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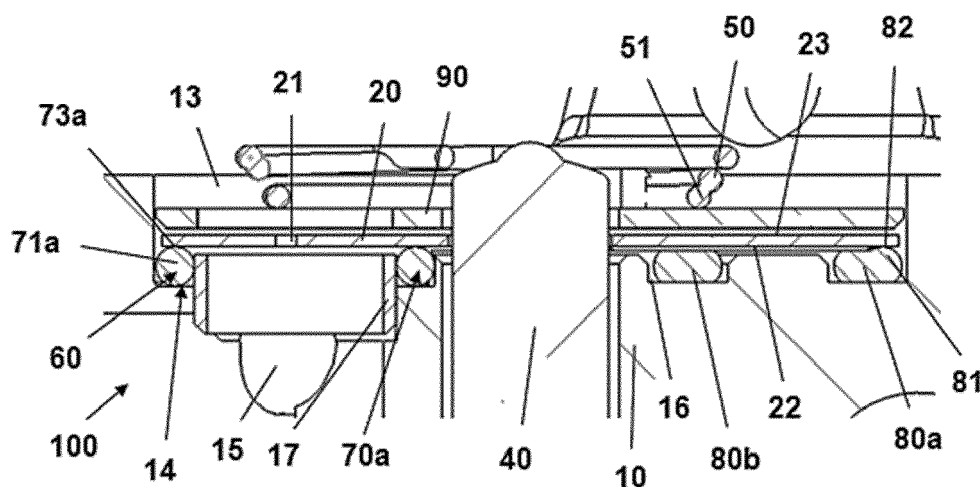
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(54) **REGULATING VALVE FOR A GAS COOKING APPLIANCE, AND GAS COOKING APPLIANCE INCORPORATING SAID REGULATING VALVE**

(57) A regulating valve with at least one gas outlet conduit (12) and a housing (13) comprising at least one outlet hole (15) fluidically communicated with the outlet conduit (12); a rotating disc (20) arranged in the housing (13); a spring (50) axially retaining the rotating disc (20) in the housing (13); and sealing means surrounding the at least one outlet hole (15). The sealing means comprise

a gasket (60) comprising a closure member (70) associated with each outlet hole (15), surrounding same, and at least one attachment arm (80) attached to the at least one closure member (70), comprising at least one protuberance (81), with the rotating disc (20) being supported on the closure members (70) and on the protuberances (81) when it presses on the gasket (60).

**FIG. 3****EP 3 828 467 A1**

Description

TECHNICAL FIELD

[0001] The present invention relates to regulating valves for a gas cooking appliance and to gas cooking appliances incorporating said regulating valves.

PRIOR ART

[0002] Regulating valves for a gas cooking appliance are known.

[0003] Document EP2299156A2 describes a regulating valve for a gas cooking appliance, comprising a valve body comprising an inlet conduit through which gas is supplied, a gas outlet conduit, and a housing fluidically communicated with the inlet conduit and the outlet conduit; a rotating disc arranged in the housing of the valve body on a contact surface of the housing, the contact surface comprising an outlet hole fluidically communicated with the outlet conduit, the rotating disc being suitable for rotating with respect to the contact surface and for regulating gas flow between the housing and the outlet conduit by means of the rotation of the rotating disc; a spring assembly which pushes the rotating disc and retains it axially at one end against the contact surface; and a gasket arranged such that it is housed on the contact surface surrounding the outlet hole, the rotating disc being supported pressing on the gasket. To improve sealing, the spring assembly is arranged on the gasket, said spring assembly comprising a disc tensioning device and a helical spring.

[0004] WO2018/216044A1 describes a regulating valve for a gas cooking appliance, comprising a valve body comprising an inlet conduit through which gas is supplied, a gas outlet conduit, and a housing fluidically communicated with the inlet conduit and the outlet conduit; a rotating disc arranged in the housing of the valve body on a contact surface of the housing, the contact surface comprising an outlet hole fluidically communicated with the outlet conduit, the rotating disc being suitable for rotating with respect to the contact surface and for regulating gas flow between the housing and the outlet conduit by means of the rotation of the rotating disc; a spring which pushes the rotating disc and retains it axially at one end against the contact surface; and a gasket arranged such that it is housed on the contact surface surrounding the outlet hole, the rotating disc being supported pressing on the gasket. The gasket of the regulating valve occupies the entire space of the contact surface of the housing of the body of the valve, except for the openings corresponding to the outlet holes of the corresponding gas, with the rotating disc being supported on said surface of the gasket.

DISCLOSURE OF THE INVENTION

[0005] The object of the invention is to provide a reg-

ulating valve for a gas cooking appliance, and a gas cooking appliance incorporating said regulating valve, as defined in the claims.

[0006] The regulating valve of the invention comprises a valve body comprising an inlet conduit through which gas is supplied, at least one gas outlet conduit, and a housing fluidically communicated with the inlet conduit and the outlet conduit; a rotating disc arranged in the housing of the valve body on a contact surface of the housing, the contact surface comprising at least one outlet hole fluidically communicated with the outlet conduit, the rotating disc being suitable for rotating with respect to the contact surface and for regulating gas flow between the housing and the outlet conduit by means of the rotation of the rotating disc; at least one spring which pushes the rotating disc and retains it axially at one end against the contact surface; and sealing means housed on the contact surface surrounding the outlet hole, the rotating disc being supported pressing on the gasket.

[0007] The sealing means comprise a gasket comprising a closure member associated with each outlet hole, the closure member comprising a body with an opening surrounding the corresponding outlet hole, and at least one attachment arm attached to the closure member, the at least one attachment arm comprising at least one protuberance, with the rotating disc being supported on the closure members and on the protuberances when it presses on the gasket.

[0008] In the regulating valves of the prior art, in order to assure sealing between the rotating disc and the outlet hole, particularly when the regulating valve is in the closed position, the pressure exerted by the spring on the rotating disc in the region above the gasket in the closure contour of the gas outlet hole is increased with devices comprising pressure discs and springs, thereby increasing valve complexity and cost. Alternatively, the surface of the gasket also increases, occupying the entire space of the contact surface of the housing of the body of the valve where the rotating disc is arranged, except for the openings corresponding to the outlet holes, with the rotating disc being supported on said surface of the gasket, requiring in these cases a spring with a greater strength to obtain suitable pressure on the gasket, given that the surface of said gasket has increased considerably, thereby making the regulating valve more expensive.

[0009] In the gasket of the regulating valve of the invention, since the rotating disc is supported pressing on the gasket, only on the closure member surrounding the gas outlet hole and on the protuberances of the attachment arms, the pressure on the gasket in the closure contour of the outlet hole increases, given that the support surface of the rotating disc on the gasket has decreased considerably. This allows the pressure for supporting the rotating disc on the gasket to increase for one and the same spring, thereby preventing tensioning discs with specific additional springs, and assuring the sealing of the regulating valve in any rotation position of the ro-

tating disc, and particularly in the closed position of the regulating valve. Also for one and the same pressure as the one used in the prior art on the gasket, a spring with a lower strength, and therefore a lower cost, can be used. It also allows reducing the surface of the gasket, given that the gasket of the regulating valve of the invention comprises at least one attachment arm attached to the closure member, instead of occupying the entire space of the contact surface of the housing of the body of the regulating valve, whereby a reduction in the cost of the gasket, and therefore of the regulating valve, is also obtained.

[0010] Other advantages of the regulating valve of the invention is that with the support surface of the gasket formed by the closure member and the protuberance of the attachment arm, equilibrium is maintained in the support of the rotating disc on the gasket, and furthermore the contact surface between the rotating disc and the gasket is minimized, such that when the rotating disc is rotated for regulating gas flow to the gas outlet conduit, friction, and therefore abrasion, between the rotating disc and the gasket is minimized, obtaining a longer service life of said gasket.

[0011] These and other advantages and features of the invention will become evident in view of the drawings and detailed description of the invention.

DESCRIPTION OF THE DRAWINGS

[0012]

Figure 1 shows a perspective view of an embodiment of the regulating valve of the invention with a safety valve, a gas outlet conduit, and a bypass.

Figure 2 shows a vertical longitudinal section view of the regulating valve of Figure 1.

Figure 3 shows a detailed section view of the housing of the body of the regulating valve with the gasket of Figure 1.

Figure 4 shows a detailed perspective view of the housing of the body of the regulating valve of Figure 1.

Figure 5 shows a perspective view of the gasket of the regulating valve of Figure 1.

Figure 6 shows a perspective view of the rotating disc of the regulating valve of Figure 1.

Figure 7 shows a perspective view of the pushing plate of the regulating valve of Figure 1.

Figure 8 shows a detailed perspective view of the housing of the body of the regulating valve of Figure 1 with the gasket, the rotating disc, the pushing plate,

and the spring assembled in said housing.

Figure 9A shows a detailed section view of the housing of the body of the regulating valve of Figure 1, with a closure member of the gasket arranged surrounding an outlet hole, the regulating valve being in a closed gas flow position.

Figure 9B shows a detailed section view of the housing of the body of the regulating valve of Figure 1, with a closure member of the gasket arranged surrounding an outlet hole, the regulating valve being in an open gas flow position.

Figure 10 shows a perspective view of a second embodiment of the regulating valve of the invention with a safety valve and two gas outlet conduits.

Figure 11 shows another perspective view of the regulating valve of Figure 10.

Figure 12 shows a detailed perspective view of the housing of the body of the regulating valve of Figure 10.

Figure 13 shows a perspective view of the gasket of the regulating valve of Figure 10.

Figure 14 shows a perspective view of the rotating disc of the regulating valve of Figure 10.

Figure 15 shows a perspective view of the pushing plate of the regulating valve of Figure 10.

Figure 16 shows a detailed section view of the housing of the body of the regulating valve of Figure 10 with the gasket.

Figure 17 shows another detailed section view of the housing of the body of the regulating valve of Figure 10 with the gasket.

Figure 18 shows a perspective view of a third embodiment of the regulating valve of the invention with a safety valve and three gas outlet conduits.

Figure 19 shows another perspective view of the regulating valve of Figure 18.

Figure 20 shows a detailed perspective view of the housing of the body of the regulating valve of Figure 18.

Figure 21 shows a perspective view of the gasket of the regulating valve of Figure 18.

Figure 22 shows a perspective view of the rotating disc of the regulating valve of Figure 18.

Figure 23 shows a perspective view of the pushing plate of the regulating valve of Figure 18.

Figure 24 shows a detailed section view of the housing of the body of the regulating valve of Figure 18 with the gasket.

Figure 25 shows another detailed section view of the housing of the body of the regulating valve of Figure 18 with the gasket.

Figure 26 shows a detailed section view of the gasket of the regulating valve of Figure 18 with a protuberance of the attachment arm and the closure protuberance and the lower protuberance of the body of a closure member.

Figure 27 shows a schematic view of an embodiment of the gas cooking appliance according to the invention.

