(19)

(12)





EP 3 896 211 A1

EUROPEAN PATENT APPLICATION

- (43) Date of publication: 20.10.2021 Bulletin 2021/42
- (21) Application number: 20169727.3
- (22) Date of filing: 15.04.2020
- (84) Designated Contracting States:
 AL AT BE BG CH CY CZ DE DK EE ES FI FR GB GR HR HU IE IS IT LI LT LU LV MC MK MT NL NO PL PT RO RS SE SI SK SM TR Designated Extension States:
 BA ME Designated Validation States:
 KH MA MD TN
- (71) Applicant: ELECTROLUX APPLIANCES AKTIEBOLAG 105 45 Stockholm (SE)

(51) Int Cl.: D06F 39/04 (2006.01) F28D 1/047 (2006.01) D06F 39/00 (2020.01)

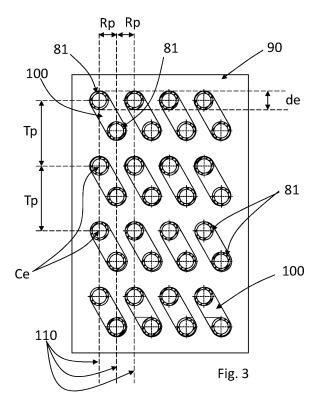
(11)

D06F 58/20 (2006.01) F28F 1/32 (2006.01) F28F 21/08 (2006.01)

- (72) Inventors:
 - CAVARRETTA, Francesco 33080 Porcia (IT)
 - GAMBARRO, Fabio 33080 Porcia (IT)
 - GOBBO, Gianni 33080 Porcia (IT)
- (74) Representative: Electrolux Group Patents AB Electrolux Group Patents S:t Göransgatan 143 105 45 Stockholm (SE)

(54) A LAUNDRY TREATING APPLIANCE HAVING A HEAT PUMP SYSTEM

(57)A laundry treating appliance (10) comprising: a cabinet (20); - a drum (30); - a heat pump system (40), using one or more flammable refrigerants; - a circulating system (60); wherein the heat pump system (40) comprises a heat exchanger (70a, 70b) comprising: - a metallic pipe (80) bent to define a coil structure comprising a plurality of straight sections (81), parallel one another and connected in twos at one end thereof by a plurality of curved sections (82); - a plurality of fins (90), stacked, spaced and parallel to one another, each provided with a plurality of through-holes (100), each through-hole (100) housing two of the straight sections (81), the plurality of straight sections (81) being distributed in such a way that, in a cross section whose cutting plane (Cp) is perpendicular to the straight sections (81), their centers (Ce) are aligned along a plurality of mutually parallel straight lines (110).



Printed by Jouve, 75001 PARIS (FR)

Description

[0001] The present invention refers to a laundry treating appliance, for example a laundry washing machine (called also washing machine), a laundry washer-drier (called also washer-drier), a tumble drier, having a heat pump system.

[0002] Traditional laundry treating appliances, for example washing machines, washer-driers, tumble driers, typically comprise a cabinet containing a rotatable drum wherein the laundry to be treated (i.e. washed and/or dried) can be loaded.

[0003] An operating fluid (e.g., depending on the kind of laundry treating appliance and on the treating process to be applied, water, water mixed with a washing/rinsing additive, air), is circulated through the drum by a circulating system (comprising, for example pumps, valves, fans, etc., depending on the kind of fluid to be circulated). [0004] In some known laundry treating appliances, the

operating fluid is heated and/or cooled/dehumidified by a heat pump system, typically comprising a compressor, an expansion valve, two heat exchangers (one operating as a condenser, and the other as an evaporator), and conduits fluidly connecting such elements in a closed circuit.

[0005] A heat pump system has an improved energy efficiency with respect to traditional heating systems using an electrical heater as heat source.

[0006] Some refrigerant flows through the compressor, the condenser, the expansion valve and the evaporator, and through the conduits connecting these to one another.

[0007] The refrigerant releases heat to the operating fluid by means of the condenser, and extracts heat and humidity from the operating fluid by means of the evaporator. The compressor converts electromechanical power to thermal power by compressing the refrigerant in the refrigerant circuit.

[0008] A known kind of heat exchanger, widely used in heat pump systems of laundry treating appliances, comprises a metallic pipe, made of copper or aluminum, wherein the refrigerant flows, bent to define a coil structure comprising a plurality of straight sections, parallel one another and connected in twos at one end thereof by a plurality of curved sections; this kind of known heat exchanger comprises a plurality of fins, typically made of aluminum and having a rectangular plane, stacked in spaced and parallel planes and comprising a plurality of through-holes, each housing two straight sections of the metallic pipe.

[0009] During the functioning of the heat pump system, the operating fluid flows through the gaps between the fins, exchanging heat with the latter, and therefore with the refrigerant flowing in the metallic pipe, which is thermally connected to the fins. Currently, the refrigerants mainly used in heat pump systems of known laundry treating appliances are hydrofluorocarbon (HFC) refrigerants, in particular the ones known as R134a and

R407C. Unfortunately, these refrigerants have a high Global Warming Potential (GWP), so alternative refrigerants start to be more and more used in different industries.

