (41) along the axis (X), each of said axles (42) having an axle portion (421), said axle portions (421) of said axles (42) respectively engaging said fixing hole (411) and said fixing recess (412) of said fixing member (41) .

**9.** The hinge as claimed in claim 8, further **characterized in that** said fixing member (41) further has two fixing grooves (413) that are respectively formed in said opposite end surfaces thereof, each of said axles (42) further having a flange portion (423) that is formed with a breach (422), said breaches (422) of said axles (42) being respectively aligned with said fixing grooves (413) of said fixing member (41), so that one of said end portions (322) of said torsion spring (32) of each of said force-providing units (3) extending through said breach (422) of said corresponding axle (42) to engage said corresponding fixing groove (413) of said fixing member (41).

**10.** The hinge as claimed in any one of claims 1 to 6, **characterized in that** each of said first barrel (211) has two inner limiting planes (2111) that are formed on an inner surrounding surface thereof, each of said force-providing units (3) including a tubular member (31) that is inserted into said first and second barrels (211, 221) and that is co-rotatable with said first leaf (21), said tubular member (31) of each of said force-providing units (3) having two outer limiting planes that are formed at an outer surrounding surface thereof and that respectively abut against said inner limiting planes (2111) of at least one of said first barrels (211).

**11.** The hinge as claimed in claim 10, further **characterized in that** said tubular member (31) of at least one of said force-providing units (3) includes a first tube section (311), and a second tube section (312) that abuts against said first tube section (311), a junction between said first tube section (311) and said second tube section (312) being located within one of said first barrels (211).

**Patentansprüche**

**1.** Scharnier, das geeignet ist, ein erstes und ein zweites Objekt (11, 12) miteinander zu verbinden, umfassend:

eine Flügeleinheit (2), die einen ersten und einen zweiten Flügel (21, 22) umfasst, die relativ zueinander gedreht werden können, wobei der erste Flügel (21) mindestens zwei erste Hülsen (211) aufweist, wobei der zweite Flügel (22) mindestens eine zweite Hülse (221) aufweist, die entlang einer Achse (X) von den ersten Hülsen (211) beabstandet ist;

zwei Kraftbereitstellungseinheiten (3), die in zwei entgegengesetzten Richtungen entlang der Achse (X) jeweils in die ersten Hülsen (211) und die zweite Hülse (221) eingesetzt sind und zusammen mit dem ersten Flügel (21) gedreht werden können; und

eine Achseinheit (4), die ein Befestigungselement (41), das herausnehmbar in der zweiten Hülse (221) des zweiten Flügels (22) angebracht ist und das zusammen mit dem zweiten Flügel (22) gedreht werden kann, und zwei Achsen (42) umfasst, die jeweils mit den Kraftbereitstellungseinheiten (3) verknüpft sind und die gemeinsam drehbar an dem Befestigungselement (41) angebracht sind, wobei das Befestigungselement (41) und die Achsen (42) als unabhängige Komponenten ausgelegt sind, wobei jede der Achsen (42) und die entsprechende Kraftbereitstellungseinheit (3) bei relativer Drehung zwischen dem ersten und dem zweiten Flügel (21, 22) relativ zueinander gedreht werden, sodass die entsprechende Kraftbereitstellungseinheit (3) eine Betätigungskraft erzeugt, die zwischen dem ersten und dem zweiten Flügel (21, 22) wirkt;

wobei das Scharnier **dadurch gekennzeichnet ist, dass**

jede der Kraftbereitstellungseinheiten (3) ein rohrförmiges Element (31), das in die erste und die zweite Hülse (211, 221) eingesetzt ist und das zusammen mit dem ersten Flügel (21) gedreht werden kann, eine Torsionsfeder (32), die in dem rohrförmigen Element (31) angeordnet ist und die die entsprechende Achse (42) umgibt, und ein Einstellelement (33) umfasst, das drehbar in dem rohrförmigen Element (31) angeordnet ist und das relativ zu dem rohrförmigen Element (31) positioniert werden kann, wobei die Torsionsfeder (32) jeder der Kraftbereitstellungseinheiten (3) zwei Endabschnitte (321, 322) aufweist, die jeweils mit dem Einstellelement (33) der Kraftbereitstellungseinheit (3) und der entsprechenden Achse (42) verbunden sind; und

mindestens eine der Kraftbereitstellungseinheit (3) weiter eine Hilfsfeder (36) und einen Gleitblock (37) umfasst, die Achse (42), die der einen der Kraftbereitstellungseinheit (3) entspricht, einen rechteckigen Hilfsachsabschnitt (424) aufweist, das Einstellelement (33) der einen der Kraftbereitstellungseinheit (3) weiter eine schräge Fläche (335) aufweist, die Hilfsfeder (36) muffenartig auf die entsprechende Achse (42) gesteckt und von der Torsionsfeder (32) der einen der Kraftbereitstellungseinheit (3) umgeben ist, der Gleitblock (37) an einem Ende der Hilfsfeder (36) anliegt und ein rechteckiges Loch

(371), das mit dem Hilfsachsabschnitt (424) der entsprechenden Achse (42) in Eingriff gebracht ist, und eine schräge Fläche (372) aufweist, die der schrägen Fläche (335) des Einstellelements (33) zugewandt ist, wobei der Gleitblock (37) zusammen mit der entsprechenden Achse (42) gedreht werden kann und entlang des Hilfsachsabschnitts (424) der entsprechenden Achse (42) entlang der Achse (X) bewegt werden kann.

2. Scharnier nach Anspruch 1, **dadurch gekennzeichnet, dass** der erste Flügel (21) weiter eine erste Auflagefläche (212), die an dem ersten Objekt (11) aufliegt, und eine erste Positionierungsfläche (213) aufweist, die parallel zur Achse (X) verläuft, die mit der ersten Auflagefläche (212) verbunden ist und die nicht mit der ersten Auflagefläche (212) koplanar ist, wobei es die erste Positionierungsfläche (213) ermöglicht, dass eine Kante (111) des ersten Objekts (11) an ihr anliegt.

3. Scharnier nach einem der Ansprüche 1 und 2, **dadurch gekennzeichnet, dass** der zweite Flügel (22) weiter eine zweite Auflagefläche (222), die an dem zweiten Objekt (12) aufliegt, und eine zweite Positionierungsfläche (223) aufweist, die parallel zur Achse (X) verläuft, die mit der zweiten Auflagefläche (222) verbunden ist und die nicht mit der zweiten Auflagefläche (222) koplanar ist, wobei es die zweite Positionierungsfläche (223) ermöglicht, dass eine Kante (121) des zweiten Objekts (12) an ihr anliegt.

4. Scharnier nach einem der Ansprüche 1 bis 3, **dadurch gekennzeichnet, dass** der erste Flügel (21) U-förmig ist und einen Aufnahmeraum definiert, wobei der zweite Flügel (22) im Aufnahmeraum des ersten Flügels (21) angeordnet ist.

5. Scharnier nach einem der Ansprüche 1 bis 4, das weiter durch eine Ringeinheit (5) gekennzeichnet ist, wobei die Ringeinheit (5) eine Vielzahl von Ringelementen (51) und eine Abstandshalterbaugruppe (52) umfasst, wobei das Ringelement (51) jeweils zwischen einer der Kraftbereitstellungseinheiten (3) und der zweiten Hülse (221), und zwischen der anderen der Kraftbereitstellungseinheiten (3) und der zweiten Hülse (221) angeordnet ist, wobei die Abstandshalterbaugruppe (52) mindestens einen Abstandshalter (521) umfasst, wobei die Abstandshalter (521) eine umgebende Wand (522), die zwischen der zweiten Hülse (221) und einer der Kraftbereitstellungseinheiten (3) angeordnet ist, und eine Flanschwand (523) aufweisen, die zwischen einer der ersten Hülsen (211) und der zweiten Hülse (221) angeordnet ist.

6. Scharnier nach Anspruch 5, **dadurch gekennzeichnet, dass** jedes aus dem ersten Flügel (21), dem zweiten Flügel (22) und dem Abstandshalter (521) aus Metall besteht, wobei jedes der Ringelemente (51) aus Polyoxymethylen (POM) oder Polytetrafluorethylen (PTFE) besteht.

7. Scharnier nach einem aus Anspruch 1, weiter **dadurch gekennzeichnet, dass** die Torsionsfeder (32) mindestens einer der Kraftbereitstellungseinheiten (3) eine mittlere Wicklung (324), zwei Endwicklungen (323), die jeweils mit zwei gegenüberliegenden Enden der mittleren Wicklung (324) verbunden sind, und zwei Endabschnitte (321, 322) aufweist, die jeweils mit den Endwicklungen (323) verbunden sind und die jeweils mit dem Einstellelement (33) und der entsprechenden Achse (42) verbunden sind, wobei jede der Endwicklungen (323) mindestens zwei Windungen aufweist, die um einen ersten Abstand (D1) voneinander beabstandet sind, wobei die mittlere Wicklung (324) eine Vielzahl von Windungen aufweist, wobei zwei benachbarte der Windungen der mittleren Wicklung (324) um einen zweiten Abstand (D2) voneinander beabstandet sind, wobei der erste Abstand (D1) kleiner ist als der zweite Abstand (D2).

8. Scharnier nach Anspruch 1, weiter **dadurch gekennzeichnet, dass** das Befestigungselement (41) ein rechteckiges Befestigungsloch (411) und eine Befestigungsausnehmung (412) aufweist, die jeweils in zwei gegenüberliegenden Endflächen des Befestigungselements (41) entlang der Achse (X) gebildet sind, wobei jede der Achsen (42) einen Achsabschnitt (421) aufweist, wobei die Achsabschnitte (421) der Achsen (42) jeweils in das Befestigungsloch (411) und die Befestigungsausnehmung (412) des Befestigungselements (41) eingreifen.

9. Scharnier nach Anspruch 8, weiter **dadurch gekennzeichnet, dass** das Befestigungselement (41) weiter zwei Befestigungsnuten (413) aufweist, die jeweils in den gegenüberliegenden Endflächen desselben gebildet sind, wobei jede der Achsen (42) weiter einen Flanschabschnitt (423) aufweist, der mit einem Durchbruch (422) ausgebildet ist, wobei die Durchbrüche (422) der Achsen (42) jeweils mit den Befestigungsnuten (413) des Befestigungselements (41) fluchten, sodass sich einer der Endabschnitte (322) der Torsionsfeder (32) jeder der Kraftbereitstellungseinheiten (3) durch den Durchbruch (422) der entsprechenden Achse (42) erstreckt, um in die entsprechende Befestigungsnut (413) des Befestigungselements (41) einzugreifen.

10. Scharnier nach einem der Ansprüche 1 bis 6, **dadurch gekennzeichnet, dass** jede der ersten Hülse (211) zwei innere Begrenzungsebenen (2111) aufweist, die an einer inneren umgebenden Fläche derselben gebildet sind, wobei jede der Kraftbereitstel-

lungseinheiten (3) ein rohrförmiges Element (31) umfasst, das in die ersten Hülsen und die zweite Hülse (211, 221) eingesetzt ist und das zusammen mit dem ersten Flügel (21) gedreht werden kann, wobei das rohrförmige Element (31) jeder der Kraftbereitstellungseinheiten (3) zwei äußere Begrenzungsebenen aufweist, die an einer äußeren umgebenden Fläche derselben gebildet sind und die jeweils an den inneren Begrenzungsebenen (2111) mindestens einer der ersten Hülsen (211) anliegen.

**11.** Scharnier nach Anspruch 10, weiter **dadurch gekennzeichnet, dass** das rohrförmige Element (31) mindestens einer der Kraftbereitstellungseinheiten (3) einen ersten Rohrabschnitt (311) und einen zweiten Rohrabschnitt (312) umfasst, der am ersten Rohrabschnitt (311) anliegt, wobei ein Übergang zwischen dem ersten Rohrabschnitt (311) und dem zweiten Rohrabschnitt (312) innerhalb einer der ersten Hülsen (211) liegt.

**Revendications**

**1.** Charnière adaptée pour interconnecter des premier et second objets (11, 12) comportant :

une unité de lames (2) comportant des première et seconde lames (21, 22) qui sont rotatives l'une par rapport à l'autre, ladite première lame (21) ayant au moins deux premiers cylindres (211), ladite seconde lame (22) ayant au moins un second cylindre (221) qui est espacé desdits premiers cylindres (211) le long d'un axe (X) ;
deux unités de fourniture de force (3) qui sont insérées dans lesdits premiers cylindres (211) et ledit second cylindre (221) respectivement dans deux directions opposées le long de l'axe (X), et qui sont co-rotatives avec ladite première lame (21) ; et
une unité d'axe d'assemblage (4) comportant un élément de fixation (41) qui est monté de manière amovible dans ledit second cylindre (221) de ladite seconde lame (22) et qui est co-rotatif avec ladite seconde lame (22), et deux axes d'assemblage (42) qui sont respectivement associés auxdites unités de fourniture de force (3) et qui sont montés de manière co-rotative sur ledit élément de fixation (41), ledit élément de fixation (41) et lesdits axes d'assemblage (42) étant configurés comme des composants indépendants, chacun desdits axes d'assemblage (42) et ladite unité de fourniture de force (3) correspondante étant mis en rotation l'un par rapport à l'autre lors d'une rotation relative entre lesdites première et seconde lames (21, 22) de sorte que ladite unité de fourniture de force (3) correspondante génère une force d'actionnement qui agit entre lesdites première et seconde lames (21, 22) ;
ladite charnière étant **caractérisée en ce que** chacune desdites unités de fourniture de force (3) comporte un élément tubulaire (31) qui est inséré dans lesdits premier et second cylindres (211, 221) et qui est co-rotatif avec ladite première lame (21), un ressort de torsion (32) qui est disposé dans ledit élément tubulaire (31) et qui entoure ledit axe d'assemblage (42) correspondant, et un élément de réglage (33) qui est disposé de manière rotative dans ledit élément tubulaire (31) et qui est en mesure d'être positionné par rapport audit élément tubulaire (31), ledit ressort de torsion (32) de chacune desdites unités de fourniture de force (3) ayant deux parties d'extrémité (321, 322) qui sont respectivement connectées audit élément de réglage (33) de ladite unité de fourniture de force (3) et audit axe d'assemblage (42) correspondant ; et
au moins une desdites unité de fourniture de force (3) comporte en outre un ressort auxiliaire (36) et un bloc coulissant (37), ledit axe d'assemblage (42) qui correspond à l'une desdites unité de fourniture de force (3) ayant une partie d'axe d'assemblage auxiliaire (424) rectangulaire, ledit élément de réglage (33) de l'une desdites unité de fourniture de force (3) ayant en outre une surface inclinée (335), ledit ressort auxiliaire (36) étant manchonné sur ledit axe d'assemblage (42) correspondant et entouré par ledit ressort de torsion (32) de l'une desdites unité de fourniture de force (3), ledit bloc coulissant (37) venant en butée contre une extrémité dudit ressort auxiliaire (36), et ayant un trou rectangulaire (371) qui est en prise avec ladite partie d'axe d'assemblage auxiliaire (424) dudit axe d'assemblage (42) correspondant, et une surface inclinée (372) qui fait face à ladite surface inclinée (335) dudit élément de réglage (33), ledit bloc coulissant (37) étant co-rotatif avec ledit axe d'assemblage (42) correspondant, et étant mobile le long de ladite partie d'axe d'assemblage auxiliaire (424) dudit axe d'assemblage (42) correspondant le long de l'axe (X).