DETAILED DISCLOSURE OF THE INVENTION

[0013] Figures 1 to 9B show an embodiment of the regulating valve 100 for a gas cooking appliance according to the invention, comprising a safety valve 30, a gas outlet conduit 12, and a minimum gas flow calibration bypass for changing the type of gas, which can be regulated from the outside of the regulating valve 100 by regulating means 5.

[0014] The regulating valve 100 comprises a valve body 10 comprising a gas inlet conduit 11 through which gas is supplied, a gas outlet conduit 12 suitable for conducting incoming gas to a burner (not shown in the drawings), and a housing 13 of the valve body 10 fluidically communicated with the inlet conduit 11 and the outlet conduit 12. The regulating valve 100 also comprises a rotating disc 20 arranged in the housing 13 of the valve body 10 on a contact surface 14 of the housing 13, the contact surface 14 comprising two outlet holes 15a and 15b fluidically communicated with the outlet conduit 12, the rotating disc 20 being suitable for rotating with respect to the contact surface 14 and for regulating gas flow between the housing 13 and the outlet conduit 12 by means of the rotation of the rotating disc 20. The regulating valve 100 also comprises a spring 50 which pushes the rotating disc 20 and retains it axially against the contact surface 14, and sealing means housed on the contact surface 14, which is surrounding the outlet holes 15a and 15b, the rotating disc 20 being supported pressing on the gasket 60.

[0015] The sealing means comprise a gasket 60 comprising two closure members 70a and 70b associated respectively with each outlet hole 15a and 15b, each closure member 70a and 70b comprising a respective body 71a and 71b surrounding the outlet hole 15a and 15b, each body 71a and 71b comprising respectively a respective opening 72a and 72b which allows the passage

of gas flow to the outlet holes 15a and 15b. The gasket 60 also comprises two attachment arms 80a and 80b radially attaching the two closure members 70a and 70b, the radius of attachment arm 80a being greater than the radius of attachment arm 80b. The attachment arm 80a comprises two protuberances 81, such that the rotating disc 20 is supported on a support surface 73a and 73b, respectively, of the bodies 71a and 71b of the closure members 70a and 70b, arranged in the upper part of said bodies 71a and 71b, and on an upper surface 82 of the protuberances 81, pressing on the gasket 60.

[0016] In this manner, the gasket 60 in this embodiment of the regulating valve 100 has a circular shape, with the two attachment arms 80a and 80b surrounding the center of the gasket 60. In this embodiment of the regulating valve 100, the attachment arm 80a extends radially to the edge of the contact surface 14 in the housing 13, such that it confers stability to the seating of said gasket 60 in the housing 13 of the valve body 10. In turn, the attachment arm 80b closely surrounds the drive shaft 40, the combination of the attachment arms 80a and 80b improving the stability of the gasket 60. The protuberances 81 are arranged on the surface of the attachment arm 80a which contacts the rotating disc 20, and radially arranged on the outer edge of said surface, said protuberances 81 being distributed in an approximate angle of 120° with respect to one another, with a center in the center of the gasket 60.

[0017] In this manner, the pressure on the gasket 60 in the closure contour of the outlet holes 15a and 15b increases, given that the support surface of the rotating disc 20 on the gasket 60 has decreased considerably with respect to the regulating valves comprising a gasket which occupies the entire contact surface of the housing of the body of the valve, given that the rotating disc 20 is only supported on the closure members 70a and 70b of the gasket 60 and on the two protuberances 81 of the attachment arm 80a. This allows, using the same spring as in a regulating valve of the prior art, with the same dimensional and gas flow characteristics, an increase in the pressure for supporting the rotating disc on the gasket, since the rotating disc is supported on a smaller surface of the gasket. Therefore, the size of the spring, and therefore the strength of said spring, can be reduced, in addition to being able to use a gasket having a smaller surface, and therefore a lower cost.

[0018] Furthermore, with respect to the regulating valves of the prior art which comprise tensioning discs with specific additional springs arranged on the rotating disc, in the support region on the gasket corresponding to the outlet hole, at least the tensioning disc can be eliminated due to the greater pressure exerted on the gasket, the overall cost being lower, despite the size of the gasket increasing to a certain extent.

[0019] The solution defined in the regulating valve of the invention allows assuring the sealing of the regulating valve in any position of rotation of the rotating disc, and particularly in the closed position of the regulating valve,

due to the increase in pressure exerted on the gasket for the same size of spring used as in the prior art.

[0020] The gas inlet conduit 11 of the regulating valve 100 is fluidically communicated with the safety valve 30, the rotating disc 20 being arranged in the housing 13 of the valve body 10, which is in turn fluidically communicated with the safety valve 30 and the outlet conduit 12. In this embodiment of the regulating valve 100, the housing 13 of the valve body 10 is open to the outside. Said regulating valve 100 comprises a closure cover 3 which is attached to the valve body 10 and closes the housing 13 in a leak-tight manner. The gasket 60 is first arranged in said housing 13 on the contact surface 14 of the housing 13, and a face 22 of the rotating disc 20 is then arranged supported on the gasket 60. The spring 50 which pushes the rotating disc 20 and retains it axially against the contact surface 14 is then arranged such that it is supported thereon, pressing on the gasket 60. The spring 50 is operatively supported at one end 51 on a face 23 of the rotating disc 20, opposite to the face 22, , and at a second end 52, opposite to the end 51, is supported on the closure cover 3 .

[0021] The regulating valve 100 also comprises a drive shaft 40 coupled at a lower end to the rotating disc 20 in a D-shaped hole 24 comprising said rotating disc 20, and comprises another upper end to which there is coupled a handle (not depicted) that can be operated by the user. Said drive shaft 40 can rotate between an initial position of rotation and a final position of rotation, and can also be moved axially to act on a transmission element 2 for opening the safety valve 30. Said transmission element 2 is housed in a housing of the cover 4, which forms the closure cover 3 with the housing 13 of the valve body 10. The transmission element 2 acts on the safety valve 30 in the housing of the cover 4. This housing of the cover 4 is fluidically communicated with a hole (not shown in the drawings) which is arranged in the valve body 10. This hole fluidically communicates with the safety valve 30. In this embodiment, the transmission element 2 is a rocker arm, and the safety valve 30 comprises a shutter closing it, such that when the drive shaft 40 is pushed axially, the lower end coupled to the rotating disc 20 pushes the transmission element 2, and this transmission element 2 acts on the shutter of the safety valve 30 through the hole of the valve body 10, opening the safety valve 30. The incoming gas from the inlet conduit 11 can therefore pass through the safety valve 30, and through the open shutter, to the housing of the cover 3, occupying the housing 13 of the valve body 10 where the rotating disc 20 is arranged.

[0022] In this embodiment, the rotating disc 20 comprises a connection opening 21 comprising a plurality of through holes for regulating gas flow from the housing 13 of the valve body 10 to the outlet conduit 12 depending on the angular position thereof. The through holes of the connection opening 21 form a radially-shaped row, such that in order to obtain the required gas flow rate in the gas outlet conduit 12, a specific number of holes of the

connection opening 21 of the rotating disc 20 are overlapped with the outlet hole 15a and/or the outlet hole 15b which is a minimum gas flow hole. Gas flow is varied by means of said holes of the connection opening 21 getting in and out of overlap with the outlet holes 15a and 15b.

[0023] The lower end of the drive shaft 40 is coupled to the rotating disc 20 in the coupling hole 24, said coupling hole 24 allowing the axial movement of the drive shaft 40 without axially moving the rotating disc 20. In turn, the gasket 60 allows the passage of the lower end of the drive shaft 40 through the space left by the attachment arm 80b around the center of the gasket 60. Finally, the lower end of the drive shaft 40 is housed in a housing 6 comprised in the valve body 10 on the contact surface 14 of the housing 13.

[0024] The regulating valve 100 comprises a spring 44 which is supported at one end on a rib of the drive shaft 40 and operatively supported at a second end on the valve body 10, the function of said spring 44 being to return the drive shaft 40 to a standby position when said drive shaft 40 is no longer axially pushed to open the safety valve 30. The regulating valve 100 comprises a cover 1 which has the drive shaft 40 going through same and closes a cavity formed between said cover 1 and the valve body 10, with the spring 44 being arranged in said cavity. The cover 1 of the regulating valve 100 comprises on the side arranged in the cavity of the spring 44 a plurality of concatenated housings, and the drive shaft 40 comprises a pin 41 arranged in an orthogonal manner with respect to said drive shaft 40, said pin 41 being pushed against the housings of the cover 1 by the action of the spring 44. In this manner, when the drive shaft 40 is rotated and arranged in the range of angular positions defining a gas flow of the regulating valve 100, it defines discrete positions of different gas flows in the gas regulating valve, which are distinguished by the user acoustically and/or sensorially.

[0025] The contact surface 14 on which the rotating disc 20 is located comprises the minimum hole 15b which is fluidically communicated with a corresponding gas minimum conduit (not shown in the drawings). This gas minimum conduit is fluidically communicated with the gas outlet conduit 12. The minimum gas flow of the regulating valve 100 is regulated by rotating the rotating disc 20 and overlapping a specific number of the through holes of the connection opening 21 with the minimum hole 15b in a specific angular position of the rotating disc 20. In this embodiment, the regulating valve 100 comprises, as shown in Figure 1, regulating means 5 suitable for regulating the minimum gas flow rate. In this embodiment of the regulating valve 100, the regulating means 5 is a screw comprising a calibrated hole defining the minimum gas flow rate to be regulated in said regulating valve 100. The regulating means 5 are arranged in a conduit going through the gas minimum conduit, the conduit of the regulating means 5 being accessible from the outside of the regulating valve 100. With the help of a tool, it can be rotated such that said regulating means 5 move axially,

which thereby allows regulating the passage of a different minimum gas flow rate depending on the position thereof. The minimum gas flow rate in the regulating valve 100 is regulated with greater precision with the help of the regulating means 5.

[0026] In this embodiment of the regulating valve 100, the protuberances 81 project from the surface of the attachment arm 80a towards the rotating disc 20 with a semi-cylindrical shape arranged on the outer edge of the surface, and defining a curved outer surface which thereby presents a minimum contact, and therefore friction, surface with the face 22 of the rotating disc 20 when said rotating disc 20 rotates as a result of the action of the drive shaft 40, and when said rotating disc 20 is supported on the protuberances 81 of the attachment arm 80a, pressing on the gasket 60.

[0027] In turn, the attachment arms 80a and 80b of the gasket 60 have a quadrangular-shaped cross-section, allowing a uniform and extended contact with the contact surface 14 of the housing 13 of the valve body 10, such that it leads to very stable seating of said gasket 60 in the regulating valve 100, despite the thrust performed by the rotating disc 20 during rotation thereof due to the action of the drive shaft 40.