- ⁵ **[0010]** Possible alternative refrigerants used for replacing hydrofluorocarbon (HFC) refrigerants in heat pump systems of laundry treating appliances are hydrocarbons refrigerants, such as propane (R290) and propylene (R1270).
- 10 [0011] These alternative refrigerants have a negligible impact on GWP and their thermo-physical properties makes them very suitable for the typical working conditions of heat pump systems of laundry treating appliances, in particular tumble driers and washer-driers.
- ¹⁵ [0012] The downside of these alternative refrigerants is that they are flammable, and therefore, for limiting possible risks, regulations (e.g. the IEC 60335-2-11 standard) limit the amount of refrigerant that can be charged in the heat pump system to 150 g (grams).
- 20 [0013] Inside the heat pump system, when the compressor is switched ON, most of the refrigerant can be found inside the condenser, since in this heat exchanger the refrigerant is at high pressure and, for a portion thereof, in liquid state, so with a very high density.
- ²⁵ [0014] The evaporator, on the contrary, works at low pressure, and the refrigerant contained therein is mainly a liquid-vapour mixture and a superheated vapour, so its density is quite low. It has been observed that limiting to 150 g the refrigerant charge could negatively affect the
 ³⁰ performances of the heat pump system, in particular its

performances of the heat pump system, in particular its energy efficiency.

[0015] There is the need, therefore, to reduce the volume of the components of the heat exchangers wherein the refrigerant flows, so as to limit the refrigerant charge required by the system. On the other hand, it's important not reducing too much the external surface area, so as to keep a good heat exchange performance.

[0016] In particular, in order to reduce the overall volume of the single duct of the heat exchanger wherein the
 refrigerant flows, there could be the possibility to reduce the number of stacked fins, and therefore the overall length of the metallic pipe; unfortunately, reducing the number of fins reduces also the overall thermal exchange surface of the heat exchanger, which reduces the energy

⁴⁵ efficiency of the heat pump system. The aim of the present invention is therefore obtaining a laundry treating appliance using a heat pump system having a reduced Global Warming Potential (GWP) and an improved efficiency.

50 [0017] Within this aim, a further object of the invention is obtaining a laundry treating appliance fulfilling the safe-ty regulations related to the refrigerants of the heat pump systems, without reducing the overall energy efficiency. Applicant has found that, by selecting, for specific ranges
 55 of external diameters of the metallic pipe of a finned heat exchanger which stacked fins have a plurality of throughholes, each housing two mutually parallel straight sections of the metallic pipe, the straight sections being dis-

35

15

20

tributed in such a way that, in a cross section whose cutting plane is perpendicular to the straight sections, their centers are aligned along a plurality of mutually parallel straight lines, specific ranges for the distances of the straight lines of adjoining straight lines, and corresponding specific ranges for the distances between the centers of couples of adjoining through-holes aligned along a same straight line, it is possible to reduce the overall internal volume of the metallic pipe of the heat exchanger wherein the refrigerant flows without reducing the length of the metallic pipe nor the number of fins, and therefore without reducing the overall thermal exchange surface.

[0018] This inventive solution allows using in the heat pump system of a laundry appliance a flammable refrigerant, like for example propane (R290) or propylene (R1270), which have a very low Global Warming Potential (GWP), but that needs to be used in small quantities, and therefore requires a reduced volume of the metallic pipe of the heat exchanger wherein the refrigerant flows. [0019] In particular, above aim is solved by a laundry treating appliance comprising:

- a cabinet;
- a drum, rotatably housed within the cabinet, in which laundry can be loaded;
- a heat pump system, using one or more flammable refrigerants, configured for exchanging heat with an operating fluid;
- a circulating system configured for circulating the operating fluid through the drum;

wherein the heat pump system comprises a heat exchanger comprising:

- a metallic pipe, wherein the flammable refrigerant flows, bent to define a coil structure comprising a plurality of straight sections, parallel one another and connected in twos at one end thereof by a plurality of curved sections;
- a plurality of fins, stacked spaced and parallel to one another, each provided with a plurality of throughholes, each through-hole housing two of the straight sections.

the plurality of straight sections being distributed in such a way that, in a cross section whose cutting plane is perpendicular to the straight sections, their centers are aligned along a plurality of mutually parallel straight lines, wherein,

if the external diameter of the metallic pipe is comprised between 5 and 9 mm, then:

the distance between the centers of two adjoining through-holes aligned along a same straight line among the mutually parallel straight lines is comprised between 30 and 40 mm, and the distance between two adjoining straight lines among the mutually parallel straight lines is comprised between 12 and 50 mm,

or

the distance between the centers of two adjoining through-holes aligned along a same straight line among the mutually parallel straight lines is comprised between 40 and 70 mm, and the distance between two adjoining straight lines among the mutually parallel straight lines is comprised between 8 10 and 50 mm.

[0020] It is underlined that stating that two straight lines are adjoining (or in other words contiguous, or neighboring, or adjacent) means that there aren't other straight lines positioned between such two straight lines.

[0021] It is underlined that stating that two throughholes are *adjoining* (or in other words contiguous, or neighboring, or adjacent) means that there aren't other through-holes positioned between such two throughholes.

[0022] Using the specific inventive ranges of distances between straight lines/through-holes, in combination with the specific inventive ranges of the external diameter of the metallic pipe, allows keeping reduced the internal vol-

25 ume of heat exchanger wherein the flammable refrigerant flows, without significantly affecting the overall dimensions of the stacked fins, and therefore without significantly affecting the heat exchange performances.