**2.** Charnière selon la revendication 1, **caractérisée en ce que** ladite première lame (21) a en outre une première surface d'accrochage (212) qui s'accroche au premier objet (11), et une première surface de positionnement (213) qui est parallèle à l'axe (X), qui est connectée à ladite première surface d'accrochage (212) et qui n'est pas coplanaire avec ladite première surface d'accrochage (212), ladite première surface de positionnement (213) permettant à un bord (111) du premier objet (11) de venir en butée contre celle-ci.

3. Charnière selon l'une quelconque des revendications 1 et 2, **caractérisée en ce que** ladite seconde lame (22) a en outre une seconde surface d'accrochage (222) qui s'accroche au second objet (12), et une seconde surface de positionnement (223) qui est parallèle à l'axe (X), qui est connectée à ladite seconde surface d'accrochage (222) et qui n'est pas coplanaire avec ladite seconde surface d'accrochage (222), ladite seconde surface de positionnement (223) permettant à un bord (121) du second objet (12) de venir en butée contre celle-ci.

4. Charnière selon l'une quelconque des revendications 1 à 3, **caractérisée en ce que** ladite première lame (21) est en forme de U et définit un espace de réception, ladite seconde lame (22) étant disposée dans ledit espace de réception de ladite première lame (21).

5. Charnière selon l'une quelconque des revendications 1 à 4, **caractérisée en outre par** une unité d'anneau (5), ladite unité d'anneau (5) comportant une pluralité d'éléments d'anneau (51) et un ensemble entretoise (52), ledit élément d'anneau (51) étant respectivement disposé entre l'une desdites unités de fourniture de force (3) et ledit second cylindre (221) et entre l'autre desdites unités de fourniture de force (3) et ledit second cylindre (221), ledit ensemble entretoise (52) comportant au moins une entretoise (521), lesdites entretoises (521) ayant une paroi d'entourage (522) qui est disposée entre ledit second cylindre (221) et l'une desdites unités de fourniture de force (3), et une paroi de bride (523) qui est disposée entre l'un desdits premiers cylindres (211) et ledit second cylindre (221).

6. Charnière selon la revendication 5, **caractérisée en ce que** chacune parmi ladite première lame (21), ladite seconde lame (22) et ladite entretoise (521) est réalisée en métal, chacun desdits éléments d'anneau (51) étant réalisé en polyoxyméthylène (POM) ou en polytétrafluoroéthylène (PTFE).

7. Charnière selon l'une de la revendication 1, **caractérisée en outre en ce que** ledit ressort de torsion (32) d'au moins l'une desdites unités de fourniture de force (3) a une bobine centrale (324), deux bobines d'extrémité (323) qui sont respectivement connectées à deux extrémités opposées de ladite bobine centrale (324), et deux parties d'extrémité (321, 322) qui sont respectivement connectées auxdites bobines d'extrémité (323) et qui sont respectivement connectées audit élément de réglage (33) et audit axe d'assemblage (42) correspondant, chacune desdites bobines d'extrémité (323) ayant au moins deux spirales qui sont espacées l'une de l'autre par une première distance (D1), ladite bobine centrale (324) ayant une pluralité de spirales, deux spirales adjacentes desdites spirales de ladite bobine centrale (324) étant espacées l'une de l'autre par une seconde distance (D2), la première distance (D1) étant plus petite que la seconde distance (D2).

8. Charnière selon la revendication 1, **caractérisée en outre en ce que** ledit élément de fixation (41) a un trou de fixation (411) rectangulaire et un évidement de fixation (412) qui sont respectivement formés dans deux surfaces d'extrémité opposées dudit élément de fixation (41) le long de l'axe (X), chacun desdits axes d'assemblage (42) ayant une partie d'axe d'assemblage (421), lesdites parties d'axe d'assemblage (421) desdits axes d'assemblage (42) se mettant en prise respectivement avec ledit trou de fixation (411) et ledit évidement de fixation (412) dudit élément de fixation (41).

9. Charnière selon la revendication 8, **caractérisée en outre en ce que** ledit élément de fixation (41) a en outre deux rainures de fixation (413) qui sont respectivement formées dans lesdites surfaces d'extrémité opposées de celui-ci, chacun desdits axes d'assemblage (42) ayant en outre une partie de bride (423) qui est formée avec une brèche (422), lesdites brèches (422) desdits axes d'assemblage (42) étant respectivement alignées avec lesdites rainures de fixation (413) dudit élément de fixation (41), de sorte que l'une desdites parties d'extrémité (322) dudit ressort de torsion (32) de chacune desdites unités de fourniture de force (3) s'étende à travers ladite brèche (422) dudit axe d'assemblage (42) correspondant pour se mettre en prise avec ladite rainure de fixation (413) correspondante dudit élément de fixation (41).

10. Charnière selon l'une quelconque des revendications 1 à 6, **caractérisée en ce que** chacun desdits premier cylindre (211) a deux plans de limitation intérieurs (2111) qui sont formés sur une surface intérieure d'entourage de celui-ci, chacune desdites unités de fourniture de force (3) comportant un élément tubulaire (31) qui est inséré dans lesdits premier et second cylindres (211, 221) et qui est corotatif avec ladite première lame (21), ledit élément tubulaire (31) de chacune desdites unités de fourniture de force (3) ayant deux plans de limitation extérieurs qui sont formés sur une surface extérieure d'entourage de celui-ci et qui viennent respectivement en butée contre lesdits plans de limitation intérieurs (2111) d'au moins un desdits premiers cylindres (211).

11. Charnière selon la revendication 10, **caractérisée en outre en ce que** ledit élément tubulaire (31) d'au moins l'une desdites unités de fourniture de force (3) comporte une première section de tube (311), et une seconde section de tube (312) qui vient en butée

contre ladite première section de tube (311), une jonction entre ladite première section de tube (311) et ladite seconde section de tube (312) étant située à l'intérieur de l'un desdits premiers cylindres (211).

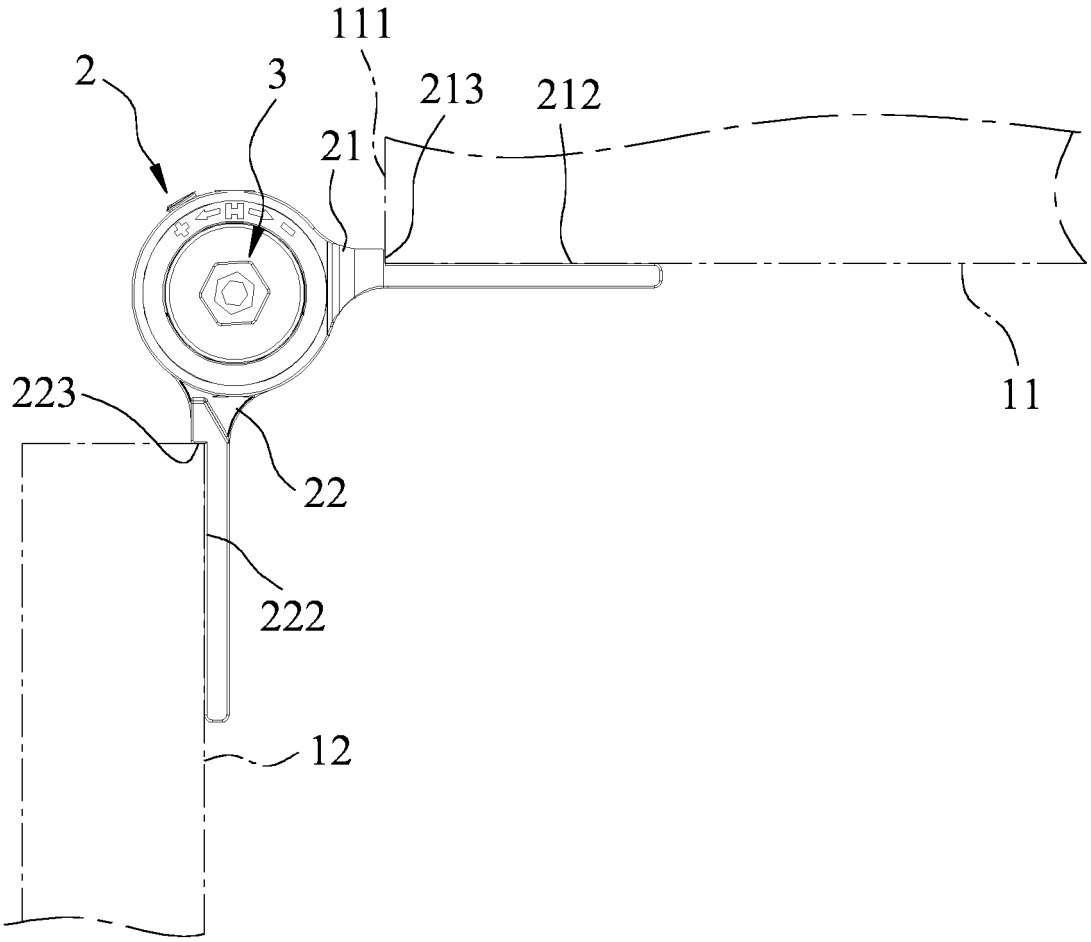
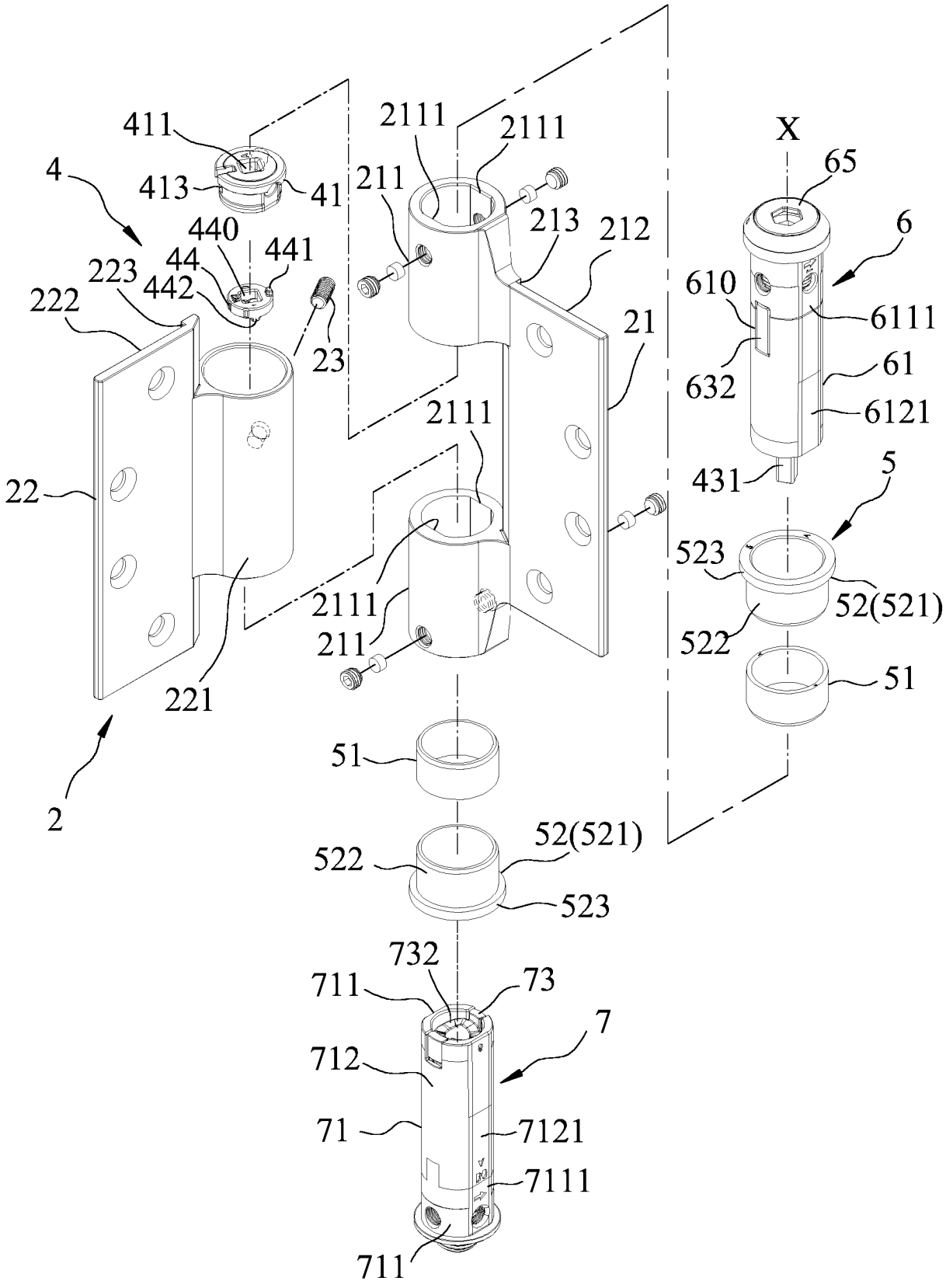