[0028] In turn, the respective body 71a and 71b of the closure members 70a and 70b of the gasket 60 has a circular cross-section, said closure members 70a and 70b being configured as oblong-shaped O-rings fitting and surrounding the respective outlet holes 15a and 15b. Like in the protuberances 81, said O-rings define a curved outer surface which thereby presents a small, and therefore lower-friction, contact surface with the face 22 of the rotating disc 20 when said rotating disc 20 rotates as a result of the action of the drive shaft 40, and when said rotating disc 20 is supported on the bodies 71a and 71b of the closure members 70a and 70b, pressing on the gasket 60.

[0029] The contact surface 14 of the housing 13 of the valve body 10 comprises, as shown in Figure 4, a planar support surface on which there is constructed, machined thereon, a channel 16 comprising different branches or arms extending over the contact surface 14 with the shape of the gasket 60, the attachment arms 80a and 80b and the closure members 70a and 70b of the gasket 60 being housed in said arms of the channel 16. The gasket 60 is housed in said channel 16, such that each attachment arm 80a and 80b and each closure member 70a and 70b are locked in said channel 16 in the walls forming each arm of said channel 16, said gasket 60 being pressed into the channel 16 as result of the support of the rotating disc 20. This housing of the gasket 60 also allows assuring the stability of said gasket 60, given that each attachment arm 80a and 80b is housed in branches of the channel 16 with side walls, but each radially opposite segment of said attachment arms 80a and 80b also abuts opposite walls of the channel 16 which keep said attachment arms 80a and 80b immobile when the rotating disc 20 is rotated.

[0030] To further assure sealing in the outlet hole 15a of the main gas flow of the regulating valve 100, and to improve gasket stability in the region of the closure members 70a and 70b of the gasket 60, said regulating valve 100 comprises, as shown in Figures 3, 9A, and 9B, a plastic bushing 17 arranged such that it is introduced in the outlet hole 15a. Said bushing 17 is partially introduced in the outlet hole 15a, with an end of the bushing 17 being supported on an inner seating of the valve body 10 inside the outlet hole 15a, and a segment of the bushing 17 projecting out of the outlet hole 15a, such that the body 71a of the closure member 70a of the gasket 60 is housed in the corresponding branch of the channel 16, laterally abutting the wall of the channel 16 itself and the projecting segment of the bushing 17. The body 71a is therefore immobilized in the channel 16, and the pressure of the rotating disc 20 on said body 71a is applied more effectively.

[0031] In this embodiment and as shown in Figure 7, the regulating valve 100 also comprises a pushing plate 90 which is supported on the rotating disc 20 occupying the entire surface thereof, the end 51 of the spring 50 which pushes the rotating disc 20 being supported on one face of the pushing plate 90, and the other face of the pushing plate 90 being supported on the face 23 of the rotating disc 20. In this embodiment of the regulating valve 100, the spring 50 is a constant load spring, given that the axis of axial movement of the drive shaft 40 does not affect the axial movement of the rotating disc 20, and therefore does not affect the spring 50. This spring 50 keeps its pressure on the rotating disc 20 constant, and does not rotate when said rotating disc 20 is angularly shifted.

[0032] The pushing plate 90 comprises a plurality of tabs 91, specifically three in this embodiment, which allow coupling the end 51 of the spring 50 to the pushing plate 90, being clipped to the tabs 91. Said tabs 91 are arranged forming a circle the center of which does not coincide with the center of the pushing plate 90 which is in the center of a hole 96 allowing the passage of the drive shaft 40, the function of this off-centering is defined below.

[0033] In this embodiment of the regulating valve 100, the pushing plate 90 has a circular shape, like the rotating disc 20 and the contact surface 14 of the housing 13. The pushing plate 90 comprises along the circular perimeter of the edge 92 a plurality of protrusions 93, specifically 3 in this embodiment, distributed on the edge 92 every 120°. The pushing plate 90 also comprises, arranged on said edge 92, a semicircular-shaped orienting protrusion 94. For its part, the housing 13 of the valve body 10 comprises a circular side wall 18 bordering the contact surface 14, said wall 18 comprising a semi-cylindrical-shaped housing 19 (see Figure 4) which houses the orienting protrusion 94 of the pushing plate 90 when the pushing plate 90 is assembled on the rotating disc 20 in the housing 13. The pushing plate 90 is thereby immobilized in the housing 13.

[0034] When said assembly is performed, the protrusions 93 of the pushing plate 90 are arranged pressing on the side wall 18, such that it makes it difficult for the pushing plate 90 to move out of the housing 13 and of its position on the rotating disc 20, particularly for operations of assembling the regulating valve 100. To improve the ease of assembly and the useful life of the pushing plate 90, said pushing plate 90 comprises a plurality of through openings 95, specifically three in this embodiment, arranged inside the pushing plate 90 confronted in association with each of the protrusions 93. The pressure exerted by the side wall 18 of the housing 13 on the protrusions 93 of the pushing plate 90 is therefore absorbed by the flexible openings 95. The pushing plate 90 also comprises two through holes 97 and 98 which provide passage, respectively, to the gas flow from the housing 13 to the outlet holes 15a and 15b through the connection opening 21 of the rotating disc 20.

[0035] In this embodiment of the regulating valve 100 comprising two outlet holes 15a and 15b, said outlet holes are adjacent and arranged close to an edge of the contact surface 14. Since the pressure that the spring 50 operatively exerts on the gasket 60, and specifically on the closure regions of said gasket 60 formed by the closure members 70a and 70b, is important, said spring 50 is arranged on the pushing plate, as shown in Figure 8, clipped to the tabs 93, and off-centered with respect to the center of the rotating disc 20 and of the pushing plate 90, such that the axial projection of the spring 50 over the gasket 60 passes, in this embodiment of the regulating valve 100, partially over the bodies 71a and 71b of the corresponding closure members 70a and 70b. Due to the dimension of the regulating valve 100 that is smaller than other valves with a higher number of gas outlet conduits, and to the arrangement of the corresponding outlet holes 15a and 15b on the contact surface 14, one spring 50 is sufficient in this regulating valve 100 to assure sealing, and particularly sealing in the closed position of said regulating valve 100.

[0036] Figures 10 to 17 show a second embodiment of the regulating valve 100' for a gas cooking appliance according to the invention, comprising a safety valve 30, and two gas outlet conduits 12a and 12b. This regulating valve 100' comprises the same features as the embodiment of the regulating valve 100, with the difference that it comprises, on the contact surface 14 of the housing 13 of the valve body 10, two outlet holes 15a and 15b in fluid communication in a corresponding manner with the two gas outlet conduits 12a and 12b, the outlet holes 15a and 15b in this embodiment being two holes for the main gas flow, and not for the minimum flow specifically with a bypass. The gas outlet conduits 12a and 12b are suitable for conducting the incoming gas to a burner with two caps, in a corresponding manner to an inner cap and an outer cap (not depicted). Gas flow to the outlet conduits 12a and 12b is regulated by overlapping the outlet holes 15a and 15b with a corresponding number of connection openings 21a and 21b of the rotating disc 20.

[0037] Figure 14 shows an embodiment of the rotating disc 20 for the regulating valve 100', comprising two through connection openings 21a and 21b for regulating gas flow from the housing 13 of the valve body 10 to the outlet conduits 12a and 12b depending on the angular position thereof. The through holes of the connection openings 21a and 21b form arcs of different radius, the radius of the opening 21b being smaller than the radius of the opening 21a, such that in order to obtain the required gas flow rate in the gas outlet conduits 12a and 12b, a specific number of holes of the connection openings 21a and 21b of the rotating disc 20 are overlapped with the outlet holes 15a and 15b.

[0038] The remaining differences of this regulating valve 100' with the previously described embodiment of the regulating valve 100 are that the gasket 60, as shown in Figure 13, also comprises two closure members 70a and 70b associated respectively with each outlet hole 15a and 15b, but in this embodiment of the regulating valve 100', the outlet holes 15a and 15b are not adjacent, but rather arranged at two ends of the gasket 60 separated by a 180°, said closure members 70a and 70b being close to an edge of the contact surface 14. The gasket 60 also comprises two attachment arms 80a and 80b radially attaching the two closure members 70a and 70b, the two attachment arms 80a and 80b virtually forming a circle, with the two attachment arms 80a and 80b surrounding the center of the gasket 60, such that the gasket 60 is substantially centered on the contact surface 14. In this embodiment, each of the arms 80a and 80b comprises a protuberance 81, such that the rotating disc 20 is supported on the support surface 73a and 73b of the bodies 71a and 71b of the closure members 70a and 70b, respectively, and on the upper region 82 of the protuberances 81, when it is arranged pressing on the gasket 60.

[0039] In this embodiment of the regulating valve 100', the attachment arms 80a and 80b extends radially almost to the edge of the contact surface 14 in the housing 13, such that it confers stability to the seating of said gasket 60 in the housing 13 of the valve body 100. The protuberances 81 are arranged on the surface of the attachment arms 80a and 80b which contacts the rotating disc 20, virtually in the center of said surface, said protuberances 81 being distributed in an approximate angle of 180° with respect to one another, with a center in the center of the gasket 60.

[0040] For the same reasons as those set forth for the first embodiment of the regulating valve 100, the solution defined in the second embodiment of the regulating valve 100' allows assuring sealing in any position of rotation of the rotating disc 20, and particularly in the closed position of the regulating valve, due to the increase in pressure exerted on the gasket 60 for the same size of spring 50 used as in the prior art, due to the smaller support surface of the rotating disc 20 on the gasket 60.

[0041] In turn, the respective body 71a and 71b of the closure members 70a and 70b of the gasket 60 has a

substantially circular cross-section, said closure members 70a and 70b being configured as oblong-shaped joints fitting and surrounding the respective outlet holes 15a and 15b. The bodies 71a and 71b comprise a closure protuberance 74a and 74b projecting in the upper part of the joint and running along the contour of the bodies 71a and 71b where the rotating disc 20 is supported. In this manner, like in the protuberances 81, said bodies 71a and 71b define a very small, and therefore lower-friction, contact surface with the face 22 of the rotating disc 20, when said rotating disc 20 rotates as a result of the action of the drive shaft 40, and also when said rotating disc 20 is supported on the bodies 71a and 71b of the closure members 70a and 70b, pressing on the gasket 60. Therefore, the surface offered by the gasket 60 for supporting the rotating disc 20 is defined only by the closure protuberances 74a and 74b and by the upper region 82 of the protuberances 81.

[0042] In this embodiment of the regulating valve 100', the bodies 71a and 71b of the closure members 70a and 70b also comprise respective lower protuberances 75a and 75b projecting in the lower part of the joint and running along the contour of the bodies 71a and 71b where the gasket 60 is supported on the contact surface 14. Like the closure protuberances 74a and 74b, the lower protuberances 75a and 75b define a very small contact surface, and therefore the pressure exerted by the gasket 60 on the contact surface 14 is greater, which assures sealing to a larger extent.