[0023] In addition, the specific combinations of inven-30 tive ranges allow obtaining a uniform heat distribution along the fins, which avoids overheating of some parts of the latter, and also improves the heat exchange.

[0024] In an advantageous embodiment, if the external diameter of the plurality of straight sections of the metallic pipe is comprised between 5 and 9 mm, then the distance

35 between the centers of two adjoining through-holes aligned along a same straight line among the mutually parallel straight lines is comprised between 30 and 40 mm, and the distance between two adjoining straight 40 lines among the mutually parallel straight lines is com-

prised between 12 and 35 mm. [0025] In a further advantageous embodiment, if the

external diameter of the plurality of straight sections of the metallic pipe is comprised between 5 and 9 mm, then the distance between the centers of two adjoining

45 through-holes aligned along a same straight line among the mutually parallel straight lines is comprised between 40 and 55 mm, and the distance between two adjoining straight lines among the mutually parallel straight lines 50 is comprised between 8 and 12 mm.

[0026] In a preferred embodiment, the distance between two adjoining straight lines among the mutually parallel straight lines is the same for all the couples of adjoining straight lines among the mutually parallel straight lines.

[0027] Preferably, the distance between the centers of two adjoining through-holes aligned along a same straight line among the mutually parallel straight lines is

55

the same for all the couples of adjoining through-holes aligned along this same straight line. More preferably, the distance between the centers of two adjoining through-holes aligned along a same straight line among the mutually parallel straight lines is the same for all the mutually parallel straight lines.

[0028] In a preferred embodiment, the metallic pipe comprises an inlet portion and an outlet portion protruding both from a same terminal fin of the plurality of stacked fins.

[0029] Preferably, the flammable refrigerant is or comprises a hydrocarbon. More preferably, the flammable refrigerant is or comprises propane (R290) or propylene (R1270).

[0030] Advantageously, the plurality of fins is made of, or comprise, aluminum or aluminum alloy, or copper, or copper alloy.

[0031] In an advantageous embodiment, the perimeter edges of the stacked fins define as a whole an envelope surface comprising at least a plane portion), and the mutually parallel straight lines are perpendicular to the at least a plane portion. Preferably, fins of the plurality of metallic fins have a width comprised between 65 mm and 145 mm, and a height comprised between 110 mm and 185 mm.

[0032] More preferably, fins of the plurality of metallic fins have a width comprised between 95 mm and 125 mm, and a height comprised between 145 mm and 165 mm.

[0033] Preferably, the overall length of the stacked fins is comprised between 330 mm and 370 mm.

[0034] More preferably, the overall length of the stacked fins is comprised between 200 mm and 250 mm. [0035] Preferably, the metallic pipe is made of, or comprises, aluminum or aluminum alloy, or copper or copper alloy.

[0036] In an advantageous embodiment, the laundry treating appliance is a tumble drier or a washer-drier, and the operating fluid is air.

[0037] In a further advantageous embodiment, the laundry treating appliance is a laundry washing machine, and the operating fluid is water, or water mixed with a washing/rinsing agent. Other advantages and features of a laundry treating appliance according to the present invention will be clear from the following detailed description, provided only as a not limitative example, in which:

Fig 1 is a schematic lateral cross section of a laundry treating appliance, in particular a tumble drier, according to the invention;

Fig. 2 is a schematic lateral view of a heat exchanger of a laundry treating appliance according to the invention:

Fig. 3 is a schematic a cross section operated according to plane Cp-Cp Fig. 2;

Fig. 4 is a rear view of the heat exchanger of Fig. 2; Fig. 5 is a schematic front view of a fin of the heat exchanger of the previous figures; Fig. 6 is a schematic lateral view of the metallic pipe of the heat exchanger of the previous figures; Fig. 7 is a lateral schematic view of the stacked fins of a heat exchanger according to the invention;

Fig. 8 is a further schematic a cross section operated according to plane Cp-Cp Fig. 2.

[0038] In the figures, same parts are indicated with the same reference numbers.

10 [0039] Advantageously, the laundry treating appliance 10 illustrated in figure 1 is a tumble drier of the "horizontal axis type"; it is however clear that the invention can be applied, without any substantial modification, also to tumble driers of the vertical axis" type, and to washing ma-

¹⁵ chines and washer driers, both of the "horizontal axis" and of the "vertical axis" type.

[0040] The laundry treating appliance (being it a tumble drier 10, or a washing machine or washer-drier, not illus-trated) comprises a cabinet 20, or housing, preferably

²⁰ parallelepiped, configured to be positioned on a horizontal surface 2, for example the floor of a building, preferably by suitable feet 21, one or more of which can have, advantageously, an adjustable height, so as to adapt to a possible not perfect planarity of the horizontal surface 2.

²⁵ [0041] Advantageously, in the frontal wall 20a of the cabinet 20 an access opening, not illustrated, is preferably obtained, advantageously selectively closable by a loading/unloading door 4, preferably hinged to the frontal wall 20a.

30 [0042] The laundry treating appliance (being it a tumble drier 10, or a washing machine or washer-drier) comprises a drum 30 rotatably housed within the cabinet 20, in which the laundry, not illustrated, can be loaded.