FIG.1
PRIOR ART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2111
211
51
522
52(521)
523
732
711
73
7
712
71
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522
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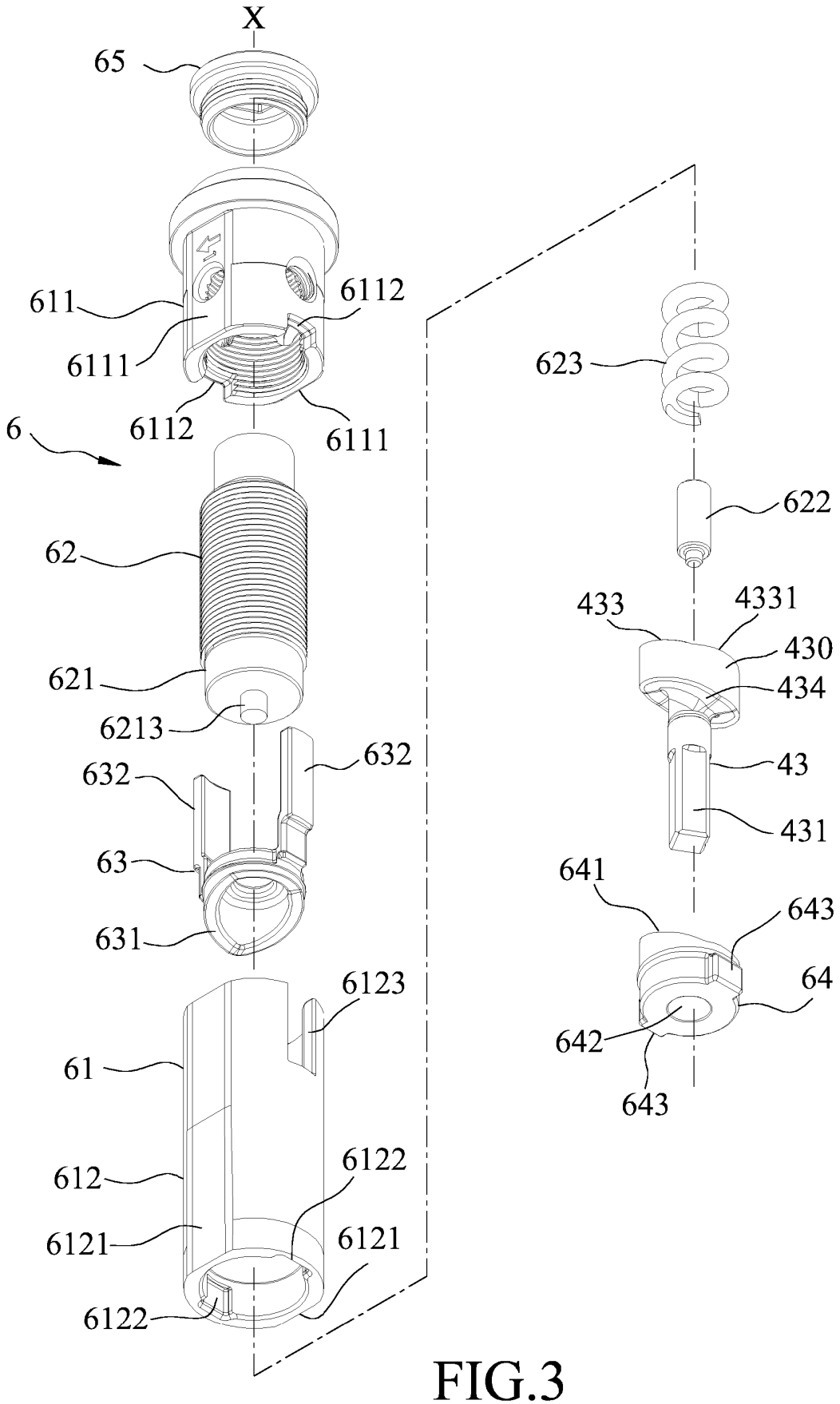
FIG.2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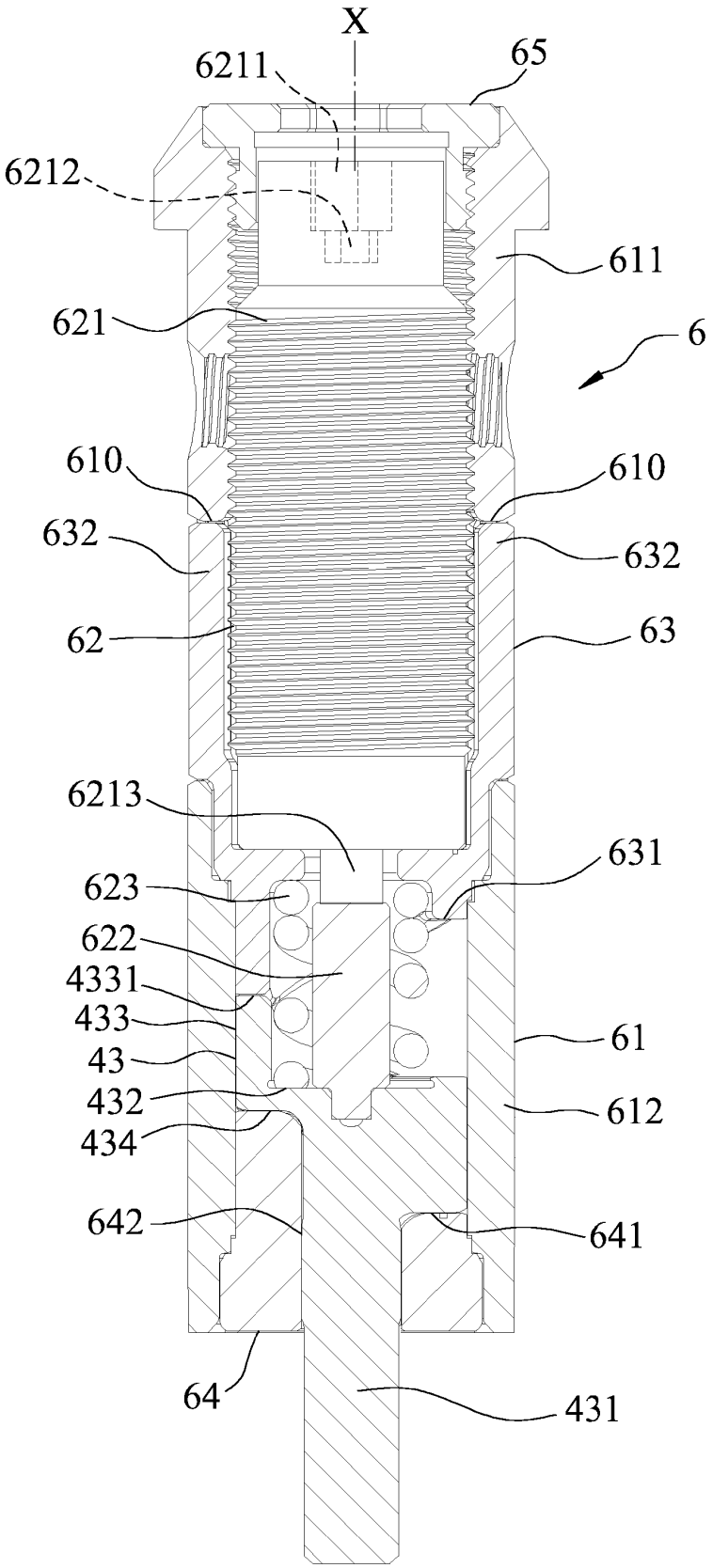
64
642
643

FIG.3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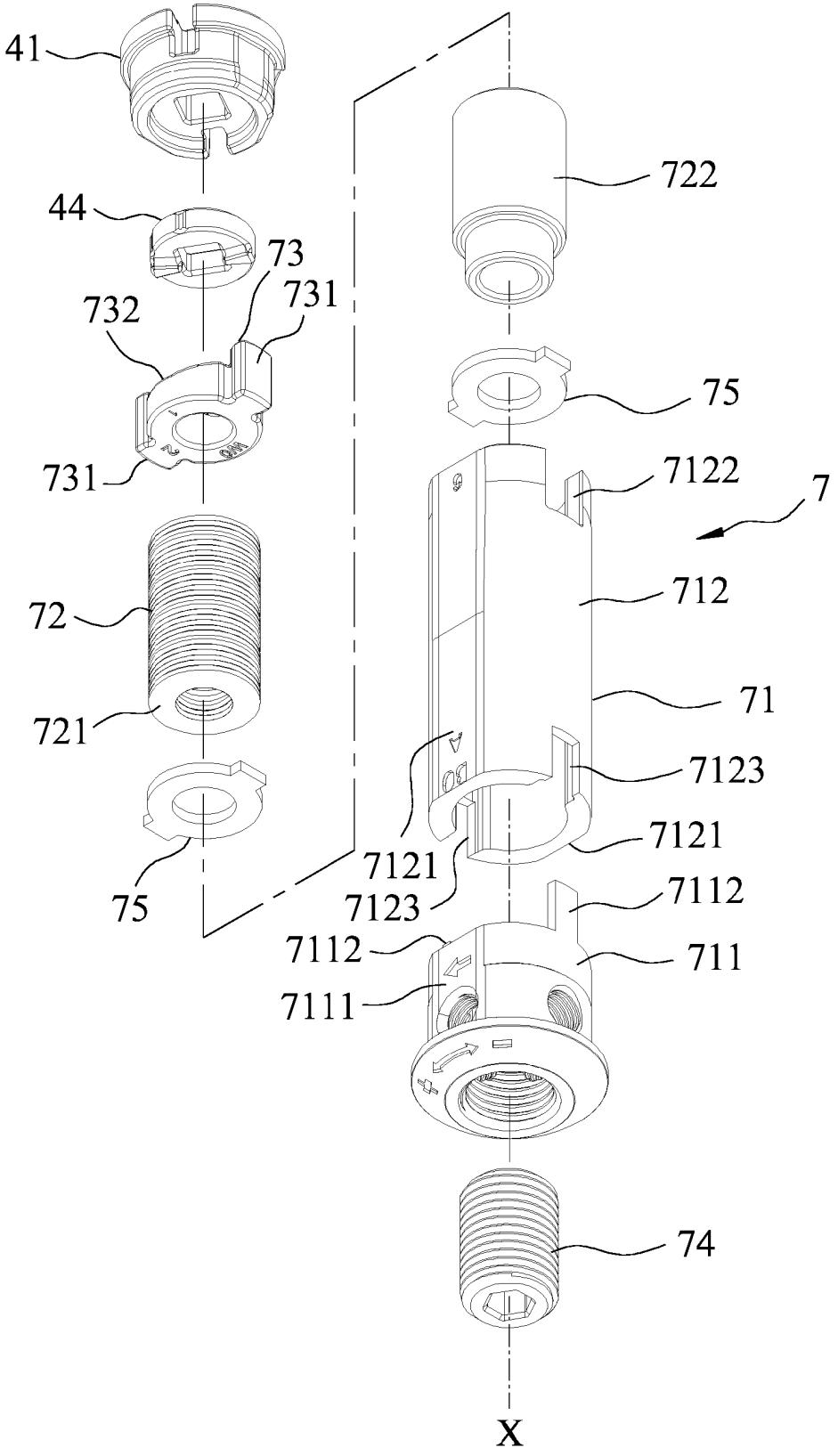
432
434
642
641
64
431

FIG.4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7121
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7112
7112
711
7111
74
X