[0043] Moreover, in this embodiment of the regulating valve 100', the protuberances 81 have a longitudinal axis which is angularly shifted with respect to the longitudinal axis of the respective attachment arms 80a and 80b. The arrangement of said protuberances 81 in the attachment arms 80a and 80b forms part of a virtual circle with a center in the center of rotation of the rotating disc 20, such that when rotating said rotating disc 20 in the contact thereof with the gasket 60, said contact occurs on the protuberances 81 in an area with a smaller contact surface.

[0044] The channel 16 on the contact surface 14 comprises two branches or arms in which the attachment arms 80a and 80b are housed, attaching two regions in which the closure members 70a and 70b of the gasket 60 are housed. This housing of the gasket 60 in the channel 16 allows assuring the stability of said gasket 60, given that each attachment arm 80a and 80b is housed in the channel 16 the side walls of which contain and immobilize said gasket 60.

[0045] In this embodiment and as shown in Figure 15, the regulating valve 100' also comprises a pushing plate 90 which is supported on the rotating disc 20 occupying the entire surface thereof, the end 51 of the spring 50 which pushes the rotating disc 20 being supported on one face of the pushing plate 90, and the other face of the pushing plate 90 being supported on the face 23 of the rotating disc 20.

[0046] In this embodiment, the pushing plate 90 also

comprises three tabs 91, which allow coupling the end 51 of the spring 50 to the pushing plate 90, clipping to the tabs 91. Said tabs 91 are arranged forming a concentric circle with the center of the pushing plate 90. The pushing plate 90 also comprises two through holes 97 and 98 which provide passage, respectively, to the gas flow from the housing 13 to the outlet holes 15a and 15b through the connection openings 21a and 21b, respectively, of the rotating disc 20.

[0047] In this embodiment of the regulating valve 100', the two outlet holes 15a and 15b are not adjacent and arranged with the outlet hole 15a closer to the edge of the contact surface 14 and the outlet hole 15b, in a diameter with the outlet hole 15a on the contact surface 14, between the center of said contact surface 14 and the edge. Since the pressure that the spring 50 operatively exerts on the gasket 60, and specifically on the closure regions of said gasket 60 formed by the closure members 70a and 70b, is important, said spring 50 is arranged on the pushing plate 90, as shown in Figures 16 and 17, clipped to the tabs 93, and centered with respect to the center of the rotating disc 20 and of the pushing plate 90, such that the axial projection of the spring 50 over the gasket 60 passes, in this embodiment of the regulating valve 100', through the inner edge of the body 71a and the outer edge of the body 71b of the corresponding closure members 70a and 70b. With this dimension of the regulating valve 100' and with this arrangement of the corresponding outlet holes 15a and 15b on the contact surface 14, one spring 50 is sufficient in this regulating valve 100' to assure sealing, and particularly sealing in the closed position of said regulating valve 100.

[0048] Figures 18 to 26 show a third embodiment of the regulating valve 100" for a gas cooking appliance according to the invention, comprising a safety valve 30, and three gas outlet conduits 12a, 12b, and 12c. This regulating valve 100" comprises the same features as the embodiment of the regulating valve 100', with the difference that it comprises two springs, a spring 50a and a second spring 50b, which pushes the rotating disc 20 and retain it axially against the contact surface 14. The spring 50a has a larger diameter than the spring 50b. Furthermore, the contact surface 14 of the housing 13 of the valve body 10 comprises three outlet holes 15a, 15b, and 15c in fluid communication in a corresponding manner with the two gas outlet conduits 12a, 12b, and 12c, the outlet holes 15a, 15b, and 15c in this embodiment being three holes for the main gas flow, and not for the minimum flow specifically with a bypass. The gas outlet conduits 12a, 12b, and 12c are suitable for conducting the incoming gas to a burner with three caps, in a corresponding manner to an inner cap, an intermediate cap, and an outer cap (not depicted). Gas flow to the outlet conduits 12a, 12b, and 12c is regulated by overlapping the outlet holes 15a, 15b, and 15c with a corresponding number of connection openings 21a, 21b, 21c of the rotating disc 20.

[0049] Figure 22 shows an embodiment of the rotating disc 20 for the regulating valve 100", comprising three through connection openings 21a, 21b, 21c for regulating gas flow from the housing 13 of the valve body 10 to the outlet conduits 12a, 12b, and 12c depending on the angular position thereof. The through holes of the connection openings 21a, 21b, and 21c each forms a radial-shaped row of different radius, the radius of the opening 21c being smaller than the radius of the opening 21b, and the radius of the opening 21b being smaller than the radius of the opening 21a, such that in order to obtain the required gas flow rate in the gas outlet conduits 12a, 12b, and 12c, a specific number of holes of the connection openings 21a, 21b, and 21c of the rotating disc 20 are overlapped with the outlet holes 15a, 15b, and 15c.

[0050] The remaining differences of this regulating valve 100" with the previously described embodiment of the regulating valve 100' are that the gasket 60, as shown in Figure 21, comprises three closure members 70a, 70b, and 70c associated, respectively, with each outlet hole 15a, 15b, and 15c, but in this embodiment of the regulating valve 100" the outlet holes 15a, 15b, and 15c are not arranged at 180°, but rather arranged in approximately a triangle of the gasket 60, with the closure member 70a being close to an edge of the contact surface 14, the closure member 70b in an intermediate region of the contact surface 14, and the closure member 70c close to the hole 6 of the contact surface 14, where the lower end of the drive shaft 14 is housed.

[0051] The gasket 60 comprises four attachment arms 80a, 80b, 80c, and 80d, respectively, radially attaching the two closure members 70a and 70b, 70a and 70b in a circular path opposite the preceding one, 70a and 70c, and 70c and 70b, the two attachment arms 80a and 80b virtually forming a circle, the attachment arms 80c and 80d transversely attaching the attachment arm 80b through the closure member 70c, and the four attachment arms 80a, 80b, 80c, and 80d surrounding the center of the gasket 60. In this embodiment, as shown in detailed in Figure 26, each of the arms 80a and 80b comprises a protuberance 81, such that the rotating disc 20 is supported on the support surface 73a, 73b, and 73c of the bodies 71a, 71b, and 71c of the closure members 70a, 70b, and 70c, respectively, and on the upper region 82 of the protuberances 81, when it is arranged pressing on the gasket 60.

[0052] In this embodiment of the regulating valve 100", the circle formed by the attachment arms 80a and 80b are slightly off-centered with respect to the center of the contact surface 14, not covering the entire contact surface 14 in the housing 13, but the opening thereof is wide enough so as to confer stability to the seating of said gasket 60 in the housing 13 of the valve body 10. The protuberances 81 are arranged on the surface of the attachment arms 80a and 80b which contacts the rotating disc 20, virtually in the center of said surface, said protuberances 81 being distributed in an approximate angle of 180° with respect to one another, with a center in the

center of the gasket 60.

[0053] For the same reasons as those set forth for the first and second embodiments of the regulating valve 100 and 100', the solution defined in the third embodiment of the regulating valve 100" allows assuring sealing in any position of rotation of the rotating disc 20, and particularly in the closed position of the regulating valve, due to the increase in pressure exerted on the gasket 60 for the same size of spring 50 used as in the prior art, due to the smaller support surface of the rotating disc 20 on the gasket 60.

[0054] In turn, the respective body 71a, 71b, and 71c of the closure members 70a, 70b, and 70c of the gasket 60 has, like in the gasket 60 of the second embodiment of the regulating valve 100', a substantially circular cross-section, said closure members 70a, 70b, and 70c being configured as oblong-shaped joints fitting and surrounding the respective outlet holes 15a, 15b, and 15c. The bodies 71a, 71b, and 71c comprise, as shown in detailed in Figure 26, a closure protuberance 74a, 74b, and 74c projecting in the upper part of the joint, and running along the contour of the bodies 71a, 71b, and 71c where the rotating disc 20 is supported. In this manner, like in the protuberances 81, said bodies 71a, 71b, and 71c define a very small, and therefore lower-friction, contact surface with the face 22 of the rotating disc 20, when said rotating disc 20 rotates as a result of the action of the drive shaft 40, and also when said rotating disc 20 is supported on the bodies 71a, 71b, and 71c of the closure members 70a, 70b, and 70c, pressing on the gasket 60. Therefore, the surface offered by the gasket 60 for supporting the rotating disc 20 is defined only by the closure protuberances 74a, 74b, and 74c and by the upper region 82 of the protuberances 81.

[0055] In this embodiment of the regulating valve 100", the bodies 71a, 71b, and 71c of the closure members 70a, 70b, and 70c also comprise respective lower protuberances 75a, 75b, and 75c projecting in the lower part of the joint, and running along the contour of the bodies 71a, 71b, and 71c where the gasket 60 is supported on the contact surface 14. Like the closure protuberances 74a, 74b, and 74c, the lower protuberances 75a, 75b, and 75c define a very small contact surface, and therefore the pressure exerted by the gasket 60 on the contact surface 14 is greater, which assures sealing to a larger extent.

[0056] Figure 26 also shows in detailed how the protuberance 81 of the gasket 60 of the regulating valves 100 and 100' is, and how the closure protuberances 74a and 74b and the lower protuberances 75a and 75b of the body 71a and 71 b of the respective closure members 70a and 70b of the regulating valve 100' are.

[0057] Moreover, in this embodiment of the regulating valve 100', like in the first embodiment of the regulating valve 100, the protuberances 81 have a longitudinal axis which is aligned with the longitudinal axis of the attachment arms 80a and 80b incorporating same.

[0058] The channel 16 on the contact surface 14 com-

prises as many branches or arms as the number of attachment arms, attaching three regions in which the closure members 70a, 70b, and 70c of the gasket 60 are housed. This housing of the gasket 60 in the channel 16 allows assuring the stability of said gasket 60, given that each attachment arm 80a, 80b, 80c, and 80d is housed in the channel 16 the side walls of which contain and immobilize said gasket 60.

[0059] In this embodiment and as shown in Figure 23, the regulating valve 100" also comprises a pushing plate 90 which is supported on the rotating disc 20 occupying the entire surface thereof, the ends 51a and 51b of the springs 50a and 50b pushing the rotating disc 20 being supported on one face of the pushing plate 90, and the other face of the pushing plate 90 being supported on the face 23 of the rotating disc 20.

[0060] In this embodiment, the pushing plate 90 comprises six tabs 91 arranged on the face on which the springs 50a and 50b are coupled, three of them being distributed in an outer circle and three of them distributed in an inner circle which allow coupling the ends 51a and 51b, respectively, of the springs 50a and 50b to the pushing plate 90, being clipped to the tabs 91. Said tabs 91 are arranged forming concentric circles with the center of the pushing plate 90. The pushing plate 90 comprises three through holes 97, 98, and 99 which provide passage, respectively, to the gas flow from the housing 13 to the outlet holes 15a, 15b, and 15c through the connection openings 21a, 21b, and 21c, respectively, of the rotating disc 20.