[0043] If the laundry treating appliance is a washing machine or a washer-drier, both not illustrated, the cabinet 20 also houses a washing tub, not illustrated, preferably suspended to the cabinet through springs and dumpers, also not illustrated, in which the drum 30 is rotatably contained.

40 **[0044]** The laundry treating appliance 10 comprises a circulating system 60 configured for circulating an operating fluid through the drum 3.

[0045] It is underlined that the circulating system 60 can define a closed circuit for the operating fluid (i.e. the

- ⁴⁵ operating fluid remains within the closed circuit during the laundry treating process, and the same fluid, opportunely treated, passed repeatedly through the drum 30), or it can define an opened circuit for the operating fluid (i.e. the operating fluid is loaded within the laundry treat-
- ⁵⁰ ing appliance 10 at a certain point of the laundry treating process, and it is drained from the laundry treating appliance 10 at another point of the laundry treating process).

[0046] In the advantageous embodiment in which the laundry treating appliance 10 is a tumble drier, like the advantageous example of figure 1, or a washer drier, not illustrated, the operating fluid is or comprises air (represented by arrows 50), and the circulating system preferably comprises an air circuit 61 and one or more fans 62 configured for circulating such air 50 through the drum 3 and the air circuit 61.

[0047] If the laundry treating appliance is a tumble drier 10, it can also advantageously comprise a lint filter 63, arranged in the air circuit 61 for trapping lint or fluff released from the laundry.

[0048] If the laundry treating appliance is a washing machine or a washer drier, both not illustrated, the operating fluid is or comprises water, or water mixed with a washing/rinsing additive, and the circulating system preferably comprises a water inlet circuit, not illustrated, adapted to feed water into the tub, also not illustrated, and a drain circuit, also not illustrated, adapted for draining washing/rinsing liquid from the machine.

[0049] The laundry treating appliance 10 advantageously comprises a heat pump system 40, configured for heating the operating fluid, for example, in case of a tumble drier, the air 50. Advantageously, the heat pump system 40 can also be configured for cooling and dehumidifying the operating fluid.

[0050] Preferably, the heat pump system 40 comprises a compressor, not illustrated, an expansion valve, also not illustrated, two heat exchangers 70a, 70b (one operating as a condenser, and the other as an evaporator), and conduits, not illustrated, fluidly connecting such elements in a closed circuit.

[0051] A flammable refrigerant flows through the compressor, the condenser 70a, the expansion valve and the evaporator 70b, and through the conduits connecting these to one another.

[0052] The flammable refrigerant releases heat to the operating fluid by means of the condenser 70a and extracts heat and humidity from the operating fluid by means of the evaporator 70b. The compressor converts electromechanical power to thermal power by compressing the flammable refrigerant in the refrigerant circuit.

[0053] The flammable refrigerant is or comprises a hydrocarbon, preferably propane (R290) or propylene (R1270).

[0054] The heat exchangers, for example the condenser 70a and/or the evaporator 70b, comprise a metallic pipe 80, wherein the flammable refrigerant flows, bent to define a coil structure comprising a plurality of straight sections 81, parallel one another and connected in twos at one end thereof by a plurality of curved sections 82.

[0055] Advantageously, the metallic pipe 80 is made of, or comprise, aluminum or aluminum alloy, or copper or copper alloy.

[0056] The straight sections 81 are distributed in such a way that, in a cross section which cutting plane Cp (illustrated in Fig. 2) is perpendicular to the straight sections 81, their centers Ce are aligned along a plurality of mutually parallel straight lines 110.

[0057] The heat exchangers, for example the condenser 70a and/or the evaporator 70b, comprise a plurality of fins 90, stacked spaced and parallel to one another, each provided with a plurality of through-holes 100, each

through-hole 100 housing two of the straight sections 81 of the metallic pipe 80. Advantageously, the fins 90 are made of, or comprise, aluminum or aluminum alloy, or copper, or copper alloy.

⁵ **[0058]** Advantageously, the through-holes 100 of any fin 90 are respectively aligned with the through-holes 100 of the rest of the stacked fins 90.

[0059] Advantageously, the straight sections 81 are fitted, with their lateral surface into close contact with the

¹⁰ border of the respective through-holes 100, so as to obtain an effective heat-exchange between them.
 [0060] Advantageously, the metallic pipe 80 comprises an inlet portion 84 and an outlet portion 85, configured for allowing the flammable refrigerant respectively to en-

¹⁵ ter/exit the metallic pipe 80; advantageously, the inlet portion 84 and the outlet portion 85 are fluidly connected or connectable to the other elements of the heat pump system 40, so as to allow circulation of the flammable refrigerant through the respective heat exchanger 70a or 70b.

[0061] Advantageously, like in the examples of attached figures, the inlet portion 84 and an outlet portion 85 protrude both from a same terminal fin 91 of the plurality of stacked fins 90, which simplifies the connection

of the single duct 83 to the other conduits of the heat pump system 40.

[0062] Advantageously, the perimeter edges 99 of the stacked fins 90 define as a whole an envelope surface, illustrated in figures 7 and 8 with a dotted line numbered 92, comprising at least a plane portion 93.

[0063] Preferably, the fins 90 have a rectangular or square plan, in which case the envelope surface 92 comprises four plane portions, corresponding to the four sides of the rectangle or square.