FIG.5

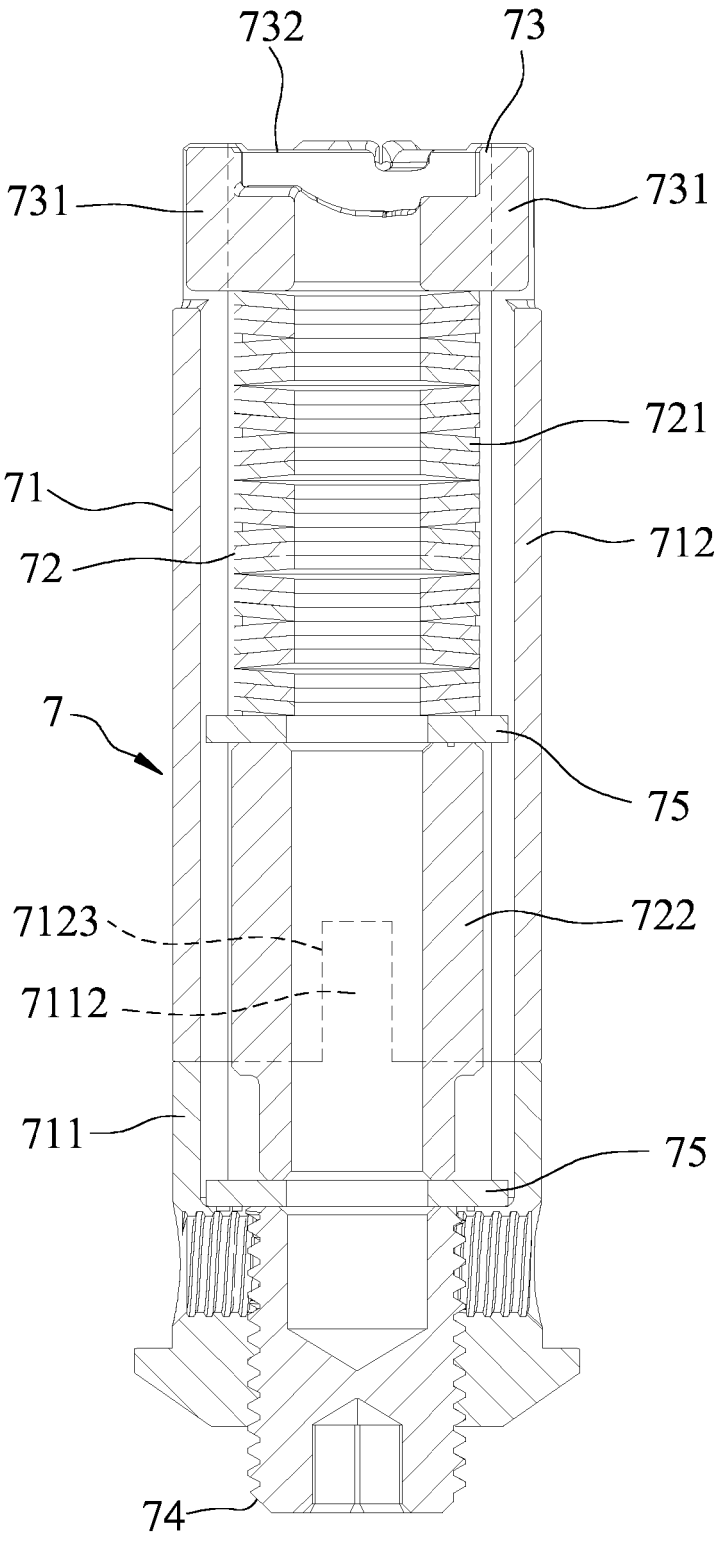



FIG.6

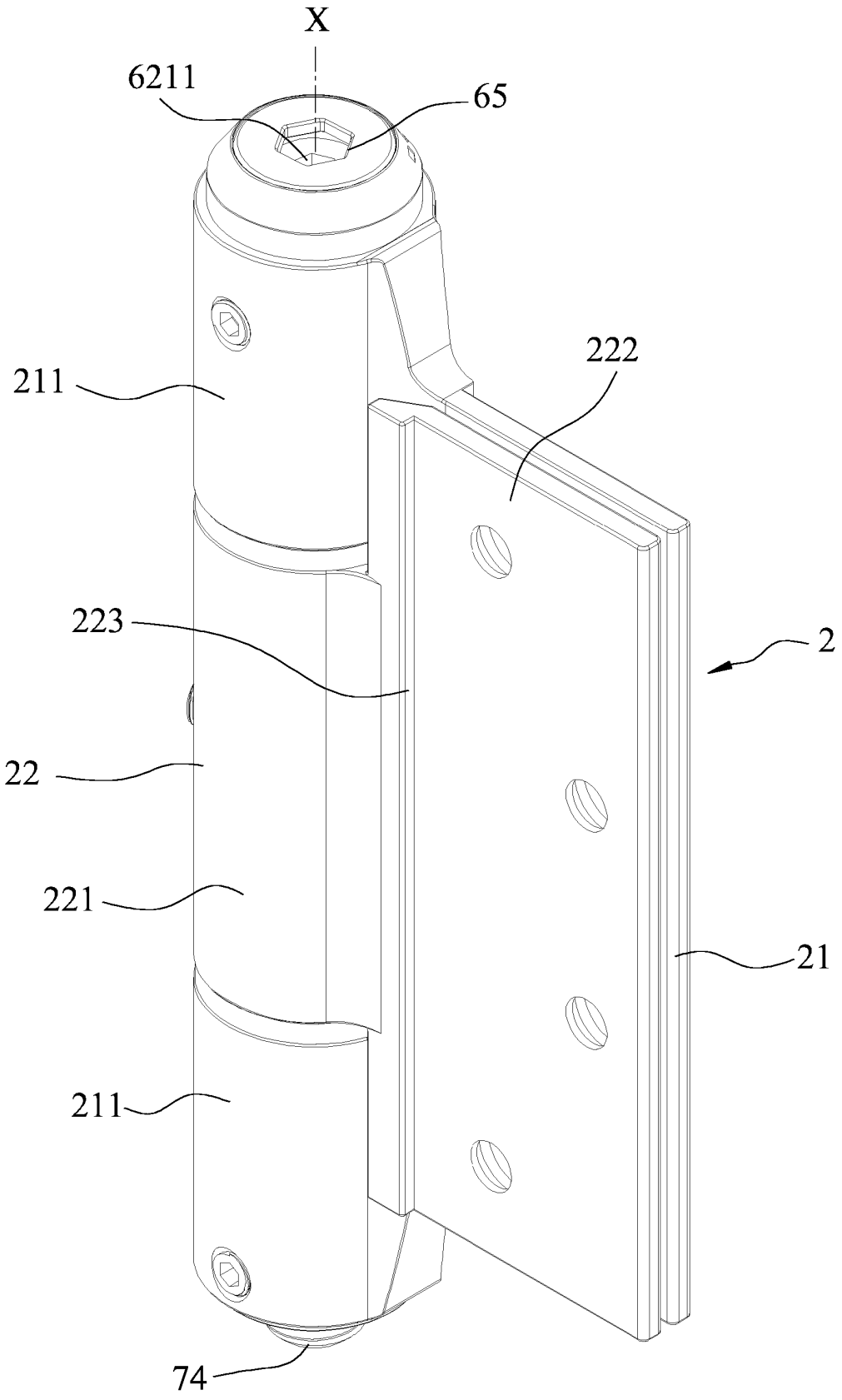
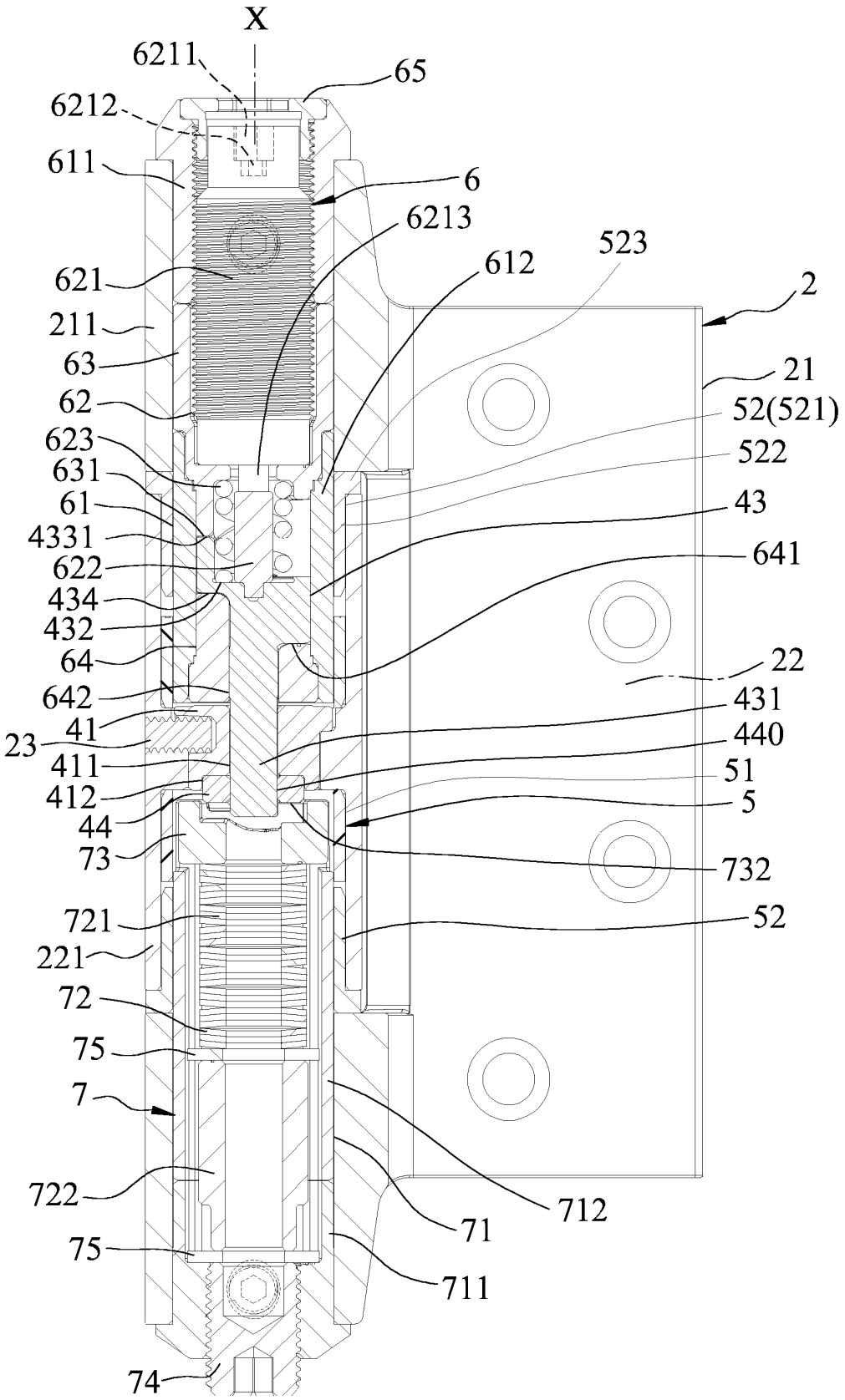



FIG.7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43
61
4331
641
622
434
432
22
64
642
431
41
440
23
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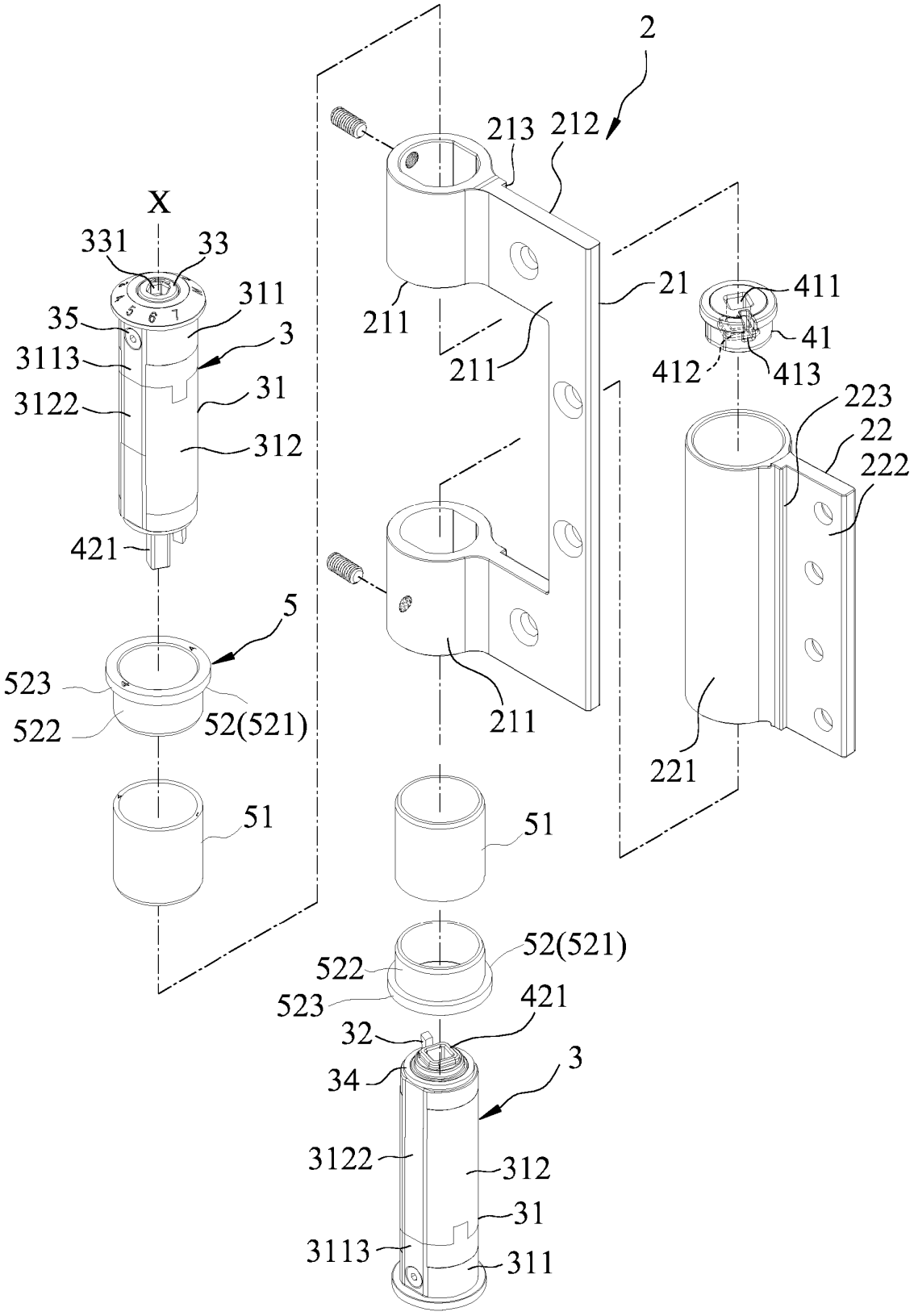
FIG.8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211
21
411
41
412
413
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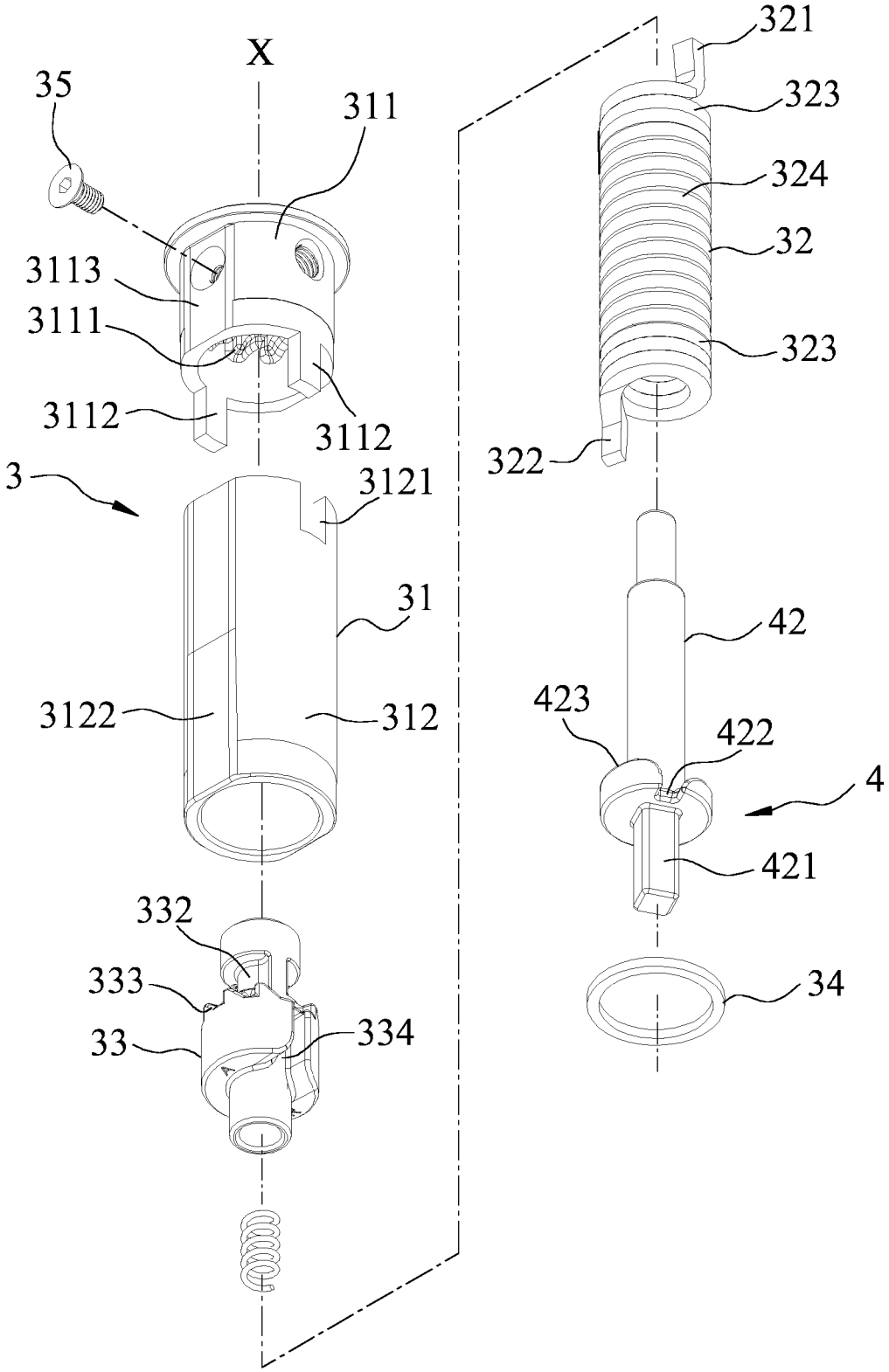
312
31
3113
311

FIG.9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421
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FIG.10

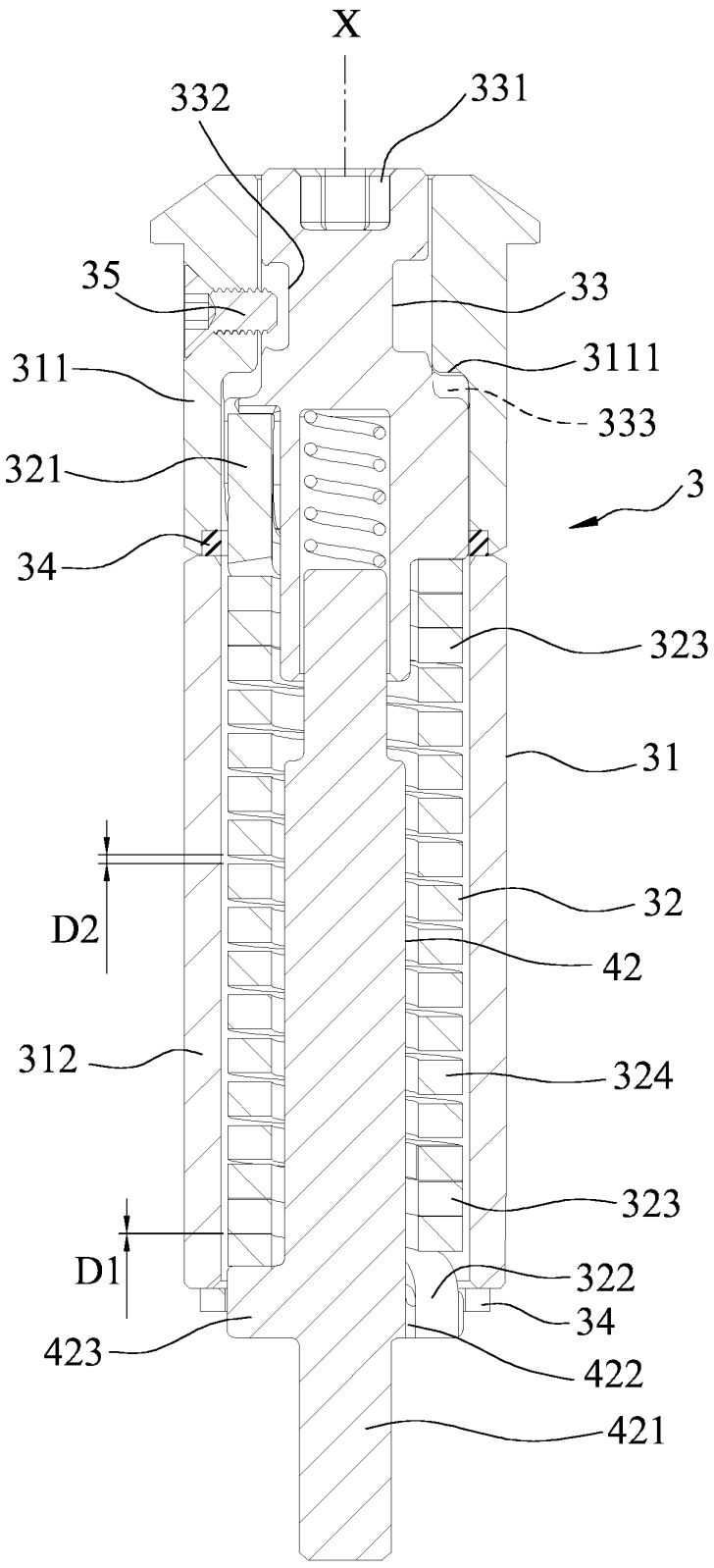
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FIG.11