[0061] In this embodiment of the regulating valve 100", the three outlet holes 15a, 15b, and 15c are not adjacent and arranged with the outlet hole 15a closer to the edge of the contact surface 14, the outlet hole 15b in a central position on the contact surface 14, and the outlet hole 15c closer to the center of the contact surface 14, in the housing 6 of the lower end of the drive shaft 40, the three outlet holes 15a, 15b, and 15c forming a triangle. Since the pressure that the springs 50a and 50b operatively exert on the gasket 60, and specifically on the closure regions of said gasket 60 formed by the closure members 70a, 70b, and 70c, is important, said springs 50a and 50b are arranged on the pushing plate 90, as shown in Figures 16 and 17, clipped to the tabs 93, and centered with respect to the center of the rotating disc 20 and of the pushing plate 90, such that the axial projection of the springs 50a and 50b on the gasket 60 passes, in this embodiment of the regulating valve 100", the spring 50a by the outer edge of the body 71a, and the second spring 50b by the inner edge of the body 71b and by the inner edge of the body 71c, of the respective closure members 70a, 70b, and 70c. This embodiment of the regulating valve 100" has a greater dimension than the first and second embodiments of the regulating valve 100 and 100', and furthermore taking into account such an open arrangement of the corresponding outlet holes 15a, 15b, and 15c on the contact surface 14, a single spring in this regulating valve 100" is not sufficient to make compres-

sion on the closure surface of the gasket 60 uniform, and thereby assure sealing, and particularly sealing in the closed position of said regulating valve 100", so two springs 50a and 50b with the characteristics defined above are arranged.

[0062] The invention also relates to a gas cooking appliance 200 incorporating at least one regulating valve 100, 100', 100" according to the invention. Figure 27 shows, in an illustrative manner, a schematic depiction of an embodiment of the gas cooking appliance 200 of the invention with four burners 300, each with its respective regulating valve 100, 100', 100". The gas cooking appliance 200 of Figure 26 also comprises an ON-OFF shut-off valve 400 at the gas supply inlet. Although the regulating valves 100, 100', 100" are manually operated in this embodiment, in other embodiments not shown in the drawings, the regulating valves 100, 100', 100" can be controlled by a control device, or both by a control device and manually.

[0063] The gas cooking appliance 200 can be, for example, a gas cooktop, a gas cooker, a gas oven, or a grill.

Claims

1. Regulating valve for a gas cooking appliance, comprising a valve body (10) comprising an inlet conduit (11) through which gas is supplied, at least one gas outlet conduit (12), and a housing (13) fluidically communicated with the inlet conduit (11) and the outlet conduit (12); a rotating disc (20) arranged in the housing (13) of the valve body (10) on a contact surface (14) of the housing (13), the contact surface (14) comprising at least one outlet hole (15) fluidically communicated with the outlet conduit (12), the rotating disc (20) being suitable for rotating with respect to the contact surface (14) and for regulating gas flow between the housing (13) and the outlet conduit (12) by means of the rotation of the rotating disc (20); a spring (50) which pushes the rotating disc (20) and retains it axially against the contact surface (14); and sealing means housed on the contact surface (14) surrounding the at least one outlet hole (15), the rotating disc (20) being supported pressing on the sealing means, **characterized in that** the sealing means comprise a gasket (60) comprising a closure member (70) associated with each outlet hole (15), each closure member (70) comprising a body (71) surrounding the corresponding outlet hole (15), and at least one attachment arm (80) attached to the at least one closure member (70), the at least one attachment arm (80) comprising at least one protuberance (81), with the rotating disc (20) being supported on the closure members (70) and on the protuberances (81) when it presses on the gasket (60).
2. Regulating valve according to claim 1, wherein the protuberance (81) projects from the attachment arm

- (80) defining a curved outer surface.
3. Regulating valve according to claim 1 or 2, wherein the attachment arm (80) has a quadrangular-shaped cross-section. 5
 4. Regulating valve according to any of claims 1 to 3, wherein the protuberance (81) has a longitudinal axis angularly shifted with respect to the longitudinal axis of the attachment arm (80), said protuberance (81) preferably forming part of a virtual circle with a center in the center of rotation of the rotating disc (20). 10
 5. Regulating valve according to any of claims 1 to 4, wherein the body (71) of the closure member (70) of the gasket (60) has a circular cross-section. 15
 6. Regulating valve according to any of claims 1 to 5, wherein the body (71) comprises a closure protuberance (74) running along the contour of the body (71) where the rotating disc (20) is supported. 20
 7. Regulating valve according to claim 6, wherein the body (71) of the closure member (70) comprises a lower protuberance (75) running along the contour of the body (71) where the gasket (60) is supported on the contact surface (14). 25
 8. Regulating valve according to any of claims 1 to 7, wherein the at least one attachment arm (80) attaches the at least one closure member (70) surrounding the center of rotation of the rotating disc (20). 30
 9. Regulating valve according to any of claims 1 to 8, wherein the contact surface (14) of the housing (13) of the body (10) comprises a planar support surface comprising a channel (16), with the gasket (60) being housed in said channel (16). 35
 10. Regulating valve according to claim 9, comprising a bushing (17) arranged such that it is introduced in the outlet hole (15), with a segment of said bushing (17) projecting from the outlet hole (15), the body (71) of the closure member (70) being housed in the channel (16) laterally abutting the wall of the channel (16) and the segment of the bushing (17). 40 45
 11. Regulating valve according to any of claims 1 to 10, comprising a pushing plate (90) which is supported on the rotating disc (20), an end (51) of the spring (50) which pushes the rotating disc (20) being supported on the pushing plate (90), the pushing plate (90) comprising a plurality of tabs (91) which allow coupling the end (51) of the spring (50) to the pushing plate (90). 50 55
 12. Regulating valve according to claim 11, wherein the pushing plate (90) comprises along the perimeter of an edge (92) a plurality of fixing protrusions (93) and an orienting protrusion (94), the housing (13) of the body (10) comprising a side wall (18) bordering the contact surface (14), said wall (18) comprising a housing (19), the orienting protrusion (94) being housed in the housing (19), and the protrusions (93) being arranged pressing on the side wall (18) when the pushing plate (90) is supported on the rotating disc (20).
 13. Regulating valve according to claim 12, wherein the pushing plate (90) comprises a plurality of openings (95) arranged inside the pushing plate (90) confronted in association with each of the protrusions (93).
 14. Regulating valve according to any of claims 1 to 13, wherein if there is an outlet hole (15) the axial projection of the spring (50) over the gasket (60) partially passes over the body (71) of the corresponding closure member (70), if there are two outlet holes (15) the axial projection of the spring (50) over the gasket (60) partially passes over the body (71) of the two corresponding closure members (70), and if there are three outlet holes (15) the axial projection of the spring (50) over the gasket (60) partially passes over the body (71) of two of the closure members (70), the regulating valve (100) comprising a second spring (50) partially passing over the body (71) of the third closure member (70).
 15. Gas cooking appliance comprising at least one regulating valve (100, 100', 100'') according to any of the preceding claims.