³⁵ [0064] In advantageous embodiments, like for example the ones illustrated in attached figures, in which the fins 90 have a rectangular or square plan, and the envelope surface 92 comprises four plane portions corresponding to the four sides of the rectangle or square,
 40 above defined straight lines 110 are advantageously par-

above defined straight lines 110 are advantageously parallel to two sides of such rectangle or square.
[0065] Preferably, the width "w" of the fins 90 is comprised between 65 mm and 145 mm, more preferably between 95 mm and 125 mm.

⁴⁵ [0066] Preferably, the height "h" of the fins 90 is comprised between 110mm and 185 mm, more preferably between 145 mm and 165 mm. Preferably, the overall length of the stacked fins 90 is comprised between 330 mm and 370 mm, more preferably between 200 mm and 50 250 mm.

[0067] According to the invention, if the external diameter de of the metallic pipe 80, and in particular of its straight portions 81 is comprised between 5 and 9 mm, then:

55

30

 the distance Tp between the centers Ce of two adjoining through-holes 100 aligned along one of the mutually parallel straight lines 110 is comprised between 30 and 40 mm, and the distance Rp between two adjoining straight lines 110 among the mutually parallel straight lines 110 is comprised between 12 and 50 mm, or

 the distance Tp between the centers Ce of two adjoining through-holes 100 aligned along one of the mutually parallel straight lines 110 is comprised between 40 and 70 mm, and the distance Rp between two adjoining straight lines 110 among the mutually parallel straight lines 110 is comprised between 8 10 and 50 mm.

[0068]In an advantageous embodiment, if the external
diameter de of the plurality of straight sections 81 of the
metallic pipe 80 is comprised between 5 and 9 mm, then15the distance Tp between the centers Ce of two adjoining
through-holes 100 aligned along one of the mutually par-
allel straight lines 110 is comprised between 30 and 40
mm, and the distance Rp between two adjoining straight
lines 110 among the mutually parallel straight lines 1102020is comprised between 12 and 35 mm.20

[0069] In a further advantageous embodiment, if the external diameter de of the plurality of straight sections 81 of the metallic pipe 80 is comprised between 5 and 9 mm, then the distance Tp between the centers Ce of two adjoining through-holes 100 aligned along one of the mutually parallel straight lines 110 is comprised between 40 and 55 mm, and the distance Rp between two adjoining straight lines 110 among the mutually parallel straight lines 110 is comprised between 8 and 12 mm. 30

[0070] In another advantageous embodiment, the distance Rp between two adjoining straight lines 110 among the mutually parallel straight lines 110 is the same for all the couples of adjoining straight lines 110 among the mutually parallel straight lines 110.

[0071] Preferably, the distance Tp between the centers Ce of two adjoining through-holes 100 aligned along one of the mutually parallel straight lines 110 is the same for all the couples of adjoining through-holes 100 aligned along a same straight line 110.

[0072] More preferably, the distance Tp between the centers Ce of two adjoining through-holes 100 aligned along a same straight line 110 is the same for all the straight line 110.

[0073] It is seen therefore how the invention achieves ⁴⁵ the proposed aim and objects, since it allows obtaining an evaporator for the heat pump system of a laundry treating appliance having a relatively small volume of the metallic pipe wherein the refrigerant flows, and a high overall thermal exchange surface; this evaporator allows ⁵⁰ using in the heat pump system a flammable refrigerant, like for example propane (R290) or propylene (R1270), which have a very low Global Warming Potential (GWP), fulfilling the regulation requirements related to flammable refrigerant charge, and keeping at the same time a high ⁵⁵ energetic efficiency.

Claims

- **1.** A laundry treating appliance (10) comprising:
 - a cabinet (20);

- a drum (30), rotatably housed within said cabinet (20), in which laundry can be loaded;
- a heat pump system (40), using one or more flammable refrigerants, configured for exchanging heat with an operating fluid (50);
- a circulating system (60) configured for circulating said operating fluid (50) through said drum (30);

wherein said heat pump system (40) comprises a heat exchanger (70a, 70b) comprising:

- a metallic pipe (80), wherein said flammable refrigerant flows, bent to define a coil structure comprising a plurality of straight sections (81), parallel one another and connected in twos at one end thereof by a plurality of curved sections (82);

- a plurality of fins (90), stacked spaced and parallel to one another, each provided with a plurality of through-holes (100), each through-hole (100) housing two of said straight sections (81),

said plurality of straight sections (81) being distributed in such a way that, in a cross section whose cutting plane (Cp) is perpendicular to said straight sections (81), their centers (Ce) are aligned along a plurality of mutually parallel straight lines (110),

characterized in that

if the external diameter (de) of said metallic pipe (80) is comprised between 5 and 9 mm, then:

- the distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is comprised between 30 and 40 mm, and the distance (Rp) between two adjoining straight lines (110) among said mutually parallel straight lines (110) is comprised between 12 and 50 mm,

or

- the distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is comprised between 40 and 70 mm, and the distance (Rp) between two adjoining straight lines (110) among said mutually parallel straight lines (110) is comprised between 8 and 50 mm.

2. A laundry treating appliance (10), according to claim 1, wherein,

if the external diameter (de) of said metallic pipe (80)

35

40

10

15

20

30

35

40

45

is comprised between 5 and 9 mm,

then said distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is comprised between 30 and 40 mm, and said distance (Rp) between two adjoining straight lines (110) among said mutually parallel straight lines (110) is comprised between 12 and 35 mm.