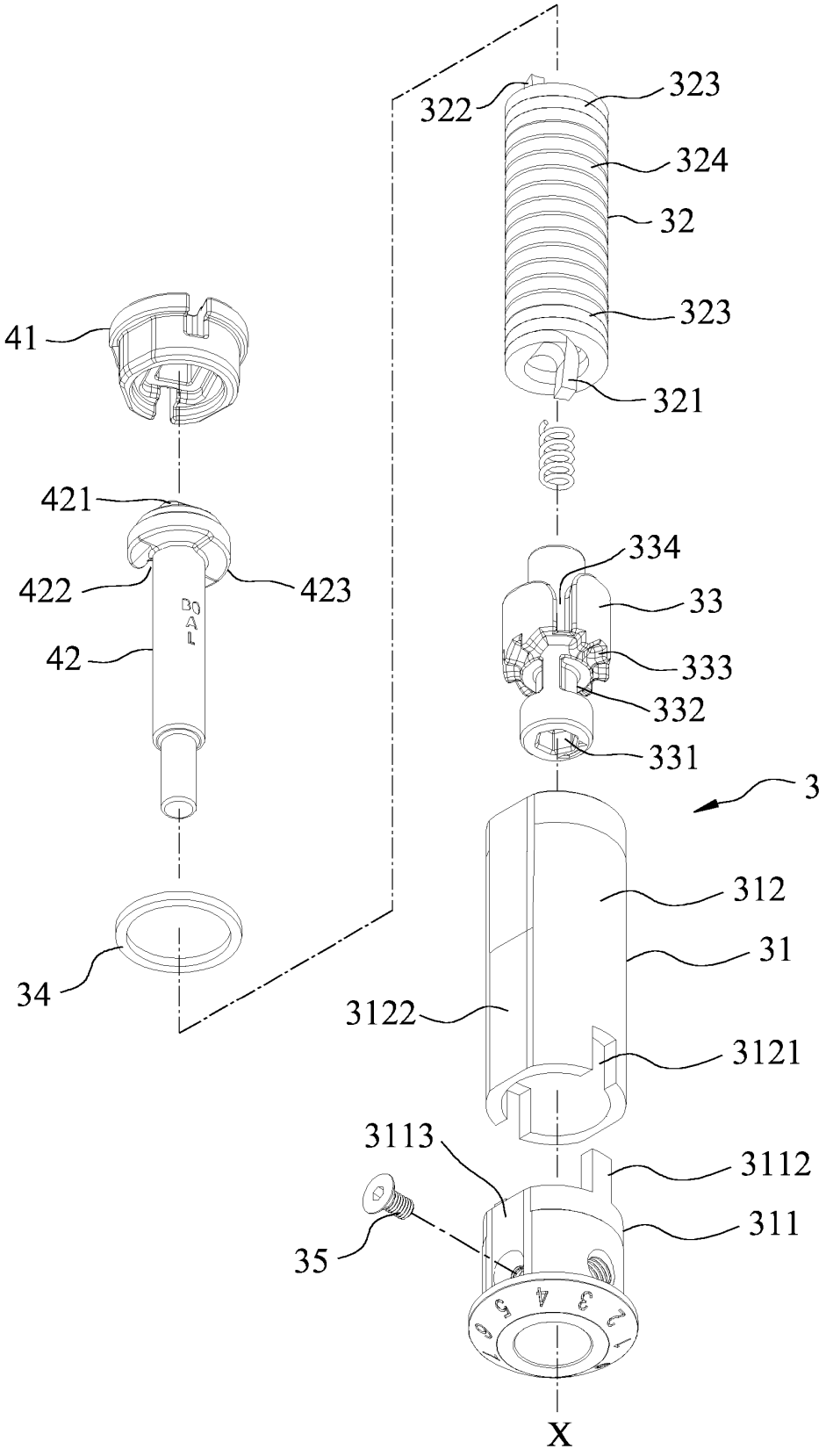
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FIG.12

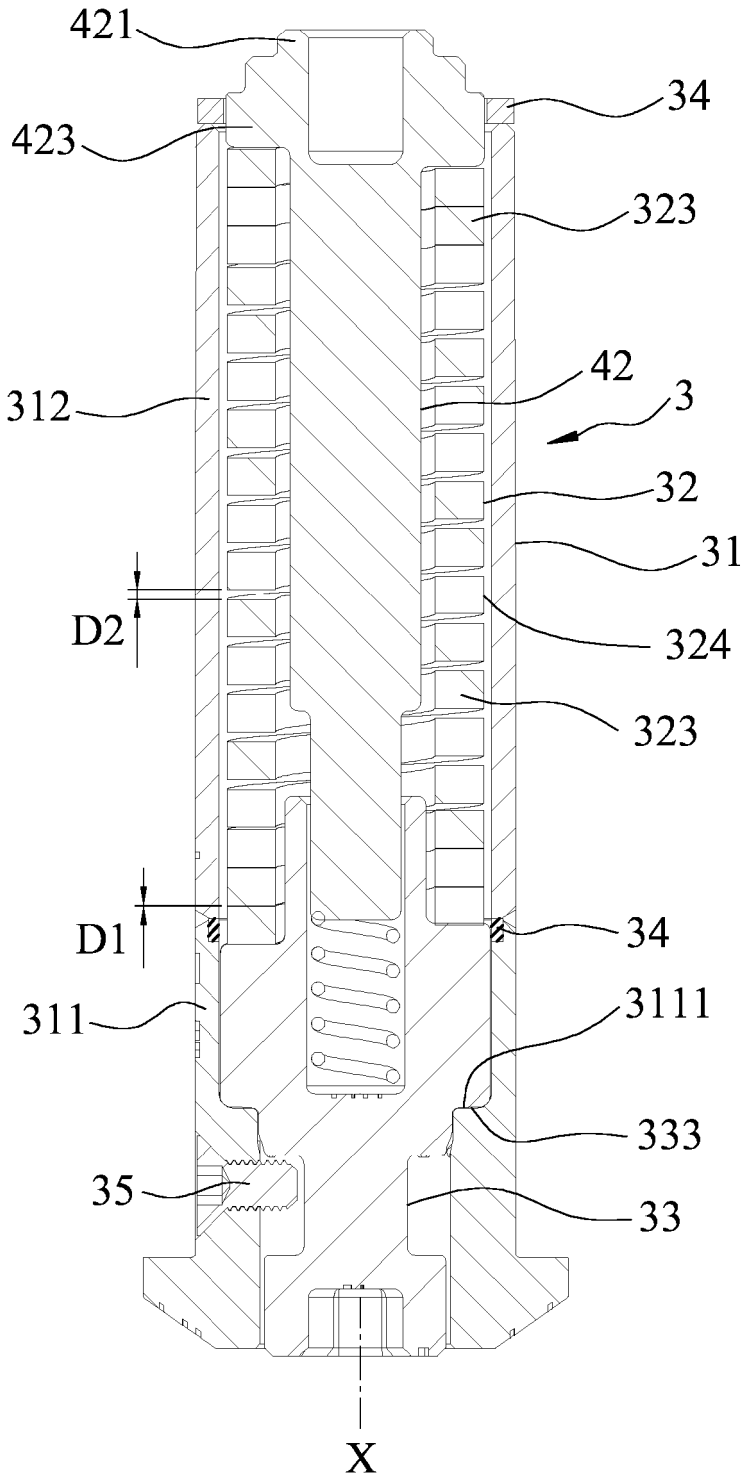


FIG.13

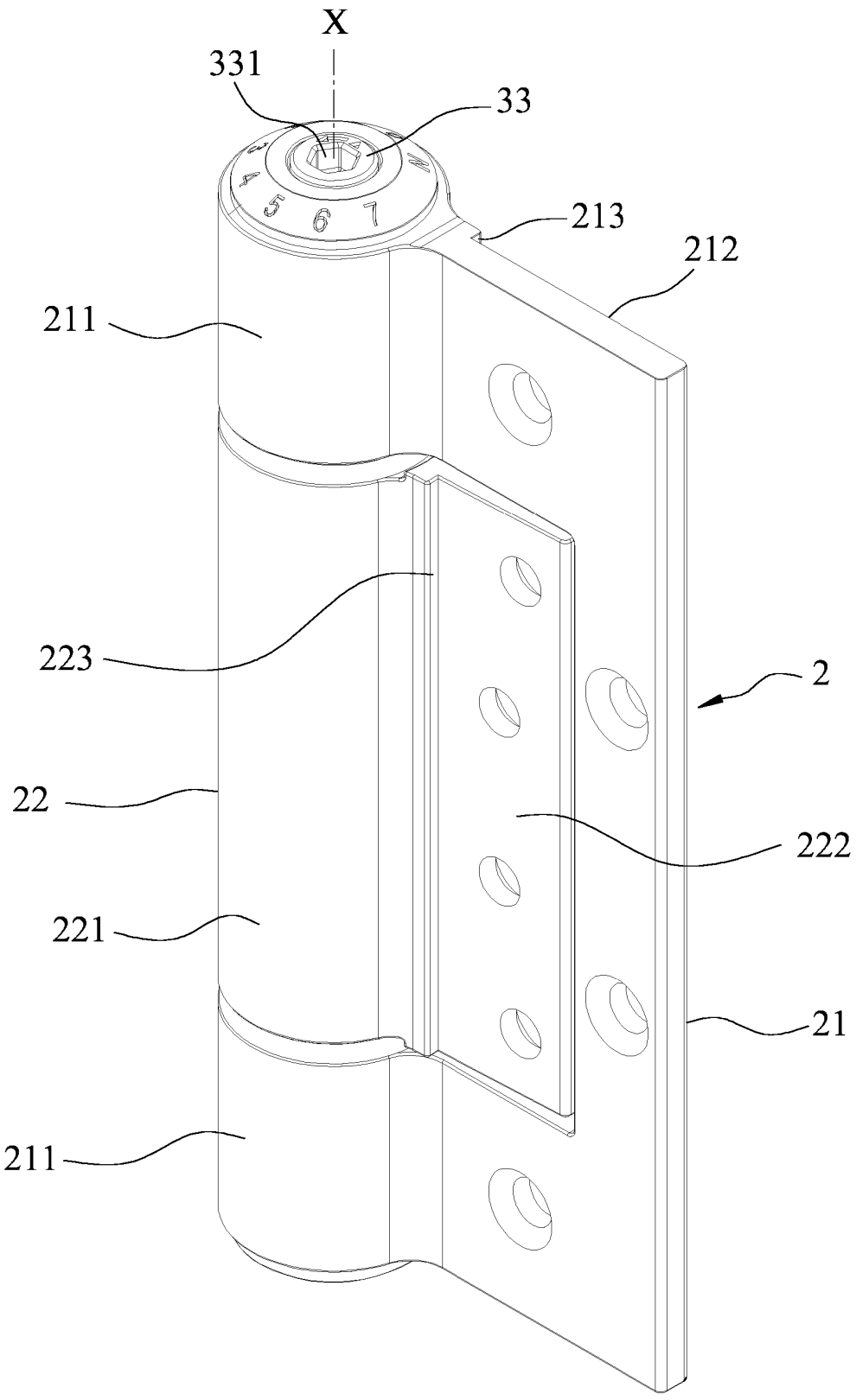


FIG.14

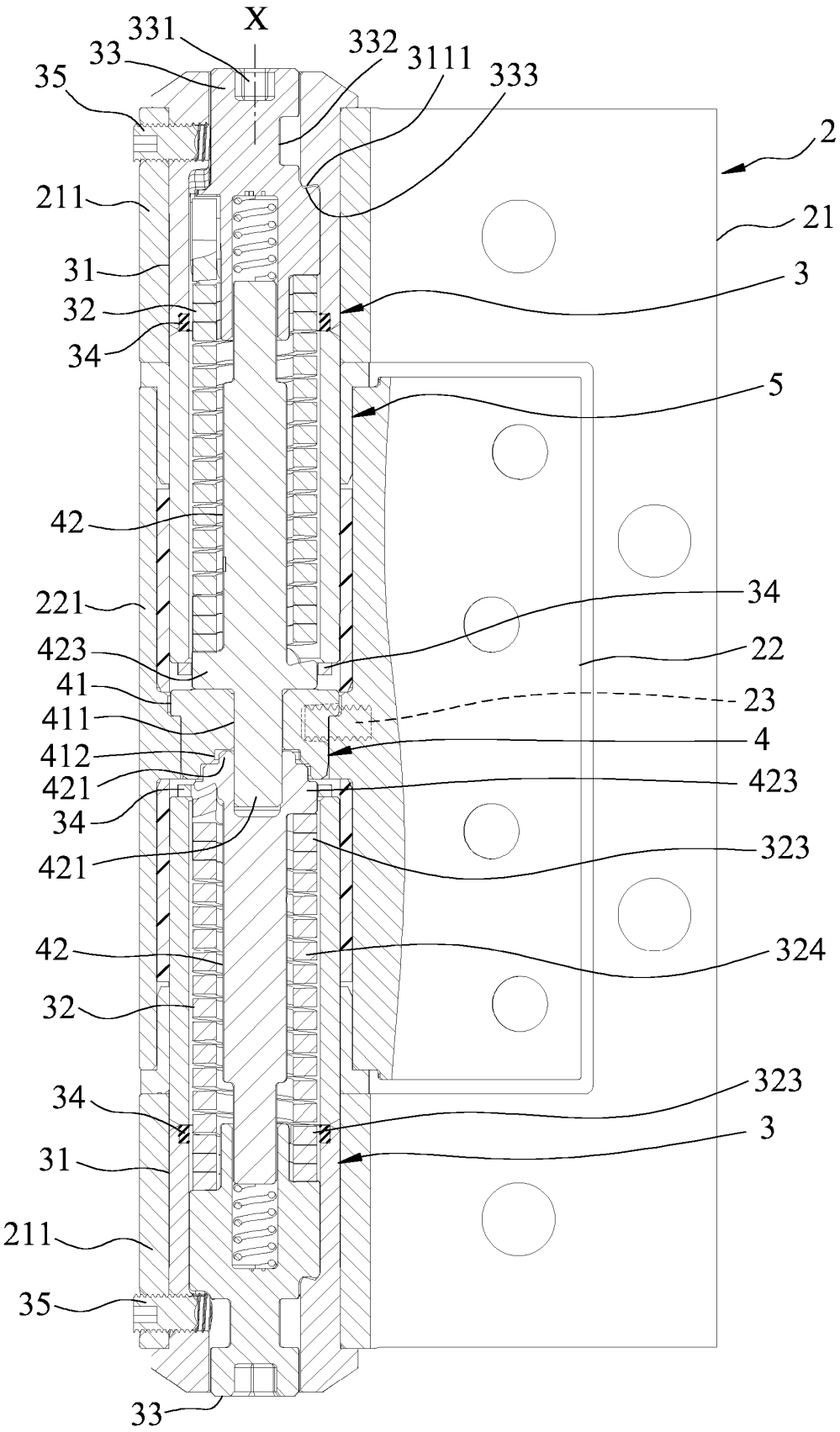
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FIG.15