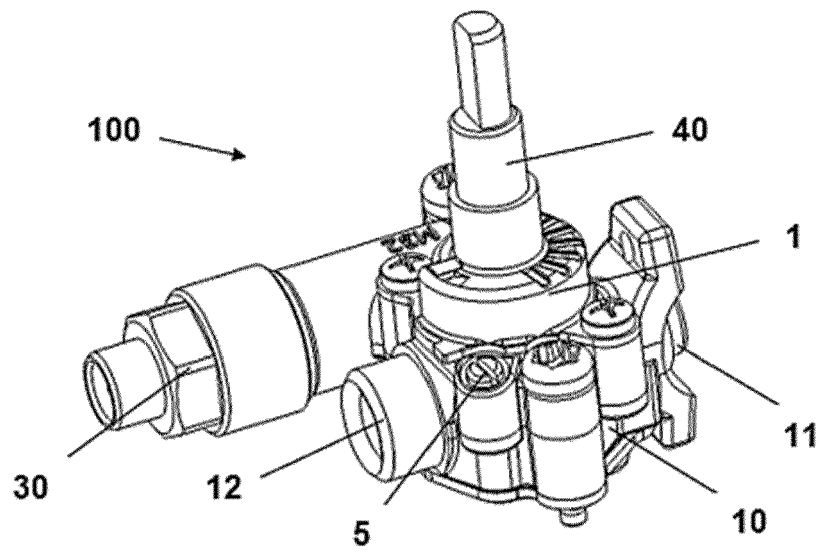


FIG. 1

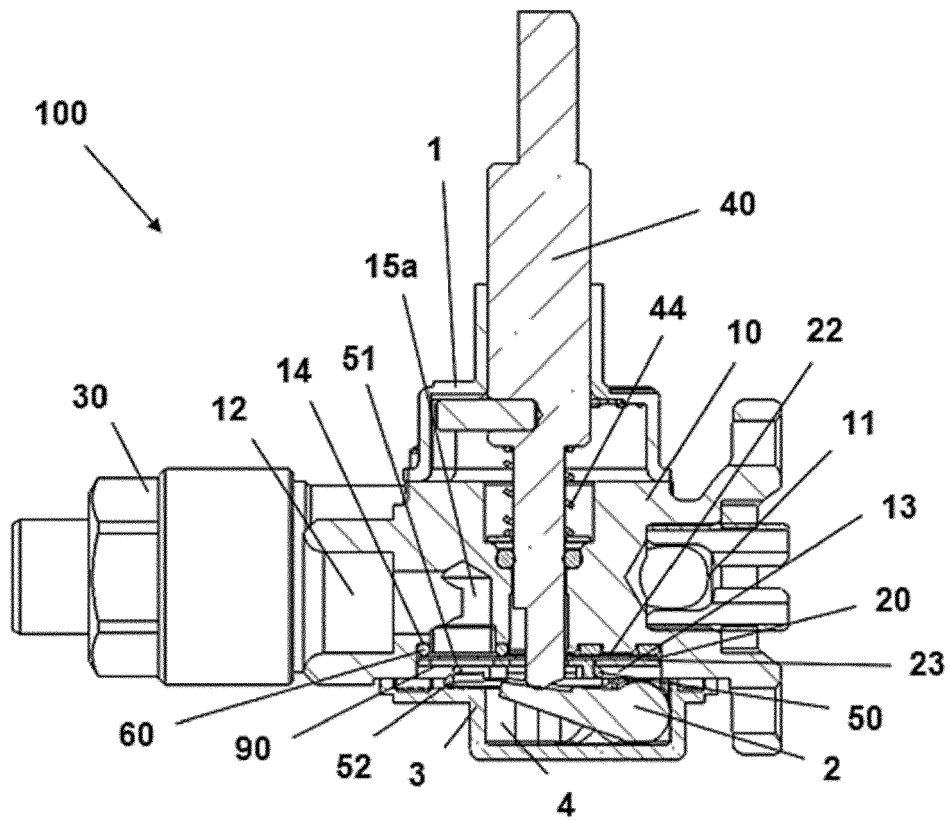


FIG. 2

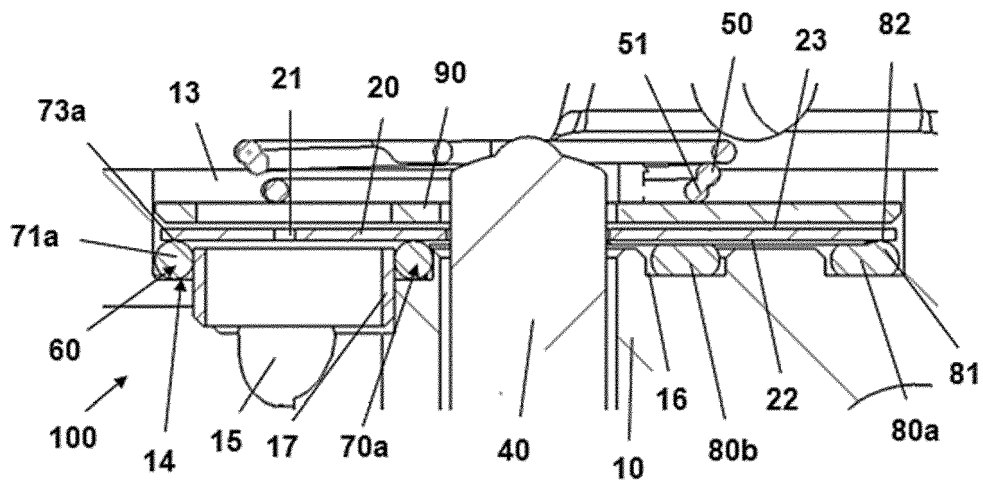


FIG. 3

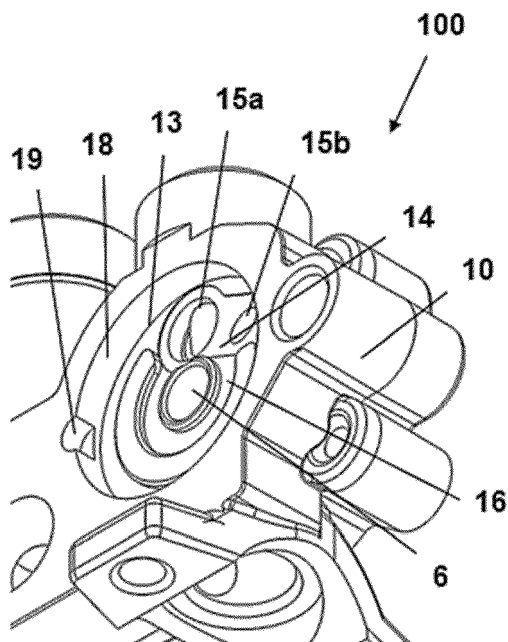


FIG. 4

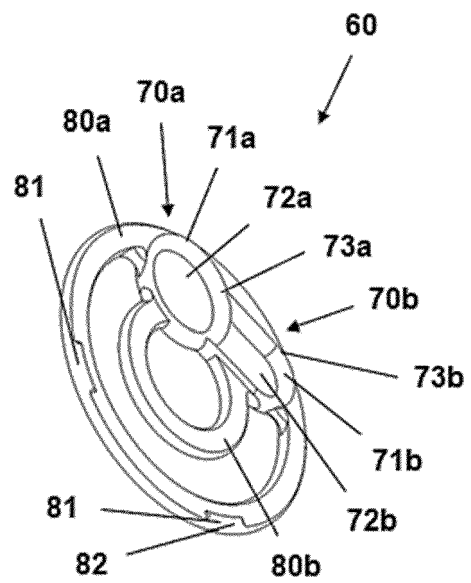


FIG. 5

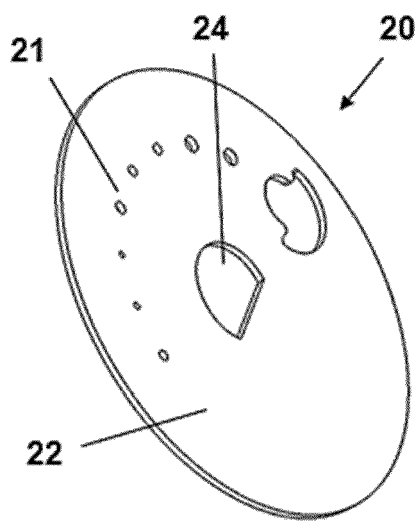


FIG. 6

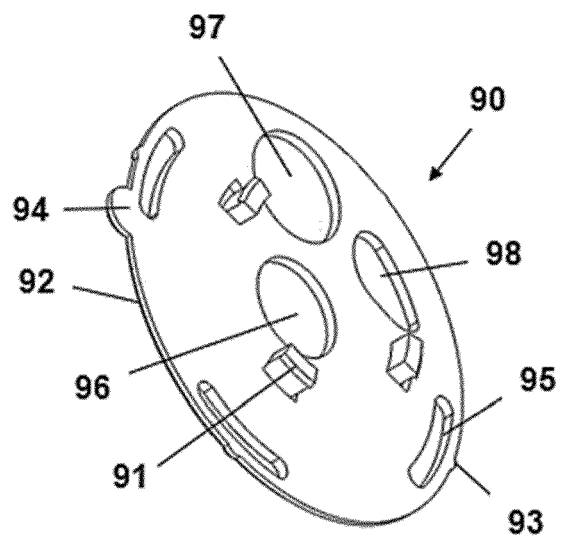


FIG. 7

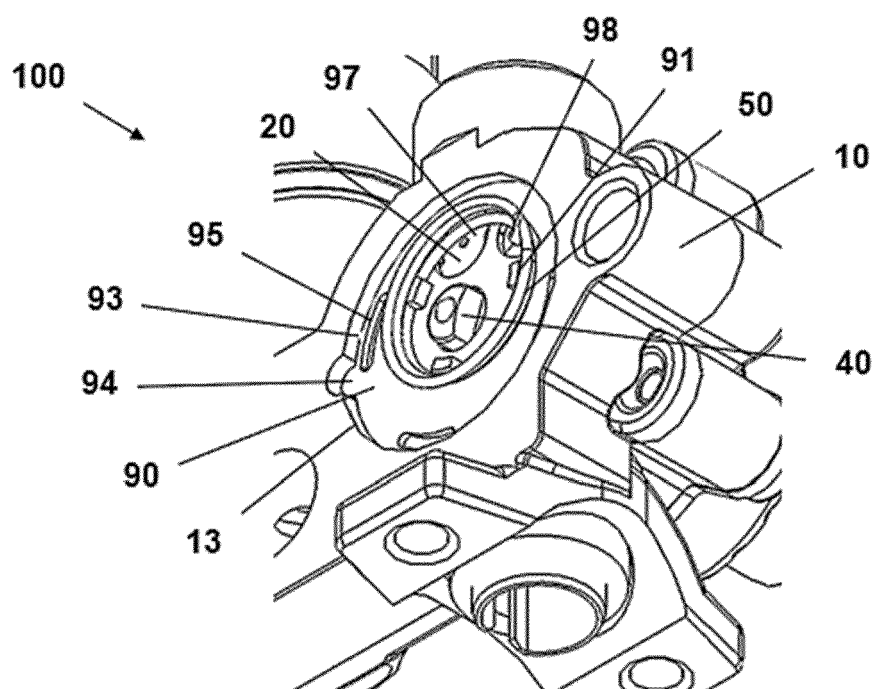


FIG. 8

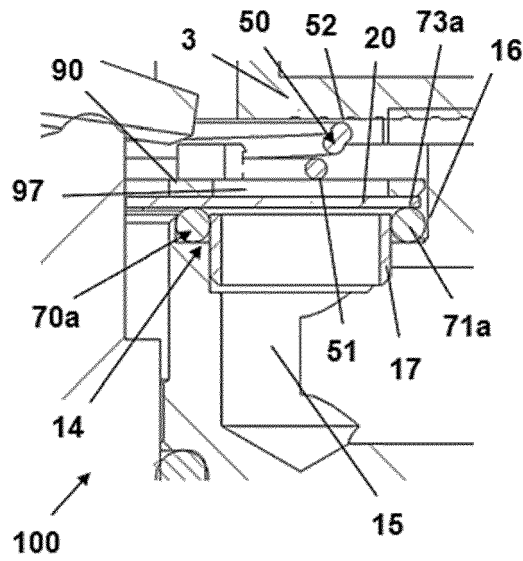


FIG. 9A

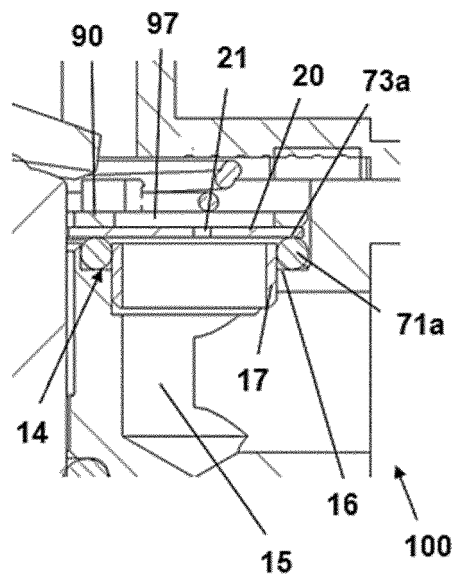


FIG. 9B

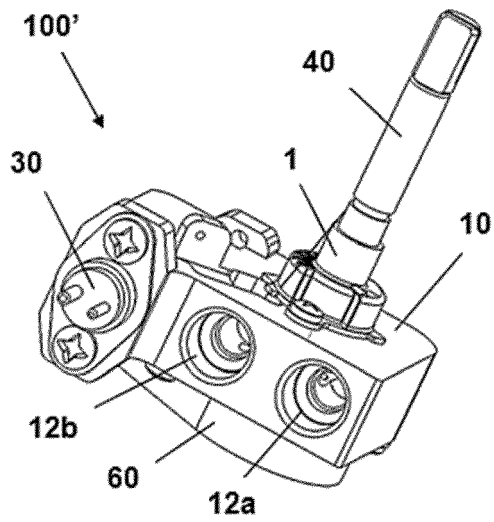


FIG. 10

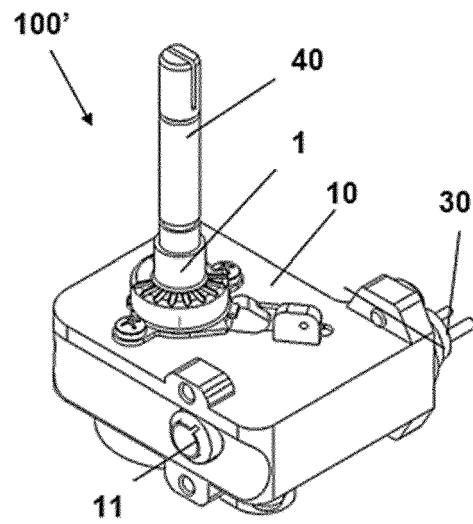


FIG. 11

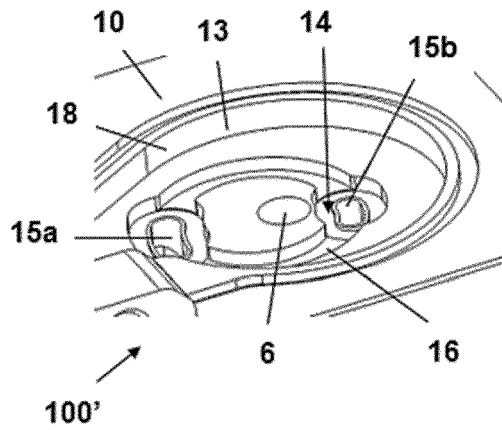


FIG. 12

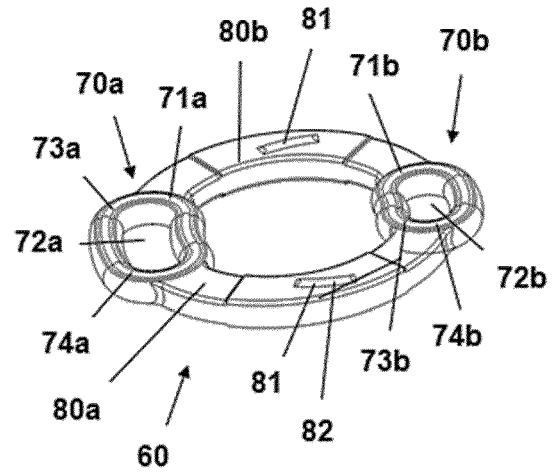


FIG. 13

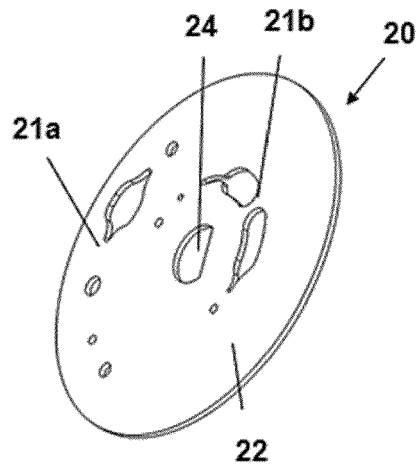


FIG. 14

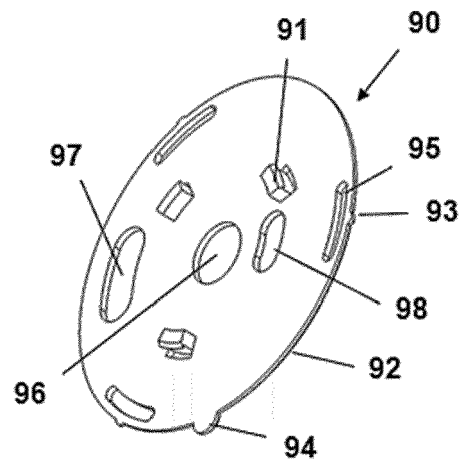


FIG. 15