3. A laundry treating appliance (10), according to claim 1, wherein,

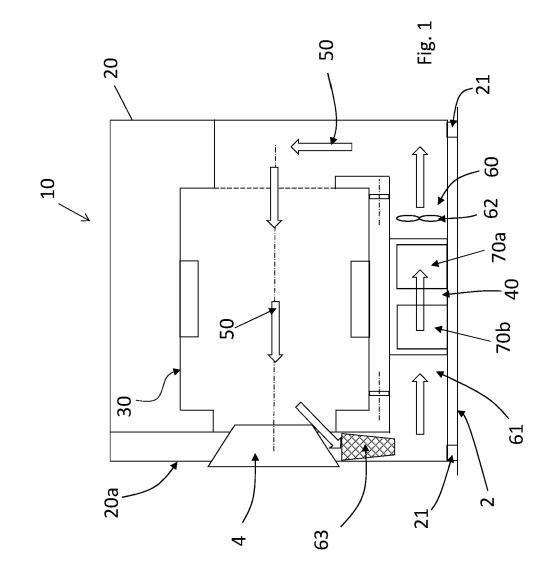
if the external diameter (de) of said metallic pipe (80) is comprised between 5 and 9 mm,

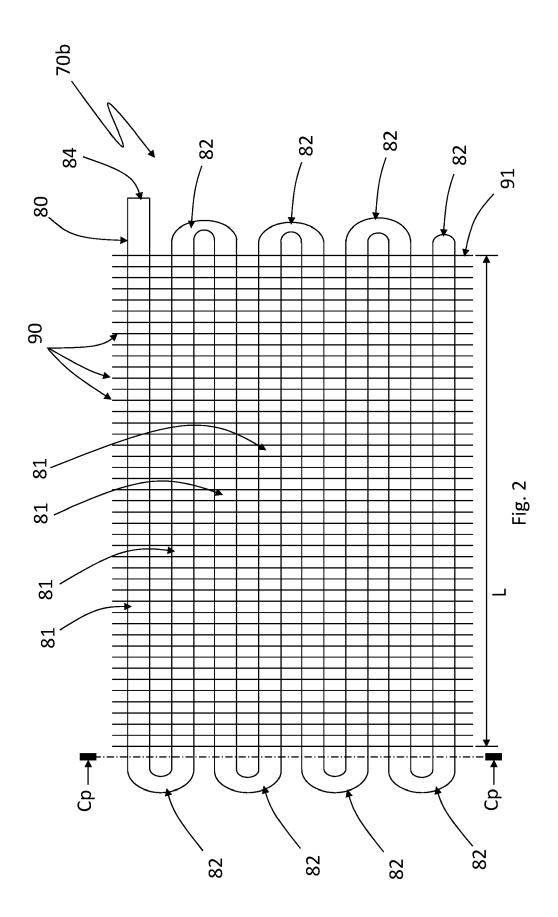
then said distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is comprised between 40 and 55 mm, and said distance (Rp) between two adjoining straight lines (110) among said mutually parallel straight lines (110) is comprised between 8 and 12 mm.

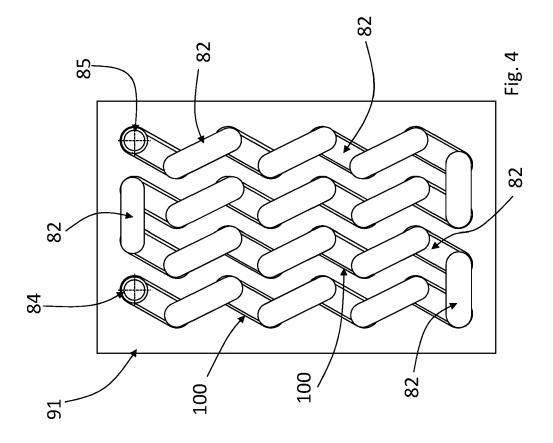
- 4. A laundry treating appliance (10) according to one or more of the previous claims, wherein said distance (Rp) between two adjoining straight lines (110) among said mutually parallel straight lines (110) is the same for all the couples of adjoining straight lines (110) among said mutually parallel straight lines (110) among said mutually parallel straight lines (110).
- A laundry treating appliance (10) according to one or more of the previous claims, wherein said distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is the same for all the couples of adjoining through-holes (100) aligned along said same straight line (110).
- 6. A laundry treating appliance (10) according to claim 5, wherein said distance (Tp) between the centers (Ce) of two adjoining through-holes (100) aligned along a same straight line (110) among said mutually parallel straight lines (110) is the same for all said mutually parallel straight lines (110).
- A laundry treating appliance (10) according to one or more of the previous claims, wherein said metallic pipe (80) comprises an inlet portion (84) and an outlet 50 portion (85) protruding both from a same terminal fin (91) of said plurality of stacked fins (90).
- A laundry treating appliance (10) according to one or more of the previous claims, wherein said flammable refrigerant is or comprises a hydrocarbon.
- **9.** A laundry treating appliance (10) according to claim

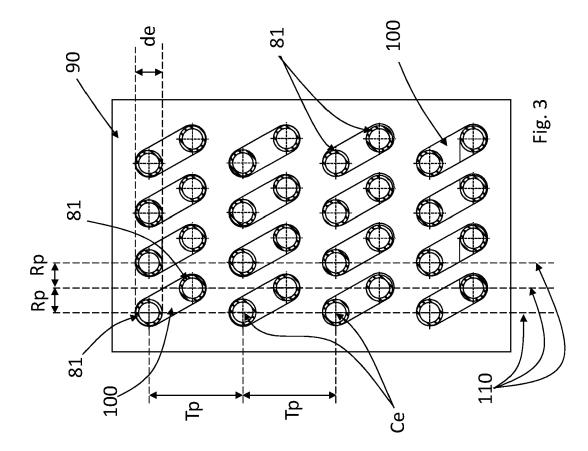
8, wherein said flammable refrigerant is or comprises propane (R290) or propylene (R1270).