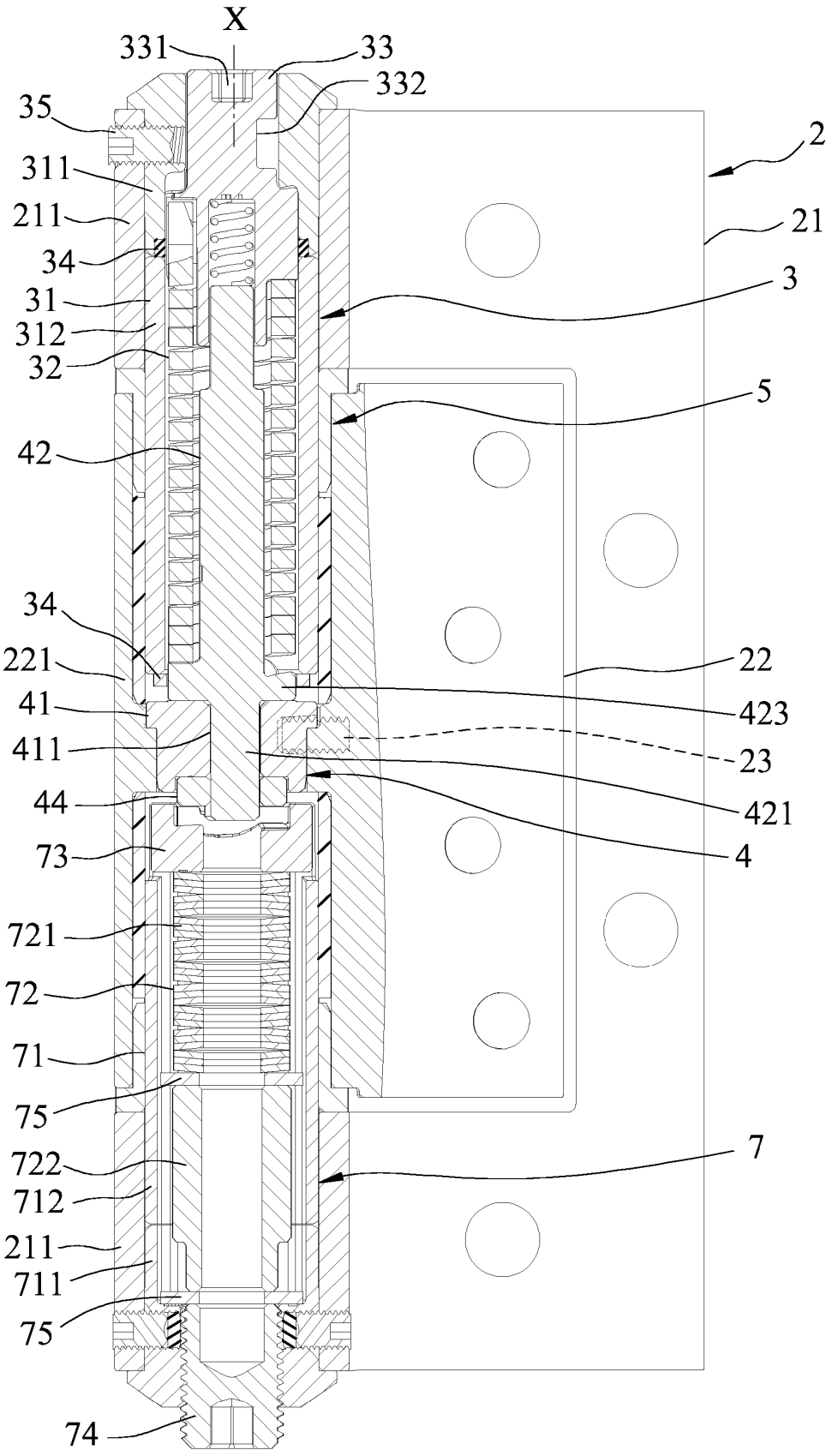
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FIG.16

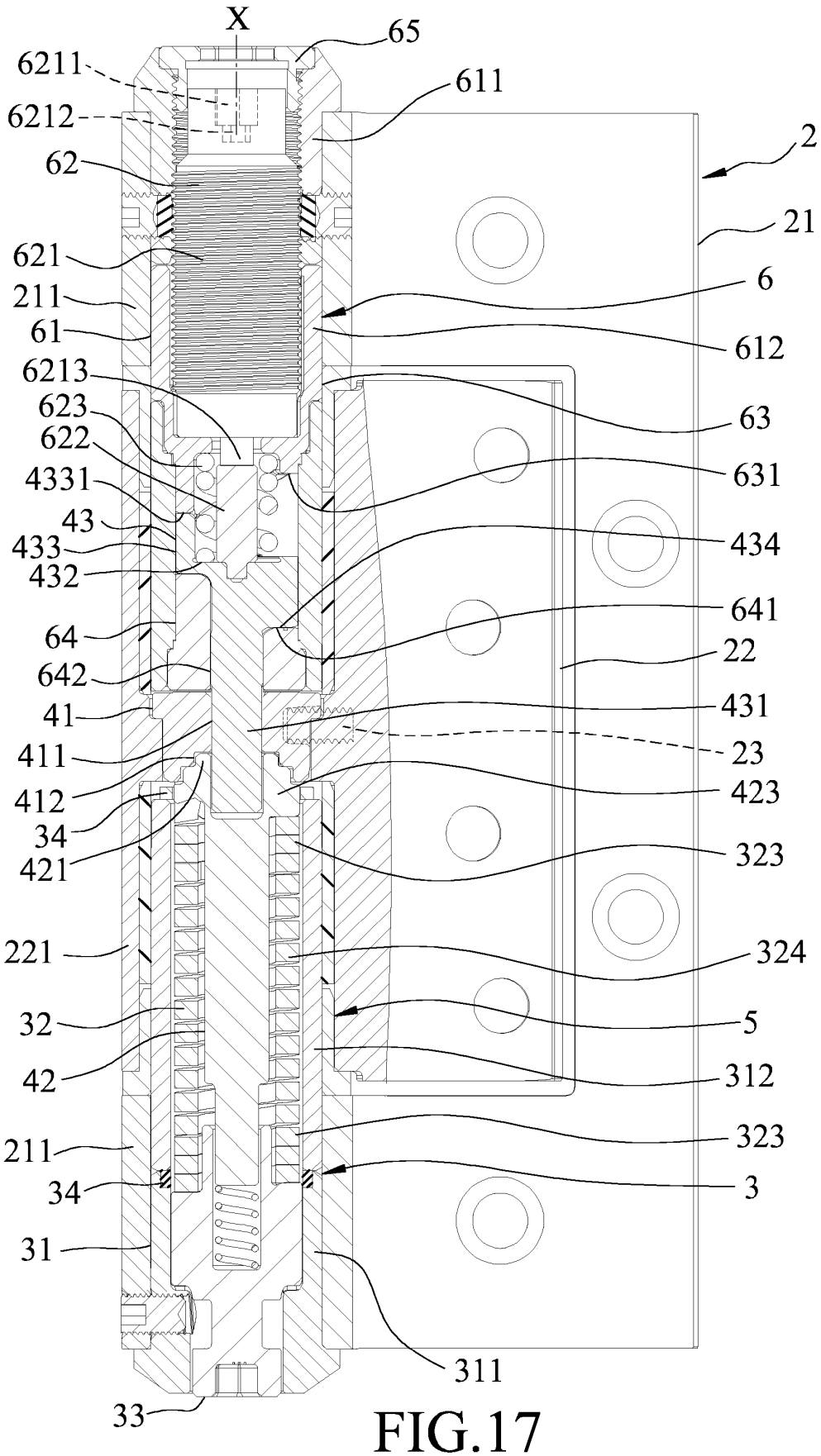
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FIG.17

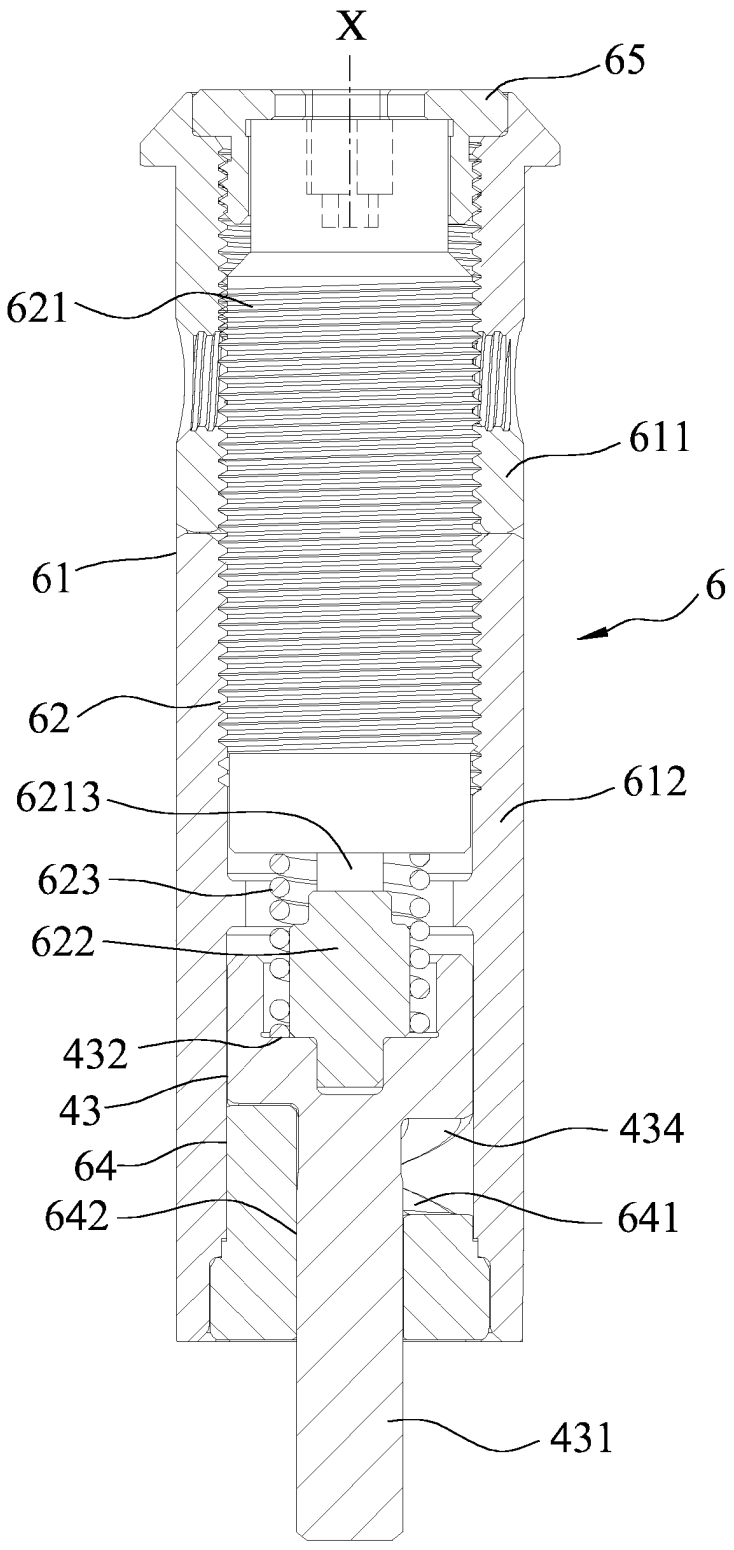


FIG.18

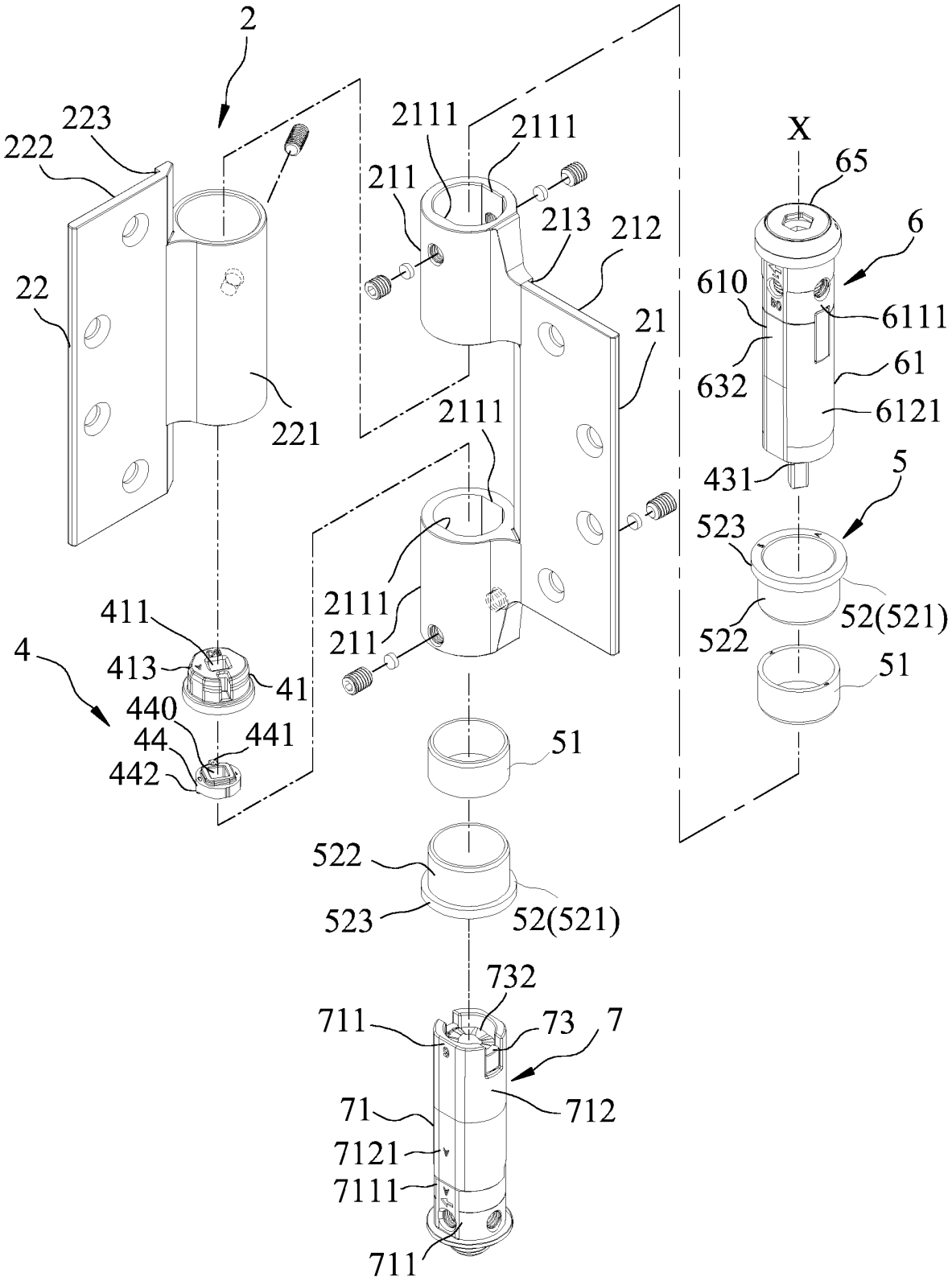
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FIG.19