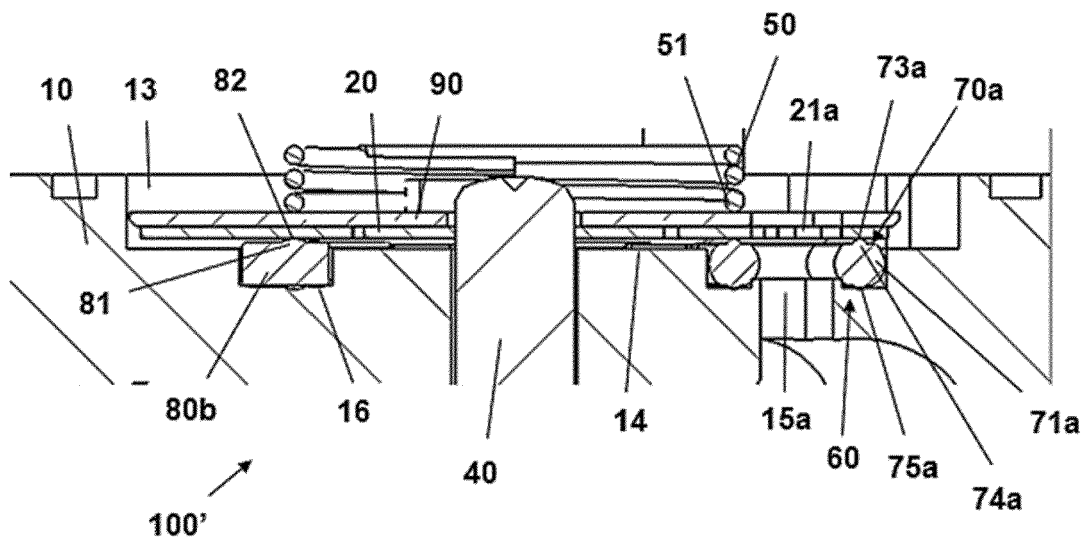


FIG. 16

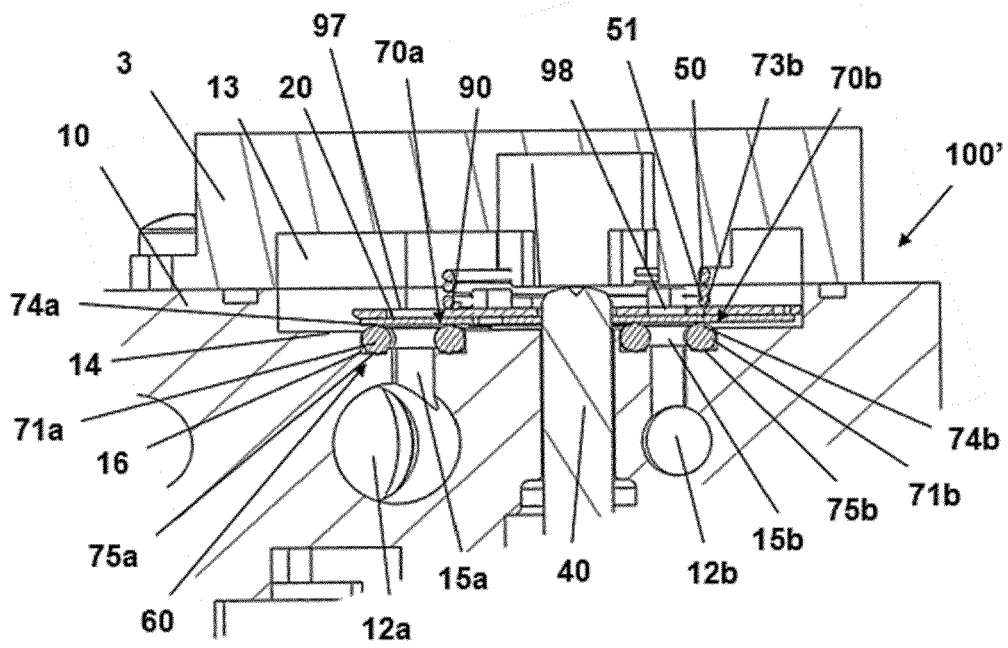


FIG. 17

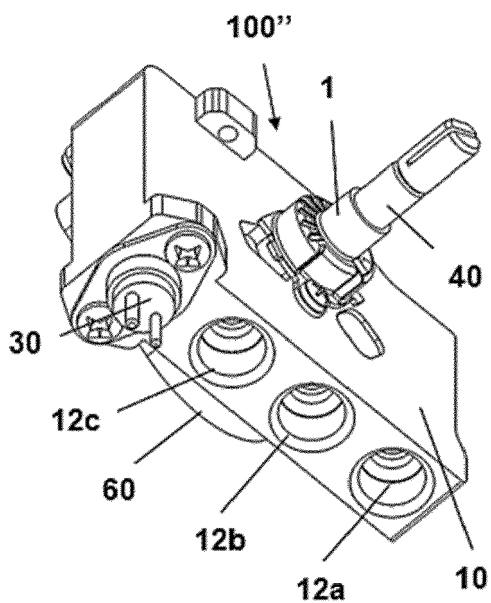


FIG. 18

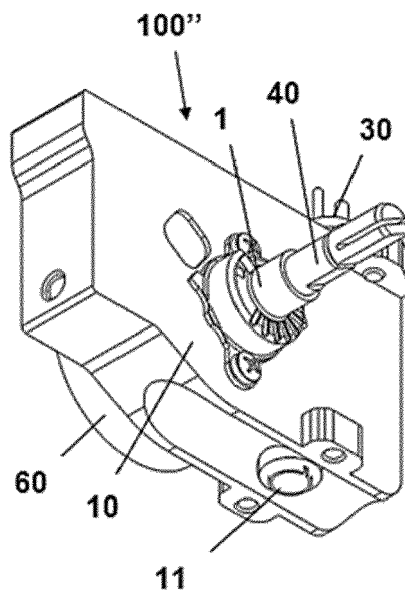


FIG. 19

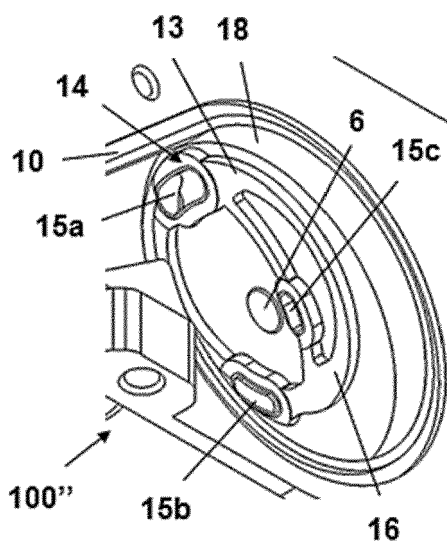


FIG. 20

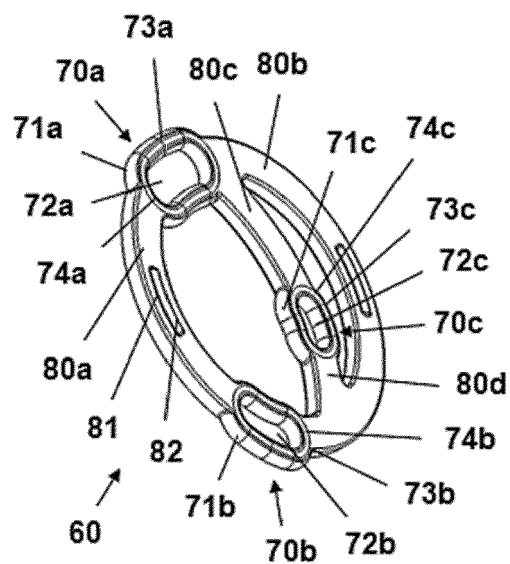


FIG. 21

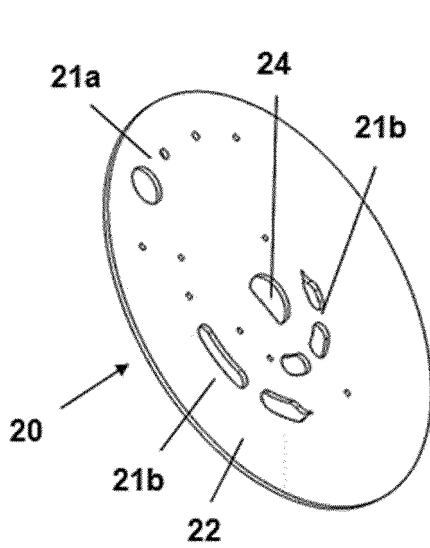


FIG. 22

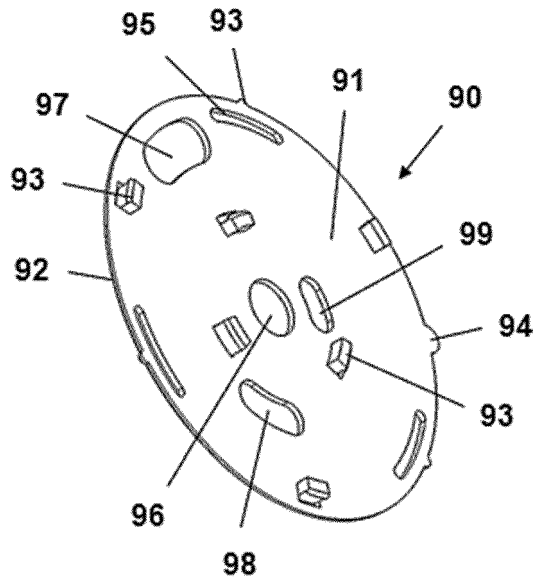


FIG. 23

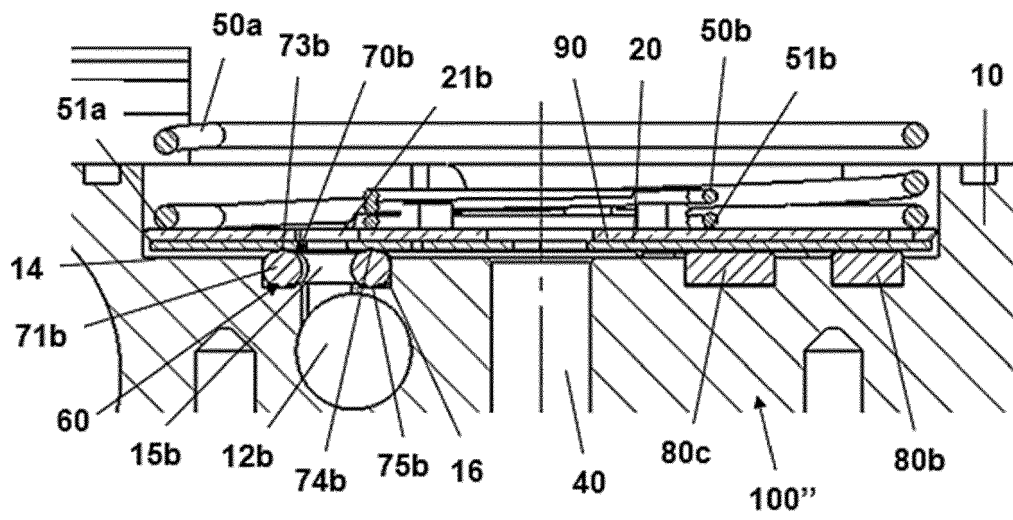


FIG. 24

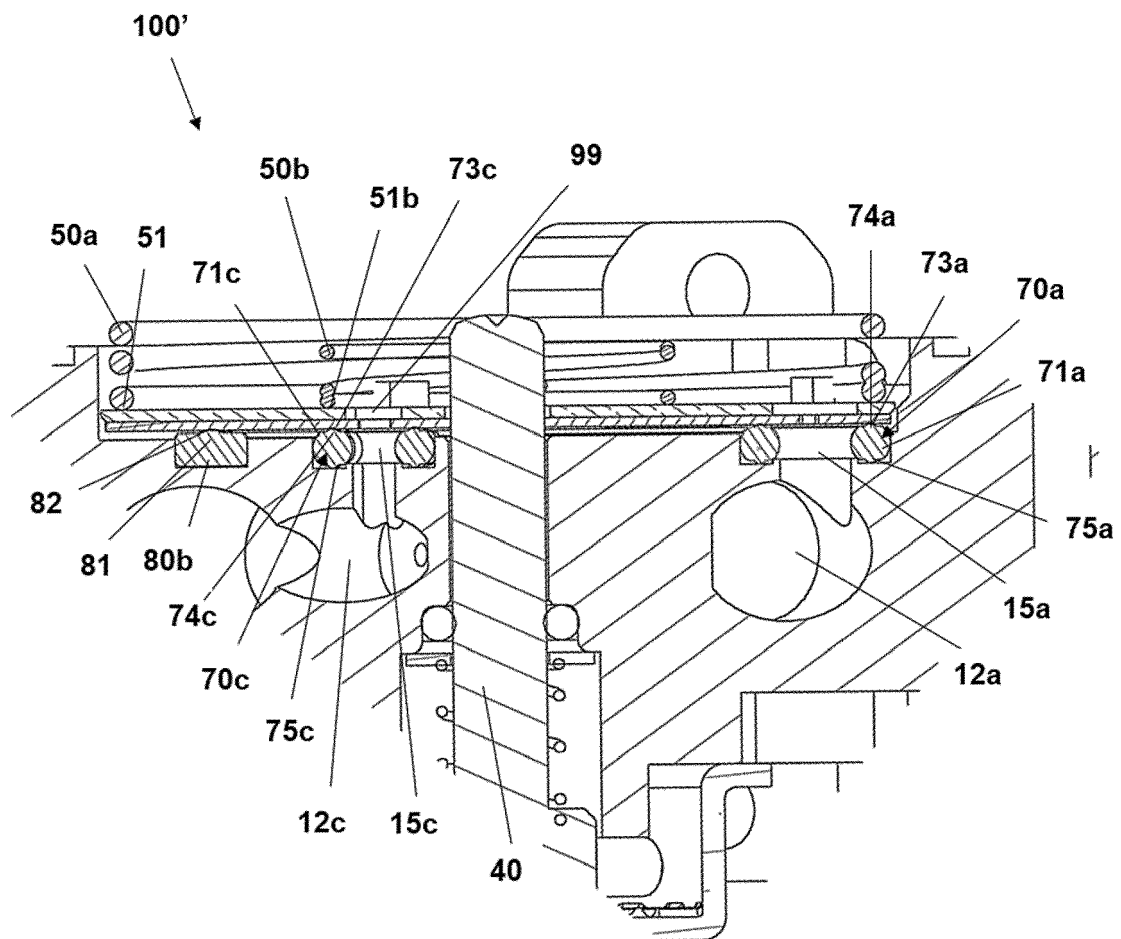


FIG. 25

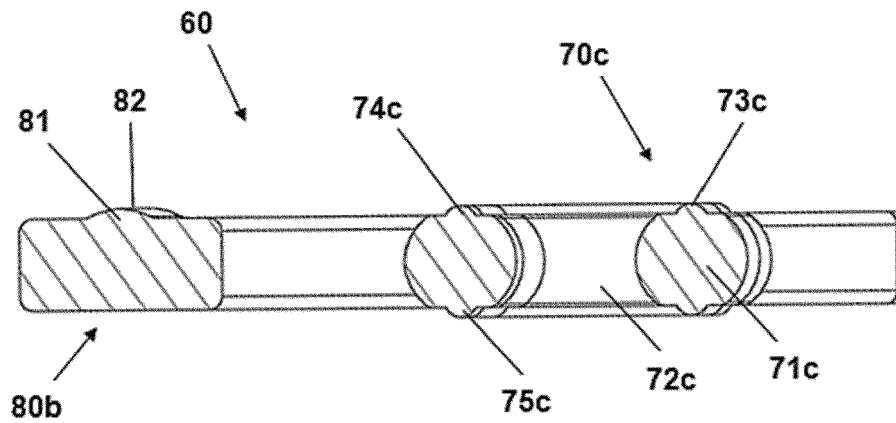


FIG. 26

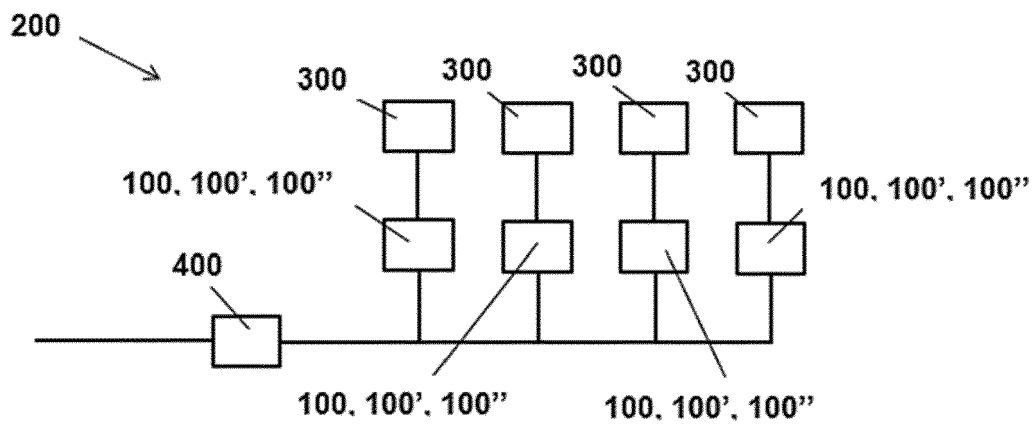


FIG. 27



EUROPEAN SEARCH REPORT

Application Number
EP 20 38 2040

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EPO FORM 1503 03.82 (P04C01)

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (IPC)
Y	US 4 328 832 A (INADA MASAMI ET AL) 11 May 1982 (1982-05-11) * column 2, line 22 - column 4, line 12 * * figures 1-6 *	1-9,14, 15	INV. F23K5/14 F16K3/08
Y	US 2005/092377 A1 (MORK STEVE O [US] ET AL) 5 May 2005 (2005-05-05) * page 1, paragraph 19 - page 3, paragraph 38 * * figures 2-7 *	1-9,14, 15	
A	US 2010/122742 A1 (LIN HSIAO-MEI [TW]) 20 May 2010 (2010-05-20) * page 1, paragraph 16 - page 2, paragraph 23 * * figures 1-5 *	1-14	
A	CN 207 229 799 U (XIAMEN ASLIN RUBBER & PLASTIC TECH CO LTD) 13 April 2018 (2018-04-13) * figures 1-3 and the relating passages of the description *	1-14	
			TECHNICAL FIELDS SEARCHED (IPC)
			F23K F23N F16K
The present search report has been drawn up for all claims			
Place of search Munich		Date of completion of the search 8 July 2020	Examiner Rudolf, Andreas
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