- **10.** A laundry treating appliance (10) according to one or more of the previous claims, wherein said plurality of fins (90) are made of, or comprise, aluminum or aluminum alloy, or copper, or copper alloy.
- **11.** A laundry treating appliance (10) according to one or more of the previous claims, wherein the perimeter edges (99) of said stacked fins (90) define as a whole an envelope surface (92) comprising at least a plane portion (93), and wherein said mutually parallel straight lines (110) are perpendicular to said at least a plane portion (93).
 - **12.** A laundry treating appliance (10) according to one or more of the previous claims, wherein fins of said plurality of fins (90) have a width (w) comprised between 65 mm and 145 mm, preferably between 95 mm and 125 mm, and a height (h) comprised between 110 mm and 185 mm, preferably between 145 mm and 165 mm.
- A laundry treating appliance (10) according to one or more of the previous claims, wherein the overall length (L) of said stacked fins (90) is comprised between 330 mm and 370 mm, preferably between 200 mm and 250 mm.
 - **14.** A laundry treating appliance (10) according to one or more of the previous claims, wherein said metallic pipe (80) is made of, or comprises, aluminum or aluminum alloy, or copper or copper alloy.
 - **15.** A laundry treating appliance (10) according to one or more of the previous claims, wherein said laundry treating appliance (10) is a tumble drier (11) or a washer-drier, and wherein said operating fluid (50) is air or, wherein said laundry treating appliance (10) is a laundry washing machine, and said operating fluid (50) is water, or water mixed with a washing/rinsing agent.









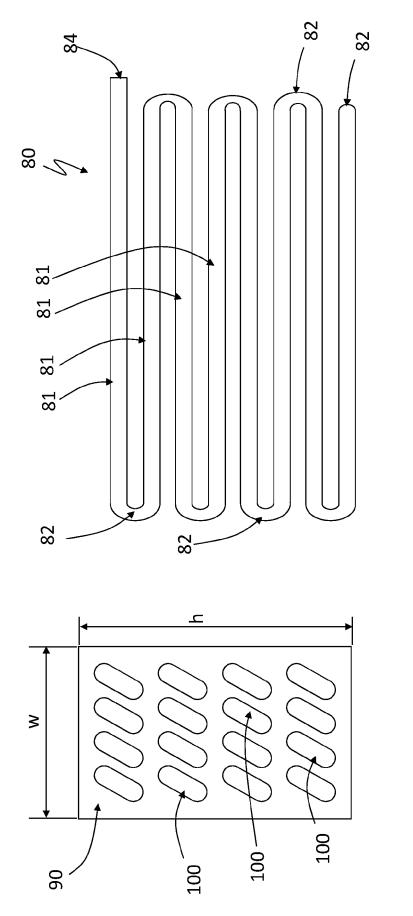
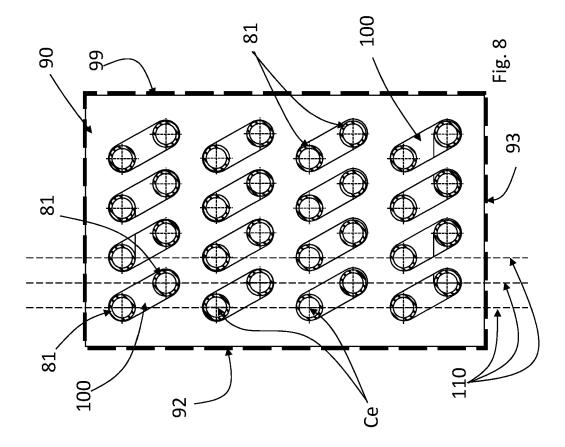
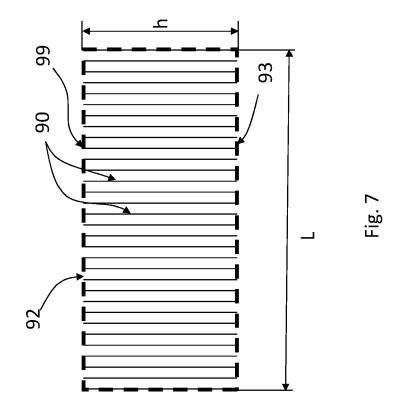