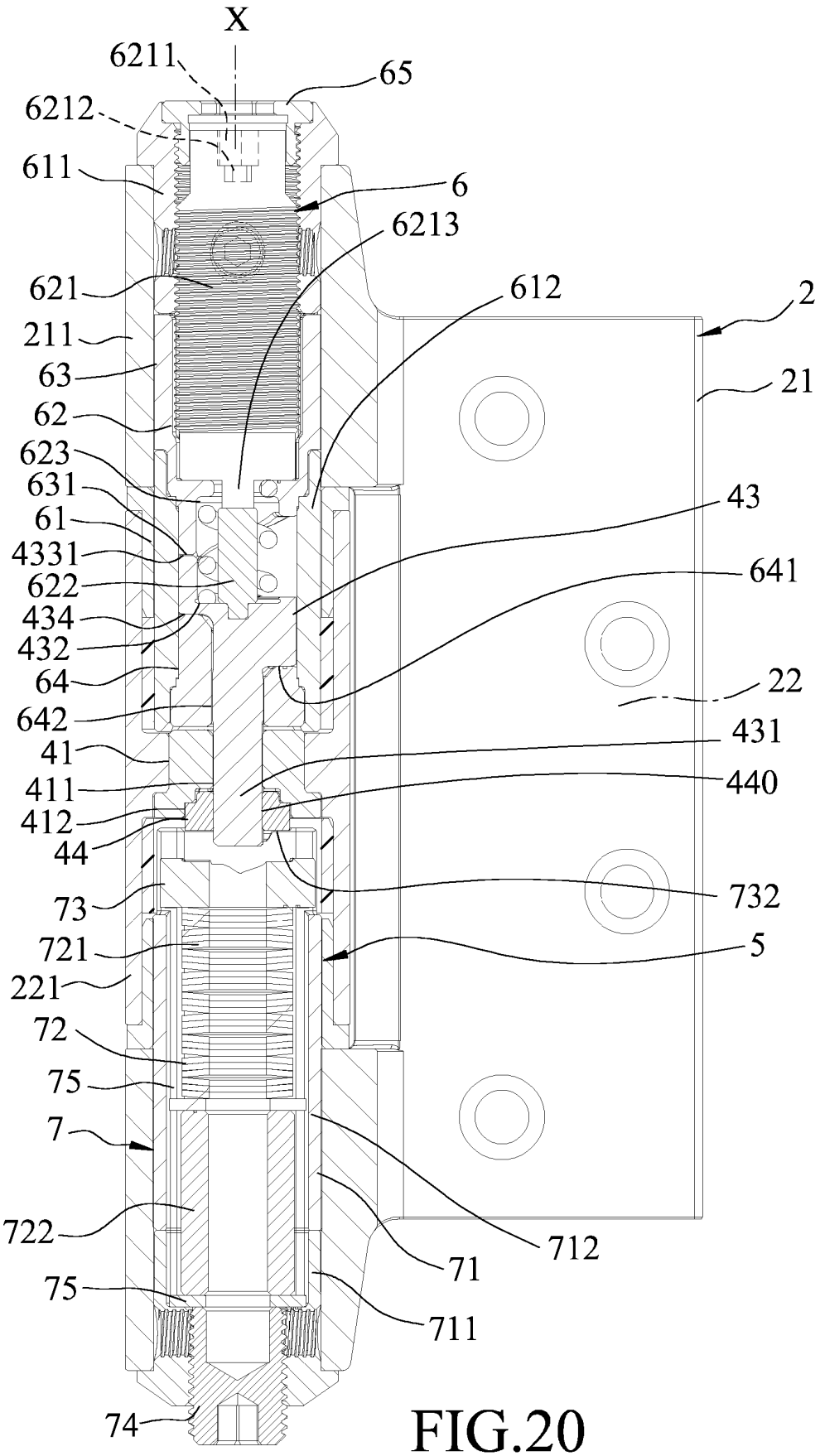
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FIG.20

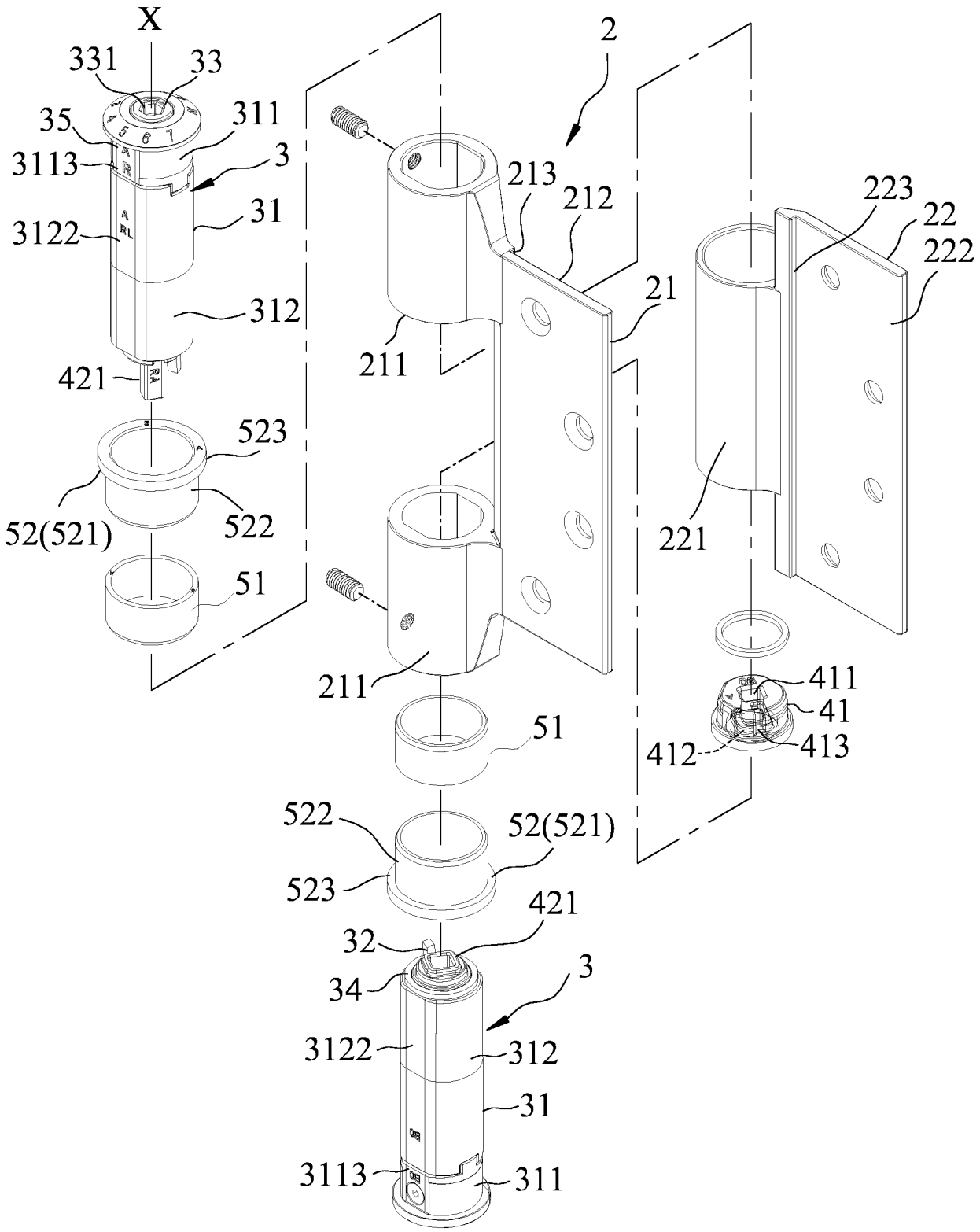
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FIG.21

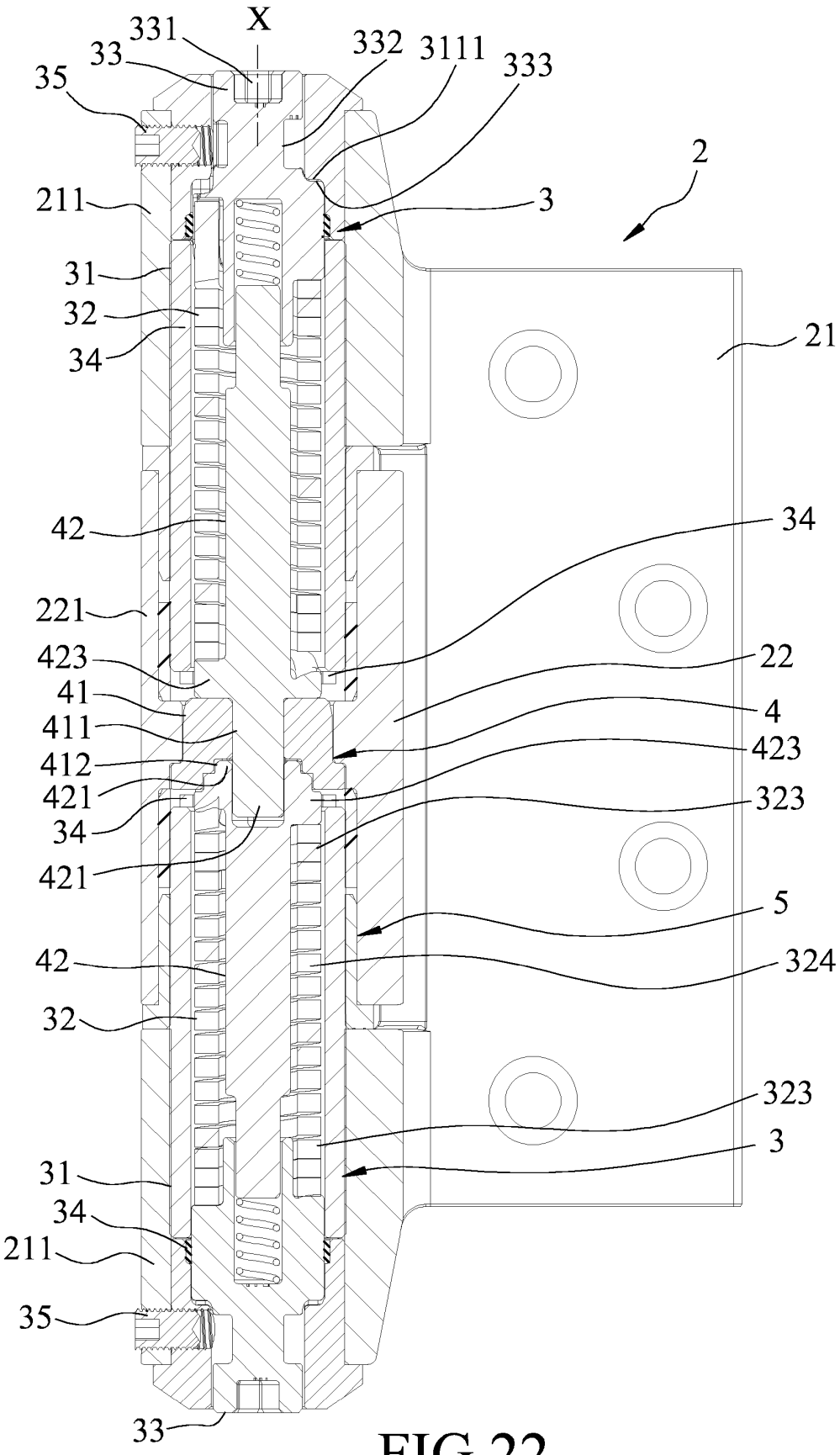
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FIG.22

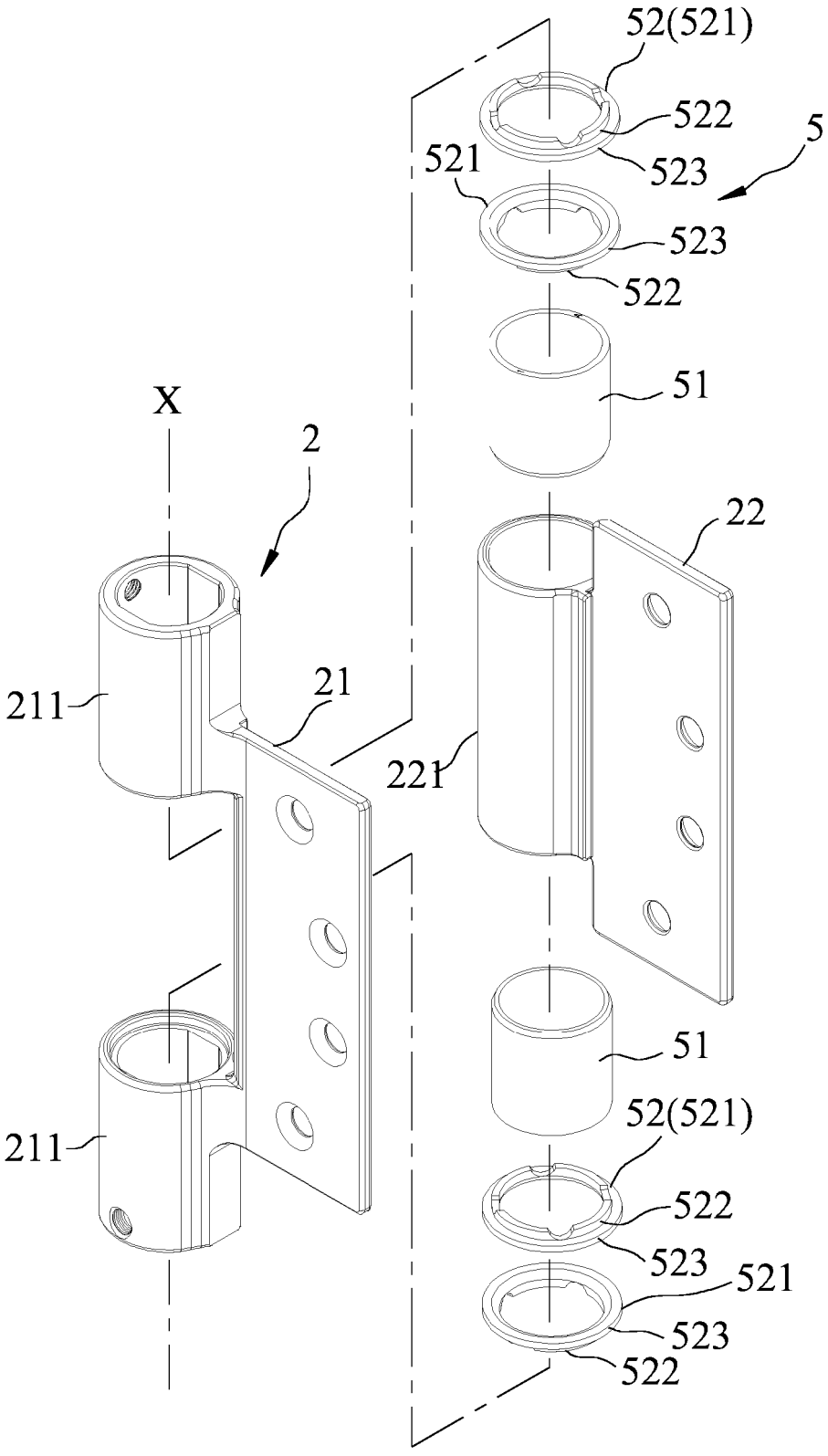
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FIG.23

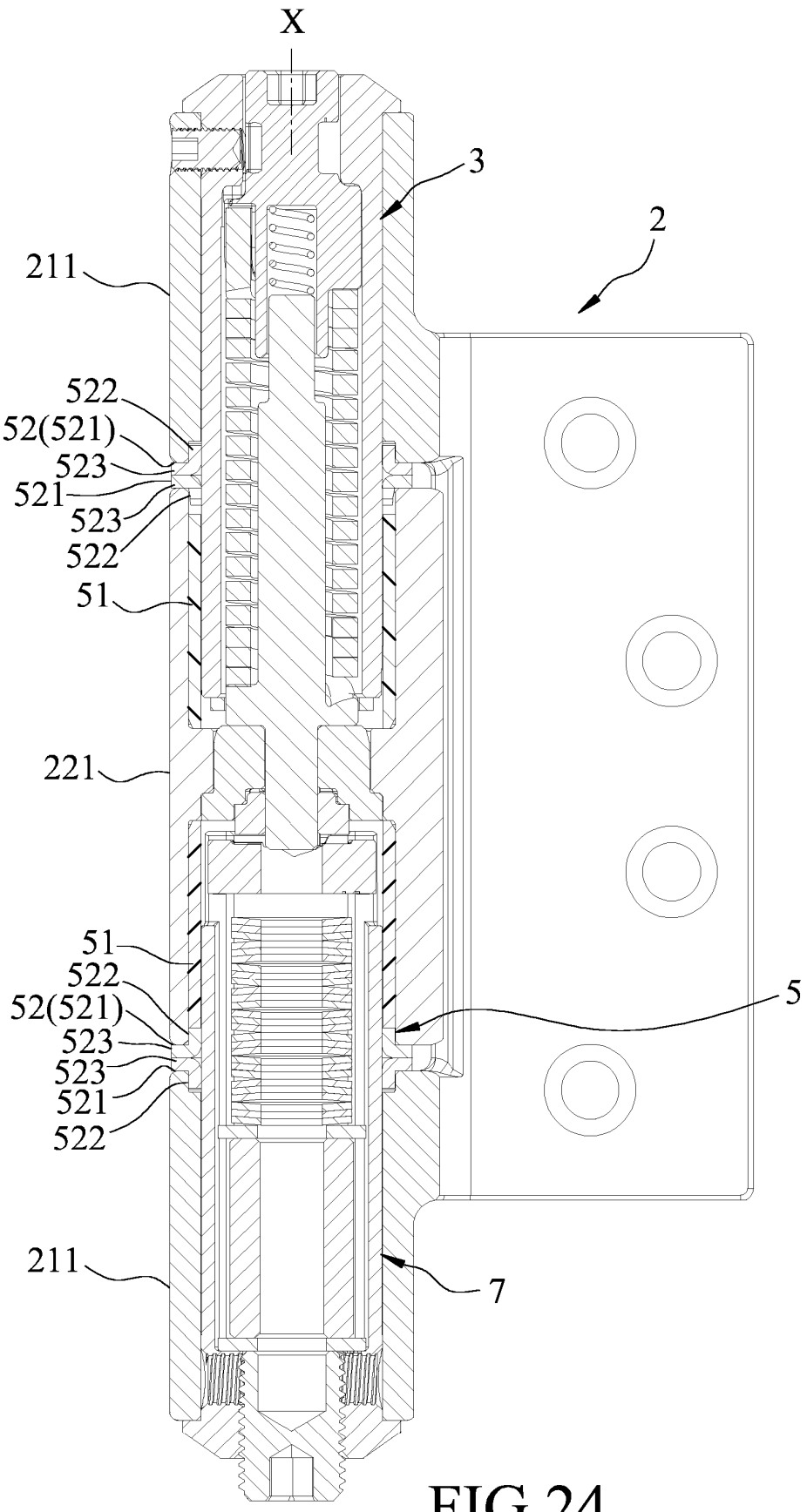
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FIG.24

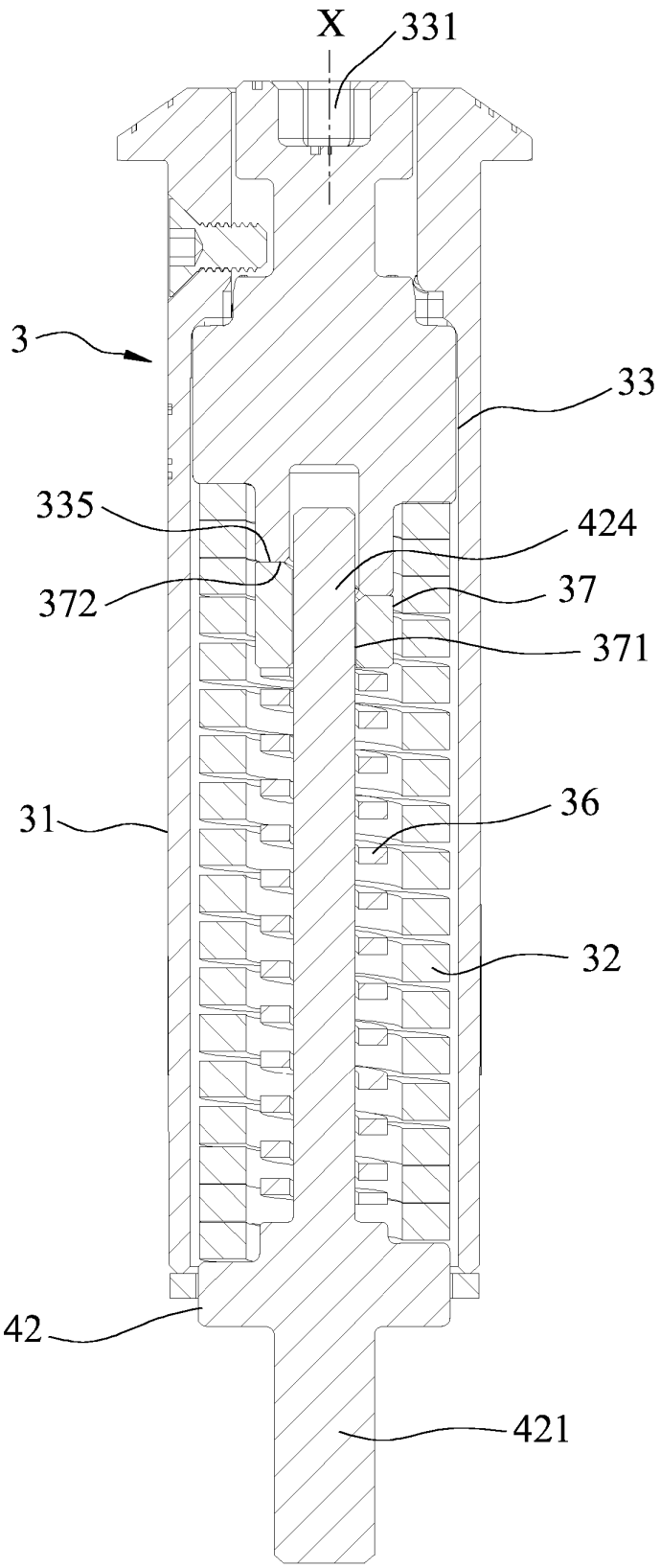
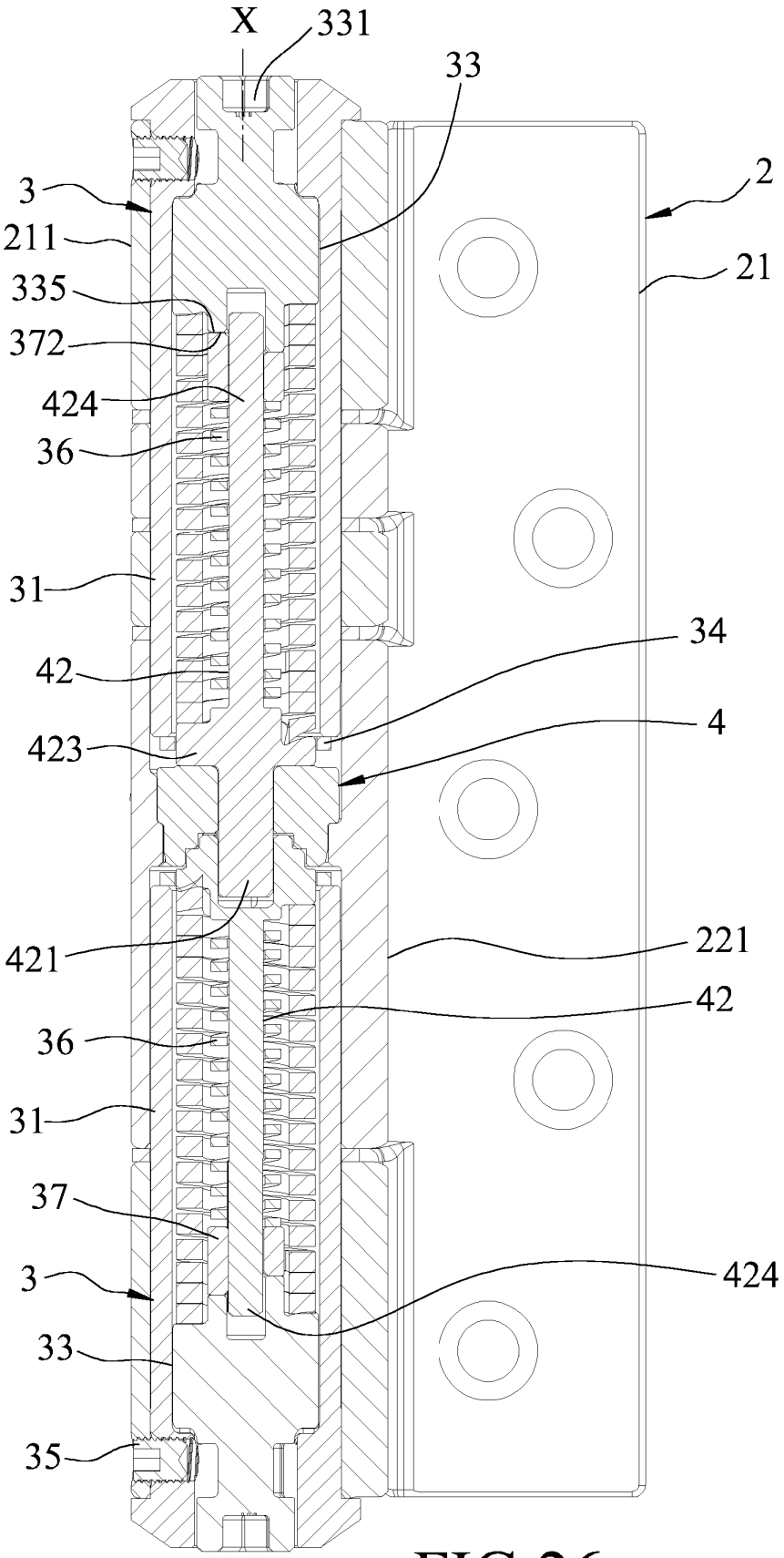



FIG.25


X
331
33
2
3
211
21
335
372
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42
4
423
421
221
42
36
31
37
3
424
33
35

FIG.26

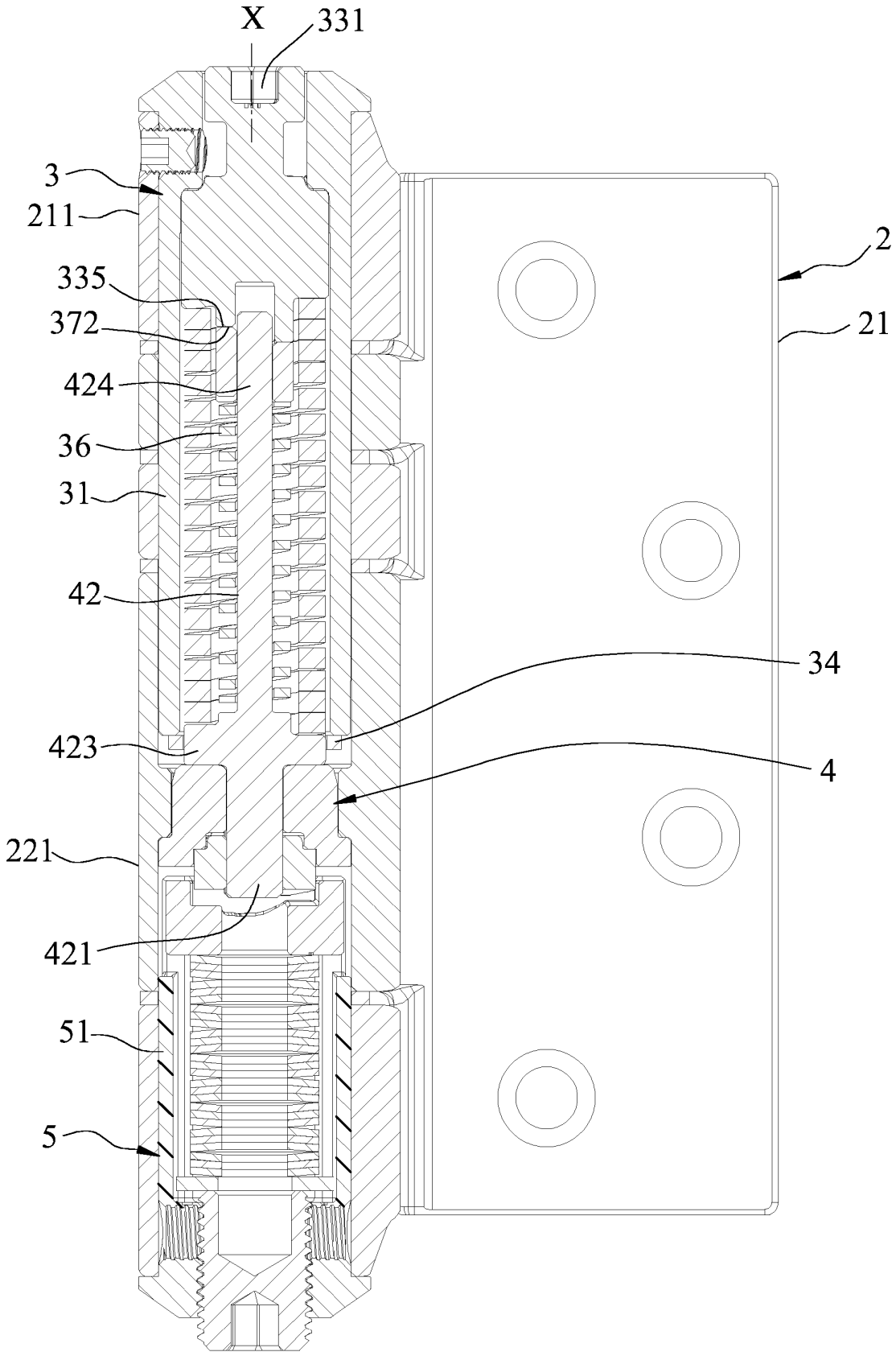


FIG.27

## REFERENCES CITED IN THE DESCRIPTION

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### Patent documents cited in the description


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- EP 3246501 A1 **[0004]**
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