Fig. 6









EUROPEAN SEARCH REPORT

Application Number EP 20 16 9727

X EP 3 066 406 A1 (BSH HAUSGERÄTE GMBH [DE]) 14 September 2016 (2016-09-14) 1-15 INV. D06F39/04 * figures 2-3, 6-8, 10 * * paragraph [0001] - paragraph [0002] * F28D1/047 * paragraph [0055] - paragraph [0057] * * paragraph [0056] * F28D1/047 * paragraph [0056] - paragraph [0057] * * paragraph [0060] - paragraph [0057] * ADD. A EP 3 617 393 A1 (ELECTROLUX APPLIANCES AB [SE]) 4 March 2020 (2020-03-04) 1-15 I-15 A DOMANSKI P A ET AL: "Performance of a finned-tube evaporator optimized for different refrigerants and its effect on system efficiency", INTERNATIONAL JOURNAL OF REFRIGERATION, ELSEVIER, AMSTERDAM, NL, vol. 28, no. 6, 1 September 2005 (2005-09-01), pages 820-827, XP004998949, ISSN: 0140-7007, D01: 10.1016/J.JJREFRIG.2005.02.003 * table 2 * 1-15 A CN 108 691 179 A (BSH HAUSGERAETE GMBH) 23 October 2018 (2018-10-23) * paragraph [0052] - paragraph [0053] * 1-15	Category	Citation of document with in of relevant pass	ndication, where appropriate, ages	Relevant to claim	CLASSIFICATION OF APPLICATION (IPC)	
 A EP 3 617 393 A1 (ELECTROLUX APPLIANCES AB [SE]) 4 March 2020 (2020-03-04) * figures 1-3a * A DOMANSKI P A ET AL: "Performance of a finned-tube evaporator optimized for different refrigerants and its effect on system efficiency", INTERNATIONAL JOURNAL OF REFRIGERATION, ELSEVIER, AMSTERDAM, NL, vol. 28, no. 6, 1 September 2005 (2005-09-01), pages 820-827, XP004998949, ISSN: 0140-7007, DOI: 10.1016/J.IJREFRIG.2005.02.003 * table 2 * A CN 108 691 179 A (BSH HAUSGERAETE GMBH) 23 October 2018 (2018-10-23) 	X	14 September 2016 (* figures 2-3, 6-8, * paragraph [0001] * paragraph [0029] * paragraph [0055] * paragraph [0060]	(2016-09-14) 10 * - paragraph [0002] * - paragraph [0031] * - paragraph [0057] *	1-15	D06F39/04 D06F58/20 F28D1/047 F28F1/32 ADD. D06F39/00	
finned-tube evaporator optimized for different refrigerants and its effect on system efficiency", INTERNATIONAL JOURNAL OF REFRIGERATION, ELSEVIER, AMSTERDAM, NL, vol. 28, no. 6, 1 September 2005 (2005-09-01), pages 820-827, XP004998949, ISSN: 0140-7007, DOI: 10.1016/J.IJREFRIG.2005.02.003 * table 2 *TECHNICAL FIELDS SEARCHED (IPCACN 108 691 179 A (BSH HAUSGERAETE GMBH) 23 October 2018 (2018-10-23)1-15	A	[SE]) 4 March 2020		1-15	F28F21/08	
820-827, XP004998949, SEARCHED (IPC) ISSN: 0140-7007, DOI: D06F 10.1016/J.IJREFRIG.2005.02.003 F28F * table 2 * F28D A CN 108 691 179 A (BSH HAUSGERAETE GMBH) 1-15 23 October 2018 (2018-10-23) 1-15	A	finned-tube evapora different refrigera system efficiency", INTERNATIONAL JOURN ELSEVIER, AMSTERDAN vol. 28, no. 6,	ator optimized for ants and its effect on NAL OF REFRIGERATION, 1, NL,	1-15		
23 October 2018 (2018-10-23)		820-827, XP00499894 ISSN: 0140-7007, DC 10.1016/J.IJREFRIG.	49,)I:		SEARCHED (IPC) D06F F28F	
	A	23 October 2018 (20	018-10-23)	1-15		
			•			
The present search report has been drawn up for all claims		Place of search Munich			Examiner Werner, Christophe	
Place of search Date of completion of the search Examiner	X : parl Y : parl doci	ATEGORY OF CITED DOCUMENTS ioularly relevant if taken alone ioularly relevant if combined with anot ument of the same category nological background	T : theory or principl E : earlier patent do after the filing dat D : document oited fu L : document cited fu	T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document oited in the application L : document oited for other reasons & : member of the same patent family, corresponding		

EP 3 896 211 A1

ANNEX TO THE EUROPEAN SEARCH REPORT ON EUROPEAN PATENT APPLICATION NO.

EP 20 16 9727

5

This annex lists the patent family members relating to the patent documents cited in the above-mentioned European search report. The members are as contained in the European Patent Office EDP file on The European Patent Office is in no way liable for these particulars which are merely given for the purpose of information.

07-09-2020

10	Patent document cited in search report	Publication date	Patent family member(s)	Publication date
15	EP 3066406 A1	14-09-2016	CN 105705899 A EP 3066406 A1 ES 2659046 T3 PL 3066406 T3 WO 2015068092 A1	22-06-2016 14-09-2016 13-03-2018 31-07-2018 14-05-2015
20	EP 3617393 A1	04-03-2020	AU 2019208188 A1 CN 110872782 A EP 3617393 A1	19-03-2020 10-03-2020 04-03-2020
	CN 108691179 A	23-10-2018	NONE	
25				
30				
35				
40				
45				
50				
55 C	For more details about this annex : see 0	Official Journal of the Euro	pean Patent Office, No. 12/82	